# THE CATHOLIC UNIVERSITY OF AMERICA 

# Experimental Study of 3D Failure Surface for Cross-Anisotropic Sand Deposits During Stress Rotation 

## A DISSERTATION

Submitted to the Faculty of the<br>Department of Civil Engineering<br>School of Engineering<br>Of The Catholic University of America<br>In Partial Fulfillment of the Requirements<br>For the Degree<br>Doctor of Philosophy<br>©<br>Copyright<br>All Rights Reserved<br>By<br>Nina Maria Rodriguez<br>Washington, D.C.<br>2012

Experimental Study of 3D Failure Surface Location for Cross-Anisotropic Sand Deposits During Stress Rotation

Nina Maria Rodriguez, Ph.D.<br>Director: Poul V. Lade, Ph.D.

A torsion shear apparatus developed at The Catholic University of America, Washington, DC was used to conduct experiments on Fine Nevada Sand in order to study the effects of cross-anisotropy, shear banding and stress rotation under three independent principal stresses on stress-strain behavior. Drained torsion shear tests were performed with constant b-values, major principal stress directions and mean confining stress. A series of 18 drained tests using a true triaxial apparatus were conducted for comparison with certain torsion shear results. Drained triaxial tests with varying mean confining stress were also performed. Additional drained conventional triaxial tests were also performed to determine parameters needed for modeling. The 3D failure surface of Fine Nevada sand is presented. Shear banding patterns and analysis is presented for the true triaxial and torsion shear tests. The failure conditions from the torsion shear results are compared to a newly developed failure criterion. The effects of the intermediate principal stress on the failure surface are analyzed. Results clearly show the effects of cross-anisotropy in Fine Nevada Sand. This is seen in variation of friction angle with differing conditions and from strain analysis in different stress paths. Non-associated flow is observed and shear band inclinations in torsion shear specimens, although scattered, follow the Coulomb equation prediction. Data from these tests provide a solid foundation for future developments of cross-anisotropic constitutive models for frictional materials.

This dissertation by Nina Maria Rodriguez fulfills the dissertation requirement for the doctoral degree in Civil Engineering approved by Poul V. Lade, Ph.D. as Director, and by George Mavroeidis, Ph.D. and Biprodas Dutta, Ph.D. as Readers.

Poul V. Lade, Ph.D., Director

George Mavroeidis, Ph.D., Reader

Biprodas Dutta, Ph.D., Reader

This thesis is dedicated to my mom and dad for their unconditional love, faith and guidance my entire life. You have been by my side in all the decisions that I have made and have shown me the true definition of love and sacrifice. You both have provided me with the foundations for a successful life and I am eternally grateful. Thank you for instilling in me the values and confidence that makes me the person I am today. I love you with all my heart. Los quiero muchisimo.

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#### Abstract

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\begin{aligned}
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& \text { normal stress. }
\end{aligned}
$$

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## 1. Introduction

The study of cross-anisotropy in sands has been of great interest for the past several decades. The definition of anisotropy can be written as a material whose properties are directionally dependent. Experiments have been performed traditionally under two dimensional conditions in the laboratory. These tests, mainly done on triaxial apparatuses, only have control of two independent stresses in two different scenarios: First, in compression testing where $\sigma_{1}>\sigma_{2}=\sigma_{3}$ and second, in extension testing where $\sigma_{1}=\sigma_{2}>\sigma_{3}$. In order to fully study cross-anisotropy, the effects of $\sigma_{2}$ should be isolated and three independent stresses must be applied to the specimen. Certain apparatuses exist in the laboratory for these conditions to be produced. In this experimental study, the use of both a true triaxial apparatus and a torsion shear machine was required. The advantages and disadvantages of both will be discussed in further detail. Test results of a comprehensive study on Fine Nevada sand will be presented and a detailed analysis of the effects that crossanisotropy has on the failure surface of this sand under a variety of conditions is presented.

While the cross-anisotropic behavior of sand plays a crucial role in the strength behavior of soil, shear banding is also an area that should not be overlooked. This experimental study also presents, analyzes and compares shear bands that develop in the testing program for both apparatuses.

### 1.1. Overview of Research

The aim of this research is to perform a comprehensive study of the cross-anisotropic effects on the failure surface of Fine Nevada Sand. In order to study this cross-anisotropy, a testing program was devised consisting of true triaxial and torsion shear tests. Under the true triaxial apparatus testing program, the aim was to determine the cross-anisotropy present under different loading conditions in the three sectors of the octahedral plane. The torsion shear experimental program consisted of looking at the failure surface under constant b-value (in which $\left.\mathrm{b}=\left(\sigma_{2}-\sigma_{3}\right) /\left(\sigma_{1}-\sigma_{3}\right)\right)$, constant inclination of principal stresses and constant mean principal stress. With both sets of data, points that overlap between the true triaxial and torsion shear tests were to be compared and studied for similar trends and results. Study of shear bands as well as the angle at which they develop was also a major aim of this research program. By analyzing shear bands in two different apparatuses over a variety of conditions, greater knowledge from the experimental data collected would be attained.

### 1.2. Experimental Program

The experimental program is separated into three main research areas: Triaxial Tests, True Triaxial Tests and Torsion Shear Tests. All tests were performed on Fine Nevada Sand. Fine Nevada Sand was chosen in order to minimize membrane penetration effects. A true triaxial apparatus was used to shear tall prismatic specimens with dimensions 7.6 cm in width, 7.6 cm in length and 19 cm in height. A series of 10 triaxial compression tests and one
conventional extension test were performed with both horizontal and vertical bedding planes in order to attain parameters for an already developed cross-anisotropic failure criterion (Lade 2007). Specimens were sheared at a variety of constant confining pressures and the stress-strain behavior was analyzed. The parameters determined were used and compared to the experimental results. Fourteen tests were performed in a of series triaxial compression experiments with varying stress paths. This set of experiments in a triaxial apparatus studied the effects that stress paths may have on the strength behavior of the soil by changing the confining pressure to follow the predetermined stress path. A series of 18 tests were performed in a true triaxial apparatus (Lade 1978). Tests were performed in all three sectors of the octahedral plane. A horizontal loading machine was used to maintain constant, but different b-values at an increment of 0.25 from 0 to 1 for each of the three sectors. These tests produced the data needed to successfully plot all points along the octahedral plane and visually see the effects of cross-anisotropy for conditions of no stress rotation.

A torsion shear apparatus was used for an additional 22 tests performed on hollow cylinder specimens. These 22 tests were used to study effects of stress rotation when compared with existing data previously attained from the true triaxial tests. The tests were performed with changing inner and outer pressures and vertical load applied to the hollow cylinder specimens, allowing for different b-values (also at increments of 0.25 from 0 to 1 ) and varying alpha values (the angle of major principal stress with the vertical axis). Alpha values ranged from 0 to 90 degrees in 22.5 degree increments. In this manner, a 3D plot showing the failure surface from the tests in torsion shear of friction angle/stress ratio at various points
along constant b -values and constant alpha could be attained. Looking at the failure surface of Fine Nevada sand over a series of conditions allows for a final and complete picture of the effect that cross-anisotropy has on the failure surface of soil. In addition, strain analyses were performed to study cross-anisotropy.

Shear banding was also studied extensively and its effect on the failure of sand in the two different testing apparatuses. As will be shown in the chapters to come, shear banding can occur both in the hardening and softening regime. With shear banding occurring in the hardening regime, the strength of the soil may be affected. This was seen in most torsion shear tests. For true triaxial tests, shear banding occurred mainly in the softening regime, following the peak stress.

By testing the same soil in two different apparatuses, a complete comparison between the true triaxial and torsion shear results is presented. The stress-strain behavior, dilation angles and strain increment directions, as well as the shear band directions are analyzed and presented in the pages that follow.

## 2. Previous Studies

### 2.1. Background and Introduction

There are many engineering situations, anywhere from the application to shallow foundations to complicated earthquake engineering problems, where soil is subjected to multi-axial loads. These loads may create increases in shear stress levels and cause the rotation of principal stress directions. One common example is that of a shallow spread footing. In this situation, there can be rotations of up to $40^{\circ}$ about the vertical direction with increasing shear stresses. This creates a highly stressed zone below the footing that is subjected to a vertical static load. Figure 2.1 .1 shows stress states along a rupture area for a situation similar to what was just described. In naturally deposited soils, cross-anisotropy can be found. The axis of symmetry is most often aligned in the vertical direction of deposition. Because of this anisotropy, soils have deformations due to both the changes in magnitudes of the principal stresses as well as their orientation. There is a well-recognized need to investigate the response of soil deformation under stress changes where there is control of orientation and rotation of principal stress directions. Situations where one can isolate the influence of the rotation of principal stresses, while other parameters are held constant can be studied in the laboratory. This of course needs the help of laboratory devices that can control both the direction and magnitudes of the principal stresses. True triaxial as well as torsion shear devices have been used for many years in order to study these conditions. With these devices that allow for the creation of three principal stresses, it is possible to create continuous controlled increments and/or rotations of principal stresses in the vertical plane of the soil specimen that is being studied and then apply it to real world conditions.


Figure 2.1.1. Stress states along rupture surface (after O'Kelly and Naughton 2009).

In the following sections, this literature review will explain true triaxial and torsion shear devices, as well as past research concerning anisotropy, shear banding, and failure surfaces that have been researched, providing the necessary background for the experimental programs that have been performed.

### 2.2. True Triaxial Apparatuses

In order to conduct three-dimensional experiments on soils in the laboratory, the conventional triaxial setup cannot be used. Traditional triaxial cells only have two acting principal stresses. In cases of triaxial compression, the intermediate principal stress, $\sigma_{2}$ is equal to the minor principal stress, $\sigma_{3}$. For triaxial extension tests, the major principal stress, $\sigma_{1}$ is equal to the intermediate principal stress, $\sigma_{2}$. Due to this shortcoming of the conventional triaxial setup, the influence of the intermediate principal stress cannot be studied. In order to have three independent principal stresses, true triaxial devices must be
designed and used. In the pages that follow, the principles of true triaxial testing will be explained, as well as several designs that have been used in the past to study the behavior of soils with three independent principal stresses. It is important to note that true triaxial testing does not allow for rotation of principal stresses. However, as will be seen in later chapters, certain techniques can be employed on the specimen, such as freezing, to rotate the bedding planes of a specimen so that the major principal stress direction can be changed from $0^{\circ}$ to $90^{\circ}$.

## Principles of True Triaxial Apparatuses

Lade, in 1978, laid out some key criteria for successful testing in true triaxial apparatuses. First, it is required that the apparatus be able to handle strains in the three directions up to the point of failure. This will allow for uniform stress and strain conditions on the specimen and not create non-uniform stresses and strains, which may affect the data. Second, the apparatus also has to have the capability of creating uniform stresses on the specimen without creating any significant shear stresses to the surfaces of the specimen, which can also affect the results.

Arthur (1988) described certain features of cubical devices with flexible boundaries. As he noted, the cubical shape is very practical on which to apply principal stresses independently. He mentioned that two ideal boundary faces, flexible and rigid could be used for cubical testing. Cubical specimens can also be stressed through a combination of flexible and rigid
boundaries. Arthur (1988) summarizes the capabilities of certain cubical sample apparatuses in the following Table 2.2.1.

Early designs of certain types with flexible, rigid and mixed boundaries are worth describing. Figure 2.2 .1 shows different boundary conditions for true triaxial apparatuses. A classic example of a rigid boundary type is Hambly's design (Hambly 1969) (see Figure 2.2.2). Both Ko and Scott (1967) and Lomize and Kryzhanovsky (1967) used flexible type boundaries. Designs with mixed boundary types were used by Green (1971) and Lade and Duncan (1973). In these designs, a pair of vertical stiff platens replaced the flexible rubber bags where the horizontal deviator stress is applied. Lade's design will be described in detail in Section 2.2.3 as it is the true triaxial apparatus used in part of the experimental program.

Hambly's design, shown in the Figure 2.2.2 allows for unlimited boundary displacement in all three axes. In this design, all six faces of the cubical specimen are loaded rigid boundaries and can slide relative to each other. However, because of its complexity only limited stress ranges can be reached. Tests on this design are done with constant mean normal stress and constant b-value.

Table 2.2.1. Capabilities of cubical sample apparatus (after Arthur 1988).

| Cubical <br> Sample <br> Apparatus |  | Boundary Surfaces Condition | All Rigid | All Flexible | Mixed <br> Surfaces |
| :---: | :---: | :---: | :---: | :---: | :---: |
| Boundary control capabilities | Stress | Average normal stress | Yes | Yes | Yes |
|  |  | Average shear stress | Yes | Yes | Yes |
|  |  | Uniform normal stress | Restricted | Yes | Yes/no |
|  |  | Uniform shear stress | No | Yes | ```according io surfaces``` |
|  | Deformation | Uniform boundary displacement | Yes | Restricted | concerned |
|  |  | Uniform sample strain | Cannot be assumed | Cannot be assumed | Cannot be assumed |
| Study applications | Material type | Homogeneous strain hardening | Yes | Yes | Yes |
|  |  | Homogeneous strain hardening ( $\rightarrow$ softening) | Yes | Yes | Yes |
|  |  | Inhomogeneous sample or material | $\begin{aligned} & \text { Restricted } \\ & \mathrm{R}^{a} \end{aligned}$ | $\begin{gathered} \text { Yes } \\ \mathrm{R} \end{gathered}$ | Restricted R |
|  | Stressstrain behavior | Drained or undrained shear | Either | Either | Either |
|  |  | Independent variation of principal stresses | Yes | Yes | Yes |
|  |  | Principal stress direction rotation | Very restricted | Planar restriction only | According to surface |
|  |  | Post peak strain (after rupture layer formed) | $\begin{gathered} \text { Restricted } \\ R \end{gathered}$ | $\begin{aligned} & \text { Restricted } \\ & \text { R } \end{aligned}$ | $\begin{aligned} & \text { Restricted } \\ & R \end{aligned}$ |
|  |  | $\begin{array}{cc}\text { Exceptional } & \text { High } \\ \text { stress levels } & \text { Low }\end{array}$ | Yes | Yes | Yes |
|  |  |  | Difficult | Yes | Difficult |
|  | Failure | Shear strength | R | R | R |
|  |  | measurement | Yes | Yes | Yes |
|  |  | Rupture layer formation (bifurcation phenomena) | $\begin{gathered} \text { Restricted } \\ R \end{gathered}$ | $\begin{gathered} \text { Yes } \\ \mathrm{R} \end{gathered}$ | $\begin{aligned} & \text { Yes } \\ & \mathrm{R} \end{aligned}$ |

" $\mathrm{R}=$ Radiography needed.


Figure 2.2.1. Types of boundary conditions employed in different true triaxial apparatus for soil testing (after Lade 2006).


Figure 2.2.2. Hambly's rigid boundary true triaxial concept (after Arthur 1988).

For tests with high stress levels, Meier et al. (1985) designed a flexible boundary device, as seen in Figure 2.2.3, where the stresses were transmitted through a vinyl membrane and then through a pad of polyurethane, followed by another pad of material which depended on the material being tested. Yamada and Ishihara (1979) designed an apparatus based on Ko and Scott's design (1967) for intermediate stress levels where they were able to study the effects of anisotropy. These types of apparatuses where all six faces of the cubical specimen are loaded by flexible boundaries are usually used to perform tests with constant mean stress and constant b-values. Yamada and Ishihara's design, shown in Figure 2.2.4. may create very high stresses along the edges if the strain of the specimen drives its edges into the frame's edge. The rigidity of the frame allows for the possibility of unlimited stress concentrations. Flexible membranes may also create very low stresses along the edges if the pressure membrane does not exert the right amount of pressure on the sample edge.

Directional shear devices have been designed to apply uniform normal and shear stresses to opposite sides of the specimen normal to the plain of strain. Varying $\sigma_{a}, \sigma_{b}, \tau_{a}$, and $\tau_{\mathrm{b}}$ controls the direction of the principal stress relative to the plane of strain. This can be seen in the apparatus developed by Arthur et al. (1977) in Figure 2.2.5.


Figure 2.2.3. High stress flexible boundary true triaxial (after Meier et al. 1985).


Figure 2.2.4. Flexible boundary true triaxial (after Yamada and Ishihara 1979).


Figure 2.2.5. Directional Shear Cell (after Arthur et al. 1977).

## True Triaxial Design used in Experimentation

Lade's cubical triaxial apparatus (1978) was designed so that two different horizontal stresses could be applied to the specimen. One is applied via the confining pressure, the second through a horizontal deviator stress via a horizontal loading system. The vertical load applied to the specimen's cap and base is transferred through a piston. The setup is in a triaxial chamber. In this chamber, the cell pressure acts as the minor principal stress, $\sigma_{3}$. The deviator stress created by the vertical load applied to the specimen and the cell pressure act as
the major principal stress, $\sigma_{1}$. The horizontal deviator stress and the cell pressure create the intermediate principal stress, $\sigma_{2}$. A schematic of the entire assembly is seen in Figure 2.2.6.


Figure 2.2.6. Cubical Triaxial Apparatus (after Lade 1978).

A brief description of the setup is as follows. More details can be found in Lade and Duncan (1973) and Lade (1978) where the entire assembly is described and illustrated. Two loading plates are connected at opposite sides of the specimen. One side is connected to an oil-filled pressure cylinder which, when compressed, allows the plates to move freely on the rails that the bottom wheels sit on. The compressibility of the plates creates an independent straining in the horizontal direction. The plates are made to be compressible in the vertical direction by alternating steel and pre-stressed balsa wood laminae. The balsa wood is pre-compressed and presoaked in water in order to decrease the strength in the directions perpendicular to the
fibers. This ensures that there is no interference with the cap and base during compression of the specimen. Lubricated ends made of $0.03-\mathrm{cm}$ rubber sheets, which are separated by a thin layer of silicon grease are also placed on the specimen base and cap, as well as on the sides of the horizontal loading plates to provide frictionless surfaces and avoid significant shear stresses. A beryllium copper load cell is embedded in the top cap, which has strain gages connected in a full bridge configuration. The wires lead through the triaxial cell and are connected to a measurement device. When loaded, the flat edge of the piston applies load to a steel ball, which sits in a ball socket on the load cell. Horizontal LVDTs are set up on the sides of the horizontal plates to measure strain in the horizontal direction. A vertical dial gage is set up outside of the triaxial cell to measure vertical strain. The top cap and base have filter stones and drainage lines to measure volume change or pore pressure. A Lucite cell is placed around the setup and six rods hold the bottom plate, Lucite cell and top plate together. Figure 2.2.7. shows the schematic of the stress control system. With this system, the $b$-value can be held constant during shearing.


Figure 2.2.7. Schematic diagram of stress control system (after Lade 1978).

### 2.3. Torsion Shear Devices

Broms and Casbarian (1965) conducted the first study of principal stress rotation effects on the strength of kaolinite clay using a hollow cylinder apparatus. They showed that the continuous rotation of the principal stress axes increases the rate of pore water pressure generation and at the same time, reduces the undrained strength of cohesive soil. In 1975, Lade designed and presented the main features of a torsion shear apparatus in order to study the shear-dilatancy effect on the stress-strain characteristics of cohesionless soil. Lade also discussed in detail the torsion shear apparatus used for his soil testing in 1981. In 1983, Hight et al. discussed in detail the development of a new hollow cylinder apparatus, as well as the principles of its operation. In 1988, Shibuya developed a servo system for the traditional Imperial College hollow cylinder apparatus. Table 2.3.1, taken from Hight et al. (1983), lists certain torsion shear apparatus and explains the tests that were performed. A more complete and recent list taken from Tastan (2009) can be found in Appendix E.

Table 2.3.1. Summary of earlier hollow cylinder apparatuses (after Hight et al. 1983).


The biggest advantage of the torsion shear test compared to other geotechnical testing apparatuses is that it allows for the inclination of the major principal stress in any direction while complimentary shear stresses are applied to the specimen. The angle between the major principal stress and vertical, $\alpha$, is tied to the intermediate principal stress parameter $b$, where

$$
\begin{equation*}
b=\frac{\sigma_{2}-\sigma_{3}}{\sigma_{1}-\sigma_{3}} \tag{Eq. 2.3.1}
\end{equation*}
$$

b indicates the relative magnitude of the intermediate principal stress $\sigma_{2} . \sigma_{1}$ and $\sigma_{3}$ are the major and minor principal stresses, respectively. For the same inner and outer pressures on the hollow cylinder specimens, the relationship between $b$ and $\alpha$ can be denoted by Eq. 2.3.2. $b=\sin ^{2} \alpha$
where $\alpha$ is the inclination of major principal stress (Lade et al. 2008).

When the inside and outside pressures are the same in a torsion shear test, the hollow specimen experiences a plane stress state. To get to this state, it is necessary to have the stresses and strains distributed uniformly throughout the specimen. This uniformity can be achieved by having appropriate dimensions for the specimen. Much advancement has been made to the torsion shear apparatus. However, the principles and overall design to date has remained the same.

## Principles and Design of Torsion Shear Apparatuses

In principle, the magnitude and direction of the major and minor principal stresses, as well as the intermediate principal stress can be controlled in a torsion shear apparatus. These are controlled under combined axial load, torque, and internal and external radial pressures. Under stress-controlled conditions, this apparatus enables controlled rotation of the principal stress directions on a surface to be attained.

A torsion shear apparatus is made up of basically four major units: 1) a specimen pressure cell, 2) a torque loading unit, 3) an axial loading unit, and 4) a data collection and control unit. Different apparatuses have variations on these basic units, but they all operate on the same principles and basic design.

## Principles on Stresses and Strains in Torsion Shear

In the laboratory, it is very difficult to reproduce controlled changes in direction and magnitude of the principal stresses. In most laboratory testing equipment, the principal stresses are fixed. One can only switch and interchange the directions. However, with the torsion shear apparatus, principal stresses can rotate and the specimens can be subjected to axial load $(\mathrm{W})$ and torque $\left(\mathrm{M}_{\mathrm{t}}\right)$ about a central vertical axis. They can also be subjected to external $\left(p_{o}\right)$ and internal $\left(p_{i}\right)$ radial pressures.

The torque applied to the specimen creates shear stresses $\left(\tau_{\theta z}\right)$ and $\left(\tau_{z \theta}\right)$. When the internal and external pressures are different, a gradient of radial stress $\left(\sigma_{\mathrm{r}}\right)$ is established across the cylinder wall. The principal stress rotation on a three dimensional element and on a hollow cylindrical specimen as well as the stresses that act on the specimen during loading can also be seen in the Figures 2.3.1 and 2.3.2.

When performing experiments on hollow cylindrical specimens, $\left(\mathrm{p}_{\mathrm{o}}\right)$ and $\left(\mathrm{p}_{\mathrm{i}}\right)$ should act through flexible membranes. This ensures that there are no shear stresses acting on the vertical boundaries. If end restrained is neglected (there are several ways to make sure that end restraint does not interfere with the sample, which will be discussed in a later section), no shear stresses exist on circumferential surfaces through the membrane walls. The shear stress due to torque produces reorientation of the principal stress directions as well as a stress state with three unequal principal stresses. If shear stresses are not applied to the specimen, the confining radial pressure, $\sigma_{\mathrm{r}}$, becomes the minor principal stress $\sigma_{3}$ in a compression test, or the major principal stress $\sigma_{1}$ in an extension test. In this case, $\sigma_{\mathrm{r}}$ is always a principal stress. With torsion shear stresses, $\sigma_{r}$ becomes the intermediate principal stress, $\sigma_{2}$. In order to determine the remaining principal stresses, it is possible to resolve the other stresses and shear stresses, $\left(\sigma_{\mathrm{o}}\right),\left(\sigma_{\mathrm{z}}\right),\left(\tau_{\mathrm{oz}}\right)$ and $\left(\tau_{\mathrm{zo}}\right)$. The magnitudes of the three principal stresses are determined by the forces that are applied, as well as the internal and external pressures and the geometry of the specimen being tested.


Figure 2.3.1. Principal stress rotation on an element (after Hight et al. 1983).


Figure 2.3.2. Idealized stress conditions in a hollow cylindrical element subject to axial load, W, torque, Mt, internal pressure, $p_{i}$ and external pressure, $p_{o}$. (a) Hollow cylinder sample; (b) stresses on an element in the wall; (c) principal stresses on an element in the wall; (d) Mohr circle representation of stress in the wall (after Hight et al. 1983).

In general, the stresses along the specimen wall will not all be uniform. Therefore, it is necessary to work in terms of average stresses. These are represented by: $\left(\sigma_{\theta}\right)_{\text {avg }},\left(\sigma_{z}\right)_{\text {avg }}$, $\left(\sigma_{\mathrm{r}}\right)_{\text {avg }}$, and $\left(\tau_{\theta z}\right)_{\text {avg. }}$. The following equations for average stresses and strains (taken from Hight et al., 1983) are shown below in Table 2.3.2. It is important to note that Hight et al.
(1983) along with many other researchers have used these equations. The average stress values, $\left(\sigma_{\theta}\right)_{\text {avg }}$ and $\left(\sigma_{z}\right)_{\text {avg }}$ are based on force equilibrium conditions. However, the average strain values $\left(\varepsilon_{\theta z}\right)_{\text {avg }}$ and $\left(\gamma_{z}\right)_{\text {avg }}$ are based on strain compatibility only. These equations do not take into account the constitutive law of the material. The equation for $\left(\sigma_{\mathrm{r}}\right)_{\text {avg }}$ is based on linear elastic stress distribution. For $\left(\varepsilon_{\mathrm{r}}\right)_{\text {avg }}$ and $\left(\varepsilon_{\theta}\right)_{\text {avg }}$, a linear variation of radial displacements across the wall is assumed. For $\left(\tau_{z \theta}\right)_{\text {avg }}$, a uniform stress distribution is assumed.

Table 2.3.2. Definitions of average stresses and strains (after Hight et al. 1983).


As stated, because normal and shear stresses vary along the thickness of a specimen's wall, average values of $\left(\sigma_{\theta}\right)$ and $\left(\sigma_{z}\right)$ are calculated. However, since a thick cylinder (where
thickness/radius $>0.1$ ) specimen is subjected to deformation, looking at equilibrium alone is not enough. It is necessary also to know the constitutive law of the material to analyze deformations.

Using linearly elastic behavior, $\left(\sigma_{\theta}\right)$ and $\left(\sigma_{r}\right)$ can be calculated (see Table 2.3.3). $\left(\sigma_{z}\right)$ can be calculated in a straight forward manner by using the vertical load across the cross section of the specimen. Corrections for membrane stiffness, piston friction and weight of the soil are also part of the calculation for $\left(\sigma_{z}\right)$.

Sayao and Vaid (1991) considered the specimen as a single element deforming as a right cylinder. They developed expressions that considered stress components assuming a linear elastic isotropic material. However, $\left(\sigma_{z}\right)$ is not dependent on the material's constitutive law. Wijewickreme (1990) found that considerations of nonlinearity in soil behavior do not affect the average stresses reached when using a linear elastic assumption. In order to get $\left(\sigma_{r}\right)$, $\left(\sigma_{\theta}\right)_{\text {avg }}$ and $\left(\tau_{z \theta}\right)$, it is necessary to average over the entire volume of the specimen. Sayao and Vaid (1991) compared the equations of both Hight et al. (1983) and Miura el al. (1986). Hight et al. (1983) and Miura et al. (1986) had averaged across the specimen wall instead of taking into account the volume of the specimen. Averaging across the volume takes into account the curvature of the wall. Hight et al. (1983) and Miura et al. (1986) also assumed a plastic constitutive law when calculating $\left(\tau_{\mathrm{z} \theta}\right)$. They found that the differences were minor,
usually less than $2 \%$. However, they stated that for consistency, it is necessary to assume one constitutive law when analyzing stresses and strains and not elastic for certain stresses and plastic for others as done as Hight et al. (1983). The strains were analyzed assuming a linear variation of displacement across the specimen wall, just like Hight et al. (1983).

Table 2.3.3. Stresses on a thick cylinder subject to internal and external pressure (a) cross section, (b) element of unit length, (c) derived stresses using linear elastic behavior (after Morshedian (1992).

(c)

$$
\begin{aligned}
& \sigma_{\pi}=\frac{p_{i} r_{i}^{2}-p_{0} r_{0}^{2}}{r_{0}^{2}-r_{j}^{2}}-\frac{\left(p_{i}-p_{0}\right) r_{0}^{2} r_{i}^{2}}{r^{2}\left(r_{0}^{2}-r_{i}^{2}\right)} \\
& \sigma_{\hat{\theta}}=\frac{p_{i} r_{i}^{2}-p_{0} r_{0}^{2}}{r_{0}^{2}-r_{i}^{2}}+\frac{\left(p_{i}-p_{0}\right) r_{0}^{2} r_{i}^{2}}{r^{2}\left(r_{0}^{2}-r_{i}^{2}\right)}
\end{aligned}
$$

The equations they developed are shown in Table 2.3.4. The Lamé equations for calculating mean values of the non-zero stress components that are induced across a specimen wall thickness were derived using equilibrium considerations assuming an isotropic, linear-elastic response.

In the case where the internal and external pressure are equal ( $p_{i}=p_{o}=p$ ), one can assume that $\left(\sigma_{\theta}\right)_{\text {avg }}$ and $\left(\sigma_{r}\right)_{\text {avg }}$ are equal to p if the hollow cylinder specimen is sufficiently tall. Any changes that occur to $\left(\sigma_{\theta}\right)_{\text {avg }},\left(\sigma_{z}\right)_{\text {avg }},\left(\sigma_{r}\right)_{\text {avg }}$, and $\left(\tau_{\theta z}\right)_{\text {avg }}$ create changes in the magnitude and orientation of $\left(\sigma_{1}\right)_{\text {avg }}$ and $\left(\sigma_{3}\right)_{\text {avg }}$. They create simultaneous changes in $\left(\sigma_{2}\right)_{\text {avg }}$, in relation to $\left(\sigma_{1}\right)_{\text {avg }}$ and $\left(\sigma_{3}\right)_{\text {avg. }}$. This relationship can be seen through the following equation for b , the parameter that indicates the relative magnitude of intermediate principal stress.
$b=\left(\frac{\sigma_{2 \text { avg }}-\sigma_{3 \text { avg }}}{\sigma_{\text {lavg }}-\sigma_{3 \text { avg }}}\right)$

Table 2.3.4. Expressions used for calculating average stresses and strains developed by Sayao and Vaid (1991).

$$
\begin{aligned}
& \sigma_{z}=\frac{F_{z}+\pi\left(P_{e} R_{e}{ }^{2}-P_{\imath} R_{i}{ }^{2}\right)}{\pi\left(R_{e}{ }^{2}-R_{\imath}{ }^{2}\right)} \\
& \sigma_{\sigma_{\theta}}^{\sigma_{r}}=\frac{P_{e} R_{e}{ }^{2}-P_{\imath} R_{i}{ }^{2}}{R_{e}{ }^{2}-R_{\imath}{ }^{2}} \\
& \mp \frac{2\left(P_{e}-P_{i}\right) R_{e}{ }^{2} R_{i}{ }^{2} \ln \left(R_{e} / R_{i}\right)}{\left(R_{e}{ }^{2}-R_{i}{ }^{2}\right)} \\
& \tau_{z \theta}=\frac{4 T_{h}\left(R_{e}{ }^{3}-R_{i}{ }^{3}\right)}{3 \pi\left(R_{e}{ }^{4}-R_{i}^{4}\right)\left(R_{e}{ }^{2}-R_{\imath}{ }^{2}\right)} \\
& \varepsilon_{z}=\frac{-\Delta H I}{H} \\
& \varepsilon_{r}=\frac{-\left(\Delta R_{e}-\Delta R_{\imath}\right)}{R_{e}-R_{\imath}} \\
& \varepsilon_{\theta}=\frac{-\left(\Delta R_{e}+\Delta R_{i}\right)}{R_{e}+R_{i}} \\
& \gamma_{z \theta}=\frac{2 \Delta \theta\left(R_{e}{ }^{3}-R_{i}{ }^{3}\right)}{3 H\left(R_{e}{ }^{2}-R_{i}{ }^{2}\right)}
\end{aligned}
$$

As stated previously in Equation 2.3.2, the b stress parameter can also be described with relation to $(\alpha)$, which is the orientation of the major principal stress $\left(\sigma_{1}\right)$ to the vertical by the equation, $b=\sin ^{2}(\alpha)$ when both internal and external pressures are the same. When the internal and external pressures are not equal, then both $b$ and $\alpha$ can be independently controlled. The three stresses $\left(\sigma_{1}\right),\left(\sigma_{2}\right)$ and $\left(\sigma_{3}\right)$, along with $(\alpha)$, can all be independently controlled.

Other stress-related parameters, besides $b$, can be determined from the four stress components induced by the specimen $\left(\sigma_{\theta}\right),\left(\sigma_{z}\right),\left(\sigma_{r}\right)$, and $\left(\tau_{\theta z}\right)$. These are:
$p=\frac{\sigma_{1}+\sigma_{2}+\sigma_{3}}{3}$
$q=\left(\sigma_{1}-\sigma_{3}\right)$
$\alpha=\frac{1}{2} \tan ^{-1}\left(\frac{2 \tau_{z \theta}}{\sigma_{z}-\sigma_{\theta}}\right)$
where p is the mean normal stress, q is the deviator stress and as mentioned before, $\alpha$ is the angle between the major principal stress direction and vertical. The effects of $\alpha$ on material behavior are direct results of cross-anisotropy, which will be discussed in detail in chapters.

## Design of a Torsion Shear Apparatus

A torsion shear apparatus can be used to study the behavior of soils while rotating the principal stress directions. This apparatus allows for the individual control of vertical normal stress, cell pressure, and the applied shear stress. When designing the specimen height, careful attention must be paid to ensure that shear bands can fully develop and that the end restraint does not cause effects on the specimen behavior.

Through the use of cap and base rings, the vertical normal stress and shear stress are transferred to the specimen. The loading system and other components can be located under
the loading table or elsewhere so as not to interfere with the specimen when being prepared and tested.

Careful attention should be paid to ensure that the apparatus is capable of accommodating large normal and shear strains so that once the specimen reaches failure, there is a minimal amount of induced non-uniformity in the stress and strain distributions. When transferring the shear stresses to the specimen, no slippage can occur between the specimen and the stress application mechanisms.

Key parts of the torsion shear apparatus include cap and base rings (with full friction surfaces to transfer shear stresses), a membrane, forming jackets, and draining lines (which allow for measurement of the volume change of the specimen). The loading system and torque loading system can differ from apparatus to apparatus. This is a design choice and can vary. Torque and a vertical load (which can be in some cases supplied by an oil-filled pressure cylinder) can be transferred through the center shaft to the base plate of the apparatus. In Lade's (1981) design, there were four pressure cylinders that allowed for torsion shear stresses in both clockwise and counterclockwise directions. A schematic of Lade's design, as well as Broms and Casabrian's design that used a turntable to shear the specimen are shown in the Figures 2.3.3 and 2.3.4.


Figure 2.3.3. Torsion Shear Apparatus (after Lade 1981).


Figure 2.3.4. Test arrangement (after Broms and Casabrian 1965).

In order to measure the vertical and shear deformations, linear motion transducers can be placed outside the cell as shown in Figure 2.3.3. The coil of a vertical linear motion transducer can be set up to measure vertical deformations. Mounted on another rod above the cell, a shear deformation transducer can be placed.

A grid of vertical and horizontal lines can be drawn on the outside surface of the hollow cylinder specimen to allow observations of shear strains. Horizontal deformations can be measured with clip gages and/or LVDTs. Figure 2.3.5 shows another example of a torsion shear apparatus built at the University College Dublin.


Figure 2.3.5. UCD hollow cylinder torsional apparatus (after O'Kelly and Naughton 2005).

A negative aspect of the torsion shear apparatus is that the tangential, horizontal normal stress in the cylinder wall $\left(\sigma_{\theta}\right)$ cannot be measured. However, during isotropic compression, $\left(\sigma_{\theta}\right)$ can be assumed to be the same as the cell pressure.

In summary, the torsion shear apparatus allows for the ability to control and change applied vertical normal stresses, cell pressures, and shear stresses. The behavior of soils can be examined by applying certain techniques with this apparatus, which produces reasonable
uniform stress states. Although designs may be different, they all work on the same principles as well as with the basic components described above.

## Influence of Specimen Geometry

Torsion shear equipment offers the benefit that no shear stresses are produced on the vertical surfaces of the specimen, while complimentary shear stresses, $\tau_{\theta z}$ are automatically generated in the specimen, and large and fairly uniform shear strains can be produced. Separate control of the vertical normal stress, the confining pressure, and the shear stress makes it possible to create various initial states of stress before the specimen is sheared. However, there are also limitations. Non-uniformity of stress and strain distributions may develop, especially in specimens with inappropriate dimensions.

Non-uniformity for a given stress state depends on the specimen's dimensions. Wall thickness, diameter and height are components of hollow cylinders. With regards to wall thickness, stress non-uniformity increases with wall thickness for a given average specimen radius where

$$
\begin{equation*}
r_{\text {avg }}=\frac{\left(r_{i}-r_{o}\right)}{2} \tag{Eq. 2.3.7}
\end{equation*}
$$

where $r_{i}$ is the inner radius and $r_{o}$ is the outer radius.

Figure 2.3.6 (taken from Sayao and Vaid 1991) shows the effect of wall thickness on nonuniformity coefficients for $\mathrm{b}=0, \alpha=45^{\circ}$ at $\mathrm{r}=2$ and 3 .


Figure 2.3.6. Effect of wall thickness non-uniformity coefficients (after Sayao and Vaid 1991).

There is a minimum thickness that specimens should have to minimize non-uniformities. Hight et al. (1983) stated two considerations. First, consideration to include a large enough number of sand grains across the wall in order to ensure a uniform sand density must be made. Second, it is important to consider the need to minimize the relative significance of potential volume change corrections due to membrane penetration. In practice, a wall thickness of 20 to 26 mm is considered to be applicable for medium and fine sands. When
considering the radii dimensions, a very large inner radius may not be practical in terms of stress path control.

Sayao and Vaid (1991) recommended a $\mathrm{r}_{\mathrm{i}} / \mathrm{r}_{\mathrm{e}}$ within 0.65 and 0.82 for specimens with a wall thickness within the range of $20-26 \mathrm{~mm}$. Specimen height must also be considered when looking at specimen geometry. Radial frictional restraint at the boundaries of the specimen causes stress non-uniformities as well as specimen curvature in the vertical direction. If the height, H to diameter, 2 R ratio is within 1.8 to 2.2 , then these non-uniformities are considered to be minimal. Any additional techniques to reduce friction on the end platens will help reduce radial friction. Lade (1981) recommends the specimen to have a height of 40 cm and an average diameter of 20 cm , with a wall thickness of 2 cm . Many experiments conducted on this size specimen have been successful and have shown that this size can provide high quality.

Table 2.3.5 shows a list of torsion shear apparatuses up until 1988. Figure 2.3.7 shows the specimen geometry for the apparatuses listed in Table 2.3.4. The boxed-in devices are those that are within the recommended dimensions for specimens (wall thickness: $\mathrm{r}_{0}-\mathrm{r}_{\mathrm{i}}=20-26 \mathrm{~mm}$, inner radius: $0.65 \leq \mathrm{r}_{\mathrm{i}} / \mathrm{r}_{\mathrm{o}} \leq 0.82$, and height: $\left.1.8 \leq(\mathrm{H} / 2) * \mathrm{r}_{\mathrm{o}} \leq 2.2\right)$.

The topic of stress non-uniformities has been an important consideration by many researchers when dealing with torsion shear apparatuses. Similar to other apparatuses used in geotechnical testing, stress non-uniformities can develop near specimen ends due to the frictional restraint of stiffness at the ends. In tests where torque is applied or where the internal and external pressures are different, the wall curvature also plays a role in producing stress non-uniformities. Sayao and Vaid (1991) and Wijewickreme and Vaid (1991) showed that the stress non-uniformities that occur are related to specimen size, and they decrease as the wall thickness is decreased and as the inner radius is increased.

Naughton and O'Kelly (2007) studied stress non-uniformities in hollow torsion shear specimens, keeping the mean principal stress, b-value and alpha constant. The hollow cylinder specimen size had an inner radius of 35.5 mm , an outer radius of 50.0 mm and a height of 200 mm . They increased the stress ratio and computed the corresponding stress nonuniformities. They found that for triaxial compression and triaxial extension, the stress distributions were completely uniform. However, for $(b-v a l u e, ~ \alpha)=\left(1,0^{\circ}\right)$ and $\left(0,90^{\circ}\right)$, there were significant stress non-uniformities. For all conditions with $\alpha=45^{\circ}$, there were significant stress non-uniformities. These locations would be where the ratio of inner to outer pressures is the maximum (for $b=1, \alpha=0^{\circ}$ ) or minimum (for $b=0, \alpha=90^{\circ}$ ). The circumferential stress (due to the applied torque) is at its maximum value for $\alpha=45^{\circ}$ values. Figure 2.3.8 shows the regions where serious stress non-uniformities can develop.

Table 2.3.5. Stress path devices using hollow cylinder specimens (after Sayao and Vaid 1991).

| No. | Reference |  | Institution |  | $\underset{(\mathrm{mm})}{\text { Specimen Dimensions }}$ |  |  | Soll <br> Type | Control Restrictions | Applications |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  |  |  |  |  | H | $R_{\text {e }}$ | $R_{i}$ |  |  |  |
| 1 | Cooling \& Smith | 1936 | Building Research Station | (ENG) | 19-38 | 50.8 | 41.3 | Clay | $F_{s}=P_{t}=P_{i}=0$ | Undrained shear strength |
| 2 | Norton | 1938 | M.I.T. | (USA) | 50.8 | 11.1 | 7.9 | Clay | $F_{s}=P_{t}=P_{t}=0$ | Torsional deformability ceramic clays |
| 3 | Geuze \& Kie | 1953 | S.M.Laboratory, Delft | (HOL) | 80.0 | 19.0 | 13.0 | Clay | $F_{s}=P_{t}=P_{t}=0$ | Underained creep |
| 4 | Kirkpatrick | 1957 | University of Glasgow | (SCOT) | 152.4 | 50.8 | 31.8 | Sand | $F_{1}=T_{n}=0$ | $\sigma_{2}$ effect on failure condition |
| 5 | Haythornthwaite | 1960 | Brown University | (USA) | ? | ? | ? | Silt | $P_{t}=P_{i}$ | $\sigma_{2}$ effect on failure condition |
| 6 | Whitman \& Luscher | 1962 | M. I. T. | (USA) | 76 or 127 | 25 or 19 | 12.7 | Sand | $T_{b}=0 ; t_{s}=0$ | Soil-structure interaction at failure |
| 7 | Wu et al. | 1963 | Michigan State University | (USA) | 127.0 | 50.8 | 38.1 | C\&S | $T_{n}=0$ | $\sigma_{2}$ effect on failure condition |
| 8 | Broms \& Ratnam | 1963 | Cornell Univeraity | (USA) | 114.3 | 76.2 | 38.1 | Clay | $T_{n}=0$ | 3-D consol effects on strength |
| 9 | Broms \& Casbarian | 1965 | Cornell University | (USA) | 254.0 | 63.5 | 38.1 | Clay | - | $\sigma_{2}$ and $\alpha$ effects on strength |
| 10 | Broms \& Jamal | 1965 | Cornell University | (USA) | 304.8 | 76.2 | 38.1 | Sand | $T_{n}=0$ | Valldity of $\sigma_{r}=\sigma_{g}$ assumption |
| 11 | Esrig \& Bemben | 1965 | Cornell University | (USA) | 203.2 | 50.8 | 38.1 | Sand | $T_{A}=0$ | $\sigma_{2}$ and $t_{2}$ effects on strength |
| 12 | Suklje \& Drnovselc | 1965 | University of Ljubljana | (YUG) | 80.0 | 32.0 | 20.0 | Clay | $T_{5}=0 ; \sigma_{8}=0$ | Deformability under plane stress |
| 13 | Jamal | 1966 | Cornell University | (USA) | 203.2 | 50.8 | 13-38 | Sand | $T_{s}=0 ; P_{s}=P_{6}$ | Wall thickness effect on strength |
| 14 | Saada \& Baah | 1967 | Case W. R. University | (USA) | 151.1 | 35.1 | 25.4 | Clay | $P_{s}=P_{s}$ | Influence of anisotropy |
| 15 | Proctor | 1967 | University of Manchester | (ENG) | 152.4 | 50.8 | 19.1 | Sand | $T_{A}=0$ | Drained shear strength |
| 16 | Lomize et al. | 1969 | Civil Engineering Institute | (USSR) | 180.0 | 155.0 | 125.0 | Clay | $P_{s}=P_{i}$ | Drained creep under 3-D stress state |
| 17 | Frydman et al. | 1971 | Israel Institute of Tech. | (ISRL) | 203.2 | 50.8 | 25.4 | Sand | $T_{k}=0$ | End restraint ; membrane penetration |
| 18 | Drnevich | 1972 | University of Kentucky | (USA) | 100.0 | 25.0 | 20.0 | Sand | $P_{t}=P_{i}$ | Torsional resonant column tests |
| 19 | Arnold \& Mitchell | 1973 | University of Adelaide | (AUS) | 142.0 | 76.0 | 51.0 | Sand | $T_{k}=0$ | 3-D stress effect on strength |
| 20 | Ishibashi \& Sherif | 1974 | University of Washington | (USA) | 13 to 25 | 50.8 | 26.4 | Sand | $P_{\text {c }}=P_{i}$ | Liquefaction characteristics |
| 21 | Tong | 1975 | University of Waterloo | (CAN) | 203.2 | 50.8 | 31.8 | Sand | - | Yield and failure criteria |
| 22 | Lade | 1975 | U. C. L. A. | (USA) | 50.0 | 110.0 | 90.0 | Sand | $P_{s}=P_{\text {i }}$ | $\alpha$ effect on stress-strain |
| 23 | Iwasaki et al. | 1978 | Inst. Ind. Science | (JAP) | 100.0 | 50.0 | 30.0 | Sand | $P_{s}=P_{i}$ | Stress-strain for $\gamma>10^{-2} \%$ |
| 24 | Lade | 1581 | U.C.L.A. | (USA) | 400.0 | 110.0 | 90.0 | Sand | $P_{s}=P_{i}$ | Influence of specimen's height |
| 25 | Dusseault | 1981 | University of Alberta | (CAN) | 200-240 | 50.8 | 25.4 | Sand | $T_{k}=0$ | Tunelling and pressuremeter paths |
| 26 | Fukushima \& Tatsuoka | 1982 | Inst. Ind. Science | (JAP) | 200.0 | 50.0 | 30.0 | Sand | $P_{4}=P_{i}$ | Deformation \& strength behaviour |
| 27 | Symes et al. | 1982 | Imperial College | (ENG) | 254.0 | 127.5 | 101.5 | Sand | - | $b$ and a effects on strain response |
| 28 | Ishihara \& Towhata | 1982 | University of Tokyo | (JAP) | 104.0 | 50.0 | 30.0 | Sand | $P_{t}=P_{i}$ | $\alpha$ effects ; liquef. characteristics |
| 29 | Ishibashi et al. | 1985 | Cornell University | (USA) | 142.0 | 35.5 | 25.4 | Sand | $P_{t}=P_{i}$ | Llquefaction characteristics |
| 30 | Miura et al. | 1986 | Hokkaido University | (JAP) | 200.0 | 50.0 | 30.0 | Sand | - | $\alpha$ effect on stress-strain \& strength |
| 31 | Alarcon et al. | 1986 | Purdue University | (USA) | 203.0 | 35.5 | 19.0 | Sand | $P_{t}=P_{i}$ | Stress-strain for $\gamma>10-2 \%$ |
| 32 | Anderson et al. | 1988 | University of sheffield | (ENG) | 150.0 | 75.0 | 12.5 | Clay | $T_{A}=0$ | Pressuremeter paths ; undrained creep |
| 33 | Chen et al. | 1988 | Cornell University | (USA) | 193.0 | 51.0 | 35.5 | Sand | $P_{t}=P_{i}$ | Dynamic shear mod. of glass spheres |
| 34 | Sayao \& Vaid | 1988 | University of B.C. | (CAN) | 304.8 | 76.2 | 50.8 | Sand | - | Effects of $a, R, b \& \sigma_{n^{\prime}}$ on strain resp. |



Figure 2.3.7. Specimen geometry of reported holly cylinder devices (after Sayao and Vaid 1991).


Figure 2.3.8 Areas where serious stress non-uniformities may arise (after Naughton and O'Kelly 2007).

### 2.4. Cross-Anisotropy

## Introduction to Anisotropy

Anisotropy in granular materials can be caused by certain reasons. The arrangement of particles and the directions of the grains major axes on the bedding plane can cause anisotropy. Anisotropy can be inherent and be caused by spatial distribution of contacts and contact forces. Anisotropy can also be induced by using carefully designed stress paths and by controlling the testing procedure. Doing so, the influence of the stress path on the strength and deformability of the soil can be explained.

When properties are the same in all directions within the horizontal plane, yet different from those in the vertical direction (the direction of deposition), soil behavior can be referred to as cross-anisotropic. When natural sand is deposited under gravity, the material structure results in a cross-anisotropic fabric. Two types of anisotropy have been defined for soils. The first type is called inherent anisotropy. This refers to the initial fabric or particle composition when the soil is in its virgin state, before loading happens. The second type of anisotropy is induced by loading and plastic deformation of an originally isotropic soil. This is caused by the non-reversible strain increments placed on a specimen when following a stress path. Any change in inherent condition would be part of this second type of anisotropy.

Lade and Abelev (2003) studied the characteristic state of dense Santa Monica Beach sand. The characteristic line is defined at the point where the volumetric strain, $\varepsilon_{v}$ is equal to zero. At this line, the volumetric behavior of a material alters from contractive to dilative. This line can be shown as a straight line in the q-p' plane and it passes the origin with a slope, $\phi_{\mathrm{CL}}$. That being stated, Lade and Abelev found that the uniqueness of a characteristic line may not be preserved for an inherently cross-anisotropic material. The location of the characteristic line for an inherently cross-anisotropic material is dependent on two factors, the relative orientation between the direction of loading and the material symmetric axis, as well as the degree of anisotropy.

When analyzing cross-anisotropy in clays, certain mechanical properties (e.g. permeability, stress-strain behavior, and strength) are related to the orientation of the plate-shaped clay particles as well as their depositional environment and preconsolidation history. Studies of clay structures have shown that clay particles tend to become oriented perpendicularly to the major principal stress direction during one-dimensional consolidation. Under these conditions, the normal to the plane is an axis of radial symmetry and the material is considered cross-anisotropic. Properties that are associated with any plane can change with the angle between the plane being looked at and the plane on which the major principal stress acted when the clay was being consolidated. When analyzing clays, the engineering behavior of clay is determined by what the structure is at the time the behavior is studied. This final structure depends on what happened to the initial structure due to additional consolidation or shearing under vertical and/or inclined loading situations.

Inherent Anisotropy:

In 1944, Casagrande and Carillo studied strength anisotropy in soils. They classified anisotropy into two groups: inherent and induced. Inherent anisotropy implied that anisotropic characteristics were present in the soil before any strains were induced in the soil. Inherent anisotropy was therefore defined as "a physical characteristic inherent in the material and entirely independent of strains." They defined induced anisotropy as "a physical characteristic due exclusively to the strain associated with an applied stress." In the 1960's, several experiments were conducted to further study inherent anisotropy in soils. Below is Table 2.4.1, which summarizes some of the early experiments performed.

Table 2.4.1. Summary of early experiments performed to study anisotropy.

| 1965 | Johansson | Conducted a thorough literature review and found evidence of <br> geometrical anisotropy occurring in soils in situ. Concluded that <br> particle shape and depositional mode affected the particle <br> orientation. |
| :--- | :--- | :--- |
| 1967 | Weindieck | Used a 2D soil model of different sized coins to show a greater <br> number of contacts normal to the vertical direction of sample <br> pouring than parallel to it. |
| 1967 | Phillips and <br> May | Constructed a shear box and changed the sample pouring <br> direction with respect to the failure plane. Dry dense samples of <br> Leighton Buzzard sand poured in air through a side or end of the <br> shear box gave max stress ratios 24\% higher than for samples <br> poured through the top of the box. There was a difference of $\phi^{\prime}$ of <br> $5^{\circ}$. |
| 1968 | Parkin et al. | Hydrostatic compression tests on triaxial samples (medium sand) <br> deposited in air showed that the radial strain of the sample was <br> always much greater than the vertical strain. Found that the long <br> dimension of the grains were aligned in the horizontal plane and <br> were symmetrically disposed about the vertical axis. |
| 1972 | Arthur and <br> Menzies | Took radiographs showing plan and elevation views of cubical <br> samples of glass ballotini and aluminum discs poured through <br> water. They also did the same with Ham River gravel, poured <br> both through air and water. (see Figure 2.4.1a and b). |



Figure 2.4.1. Radiographs of a cube deposited aluminum discs (a) elevation (b) plan (after Arthur and Menzies 1972).

Early studies were performed with specially built shear boxes both by Phillips and May (1967) and by Duncan and Dunlop (1969). In these tests, inherent anisotropy of sand was studied by pouring sand in different directions with respect to the horizontal failure plane. The tests showed that the shear strength of sand is dependent on the inclination of the bedding planes. Phillips and May (1967) found that for dry dense samples of Leighton Buzzard sand poured through the side or end of the shear box constructed, a stress ratio of $24 \%$ higher was attained than when compared to the specimens poured through the top of the box. This corresponded to a variation of friction angle of 5 degrees.

In order to further study inherent anisotropy and not be limited by the shortcomings of the shear box used by Phillips and May (1967), Arthur and Menzies (1972) developed an apparatus that would allow for the deposition of sand in any direction relative to the applied principal stress directions. They created a prism shaped sample and used a true triaxial stress system that worked for a range of stress paths. The cubical samples were prepared while the molds were tilted at various angles to the direction of pouring. Figure 2.4 .2 shows the preparation of the specimens.

Samples were prepared with the same porosity and were deposited at angles of tilt, $\theta$ of $0^{\circ}$, $20^{\circ}, 30^{\circ}, 40^{\circ}$ and $90^{\circ}$. They were tested to failure in drained triaxial compression. Figure 2.4.3 shows the strength results. As can be seen from the figure, there is a strength anisotropy of $10 \%$ in the maximum principal stress ratio corresponding to over $2^{\circ}$ in $\phi^{\prime}$. This occurs between $\theta=0^{\circ}$ and $\theta=90^{\circ}$. In the region where $20^{\circ}<\theta<40^{\circ}$, there is a discontinuity (which corresponds to the orientation plane of maximum stress obliquity, $45^{\circ}-\phi$ ' $/ 2$ of approximately $26^{\circ}$ ).


Fig. 17(a)-(h). Preparation of
tilted sample

Figure 2.4.2. Preparation of tilted sample (after Arthur and Menzies 1972).


Figure 2.4.3. Variation in drained strength with angle of tilt (after Arthur and Menzies 1972).

Figures 2.4.4a and 2.4.4b show that the magnitudes of the lateral (intermediate) principal strains are different even though they are tied with the same magnitude of the principal stress $\left(\sigma^{\prime}{ }_{2}=\sigma^{\prime}{ }_{3}\right)$. As can be seen in the figures, the lateral strain in the direction of layering, $\varepsilon_{2}$ is in every case lower than the other lateral principal strain, $\varepsilon_{3}$.


Figures 2.4.4. (a) and (b) Stress ratios-lateral principal strains, (after Arthur and Menzies 1972).

Arthur and Menzies (1972) concluded that in their experiments, the anisotropic behavior of the sand was caused by deposition alone. They stated that the deposition history as well as the stress history was needed to model stress-strain behavior of granular materials. They also noted that the variation in strength was associated with the changes in relative magnitude of the intermediate principal stress.

Arthur and Phillips (1975) found that changing the relation of the plane of deposition to the major principal stress showed inherent anisotropy. They tested homogenous, layered and multilayered samples. For the homogeneous samples, although testing two different sands (Leighton Buzzard and Ham River), they saw that both sands had a rapid change in stress ratio for a small change in principal stress direction. They speculated that the ratio jump might be due to the difference in particle shape or surface texture. They concluded that inherent strength anisotropy could differ considerably among different granular materials.

As defined in the previous section by Casagrande and Carillo (1944), induced anisotropy could be defined as "a physical characteristic due exclusively to the strain associated with an applied stress." Therefore, induced anisotropy is an integral part of the straining process of a soil. However, it is difficult to study induced anisotropy due to the need to have controlled rotation of principal stress directions during shear.

In 1977, Arthur et al. created an apparatus (see Figures 2.4.5a. and 2.4.5b.) to study the controlled changes of principal stress directions in dense sand samples undergoing plane strain. With the new apparatus they were able to create a single sudden rotation of principal stress directions (from $0^{\circ}$ to $90^{\circ}$ ) to each sample of sand. To get rid of any inherent anisotropy, so that only induced anisotropy could be studied, the samples were deposited in the direction of the subsequently applied intermediate principal stress $\left(\Delta \varepsilon_{2}=0\right)$. Their tests showed that induced anisotropy caused a large effect on the magnitude of strain increments. It also caused a quick and small reduction in the deviation of the principal axes of stress and strain increments.


Figure 2.4.5. (a) Application of both normal and shear stresses to the sample. (b) Pulling sheets in place around sample under load (after Arthur et al. 1977).

There was a significant increase in major principal strain with reloading after a principal stress direction with a rotation of $\theta$. A significant amount of strain occurs before reaching the stress ratio of two once $\theta$ passes $50^{\circ}$. The major principal strains up until the stress ratio reaches 6 during the reloading path have been plotted against $\theta$ in Figure 2.4.6. The peak seen on this graph shows the strain when the rotation is at $75^{\circ}$. At this particular rotation, one no-extension direction of the reloading coincides with one no-extension direction of the first loading. These no-extension directions appear to be the axes of induced anisotropy that define the two directions where the major principal stress would have the minimum amount of stiffness.


Figure 2.4.6. Major principal strain to achieve a stress ratio of 6 on reloading plotting against rotation of principal stress direction (after Arthur et al. 1977).

Arthur et al. (1977) concluded that there would be three axes of induced anisotropy, two of which were minima and one of which would be a maximum. The maximum would be in the direction of the previously applied principal stress. They found that induced anisotropy did not greatly impact the angle of shearing resistance, $\varphi^{\prime}$. However as seen above, induced anisotropy can have a great influence on the strain needed to attain a certain stress ratio, as well as it can affect the secant modulus when reloading after a rotation of the principal stress direction.

Further studying anisotropy, Lade and Wasif (1988) performed tests on Cambria sand with height to diameter (H/D) ratios of 1.0 and 2.5 . The specimens were prepared in a specially designed mold, which was tilted at certain angles (see Figure 2.4.7). The sand grains were poured and shaken in different layers and then frozen. When ready, they were thawed and
sheared. Since the specimens were tilted, the major principal stress could be applied at different angles in relation to the bedding planes. Figure 2.4.8 shows the coordinate system for indication of the initial bedding plane inclination of the cross-anisotropic specimens prepared.


Figure 2.4.7. Mold for Preparation of specimens with inclined bedding planes (after Lade and Wasif, 1988).

Clear effects of cross-anisotropy were seen in tests that were inclined and that had vertical bedding planes. The stress strain curves of the tests performed are shown in Figure 2.4.9. As can be seen, tests with H/D ratio of 1.0 varied very little in strength with varying inclinations. However, a 5.5 degree difference in friction angle was seen in tests with a H/D ratio of 2.5 . This can be seen clearly in Figure 2.4.10.


## 1, 2, 3 : PRINCIPAL STRESS AXES

X, Y, Z: MATERIAL AXES
Figure 2.4.8. Coordinate system for the indication of initial bedding plane inclination of cross-anisotropic specimens (after Lade and Wasif, 1988).


Figure 2.4.9. Stress-strain and volume change characteristics obtained in triaxial compression tests on Cambria sand with cross-anisotropic fabric (after Lade and Wasif, 1988).


Figure 2.4.10. Variation of friction angles with bedding plane inclination in triaxial compression tests on Cambria Sand with cross-anisotropic fabric (after Lade and Wasif, 1988).

## Cross-Anisotropy Results using a true triaxial apparatus

Yamada and Ishihara (1979) used a cubical triaxial apparatus to study a wide range of one-directional loading conditions to further study the effects of anisotropic deformability of sand. Two specimens were sheared by increasing the major principal stress while decreasing the two minor principal stresses simultaneously. Although the stress conditions were identical in the tests, there were differences in deformation characteristics. The strain component in the direction of the major principal stress was larger in the second test than in the first. This showed that the specimen was less compressible in the direction of deposition than perpendicular to the direction of deposition. After looking at similar results from other samples, they concluded that a specimen prepared by vertical deposition of sand generally exhibited anisotropic deformation characteristics in such a way that the specimen is more
resistant when compressed in the vertical direction than in the horizontal direction. Two causes can be used to explain these deformation characteristics. Firstly, gravitational force may have an effect and compress the specimen to some extent, making the sand slightly more resistant to vertical deformation. Secondly, inherent anisotropy may have developed when the specimen was prepared underwater. They concluded that the effects were due to inherent anisotropy that formed in the specimen when the sand was deposited under water in horizontal layers.

When looking at volumetric strain, if the sand specimen was isotropic, then all the curves in Figure 2.4.11 would coincide, since the same stress conditions were applied. However, the difference in the volume contraction can be attributed to the difference in the mode of each straining occurring in different directions; a direct result of the inherent anisotropy of the specimens.


Figure 2.4.11. Volumetric strain versus stress ratio of RS $15^{\circ}, \operatorname{RS} 105^{\circ}$ and RS $135^{\circ}$ tests (after Yamada and Ishihara 1979).

The anisotropic behavior of specimen can be interpreted using the representation of the strain increment vector plotted in the principal stress space (see Figure 2.4.12). If the sand specimen were to be isotropic, then the strain vectors shown should be oriented symmetrically with respect to XE- and YC- directions. When looking closely at Figure 2.4.12, one can see that the strain increment vectors are oriented more in the clockwise direction. This shows that the shear strain of the specimen is anisotropic in nature. As the stress ratio became large enough to produce maximum volume contraction and failure, the effects of inherent anisotropy present in the sand specimens seemed to disappear. This can be seen in the plane strain increment vectors in Figure 2.4.12. As seen on the strain increment
vectors at failure for the XE and YC directions, the vectors are symmetrically oriented. This shows that the inherent anisotropic characteristics cannot be seen at large stress ratios causing failure in the specimen.


Figure 2.4.12. Representation of measured strains on the octahedral plane (after Yamada and Ishihara 1979).

Cross-Anisotropy Results using a torsion shear apparatus
Miura et al. (1986) studied the drained deformation characteristics of sand that had an anisotropic fabric, which was formed during deposition. This occurs due to the parallel alignments of particles during the deposition process. The specimens were exposed to a continuous rotation of the principal stress axes with the three principal stresses kept constant. By keeping them constant, it is possible to see the deformation characteristics due only to the
rotation of the principal stress axes. Tests revealed that the shear deformation characteristics and volume change due to this rotation are not as small as those that happen with irrotational shear. Depending on whether or not directional change of the principal stress occurred, the effects of the anisotropic fabric were different. This difference can be explained due to the predominant sliding on the bedding plane having the lower resistance value against shear stress. The schematic explanation of this is depicted in Figure 2.4.13.


Figure 2.4.13. Schematic explanation for the lowest resistance against sliding on bedding plane (after Miura et al. 1986).

Research done by Oda et al. (1978) showed that the shear plane nearly parallel to the bedding plane appeared when failure strength was at a minimum. Due to this, it can be presumed that the shear stress resistance along the bedding planes is at a minimum. The interlockings between elongated sand particles with their long axes laid horizontally are the poorest on the bedding plane. Because contact planes between particles are parallel to the bedding plane, the largest sliding displacement occurs on the bedding plane. Deformation behavior of anisotropic sand can be predicted by the largest displacements on potential sliding planes.

This can be predicted even under the most general stress conditions involving principal stress rotation.

Miura et al. (1986) also concluded that despite having values of the three principal stresses kept constant, specimens tended to contract accumulatively because of the rotation of principal stress axes, even though they expand due to the increase in shear stress involving no rotation of principal stress axes within the same stress domain. When rotating the principal stress axes, the direction of the principal strain increment axes is located between the principal stress and principal stress direction. It approaches the principal stress axes with increase in the shear strain increment. The magnitude of the strain increment becomes larger at $2 \alpha_{\mathrm{de}}= \pm 90^{\circ}$. The direction of the strain increment axes changes to this same direction as well as under the irrotational stress condition.

## Three Dimensional Failure Criterion for Cross-Anisotropic Soils

Abelev and Lade (2004) also used true triaxial tests to study the effects of crossanisotropy on Santa Monica Beach sand. They performed a total of 37 drained, true triaxial tests on cubical specimens. The specimens were oriented in a way so that tests with $b$-values in all sections of the octahedral plane (Figure 2.4.14) could be performed. When comparing the results from the results of previous studies done on a different type of sand (Cambria Sand), more pronounced effects due to cross-anisotropy of the Santa Monica Beach sand were seen. The major principal stress was always perpendicular or parallel to the bedding
planes. Experiments in all three sectors of the octahedral plane showed that air pluviated sand with horizontal bedding planes showed cross-anisotropy with lower strength in the horizontal direction than in the vertical direction. The failure surface was symmetric along the vertical principal stress axis on the octahedral plane.


Figure 2.4.14. Specimen orientation (a) in Cartesian coordinate system and (b) as installed in cubical triaxial apparatus (after Abelev and Lade 2004).

In order to create a failure criterion to take into account cross-anisotropy, Abelev and Lade (2004) used the isotropic 3D failure criterion, where
$\left(\frac{I_{1}^{3}}{I_{3}}-27\right)\left(\frac{I_{1}}{p_{a}}\right)^{m}=\eta_{1}$
Eq. 2.4.1
in which $\mathrm{I}_{1}$, the first stress invariant is calculated by:
$I_{1}=\sigma_{1}+\sigma_{2}+\sigma_{3}$
and where $\mathrm{I}_{3}$, the third stress invariant is calculated by:
$I_{3}=\sigma_{1} * \sigma_{2} * \sigma_{3}$
Eq. 2.4.3
and $p_{a}$ is the atmospheric pressure. $\eta_{1}$ and m are constant dimensionless parameters. They created a pseudo isotropic failure surface where the hydrostatic axis was shifted by an angle, $\alpha$ around the stress origin in the vertical triaxial plane. This can be seen in Figure 2.4.15.


Figure 2.4.15. Rotation of principal stress space around stress origin in vertical triaxial plane to capture cross-anisotropic strength observed in cubical triaxial tests (after Abelev and Lade 2004).

Once new parameters $\eta_{1}$, m and $\alpha$ were chosen, the new cross-anisotropic failure criterion could be plotted on the octahedral plane. See Figure 2.4.16.


Figure 2.4.16. Octahedral plane with comparison of test data for cross-anisotropic Santa Monica Beach sand and isotropic as well as cross-anisotropic failure criteria (after Abelev and Lade 2004).

As can be seen, the cross-anisotropic failure surface keeps all the properties of the isotropic criterion, except that it has a center axis of $\alpha$ degrees from the hydrostatic axis. Also, where shear banding would occur, failure can occur in the hardening regime. Test data from other tests performed on San Francisco Bay Mud and Toyoura sand were also plotted on the octahedral plane with the cross-anisotropic failure surface. These plots can be seen in Figures 2.4.17a and 2.4.17b.

Further analysis (presented in Lade and Abelev 2004) showed that the variation of maximum dilation illustrated the effects of cross-anisotropy in the specimen due to the initial direction of deposition. They found that the characteristic line of sand in the principal stress space may not be a unique feature in sands with very high degrees of structural cross-anisotropy.

Besides the degree of anisotropy, the location of the characteristic line is dependent on the orientation between the direction of loading and the material symmetry axis.
(a)

(b)


Figure 2.4.17. (a) Octahedral plane with comparison of test data or cross-anisotropic San Francisco Bay Mud and isotropic as well as cross-anisotropic failure criteria; (b) Octahedral plane with comparison of test data for air-pluviated dense Toyoura sand and isotropic as well as cross-anisotropic failure criteria (after Abelev and Lade 2004).

A shortcoming of the previously described failure criterion for cross-anisotropic soils is that it cannot take into account stress rotation. In 2007, Lade modeled the behavior of failure in cross-anisotropic frictional materials, taking into account rotation of the principal stresses. Lade $(2007,2008)$ developed a newer model that took into account the effects of stress rotation in cross-anisotropic soil, which the previous model by Abelev and Lade (2004) could not account for. This model incorporates the direction of loading relative to the microstructure directions of the material. Lade (2008) combined the isotropic failure criterion (Lade and Duncan, 1975, Lade 1977) with an expression including rotation by Pietruszack and Mroz $(2000,2001)$ as follows:

$$
\begin{equation*}
f=\left(\frac{I_{1}^{3}}{I_{3}}-27\right)\left(\frac{I_{1}}{p_{a}}\right)^{m}=\eta_{0}\left[1+\Omega_{1}\left(1-3 l_{2}^{2}\right)\right] \tag{Eq. 2.4.4}
\end{equation*}
$$

where $I_{1}$ and $I_{3}$ are the first and third invariants of the stress tensor, $p_{a}$ is the atmospheric pressure, and $m$ is a constant determined for specific soils. $\eta_{0}$ describes the three dimensional variation of a scalar over a sphere and $\Omega$ describes the deviation in three dimensions from the sphere. $1_{2}$ is the loading direction which can be defined for cross-anisotropic materials tested in 3D laboratory experiments as:

$$
\begin{equation*}
l_{2}=\sqrt{\frac{\sigma_{y}^{2} \sin ^{2} \beta+\sigma_{z}^{2} \cos ^{2} \beta}{\sigma_{x}^{2}+\sigma_{y}^{2}+\sigma_{z}^{2}}} \tag{Eq. 2.4.5}
\end{equation*}
$$

where $\sigma_{1}, \sigma_{2}, \sigma_{3}$ are the principal stresses and $\beta$ is the major principal stress direction with the vertical.

Three material parameters were determined from three conventional triaxial compression tests on vertical specimens and either two triaxial compression tests on horizontal specimens or two conventional triaxial extension tests on vertical specimens. The shear strengths obtained from these experiments in the mid ranges of b-values can represent shear banding and therefore, show a break in the homogeneous deformation of cross-anisotropic soils. Because of this break in the range of middle b-values, the data obtained from the test and used for parameter determination should be gathered at $b=0$ and/or $b=1$. The failure criterion established using data from Santa Monica Beach sand can be seen in Figure 2.4.18 and 2.4.19.


Figure 2.4.18. Comparison of failure criterion with true triaxial test data for crossanisotropic, dense Santa Monica Beach sand tested in all three sectors of the octahedral plane (after Lade 2007).


Figure 2.4.19. Comparison of failure criterion with torsion shear test data for medium dense, cross-anisotropic Santa Monica Beach sand. Shear band in the hardening regime reduces friction angles in mid-ranges of b-values (after Lade 2007).

However, not all sets of data can be represented with the proposed failure criterion for crossanisotropic frictional materials that takes into account stress rotation because special distinctive aspects of the material can control the measured behavior of the material. In particular, sea ice and San Francisco Bay mud could not be modeled using this failure criterion, even though anisotropy was clearly present in their soil fabric. The particular structures of their micro fabrics caused the materials to behave in ways that were not in accordance with the failure criterion and this model could not be applied.

## Conclusion

Many experimental tests have been performed to determine and study cross-anisotropy in soils. As previously discussed, anisotropy can be both inherent and induced. In order to
study both inherent and induced anisotropy, several experiments including the rotation of principal stresses have been performed. Induced anisotropy can have a great influence on the strain needed to attain a certain stress ratio, and it can affect the secant modulus when reloading after a rotation of the principal stress direction. Several experimental data has been presented in this paper to show how true triaxial shear boxes as well as torsion shear apparatuses have been used to see the effects that anisotropy has on the behavior of soils when reaching failure. A newly developed cross-anisotropic failure criterion for soils has also been discussed to show the difference between isotropic and cross-anisotropic failure surfaces.

### 2.5. Shear Bands

## Introduction to Shear Bands

Localization occurs when the specimen deformation divides from a homogeneous mode to a mode localized where one or more groups of grains form. This coalescence can be referred to as a dominant shear band. Figure 2.5.1 was taken from Saada et al. (1999). Digitized grids at various increments during the test clearly show the formation of shear bands and localization. Shear bands are referred to as "bands" because of the thickness of the zones when they form as well as their change in size with the deformation of the specimen. Localization is said to start at the point where a dominant shear band will grow. If there is no boundary to inhibit it from stopping (as sometimes is the case with certain testing equipment as will be discussed later), the shear band will continue to travel around the specimen. This is usually seen in hollow cylindrical specimens. Complete localization occurs around the peak
stress as the critical state is approached. Beyond this point, any further deformation occurring can be referred to as rigid body motion of one part of the specimen compared to the other part of the specimen. Depending on whether sand particles move into or out of the shear band, the band can grow or shrink in thickness. During stress relaxation, the moving blocks of specimen can experience elastic rebound. This can occur while the shear bands are deformed. Due to this motion, the lines that are within shear bands can be considered lines of zero extension. The direction of the shear bands can be identified as directions of propagation of the shear bands.


Figure 2.5.1. Digitized grids at various time increments, (a)t=0s, (b)t=1000s, (c) $\mathbf{t}=2000 \mathrm{~s}$, (d) $t=3000$ s (after Saada et al 1999).

Orientation of Shear Bands/Theory of Strain Localization

Strain-localization theory can be used to specify the conditions in which shear bands can emerge within materials that are uniformly stressed and strained. This theory also helps determine the orientation of shear bands when they emerge within a specimen. The theory considers a homogeneously strained material that is able to sustain a uniform Cauchy stress, $(\sigma)$. The velocity field, $\left(v^{0}\right)$ has a homogeneous spatial gradient, $\mathrm{L}^{0} . \mathrm{D}^{0}$ is the homogeneous rate of deformation. $\mathbf{W}^{0}$ denotes the initial spin tensor.

The strain localizes at the point where a velocity field $(v)$ different from $\left(v^{0}\right)$ forms in a planar shear band. The velocity gradient outside the shear band remains equal to $L^{0}$. However, the velocity gradient inside becomes $L^{0}+\mathbf{g n}$. (n) is the unit vector that is normal to the shear band and (g) is a vector function of the distance across a planar band that vanishes outside the band. The rate of deformation (D) and spin tensor $(\mathbf{W})$ inside the band can be quantified with the following equations:

$$
\begin{equation*}
\mathrm{D}=\mathrm{D}^{0}+1 / 2(\mathbf{g} \mathbf{n}+\mathbf{n g}) \tag{Eq. 2.5.1}
\end{equation*}
$$

$\mathbf{W}=\mathbf{W}^{0}+1 / 2(\mathbf{g n}-\mathbf{n g})$

There are certain major limitations to the strain-localization theory. Although it can predict the emergence of shear bands, it cannot analyze the actual development of the shear bands. It
also cannot guarantee that a shear band will form. Because there is no length dimension involved in the theory, it also does not predict the thickness of the shear band.

Besides these shortcomings, the theory of strain localization can be applied to the elastoplastic Mohr-Coulomb model. Doing so, the orientation of shear bands in sands can be predicted. Bardet (1991) showed that the Mohr-Coulomb model overestimates the inclination $(\theta)$ of shear bands in granular materials. An extended Mohr-Coulomb model may be used in order to predict more accurate values of $(\theta)$. The extended model that was developed uses an additional plastic mechanism to soften the transverse modulus. Because its mathematical structure is simple, an analytical expression can be derived for the plastic modulus and shearband orientation at the beginning of strain localization.

## Different Approaches developed to determine Orientation Angle

Zitouni (1988) stated that the orientation of a dominant shear band can be obtained by using an approach that involves a state of stress (statics) or by using a state of deformation (kinematics). Traditionally, the approach based on statics assumes the Coulomb plasticity criterion. This criterion states that the bands will develop in a direction making an angle ( $\alpha_{C}$ ) with the major principal stress where:

$$
\alpha_{c}=45+\left(\frac{\phi}{2}\right)
$$

The value of $\alpha$ is obtained on the basis of conditions for force equilibrium. It is assumed that the intermediate principal stress has no influence on the plastic behavior of the material. Figure 2.5.2 (taken from Saada et al. 1999) depicts Mohr's circle used in this approach. The point on (a) labeled " M " represents the magnitude of the normal and shearing stresses on planes that are found along the shear band. Although the intermediate principal stress $\left(\sigma_{2}\right)$ is ignored, that does not necessarily mean that plane stress or a plane strain behavior is achieved.


Figure 2.5.2. Mohr circles (a) stress; (b) strain increments (after Saada et. al. 1999).

A second approach based on kinematics was developed by Roscoe in 1970. This approach considers that the shear bands along a principal plane are located along lines where the rate of extension is equal to zero. From Mohr's circle, the direction of zero extension is
represented by the angle, $\alpha_{R}$. The shear bands develop in a direction making an angle with that of the major principal strain increment by the following equation:

$$
\begin{equation*}
\alpha_{R}=45+\frac{\psi}{2} \tag{Eq. 2.5.4}
\end{equation*}
$$

where $(\Psi)$ is referred to as the angle of dilation.

This angle is used to describe the orientation of the shear band in the plane $\mathrm{d} \varepsilon_{1}, \mathrm{~d} \varepsilon_{3}$. However, this name (angle of dilation), which implies that there is a volume change occurring, can only be justified under plane strain conditions. The value of $(\Psi)$ is acquired when the initiation of the dominant shear band is first seen. $(\Psi)$ is also attained at the peak of the strength curve once the band has completely surrounded the specimen. The peak happens at the point of inflexion of the curve, giving the volume change compared to $\varepsilon_{1}$.

The angle $(\Psi)$ can be calculated by the method used by Zitouni (1988). This method is based on the formula:

$$
\begin{equation*}
\sin \psi=-\frac{1+\left(\Delta \varepsilon_{3} / \Delta \varepsilon_{1}\right)}{1-\left(\Delta \varepsilon_{3} / \Delta \varepsilon_{1}\right)}=-\frac{1-\tan \theta}{1+\tan \theta}=\tan \left(\theta-\frac{\pi}{4}\right) \tag{Eq 2.5.5}
\end{equation*}
$$

$(\theta)$ is given by the slope of the line AB in a plot of $\varepsilon_{1}$ versus $\varepsilon_{3}$. This slope can be seen in the Figure 2.5.3 (taken from Saada et al. 1999). Localization is said to occur where the linear part of the curve stops.


Figure 2.5.3. Zitouni's method for obtaining $\Psi$ (after Saada et al. 1999).

Another method was developed based on direct shear tests by Arthur et al. (1977) and Arthur and Dunstan (1982). They found that the orientation of shear bands was inclined between the Coulomb and the Roscoe directions. They suggested that the average of these two angles should be used to define the direction of shear bands. The equation for the Arthur et al. angle can be seen below and is referred to as "half angle":
$\alpha_{\mathrm{R}}=45-\left(\frac{\phi+\psi}{4}\right)$
Eq 2.5.6

Vardoulakis (1980) supported this suggestion through his experimental findings. Later in 1990, Koenders suggested that the inclination of shear bands ( $\alpha$ ) depended on the average grain size. Figure 2.5.4 shows the relation of Roscoe, Arthur's (half angle) and Coulombs inclination angles versus the average grain size. Further experiments showed that coarse sands created shear bands in the direction of zero extension (as proposed by Roscoe) and fine sands produced shear bands in the direction proposed by Coulomb. Medium sands, with grain sizes in a relatively narrow range produced shear bands that were inclined at Arthur's half angle.


Figure 2.5.4. Proposed Shear band inclination as a function of average grain size (after Lade et al. 1996).

Scarpelli and Wood (1982) conducted simple shear tests on sand, from which they suggested that shear band inclination was affected by the degree on constraint felt by the sand. This was cased by the boundary conditions on the testing apparatus being used.

Vermeer (1990) proposed a theoretical analysis to obtain the shear band inclination. He suggested that in plane-strain tests, the orientation of the shear band would be between Coulomb's and Roscoe's angle. He also said that the boundary conditions (i.e. the membrane which surrounds the sample) could affect the shear band direction. When dealing with fine sands, the membrane plays a minor role. In fine sands, a shear band is usually very thin, about 10-20 times the average grain diameter. In this situation, the Coulomb angle is the most critical inclination for the development of shear bands. However, when dealing with coarse sands, the shear bands emerge thicker. Since they are thicker, they take up a larger proportion of the specimen and in this case, boundary conditions prove to be more important. With shear band development in coarse sands, incorrect stresses at the end of the shear bands can be formed due to the flexible rubber membranes used around specimens. Therefore, the Roscoe angle orientation is obtained in tests with coarse sands. As previously stated, when dealing with medium sands, shear bands develop between these two conditions. Vermeer's conclusions imply that the apparatus used for testing and the boundary conditions set, can influence the results. There exists the possibility that if a specimen is large enough and the boundary conditions do not play an important role, Coulomb's inclination angles can be used.

Figure 2.5 .5 a shows the development of a shear band in triaxial compression tests with a short specimen (where $\mathrm{H}=\mathrm{D}$ ) and a tall specimen (where $\mathrm{H}>\mathrm{D}$ ). The shear plane transcends a length of $\mathrm{D}^{*} \tan (45+\phi / 2)$ for Figure 2.5 .5 b. The shear band can be seen to fully develop in the taller specimen.


Figure 2.5.5. Testing techniques employed in triaxial compression tests (a) short specimen where $\mathrm{H}=\mathrm{D}$ with lubricated ends and (b) conventional tall specimen (after Lade et al. 1996).

In Figure 2.5.6, one can see the effect that the specimen boundaries have on the development of shear bands and the inclination angle on different sized specimens. Figure 2.5 .7 shows the effect that lubricated ends have on the development of shear bands.

Han and Drescher (1993) conducted plane-strain biaxial compression tests on dry coarse sand at various confining pressures. Figure 2.5.8 shows a schematic of the biaxial


Figure 2.5.6. Shear band inclination and interception in (a) conventional extension tests and (b) extension tests on very short specimens (after Lade et al. 1996).


Figure 2.5.7. Merging of shear bands with lubricated ends (after Lade et al. 1996).
apparatus used and the incremental deformation that exists inside a shear band. These tests investigated the state of shear band formation, shear band orientation and shear band growth. Their objective was to research the state of shear band formation in poorly graded course sands. There was also an aim to compare the results from the experiments with predictions provided by the equilibrium bifurcation theory for a number of local incremental constitutive equations. Finally, they wanted to investigate the progressive growth of shear bands.


Figure 2.5.8. Biaxial apparatus (a) schematic, (b) incremental deformation in a shear band (after Han and Drescher 1993).

As stated in the previous section, equilibrium bifurcation theory helps explain the formation of shear bands. This theory gives a reason for why shear bands form in tests on specimens despite "ideal" boundary conditions. For a material described by an incremental constitutive equation, the development of shear bands is identified with the presence of a non-uniform deformation field that occurs even under ideal loading conditions. Bifurcation theory gives information on strains at the time of shear banding, shear band orientation and the thickness of shear bands.

In the Han and Drescher study, all of the experimental tests were conducted with displacement controlled axial loading at the displacement rate of $0.2 \mathrm{~mm} / \mathrm{min}$. The specimens were subjected to constant confining pressures of $50,100,200$, or 400 kPa . Constitutive models were set up to make predictions of the experiments. As seen in Figure 2.5.9, the biaxial experiments showed that both the shear strain at shear banding and the shear band orientation in dry, poorly-graded coarse sand are dependent on the magnitude of the confining pressure. When the confining pressure increases, the shear strain increases. The shear band inclination angle measured (with respect to the direction of the major principal stress) decreased as the confining pressure increased.

The shear band orientation was much lower than calculated when using the Coulomb formula (as seen in Figure 2.5.10). When lower confining pressures were applied, the
orientation was about 2 degrees higher. At higher confining pressures, the angle was equal to that predicted by the Roscoe formula.


Figure 2.5.9. Test results and predictions; (a) shear strain at shear banding vs. confining pressure; (b) shear band inclination angle vs. confining pressure (after Han and Drescher 1993).


Figure 2.5.10. Test results and classical predictions for the shear band inclination angle (after Han and Drescher 1993).

When comparing the bifurcation analysis for local incremental constitutive equations and Mohr-Coulomb type yield condition to the results, the predictions from these equations gave much smaller shear strains at shear banding than what was shown by the experimental data. The theory does not adequately describe the orientation angle at higher confining pressure or the experimentally observed trend in shear band orientation in relation to the magnitude of confining pressure. It was unknown if using other local constitutive equations and not modifying the yield conditions would get rid of these discrepancies. Shear band growth analysis shows that a non-local constitutive equation and flow theory provide a shear band thickness (seen in Figure 2.5.11) at localization very near what the experimental data showed in the results. The thickness that was seen can be explained by taking into account material dilation and abrasion, which formed
parallel shear bands. That implied periodicity is a phenomenon that can be observed in tests on water-saturated sands.


Figure 2.5.11. (a) Shear band thickness vs. shear band relative displacement; (b) Variation of ratio of voids and grains across shear band (after Han and Drescher 1993).

Figure 2.5.12 shows contours and vectors of incremental Digital Image Correlation (DIC) displacements during a biaxial compression test on dense masonry sand performed by Rechenmacher (2005). The image locations are indicated at certain points along the stress-strain curve. Over 10,000 displacement data points across the area being analyzed were produced using DIC analyses for this biaxial test. The images captured over $80 \%$ of the specimen height and width. Membrane discoloration and shadowing didn't allow for the entire $100 \%$ to be captured. About $3 \%$ of the subset displacement vectors are shown for clarity. The scale bar and tick marks on the axes in the figure shown below indicate $20-\mathrm{mm}$ distances horizontally and vertically across the specimen face.


Figure 2.5.12. Local DIC-Derived displacements during biaxial compression test on Mason sand (after Rechenmacher 2005).

From the image taken, shear banding naturally tends to initiate along conjugate planes. When images were compared between biaxial and triaxial tests, the patterns in shear band formation were partly due to the different boundary conditions. When the boundary conditions do not create constraints on the specimen nor do they affect the way that it behaves, as seen in the biaxial apparatus, one dominant, nearly linear shear band is formed. A possible reason for a slower, less abrupt shear band formation in triaxial testing compared to biaxial testing can be explained due to the deformational constraints created from a laterally fixed, non-lubricated top and bottom soil base and cap. The images of the triaxial test that was performed can be seen in Figure 2.5.13.


Figure 2.5.13. Local displacements and global behavior during triaxial test on levering sand (after Rechenmacher 2005).

## Results from Triaxial Testing Shear Banding and Strain Softening Correlations:

Suzuki and Yamada (2006) conducted a series of experiments in triaxial equipment to study shear behavior of sand from a macromechanical point of view. They also wanted to discuss a possible mechanism of shear behavior. The drained triaxial tests were conducted on Toyoura sand and samples with two distinct void ratios $(\mathrm{e}=0.65$ and $\mathrm{e}=0.80)$ were prepared. Figure 2.5 .14 shows the typical stress-strain curves for two initial void ratios and two types of stress paths. The specimens were all 5 cm in diameter and 10 cm in height. The samples were isotropically consolidated to three values of initial confining stresses.


Figure 2.5.14. Typical stress-strain curves for two initial void ratios and two types of stress paths (after Suzuki and Yamada 2006).

In the experiments that were conducted, three typical failure shape patterns were seen for axial strains of up to $30 \%$. These consisted of 1) weakly developed shear bands, 2) fully
developed shear bands, and 3) fully developed shear bands that cross diagonally. These are represented in Figure 2.5.15. The results from the stress-strain curves could also be classified into three groups: a) no strain softening, b) strain stiffening after strain softening, and c) two intervals of strain softening (sometimes referred to as double strain softening).


Figure 2.5.15. Failure shape patters: (a) weakly developed shear band, (b) fully developed shear band, (c) diagonally crossing shear bands (after Suzuki and Yamada 2006).

Comparing the data showed that when strain localization was relatively faint, (occurring in weakly developed shear bands), the stress-strain curve exhibits no strain softening. However, when strain localization is strong (occurring in a fully developed shear band), the stress-strain curve exhibits strain softening and subsequent stiffening. It can be stated that the failure shape pattern can be an indicator of the ultimate condition of strain localization. The pattern of the stress-strain curve is strongly related to the failure shape pattern.

This set of data also showed that the stress-strain curve exhibits strain softening when the extent of strain localization is relatively large. Also, strain softening becomes greater as the maximum dilatancy index defines the extent of strain localization. The dilatancy index
determines not only the ratio of the volumetric strain increment to the axial strain increment, but also the condition of the strain localization.

Once the peak strength is reached, the dilatancy index gradually decreases. This causes the dilative volumetric strain increment rate to begin to decrease, while the volumetric strain increment is still dilative. An important qualitative change occurs around the peak strength. Due to this behavior, it can be seen that strain softening is strongly correlated to the appearance of a shear band. One possibility is that the peak strength occurs when a shear band begins to form. Two different regimes for the dilatancy index exist, before and after the peak strength is reached. Before the peak strength is reached, the dilatancy index can be produced without dominant strain localization. Beyond the peak strength, the dilatancy index is influenced by the extent of sliding within the shear band. Figure 2.5 .16 shows the typical progressive failure of sands in drained triaxial tests. At different points along the curve, the development of shear bands can be seen.


Figure 2.5.16. Illustration of progressive failure of sands in drained triaxial tests (a) initial condition, (b) occurrence of dilative strain areas, (c) continuous linking of dilative strain areas, (d) fully developed shear band, (e) further occurrence of dilative strain areas, (f) continuous link of dilative strain areas, (g) "diagonally crossing shear bands, (h) q vs ea (after Suzuki and Yamada 2006).

The failure process can be summarized by following the points on the curve. At point (a), the initial condition of stress and strain is assumed to be the same throughout the entire specimen. Then at point (b), small compressive and dilative strain areas occur at various locations throughout the soil sample. The dilative strain begins to link continuously at point (c), and the development of a shear band starts to occur. The beginning of the shear band formation coincides with the peak strength. At this point, part of the specimen starts to slide along the developing shear band. When the shear band slides, most of the deformation is absorbed and the dilative strain increment areas of the sample decrease as the shear band develops further. The dilatancy becomes smaller. At point (d), the volumetric strain increment becomes zero, even at points within the shear band. At this point, the dilatancy
becomes zero as well and a new equilibrium is reached. At point (e), a new shear band can occur and dilative strain areas develop in the soil sample once again, but only a small stress ratio increment is now needed. Point (f) shows the dilative strain area beginning to link continuously to create a shear band that is more developed. Diagonally crossing shear bands can sometimes develop at this point. Point $(\mathrm{g})$ shows the point where the dilatancy index becomes zero once again, and equilibrium is reached once again.

## Results from Triaxial Extension Tests

For materials that dilate under conditions of triaxial compression, plastic strain localization in granular materials leads to the development of shear bands in the post peak softening regime. For materials that do not dilate, meaning that they compress under triaxial compression, the development of shear bands has not been observed. For plane strain conditions in materials that dilate, shear bands develop very close to the point of peak failure. In this situation, the developing shear band negligibly affects the peak friction angle.

When performing triaxial extension tests with uniform stress conditions, the orientation of the shear band occurs at an angle of $45-\phi / 2$ to the planes of the cap and base. If the soil specimen is taller than $\mathrm{D}^{*} \tan (45+\phi / 2)$, then shear bands can occur in extension. If $\mathrm{H}<$ $D^{*} \tan (45+\phi / 2)$, then it is possible to stop the development of shear bands in a normal triaxial extension test. The friction angle that is measured for short specimens would represent uniform strain behavior. If the specimen is taller, it is possible for shear bands to occur in extension tests. To achieve uniform stress behavior when performing triaxial
extension tests on cubical specimens, one must apply equal stress differences. Therefore, $\left(\sigma_{1}-\sigma_{3}\right)$ must equal $\left(\sigma_{2}-\sigma_{3}\right)$. Since the height of the cylindrical specimen is important, it may have to be shorter than $\mathrm{D}^{*} \tan (45+\phi / 2)$ to achieve uniform strain behavior. This is because shear bands usually have a thickness of about 10 to 20 grain size diameters.

Figure 2.5 .17 shows that a transition zone exists in which the friction angle decreases as the height approaches $\mathrm{D}^{*} \tan (45+\phi / 2)$. There is a drop in value as the sample goes from uniform strain behavior to lower, more erratic values that were seen during early development of shear bands.


Figure 2.5.17. Expected variation of friction angle with $\mathrm{H} / \mathrm{D}$ ratio in extension tests (after Lade et al. 1996).

The stress-strain and volume change for two cylindrical specimens under triaxial extension are shown in Figure 2.5.18. Both specimens, one under uniform strain, the other under
conventional strain, show volumetric compression. Because necking occurred in one specimen, this caused a lower strength compared to the second specimen. The second specimen had membranes that were reinforced with small curved plates to slow down the development of necking. Since necking was prevented in this test, the strength was higher. In triaxial extension tests, if necking and/or shear bands that occur after necking but form before peak failure are present, then premature peak failure can be seen.


Figure 2.5.18. Comparison of results from drain uniform strain and conventional triaxial extension tests on cylindrical specimens of dense Cambria sand with confining pressure of 17.5 MPa (after Lade et al. 1996).

Tests on Cambria sand, medium silica sand and fine silica sand were presented by Lade et al. (1996). The Figures 2.5.19(a)-(c) show the relation of shear band inclination (in degrees) with the height/diameter ratio of the specimens tested in extension tests. All three cases are closer to the Coulomb orientation angle. A possible explanation for the deviation from
previous speculation, which states that coarse sand should be closer to Roscoe's number and the medium silica sand to the half angle value could be explained by the anisotropy of the soil. When specimens are formed using the dry pluviation technique, cross-anisotropic samples are formed. The way that the grain axes line up (forming horizontal directions while the grain contact points are vertical), tend to form shear bands that appear more horizontal under triaxial extension loading.

## Results of Shear Banding in True Triaxial Tests

When conducting true triaxial tests, the stress-strain relationship obtained from experimental results often show a continuously decreasing strain to failure with increasing bvalue. There is a more rapid strength reduction and a pointed peak for $b$-values $\left(b=\left(\sigma_{2}-\right.\right.$ $\left.\left.\sigma_{3}\right) /\left(\sigma_{1}-\sigma_{3}\right)\right)$ near and larger than the value at plane strain. Due to this, shear bands that form before failure are prevalent over a range of $b$-values. When stresses are in this range, failure may result due to the shear band that has formed under these stress conditions. Thus, the peak strength may be dependent on the critical condition at which shear banding forms. With true triaxial equipment one can produce uniform stresses and strains over the full range of the intermediate principal stress. Therefore, one can study the influence that the formations of shear bands have on the stress-strain behavior of granular materials.


Figure 2.5.19. Figure 2.5.19. (a) Comparison of measured and proposed shear band inclinations with $\mathrm{H} / \mathrm{D}$ ratio in extension test on dense specimens of (a)Cambria sand of (b) medium silica sand and (c) fine silica sand (after Lade et al. 1996).

Bifurcation marks the initiation of the formation of a shear band in a sand that dilates. This is a result of continued shearing under uniform stress and strain. At a certain point in this process of continued stress and strain being imposed on to the soil specimen, the stress and strain states allow a localized displacement in order to satisfy equilibrium, compatibility, boundary conditions and constitutive relations. In order to investigate shear banding under 3D conditions, this occurrence has been applied to study existing constitutive models that predict the formation of shear bands, as well as to study the conditions that are present in shear band formation.

When performing bifurcation analysis for 3D stress and strain states, a set of critical hardening moduli, $\mathrm{H}_{\mathrm{c}}$, is normalized with Young's modulus of elasticity, E , as seen in Figure 2.5.20. The values vary with changing values of $b$.


Figure 2.5.20. Schematic diagram of variation of normalized, critical hardening modulus with $b$ according to Rudnicki and Rice (1975) and Rice (1976) (after Lade and Wang 2001).

In 1975, Rudnicki and Rice proposed that shear banding could occur over a range of stress states where the hardening modulus is positive. The means that shear banding can occur during increasing loading before smooth peak failure. The normalized critical hardening modulus can be calculated for true triaxial tests at the point where shear banding starts. In Figure 2.5.21, one can see the variation of the normalized critical hardening moduli for true triaxial tests on dense, medium dense and loose Santa Monica beach sand.


Figure 2.5.21. Variation of Normalized, Critical Hardening Modulus, Hc/E, with b for True Triaxial Tests on Prismatic specimens of Dense, Medium Dense and Loose Santa Monica Beach Sand with $\sigma^{\prime}{ }_{3}=49 \mathrm{kPa}$ (after Lade and Wang 2001).

For tests conducted on tall rectangular prismatic specimens with a height to diameter ratio $H / D=2.47$, shear bands developed without interference with the lubricated cap and base. For tests with $\mathrm{b}=1$, failure occurred in the horizontal direction due to cross-anisotropy of the specimen and the corresponding pattern can be seen in Figure 2.5.22. The angles of shear
band inclination with respect to the direction of the minor principal stress were determined. In most cases, the inclinations angles were between $60^{\circ}$ and $70^{\circ}$. They were not necessarily affected by b-values but did increase a little with increasing density.

After analyzing the experimental results, the stress-strain curves and failure surfaces showed shear banding in the hardening regime on the peak strength of the material in the midrange of $b$-values. Shear banding was seen to start in the hardening regime when $b$-values reached $a$ range of 0.18 to 0.85 . In this region, failure does not occur due to peak failure, but does however occur due to the formation of shear bands.


Figure 2.5.22. Modes of Shear banding observed in prismatic specimens of Santa Monica beach sand (a) failure in vertical direction for tests with $0 \leq b \leq 1$; (b) failure in horizontal direction for tests with $\mathrm{b}=1$ (after Lade and Wang 2001).

In order to make sure that this was the reason, the shear band formation analysis proposed by Rudnicki and Rice in 1975 was used for each test. The hardening modulus prior to the formation of shear banding was calculated. For b-values in the ranges from $0-0.18$, and also from 0.85-1, negative hardening moduli were found. This meant that conditions for shear
banding were fulfilled in the softening regime. Failure in these regions occurs due to smooth peak failure. In the range of $b$-values from 0.18 to 0.85 , positive values of the hardening modulus were obtained. This occurred immediately before shear bands formed. This indicated that the condition for shear banding is fulfilled on the hardening portion of the stress-strain curve. Peak failure, in this middle range of b-values, is caused by shear banding. Because this occurs, a smooth, continuous 3D failure surface is therefore generally not achieved for soils. Figure 2.5 .23 shows the Mohr-Coulomb failure surface as well as the Lade failure surface. The results have higher friction angles than predicted by MohrCoulomb. However, Lade's failure surface is quite accurate until the middle range of bvalues is reached. In this middle b -value range, shear banding occurs in the hardening regime. Figure 2.5 .24 shows the same tests but on the octahedral plane.

Experimental shear band orientations are compared in Figure 2.5 .25 with the three theoretical values explained previously. The measured shear band inclinations are between the Coulomb and the Arthur inclinations. With denser sand, the shear bands fit better with the Coulomb inclination. For dense Santa Monica Beach sand, the inclinations are almost equal to the Coulomb inclination angle. In loose sand, there is considerable scatter. This scatter can be explained by the boundary conditions that have an effect on the shear band directions.


Figure 2.5.23. Comparison of Mohr-Coulomb failure surface and Lade's failure surface with measured friction angles for true triaxial tests on prismatic specimens of dense, medium dense and loose santa monica beach sand with $\sigma_{3}=49 \mathrm{kPa}$ (effect of shear banding indicated in mid-ranges of b-values) (after Lade and Wang 2001).


Figure 2.5.24. Experimental failure surfaces for true triaxial tests with $\sigma^{\prime}{ }_{3}=49 \mathrm{kPa}$ compared with Mohr-Coulomb failure surface and Lade's failure surface in octahedral plane on prismatic specimens of Santa Monica Beach Sand (a) dense, (b)medium dense, (c) loose (after Lade and Wang 2001).

## Results from Torsion Shear Testing

Lade et al. (2008) conducted 34 drained torsion shear tests with rotation of principal stress directions in order to study shear banding in Santa Monica beach sand. All the tests were conducted with the same constant internal and external confining pressures. Therefore, the b value and the inclination angle of the major principal stress were connected. Shear bands were able to develop freely without noticeable restraint from the rubber membranes. Strain localization and shear banding was noticed in the hollow cylindrical specimens.

In the torsion shear tests conducted, the stress-strain curves did not show an abrupt drop off in load carrying capacity at the formation of shear bands, as was observed in the corresponding three triaxial tests. Most stress-strain curves continued with small changes in strength. There was also very little change in the stress-strain curve and the volume change patterns once the shear band was observed to develop. After studying the specimens more closely, it was noticed that some stress-strain curves had a slight upwards rise following the shear band formation. The sliding of one part of the soil specimen on top of the other part ended in one of the parts being restrained at the end rings. Due to this, the specimen picked up some more load. This created a slight increase in stresses. As this occurred after the shear banding, it is not part of the uniform deformation pattern.


Figure 2.5.25. Experimental shear band directions compared with three theoretical values for Santa Monica Beach Sand (a)dense, (b) medium, (c) loose (after Lade and Wang 2001).

Shear bands that developed in torsion shear testing can be seen in Figures 2.5.26a and 2.5.26b. The shear band lines were drawn on the rubber membrane where kinks were formed. The shear band inclination angle was measured directly on the specimen and then recorded with photographic evidence.


Figure 2.5.26. (a) Picture of 40 cm tall torsion shear specimen after failure, (b) picture of 25 cm tall torsion shear specimen after failure (after Lade et al. 2008).

After the shear band inclination angle relative to the $\sigma_{1}-$ plane was measured on the specimen, this angle, $(\omega)$ is added to the angle $(\beta)$ of $\sigma_{1}$ that is obtained from the equation, $b=\sin ^{2} \beta$. Therefore, as seen in Figure 2.5.27, the angle of the shear band with relation to the $\sigma_{1}$ plane is $\beta+\omega$.


Figure 2.5.27. Inclination of shear bands relative to plane of major principal stress in torsion shear tests (after Lade et al. 2008).

The shear band inclination angles that were measured were compared to the inclination angles of Coulomb, Roscoe and Arthur's orientations. These results can be seen in Figure 2.5.28. This figure shows decreasing shear band inclinations with increasing $b$-values. This is probably due to the horizontal bedding planes that influence the shear bands. As previously discussed, the anisotropy of the specimen influences the direction of shear bands in torsion shear tests as well as in triaxial tests.


Figure 2.5.28. Comparison of experimental shear band inclinations with theoretical values (after Lade et al. 2008).

### 2.6. Yield and Failure Surfaces

## Introduction:

Yield surfaces of sands can be established from stress-strain data. It is also possible to determine the effects of certain factors such as density, consolidation stresses, anisotropy, material fabric, etc. on yielding behavior. To do so is not an easy task. In order to construct yield surfaces, tests that produce data under complex and unconventional stress paths are performed on the soil. Sometimes, these yield points have to be approximated. Oftentimes, loading the soil causes changes in properties making it difficult to get different yield points along the same yield locus. For materials such as clays, ellipses have often been used to draw the shapes of the yield surfaces. For sands, a wide range of shapes have been used, as it is often more difficult to characterize the yield shapes due to the difficulties just stated. This
section will explain the basis and history in the development of yield surfaces and their determination, as well as report experimental research that has been presented in the literature that has helped expand the current knowledge on yield surfaces for sands and clays.

## Yielding of Soils

Hvorslev in 1937 showed that the peak shear stress at failure, $\tau_{f}$ of remolded saturated cohesive soils is a "function of the effective normal stress, $\sigma^{\prime}$ ', on and of the voids ratio, $e_{\mathrm{f}}$ in, the plane of failure at the moment of failure". This function is independent of the sample's stress history. Using Hvorslev's approach, it was only possible to predict the strength at failure when knowing the values of the normal stress and the void ratio at failure. In 1953, Roscoe designed an apparatus for the purpose of imposing uniform shear strains on soil samples. When uniform strains were applied to the samples using this apparatus, it was possible to find out the void ratios at all times during the test by measuring the average void ratio of the entire sample. This strain-controlled apparatus allowed for the measurement of strains beyond failure. They used this simple shear apparatus to study the yielding of cohesionless media and the critical void ratios of those media. Yielding in this case meant what is now referred to as failure.

Hvorslev's failure surface was based upon results of fully drained tests on saturated clays in shear boxes. Refinements to this criterion were introduced by Roscoe et al. (1958) after they performed different tests. The shear tests that were performed by Hvorslev were stress
controlled. Therefore, any further study of the conditions of the sample after the peak shear stress could not be studied. Figure 2.6.1 shows Hvorslev's surface.

## Critical Voids Ratio and the Ultimate State Surface

Continuous yielding of a specimen can happen when a loading path gets to the yield (failure) surface and then remains on the failure surface. Therefore, it may be difficult to find out if that loading path ends at a specific point. Once the loading path reaches this specific point, it can be said to be at the critical voids ratio state.

The term "critical voids ratio" is usually applied to a particular state of sand. Roscoe et al. (1958) gave two definitions for two states where this term can be applied. The first definition deals with changes of volume in drained tests (per Casagrande 1938). The second deals with changes in effective stress and strength in undrained tests (per Taylor 1948). The critical voids ratio concept is also valid for clays.

In the case of a drained test, the critical voids ratio state is defined by Roscoe as "the ultimate state of a sample at which any arbitrary further increment of shear distortion will not result in any change of the voids ratio". For any series of drained tests that are performed, the critical void ratio points are expected to lie in or near a line of the drained yield surface (failure surface).


Fig. 1. Two isometric views of the Hvorslev surface for Kleinbelt Ton


Figure 2.6.1. Geometry of Hvorslev surface for Kleinbelt Ton (after Roscoe et al. 1958).

When dealing with undrained tests, the sample stays at a constant voids ratio. However, the effective stress, $\mathrm{p}^{\prime}$, will change to bring the sample into an ultimate state. This state will be such that the voids ratio remaining during shear becomes the critical voids ratio. For any series of undrained tests, the critical void ratio points are expected to lie in or near the undrained yield surface (failure surface).

When comparing results from drained and undrained tests, if there is a unique line where all of the loading paths converge then that line is called the critical voids ratio line (C.V.R.). The drained and undrained yield (failure) surfaces occur on the same C.V.R. line and may be identical, forming a common yield (failure) surface.

As shown in the previous section, in the beginning stages of discussion of yield/failure surface parameters, the quantities used to represent certain surfaces (like Hvorslev's Surface) were p', q and e. Roscoe et al. (1958) labeled the C.V.R. line discussed in the paragraph above. Later, it was pointed out that the frame of reference that Roscoe used was a state space and the C.V.R. should be called a Critical State Line. When this line is transformed to a three dimensional space, it becomes a surface. This surface is referred to by Poorooshasb (1989) as the "Ultimate State Surface."

The ultimate state surface can be defined as a surface in the state space for which the relation:

$$
\begin{equation*}
\frac{\mathrm{dp}}{\mathrm{~d} \varepsilon}=\frac{\mathrm{dq}}{\mathrm{~d} \varepsilon}=\frac{\mathrm{d} \theta}{\mathrm{~d} \varepsilon}=\frac{\mathrm{de}}{\mathrm{~d} \varepsilon}=0 \tag{Eq. 2.6.1}
\end{equation*}
$$

The above relation should hold true at every point on this surface in the state space. $\varepsilon$ is a shear strain derived from the second invariant of the strain deviator tensor and is a measure of sample distortion.

According to Poorooshasb (1989), once the ultimate state is reached, if a unique relation can be attained between two of the state parameters, a three dimensional space may be used to represent the Ultimate State Surface. For example, if a simple relation exists between the void ratio and the mean normal stress, $p^{\prime}$, then a space made up of $(q, \theta, e)$ could be used. This relationship would be governed by the Casagrande's equation:

$$
\begin{equation*}
\mathrm{e}_{\mathrm{c}}=\mathrm{e}_{0} \lambda \ln \left(\frac{p}{p_{0}}\right) \tag{Eq. 2.6.2}
\end{equation*}
$$

## State Boundary Surface

The state boundary surface was defined by Poorooshasb (1989) as "the surface in the state space enveloping all the possible states a sample of a granular medium may assume." Not all state points enclosed by the state boundary surface can be reached. However, those
that are not enclosed are not accessible. Figure 2.6 .2 shows the Ultimate State Surface of Sacramento River Sand.


Figure 2.6.2. Isometric View of the Ultimate State Surface for Sacramento River Sand (after Poorooshasb 1989).

Under triaxial compression, the surface assumes the form shown in Figure 2.6.3. The Ultimate State Surface creates only a trace of the State Boundary Surface which lies on the $\mathrm{e}=0$ plane.


Figure 2.6.3. State Boundary Surface in the (p,q,e') space, $g(\theta)=1$ (after Poorooshasb 1989).

When e is constant, a bullet shaped surface with its apex at the origin of the stress space and its axis along the diagonal is formed. In this situation, the Ultimate State Surface traces a band on the State Boundary Surface, which can be seen in Figure 2.6.4.


Figure 2.6.4. State Boundary Surface for Sacramento River Sand, e=5 (after Poorooshasb 1989).

## Bounding Surface and Yield Surface:

In 1975, Dafalias and Popov introduced the concept of bounding surface. This surface distinguishes between virgin loading and secondary loading. The bounding surface expands and moves during virgin loading as the stress point goes through a certain curve in the stress space. In non-virgin loading, the stress point no longer is touching the bounding surface. The State Boundary Surface encloses the bounding surface and the bounding surface has the same curvature as the state boundary surface. The position of the bounding surface is controlled by the current state of the sample when the loading is in its virgin state, and therefore, will have a similar curvature as the State Boundary Surface. This can be seen in Figure 2.6.5.


Figure 2.6.5. Bounding Surface and its relation to the state boundary surface in the (p,q,e') space, $g(\theta)=0$ (after Poorooshasb 1989).

When the sample undergoes virgin loading, the yield surface is tangential to the bounding surface and moves with it. The yield surface encloses the set of points in the state space for which the behavior of the sample is elastic/reversible.

Figures 2.6.6 and 2.6.7 show the State Boundary Surface alongside the bounding surface and the yield surface for various states with constant void ratios. The yield surfaces are all curved. All three surfaces start from the origin. During isotropic consolidation, both the bounding surface and the yield surface shrink to a line that is along the space diagonal. Because of this, no yielding takes place and therefore, no plastic deformations can occur. The material response is only elastic. If the sample experienced anisotropic consolidation (for example, the $K_{0}$ condition), then there would be a large magnitude of irreversible strains. This is because the stress path is passing various yield surfaces as it is loaded.

Poorooshasb (1989) explained three kinematic constraints of the yield surface. They are quoted as follows:

1. The control meridian, the axis of the yield surface and the space diagonal are at all stages coplanar.
2. As the state of stress changes from $\sigma$ to $\sigma+\delta \sigma$, the axis of the yield surface must move along the guide plane associated with $\sigma$.
3. The new yield surface must contain the new stress point representing the state of stress of $\sigma+\delta \sigma$.

## Plastic Potential

A material will yield if the stress point that is located on the yield surface moves outside the yield domain. This causes plastic deformation. The global plastic potential may be used to analyze the direction of the principal strain increment vector if the stress point is located on a control meridian. The local plastic potential can be derived from the global plastic potential and is the same along the control meridian. The local plastic potential surface cuts the $\theta=$ constant plane along a curve seen in b-b' in the Figure 2.6.8.

(a)

(b)


Figure 2.6.6. (a) Yield and Bounding Surfaces for Sacramento River sand, e=0.55 (b) Yield and Bounding Surfaces for Sacramento River Sand, e=0.75 (after Poorooshasb 1989).
(a)

(b)

Figure 2.6.7. (a) The two dimensional representation of the state space (b) Yield and conjugate surfaces simplified scheme (after Poorooshasb 1989).

 state space (after Poorooshasb 1989).

According to Poorooshasb et al. (1967), "the gradient of the irreversible component of the strain increment vector is independent of the gradient of the stress increment being a function only of the state of the element." Therefore, $\mathrm{d} v / \mathrm{d} \gamma=f(\tau, \sigma, e)$. The plastic potential function, $\psi$ serves to define the components of the strain increment vector. It is given by the following equation:

$$
\mathrm{d} \varepsilon_{\mathrm{ij}}^{p}=\frac{d \psi}{d \sigma_{i j}}<d \lambda>
$$

Eq. 2.6.3

The scalar $\mathrm{d} \lambda$ depends on the state of the element and its mode of yielding. Poorooshasb concluded that the plastic potential curves for a given value of e should form a family of geometrically similar curves. This can be seen in Figure 2.6.9.


Figure 2.6.9. Establishment of plastic potential curves from the inclination of plastic strain increments, with three values for e (after Poorooshasb 1967).

The flow rule is the relation between the plastic strain increments and the stresses.

$$
\begin{equation*}
\Delta \varepsilon_{\mathrm{ij}}^{p}=\Delta \lambda \frac{d g}{d \sigma_{i j}} \tag{Eq. 2.6.4}
\end{equation*}
$$

$\Delta \lambda$ is a constant. This equation shows the strain increments being proportional to the derivatives of the plastic potential, where the plastic potential according to Poorooshasb (1967) is defined by

$$
\begin{equation*}
g=\left(I_{1}\right)^{3}-\kappa_{2} I_{3} \tag{Eq. 2.6.5}
\end{equation*}
$$

The value of $\kappa_{2}$ determines the relative magnitudes of the plastic strain increments. $\Delta \lambda$ determines the absolute magnitudes.

Yielding can be defined and explained by taking a look at the state of an element. Poorooshasb et al. (1967) described this state as being represented by a point $P$ in the state space (also called state point). One can consider a sphere, which has a radius, $r$ that contains point P in its center. When something perturbs this sphere, it produces a small change in the state of the element. This change, the state increment vector, can be denoted $\delta$, such that $|\delta| \leq$ r. When the cause for this change is removed, a new position of the stress point is reached, P'. There are different categories that the element can be classified, depending on the position of $\mathrm{P}^{\prime}$. If $\left|\mathrm{PP}^{\prime}\right|>\mathrm{r}$, then the element is said to be unstable. However, if $\left|\mathrm{PP}^{\prime}\right| \neq 0 \leq \mathrm{r}$, then it is stable-yielding. Moreover, if $\left|\mathrm{PP}^{\prime}\right|=0$, then it is considered stable-nonyielding. If P has identical elements (identical states who have experienced similar loading histories), then loading of the elements will cause the state points to trace a curve (also referred to as a state path). This path would depend on the loading conditions. Figure 2.6 .10 shows these concepts in an illustration.


Figure 2.6.10. Illustration of yielding (after Poorooshasb 1967).

The yield surface for a sand element can be written

$$
f\left(\sigma_{\mathrm{ij}}, e\right)=c
$$

Eq. 2.6.7
c is a constant describing the loading history for a particular sand element. Poorooshasb et al. (1967) concluded that a simple formulation of the yield function $f$, of the form $f=\overline{\mathrm{c}} \eta$ ( $\overline{\mathrm{c}}$ is a function less than but close to 1) was consistent for the behavior of isotropic cohesionless granular materials composed of non-breakable hard particles. Figure 2.6 .11 shows a schematic picture of the intersection of a yield surface and of $\eta=$ constant.


Figure 2.6.11. Intersection of a yield surface and an $\eta=$ constant surface (after Poorooshasb et al. 1986).

## Yielding and Yield Surfaces from Triaxial Tests

Lade and Prabucki (1995) studied the plastic yield surface for soils in the post-peak softening regime. They were interested in the movement of the yield surface due to preshearing and wanted to model the soil behavior in the hardening and softening regimes near peak failure. The sand specimens were tested in triaxial compression using certain stress paths where they would be able to establish the yield surface, and then keep searching for another yield surface at another location in the stress space.

As mentioned in the explanation of yield surfaces, the yield surface (expressed in terms of the stress invariants) shows the locus at which the total plastic work is constant. The total plastic work (due to shear strains and volumetric strains) acts as the hardening parameter. It is used to delineate the location and shape of the yield surface. They developed a constitutive model for a single, isotropic yield surface that expressed a contour of constant plastic work, measured from the origin of stress. The isotropic yield surface is shaped as an asymmetric
teardrop with a pointed apex at the origin of the principal stress. This can be seen in Figure 2.6.12.

Looking at Figure 2.6.12, one can see that for an isotropic material, the yield surface is perpendicular to the hydrostatic axis, bending towards the origin, and crossing the failure surface at sharp angles. At these points outside the failure line, the stress level is greater than one. Lade and Prabucki also studied the effects of pre-shearing and found that following preshearing to peak failure, the yield surface in the region of lower confining pressures moved beyond the failure surface for normally consolidated sand. The sand showed softening beyond the yield surface, which became the failure surface of the region.


Figure 2.6.12. Pattern of yield surfaces for isotropic granular materials (after Lade and Prabucki 1995).

Yasufuku et al. (1991) also studied the yield characteristics of aniostropically consolidated sand using drained triaxial testing. They tested Aio sand under low and high stresses with different stress paths. They found that the overall shapes of the yield curves under low and high stress levels are similar and by using ellipses that are not symmetrical about the stress path during consolidation, the shapes of the yield surfaces can be drawn. They also found that the relationship between the tangent slopes of the yield curves and stress ratios can be expressed as a unique function of the applied stress ratio, no matter what proportional loading path history was used. They also proposed a yield function for anisotropically consolidated sand. They defined the yield point as the state of stress at a marked change in slope of each stress-strain curve. The yield stress was taken to be the point of maximum curvature, corresponding to the start of fully plastic deformation.

Figure 2.6.13 shows the method that Yasufuku et al. (1991) used which was based on the method used by Poorooshasb et al. (1967) and Miura and Yamamoto (1982). This method involves locating the yield point by an intersection of two simple straight lines. The developed yield curves can be seen on Figure 2.6.14. It is seen that the yield surface shape differ from each other. This difference seems to be related to the dependency of the failure line on the confining pressure. However, the general shape for anisotropically consolidated sand is approximately an elliptical one, which is not symmetrical about the stress path during consolidation.


Figure 2.6.13. Typical stress-strain curves for tests of Type A in the low stress level (after Yasufuku et al. 1991).



Figure 2.6.14. Experimental yield curves obtained from tests of Type A in the (a) low stress level, (b) high stress level (after Yasufuku et al. 1991).

Yielding and Flow of Sand in Torsion Shear Tests
Pradel et al. (1990) studied the influence of inherent anisotropy on the yielding and plastic flow of loose and dense Toyoura sand with a torsional shear apparatus. The loading path that was followed can be seen in Figure 2.6.15a and 2.6.15b.


Figure 2.6.15. (a) Loading path for the study of yielding (b) stress paths used for the determination of yield loci (after Pradel et al.1990).

When finding yield points, they experienced that there was not always a sharp change between the elastic and plastic response that was easily observed. Figure 2.6.16 shows the idealized stress-strain behavior during reloading where it is quite easy to get the yield point. However, when the plastic strain in the transition zone is too large, the derivation of the yield point can be subject to some errors and requires assumptions about the stress-strain behavior.


Figure 2.6.16. Idealized stress-strain behavior during reloading (after Pradel et al. 1990).

Pradel et al. (1990) concluded that yielding occurred when stresses reached a particular level of stress for a given direction of principal stresses. The yield loci obtained from different stress-strain curves were not identical, and all the yield loci defined smooth curves of elliptical shape that were independent of density. Pradel et al. (1990) also showed that the direction of principal plastic strain increment in sand during principal stress rotation is very much dependent on the stress increment direction. This response violates the postulate of uniqueness in flow of plasticity theory. Gutierrez et al. (1991) conducted stress probe tests under different states of stress and they found that the plastic strain increment direction depended not only the current state of stress but also on the direction that the stress increment was executed. Based on experimental observations, they saw that the flow of sand is dependent on the stress increment direction as well as on the level of shear stress and then proposed a plastic potential theory (shown in Figure 2.6.17). The new theory proposed to determine the plastic strain increment direction as the normal to the failure surface at the point of intersection of the failure surface and the stress increment vector extended.

Pradel et al. (1990) used a Torsion Shear Apparatus to study the effects of reorienting the major principal stress on the yield behavior of sand. However, they did this only for limited regions of the stress space. Their experimental data showed that there were combined effects of changes in both $\alpha_{\circ}$ and the intermediate principal stress parameter, b .

O'Kelley and Naughton (2009) studied the effects of different orientations of the major principal stress and different magnitudes of the $b$ parameter on the yield behavior of sand separately. The stress paths that they determined used the method of finding yield points and yield loci by Tatsuoka and Ishihara (1974). Figure 2.6 .18 shows the process.


Figure 2.6.17. Stress and plastic strain increment representation in the $\mathrm{X}-\mathrm{Y}$ stress space (after Gutierrez et al. 1991).


Figure 2.6.18. Stress probing to identify yield loci (after O’Kelly and Naughton, 2009).

Once isotropic consolidation was achieved to the same mean effective confining stress, the sand specimens were anisotropically consolidated to achieve different values of $\alpha_{c}\left(\alpha_{s}=0^{\circ}\right.$, $30^{\circ}, 60^{\circ}$, and $90^{\circ}$ ) and the b parameter ( $\mathrm{b}=0$ or increased from $\mathrm{b}=0$ to 0.5 ). This created a new fabric in the specimen. By increasing the major to minor effective principal stress ratio $\left(\sigma_{1}^{\prime} / \sigma_{3}\right)$ until the start of plastic straining, points on the yield surface were found. Once the yield point was established, the principal stress ratio was reduced and the specimen was reconsolidated in order to increase the $b$ parameter by 0.25 . Once the $b$ value was reached, the ratio was increased again to search for a second yield point. This allowed a yield locus to be established between the two points. This procedure was repeated to establish segments of the yield loci for different stress states for different specimens. By using this method, it was possible to study the effects of reorienting the major principal axis as well as changes in the magnitude of $b$ separately.

Figure 2.6.19(c) and 2.6.19(d) show the strain responses recorded during the load-unload and reconsolidation stages of the stress path (changes in the $\mathrm{R}^{\prime}$ (principal stress ratio) and the b value). The stress-strain responses seen in Figure 2.6.19(c) were approximately linear-elastic along the unload paths and along the initial phase of the reload paths. The strains during the reconsolidation stages were negligible. This shows that the soil fabric induced in the specimens by the end of anisotropic consolidation (after the second stage) would have remained largely intact for the remainder of the stress path. Near a yield point, the stress-
strain response became increasingly non-linear. Due to Pradel et al.'s (1990) suggestion that the yield points derived from the intersection of best fit lines to the initial pseudo-elastic and post-yield slopes provided better approximations of the yield loci for the MatsuokaNakai and Lade yield criterion, the yield points were identified by curve fitting. A part of the yield locus was drawn by joining the yield point established for higher values of the $b$ parameter back to the initial yield point.


Figure 2.6.19. Stress probing to determine yield points (after O'Kelly and Naughton 2009).

Figure 2.6.20 shows the experimental yield data found plotted on a three dimensional plot with the axes representing, $\alpha, b$, and the deviator stress, $q$, where

$$
\begin{equation*}
q=\sqrt{\left(\sigma_{1}^{\prime}-\sigma_{3}^{\prime}\right)^{2}+\left(\sigma_{1}^{\prime}-\sigma_{2}^{\prime}\right)^{2}+\left(\sigma_{2}^{\prime}-\sigma_{3}^{\prime}\right)^{2}} \tag{Eq. 2.6.7}
\end{equation*}
$$

The data shows that the magnitude of the deviator stress corresponds to the yield loci in a continuous manner. The deviator stress was dependent of the initial value of the $b$ parameter but was independent of $\alpha$, the rotation that happened during the anisotropic consolidation stage of the stress path. When looking at the b parameter, two trends can be seen. The first observation is for the stress paths that had re-oriented the major principal stress and had the intermediate principal stress parameter initially set at $b=0$. For this situation, the deviator stress at yield decreased by about $15-20 \%$ because of the increase from $b=0$ to $b=0.5$. For the stress paths that reoriented the major principal stress and had the $b$ originally set at 0.5 , the magnitude of the deviator stress at yield either remained constant or increased slightly due to the increase from $b=0.5$ to $b=1.0$.


Figure 2.6.20. Variation of deviator stress on yield loci (after O'Kelly and Naughton 2009).

The yield loci were normalized and compared to the Matsuoka-Nakai and Lade criteria. The yield criterion was a bit less for the onset of $b$ values $(0>b>0.5)$ and slightly over for $(0.5>b>1.0)$. Taking account the scatter in the results, they concluded that both criterion were satisfactory for the results (see Figure 2.6.21).


Figure 2.6.21. Experimental and theoretical values of points on yield loci (after O'Kelly, and Naughton 2009).

## Conclusion on Yield and Failure Surfaces

Research into yield surfaces first began by establishing the failure surface, which in the beginning was referred to as the yield surface. Experiments were performed to determine the critical void ratio and ultimate surface, to find the state boundary surface, then bounding surface and the yield surface. Certain techniques have been developed in order to successfully find the yield surfaces of sands and clays, however, construction of the yield surfaces often requires tests in which unconventional stress paths are created on the soil. In some cases, yield points have to be approximated. Loading of soils often changes the soil properties, making it difficult to reach different yield points belonging to the same yield
locus, while the soil retains the same properties (i.e. void ratio, anisotropy). For clays, ellipses have been used as yield surfaces. For sands, more complex yielding behavior has been seen and a wide range of shapes has bee used to show the yielding of sands. Many triaxial tests, alongside fewer torsion shear tests have been performed in order to study yield surfaces. Research still continues on studying the effects of anisotropy and principal stress direction for cross-anisotropic sand deposits.

## 3. Triaxial Compression and Extension

### 3.1 Tests Material Used

In this experimental program, all triaxial, true triaxial and torsion shear tests were conducted on Fine Nevada Sand. The sand used is mostly made up of Silica Dioxide, $\mathrm{SiO}_{2}$, i.e. quartz. X-ray diffraction was performed on the sand and a complete list of the chemical make-up of the sand is presented in Table 3.1.1. The graph format showing the wave intensity is shown in Appendix A.

Table 3.1.1. CUA Vitreous State Laboratory X-Ray Diffraction Analysis on Fine Nevada Sand

| Oxide | $\mathbf{w t \%}$ |
| :---: | :---: |
| $\mathrm{Al}_{2} \mathrm{O}_{3}$ | 1.67 |
| CaO | 0.033 |
| CoO | 0.01 |
| $\mathrm{Cr}_{2} \mathrm{O} 3$ | 0.006 |
| CuO | 0.003 |
| $\mathrm{Fe}_{2} \mathrm{O}_{3}$ | 0.073 |
| $\mathrm{~K}_{2} \mathrm{O}$ | 0.157 |
| MgO | 0.058 |
| $\mathrm{Na}_{2} \mathrm{O}$ | 0.11 |
| P | 0.009 |
| S | 0.01 |
| $\mathrm{Sb}_{2} \mathrm{O}_{3}$ | 0.017 |
| $\mathrm{SiO}_{2}$ | 97.73 |
| $\mathrm{TeO}_{2}$ | 0.028 |
| $\mathrm{TiO}_{2}$ | 0.051 |
| $\mathrm{~V}_{2} \mathrm{O}_{5}$ | 0.023 |
| $\mathrm{ZrO}_{2}$ | 0.006 |

The specific gravity was also determined in the laboratory through the liquid submersion technique. A detailed description of the procedures can be found in Germaine and Germaine (2009). In order to account for variations associated with temperature, a volumetric flask was calibrated. Distilled water was also used to ensure accuracy in the results. Calculations are provided in Appendix B. The calculated specific gravity for the Fine Nevada Sand used in the experiments was 2.651 .

The minimum void ratio was determined by air-pluviating a predetermined amount of sand into a graduated cylinder. This was performed very slowly by dropping a spoonful of sand at a time from a height of 40 cm . This ensured that the sand grain structure could reach its densest state. The volume was then measured in order to determine the minimum void ratio. The $\mathrm{e}_{\text {min }}$ was calculated to be 0.507 . The maximum void ratio was determined by sealing the top of the graduated cylinder and slowly turning it upside down and right side up. This was performed several times to ensure that a constant maximum void ratio was achieved. The $\mathrm{e}_{\max }$ was calculated to be 0.771 . The calculations and test data are presented in Appendix C.

The grain size distribution was attained by performing a sieve and hydrometer analysis on the sand. Fine Nevada Sand was sieved through a 0.850 mm opening (Sieve No. 20) to ensure that no larger sand grains or particles were tested for all triaxial, true triaxial and torsion shear tests. The grain size distribution curve is shown in Figure 3.1.1. The sieve and hydrometer data can be found in Appendix D.

It should be noted that the sand used in the triaxial and true triaxial tests was not reused. However, with the torsion shear tests, it was necessary to reuse the sand because over 8800 grams of sand were used in each experiment. As the specimens were sheared under low confining pressures, the sand did not experience any crushing. After each test, the sand was dried in an oven and then separated into containers to ensure that it was not used again until the whole supply had been used up. A sieve analysis on the used sand confirmed that the grain size distribution did not change and that all tests were comparable. Table 3.1.2 summarizes the properties of the Fine Nevada Sand used in the experimental program.


Figure 3.1.1. Grain Size distribution curve including hydrometer analysis on Fine Nevada Sand.

Table 3.1.2. Properties of Fine Nevada Sand.

| Specific gravity, Gs | 2.651 |
| :--- | :---: |
| Min. void ratio, emin | 0.507 |
| Max. void ratio, emax | 0.771 |
| D10 (mm) | 0.114 |
| D30 (mm) | 0.169 |
| D60 (mm) | 0.238 |
| Coefficient of uniformity, Cu | 2.076 |
| Coefficient of curvature, Cc | 1.049 |

### 3.2 Triaxial Compression and Extension Tests

A total of ten triaxial compression and one conventional extension tests were performed on Fine Nevada sand in order to attain the parameters that are needed and used in Lade's failure criterion (Lade 2008). For these specific tests, the true triaxial apparatus developed by Lade and Duncan (1973) was used. Four tests were performed with horizontal bedding planes, $\alpha=0^{\circ}$. An additional six tests were performed on specimens that had vertical bedding planes, $\alpha=90^{\circ}$. One conventional extension test was performed as well. The details and results are presented in the sections that follow. As a point of reference, Figure 3.2.1 shows the Mohr's circle for triaxial compression and extension tests.


Figure 3.2.1. Mohr's Circle for Triaxial Compression and Extension Tests.

## Triaxial Tests Set-up

Lade and Duncan's (1973) true triaxial apparatus (described in Section 2.2) was slightly modified to perform traditional triaxial tests. The horizontal loading system was removed and any openings were plugged to create a conventional set-up. The confining
pressure, which acted on all sides of the specimen, was applied by air pressure through a fitting on the base plate and was measured directly from the cell with a pressure transducer. The air pressure inside the chamber created the minor principal stress, $\sigma_{3}$. The vertical load was applied to the specimen through the top cap and base. It was transferred through a stainless steel piston rod that touched a ball placed in a ball socket in the top cap load cell. The flat end of the piston was used to touch the ball, ensuring that the point load was always centered on the ball. The deviator stress from this load with the addition of the confining pressure, $\sigma_{3}$, provided the major principal stress, $\sigma_{1}$. A schematic of the set-up (without the loading plates) can be seen in Figure 3.2.2.


Figure 3.2.2. Triaxial Set up (Modified for Sand) after Lade (1978).

Two load cells were used during conventional triaxial testing. The interior load cell embedded in the top cap was made of beryllium copper. It was supplied with four strain gages glued to the surface and connected in a full bridge. Details can be found in Lade (1978). The cable with the wires connected to the strain gages passed through a sealed opening at the base of the cell and was connected to a Vishay Measurement Group P-3500 Strain Indicator Box. The gage factor was set appropriately and any time that the load cell was taken apart, it was recalibrated. The exterior load cell was a Futek LCF-450 and has a capacity of 226.8 kg . The Futek LCF-450 was connected to a Futek IPM500 signal conditioned digital display. Both load cells were calibrated independently of each other and independent measurements were taken during all tests. Measuring the vertical load by two load cells served as a way to confirm that the readings were consistent and correct.

Filter stones and drainage lines on the top and bottom caps allowed for water to flow to the volume change device while the saturated specimen was isotropically consolidated and sheared. In order to measure the saturation of each sample and obtain the Skempton B-Value, the volume change device had a 3-way valve that was connected to a 3447 kPa pressure transducer and a Micro Meters signal conditioned digital display. This 3-way valve made it possible to measure the cell pressure as well as the back pressure on the specimen. A schematic of the volume change device that was used is shown in Figure 3.2.3.


Figure 3.2.3. Schematic of Volume change and Pressure Measuring Device (after Kirkgard 1988).

The triaxial set-up was placed on a Humboldt Master Loader HM-3000. A strain rate of $0.1 \mathrm{~mm} / \mathrm{min}$ was used for the set of tests described in this section.

## Specimen Preparation

All specimens were approximately 7.6 cm in length, 7.6 cm in width and 19 cm in height. A void ratio, e, of 0.53 was targeted for each test. This corresponds to $91 \%$ relative density. This void ratio and relative density was chosen in order to create a dense specimen to see the effects of shear banding. In general, two factors affect the void ratio when pouring the sand: drop height and rate of sand pouring. In order to ensure the same void ratio for each specimen, the specimen was poured into a funnel with a small tube inside the mouth of the funnel ensuring a constant flow rate of sand. The sand was dropped into a mold that was created of block pieces. These pieces had a thin layer of grease between each piece to ensure that no leaks were created. A bottom copper plate with a hole that was used during the saturation process was also part of the mold. The top face of the mold was left open while the sand was rained in. The funnel was raised as the sand fell to maintain a drop height of 35 cm . It was determined empirically that a drop height of 35 cm at the flow rate established would create the desired void ratio. As the sand was deposited, the funnel was carefully raised to ensure even bedding planes into the previously assembled mold. A picture of the assembled mold is shown in Figure 3.2.4.


Figure 3.2.4. Picture of mold used to create frozen true triaxial specimens.

For tests with $\alpha=0^{\circ}$, the mold would stand vertically and would be open at the top. The mold had a base plate made of copper and its sides were made of plastic pieces (PVC). Rods going through the plastic pieces were fastened and ensured that the mold would not be loose or break apart. The interior of the mold was lightly greased with vacuum grease before depositing the sand in order to easily slide out the specimen. For tests with $\alpha=90^{\circ}$, the mold was set on its side and the topside face was left open where the sand could be deposited. Once the sand was deposited, special care was taken to not move the specimen until after it was completely saturated. It was not moved in order to make sure that the bedding planes
were not shifted in any way. When the sand was completely deposited, the last and open side of the mold was closed with the plastic pieces and secured with additional rods. On whichever side was facing up, a special piece, which included a fitting, was used. This was done so that any water that expanded during freezing could exit through the fitting and not affect the void ratio of the specimen. After it was closed and sealed, gaseous $\mathrm{CO}_{2}$ was slowly passed from the bottom up through the specimen, pushing any trapped air out of the fitting in the uppermost plastic plate. Deaired water was then slowly passed through the specimen until it was completely saturated. Any remaining gaseous $\mathrm{CO}_{2}$ would dissolve in the water, because Henry's coefficient of solubility is about $1.0 \mathrm{vol} / \mathrm{vol}$ at room temperature. When the $\mathrm{CO}_{2}$ saturation process was complete, the specimen was carefully moved to a freezer. The specimen with the copper plate at the bottom was set on an aluminum block inside the freezer to facilitate the freezing process. All specimens were left overnight in the freezer to ensure that they were completely frozen. Any excess water would escape from the fitting located on the top plastic piece of the mold and therefore, the void ratio of the specimen was not changed during the freezing process.

Once the specimen was frozen, it was ready to be set up in the triaxial cell. Special sized membranes that had been formed by dipping a mandrel into fluid latex rubber and molded to the correct dimensions were used for all triaxial and true triaxial tests. All of the membranes were approximately 0.03 cm thick. Lubricated ends were placed on the top and bottom caps to ensure that shear stresses between the specimen and end plates did not develop. Two sheets $(0.01 \mathrm{~cm}$ thick) were used. A thin layer of vacuum grease was placed on the aluminum
side of the top and bottom cap. Then, one sheet was placed and then another thin layer of vacuum grease was spread between the top and final sheet. Two sheets were found to be adequate in the testing program and allowed for enough lubrication to ensure that the specimens remained as vertical as possible at peak failure. A bulged specimen can affect the direction of the measured shear bands and this is why special care was taken to ensure the lubricated ends worked correctly.

The mold was then carefully disassembled and the frozen specimen was placed on the bottom cap. The membrane was stretched over the specimen and secured with two O-rings at each of the top and bottom cap. The drainage lines connected to a vacuum held at 25 kPa effective confining pressure. The vacuum was applied through a bubble chamber and therefore, any leaks in the membrane would be indicated. Liquid latex rubber was then painted on to the sample to ensure that no holes and leaks existed. Once the liquid latex rubber dried, the Lucite cell wall and the top lid of the triaxial cell were installed and secured to the base plates with six tie rods. The piston was also inserted through the top lid and rested lightly on the ball placed in the ball socket of the interior load cell. The specimen was then left overnight to ensure that all ice had thawed completely.

Once thawed, deaired water was filled into the outer cell and a confining pressure of 25 kPa was applied to the cell while the vacuum was removed. For tests using the inner top cap load cell, only air pressure was used, as the load cell was not water proofed. Deaired water was
again introduced into the specimen in order to saturate the drainage lines. Once saturated, the top and bottom drainage lines were hooked up to the volume change device and back pressure was added to the specimen, keeping the effective confining pressure at 25 kPa . A back pressure of 100 kPa helped in saturating the specimen. The saturation was tested after the back pressure was left on the specimen for several hours. The specimen was then isotropically consolidated and sheared.

## Calibration of Measurement Devices

Several measuring devices had to be calibrated to ensure accurate readings throughout testing. First, in order to ensure that all pressure gages were reading accurately, the gages were hooked up in parallel to the air supply line. Various pressures were applied and the pressure gages were each connected to ensure that the same readings were displayed. No problems were experienced with the pressure gages. A Mitutoyo digital dial gage was used in order to measure axial deflections. The accuracy of the dial gage was tested with a Mitutoyo digital micrometer. Both readings on the dial gage and micrometer were in unison. The volume change device was also calibrated in order to convert the readings recorded into actual volume. The calibration was done by opening one side of the drainage valve and letting the water drain into a graduated cylinder. By recording the measured water and the readings on the volume change device, a calibration constant was attained.

In order to calibrate both the internal embedded top cap load cell and the external load cell, a proving ring was used. Although the manufacturer of the proving ring provided a calibration
sheet, the calibration was double checked in the laboratory prior to the calibration of the load cells. By using a hanger system set-up (shown in Figure 3.2.5), the deflection on the proving ring was measured as weights were added to the hanger. The additional weight of the system was excluded and the deflection and load were calibrated. It was found that the manufacturer's proving ring calibration sheet corresponded to the experimental results.


Figure 3.2.5. Calibration of Proving Ring Set-up.

Once the proving ring was confirmed to be accurate, the external load cell was calibrated in compression with the proving ring by setting it up in a loading machine. With the load cell and proving ring in place, the applied load was calculated with the proving ring calibration sheet and the corresponding reading on the external electronic load signal measuring device. With these readings, the calibration curve was attained. A similar procedure was employed for the embedded top cap load cell. However, if at any time, the load cell was unscrewed from the top cap, the calibration was checked and changed as needed. Any changes made to the top cap can affect the calibration of the embedded load cell and therefore, a new calibration was performed. For the one extension test performed, the load cell was also calibrated in extension.

## Corrections and Stress-Strain Calculations

The cross sectional area was corrected during consolidation and shearing of the specimen. This correction assumed that the specimen deformed as a right prism. Lubricated ends were placed on the specimen top and base caps to keep the shape of the specimen as a rectangular prism as much as possible, minimizing bulging. The area correction is given by equation 3.2.1.

$$
\begin{equation*}
A_{c}=A_{o}\left(\frac{1-\varepsilon_{v}}{1-\varepsilon_{1}}\right) \tag{Eq. 3.2.1}
\end{equation*}
$$

where $A_{c}$ is the current area of the specimen, $A_{o}$ is the initial area of the specimen, $\varepsilon_{v}$ is the volumetric strain and $\varepsilon_{1}$ is the axial strain.

It is well established that membranes provide resistance to the applied loads during specimen shearing. Therefore, it may be necessary to account for this resistance.

According to Degroff et al. (1988), a Poisson's ratio of 0.5 and a Young's modulus of 1400 kPa can be used for membrane corrections. Assuming the membrane is volumetrically incompressible, the following formula can be used to attain the axial load carried by the membrane:

$$
\begin{equation*}
F_{m}=\frac{E_{m} \varepsilon_{m} A_{m}}{\left(1-\varepsilon_{m}\right)} \tag{Eq. 3.2.2}
\end{equation*}
$$

where $\mathrm{F}_{\mathrm{m}}$ is the axial load carried by the membrane, $\mathrm{E}_{\mathrm{m}}$ is the elastic modulus which is 1400 $\mathrm{kPa}, \varepsilon_{\mathrm{m}}$ is the axial strain, and $\mathrm{A}_{\mathrm{m}}$ is the initial cross-sectional area of the membrane.

The membranes used for triaxial testing were 0.03 cm thick and 10 cm in diameter. The difference in friction angle at $5 \%$ axial strain was less than $.09^{\circ}$ and therefore, the correction was found to be negligible.

In the only conventional triaxial extension test performed, the piston was rigidly screwed into the piston top cap. Therefore, the chamber pressure applied to the specimen was absent across the area of the piston that is going out from the top cap. This caused an uplift force and a reduction of axial stress on the specimen. The piston uplift can be calculated by

$$
\begin{equation*}
F_{p}=\sigma_{3} A_{p}=\left(u_{b}+\sigma_{3}^{\prime}\right) A_{p} \tag{Eq. 3.2.3}
\end{equation*}
$$

where $F_{p}$ is the piston uplift and $\sigma_{3}$ is the chamber pressure which equals the sum of the back pressure, $u_{\mathrm{b}}$ and the effective confining pressure, $\sigma_{3}$. The weight of the load cell as well as the weight of the top cap was also included in calculating the vertical forces acting on the specimen during triaxial extension.

With the lubricated ends on top and bottom caps, uniform deformations developed well into the post-peak region. Therefore, the effect of non-uniform deformations did not affect the strength of the sand and no corrections were made. The bushing on the true triaxial machine allowed for the piston to slide easily up and down and therefore, no friction on the piston was accounted for in the calculations.

Membrane penetration effects in flexible membranes can sometimes cause a systematic error in calculations and must be checked. Fine Nevada sand was chosen in order to minimize membrane penetration. For drained tests, membrane penetration affects only the volumetric
strain and does not affect the effective stress state of the sand. In general certain aspects affect membrane penetration. Particle size is one of the most important factors. The mean grain size, $\mathrm{D}_{50}$ is assumed to be representative of the soil being used. Particle shape and the soil's relative density have a minor affect per Baldi and Nova (1985). The effective lateral stress, $\sigma_{3}{ }^{\prime}$ can play a large role for membrane penetration. Baldi and Nova (1985) derived an equation to measure the effects of volume change due to membrane penetration considering the average diameter grain size, effective confining pressure, diameter of the specimen, Young's Modulus and the thickness of the membrane. The equation is as follows:

$$
\begin{equation*}
V_{m}=\frac{d_{g}}{D} V_{0}\left[\frac{\sigma_{3}^{\prime} d_{g}}{E_{m} t_{m}}\right]^{1 / 3} \tag{Eq. 3.2.4}
\end{equation*}
$$

where $\mathrm{V}_{\mathrm{m}}$ is the membrane penetration volume; $\mathrm{d}_{\mathrm{g}}$ is the average diameter grain size (taken from $\mathrm{D}_{50}$ in a grain size distribution analysis); D is the initial diameter of the specimen; $\mathrm{V}_{0}$ is the initial volume; $\sigma_{3}{ }^{\prime}$ is the effective confining pressure; $\mathrm{E}_{\mathrm{m}}$ is Young's modulus for the membrane and $\mathrm{t}_{\mathrm{m}}$ is the thickness of the membrane.

The $\mathrm{D}_{50}$ of the Fine Nevada Sand used in the experimental program was calculated to be 0.022 cm (see Figure 3.1.1). Since the specimens were square, an equivalent diameter for the area of the specimen was calculated to be 8.56 cm . The initial volume was $1097.44 \mathrm{~cm}^{3}$. The final effective confining pressure for most of the true triaxial tests was 50 kPa . As previously noted, the thickness of the membranes were 0.03 cm and Young's modulus was taken as 1400 kPa . With these parameters, the $\mathrm{V}_{\mathrm{m}}$ is $0.41 \mathrm{~cm}^{3}$, where $\Delta \sigma_{3}{ }^{\prime}=25 \mathrm{kPa}$. Depending on the total
volume change for different tests, this value varies from 1 to $3 \%$ of the total volume change. Thus, membrane penetration effects were considered to be negligible for triaxial and true triaxial tests.

Although all efforts were made to have specimens with the same void ratio, in some cases, it was not possible. Any slight deviation in method during air pluviation can affect the amount of sand that went into the molds during the specimen preparation. Nonetheless, an equation relating friction angles to void ratio allows for all friction angles to be calculated according to a certain void ratio. By using the equation,
$\mathrm{e} \bullet \tan \varphi=$ constant
where e is the void ratio, $\varphi$ is the friction angle and c is a constant, corrected friction angles could be attained. All friction angles were corrected to a void ratio of 0.53 .
3.3 Triaxial Compression Tests with $\alpha=0^{\circ}$ and $\alpha=90^{\circ}$

Lade (2007) suggests that is preferable to have three triaxial compression tests on vertical specimens and three triaxial compression tests on horizontal specimens in order to be able to determine the parameters needed for Lade's failure criterion for soils (see Equation 2.4.4). These three parameters are: $\eta_{0}, \mathrm{~m}, \Omega_{1}$. Therefore, drained compression tests were performed at different confining pressures in both the first and third sector, i.e. on vertical and horizontal specimens.

Preliminary tests were run as trial experiments while setting up the triaxial apparatus. These tests experienced serious problems, not producing reliable results and are therefore not presented. However, Tests 1, 2, 3 and 4 are summarized in Table 3.3.1. Test 1 is also Test TT\#1 that will be presented in Chapter 5.

Table 3.3.1. Triaxial Compression Tests with $\alpha=0^{\circ}$.

| Date | Test | $\alpha$ | $\sigma_{3}^{\prime}$ | e | b-value | $\sigma_{1} / \sigma_{3}$ |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| $3 / 21 / 11$ | $1, \mathrm{TT} \# 1$ | $0^{\circ}$ | 50.00 | 0.522 | 0.00 | 5.28 |
| $4 / 1 / 11$ | 2 | $0^{\circ}$ | 25.00 | 0.537 | 0.00 | 6.29 |
| $4 / 4 / 11$ | 3 | $0^{\circ}$ | 130.00 | 0.542 | 0.00 | 4.57 |
| $4 / 5 / 11$ | 4 | $0^{\circ}$ | 70.00 | 0.542 | 0.00 | 4.84 |

The stress strain and volume change curves are presented in Figure 3.3.1 and Figure 3.3.2, respectively.

As the confining pressure increases, the friction angle decreases. With lower confining pressure, there is also more total dilation. The rates of dilation are all very similar in the four tests. Tests 1, 2, 3 and 4 had horizontal bedding planes (where $\alpha=0^{\circ}$ ) and are located in the first sector in the octahedral plane.


Figure 3.3.1. Triaxial Compression Stress-Strain Curve for $\alpha=0^{\circ}$ Tests.


Figure 3.3.2. Triaxial Compression Volumetric Strain for $\alpha=0^{\circ}$ Tests.

Table 3.3.2. Compression Tests with $\alpha=90^{\circ}$.

| Date | Test | $\alpha$ | $\sigma_{3}^{\prime}(\mathrm{kPa})$ | e | b -value | $\sigma_{1} / \sigma_{3}$ |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| $11 / 19 / 10$ | 5 | $90^{\circ}$ | 25.00 | 0.533 | 0.0 | 4.47 |
| $11 / 24 / 10$ | 6 | $90^{\circ}$ | 75.00 | 0.538 | 0.0 | 4.24 |
| $12 / 4 / 0$ | 7 | $90^{\circ}$ | 130.00 | 0.532 | 0.0 | 4.01 |
| $2 / 25 / 11$ | $8, \mathrm{TT} \# 13$ | $90^{\circ}$ | 50.00 | 0.540 | 0.0 | 4.13 |
| $1 / 17 / 11$ | $9, \mathrm{~W} 1$ | $90^{\circ}$ | 101.00 | 0.534 | 0.0 | 4.15 |
| $12 / 21 / 10$ | $10, \mathrm{~A} 7$ | $90^{\circ}$ | 99.00 | 0.532 | 0.0 | 4.06 |

Six additional tests were performed with vertical bedding planes (where $\alpha=90^{\circ}$ ). Table 3.3.2 summarizes these tests. Test 8 is also presented as Test TT\#13 in the True Triaxial Tests in Chapter 5. Tests 9 and 10 are also presented as Tests W1 and A7 in Chapter 4, respectively. Since these tests were typical compression tests with $\alpha=90^{\circ}$ bedding planes, the data was also used to supplement the other tests in determining the parameters.

The stress strain and volume change curves are presented in Figure 3.3.3 and Figure 3.3.4, respectively.


Figure 3.3.3. Triaxial Compression Stress-Strain Curve for $\alpha=90^{\circ}$.


Figure 3.3.4. Triaxial Compression Volumetric Strain for $\alpha=90^{\circ}$ Tests.

Tests 8(TT\#13), Test 6, Test 9(W1) and Test 10(A7) have very similar stress-strain behavior even though they have a difference of up to 50 kPa in effective confining pressure. The same trend is seen in the $\alpha=90^{\circ}$ tests as in the $\alpha=0^{\circ}$ tests, i.e. decreasing friction angles with increasing confining pressures. In general, greater dilation is seen with lower confining stress for the $\alpha=90^{\circ}$ tests. When comparing and analyzing the tests with horizontal versus vertical bedding planes, it was seen that tests with vertical bedding planes $\left(\alpha=90^{\circ}\right)$ have lower stress ratios and friction angles than those with horizontal bedding planes $\left(\alpha=0^{\circ}\right)$. At $\sigma^{\prime}{ }_{3}=25 \mathrm{kPa}$, the friction angles change from $46.6^{\circ}$ to $39.4^{\circ}$ (horizontal to vertical). At $\sigma^{\prime}{ }_{3}=130 \mathrm{kPa}$, the friction angles change from $39.9^{\circ}$ to $36.9^{\circ}$. Thus, the cross-anisotropy is more pronounced at low, rather than high, confining pressures. This shows the cross-anisotropy of the soil and is consistent with results from previous studies that have been presented in Chapter 2.

### 3.4 Conventional Extension Test

One extension test (Test 11, W5) was performed where the confining pressure was kept constant (at 101 kPa ) and the specimen was extended instead of traditionally compressed. This specimen had vertical bedding planes where $\alpha=90^{\circ}$. The set-up for this test changed slightly from the previous triaxial compression tests that were performed. In this particular test, a load cell was hung from the triaxial loading machine by a metal rod. Some movement was allowed so that the rod was freely able to rotate to minimize any moment on the specimen. The rod connected to the top of the load cell, was recalibrated for extension. On the bottom side of the load cell, an adaptor piece connected the load cell to the piston. The load cell located in the top cap was removed and a special piece where the piston could be
screwed on was fastened into the top cap. The triaxial set up was clamped down at three points to the loading machine to ensure that the cell did not get lifted off the pedestal of the loading machine. The same procedure was performed for the freezing process and set-up of the specimen as was described previously. However, water pressure was used instead of air pressure for the confining cell pressure and a slower strain rate of $0.05 \mathrm{~mm} / \mathrm{min}$ was used. The stress strain and volume change curves are presented in Figure 3.4.1 and Figure 3.4.2, respectively. Test 11 data is also presented in Chapter 4 as Test W5.

The weight of the load cell, the top cap and the vertical uplift on the piston were considered in calculating the total load acting on the specimen. There was no visible necking at the end of the test and shear bands along the top half of the specimen developed at about $1.3 \%$ strain. As can be seen from Figures 3.4.1, failure clearly occurred around $1.3 \%$ axial strain and the specimen had a friction angle of $35.9^{\circ}$. Conventional extension tests performed in this manner are notoriously unreliable and inherently unstable due to the concentration of stresses along the weakest part of the specimen. Lade and Wang (2012) describe and explain attempts made to achieve uniform stresses and strains on sand specimens in conventional extension tests.


Figure 3.4.1. Triaxial Extension Test 11,W5 Stress-Strain Curve for $\alpha=90^{\circ}$.


Figure 3.4.2. Triaxial Extension Test 11,W5 Volumetric Strain for $\alpha=90^{\circ}$.


Figure 3.4.3. Triaxial Compression and Extension Stress-Strain Curve for $\alpha=90^{\circ}$ for Tests 5-11.

Test 11(W5) showed substantially different stress-strain behavior and strength as compared to the previously discussed triaxial compression tests with $\alpha=90^{\circ}$ (Tests 5-8). Figure 3.4.3 shows that Test 11 has lower effective stress ratio at failure corresponding to lower friction angle than any of the corresponding triaxial compression Tests 5-10. Because of the conclusion from many investigators (such as Roscoe et al. (1963), Yamamuro and Lade (1995), Lade et al. (1996)) in which they explain that conventional triaxial extension tests results cannot be relied upon to produce accurate and high quality results, only Tests 5-8 data
will be used to determine the parameters necessary for the cross-anisotropic failure criterion for Fine Nevada Sand.

### 3.5 A Comment for Specimen B-value and Saturation

When performing triaxial tests, the $B$-value, which measures the saturation of the specimen, was determined. This was done once the specimen was completely thawed and a back pressure of 100 kPa had been applied for a minimum of two hours. At this point, the cell and specimen pressure readings on the transducer (set up on the volume change device) were recorded. Then, the valve to the volume change device was closed (creating an undrained condition) and the cell pressure was raised by about 5 kPa . The new readings were recorded and then the cell pressure was lowered to the original cell pressure. The drainage valve was re-opened. The B-value was determined using the following equation:

$$
\begin{equation*}
B=\frac{\Delta u}{\Delta \sigma_{3}} \tag{Eq. 3.5.1}
\end{equation*}
$$

where $\Delta \sigma_{3}$ is the imposed change in all around isotropic cell pressure and $\Delta \mathrm{u}$ is resulting change in pore pressure obtained from undrained conditions.

A B-value of 1.0 shows a fully saturated soil. However, a number of factors strongly affect the value of B and these factors may cause a B lower than 1 . In the triaxial compression tests, as well as the tests performed in the true triaxial apparatus, B-values may be lower than unity
when calculated. These values measured in the initially frozen specimens ranged from 0.2 to 0.8 for different tests. As described previously, the specimens were saturated fully prior to freezing. When thawed, they were placed under vacuum. During this procedure, it is possible that some small amounts of water could have been sucked out into the bubble chamber that was hooked up to the vacuum. De-aired water was introduced into the sample to get rid of any air that might have been trapped in the specimen, as well as to fully saturate the lines leading to the volume change device. During this procedure an air bubble might have stayed in the specimen and not dissolved under the 100 kPa back pressure. This would definitely affect the calculated B value. In order to determine the degree of saturation of the specimens, a slightly different procedure was performed.

The B-value was calculated in the same manner as was previously described but instead of lowering the cell pressure back down to the original value, the drainage valve was opened and the volume change that occurred was measured. The saturation of the specimen could then be calculated by using the equation:
$S=\left[1-\left[\frac{u_{2}}{n}\left(c_{d}+\frac{f_{m}}{V_{o}}\right)\left(\frac{1+B}{B}\right)\right]-\frac{f_{s}}{V_{o}}\right]$
where $\mathrm{u}_{2}$ is the pore pressure measured after increasing the cell pressure, n is porosity of the sand, $\mathrm{c}_{\mathrm{d}}$ is the volume compressibility of the soil skeleton, $\mathrm{V}_{0}$ is the initial volume of the specimen, $B$ is the measured $B$-value (from Eq. 3.6.1) and $f_{s}$ is the flexibility of the pore
pressure measuring system. Tubings, valves, etc. introduce factors that affect $f_{s}$. The measured volume change part of Eq. 3.5.2 can be substituted as follows:
$\left(c_{d}+\frac{f_{m}}{V_{o}}\right)=\frac{\Delta V_{\text {meassured }}}{V_{o} \Delta \sigma_{3}}$
Eq. 3.5.3

The porosity of Fine Nevada Sand was calculated to be 0.35 and $\mathrm{f}_{\mathrm{s}}=5 \times 10^{\wedge}-7 \mathrm{~cm} / \mathrm{kPa}$ was used to determine the saturation of the specimens after attaining the B-value. All specimens had a saturation of over $96 \%$, which shows that the specimens were sufficiently to produce reliable test results.

### 3.6 Conclusion

In conclusion, ten drained triaxial compression test and one drained triaxial extension tests were performed in order to attain certain parameter data needed for modeling. Through a special preparation, saturation and freezing technique, tests were performed with both vertical and horizontal bedding planes. The triaxial compression tests, performed at increasing confining pressures, all showed typical stress-strain and volumetric change behavior. Triaxial compression tests with $\alpha=90^{\circ}$ all showed very similar friction angles. One traditional extension test performed where $\alpha=90^{\circ}$ failed at a very short strain to failure and lower friction angle.

## 4. Triaxial Compression Tests with Varying Confining Pressure

### 4.1 Introduction and Background

Section 3.2 and 3.3 described Tests 1-8 on specimens with both horizontal $\left(\alpha=0^{\circ}\right)$ and vertical $\left(\alpha=90^{\circ}\right)$ bedding planes. When studying anisotropy, the difference in friction angle for the specimens with vertical bedding planes under different confining pressures was not that large, only $3.0^{\circ}$ compared to a $7.2^{\circ}$ difference for tests with horizontal bedding planes.

A possible explanation for the small difference in friction angle may be found in the amount of strain to failure of the specimens. Tests 5, 6 and 7 failed between 5 and $6 \%$ axial strain. It was hypothesized that the possible rotation and movement of sand grains during this large amount of strain could have allowed the sand grains to rearrange themselves before failure and therefore, the effects of anisotropy could appear to be less significant. In order to test this hypothesis, tests with varying stress paths and consequent varying strain-to-failure were performed.

In the literature, almost no previous research was found on tests performed with varying stress paths. Only theoretical and numerical results have been presented by $\operatorname{Ng}(2004,2005)$. Ng (2005) studied the behavior of granular material made up of different densities subjected to different stress paths using numerical models. The discrete element method was used to simulate experiments on specimens that were made up of ellipsoid particles.

Different densities were used to show the air pluviation method for a specimen in a rectangular prism and then different stress paths were employed to study the behavior and strength of the specimens.

Ng (2005) found that the axial compression simulation resulted in the lowest shear strength. The plane strain simulation had the highest friction angle for all samples and the difference between the plane strain and axial compression strengths increased with the density of the sample. He also observed that the dilatancy component of shear strength depended on the mean stress at failure. A greater dilatancy component was obtained when there was a large difference between the mean stress at failure and the initial mean stress.

With only this limited numerical research available, an experimental program was designed to study the effects that the stress path may have on the shear strength of soil. In the sections that follow, the experimental program will be described and the results will be presented.

### 4.2 Experimental Program

With the same triaxial apparatus that was used for the tests described in Chapter 3, a series of 13 tests were performed on tall prismatic specimens with vertical $\left(\alpha=90^{\circ}\right)$ bedding planes. The specimens had the same dimensions as in the previous tests described $(7.6 \mathrm{~cm}$ in length, 7.6 cm in width and 19 cm in height). The deposition, freezing and set-up of the specimens were also done the same way as previously presented in Section 3.1. The experimental program can be divided into two main categories: 1) 8 tests performed with
confining pressure supplied by compressed air and 2) 5 tests performed with confining pressure supplied by water. Stress paths with different directions in the triaxial plane were employed for each test. The stress path employed for each specimen was varied by changing the confining pressure in the triaxial cell.

In order to accurately change the confining pressure in the triaxial cell, the cell pressure (whether air or water) was measured by a pressure transducer, which was hooked up to a digital display. This pressure transducer and display was also used in calculating the saturation of the specimen. A pressure regulator was manually regulated to certain pressures and the corresponding readings on the display were recorded. With these recordings, a calibration curve was attained and the confining pressure could be changed according to the desired stress path.

A program was set up to calculate the real time mean stress on the specimen, taking account of vertical load, axial strain, and volume change. Readings from the vertical deformation gage, the vertical load cell and the volume change device were recorded. With these readings, the current area, resulting deviator stress and major principal stress were calculated. Then, the current mean principal stress was calculated for each step in loading. Depending on what stress path direction was desired, the required confining pressure was calculated. An equation for the change in confining pressure required, $\Delta \sigma_{3}$, was used for its calculation in terms of the increment in mean confining stress, $\Delta \sigma_{\mathrm{m}}$, and the increment of major principal stress, $\Delta \sigma_{1}$. The pressure gage was manually adjusted until the digital read-out displayed the
correct corresponding pressure. This cycle was performed throughout the entire test as fast as physically possible. The strain rate for all tests varied from $0.01 \mathrm{~mm} / \mathrm{min}$ to $0.05 \mathrm{~mm} / \mathrm{min}$ ensuring a slow enough strain rate to manually input the data and adjust the confining pressure.
4.3 Stress Paths for Tests performed with varying $\sigma_{3}$ with air

The first eight tests had a variety of stress paths along the hydrostatic axis of the triaxial plane. Figure 4.3 .1 shows the stress paths that were followed for Tests A1-A8. The order by which they are plotted is by increasing mean principal stress. Tests A5 and A8 overlay each other as they follow the same stress path. This is also true for Tests A1 and A6. These tests were repeated to ensure repeatability in the testing.


Figure 4.3.1. Stress paths along hydrostatic axis for Tests A1-A8 on Triaxial Plane.
4.4 Stress Paths for Tests performed with varying $\sigma_{3}$ with water

When compared to water pressure, air pressure takes a little bit of time to fully be applied in the cell. There is somewhat of a buffer time between the increase/decrease from the manual regulator and the actual applied pressure inside the cell. In hopes that the testing would be more stable and to get rid of this delay, tests were re-run on certain similar stress paths with water in the triaxial cell. The inside load cell was removed and the cell was completely sealed to be watertight. The line to the transducer was also completely saturated and careful attention was paid to ensure that no air bubbles were present in the line. This was important as any bubbles might affect the displayed pressure transducer reading for the confining pressure and the cell pressure was adjusted in accordance with the displayed reading. The stress paths that were followed for the tests with water in the triaxial cell are presented in Figure 4.4.1.


Figure 4.4.1. Stress paths along hydrostatic axis for Tests W1-W5 on Triaxial Plane.
4.5 Instability of Tests with varying $\sigma_{3}$ when approaching Failure

When performing the tests described above, large instabilities were encountered as the specimen approached failure. These instabilities were seen for tests where the confining pressure, $\sigma_{3}$ was varied. Traditional compression and extension tests (tests that maintain $\sigma_{3}$ constant) did not experience this instability. Because of the instability that was seen in the initial trial test, the tests were performed at very slow strain rates, from $0.01 \mathrm{~mm} / \mathrm{min}$ to $0.05 \mathrm{~mm} / \mathrm{min}$. With a slower strain rate, the resulting incremental deviator stress was in turn less and the confining pressure was adjusted accordingly with smaller incremental changes. As the specimen's strength increased, it behaved very stably with changes in confining pressure that were required to follow the intended stress path. For all of the stress paths described in Section 4.3 and 4.4 (besides the conventional triaxial compression and extension tests), as the deviator stress increased, the confining pressure was lowered. How much the confining pressure was lowered, depended on the stress path being followed. As long as the deviator stress increased and the specimen was able to take additional stress, there were no instabilities experienced. However, as the specimen began to fail, the resulting deviator stress became lower and in order to stay on the same stress path, the confining pressure should be increased. With this increase in confining pressure, the specimen gained strength. As the specimen's new strength was recorded and the calculated $\sigma_{3}$ was known, then the confining pressure once again had to be lowered. Since it was already at/near failure, the specimen would lose strength after experiencing the lower confining pressure and would therefore need a higher confining pressure to stay along stress path. Examples of this behavior are shown in

Figures 4.5.1 and 4.5.2. These figures compare two types of tests, one done with air as confining pressure, the other with water, both of which follow the same stress path.


Figure 4.5.1. Stress-strain plot showing initiation of instability of Tests W4 and A3, both of which are along the same stress path.

These cycles occurring for all of the tests, created a zig-zag pattern in the stress-strain and volume change curves. The specimens were strained as far along as possible without experiencing too great a fluctuation in stress ratios. Once it was too difficult to continue along the desired stress path, the test was stopped. The lower the mean principal stress was, the harder it was to control the stress path. No shear bands were seen, as the specimens were not sheared post peak failure. In order to correct for this instability, the average was taken between fluctuating points and the friction angle was calculated from a point were there was
little or no instability. An example of the corrected stress-strain curve for Test A3 is shown in Figure 4.5.3. In the sections that follow, only the corrected data is presented.

As can be seen from Figures 4.5 .1 and 4.5.2, tests performed with air in the confining cell experienced greater instability than those performed with water. Although they were corrected in the manner described above, the results had too much scatter to be considered in the analysis. Therefore, only test data for tests W1 through W5 is presented in the section that follows.


Figure 4.5.2. Stress-strain plot showing initiation of instability of Tests A1, A6, W2 and W3, all of which are along the same stress path.


Figure 4.5.3. Stress-Strain Plot of Corrected and Uncorrected Test A3.
4.6 Stress-Strain and Volume Change behavior and of Tests with varying $\sigma_{3}$

The stress-strain and volume change curves for Tests W1 through W5 are presented in Figures 4.6.1 and 4.6.2. Test W2 failed with less axial strain because its confining pressure was varied. Test W1 was a conventional triaxial compression test and therefore, failed at larger strain. Test W1 is omitted from Figures 4.6 .3 and 4.6.4 in order to see Tests W2-W5 in more detail. As seen from the figures, W2 through W4 have very similar stress-strain curves. W1 is a traditional compression test and W5 is a conventional extension test that has been described in Chapter 3, Section 3.4.


Figure 4.6.1. Stress-Strain Plots for Tests W1-W5 with Varying $\sigma_{3}$.


Figure 4.6.2. Volume Change for Tests W1-W5 with Varying $\sigma_{3}$.


Figure 4.6.3. Stress-Strain Plots for Tests W2-W5 with varying $\sigma_{3}$.


Figure 4.6.4. Volume Change for Tests W2-W5 with varying $\sigma_{3}$.

Tests with varying $\sigma_{3}\left(\right.$ at $\left.\sigma_{m}=101 \mathrm{kPa}\right)$ for $\alpha=90^{\circ}$

In order to be able to compare tests results from the true triaxial testing apparatus and the torsion shear apparatus (which will be discussed in great detail in Chapter 6), triaxial compression tests with constant mean normal stress at about 101 kPa were performed. As will be described later, the torsion shear tests were all performed with constant mean normal stress of 101 kPa . Therefore, several tests with stress paths perpendicular to the hydrostatic axis were performed at 101 kPa were performed with the same method described in Section 4.2 , by changing the confining pressure in the triaxial cell. Because the tests done with air pressure in the triaxial cell were extremely unstable when reaching failure (as described in Section 4.5), the tests were repeated twice with water in the triaxial cell. The stress-strain curves for the tests are presented in Figure 4.6.5. For comparison and reference, the data for the torsion shear test under triaxial compression at $b=0$ and $\alpha=90^{\circ}$ is also presented in Figure 4.6.5. The torsion shear test results are comprehensively analyzed in Chapter 7. The volume change versus axial strain is presented in Figure 4.6.6.

As can be seen from the stress-strain and volumetric strain plots, there is scatter among the three different tests. It must be kept in mind that two different testing apparatuses were used. Test W2 fails at about $0.8 \%$ more strain than W3. The torsion shear test fails at the least amount of major principal strain. As seen in Table 4.6.1, friction angle differences of 2.0 and 4.2 degrees are seen between Test W2 and W3 and the torsion shear test.


Figure 4.6.5. Stress-Strain curve for Tests with $\sigma^{\prime}{ }_{m}=101 \mathrm{kPa}$ for $\alpha=90^{\circ}$.


Figure 4.6.6. Volumetric strain curves for Tests with $\sigma^{\prime}{ }_{m}=101 \mathrm{kPa}$ for $\alpha=90^{\circ}$.

Table 4.6.1 Torsion Shear and Triaxial Tests with $\sigma_{m}=101 \mathrm{kPa}$ for $\alpha=90^{\circ}$ Tests.

| Test No. | $\varphi(\mathrm{deg})$ | $\sigma_{\mathrm{m}}(\mathrm{kPa})$ | $\varepsilon_{1}(\%)$ |
| :---: | :---: | :---: | :---: |
| W3 | $35.27^{\circ}$ | 100.77 | 3.10 |
| W2 | $37.51^{\circ}$ | 96.54 | 3.94 |
| TS 41 $\left(\alpha=90^{\circ}\right)$ | $33.27^{\circ}$ | 100.8 | 2.27 |

Stress-Strain behavior of Tests with varying $\sigma_{3}$ at $\sigma_{m}=101 \mathrm{kPa}$ for $\alpha=0^{\circ}$

For further comparison to the torsion shear tests performed, one additional test (Test W6) was performed with horizontal bedding planes $\left(\alpha=0^{\circ}\right)$ in the triaxial apparatus with a mean stress of about 101 kPa . This test was compared to the torsion shear test under normal triaxial compression with $b=0$ and $\alpha=0^{\circ}$ conditions. In Figure 4.6.7 and 4.6.8, the stress strain and volumetric change versus axial strain curves are presented. When looking at both curves, it is seen once again that the torsion shear test fails at a much smaller axial strain than the triaxial test on the rectangular prismatic specimen. In this torsion shear test failure occurred with the development of a shear band. However, the friction angles at failure are close to each other with a $1.2^{\circ}$ difference. Table 4.6.2 compares these results.

Table 4.6.2 Torsion Shear and Triaxial Tests with $\sigma_{m}=101 \mathrm{kPa}$ for $\alpha=0^{\circ}$ Tests.

| Test No. | $\varphi(\mathrm{deg})$ | $\sigma_{\mathrm{m}}(\mathrm{kPa})$ | $\varepsilon_{1}(\%)$ |
| :---: | :---: | :---: | :---: |
| W6 | 39.83 | 98.35 | 3.240 |
| TS 23 $(\alpha=0)$ | 40.99 | 101.36 | 1.542 |



Figure 4.6.7. Stress-Strain curve for Tests with $\sigma^{\prime}{ }_{m}=101 \mathrm{kPa}$ for $\alpha=0^{\circ}$.


Figure 4.6.8. Volumetric strain curves for Tests with $\sigma^{\prime} \mathrm{m}=101 \mathrm{kPa}$ for $\alpha=0^{\circ}$.

### 4.7 Summary of Friction Angles from Tests W1-W4

Table 4.7.1 summarizes the strength results for all of the tests performed for this study. As can be seen by the void ratio, e column, the tests were all of similar density and therefore, results could easily be compared. As mentioned previously, all tests in this series had vertical bedding planes. The friction angle and stress ratios at failure are summarized, as well as the mean stress at failure. Tests W1 and W2 were very close in friction angle despite the significant difference in mean stress. Test W4's stress path kept $\sigma_{1}$ constant while $\sigma_{3}$ was lowered. Since the strain to failure is lowest for the tests with lowest confining pressure at failure, the fabric has not changed much and the lowest strength is obtained.

Table 4.7.1. Summary of Test Performed with Varying $\sigma_{3}$.

| Test <br> No. | $\alpha(\mathrm{deg})$ | e | $\varphi(\mathrm{deg})$ | $\sigma_{1} / \sigma_{3}$ | $\sigma_{\mathrm{m}(\text { failure) }}$ <br> $(\mathrm{kPa})$ | $\varepsilon_{1 \text { (failure) }}(\%)$ |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| W1 | $90^{\circ}$ | 0.534 | 37.904 | 4.15 | 206.97 | 5.50 |
| W2 | $90^{\circ}$ | 0.528 | 37.506 | 4.13 | 96.54 | 3.94 |
| W3 | $90^{\circ}$ | 0.530 | 35.271 | 3.73 | 100.77 | 3.10 |
| W4 | $90^{\circ}$ | 0.554 | 35.935 | 3.65 | 52.29 | 2.50 |

Figure 4.7.1 shows the variation of friction angle with increasing mean stress for triaxial compression tests with water in the triaxial cell. For Tests W1-W4, it is seen that the friction angle increases with increasing confining pressure at failure, but more importantly, the axial strain-to-failure decreases with decreasing confining pressure at failure. Thus, the tests results tend to confirm the hypothesis that the effect of initial cross-anisotropic fabric affects
the strength. Usually, the highest friction angles are obtained at low confining pressures, but these results show increasing friction angles with increasing confining pressure at failure.


Figure 4.7.1. Variation of Friction Angle with Mean Principal Stress for Tests W1-W5.

### 4.8 Summary of Dilation Angles from Tests W1-W4

In order to see any effects on friction angle behavior due to dilation angle, an analysis was performed. The angle of dilation, $\Psi$ can be calculated by the following equation:

$$
\begin{equation*}
\Psi=\sin \left(\frac{\left(\frac{\Delta \varepsilon_{v}}{\Delta \varepsilon_{1}}\right)}{\left(\frac{\Delta \varepsilon_{v}}{\Delta \varepsilon_{1}}\right)-2}\right) \tag{Eq. 4.8.1}
\end{equation*}
$$

where $\Delta \varepsilon_{\mathrm{v}}$ is the increment in volume change and $\Delta \varepsilon_{1}$ is the increment in axial strain.

As can be seen in Table 4.8.1 and Figure 4.8.1, there is little variation in the angle of dilation. Therefore, it can be concluded that the angle of dilation does not change regardless of stress path.

Table 4.8.1. Summary of Tests W1-W4 showing Friction Angle, Mean Principal Stress and Dilation Angle.

| Test No. | $\boldsymbol{\phi}(\mathbf{d e g})$ | $\boldsymbol{\sigma}_{\mathbf{m}}(\mathbf{k P a})$ | $\boldsymbol{\Psi}(\mathbf{d e g})$ |
| :---: | :---: | :---: | :---: |
| W1 | 37.904 | 206.97 | 9.1800 |
| W2 | 37.506 | 96.54 | 9.0100 |
| W3 | 35.271 | 100.77 | 7.6261 |
| W4 | 35.935 | 52.29 | 7.2200 |



Figure 4.8.1. Variation of Dilation Angle with Mean Principal Stress for Tests W1-W5.

### 4.9 Conclusion

In conclusion, triaxial compression tests where $\alpha=90^{\circ}$ were done to study the effects of stress paths on cross-anisotropic specimens. By varying the confining pressure, specimens were strained at different stress paths along the triaxial plane. Experiments were done with first air pressure in the triaxial cell, and then with water. Instabilities were seen at failure. However, once the stress-strain behavior was corrected at failure, results showed that for increasing mean principal stress, increasing friction angles are obtained. Dilation angles stayed constant for all tests.

## 5. True Triaxial Tests on Tall Specimens

### 5.1 Introduction

An experimental program was designed to study the effects of cross-anisotropy under three-dimensional conditions on Fine Nevada Sand. Eighteen tests were performed using Lade's (1977) true triaxial apparatus. The stress-strain, strength and failure behavior under various loading conditions on the octahedral plane are presented. The occurrence of shear banding and their influence on the strength behavior of the sand was also studied, but will be described in further detail in Chapter 9. The present chapter will focus on the failure surface attained under different $b$-values where $b=\left(\sigma_{2}-\sigma_{3}\right) /\left(\sigma_{1}-\sigma_{3}\right)$ in the three Sectors of the octahedral plane.

Figure 5.1.1 shows the specimen orientation on the Cartesian coordinates as well as on the octahedral plane. As can be seen from Figure 5.1.1, depending on the orientation of the specimen, the bedding planes are subject to different stresses. Changing b-values allows for a variety of points in each Sector. Tests were performed in about 0.25 increments of $b$-values ranging from zero to one for each of the three Sectors. All specimens had the same dimensions as described previously in Chapter 3 ( 7.6 cm width, 7.6 cm length and 19 cm height, $\mathrm{H} / \mathrm{D}$ ratio $=2.5$ ) and were prepared using the same deposition, freezing and thawing techniques also described in Section 3.1.1.


Figure 5.1.1. Specimen orientation in (a) Cartesian coordinate system and (b) on Octahedral plane.

### 5.2 True Triaxial Apparatus Assembly

The triaxial apparatus described in Section 3.1 was slightly modified in order to be able to produce tests under three-dimensional conditions. However, as described in detail by Lade (1978) a few more parts were added in order to use the horizontal loading device shown in Figure 5.2.1.


Figure 5.2.1. Schematic view of the cubical device (after Lade 1977).

Once the specimen was set up as described previously in Section 3.1, the horizontal loading system had to be assembled carefully around the specimen. The horizontal loading system sits on two guide rails on opposite sides of the specimen. These guide rails are positioned precisely to allow the horizontal plates to compress without hitting the top and bottom caps and are secured to the bottom plate of the triaxial cell. With these guide rails set in place, the
horizontal load is only applied to the specimen. Ball bearings on the bottom and top of the side plates allowed the plates to roll on the guide rails when the plates were being squeezed towards each other. On each of the sides where the horizontal loading system was going to be placed, one sheet of latex rubber membrane was placed on top of a thin layer of vacuum grease covering the entire length and width of the specimen.

The two vertically compressible plates were assembled of alternating pre-stressed and soaked balsa wood and steel laminae. Prior to testing, the balsa wood pieces were compressed three times using a vice and then were soaked in water. This allowed for the strength in the direction perpendicular to the fibers of the balsa wood to be significantly decreased. The same set of balsa wood laminae was used during all true triaxial tests. The balsa wood was also kept soaked at all times between testing. In Figure 5.2.2, the configuration of the balsa wood and steel laminae in relation to the specimen can be seen.


Figure 5.2.2. Horizontal and vertical sections of the horizontal loading plates (after Lade 1978).

Once the balsa and wood laminae were assembled, the two plates were interconnected and secured with screws around the specimen. Once the side plates were placed, horizontal LVDTs were positioned in order to measure horizontal deflections. At this point, the horizontal loading system was slightly squeezed by applying a small amount of pressure to the system to ensure that the plates were lightly touching the specimen. Doing so avoided a false horizontal strain measurement at the beginning of the test. The horizontal load was applied to the specimen by an oil-filled pressure cylinder. A pressure transducer connected to the oil line on the cylinder outside of the triaxial cell was hooked up to a strain gage indicator box and the box provided a digital display of the pressure being applied. A picture of the entire assembly can be seen in Figure 5.2.3.

To complete the entire triaxial assembly, a frame with four legs was attached to the piston by a setscrew. This frame allowed for vertical compression of the side plates. The frame and piston were carefully set while placing the top lid. They were precisely positioned so that the four-legged frame sat on the top ball bearings of the horizontal plates. The flat end of the piston was also positioned to slightly touch the steel ball sitting on the top cap load cell (also described in Section 3.1). With everything in place, the cell was closed and sealed with six tie rods. A digital dial gage was secured on the piston to measure axial deformations. Also, the external load cell (described in Section 3.1) was left in place on the triaxial loading machine in order to compare loads between the external and embedded top cap load cell. A schematic of the horizontal loading system is shown in Figure 5.2.4.


Figure 5.2.3. Picture of True Triaxial Horizontal Loading System with Horizontal LVDTs.


Figure 5.2.4. Schematic of Horizontal Loading System.

### 5.3 Calibration of Horizontal Loading System

All measurement devices were calibrated to ensure correct measurements. Certain measurement devices such as the volume change, dial gage and the load cells were previously calibrated for the tests described in Chapters 3. However, with the addition of a new system to the assembly, two new devices had to be calibrated: the horizontal loading system and the horizontal LVDTs.

In order to calibrate the horizontal loading system, the horizontal load is calibrated as a function of the pressure transducer readings on the P-3500 strain indicator box. In order to calibrate the pressure transducer readings, a load cell was placed between the two vertical
plates. The balsa wood and steel laminae were not assembled for the calibration. The load cell was carefully placed in the middle of the plates in order to minimize any moment effects from any eccentricity. Steel plates were set around the load cell in order to prevent any compression of the plates. Any compression of the plates would provide a false calibration of pressure and load. Once set, pressure was applied to the pressure cylinder and in turn, the plates. Pressure was applied and the reading displayed on the indicator box alongside the load cell reading was recorded for five loading and unloading cycles reaching 200 kPa . The load cell reading was converted to pounds (by a previous calibration described in Section 3.1.3). Using these readings, a calibration curve was determined and applied for the experiments. A picture of the set up for the horizontal loading calibration is shown in Figure 5.3.1. The strain indicator box, pressure cylinder and load cell can be seen.


Figure 5.3.1. Picture of set-up for the Horizontal Loading System calibration.

Two horizontal Linear Variable Differential Transformers (LVDTs) were used on each side of the specimen in order to measure horizontal deformations. In order to calibrate the horizontal LVDTs, they were hooked up to an MP2000 signal display unit that displayed real time readings. The LVDTs were secured on stands especially made to calibrate LVDTs, and they were calibrated using Mitutoya Digimac Micrometers. Each horizontal LVDT was calibrated individually and two calibration curves were calculated. A picture of the calibration set up is shown in Figure 5.3.2.


Figure 5.3.2. Picture of set-up for two horizontal LVDTs calibration.

### 5.4 Experimental Program

As mentioned before, 18 tests were performed on tall rectangular prismatic specimens. The summary of the tests performed according to Sector and b-values is shown in Table 5.4.1. The b-values are not precisely at 0.25 increments for certain tests because of certain calibration and testing conditions that were particular to each test. For these specific cases, details will be explained when analyzing the stress-strain behavior. However, in most of the cases, the $b$-values were maintained constant as the specimen was sheared.

For the first Sector tests, seven specimens with horizontal bedding planes $\left(\alpha=0^{\circ}\right)$ were prepared. Five specimens in the second Sector and six specimens in the third Sector were prepared with vertical bedding planes $\left(\alpha=90^{\circ}\right)$. The specimens were rotated so that the bedding planes and specimen sides were correctly oriented in accordance with Figure 5.1.1.

Table 5.4.1. Summary of alpha and b-values of True Triaxial Tests in all Three Sectors.

| Test No. | $\alpha$ | b-value |
| :---: | :---: | :---: |
|  |  |  |
| Sector I |  |  |
| TT\#1 | 0 | 0.00 |
| TT\#2 | 0 | 0.24 |
| TT\#3 | 0 | 0.51 |
| TT\#4 | 0 | 0.75 |
| TT\#5 | 0 | 0.70 |
| TT\#6 | 0 | 0.72 |
| TT\#7 | 0 | 1.00 |
| Sector II |  |  |
| TT\#8 | 90 | 0.25 |
| TT\#9 | 90 | 0.49 |
| TT\#10 | 90 | 0.70 |
| TT\#11 | 90 | 0.69 |
| TT\#12 | 90 | 1.0 |
| Sector III |  |  |
| TT\#13 | 90 | 0.00 |
| TT\#14 | 90 | 0.25 |
| TT\#15 | 90 | 0.49 |
| TT\#16 | 90 | 0.72 |
| TT\#17 | 90 | 0.72 |
| TT\#18 | 90 | 0.95 |

All tests were performed with constant confining pressure. A confining pressure of 50 kPa was set for most tests. Tests with $\mathrm{b}=1.0$ were set at a lower confining pressure of 30 kPa .

As confining pressure was applied in the triaxial cell, the horizontal loading bellofram diaphragm was also subject to this cell pressure. In order to keep the piston placed correctly and to prevent any oil from running back into the pressure cylinder, the non-diplacement valve was closed to the pressure cylinder. Once the final total confining pressure was set in the triaxial cell, the amount of pressure required to produce the initial zero reading in the oil pressure cylinder transducer was applied. With higher confining pressures, a greater initial pressure is required to zero the cylinder before the test. The air pressure supply line only reached about 800 kPa , and this pressure is not sufficient to increase the load in the horizontal loading system when running tests with higher $b$-values where a larger horizontal load was required. Therefore, the confining pressure in the triaxial cell was set to a lower pressure.

A program was set up to maintain a constant b-value throughout the entire test. After isotropic compression, the new area and volume were calculated. With a shearing rate of $0.1 \mathrm{~mm} / \mathrm{min}$, the specimen was sheared on the loading machine. Readings were taken from the vertical dial gage, two horizontal LVDTs, and both vertical load cells. With these values input to the program, the major principal stress was calculated according to the corrected area of the specimen. The corresponding calibration for the horizontal load that produced the required intermediate principal stress for the desired $b$-value was calculated. With this known, pressure to the horizontal loading system was added manually with the pressure
regulator until the desired reading was reached on the P-3500 strain indicator box. At this point, the appropriate intermediate principal stress was applied. New readings were attained immediately and the cycle was repeated until the specimen was sheared and shear bands were observed.

### 5.5 Calculation of Horizontal Force required and Corrections to Horizontal Strains

In order to calculate the required horizontal load for any $b$-value, a relationship between $b$, the areas and the vertical force was derived. As previously stated, the intermediate principal stress, $\sigma_{2}$ can be expressed by the following equation:

$$
\begin{equation*}
b=\frac{\left(\sigma_{2}-\sigma_{3}\right)}{\left(\sigma_{1}-\sigma_{3}\right)} \rightarrow\left(\sigma_{2}-\sigma_{3}\right)=b\left(\sigma_{1}-\sigma_{3}\right) \tag{Eq. 5.5.1}
\end{equation*}
$$

Since intermediate principal stress is a summation of the confining pressure, $\sigma_{3}$ and the horizontal deviator stress, $\sigma_{2 \mathrm{~d}}$, and because stress is a force divided by an area, it can be written that,

$$
\sigma_{2}=\left(\sigma_{2 D}+\sigma_{3}\right) \rightarrow \sigma_{2 D}=\left(\sigma_{2}-\sigma_{3}\right)
$$

where
$\sigma_{2 D}=\frac{F_{2}}{A_{2}}$
Eq. 5.5.2
where $F_{2}$ is the horizontal force and $A_{2}$ is the area of the sides of the specimen (length times height).

A similar equation can be derived for the major principal stress including the vertical deviator stress such that,

$$
\begin{align*}
& \sigma_{1}=\left(\sigma_{1 D}+\sigma_{3}\right) \rightarrow \sigma_{1 D}=\left(\sigma_{1}-\sigma_{3}\right) \\
& \text { where }  \tag{Eq. 5.5.3}\\
& \sigma_{1 D}=\frac{F_{1}}{A_{1}}
\end{align*}
$$

where $F_{1}$ is the vertical force and $A_{1}$ is the area of the top of the specimen (length times width).

By combining equations 5.5.1, 5.5.2 and 5.5.3, the following relationship for the horizontal force can be derived where,

$$
\begin{equation*}
\frac{F_{2}}{A_{2}}=b\left(\frac{F_{1}}{A_{1}}\right) \rightarrow F_{2}=b\left(\frac{A_{2}}{A_{1}}\right) F_{1} \tag{Eq. 5.5.4}
\end{equation*}
$$

Using equation 5.5.4, the force required for the horizontal loading system was calculated. By knowing the force required, the reading for the pressure transducer was calculated with the
calibration curve previously established. In this manner, adequate pressure was applied to the specimen in order to maintain the b -value for the specific experiment.

As described previously, when setting up the horizontal loading system, a small amount of pressure was applied to the horizontal loading system to move the plates as close to the specimen as possible. The reading on the display was recorded and the non-displacement valve on the pressure cylinder was closed. After the total confining pressure was applied, the new pressure transducer reading was recorded. Since the pressure transducer was located between the non-displacement shut off valve and the horizontal loading cylinder inside the triaxial cell, the transducer could display the new readings with the addition of the cell pressure (see Figure 5.2.4). The valve was then opened and pressure was manually increased so that the pressure in the oil filled cylinder was balanced with the horizontal loading cylinder. This prevented the oil in the bellofram diaphragm from running back into the oil cylinder. With this process completed, the plates of the horizontal loading system were still in the original position as close to the specimen as possible. Since the plates did not touch the specimen, no stress was exerted prior to testing.

At the start of the test, the initial gap between the specimen and the horizontal plates was seen in the movement of the horizontal LVDTs. This created some false deformation measurements. To exclude this initial movement of the plates, the strains in the horizontal $\varepsilon_{2}-$ direction were plotted versus the strains in the axial $\varepsilon_{1}$-direction. Large strains seen at the
beginning of shearing which were obviously due to this initial gap between the plates and specimen were subtracted. This allowed for the true horizontal strain to be analyzed. Figure 5.5.1 shows an example of the correction applied to the $\varepsilon_{2}$ strain calculations for all of the true triaxial tests.


Figure 5.5.1. Example of the correction applied when calculating $\varepsilon_{2}$ for true triaxial tests.

### 5.6 Correction to Friction Angle and b-values due to Measurement Errors

In order to account for any measurement errors, an analysis was done for both friction angles and b-values. This analysis used the Least Squares Method to determine the error for true triaxial tests, assuming certain values of error in vertical force, $\mathrm{F}_{\mathrm{v}}$, inner and outer
pressures, $p_{i}$ and $p_{o}$, inner and outer radii, $r_{i}$ and $r_{o}$ and alpha values. A table with the summary and the effect on friction angle and b-values for different ranges of errors is presented in Table 5.6.1. As is seen, the friction angle varies from 0.2 to 2.5 degrees for the measurement inaccuracies presented. The worst effect occurs for the condition $b=1$ and $\alpha=0^{\circ}$. The major contributor of the error is seen to be the cell pressure. The force and area deviation are seen to play minor roles in the friction angle error. A detailed description of how these values were calculated can be found in Appendix J.

When looking at the error in b-value, the cell pressure does not affect the error analysis. The only terms that are important are the areas in the vertical and horizontal direction on which the forces in the horizontal and vertical directions are applied. The analysis of varying measurement errors on the b-value can be seen in Table 5.6.2. As can be seen, the variation of b-value is very small, ranging from 0.002 to 0.026 .

Table 5.6.1. Comparison of measurement Errors in friction angle for different values of $\Delta \mathrm{Fv}, \Delta \mathrm{Fv}, \Delta \mathrm{Av}$ and $\Delta \mathrm{Ah}$ for four true triaxial tests.

| \# | b | $\phi\left({ }^{\circ}\right)$ | F (kg) | $\Delta \mathrm{F}$ | A (cm ${ }^{2}$ ) <br> $\left(\mathrm{cm}^{2}\right)$ | $\Delta \mathrm{A}$ | $\begin{gathered} \sigma_{\text {cell }} \\ (\mathbf{k P a}) \end{gathered}$ | $\Delta \sigma_{\text {cell }}$ | $\Delta \phi\left({ }^{\circ}\right)$ |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 1 | 0 | 42.97 | 133.1 | 0.5 | 60.92 | 0.5 | 50 | 0.5 | 0.226 |
| 7 | 1 | 50.36 | 121.4 | 0.5 | 59.22 | 0.5 | 30 | 0.5 | 0.300 |
| 13 | 0 | 37.6 | 97.7 | 0.5 | 61.24 | 0.5 | 50 | 0.5 | 0.236 |
| 18 | 0.95 | 46.8 | 163.2 | 0.5 | 59.62 | 0.5 | 30 | 0.5 | 0.219 |
| \# | b | $\phi\left({ }^{\circ}\right)$ | F (kg) | $\Delta \mathrm{F}$ | $\mathrm{A}\left(\mathrm{~cm}^{2}\right)$ | $\Delta \mathrm{A}$ | $\begin{gathered} \sigma_{\text {cell }} \\ (\mathbf{k P a}) \end{gathered}$ | $\Delta \sigma_{\text {cell }}$ | $\Delta \phi\left({ }^{\circ}\right)$ |
| 1 | 0 | 42.97 | 133.1 | 1 | 60.92 | 0.5 | 50 | 1 | 0.383 |
| 7 | 1 | 50.36 | 121.4 | 1 | 59.22 | 0.5 | 30 | 1 | 0.552 |
| 13 | 0 | 37.6 | 97.7 | 1 | 61.24 | 0.5 | 50 | 1 | 0.406 |
| 18 | 0.95 | 46.8 | 163.2 | 1 | 59.62 | 0.5 | 30 | 1 | 0.367 |
| \# | b | $\phi\left({ }^{\circ}\right)$ | F (kg) | $\Delta \mathrm{F}$ | $\mathrm{A}\left(\mathrm{~cm}^{2}\right)$ | $\Delta \mathbf{A}$ | $\begin{gathered} \sigma_{\text {cell }} \\ (\mathbf{k P a}) \end{gathered}$ | $\Delta \sigma_{\text {cell }}$ | $\Delta \phi\left({ }^{\circ}\right)$ |
| 1 | 0 | 42.97 | 133.1 | 1 | 60.92 | 0.5 | 50 | 2 | 0.639 |
| 7 | 1 | 50.36 | 121.4 | 1 | 59.22 | 0.5 | 30 | 2 | 1.056 |
| 13 | 0 | 37.6 | 97.7 | 1 | 61.24 | 0.5 | 50 | 2 | 0.711 |
| 18 | 0.95 | 46.8 | 163.2 | 1 | 59.62 | 0.5 | 30 | 2 | 0.671 |
| \# | b | $\phi\left({ }^{\circ}\right)$ | F (kg) | $\Delta \mathrm{F}$ | $\mathrm{A}\left(\mathrm{~cm}^{2}\right)$ | $\Delta \mathbf{A}$ | $\begin{gathered} \sigma_{\text {cell }} \\ (\mathbf{k P a}) \end{gathered}$ | $\Delta \sigma_{\text {cell }}$ | $\Delta \phi\left({ }^{\circ}\right)$ |
| 1 | 0 | 42.97 | 133.1 | 1 | 60.92 | 1 | 50 | 5 | 1.677 |
| 7 | 1 | 50.36 | 121.4 | 1 | 59.22 | 1 | 30 | 5 | 2.506 |
| 13 | 0 | 37.6 | 97.7 | 1 | 61.24 | 1 | 50 | 5 | 1.701 |
| 18 | 0.95 | 46.8 | 163.2 | 1 | 59.62 | 1 | 30 | 5 | 1631 |

Table 5.6.2. Comparison of Measurement Errors in b-values for different values of $\Delta F_{v}$, $\Delta A_{v}$ and $\Delta A_{h}$.


After performing the analysis of the importance of measurement errors, errors were assigned to the measured values for all true triaxial tests. These errors were calculated for both friction angles and b -values. The cell pressure error was estimated to be at $0.5 \mathrm{kPa}\left(0.005 \mathrm{~kg} / \mathrm{cm}^{\wedge} 2\right)$. This is about $1 \%$ of the total cell pressure applied. The error on the vertical and horizontal forces were both estimated to be 1 kg (also about $1 \%$ of the applied vertical load) and the area measurement error was estimated at $0.5 \mathrm{~cm}^{\wedge} 2$ (almost $1 \%$ of the horizontal area). Although
these errors are provided, the integrity of the tests performed is strong and it is believed that these errors are on the conservative side.

The results from the analysis are shown in Table 5.6.3. The worst case of error in friction angle occurs in Sector II where $\alpha=90^{\circ}, \mathrm{b}=1.0$. The measurement errors result in only 0.325 degrees. A summary of friction angle errors and $b$-value errors is presented in Table 5.6.3. To get a better visual understanding of the deviation with respect to $b$-values, Figures 5.6.1 and 5.6.2 are also presented.

Table 5.6.3. Summary of Friction Angle and b-value Measurement Errors for all Ttrue Triaxial tests.

| $\begin{array}{\|l\|} \text { Test } \\ \text { No. } \end{array}$ | b | $\alpha$ | $\phi\left({ }^{\circ}\right)$ | $\begin{gathered} \mathrm{Fv} \\ (\mathrm{~kg}) \end{gathered}$ | $\begin{gathered} \Delta_{\mathrm{Fv}} \\ (\mathrm{~kg}) \\ \hline \end{gathered}$ | $\begin{gathered} \mathrm{Ah} \\ \left(\mathrm{~cm}^{2}\right) \end{gathered}$ | $\begin{gathered} { }^{\Delta} \mathrm{Ah} \\ \left(\mathrm{~cm}^{2}\right) \\ \hline \end{gathered}$ | $\begin{array}{\|c\|} \hline \sigma_{\text {cell }} \\ \left(\mathrm{kg} / \mathrm{cm}^{2}\right. \\ \hline \end{array}$ | $\begin{gathered} \Delta \sigma_{\text {cell }} \\ \left(\mathrm{kg} / \mathrm{cm}^{2}\right) \\ \hline \end{gathered}$ | $\Delta \phi \quad\left({ }^{\circ}\right)$ | $\begin{gathered} \text { Fh } \\ (\mathrm{kg}) \end{gathered}$ | $\begin{aligned} & \Delta_{\mathrm{Fh}} \\ & (\mathrm{~kg}) \end{aligned}$ | $\begin{gathered} \mathrm{A} v \\ \left(\mathrm{~cm}^{\wedge} 2\right) \\ \hline \end{gathered}$ | ${ }^{\Delta} \mathrm{Av}$ | Db-value |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| Sector I |  |  |  |  |  |  |  |  |  |  |  |  |  |  |  |
| 1 | 0 | 0 | 42.97 | 133.1 | 1 | 60.9 | 0.5 | 0.510 | 0.005 | 0.252 | 0.0 | 1 | 142.52 | 0.5 | 0.003 |
| 2 | 0.2 | 0 | 46.42 | 161.2 | 1 | 60 | 0.5 | 0.510 | 0.005 | 0.237 | 92.8 | 1 | 147.03 | 0.5 | 0.004 |
| 3 | 0.5 | 0 | 50.35 | 203.0 | 1 | 59.4 | 0.5 | 0.510 | 0.005 | 0.220 | 244.2 | 1 | 150.39 | 0.5 | 0.006 |
| 4 | 0.8 | 0 | 51.73 | 221.3 | 1 | 59.5 | 0.5 | 0.510 | 0.005 | 0.214 | 404.5 | 1 | 151.98 | 0.5 | 0.009 |
| 5 | 0.7 | 0 | 52.84 | 236.6 | 1 | 59 | 0.5 | 0.510 | 0.005 | 0.209 | 398.8 | 1 | 151.95 | 0.5 | 0.008 |
| 6 | 0.7 | 0 | 52.44 | 229.6 | 1 | 59.9 | 0.5 | 0.510 | 0.005 | 0.211 | 403.5 | 1 | 151.9 | 0.5 | 0.008 |
| 7 | 1 | 0 | 50.36 | 121.4 | 1 | 59.2 | 0.5 | 0.306 | 0.005 | 0.320 | 297.9 | 1 | 154.9 | 0.5 | 0.014 |
| Sector II |  |  |  |  |  |  |  |  |  |  |  |  |  |  |  |
| 8 | 0.2 | 90 | 46.29 | 159.1 | 1 | 59.9 | 0.5 | 0.510 | 0.005 | 0.238 | 93.6 | 1 | 148.5 | 0.5 | 0.004 |
| 9 | 0.5 | 90 | 49.13 | 187.7 | 1 | 59.2 | 0.5 | 0.510 | 0.005 | 0.226 | 217.4 | 1 | 150.25 | 0.5 | 0.006 |
| 10 | 0.7 | 90 | 52.47 | 232.6 | 1 | 59.4 | 0.5 | 0.510 | 0.005 | 0.210 | 383.4 | 1 | 152.18 | 0.5 | 0.008 |
| 11 | 0.7 | 90 | 52.45 | 233.6 | 1 | 60 | 0.5 | 0.510 | 0.005 | 0.210 | 383.7 | 1 | 137.12 | 0.5 | 0.009 |
| 12 | 1 | 90 | 50.49 | 117.3 | 1 | 58.7 | 0.5 | 0.306 | 0.005 | 0.325 | 281.2 | 1 | 151.96 | 0.5 | 0.014 |
| Sector III |  |  |  |  |  |  |  |  |  |  |  |  |  |  |  |
| 13 | 0 | 90 | 37.6 | 97.7 | 1 | 61.2 | 0.5 | 0.510 | 0.005 | 0.281 | 0.0 | 1 | 140.82 | 0.5 | 0.004 |
| 14 | 0.3 | 90 | 45.21 | 146.9 | 1 | 59.1 | 0.5 | 0.510 | 0.005 | 0.245 | 90.1 | 1 | 145.77 | 0.5 | 0.004 |
| 15 | 0.5 | 90 | 44.3 | 140.8 | 1 | 59.7 | 0.5 | 0.510 | 0.005 | 0.248 | 165.2 | 1 | 148.98 | 0.5 | 0.007 |
| 16 | 0.7 | 90 | 49.57 | 195.8 | 1 | 60.4 | 0.5 | 0.510 | 0.005 | 0.222 | 331.4 | 1 | 152.18 | 0.5 | 0.009 |
| 17 | 0.7 | 90 | 50.07 | 200.9 | 1 | 60.2 | 0.5 | 0.510 | 0.005 | 0.222 | 347.0 | 1 | 151.03 | 0.5 | 0.009 |
| 18 | 0.9 | 90 | 46.8 | 163.2 | 1 | 59.6 | 0.5 | 0.510 | 0.005 | 0.236 | 370.3 | 1 | 153.74 | 0.5 | 0.012 |



Figure 5.6.1. Summary of b-value Measurement Errors for True Triaxial tests plotted.


Figure 5.6.2. Summary of Friction Angle Measurement Errors for True Triaxial tests. plotted.
5.7 Translating Failure Points to the Same Octahedral Plane

The octahedral plane is a plane whose normal vector, the hydrostatic axis creates equal angles with each of the principal axes. The octahedral plane can be used to present results of tests with three unequal stresses. The center point of the octahedral plane represents the hydrostatic axis, where $\sigma_{1}=\sigma_{2}=\sigma_{3}$. The three axes (120 degrees apart) represent the projections of the principal stress axes in 3D on the octahedral plane. The axes are located symmetrically such that $\sigma_{1}=\sigma_{2}, \sigma_{2}=\sigma_{3}$, and $\sigma_{3}=\sigma_{1}$. A diagram of the octahedral plane and the corresponding stresses in each sector is shown in Figure 5.7.1 and in Figure 5.7.2.

The octahedral plane serves as a good diagram on which to represent the three principal stresses of the true triaxial tests. However, because the tests were not under the same constant mean normal stress at failure, they do not all fall on the same octahedral plane. In order to analyze all data from the true triaxial tests across the same conditions, the failure points were shifted to the same octahedral plane where the mean normal stress is $\sigma_{m}^{\prime}=100 \mathrm{kPa}$ corresponding to a value of the first stress invariant of $\mathrm{I}_{1}=300 \mathrm{kPa}$. This plane was chosen because the torsion shear tests to be presented later were all done with $\mathrm{I}_{1}=300 \mathrm{kPa}$ and it was the intent to compare the results of the two types of tests. A detailed description of how to project stress points onto the same octahedral plane is provided in Appendix F.

In the sections that follow, all presented stress points for true triaxial tests have been projected onto the $\mathrm{I}_{1}=300 \mathrm{kPa}$ octahedral.


Figure 5.7.1. Octahedral Plane in 2D showing the stress states in each part of the plane.


Figure 5.7.2. Principal Stress space in 3D showing Octahedral Plane and Hydrostatic Axis.

### 5.8 Sector I Tests

Specimens tested in Sector I of the octahedral plane have horizontal bedding planes, where $\alpha=0^{\circ}$. In this part of the experimental study, seven specimens were prepared and tested with different b-values. A summary of the tests as well as the stresses at failure is presented in Table 5.8.1 and Figure 5.8.1. The friction angle values have been corrected for void ratio
variation (as mentioned in Section 3.2.2) and have all been corrected to a void ratio of 0.53 . The stresses have also been shifted to the $\mathrm{I}_{1}=300 \mathrm{kPa}$ plane on the Octahedral Plane. For bvalues from 0 to 0.69 , the friction angle increases. However, the friction angle decreases once it passes about 0.69 and drops by about two degrees as it reaches 1 . This trend can be seen in Figure 5.8.6.

Table 5.8.1 Summary of Sector I Tests with stresses at failure.

| Test <br> No. | b-value | $\boldsymbol{\sigma}_{\mathbf{1}} / \boldsymbol{\sigma}_{3}$ | $\boldsymbol{\phi}\left(^{\circ}\right)^{*}$ |
| :---: | :---: | :---: | :---: |
| TT\#1 | 0.00 | 5.49 | 43.34 |
| TT\#2 | 0.24 | 7.16 | 48.85 |
| TT\#3 | 0.51 | 9.23 | 53.41 |
| TT\#4 | 0.75 | 9.99 | 54.78 |
| TT\#5 | 0.70 | 10.59 | 55.84 |
| TT\#6 | 0.72 | 10.32 | 55.42 |
| TT\#7 | 1.00 | 8.83 | 52.80 |

> *Friction angles listed in the table have been corrected for void ratio variation and have been shifted to $\mathrm{I}_{1}=300 \mathrm{kPa}$


Figure 5.8.1. Varying Friction Angle with b-values for Sector I Tests.

As seen from the table, there are three tests with a b-value near 0.75 . Test TT\#4 was sheared at a constant b-value of 0.75 for the entire duration of the test. However, as mentioned previously in Section 5.4, for tests with higher b-values the confining pressure had to be lowered. Before knowing this, Tests TT\#5 and TT\#6 were conducted at b-values of 1.0 until the air pressure supply line was maxed out. Once this occurred, the horizontal stress could not be kept at the corresponding $b$-value of 1.0 and therefore, dropped as the vertical deviator stress increased. Because of this circumstance, the horizontal stress was kept constant while the specimens were sheared. The b -values listed in Table 5.1.1 are the b -values that were
calculated according to the applied horizontal and axial stress at failure. In Test TT\#7, the confining pressure was lowered to 30 kPa and the test was able to hold a constant b -value of 1.0 during the entire shearing of the specimen.

Figure 5.8.2 shows the stress paths followed for the different tests in Sector I according to bvalue. As is seen, Tests 5 and 6 deviate from the intended b-value and as the horizontal pressure is kept constant and the vertical load is increased, the b-value decreases until failure and stays constant at that value until the softening regime is reached.


Figure 5.8.2. b-value stress paths for Tests TT\#1-TT\#7.

Even though the stress paths changed slightly towards the end, it is seen that the friction angles and stress ratios of all tests that failed at a b-value of 0.75 are extremely close to each other. This confirms the repeatability of the results.

The stress-strain and volume change behavior for Tests TT\#1 through TT\#7 are presented in Figure 5.8.3 and 5.8.4. As the b-value increases, the axial strain at failure also decreases. After peak failure, certain specimens developed shear bands. The beginning of the shear bands can be seen where there is a sudden drop in stress ratio. However, shear banding and its effects will be discussed in detail in Chapter 9. When looking at the volume change behavior, there is also a trend that shows more dilation with increasing b-values. The angle of dilation is presented in Figure 5.8.5. The angles for all tests were plotted. As can be seen from the figure, there is some scatter among the results for $\mathrm{b}=0.75$, particularly at TT\#5 and TT\#6. This is due to the changes in b-value stress path described earlier. The angle of dilation is constant until about 0.5 and then increases with increasing b-value. Table 5.8.2 lists the angle of dilation for each test in Sector I. The dilation angle was calculated by analyzing the slope of the volume change at failure. The equation for angle of dilation, $\Psi$ is written as
$\Psi=\sin ^{-1}\left(\frac{\frac{\Delta \varepsilon_{v}}{\Delta \varepsilon_{1}}}{\frac{\Delta \varepsilon_{v}}{\Delta \varepsilon_{1}}-2}\right)$
where $\Delta \varepsilon_{\mathrm{v}}$ is the incremental volumetric strain and $\Delta \varepsilon_{1}$ is the incremental axial strain.

The axial strain and horizontal strain was also analyzed. In Figure 5.8.6, it is apparent that as b-value increases, the horizontal strain also increases. This is due to the increase in horizontal deviator stress applied by the horizontal loading system. For tests TT\#5 and TT\#6 which were originally $b=1.0$ tests, the horizontal strain overlaps that of test TT\#7 where $b=1.0$. However, as the b-value starts to decrease, the horizontal strain also decreases and follows very closely the strain shown by TT\#4 where $\mathrm{b}=0.75$. The horizontal strain for Test TT\#7 where $\mathrm{b}=1.0\left(\sigma_{2}=\sigma_{1}\right)$ does not level off towards failure like the other tests. The horizontal strain increases linearly until failure is reached.


Figure 5.8.3. Stress-strain curves for Tests TT\#1-TT\#7 in Sector I.


Figure 5.8.4. Volume Change curves for Tests TT\#1-TT\#7 in Sector I.


Figure 5.8.5. Angle of Dilation for Sector I Tests.

Table 5.8.2. Summary Table of Dilation Angles for Sector I Tests.

| Sector 1 |  |  |
| :---: | :---: | :---: |
| Test No. | b-value | Dilation Angle <br> $\left({ }^{\circ}\right)$ |
| TT\#1 | 0 | 13.17 |
| TT\#2 | 0.24 | 13.48 |
| TT\#3 | 0.51 | 14.30 |
| TT\#4 | 0.75 | 17.02 |
| TT\#5 | 0.70 | 13.40 |
| TT\#6 | 0.72 | 15.01 |
| TT\#7 | 1.00 | 28.07 |



Figure 5.8.6. Comparison of horizontal and vertical strains for Sector I tests.

### 5.9 Sector II Tests

A total of five specimens were sheared in order to attain test data for Sector II of the octahedral plane. Specimens in this Sector have vertical bedding planes where $\alpha=90^{\circ}$. As seen in Figure 5.1.1, the face of the specimen that is perpendicular to the bedding planes is placed in the $\sigma_{2}$ direction. A summary of the tests performed is provided in Table 5.9.1. This summary includes Test TT\#13 where $\mathrm{b}=0$ and $\alpha=90^{\circ}$. TT\#13's test conditions are shared on
the octahedral plane between Sector II and III and are therefore presented in both sections. Test TT \#7 and TT \#12 are also shared and are compared later in Figure 5.9.7.

As described before with TT\#5 and TT\#6, when running Test TT\#11, the pressure in the air supply line reached its maximum. Therefore, instead of being able to fail the specimen at the desired $b$-value of 1 , the horizontal stress was kept constant while the axial stress reached its maximum and sheared the specimen. Therefore, Tests TT \#10 and TT\#11 both were sheared at almost the same exact b-value at failure. The friction angle difference for both tests is 0.6 degrees, showing once again the repeatability of the results of the tests. In order to shear the specimen at a constant b-value of 1.0 , the confining pressure was lowered to 30 kPa . The results of the six tests in Sector II are summarized in Table 5.9.1 and Figure 5.9.1. The stresses presented are stresses at failure. The stress paths with relation to b-value and major principal stress are presented in Figure 5.9.2.

Table 5.9.1. Summary of Sector II Tests with stresses at failure.

| Test <br> No. | $\mathbf{b}-$ <br> value | $\boldsymbol{\sigma}_{\mathbf{1}} / \boldsymbol{\sigma}_{\mathbf{3}}$ | $\left.\boldsymbol{\phi} \mathbf{(}^{\circ}\right)^{*}$ |
| :---: | :---: | :---: | :---: |
| TT\#13 | 0.00 | 4.16 | $38.02^{\circ}$ |
| TT\#8 | 0.25 | 7.15 | 48.99 |
| TT\#9 | 0.49 | 8.65 | 52.44 |
| TT\#10 | 0.70 | 10.34 | 55.45 |
| TT\#11 | 0.69 | 10.36 | 55.49 |
| TT\#12 | 1.00 | 8.60 | 52.34 |

*Friction angles listed in the table have been corrected for void ratio variation and have been shifted to $\mathrm{I}_{1}=300 \mathrm{kPa}$


Figure 5.9.1. Varying Friction Angle with b-values for Sector II Tests.


Figure 5.9.2. b-value stress paths for Tests TT\#8-TT\#13.

The stress-strain and volume change curves for Tests TT\#8 through TT\#13 are presented in Figures 5.9.3 and 5.9.4. As the b-value increases from 0 to 0.75 , the stress ratio also increases. With $a b=1.0$, the stress value decreases. The axial strain to failure also decreases as the b-value increases. The stress-strain behavior of TT\#10 and TT\#11 follow the same curve and fail at almost the same stress-ratio and friction angle. TT\#11 begins to drop b value at about $0.5 \%$ axial strain. Due to the initial increase in strain under $b=1.0$ conditions for TT\#10, the total volume change begins to show dilation earlier than TT\#11. However, once they reach the same $b=0.75$ value at about $1 \%$ axial strain, it can be seen that the curves follow the same exact slopes. Shear banding occurs sooner in TT\#11 due to the experienced increase in dilation and change in stress path.

The angles of dilation for Sector II tests are presented in Figure 5.9.5 and a summary of the values is presented in Table 5.9.2. When looking through the scatter for the repeated tests at $\mathrm{b}=0.75$, there is an upward trend as b -increases. This is apparent when looking at the slopes of the volume change shown in Figure 5.9.4.


Figure 5.9.3. Stress-strain curves for Tests TT\#8-TT\#13 in Sector II.


Figure 5.9.4. Volume Change curves for Tests TT\#8-TT\#13 in Sector II.


Figure 5.9.5. Angle of Dilation for Sector II Tests.

Table 5.9.2. Summary Table of Dilation Angles for Sector II Tests.
Sector II

| Test No. | b-value | Dilation <br> Angle |
| :---: | :---: | :---: |
| TT\#13 | 0 | 8.47 |
| TT\#8 | 0.24 | 15.04 |
| TT\#9 | 0.48 | 11.84 |
| TT\#10 | 0.68 | 17.57 |
| TT\#11 | 0.68 | 15.52 |
| TT\#12 | 0.99 | 23.84 |

As shown with Sector I, the axial and horizontal strains were graphed and studied. The horizontal strains are similar to those seen in Sector I. With increasing b-value, the horizontal load is also increased. Therefore, more strain in the horizontal direction will be seen. When comparing Sector I and Sector II, more horizontal strain is seen in Sector II for $b=0.25,0.5$ and 1.


Figure 5.9.6. Comparison of horizontal and vertical strains for Sector II tests.

As a way to double-check the results from Sector I, Specimen TT\#12 was sheared at $b=1.0$. This point is shared on the octahedral plane between Sectors I and II. Looking back at the
results from Test TT \#7, there is only a slight difference in friction angle of 0.17 degrees. The overlay of the plots is presented in Figures 5.9.7 and 5.9.8. In order to see more details of the graphs, the axial strain on the figures is shown only to $2 \%$. The axial strains to failure for both tests are the same. Also, the peak stress ratios are very near each other. Therefore, the two tests can confirm each other.


Figure 5.9.7. Stress Ratio Comparison of Sector I and II Tests with $b=1.0$.


Figure 5.9.8. Volume Change Comparison of Sector I and II Tests with $b=1.0$.

### 5.10 Sector III Tests

A total of six specimens were sheared with vertical bedding planes of $\alpha=90^{\circ}$ that correspond to the third Sector. The intermediate principal stress was applied to the face of the specimen that was parallel to the vertical bedding planes. Therefore, the face perpendicular to the bedding planes was exposed to the confining pressure, $\sigma_{3}$. A summary of Tests TT\#13 through TT\#18 is presented. Test TT\#17 began with a b-value of 1.0 but due to having insufficient air pressure once again, the specimen sheared at a b-value of 0.72 . As can be seen in Table 5.10.1 and Figure 5.10.1, a difference of only 0.5 degrees is seen between Tests TT\#16 and TT\#17. Figure 5.1.2 shows the stress paths followed for the Sector III tests. The test data shows an upward trend until 0.5 . At $\mathrm{b}=0.5$, there is a low friction angle which was due to the occurrence of shear banding during the hardening regime. Then as $b$-value increases from 0.5 to one, the similar trend seen earlier is seen for Sector III.

Table 5.10.1. Summary of Sector III Tests with stresses at failure.

| Test <br> No. | $\mathbf{b -}$ <br> value | $\boldsymbol{\sigma}_{\mathbf{1}} / \boldsymbol{\sigma}_{\mathbf{3}}$ | $\boldsymbol{\phi}\left({ }^{\circ}\right)^{\boldsymbol{*}}$ |
| :---: | :---: | :---: | :---: |
| TT\#13 | 0.00 | 4.16 | 38.02 |
| TT\#14 | 0.25 | 6.74 | 47.88 |
| TT\#15 | 0.49 | 6.78 | 48.15 |
| TT\#16 | 0.72 | 8.94 | 53.17 |
| TT\#17 | 0.72 | 9.18 | 53.62 |
| TT\#18 | 0.95 | 7.76 | 50.29 |

*Friction angles listed in the table have been corrected for void ratio variation \& have been shifted to $\mathrm{I}_{1}=300 \mathrm{kPa}$


Figure 5.10.1. Varying Friction Angle with b-values for Sector III Tests.

The stress-strain and volumetric change graphs are presented in Figures 5.10.3 and 5.10.4. When looking at the behavior of the stress-strain curves for Tests TT\#16 and TT \#7, they look very similar. Because the b -value $=1.0$ for Tests TT\#17 during the beginning of the test, the specimen experienced a greater intermediate principal stress compared to TT\#16. This caused a faster increase in stress with less axial strain (i.e. a steeper curve in the beginning of the test). Once the b -value started to drop, the specimen behaved similarly to Test \#17. Around and after peak failure both curves are almost the same. This behavior is also seen in the volume change graphs.


Figure 5.10.2. b-value stress paths for Tests TT\#13-TT\#18.

Similar to Sectors I and II, the strength increases from $b=0$ to $b=0.25$. However, as seen in Lade and Abelev (2003) the strength drops for Test TT\#15 due to the occurrence of shear bands. The development of shear bands, causes the specimen to shear prematurely and this is reflected in the peak friction angle at failure. The friction angle increases once again for TT\#16 and TT\#17 (where $\mathrm{b}=0.71$ and 0.72 , respectively) and lowers for TT\#18 at $\mathrm{b}=0.94$. The patterns in strength with increasing b-values is similar to those seen in Sectors I and II, with the exception of TT\#15. When comparing $\mathrm{b}=0$ and $\mathrm{b}=1.0$, the largest difference in friction angle $\left(9.2^{\circ}\right)$ is seen in Sector III. Sectors I and II had a difference of $7.9^{\circ}$ and $6.2^{\circ}$, respectively. An increase in angle of dilation is seen once again with tests in Sector III and is presented in Figure 5.10.5. The summary of the dilation angles is presented in Table 5.10.2.


Figure 5.10.3. Stress-strain curves for Tests TT\#13-TT\#18 in Sector III.


Figure 5.10.4 Volume Change curves for Tests TT\#13-TT\#18 in Sector III.


Figure 5.10.5. Angle of Dilation for Sector III Tests.

Table 5.10.2. Summary Table of Dilation Angles for Sector III Tests.

| Sector III |  |  |
| :---: | :---: | :---: |
| Test No. | b-value | Dilation Angle |
| TT\#13 | 0 | 8.78 |
| TT\#14 | 0.25 | 9.03 |
| TT\#15 | 0.49 | 8.80 |
| TT\#16 | 0.72 | 13.80 |
| TT\#17 | 0.72 | 14.34 |
| TT\#18 | 0.95 | 25.85 |

The comparison of vertical and horizontal strain for Sector III tests is presented in Figure 5.10.6. Once again it is seen that with increasing $b$-value the horizontal strains also increase.

At the points where the horizontal strain levels off, there is less axial strain and greater horizontal strain than in Sector II.


Figure 5.10.6. Comparison of horizontal and vertical strains for Sector II tests.

### 5.11 Conclusion

In order to study cross-anisotropy and its effect on the strength behavior of Fine Nevada sand, Sectors I, II and III must be analyzed together. Figure 5.11 .1 shows the varying strengths according to Sector for various b -values. These results are similar to those by Lade and Abelev (2003) where strength decreases when moving from Sector I to III. Because of the deposition of the sand grains, as explained in Chapter 2, the grains are strongest in the horizontal bedding plane $\left(\alpha=0^{\circ}\right)$ direction where the major principal stress is perpendicular to the long axes of the grains. This is seen in the results from Sector I. They are the weakest in the vertical direction $\left(\alpha=90^{\circ}\right)$ when the major principal stress is parallel to the long axes of the grains (seen in Sector III). Results from Sector II show that the effect of the intermediate principal stress, $\sigma_{2}$, when perpendicular to the bedding planes, (see Figure 5.1.1) is not as pronounced as when $\sigma_{2}$ is parallel to the bedding planes.


Figure 5.11.1. Summary of friction angle versus b-value for all Sectors.

In order to see the clear difference among the three Sectors, Table 5.11 .1 summarizes the friction angle by b-values. The difference presented in the last column of the table is calculated by subtracting each of the friction angles from the largest friction angle for that particular b-value. As can be seen, the largest difference is between tests in Sectors I and III. Friction angles for Sector II are closer to Sector I than Sector III for all b-values.

A summary of the calculated dilation angles for all sectors is presented in Figure 5.11.2. As can be seen, for all sectors, the dilation angle increases with $b$-value. The increase for lower
$b$-values is less steep from $b=0$ to $b=0.5$. However, for tests with $b=0.75$ and $b=1.0$, there is a great increase in dilation angle and volumetric change. Tests with varying stress paths have been omitted from this figure as their b-values were not constant and produced some scatter in the dilation angle results. However, they have been previously plotted for their individual sectors.

Table 5.11.1. Summary of Friction angle by b-value for Sectors I, II and III.

$$
\text { Sector }
$$

$\mathbf{b}=\mathbf{0}$

| b-value | $\boldsymbol{\sigma}_{\mathbf{1}} / \boldsymbol{\sigma}_{\mathbf{3}}$ | $\boldsymbol{\phi}$ | $\boldsymbol{\Delta}$ |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: |
| TT\#1 | I | 0.00 | 5.49 | 43.34 | 0.00 |
| TT\#13 | III | 0.00 | 4.16 | 38.02 | -5.32 |

$\mathrm{b}=\mathbf{0 . 2 5}$

| TT\#2 | I | 0.24 | 7.16 | 48.85 | -0.14 |
| :---: | :---: | :---: | :---: | :---: | ---: |
| TT\#8 | II | 0.25 | 7.15 | 48.99 | 0.00 |
| TT\#14 | III | 0.25 | 6.74 | 47.88 | -1.11 |

## $b=0.5$

| TT\#3 | I | 0.50 | 9.23 | 53.41 | 0.00 |
| :---: | :---: | :---: | :---: | :---: | ---: |
| TT\#9 | II | 0.48 | 8.65 | 52.44 | -0.97 |
| TT\#15 | III | 0.48 | 6.78 | 48.15 | -5.26 |

$$
b=0.75
$$

| TT\#4 | I | 0.75 | 9.99 | 54.78 | -1.06 |
| :---: | :---: | :---: | :---: | :---: | :---: |
| $\mathrm{TT} \# 5$ | I | 0.69 | 10.59 | 55.84 | 0.00 |
| TT\#6 | I | 0.72 | 10.32 | 55.42 | -0.42 |
| TT\#10 | II | 0.68 | 10.34 | 55.45 | -0.39 |
| TT\#11 | II | 0.68 | 10.36 | 55.49 | -0.35 |
| TT\#16 | III | 0.71 | 8.94 | 53.17 | -2.67 |
| TT\#17 | III | 0.72 | 9.18 | 53.62 | -2.22 |

$\mathrm{b}=1.0$

| TT\#7 | I | 1.00 | 8.83 | 52.80 | 0.00 |
| :---: | :---: | :---: | :---: | :---: | :---: |
| TT\#12 | II | 0.99 | 8.60 | 52.34 | -0.46 |
| TT\#18 | III | 0.94 | 7.76 | 50.29 | -2.51 |

[^0]

Figure 5.11.2. Summary of Dilation Angle for All Sectors.

In order to plot all of the strength points all in the same octahedral plane, the principal stresses were modified according to the following equation,

$$
\begin{equation*}
\left(\sigma_{1}^{*}, \sigma_{2}^{*}, \sigma_{3}^{*}\right)=\left(\sigma_{1}, \sigma_{2}, \sigma_{3}\right)\left(\frac{\sigma_{o c t}}{\frac{1}{3}\left(\sigma_{1}+\sigma_{2}+\sigma_{3}\right)}\right) \tag{Eq. 5.11.1}
\end{equation*}
$$

where $\sigma_{1}, \sigma_{2}$, and $\sigma_{3}$ are the principal stresses at failure, and $\sigma_{\text {oct }}$ is the octahedral normal stress corresponding to the plane where the results will be plotted. Since only the magnitudes
of the principal stresses are modified, the stress ratios remain constant and therefore, there is no effect on the friction angle. The results were all plotted on a plane corresponding to $\mathrm{I}_{1}=300 \mathrm{kPa}$. Results from all of the tests performed are plotted on the octahedral plane in Figure 5.11.3.

This value of $I_{1}=300 \mathrm{kPa}$ was within the range in which the torsion shear tests were performed, making it a suitable plane for further comparison of test results. The results are plotted in Figure 5.11.4. It is important to note that although the torsion shear tests were performed at $I_{1}=300 \mathrm{kPa}$, most of the true triaxial tests fails around $I_{1}=500 \mathrm{kPa}$. The two planes can be seen in Figure 5.11.5.


Figure 5.11.3. True Triaxial Test Results plotted on the Octahedral Plane.


Figure 5.11.4. Plot of true triaxial failure stresses on $\mathrm{p}_{\mathrm{a}} / \mathrm{I}_{1}$ vs $\left(\mathrm{I}_{1}{ }^{3} / \mathrm{I}_{3}-27\right)$ diagram.

## 6. Torsion Shear Experimental Program

### 6.1 Introduction

Chapter 5 presented tests that were performed in order to fully study the failure surface of Fine Nevada Sand in three dimensions. The true triaxial apparatus allows for application of the three principal stresses, and by the freezing technique implemented, the specimen could be placed in different configurations in order to study its strength at two values of alpha, $0^{\circ}$ and $90^{\circ}$. As already described in Chapter 2, a shortcoming of the true triaxial apparatus is the inability to change principal stress direction. It is to say; different values of alpha besides $0^{\circ}$ and $90^{\circ}$ cannot be applied. However, with a torsion shear apparatus, changes in the principal stress direction can be easily applied.

With the ability to change alpha as well as b-values, torsion shear tests serve as a way to truly study the cross-anisotropic behavior of sand. An experimental program was created consisting of a total of 22 torsion shear tests. These tests are labeled TS 23-44. Each test had a certain alpha and b-value held constant. The mean normal stress was also held constant at 101 kPa . Alpha values ranged from $0^{\circ}$ to $90^{\circ}$ at $22.5^{\circ}$ increments and b-values from 0 to 1 at 0.25 increments.

Test data from 22 previously performed torsion shear tests (Van Dyck 2012) are also included in the experimental results to show the full failure surface of Fine Nevada Sand under all loading conditions. They are labeled as Tests 1-22.

The torsion shear apparatus can be broken down into three main parts: (1) the torsion shear specimen and cell which sits on a rotary turntable, (2) the main panel board which displays inner and outer pressures, measures volume change and applies the back pressure, load and vacuum and (3) the data acquisition system which uses LabView 8.5.1 to change the internal and external pressures and the load for the desired stress path as well as measures and records readings.

### 6.2 Calibration of measurement devices

All test equipment was calibrated in a similar manner to that described in Section 3.2.3. However, the torsion shear apparatus has several components different from the true triaxial apparatus. The torque right and left load cell, along with the vertical load cell were calibrated with a loading machine and proving ring. The volume change devices for both the inner cell and specimen cell (the volume of the outer cell was not measured) were calibrated also. Using LabVIEW, a reading was recorded at the full height of the volume change device. This represented the maximum amount of volume change for the cylinder. The de-aired water was let out of the volume change cylinder and captured in a graduated cylinder. It was weighed and the volume of water captured was recorded. The reading for the empty volume change cylinder was also recorded. By knowing the full span of the volume change cylinder
and the volume it held inside, a calibration was found and the program was adjusted. This procedure was also done for the specimen volume change cylinder.

Inner and outer pressure gage sensors were also calibrated by recording the readings by LabVIEW and comparing them to the readings physically shown on the pressure gages. By adding pressure manually to the gage, different readings were recorded and any offsets in readings were calibrated in the computer program.

Both horizontal and vertical LVDTs were calibrated using a micrometer similar to the method described in Section 3.2.3 for calibration of LVDTs. The entire length of the LVDT cores was calibrated and only the linear portion of the LVDT was used during testing. As a double check on the Horizontal LVDT calibration, a mechanical dial gage was used during testing. Readings were recorded by hand and then were compared to what was recorded during the test. In no instances, were the dial gage and HLVDTs off from each other.

### 6.3 Corrections to the Recorded Data

In a manner similar to that described in Section 3.2.3, certain corrections had to be looked at in order to determine if they would affect the measured stress-strain behavior of the specimens.

When considering membrane strength effects, unlike the triaxial and true triaxial tests, torsion shear specimens use two membranes. Therefore, the membrane effects of both the inner and outer membrane should be accounted for. Torsional membrane strength effects also should be considered. By using the theory of elasticity and assuming that both the inner and outer membranes stay upright during shearing, the torsional membrane strength effects can be found (Tatsuoka et al. 1986). Taking the previously assumed Poisson's Ratio for rubber as 0.5 , the following equations were used to determine membrane strength effects.

$$
\begin{align*}
& \Delta \sigma_{a}=-\frac{4}{3} \frac{E_{m} t_{m}}{r_{o}^{2}-r_{i}^{2}}\left[r_{o}\left\{2\left(\varepsilon_{a m}\right)_{0}+\left(\varepsilon_{\theta m}\right)_{0}\right\}+r_{i}\left\{2\left(\varepsilon_{a m}\right)_{i}+\left(\varepsilon_{\theta m}\right)_{i}\right\}\right.  \tag{Eq. 6.3.1}\\
& \Delta \sigma_{t}=-\frac{2}{3} \frac{E_{m} t_{m}}{r_{0}-r_{i}}\left[\left\{\left(\varepsilon_{a m}\right)_{0}+2\left(\varepsilon_{\theta m}\right)_{0}\right\}+\left\{\left(\varepsilon_{a m}\right)_{i}+2\left(\varepsilon_{\theta m}\right)_{i}\right\}\right.  \tag{Eq. 6.3.2}\\
& \Delta \sigma_{r}=-\frac{2}{3} \frac{E_{m} t_{m}}{r_{0}+r_{i}}\left[\left\{\left(\varepsilon_{a m}\right)_{0}+2\left(\varepsilon_{\theta m}\right)_{0}\right\}-\left\{\left(\varepsilon_{a m}\right)_{i}+2\left(\varepsilon_{\theta m}\right)_{i}\right\}\right.  \tag{Eq. 6.3.3}\\
& \Delta \tau_{\alpha t}=-2 E_{m} t_{m} \frac{r_{0}^{3}+r_{i}^{3}}{\left(r_{0}^{3}-r_{i}^{3}\right)\left(r_{0}+r_{i}\right)} \gamma_{\alpha t} \tag{Eq. 6.3.4}
\end{align*}
$$

where $\Delta \sigma_{\mathrm{a}}, \Delta \sigma_{\mathrm{t}}, \Delta \sigma_{\mathrm{r}}$, and $\Delta \tau_{\mathrm{ct}}$ are the axial, circumferential, radial and shear membrane strengths, respectively. $E_{m}, t_{m}, r_{o}$ and $r_{i}$ are the Young's modulus, membrane thickness, outer radius and inner radius, respectively. $\varepsilon_{\mathrm{a} \theta}$ and $\varepsilon_{\theta \mathrm{m}}$ are the strains in the membrane and they can be calculated by adding the initial membrane strains and the axial and radial strains during isotropic compression and during torsional shear.

The diameters of the inner and outer membranes were 18 cm and 22 cm , respectively. The thickness of the membrane varied from the top to the bottom of the membrane due to its fabrication process. The average thickness was calculated to be 0.047 cm . A Young's modulus of 1400 kPa was used for rubber. The calculated friction angle difference at the maximum strain due to membrane strength effects was 0.005 degrees and therefore, was not included in the calculations.

In tests with changes in effective confining pressure, the volumetric deformation consists of changes in the soil skeleton as well as changes in the volume due to membrane penetration. In cases of hollow cylinder specimens, both inner and outer membranes are used so membrane penetration must be analyzed. Therefore, to accurately measure volume change, the effects of membrane penetration must be analyzed. Per Wong et al. (1975) and Martin et al. (1978), the use of large test specimens will reduce the effects of membrane penetration but will not totally eliminate them. Using Equation 3.2.4, membrane penetration effects were calculated for both the inner and outer membranes. This accounts for about $1 \%$ of the total volume change for the torsion shear tests. Therefore, membrane penetration effects are negligible and not considered in corrections.

Because the piston on the torsion shear apparatus is part of the top cap, piston uplift has to be accounted for. As stated in Section 3.2.2, the vertical uplift can be calculated by multiplying the area of the piston by the total pressure in the outer cell. The uplift force is then subtracted
from the total force applied to the specimen. For tests under compression the uplift force is subtracted from the total downward force. For extension tests, where the vertical force is already in the upward direction the negative uplift force acts as an additional load and therefore increases the total vertical negative load. A free body diagram of the piston uplift that acts on the top cap is shown in Figure 6.3.1.

In certain tests of the experimental program, the uplift pressure was not included in the calculated and applied forces and pressures while shearing the specimen. Therefore, once this correction was added to the recorded test data, the b-values and alpha values were not the targeted values. Once this uplift was corrected in the computer program before shearing, the applied forces and pressures were on target. For tests that are in increments of 0.25 for bvalues from 0 to 1 and increments of $22.5^{\circ}$ of alpha from $0^{\circ}$ to $90^{\circ}$, the correct uplift pressure was accounted for prior to testing. The tests that are somewhat off these incremental targets have been corrected after shearing. Tests where the uplift was not correctly accounted for before shearing have been designated with a * behind their test number. A list of these tests is presented in Table 6.3.1.

Three tests in particular (7*, 13*, and $16^{*}$ ) exhibited an unexpected major principal stress in the radial direction (where $\sigma_{r}>\sigma_{1}$ ). Due to the uniqueness of this condition, the test results do not exactly fit the same conditions as the rest of the tests. These three tests experienced failure in the $\sigma_{z}-\sigma_{\mathrm{r}}$ plane instead of the $\sigma_{\mathrm{z}}-\sigma_{\theta}$ plane, which is where the shear stress was
applied. These tests will be discussed in great detail in Chapter 10. For consistency in presenting the test data, they will be excluded from the failure plots and failure surface results.

Table 6.3.1. List of Torsion Shear Tests. Tests designated with * did not have uplift correction prior to shearing.

| Test No. |  | alpha ( ${ }^{\circ}$ ) | b-value | Test No. |  | alpha $\left(^{\circ}\right.$ ) | b-value |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 1 | * | 0.00 | 0.00 | 23 |  | 0.00 | 0.00 |
| 2 |  | 0.00 | 0.75 | 24 | * | 0.00 | 0.27 |
| 3 |  | 22.41 | 0.00 | 25 | * | 0.00 | 0.55 |
| 4 | * | 23.97 | 0.27 | 26 |  | 0.00 | 1.00 |
| 5 | * | 23.53 | 0.27 | 27 | * | 24.04 | 0.02 |
| 6 |  | 22.48 | 0.50 | 28 |  | 23.69 | 0.23 |
| 7 | * | 20.13 | 0.87 | 29 |  | 22.21 | 0.75 |
| 8 |  | 22.47 | 0.99 | 30 | * | 22.92 | 0.85 |
| 9 |  | 44.98 | 0.25 | 31 |  | 44.71 | 0.02 |
| 10 | * | 42.60 | 0.54 | 32 | * | 31.76 | 0.18 |
| 11 |  | 44.98 | 0.75 | 33 |  | 44.99 | 0.50 |
| 12 | * | 42.26 | 0.80 | 34 |  | 44.95 | 1.00 |
| 13 | * | 33.86 | 0.94 | 35 | * | 69.89 | 0.16 |
| 14 |  | 67.33 | 0.00 | 36 |  | 67.80 | 0.25 |
| 15 |  | 67.47 | 0.50 | 37 | * | 69.91 | 0.55 |
| 16 | * | 72.06 | 0.96 | 38 |  | 67.42 | 0.75 |
| 17 |  | 68.21 | 1.00 | 39 | * | 70.56 | 0.79 |
| 18 |  | 90.00 | 0.04 | 40 | * | 73.08 | 0.80 |
| 19 | * | 90.00 | 0.07 | 41 |  | 90.00 | 0.00 |
| 20 |  | 90.00 | 0.54 | 42 | * | 90.00 | 0.32 |
| 21 |  | 90.00 | 0.78 | 43 | * | 90.00 | 0.78 |
| 22 |  | 90.00 | 0.99 | 44 | * | 90.00 | 0.99 |


(a) Compression

$$
\begin{aligned}
& \Sigma \mathrm{F}_{\mathrm{z}}=0 \\
& \mathrm{~F}_{\mathrm{v}}+\mathrm{p}_{\mathrm{o}}\left(2 \mathrm{r}_{\mathrm{i}}-2 \mathrm{r}_{\mathrm{p}}\right)-\mathrm{p}_{\mathrm{i}}\left(2 \mathrm{r}_{\mathrm{i}}\right)=0
\end{aligned}
$$


(b) Extension

$$
\begin{aligned}
& \Sigma \mathrm{F}_{\mathrm{z}}=0 \\
& -\mathrm{F}_{\mathrm{v}}+\mathrm{p}_{\mathrm{o}}\left(2 \mathrm{r}_{\mathrm{i}}-2 \mathrm{r}_{\mathrm{p}}\right)-\mathrm{p}_{\mathrm{i}}\left(2 \mathrm{r}_{\mathrm{i}}\right)=0
\end{aligned}
$$

Figure 6.3.1. Free body diagram of forces acting in the vertical direction on the top cap of the torsion shear specimen for (a) compression and (b) extension tests.

Since the bushing that housed the piston was not completely frictionless, this friction had to be accounted for as well. By doing a simple friction test which consisted of rotating the torsion shear assembly (without a specimen in place) with the piston in place, it was determined that 9 in-lbs of torque were present in piston friction while rotating the assembly. This load was subtracted from the total torque applied to the specimen during testing. A vertical piston friction of 3 pounds was calculated as well. This friction was also deducted from the vertical load applied to the specimen.

Although all efforts were made to have specimens with the same void ratio, in some cases, it was not possible. Any slight deviation in method during air pluviation can affect the amount of sand that enters into the molds during the specimen preparation. Nonetheless, an equation relating friction angles to void ratio allows for all friction angles to be calculated according to a certain void ratio. By using the equation,
$\mathrm{e} \cdot \tan \varphi=$ constant
Eq. 6.3.5
where e is the void ratio, $\varphi$ is the friction angle and c is a constant, corrected friction angles could be attained. All friction angles were corrected to a void ratio of 0.53 .

The stresses were calculated at mid-height of the specimen and therefore, half of the weight of the soil was added to the vertical load. Because the piston, top cap, torque arms and the
horizontal bar that attached the torque arms to the specimen weighed a considerable amount, this load was also added to the total applied vertical force.

### 6.4 Corrections for Measurement Errors

In order to account for any measurement errors, an analysis was done for both friction angles and b-values. This analysis used the Least Squares Method to determine the error for torsion shear tests, assuming certain values of error in vertical force, $\mathrm{F}_{\mathrm{v}}$, inner and outer pressures, pi and $p_{o}$, inner and outer radii, $r_{i}$ and $r_{o}$ and moment, $M$ values. Details of the equations and methods used to derive the measurement errors are provided in Appendix I.

A table with the summary and the effect on friction angle and b-values is presented in Table 6.4.1. This table summarizes different friction angle variations at the four extremes of the experimental program $\left(\alpha=0^{\circ}, b=0 ; \alpha=0^{\circ}, b=1 ; \alpha=90^{\circ}, b=0\right.$ and $\left.\alpha=90^{\circ}, b=1\right)$. As is seen the friction angle varies from 0.04 to 12.6 degrees for the measurement inaccuracies depending on which measurement errors are assumed. The $b$-value measurement inaccuracies are very small (see Table 6.4.2). They range from 0 to 0.02 . It is important to note that the deviation of inner and outer pressures causes the greatest increase in inaccuracy for friction angle, b value and alpha. The error assumed for these measurements is therefore, the most important.

Table 6.4.1 Table showing effect of various measurement inaccuracies on friction angle for Torsion Shear Tests.

| Test <br> No. | b | $\begin{gathered} \phi \\ \left(^{\circ}\right) \\ \hline \end{gathered}$ | F (kg) | $\Delta F$ | $\begin{gathered} \alpha \\ \left({ }^{\circ}\right) \\ \hline \end{gathered}$ | $\Delta \alpha$ |  | $\Delta p_{0}$ |  | $\Delta p_{i}$ | $\begin{gathered} r_{o} \\ (\mathrm{~cm}) \end{gathered}$ | $\Delta r_{0}$ | $\begin{gathered} r_{i} \\ (\mathrm{~cm}) \end{gathered}$ | $\Delta r_{i}$ | $\Delta \phi\left({ }^{\circ}\right)$ |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 23 | 0 | 41 | 232.3 | 0.5 | 0 | 1 | 0.446 | 0.001 | 0.445 | 0.001 | 11.194 | 0.001 | 9.121 | 0.001 | 0.040 |
| 26 | 1 | 53 | 84.6 | 0.5 | 0 | 1 | 1.354 | 0.001 | 1.617 | 0.001 | 11.115 | 0.001 | 9.096 | 0.001 | 0.252 |
| 18 | 0.1 | 36 | -93.9 | 0.5 | 90 | 1 | 0.704 | 0.001 | 0.423 | 0.001 | 10.798 | 0.001 | 8.763 | 0.001 | 0.083 |
| 22* | 1 | 36 | -130.6 | 0.5 | 90 | 1 | 1.332 | 0.001 | 1.331 | 0.001 | 10.945 | 0.001 | 8.948 | 0.001 | 0.200 |
| Test No. | b | $\begin{gathered} \phi \\ \left(^{\circ}\right) \end{gathered}$ | F (kg) | $\Delta F$ | $\begin{gathered} \alpha \\ \left({ }^{\circ}\right) \end{gathered}$ | $\Delta \alpha$ |  | $\Delta p_{0}$ | $\begin{array}{\|c\|} p_{i} \\ (\mathbf{k g} / \\ \left.\mathbf{c m}^{\wedge} 2\right) \end{array}$ | $\Delta p_{i}$ | $\begin{gathered} \mathbf{r}_{\mathrm{o}} \\ (\mathrm{~cm}) \end{gathered}$ | $\Delta r_{0}$ | $\begin{gathered} r_{i} \\ (\mathrm{~cm}) \end{gathered}$ | $\Delta r_{i}$ | $\Delta \phi\left({ }^{\circ}\right)$ |
| 23 | 0 | 41 | 232.3 | 1 | 0 | 1 | 0.446 | 0.050 | 0.445 | 0.050 | 11.194 | 0.050 | 9.121 | 0.050 | 1.814 |
| 26 | 1 | 53 | 84.6 | 1 | 0 | 1 | 1.354 | 0.050 | 1.617 | 0.050 | 11.115 | 0.050 | 9.096 | 0.050 | 12.582 |
| 18 | 0.1 | 36 | -93.9 | 1 | 90 | 1 | 0.704 | 0.050 | 0.423 | 0.050 | 10.798 | 0.050 | 8.763 | 0.050 | 1.592 |
| 22* | 1 | 36 | -130.6 | 1 | 90 | 1 | 1.332 | 0.050 | 1.331 | 0.050 | 10.945 | 0.050 | 8.948 | 0.050 | 7.218 |
| Test <br> No. | b | $\begin{gathered} \phi \\ \left(^{\circ}\right) \\ \hline \end{gathered}$ | F (kg) | $\Delta F$ | $\begin{array}{\|c} \alpha \\ \left({ }^{\circ}\right) \\ \hline \end{array}$ | $\Delta \alpha$ |  | $\Delta p_{0}$ |  | $\Delta p_{i}$ | $\begin{gathered} \mathbf{r}_{\mathrm{o}} \\ (\mathrm{~cm}) \\ \hline \end{gathered}$ | $\Delta r_{\text {o }}$ | $\begin{gathered} \mathbf{r}_{\mathbf{i}} \\ (\mathrm{cm}) \end{gathered}$ | $\Delta r_{i}$ | $\Delta \phi\left({ }^{\circ}\right)$ |
| 23 | 0 | 41 | 232.3 | 1 | 0 | 1 | 0.446 | 0.010 | 0.445 | 0.010 | 11.194 | 0.010 | 9.121 | 0.010 | 0.364 |
| 26 | 1 | 53 | 84.6 | 1 | 0 | 1 | 1.354 | 0.010 | 1.617 | 0.010 | 11.115 | 0.010 | 9.096 | 0.010 | 2.517 |
| 18 | 0.1 | 36 | -93.9 | 1 | 90 | 1 | 0.704 | 0.010 | 0.423 | 0.010 | 10.798 | 0.010 | 8.763 | 0.010 | 0.411 |
| 22* | 1 | 36 | -130.6 | 1 | 90 | 1 | 1.332 | 0.010 | 1.331 | 0.010 | 10.945 | 0.010 | 8.948 | 0.010 | 1.469 |
| Test No. | b | $\begin{gathered} \phi \\ \left(^{\circ}\right) \end{gathered}$ | F (kg) | $\Delta \mathrm{F}$ | $\begin{gathered} \alpha \\ \left({ }^{\circ}\right) \\ \hline \end{gathered}$ | $\Delta \alpha$ |  | $\Delta p_{0}$ |  | $\Delta p_{i}$ | $\begin{gathered} r_{\mathrm{o}} \\ (\mathrm{~cm}) \\ \hline \end{gathered}$ | $\Delta r_{0}$ | $\begin{gathered} \mathbf{r}_{\mathbf{i}} \\ (\mathrm{cm}) \end{gathered}$ | $\Delta r_{i}$ | $\Delta \phi\left({ }^{\circ}\right)$ |
| 23 | 0 | 41 | 232.3 | 1 | 0 | 1 | 0.446 | 0.020 | 0.445 | 0.020 | 11.194 | 0.020 | 9.121 | 0.020 | 0.726 |
| 26 | 1 | 53 | 84.6 | 1 | 0 | 1 | 1.354 | 0.020 | 1.617 | 0.020 | 11.115 | 0.020 | 9.096 | 0.020 | 5.033 |
| 18 | 0.1 | 36 | -93.9 | 1 | 90 | 1 | 0.704 | 0.020 | 0.423 | 0.020 | 10.798 | 0.020 | 8.763 | 0.020 | 0.652 |
| 22* | 1 | 36 | -130.6 | 1 | 90 | 1 | 1.332 | 0.020 | 1.331 | 0.020 | 10.945 | 0.020 | 8.948 | 0.020 | 2.898 |

Table 6.4.2 Table showing effect of various measurement inaccuracies on b-value for Torsion Shear Tests.

| Test No. | b | $\phi\left({ }^{\circ}\right)$ | F (kg) | $\Delta \mathrm{F}$ | $\alpha\left({ }^{\circ}\right)$ | $\Delta \alpha$ | $\underset{\substack{p_{0}\left(k g / \\ c m^{\wedge}\right.}}{ }$ | $\Delta \mathrm{p}_{\text {o }}$ | $\left\lvert\, \begin{array}{cc} \mathbf{p}_{\mathrm{i}} & (\mathrm{~kg} / \\ \mathrm{cm} \wedge 2) \end{array}\right.$ | $\Delta p_{i}$ | $\mathrm{r}_{0}(\mathrm{~cm})$ | $\Delta r_{\text {o }}$ | $\mathrm{r}_{\mathrm{i}}(\mathrm{cm})$ | $\Delta r_{i}$ | $\Delta \mathrm{b}$-value |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 23 | 0 | 41.4 | 232.3 | 0.5 | 0 | 1 | 0.446 | 0.001 | 0.445 | 0.001 | 11.194 | 0.001 | 9.121 | 0.001 | 0.000 |
| 26 | 1 | 53.1 | 84.6 | 0.5 | 0 | 1 | 1.354 | 0.001 | 1.617 | 0.001 | 11.115 | 0.001 | 9.096 | 0.001 | 0.001 |
| 18 | 0 | 36.1 | -93.9 | 0.5 | 90 | 1 | 0.704 | 0.001 | 0.423 | 0.001 | 10.798 | 0.001 | 8.763 | 0.001 | 0.001 |
| 22* | 1 | 36.1 | -130.6 | 0.5 | 90 | 1 | 1.332 | 0.001 | 1.331 | 0.001 | 10.945 | 0.001 | 8.948 | 0.001 | 0.001 |
| Test No. | b | $\phi\left({ }^{\circ}\right)$ | F (kg) | $\Delta \mathrm{F}$ | $\alpha\left({ }^{\circ}\right)$ | $\Delta \alpha$ | $\begin{gathered} \mathrm{p}_{\mathrm{o}}(\mathrm{~kg} / \\ \left.\mathrm{cm}^{\wedge}\right)^{2} \end{gathered}$ | $\Delta \mathrm{p}_{\text {o }}$ | $\begin{array}{\|cc\|} \hline \mathrm{p}_{\mathrm{i}}(\mathrm{~kg} / \\ \mathrm{cm} \wedge 2) \end{array}$ | $\Delta p_{i}$ | $\mathrm{r}_{0}(\mathrm{~cm})$ | $\Delta r_{0}$ | $\mathrm{r}_{\mathrm{i}}(\mathrm{cm})$ | $\Delta r_{i}$ | $\Delta b$-value |
| 23 | 0 | 41.4 | 232.3 | 1 | 0 | 1 | 0.446 | 0.050 | 0.445 | 0.050 | 11.194 | 0.050 | 9.121 | 0.050 | 0.011 |
| 26 | 1 | 53.1 | 84.6 | 1 | 0 | 1 | 1.354 | 0.050 | 1.617 | 0.050 | 11.115 | 0.050 | 9.096 | 0.050 | 0.012 |
| 18 | 0 | 36.1 | -93.9 | 1 | 90 | 1 | 0.704 | 0.050 | 0.423 | 0.050 | 10.798 | 0.050 | 8.763 | 0.050 | 0.016 |
| 22* | 1 | 36.1 | -130.6 | 1 | 90 | 1 | 1.332 | 0.050 | 1.331 | 0.050 | 10.945 | 0.050 | 8.948 | 0.050 | 0.030 |
| Test No. | b | $\phi\left({ }^{\circ}\right)$ | F (kg) | $\Delta \mathrm{F}$ | $\alpha\left({ }^{\circ}\right)$ | $\Delta \alpha$ | $\begin{aligned} & \hline \mathrm{p}_{0}(\mathrm{~kg} / \\ & \mathrm{cm} \wedge \end{aligned}$ | $\Delta \mathrm{p}_{\text {o }}$ | $\begin{array}{\|cc\|} \hline p_{i} \quad(\mathrm{~kg} / \\ \mathrm{cm} \wedge 2) \end{array}$ | $\Delta \mathrm{p}_{\mathrm{i}}$ | $\mathrm{r}_{0}(\mathrm{~cm})$ | $\Delta \mathrm{r}_{\text {}}$ | $\mathrm{r}_{\mathrm{i}}(\mathrm{cm})$ | $\Delta r_{i}$ | $\Delta \mathrm{b}$-value |
| 23 | 0 | 41.4 | 232.3 | 1 | 0 | 1 | 0.446 | 0.010 | 0.445 | 0.010 | 11.194 | 0.010 | 9.121 | 0.010 | 0.002 |
| 26 | 1 | 53.1 | 84.6 | 1 | 0 | 1 | 1.354 | 0.010 | 1.617 | 0.010 | 11.115 | 0.010 | 9.096 | 0.010 | 0.003 |
| 18 | 0 | 36.1 | -93.9 | 1 | 90 | 1 | 0.704 | 0.010 | 0.423 | 0.010 | 10.798 | 0.010 | 8.763 | 0.010 | 0.004 |
| 22* | 1 | 36.1 | -130.6 | 1 | 90 | 1 | 1.332 | 0.010 | 1.331 | 0.010 | 10.945 | 0.010 | 8.948 | 0.010 | 0.006 |
| Test No. | b | $\phi\left({ }^{\circ}\right)$ | F (kg) | $\Delta \mathrm{F}$ | $\alpha\left({ }^{\circ}\right)$ | $\Delta \alpha$ | $\begin{gathered} \mathrm{p}_{\mathrm{o}}(\mathrm{~kg} / \\ \left.\mathrm{cm}^{\wedge} 2\right) \end{gathered}$ | $\Delta \mathrm{p}_{\text {o }}$ | $\begin{array}{\|cc} \mathrm{p}_{\mathrm{i}}(\mathrm{~kg} / \\ \mathrm{cm} \wedge 2) \end{array}$ | $\Delta p_{i}$ | $\mathrm{r}_{0}(\mathrm{~cm})$ | $\Delta r_{\text {o }}$ | $\mathrm{r}_{\mathrm{i}}(\mathrm{cm})$ | $\Delta r_{i}$ | $\Delta b$-value |
| 23 | 0 | 41.4 | 232.3 | 1 | 0 | 1 | 0.446 | 0.020 | 0.445 | 0.020 | 11.194 | 0.020 | 9.121 | 0.020 | 0.004 |
| 26 | 1 | 53.1 | 84.6 | 1 | 0 | 1 | 1.354 | 0.020 | 1.617 | 0.020 | 11.115 | 0.020 | 9.096 | 0.020 | 0.005 |
| 18 | 0 | 36.1 | -93.9 | 1 | 90 | 1 | 0.704 | 0.020 | 0.423 | 0.020 | 10.798 | 0.020 | 8.763 | 0.020 | 0.007 |
| 22* | 1 | 36.1 | -130.6 | 1 | 90 | 1 | 1.332 | 0.020 | 1.331 | 0.020 | 10.945 | 0.020 | 8.948 | 0.020 | 0.012 |

After consideration of the torsion shear apparatus used and the consistency of running the experiments, measurement errors for the tests performed were selected. These are summarized Table 6.4.3. Since each test is different, there is no constant measurement at failure. Therefore, an average of about $0.5 \%$ of the measurements taken at failure has been chosen for the measurement errors assumed in this analysis.

Table 6.4.3 Summary of Measurement Errors estimated for Torsion Shear Tests.

| Measurement Errors |  |
| :---: | :---: |
| $\Delta \mathrm{F}_{\mathrm{v}}(\mathrm{kN})$ | 0.010 |
| $\Delta \mathrm{M}(\mathrm{kN}-\mathrm{cm})$ | 0.049 |
| $\Delta \mathrm{P}_{\mathrm{o}}(\mathrm{kPa})$ | 0.490 |
| $\Delta \mathrm{P}_{\mathrm{i}}(\mathrm{kPa})$ | 0.490 |

The resulting measurement errors for friction angle, b-value and alpha are shown in Table 6.4.4 for each test performed. The friction angle varies from 0.46 to 3.6 degrees. Figures 6.4.1, 6.4.2 and 6.4.3 plot the inaccuracies for all tests. They are separated by alpha values. As can be seen on the figures, the highest friction angle inaccuracy occurs at $\alpha=0^{\circ}$ at higher b-values. As alpha increases, the inaccuracies decrease. Overall, they tend to stay at around one to two degrees. The b -value error shows an increase in measurement variation as the b value increases. The alpha measurement errors vary from 0.1 to 0.5 degrees and show a slight increase with higher b-values.

Table 6.4.4. Measurement inaccuracies of friction angle and $b$-value for all Torsion Shear Tests.

| Test No. | b | $\phi\left({ }^{\circ}\right)$ | $\alpha\left({ }^{\circ}\right)$ | $\mathrm{F}(\mathrm{kN})$ | $\Delta \mathrm{F}(\mathrm{kN})$ | $\mathrm{M}\left(\mathrm{kN*}{ }^{*} \mathrm{~cm}\right)$ | $\Delta \mathrm{M}\left(\mathrm{kN} \mathrm{N}^{*} \mathrm{~cm}\right)$ | $\mathrm{p}_{0}(\mathrm{kPa})$ | $\Delta \mathrm{p}_{0}(\mathrm{kPa})$ | $\mathrm{p}_{\mathrm{i}}(\mathrm{kPa})$ | $\Delta \mathrm{p}_{\mathrm{i}}(\mathrm{kPa})$ | $\Delta \phi\left({ }^{\circ}\right)$ | $\Delta b$-value | $\Delta \alpha\left({ }^{\circ}\right)$ |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 23 | 0.00 | 41.57 | 0 | 2.279 | 0.01 | 0.000 | 0.049 | 43.752 | 0.490 | 43.683 | 0.490 | 1.494 | 0.014 | 0.123 |
| 1* | 0.01 | 37.36 | 0 | 1.928 | 0.01 | 0.000 | 0.049 | 46.583 | 0.490 | 46.877 | 0.490 | 1.428 | 0.017 | 0.145 |
| 24* | 0.27 | 45.89 | 0 | 1.723 | 0.01 | 0.000 | 0.049 | 69.081 | 0.490 | 77.540 | 0.490 | 2.054 | 0.016 | 0.141 |
| 25* | 0.55 | 53.51 | 0 | 1.414 | 0.01 | 0.000 | 0.049 | 93.763 | 0.490 | 110.723 | 0.490 | 3.054 | 0.017 | 0.144 |
| 2 | 0.75 | 57.02 | 0 | 1.208 | 0.01 | 0.000 | 0.049 | 116.245 | 0.490 | 138.865 | 0.490 | 3.564 | 0.017 | 0.146 |
| 26 | 1.00 | 53.27 | 0 | 0.828 | 0.01 | 0.000 | 0.049 | 132.702 | 0.490 | 158.569 | 0.490 | 3.488 | 0.020 | 0.172 |
|  |  |  |  |  |  |  |  |  |  |  |  |  |  |  |
| 3 | 0.00 | 39.4 | 22.38 | 1.689 | 0.01 | 7.753 | 0.049 | 48.129 | 0.490 | 43.096 | 0.490 | 1.321 | 0.015 | 0.239 |
| 27* | 0.02 | 36.34 | 24.04 | 1.428 | 0.01 | 7.243 | 0.049 | 52.335 | 0.490 | 47.853 | 0.490 | 1.278 | 0.017 | 0.279 |
| 28 | 0.26 | 44.48 | 22.97 | 1.310 | 0.01 | 7.497 | 0.049 | 73.711 | 0.490 | 77.227 | 0.490 | 1.749 | 0.016 | 0.257 |
| 4* | 0.27 | 47.03 | 23.73 | 1.282 | 0.01 | 7.719 | 0.049 | 75.228 | 0.490 | 71.780 | 0.490 | 1.233 | 0.017 | 0.292 |
| 5* | 0.27 | 41.39 | 24.18 | 1.168 | 0.01 | 6.984 | 0.049 | 74.124 | 0.490 | 77.365 | 0.490 | 1.655 | 0.017 | 0.285 |
| 6 | 0.51 | 46.31 | 22.01 | 0.993 | 0.01 | 6.611 | 0.049 | 96.465 | 0.490 | 107.360 | 0.490 | 2.126 | 0.017 | 0.276 |
| 29 | 0.75 | 46.75 | 22.77 | 0.679 | 0.01 | 6.131 | 0.049 | 116.393 | 0.490 | 132.390 | 0.490 | 2.341 | 0.017 | 0.310 |
| 7* | 0.83 | 40.7 | 24.11 | 0.215 | 0.01 | 4.709 | 0.049 | 129.218 | 0.490 | 148.663 | 0.490 | 2.374 | 0.026 | 0.438 |
| 30* | 0.85 | 46.18 | 22.92 | 0.554 | 0.01 | 5.671 | 0.049 | 116.737 | 0.490 | 133.355 | 0.490 | 2.476 | 0.021 | 0.340 |
| 8 | 0.97 | 41.87 | 22.87 | 0.416 | 0.01 | 5.187 | 0.049 | 130321 | 0.490 | 149.559 | 0.490 | 2.211 | 0.023 | 0.361 |
|  |  |  |  |  |  |  |  |  |  |  |  |  |  |  |
| 31 | 0.00 | 36.25 | 44.91 | 0.476 | 0.01 | 9.458 | 0.049 | 59.162 | 0.490 | 43.854 | 0.490 | 0.892 | 0.017 | 0.360 |
| $32 *$ | 0.18 | 40.58 | 31.76 | -0.117 | 0.01 | 8.385 | 0.049 | 63.230 | 0.490 | 49.646 | 0.490 | 1.186 | 0.019 | 0.371 |
| 9 | 0.24 | 38.85 | 44.99 | 0.232 | 0.01 | 9.223 | 0.049 | 80.675 | 0.490 | 73.366 | 0.490 | 1.136 | 0.018 | 0.369 |
| 33 | 0.50 | 45.02 | 44.99 | 0.008 | 0.01 | 9.319 | 0.049 | 101.430 | 0.490 | 101.292 | 0.490 | 1.559 | 0.018 | 0.366 |
| 10* | 0.55 | 40.55 | 41.96 | -0.161 | 0.01 | 7.743 | 0.049 | 101.430 | 0.490 | 101.292 | 0.490 | 1.330 | 0.021 | 0.435 |
| 11 | 0.75 | 40.36 | 44.98 | -0.184 | 0.01 | 7.536 | 0.049 | 118.323 | 0.490 | 124.184 | 0.490 | 1.451 | 0.022 | 0.452 |
| 12* | 0.81 | 41 | 41.54 | -0.348 | 0.01 | 7.079 | 0.049 | 117.565 | 0.490 | 123.219 | 0.490 | 1.527 | 0.023 | 0.475 |
| 13* | 0.94 | 39.2 | 33.86 | -0.486 | 0.01 | 6.143 | 0.049 | 12.563 | 0.490 | 139.354 | 0.490 | 1.518 | 0.026 | 0.543 |
| 34 | 1.00 | 35.61 | 44.95 | -0.307 | 0.01 | 6.230 | 0.049 | 129.356 | 0.490 | 139.285 | 0.490 | 1.331 | 0.026 | 0.547 |
|  |  |  |  |  |  |  |  |  |  |  |  |  |  |  |
| 14 | 0.00 | 35.01 | 67.50 | -0.501 | 0.01 | 6.266 | 0.049 | 65.367 | 0.490 | 40.062 | 0.490 | 0.563 | 0.018 | 0.286 |
| 35* | 0.16 | 37.74 | 69.85 | -0.585 | 0.01 | 5.544 | 0.049 | 68.884 | 0.490 | 46.819 | 0.490 | 0.639 | 0.019 | 0.293 |
| 36 | 0.25 | 37.57 | 67.68 | -0.705 | 0.01 | 6.107 | 0.049 | 86.122 | 0.490 | 69.091 | 0.490 | 0.734 | 0.019 | 0.298 |
| 15 | 0.50 | 39.52 | 67.40 | -0.862 | 0.01 | 5.765 | 0.049 | 105.636 | 0.490 | 96.327 | 0.490 | 0.897 | 0.020 | 0.319 |
| 37* | 0.56 | 38.19 | 70.03 | -0.899 | 0.01 | 4.767 | 0.049 | 104.878 | 0.490 | 96.948 | 0.490 | 0.887 | 0.022 | 0.325 |
| 38 | 0.75 | 32.98 | 67.54 | -0.836 | 0.01 | 4.501 | 0.049 | 119.220 | 0.490 | 117.151 | 0.490 | 0.743 | 0.026 | 0.408 |
| 39* | 0.79 | 34.68 | 70.62 | -0.915 | 0.01 | 3.877 | 0.049 | 117.289 | 0.490 | 115.427 | 0.490 | 0.815 | 0.026 | 0.383 |
| 40* | 0.80 | 34.97 | 73.06 | -0.987 | 0.01 | 3.558 | 0.049 | 127.632 | 0.490 | 129.907 | 0.490 | 0.887 | 0.028 | 0.406 |
| 16* | 0.80 | 30.68 | 74.14 | -0.923 | 0.01 | 3.081 | 0.049 | 125.494 | 0.490 | 127.839 | 0.490 | 0.720 | 0.031 | 0.437 |
| 17 | 1.00 | 37.11 | 68.26 | -1.028 | 0.01 | 4.610 | 0.049 | 135.217 | 0.490 | 138.389 | 0.490 | 1.012 | 0.025 | 0.396 |
|  |  |  |  |  |  |  |  |  |  |  |  |  |  |  |
| 41 | 0.00 | 34.27 | 90.00 | -0.865 | 0.01 | 0.000 | 0.049 | 68.532 | 0.490 | 39.710 | 0.490 | 0.460 | 0.019 | 0.159 |
| 18 | 0.06 | 36.32 | 90.00 | -0.921 | 0.01 | 0.000 | 0.049 | 69.008 | 0.490 | 41.468 | 0.490 | 0.535 | 0.018 | 0.158 |
| 19* | 0.08 | 40.37 | 90.00 | -1.035 | 0.01 | 0.000 | 0.049 | 66.815 | 0.490 | 36.959 | 0.490 | 0.639 | 0.017 | 0.143 |
| 42* | 0.32 | 45.1 | 90.00 | -1.230 | 0.01 | 0.000 | 0.049 | 87.777 | 0.490 | 66.815 | 0.490 | 0.904 | 0.017 | 0.149 |
| 20* | 0.52 | 41.81 | 90.00 | -1.277 | 0.01 | 0.000 | 0.049 | 108.394 | 0.490 | 97.500 | 0.490 | 0.970 | 0.020 | 0.173 |
| 43* | 0.78 | 40.41 | 90.00 | -1.252 | 0.01 | 0.000 | 0.049 | 120.185 | 0.490 | 114.945 | 0.490 | 0.916 | 0.023 | 0.198 |
| $21^{*}$ | 0.78 | 41.27 | 90.00 | -1.339 | 0.01 | 0.000 | 0.049 | 121.702 | 0.490 | 116.048 | 0.490 | 1.038 | 0.022 | 0.185 |
| $22^{*}$ | 0.99 | 39.36 | 90.00 | -1.281 | 0.01 | 0.000 | 0.049 | 137.527 | 0.490 | 130.500 | 0.490 | 0.573 | 0.022 | 0.187 |
| 44* | 0.99 | 35.72 | 90.00 | -1.240 | 0.01 | 0.000 | 0.049 | 129.425 | 0.490 | 129.287 | 0.490 | 0.876 | 0.026 | 0.225 |



Figure 6.4.1. Torsion Shear measurement inaccuracies in friction angle for all tests.


Figure 6.4.2. Torsion Shear measurement inaccuracies in b-value for all tests.


Figure 6.4.3. Torsion Shear measurement inaccuracies in alpha value for all tests.

### 6.5 Description of Torsion Shear Specimen and Cell

## Specimen preparation

In order to attain comparable results, all specimens were made using a prefabricated inner and outer mold, which created a specimen with the same dimensions. The inner and outer radius of the specimens, were 9 cm and 11 cm , respectively. The specimen height was 40 cm . For each test, the inner and outer radius, as well as the height of the specimen, was physically measured in order to calculate the void ratio of the specimen. A list of the tests as well as the void ratio and relative density is provided in Table 6.5.1.

Table 6.5.1. Summary of Torsion Shear Tests at different alphas and b-values.

| Test <br> No. | alpha <br> $\left({ }^{\circ}\right)$ | b- <br> value | void <br> ratio | relative <br> density <br> $(\%)$ | Test <br> No. | alpha <br> $\left({ }^{\circ}\right)$ | b- <br> value | void <br> ratio | relative <br> density <br> $(\%)$ |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| $1 *^{*}$ | 0.00 | 0.00 | 0.510 | 98.86 | 23 | 0.00 | 0.00 | 0.531 | 90.91 |
| 2 | 0.00 | 0.75 | 0.530 | 91.29 | 24 | $*$ | 0.00 | 0.27 | 0.530 |
| 91.29 |  |  |  |  |  |  |  |  |  |
| 3 | 22.41 | 0.00 | 0.523 | 93.94 | 25 | $*$ | 0.00 | 0.55 | 0.530 |
| 91.29 |  |  |  |  |  |  |  |  |  |
| 4 | $*$ | 23.97 | 0.27 | 0.548 | 84.47 | 26 | 0.00 | 1.00 | 0.532 |
| 90.53 |  |  |  |  |  |  |  |  |  |
| 5 | $*$ | 23.53 | 0.27 | 0.524 | 93.56 | 27 | $*$ | 24.04 | 0.02 |

Before building each specimen, dry Fine Nevada sand was sieved through a No. 20 sieve ( $850 \mu \mathrm{~m}$ opening), to ensure that no larger grains or particles were in the specimen. An inner membrane was placed tightly around an inner form, which was secured to the inner radius of the bottom base ring using an O-ring. A thin layer of 2-ton epoxy was spread along the top of the base ring to allow the sand grains that were deposited to interlock with the base ring. The bottom base ring had four drainage paths covered by filter stones. The bottom base ring was screwed into the base turntable where the drainage holes on the turntable were located. This sealed the membrane and drainage lines. The inner membrane fit tight enough around the inner form so that no vacuum was necessary to hold it up. At this point, the radius of the inner membrane form was measured using a Pi Tape and recorded. The outer membrane was then placed over the inner form and secured to the outer radius of the bottom ring with an Oring. The outer forming jacket was assembled around the outer radius of the base ring. A vacuum was applied to the forming jacket so that the outer membrane was held tightly around the outer form. With the two membranes in place, the measured sand was put into a funnel above the torsion shear apparatus and slowly deposited using the air pluviation method. The sand exited the funnel by a small tube, which was constantly moved around the entire circumference of the specimen walls. Empirical tests showed that a drop height of 35 cm created the desired void ratio $(\mathrm{e}=0.53)$.

A small 35 cm rod at the end of the deposition tube and a flashlight were used to ensure that the drop height was kept constant. The funnel was lifted ensuring that the 35 cm rod never touched the surface of the sand. Bedding planes were kept as horizontal as possible by carefully watching the deposition of sand as well as moving the tube and funnel throughout the deposition process. Splashguards were also placed on the inner and outer membrane in order to prevent losing any sand. Once all of the sand was deposited, any grains that might have fallen outside the specimen were vacuumed, weighed and recorded to ensure a proper calculation of the void ratio. The top surface was leveled and a thin layer of 2-ton clear epoxy was placed underneath the top ring before placing it on the specimen. The top ring was pressed down on the upper layer of sand grains, interlocking the top ring to the sand grains. The epoxy was used in order to transfer shear stresses from the top and base rings to the specimen, avoiding slippage at the interfaces. Using two O-rings, the inner and outer membranes were secured to the top cap. Similar to the base ring, the top ring had four drainage paths that were covered by filter stones. Drainage lines, which led to the volume change device, were connected to the top ring. A bubble chamber, which was connected to these drainage lines, was at this time also connected.

A small vacuum of 48 kPa was applied to the specimen and the outer and inner forms were removed. With the bubble chamber attached, any holes in the membranes were indicated once the forms were removed. The specimen inner and outer membranes were painted with rubber latex liquid glue until any holes were plugged. The rubber latex liquid glue was allowed to completely dry before any additional layers were painted on the membranes. Once
any leaks were stopped, a grid of horizontal and vertical lines was drawn on the outer front membrane using a felt pen with waterproof ink or a permanent marker. This allowed for a visual check during shearing and as a way to notice shear bands as they developed. A picture of each specimen was taken prior to shearing and the height at three places as well as the outer radius were measured and recorded. Finally, a top cap with four toggles was screwed on to the top ring. The top cap had a greased O-ring that sealed the connection between the top cap and the top ring. The top cap also had a small rod to which the piston would then be attached later. A picture of a prepared specimen standing under a vacuum of 48 kPa with the grid lines drawn can be seen in Figure 6.5.1.


Figure 6.5.1. Picture of a torsion shear specimen prior to saturation.

With the specimen ready for the saturation process, gaseous Carbon Dioxide $\left(\mathrm{CO}_{2}\right)$ was slowly introduced into the inner cell. The four toggles on the top cap were left open for about 15 minutes to let the $\mathrm{CO}_{2}$ push any air out of the inner cell. After approximately 15 minutes, de-aired water slowly filled the inner cell. Once the deaired water came out of the top cap toggles, three toggles were closed, the deaired water was stopped and the fourth toggle was then closed. This ensured that water filled the entire inner cell and that no pressure was trapped in the inner cell.

Once the inner cell was completely saturated, the outer cell acrylic cylinder was placed around the specimen. The acrylic cylinder sat over a greased O-ring that was placed in a groove in the outer circumference of the base plate. A top lid was carefully placed on top of the acrylic cylinder. The top lid also had a greased O-ring inside a groove and sat precisely over the acrylic cylinder. It was important to check the O-rings as any slips or discontinuities in the O-rings could cause the outer cell to leak during shearing.

The piston was slightly greased and inserted through the top lid piston sheath and secured with a 17 mm bolt so that the piston was attached to the top cap that was resting on the specimen. The greased piston sheath was designed to minimize friction through ball bearings inside the piston sheath. Six vertical tie rods connected the top lid to the base plate. The tie rods were tightened with threaded bolts. It was also important to have these as tight as
possible to ensure that no water was able to leak out of the outer cylinder during testing. Any leaks affect the pressures applied during testing and can severely affect the results.

With the entire outside cell sealed, deaired water was slowly introduced to the system. Once full and some water had leaked out of the top cap drainage vents, the water was stopped and the vents were closed. At this point, pressure was applied to the inner and outer cell simultaneously. This could be done manually or using the LabView program. As the pressure was applied, the vacuum on the specimen was lowered. Once 48 kPa was applied to both inner and outer cells, the bubble chamber was disconnected and the backpressure line was directly connected to the main control board, in order to saturate the specimen.

Using the $\mathrm{CO}_{2}$ method, gaseous $\mathrm{CO}_{2}$ was slowly passed from the bottom through the specimen and out the top drainage line. A small line with a three-way valve was connected to the drainage lines and led out of the base plate. When flipped in one direction, the three-way valve led to a small pig-tail that was placed in a glass of water in order to see how fast the $\mathrm{CO}_{2}$ was flowing through the specimen. The specimen was saturated with $\mathrm{CO}_{2}$ for 15 minutes. Then, deaired water was introduced through the bottom ring, filled the specimen, went out of the top ring, through the drainage lines and into the glass of water. Water was allowed to flow through the specimen for about 15 minutes to ensure full saturation. Then, the three-way valve was flipped to a pressure transducer, which would be used in measuring
the saturation of the specimen when performing the B-value test to check the degree of saturation.

A back pressure of 48 kPa was applied to the specimen while the inner and outer cell pressures were simultaneously raised to 98 kPa and left on for a minimum of two hours. In most cases, the specimen was left overnight with the back pressure applied to let the specimen fully saturate. Skempton's B-value saturation test was performed to check the degree of saturation by recording the initial value of the pressure transducer, closing the drainage lines, and increasing both the inner and outer cell pressure simultaneously. The new pressure transducer reading and pressures were recorded and the pressures were brought back down to their values prior to the increase. The drainage lines were again opened, and the Bvalue was calculated according to the following equation, which was also described in Section 3.5.1.

$$
\begin{equation*}
B=\frac{\Delta u}{\Delta \sigma_{3}} \tag{Eq. 6.5.1}
\end{equation*}
$$

where $\Delta u$ is the change in back pressure recorded from the pressure transducer and $\Delta \sigma_{3}$ is the change in both inner and outer pressures.

Once this was recorded, the three-way valve was closed and the preparation for the test was continued. A picture of the torsion shear specimen held under inner and outer pressure of 48 kPa and fully saturated can be seen in Figure 6.5.2.


Figure 6.5.2. Picture of Torsion Shear Specimen under 48 kPa water pressure after it has been fully saturated.

### 6.6 Set-up of Torsion Shear Apparatus

Connection of Instrumentation
With the specimen ready for testing, several more components had to be added before the specimen could be sheared. Measuring devices such as the vertical LVDT, horizontal

LVDT, vertical load cell, torque arms and torque load cells had to be correctly placed on the assembly prior to testing.

Before any of the instrumentation was assembled, a reinforced rigid body frame was moved up and bolted down to the base of where the turntable was fastened. This created an entire rigid body frame, allowing for accurate measurement of loads for both the vertical load and torque load cells.

A vertical LVDT was secured to the piston in order to measure any axial deformations. An extension rod was placed on a top bolt on the top lid in order to set the horizontal LVDT. The horizontal LVDT assembly had several parts. First, a pie shaped plate with a groove along its curved edge was secured on the piston. One end of the core of the horizontal LVDT was connected to the pie with a radio wire cord. The radio wire cord was fastened with a setscrew and ran along the groove of the pie. Once it exited the LVDT core, the other end of the horizontal LVDT core was connected to another piece of radio wire, which had a weight at the end of it. A frictionless wheel set at the end of the assembly allowed the weight to drop down vertically. The assembly was positioned so that a straight line was established from the far edge of the pie shaped plate, through the LVDT core and to the frictionless wheel.

A load cell was placed on a piston adaptor that allowed the load cell to be set between the piston and an air pressure cylinder located at the top of the rigid body frame. If the piston was not precisely in line with the air pressure cylinder collar, the collar was moved so that no additional torque was applied.

In order to measure the shear stresses, a torque arm assembly was placed perpendicular to the piston. The assembly was secured around the piston and two torque arms with load cells were connected to a back plate of the rigid body frame. Springs were placed between the torque load cells and the rigid body frame to allow for compression and extension of the arms. A picture of a typical torsion shear specimen with the instrumentation set up is shown in Figure 6.6.1.


Figure 6.6.1. Picture of typical torsion shear specimen with all of its instrumentation set up (HLVDT, VLVDT, Torque sensors, and Load cell).

## Main Panel Board

Section 6.5 described the preparation, assembly and instrumentation devices used for the torsion shear tests that were performed. A main panel board applied pressures and loads, as well as measured the volume change of the specimen during shearing. Two pressure gages displayed the inner and outer pressures applied to the system. This pressure could be applied both manually with a manual regulator and automatically with the computer if the LabView program was turned on. An auto on/off switch valve allowed for the switch between automatic and manual. The back pressure was applied manually to the specimen with a manual regulator and pressure gage. The vertical load could also be applied both manually and automatically to the specimen. Lower and upper air pressure lines were connected to the air pressure cylinder allowing for both compression and extension tests to be performed. Two additional manual regulators were provided to apply vacuum to the specimen and forming jacket during assembly and disassembly. A gage connected to a switch valve between the forming jacket and the specimen displayed the respective applied vacuum.

Two deaired water tanks were also connected to the main panel by lines that led to either the volume change device or the inner and outer pressure lines leading to the torsion shear cell. Pressure sensors on the inner and outer pressure lines allowed for accurate measurements with the computer. A visual check was provided by the previously described inner and outer pressure gages. Although three volume change devices were used for inner, outer and
specimen volume change, sensors were only placed to measure volume change for the inner and specimen volume. The outer volume change is not accounted for in any calculations. A picture of the main panel board and entire assembly can be seen in Figure 6.3.2.


Figure 6.6.2. Typical Picture of a saturated Torsion Shear Specimen before shearing (showing the main panel board, data acquisition devices, air pressure cylinder, stability frame and power supply).

### 6.7 Equations used for Stress Paths and Strain Calculations

The experimental program designed to study the failure surface of Fine Nevada sand required that a certain b-value, alpha value and mean normal stress were all three held
constant during shearing of the specimen. By using the equations presented in Chapter 2, section 2.3.2 and combining them, a set of three equations and three unknowns can be derived and used for the conditions desired for the testing program (i.e. constant alpha, b values and mean normal stress).

A Mohr circle representation of principal stresses for a hollow cylinder specimen is shown in Figure 6.7.1. A stress block from the hollow cylinder torsion shear specimen is shown in Figure 6.7.2. This stress block shows the main stresses: $\sigma_{r}, \sigma_{z}, \sigma_{\theta}$ and $\tau_{z \theta}$.


Figure 6.7.1. Mohr Circle Representation of Stress in the Wall of a hollow cylinder specimen (after Hight et al. 1983).


Figure 6.7.2. Stress and deformation states for an hollow cylindrical specimen (a) applied loads and pressures (b) deformations (c) induced stresses and (d) principal.

From Mohr's cicles, the principal stresses can be derived in terms of $\sigma_{r}, \sigma_{z}, \sigma_{\theta}$ and $\alpha$. The equations for the major, intermediate, minor principal stresses and principal stress direction, discussed in Chapter 2, Section 2.3.2 are restated as:

$$
\begin{equation*}
\sigma_{1}=\frac{\sigma_{z}+\sigma_{\theta}}{2}+\sqrt{\left(\frac{\sigma_{z}-\sigma_{\theta}}{2}\right)^{2}+\tau_{z \theta}^{2}} \tag{Eq. 6.7.1}
\end{equation*}
$$

$\sigma_{2}=\sigma_{r}$
Eq. 6.7.2
$\sigma_{1}=\frac{\sigma_{z}+\sigma_{\theta}}{2}-\sqrt{\left(\frac{\sigma_{z}-\sigma_{\theta}}{2}\right)^{2}+\tau_{z \theta}^{2}}$
Eq. 6.7.3
$\alpha=\frac{1}{2} \tan ^{-1}\left(\frac{2 \tau_{z \theta}}{\sigma_{z}-\sigma_{z \theta}}\right)$
Eq. 6.7.4

The intermediate principal stress ratio can be written as
$b=\left(\frac{\sigma_{2}-\sigma_{3}}{\sigma_{1}-\sigma_{3}}\right)$
Eq. 6.7.5

The mean normal stress is calculated as
$\sigma_{m}=\frac{1}{3}\left(\sigma_{1}+\sigma_{2}+\sigma_{3}\right)$
Eq. 6.7.6

It is also necessary to know the four basic equations of induced stresses with respect to inner and outer pressures, inner and outer radiuses, vertical load and torque. These equations are based on equilibrium considerations and are independent of the constitutive law of the material being tests (see Section 2.3.2.1). Figure 6.7 .3 shows the free body diagram of a cross-section of the specimen.

(a) Plan View
(b) Side view

Figure 6.7.3. Torsion Shear Specimen showing Stresses and Pressures (a) Plan View and (b) Side view.

It is seen in that when the sum of the forces in the x direction is set to zero,
$2 \sigma_{\theta}(t) d z+p_{i} 2 r_{i} d z-p_{o} 2 r_{o} d z=0$
Eq. 6.7.7
where $t=r_{o}-r_{i}$ and $d z=1$ yields,
$\sigma_{\theta}\left(r_{o}-r_{i}\right) d z=p_{o} r_{o}-p_{i} r_{i}$
Eq. 6.7.8

Solving for $\sigma_{\theta}$ gives,

$$
\begin{equation*}
\sigma_{\theta}=\frac{p_{o} r_{o}-p_{i} r_{i}}{r_{o}-r_{i}} \tag{Eq 6.7.9}
\end{equation*}
$$

When looking at stresses in the vertical direction, it is necessary to look at the free body diagram of the forces acting on the specimen in the z -direction. In this case, the areas of the inner and outer specimen have to be multiplied by the stress imposed by the inner and outer pressures. The stress in the element also has to be multiplied by the area of the element. Vertical force is also added to the calculation. Figure 6.7.4 illustrates the free body diagram.

where

$$
\begin{aligned}
& \mathrm{A}_{\mathrm{s}}=\pi\left(\mathrm{r}_{\mathrm{o}}^{2}-\mathrm{r}_{\mathrm{i}}^{2}\right) \\
& \mathrm{A}_{\mathrm{i}}=\pi \mathrm{r}_{\mathrm{i}}^{2} \\
& \mathrm{~A}_{\mathrm{o}}=\pi \mathrm{r}_{\mathrm{o}}^{2}
\end{aligned}
$$

Figure 6.7.4. Vertical forces acting on a Torsion Shear Specimen.

Since force is stress times area, the stresses on the specimen are summed up so that the forces in the $z$-direction equal zero. It can be said then that,
$p_{o}\left(\pi r_{o}^{2}\right)+F_{v}-p_{i}\left(\pi r_{i}^{2}\right)=\sigma_{z}\left(r_{o}^{2}-r_{i}^{2}\right)$
Eq. 6.7.10

Solving for $\sigma_{z}$ gives
$\sigma_{z}=\frac{F_{v}}{\pi\left(r_{o}^{2}-r_{i}^{2}\right)}+\frac{p_{o} r_{o}^{2}-p_{i} r_{i}^{2}}{\left(r_{o}^{2}-r_{i}^{2}\right)}$
Eq. 6.7.11

In a similar manner, the radial stress can be derived by summing up the stresses in the $y$ direction. The free body diagram is shown in Figure 6.7.5.


Figure 6.7.5. Radial forces acting on a Torsion Shear Specimen.
$p_{i} \frac{2 \pi r_{i}}{2} d z+p_{o} \frac{2 \pi r_{o}}{2} d z=\sigma_{r} 2 \pi\left(\frac{r_{i}+r_{o}}{2}\right) d z$
Eq. 6.7.12

When $\mathrm{dz}=1$, then,
$p_{i} r_{i}+p_{o} r_{o}=\sigma_{r}\left(r_{i}+r_{o}\right)$
Eq. 6.7.13
and $\sigma_{r}$ can be written as,

$$
\begin{equation*}
\sigma_{r}=\frac{p_{i} r_{i}+p_{o} r_{o}}{r_{i}+r_{o}} \tag{Eq. 6.7.14}
\end{equation*}
$$

From Logan (1981), the equation for the moment exerted on a hollow circular shaft can be written as,

$$
\begin{equation*}
M=\int \tau_{z \theta} \rho d A \tag{Eq. 6.7.15}
\end{equation*}
$$

where $A=\mathrm{pr}^{2}, d A=2$ pdr and $r$ is the thickness of the specimen $\left(r_{0}-r_{i}\right)$. Integrating over the thickness of the specimen yields

$$
M=2 \pi \int \tau \rho^{2} d \rho=\frac{2 \pi \tau_{z \theta}}{3}\left(r_{o}^{3}-r_{i}^{3}\right)
$$

Therefore,
$\tau_{z \theta}=\frac{3 M}{2 \pi\left(r_{o}^{3}-r_{i}^{3}\right)}$

By substituting Equations 6.7.9, 6.7.11, 6.7.14 and 6.47.16 into Equations 6.7.4 through 6.7.6, terms can be rearranging so that $b, \alpha$ and $\sigma_{m}$ are expressed by only $p_{i}, p_{o}, r_{i}, r_{o}, F_{v}$ and M. Since $r_{i}$ and $r_{o}$ are measured and known, three equations and unknowns are left. As constants in the tests, $\mathrm{b}, \alpha$ and $\sigma_{\mathrm{m}}$ are also known and therefore, $\mathrm{p}_{\mathrm{i}}, \mathrm{p}_{\mathrm{o}}$, and $\mathrm{F}_{\mathrm{v}}$ can be calculated. Detailed substitutions of these equations are shown in Appendix H. The equations for $\mathrm{p}_{\mathrm{i}}, \mathrm{p}_{\mathrm{o}}$, and $\mathrm{F}_{\mathrm{v}}$ are summarized below:

$$
p_{i}=\frac{3\left(r_{o}^{2}-r_{i}^{2}\right) \sigma_{m}-\left[\frac{3 r_{o}}{3 r_{o}+r_{i}}+\frac{3 \cos 2 \alpha-2 b+1}{1-2 b-\cos 2 \alpha} * \frac{r_{i}}{3 r_{o}+r_{i}}\right]\left(3\left(r_{o}^{2}-r_{i}^{2}\right) \sigma_{m}-\left(\frac{3\left(r_{o}^{2}-r_{i}^{2}\right)}{\pi\left(r_{o}^{3}-r_{i}^{3}\right)}\right)\left(\frac{M}{\tan 2 \alpha}\right)\right)}{\frac{3 r_{i}\left(r_{o}^{2}-r_{i}^{2}\right)}{3 r_{o}+r_{i}} * \frac{4 \cos 2 \alpha}{\cos 2 \alpha+2 b-1}}
$$

Eq. 6.7.17

$$
\begin{align*}
& p_{o}=\frac{3\left(r_{o}^{2}-r_{i}^{2}\right)}{r_{o}\left(3 r_{o}+r_{i}\right)} \sigma_{m}-\left(\frac{3\left(r_{o}^{2}-r_{i}^{2}\right)}{\pi\left(r_{o}^{3}-r_{i}^{3}\right)}\right)\left(\frac{1}{r_{o}\left(3 r_{o}+r_{i}\right)}\right)\left(\frac{M}{\tan 2 \alpha}\right)+\left(\frac{r_{i}\left(3 r_{i}+r_{o}\right)}{r_{o}\left(3 r_{o}+r_{i}\right)}\right) p_{i}  \tag{Eq. 6.7.18}\\
& F_{v}=\frac{3\left(r_{o}^{2}-r_{i}^{2}\right)}{r_{o}\left(3 r_{o}+r_{i}\right)} \pi \sigma_{m}-3 \pi r_{o}^{2} p_{o}+3 \pi r_{i}^{2} p_{i} \tag{Eq. 6.7.19}
\end{align*}
$$

The strains in torsion shear were calculated by measuring the horizontal LVDT, vertical LVDT, the volume change of the specimen and the volume change of the inner cell. Average strains were obtained from the assumption that the specimen remained a right cylinder
throughout loading. Visual inspection throughout the testing proved this to be true throughout shearing. The following strain equations were used to measure shear strains in torsion shear tests:
$\varepsilon_{z}=\frac{\Delta h}{h_{o}}$
Eq. 6.7.20

For a given radius, $r$, the radial displacement increment, $\Delta \mathrm{u}$, is assumed to be proportional to the radius through the following relationship (Pradhan et al.,1988) :

$$
\begin{equation*}
\Delta u=\frac{\Delta r_{o}-\Delta r_{i}}{r_{o}-r_{i}}\left(r-r_{i}\right)+\Delta r_{i} \tag{Eq. 6.7.21}
\end{equation*}
$$

With this relationship, the radial and circumferential strains can be calculated through
$\varepsilon_{r}=-\left[\int_{r_{i}}^{r_{o}} \frac{\partial \Delta u}{\partial r} r d r / \int_{r_{i}}^{r_{o}} r d r\right]=-\frac{\Delta r_{o}-\Delta r_{i}}{r_{o}-r_{i}}$
Eq. 6.7.22
$\varepsilon_{\theta}=-\left[\int_{r_{i}}^{r_{o}} \frac{\Delta u}{r} r d r / \int_{r_{i}}^{r_{o}} r d r\right]=-\frac{\Delta r_{o}+\Delta r_{i}}{r_{o}+r_{i}}$
Eq. 6.7.23
$\Delta r_{i}=\sqrt{\left(\frac{\pi\left(r_{i}^{2} h_{o}\right)+\Delta I_{\text {vol }}}{\pi h}\right)}-r_{i}$
$\Delta r_{o}=\sqrt{\left(\frac{\pi\left(r_{o}^{2} h_{o}\right)+\Delta I_{v o l}+\Delta B P_{v o l}}{\pi h}\right)}-r_{o}$
Eq. 6.7.25

$$
\left.\begin{array}{l}
\varepsilon_{z \theta}=\frac{1}{\pi\left(r_{o}^{2}-r_{i}^{2}\right)} \int_{r_{i}}^{r_{o}} \int_{0}^{2 \pi} \varepsilon_{z \theta} \cdot(r) \cdot r \cdot d \theta d r \\
\varepsilon_{z \theta}=\frac{1}{\pi\left(r_{o}^{2}-r_{i}^{2}\right)} \int_{r_{i}}^{r_{0}} \frac{r \cdot \Delta \theta}{2 h} 2 \pi r \cdot d r  \tag{Eq. 6.7.26}\\
\varepsilon_{z \theta}=\frac{\Delta \theta\left(r_{o}^{3}-r_{i}^{3}\right)}{3 h\left(r_{o}^{2}-r_{i}^{2}\right)}
\end{array}\right\}
$$

where $\Delta \mathrm{h}$ is the change in specimen height measured by the vertical LVDT, $\mathrm{h}_{\mathrm{o}}$ is the original measured height of the specimen measured prior to testing, h is the current height of the specimen, $r_{o}$ is the outer radius, $\Delta r_{o}$ is the change in outer radius, $r_{i}$ is the inner radius, $\Delta r_{i}$ is the change in inner radius, $\Delta \mathrm{I}_{\mathrm{vol}}$ is the change in inner volume, and $\Delta \mathrm{BP}_{\mathrm{vol}}$ is the change in specimen volume recorded.
$\Delta \theta$ is the angular displacement of the specimen in radians. The angular displacement is calculated by

$$
\begin{equation*}
\Delta \theta=\frac{\Delta H L V D T}{r_{\text {meausrementplate }}} \tag{Eq. 6.7.26}
\end{equation*}
$$

where $\triangle$ HLVDT is the recorded change in HLVDT reading and $r_{\text {measurement plate }}$ is the radius of the pie shaped measurement plate to the radio wire cord.

Using Mohr's circle, these strains are converted to major, intermediate and minor principal strains. The total volumetric strain is the summation of these three strains.

### 6.8 LabView Program

Only three components of the torsion shear test were not automated: the turntable, the B-value saturation test measurement sensor and the back pressure applied to the specimen. As described previously, back pressure was manually applied to the specimen using the manual regulator and pressure gage. A pressure transducer with a display unit was used in measuring the saturation of the specimen. The turntable was rotated with a Maxon gear motor. The voltage to spin the motor was supplied by an external Hewlett Packard triple outlet DC power supply.

However, the internal and external pressure sensors, internal and specimen volume change sensors, load cell, horizontal LVDT, vertical LVDT and the two torque load cells were all connected to a data acquisition unit and displayed in LabView. LabView is a computer program that uses a graphical programming environment in order to develop measurement, test and control systems. It uses graphical icons and wires in the form of a flowchart to allow for easy programming.

The data acquisition system read voltage signals from the measurement devices described in the paragraph above. With this, the program read these signals and through calibrations that
were input into the program, the signals were converted to pressures, loads and distances. A graphical front panel allowed the user to input the conditions of the test desired. In this experimental program, b-values, alpha values and a constant mean normal stress were input. The front panel was also used for the initial application of pressures to the inner and outer cells of the specimen as well as for isotropic consolidation.

For tests with rotation, the torque applied from the torque right and left load cells was measured once the table started to rotate. Once a measurement was acquired by the data acquisition system and LabView program, the internal and external pressures and vertical load were changed to ensure the designated $b$-value and alpha value are applied to the specimen, all while keeping the mean normal stress constant. The applied pressures and load were recorded, as well as the inner volume, specimen volume, horizontal LVDT and vertical LVDT readings. These readings provided the necessary data to study the behavior of the sand under different stress paths.

Tests without rotation, where $\alpha=0^{\circ}$ and $\alpha=90^{\circ}$, do not require the torque load cells and the horizontal LVDT, and they are stress controlled tests. For these tests, the vertical load was applied at desired load increments and pressures are changed accordingly to create different $b$-value conditions with a constant mean normal stress. The desired $\alpha$ and $b$-values are still set on the front panel so that the correct equations throughout the test are used.

After the specimen reached failure, the rotating table was stopped (for tests with rotation). The backpressure line and the internal and external volume change lines were closed in order to prevent more volume change from occurring. The specimen was then exposed to and held under vacuum. The vertical load, back pressure and internal and external pressures were brought back down to zero and the torsion shear assembly was disassembled. With the specimen held under vacuum, any shear bands were recorded and the sheared specimen was photographed. A typical example of a specimen held under vacuum once it had been sheared is shown in Figure 6.8.1.


Figure 6.8.1. Typical Picture of Torsion Shear Specimen after failure held up by a vacuum of 48 kPa in order to measure and photograph any developed shear bands.

A series of 22 tests were performed in this experimental program in order to study the strength behavior and failure surface of Fine Nevada sand while keeping alpha, $b$ and $\sigma_{m}$ constant. As described in details in Sections 6.2 the sand was deposited using the airpluviation method and saturated using the $\mathrm{CO}_{2}$ method. Skemptom's B -value for all specimens was found and all specimens showed B-values above 0.94 . This saturation number is acceptable for highly dense fine sand. Once ready to begin testing, the specimens underwent isotropic consolidation in 6.9 kPa increments starting at 48.3 kPa to 101 kPa effective confining pressure.

Shearing began for all specimens at initial inner and outer effective confining pressures of 101 kPa . Depending on the stress path indicated for the particular test, the inner and outer confining pressures either increased or decreased. The vertical force also was either in compression or extension. Table 6.9 .1 shows the behavior of inner and outer pressures, as well as the vertical force, according to different $b$-value and alpha configurations. In the $\mathrm{p}_{\mathrm{o}}$ and $p_{i}$ columns, a (-) sign shows a decrease and a ( + ) sign shows an increase in effective confining pressure throughout shearing. The bolded signs indicate the greater pressure between the two for the condition shown. $\mathrm{A}(+)$ sign in the $\mathrm{F}_{\mathrm{v}}$ column shows that the specimen was under compression (positive downward force) and a (-) sign shows the specimen was under extension (negative upward force). An ( $=$ ) sign (seen for $\mathrm{b}=0.5$ and
$\alpha=90^{\circ}$ ) shows constant values (i.e. no changes in original effective confining pressure or vertical load).

Table 6.9.1. Varying Pressures and Vertical Loads for different Torsion Shear Test Configurations.

| $\alpha$ (degrees) | b-value | $\mathbf{p}_{\text {i }}$ | $\mathbf{p}_{0}$ | $\mathbf{F}_{\mathbf{v}}$ |
| :---: | :---: | :---: | :---: | :---: |
| $0^{\circ}$ | 0 | - | - | + |
| $0^{\circ}$ | 0.25 | - | - | + |
| $0^{\circ}$ | 0.5 | $+$ | - | + |
| $0^{\circ}$ | 0.75 | $+$ | + | + |
| $0^{\circ}$ | 1 | $+$ | + | + |
| $22.5{ }^{\circ}$ | 0 | - | - | + |
| $22.5{ }^{\circ}$ | 0.25 | - | - | + |
| $22.5{ }^{\circ}$ | 0.5 | $+$ | - | + |
| $22.5{ }^{\circ}$ | 0.75 | $+$ | + | + |
| $22.5{ }^{\circ}$ | 1 | $+$ | + | + |
| $45^{\circ}$ | 0 | - | - | + |
| $45^{\circ}$ | 0.25 | - | - | + |
| $45^{\circ}$ | 0.5 | $=$ | $=$ | $=$ |
| $45^{\circ}$ | 0.75 | $+$ | + | - |
| $45^{\circ}$ | 1 | $+$ | + | - |
| $67.5^{\circ}$ | 0 | - | - | - |
| $67.5^{\circ}$ | 0.25 | - | - | - |
| $67.5^{\circ}$ | 0.5 | - | - | - |
| $67.5^{\circ}$ | 0.75 | $+$ | + | - |
| $67.5^{\circ}$ | 1 | $+$ | + | - |
| $90^{\circ}$ | 0 | - | - | - |
| $90^{\circ}$ | 0.25 | - | - | - |
| $90^{\circ}$ | 0.5 | $+$ | + | - |
| $90^{\circ}$ | 0.75 | $+$ | $+$ | - |
| $90^{\circ}$ | 1 | $+$ | + | - |

All efforts were made to stay as close to the targeted alpha and $b$-values as possible during testing. However, for certain tests due to vertical uplift corrections applied to the data after the test was run, the $b$-values and alpha values were not exactly on the targeted value. The
total stress paths were different for these tests. However, as stress path does not affect failure, the failure points can be presented for all tests. A figure showing the boundary between compression and extension tests while having constant mean stress, b-value and alpha is shown in Figure 6.9.1. Figure 6.9 .2 shows the test area where the inner pressure is greater than the outer pressure and vice versa. The arrows show the direction of increasing difference in internal and external pressures for varying b-value and alpha values.


Figure 6.9.1. Boundary area between extension and compression tests for Torsion Shear Tests held at constant alpha, b-value, and mean normal stress.


Figure 6.9.2. Boundary area for inner and outer pressure conditions for Torsion Shear Tests held at constant alpha, b-value, and mean normal stress.

## 7. Torsion Shear Test Results

### 7.1 Friction Angle Summary and Failure Surface Plots

To provide ease to the reader when presenting the data, the final friction angles will be analyzed both in the $\mathrm{b}=$ constant direction and alpha=constant direction. Stress strain and volume change curves for the tests performed as part of this experimental program as well as previous tests in Van Dyck (2012) will be presented at $\alpha=$ constant intervals.

Table 7.1.1 summarizes the tests performed as part of this experimental program and provides the alpha value, b -value, void ratio, and corrected friction angle for each test. The friction angles have been corrected for any changes in void ratio, as well as have been all translated to the same octahedral plane, where $I_{1}=300 \mathrm{kPa}$. As mentioned in Chapter 6, the data from Tests 1-22 has been included to give an overall and complete picture of the failure surface but these tests were not part of the author's experimental study. Tests designated with a * had the uplift correction applied after shearing and a different stress path was followed than the one originally intended. However, because stress path does not affect failure, they are included in the set of data presented in the following pages.

Tests $7^{*}, 13^{*}$, and 16* failed in the radial-vertical plane, a different plane (verticalcircumferential) from the other tests and therefore have been separated out from the test failure results presented in this chapter.

Figures 7.1.1 through 7.1.5 show the failure points plotted versus b-value and Figures 7.1.6 through 7.1.10 show the failure points plotted versus alpha values so that the results can be analyzed through two different variables. Figure 7.1 .11 shows the 3D surface plotted with only points chosen at the corresponding intersections at increments of $b=0.25$ and $\alpha=22.5^{\circ}$. As described in Section 6.1.3, the corresponding measurement error bars have been drawn on the failure surface points for Figures 7.6.1 through 7.6.10. Figures 7.1.1 and 7.1.5 also have the true triaxial data results plotted as a reference and comparison.

Table 7.1.1. Summary of Test Results sorted by alpha=constant and b-values.

| Test <br> No. | $\alpha$ | bvalue | e | $\phi\left({ }^{\circ}\right)$ | Test <br> No. | $\alpha$ | b-value | e | $\phi\left({ }^{\circ}\right)$ |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 23 | 0.00 | 0.00 | 0.531 | 41.21 | 14 | 67.50 | 0.00 | 0.538 | 35.06 |
| 1 * | 0.00 | 0.00 | 0.510 | 36.78 | 35 * | 69.89 | 0.16 | 0.531 | 37.64 |
| $24 *$ | 0.00 | 0.27 | 0.530 | 45.84 | 36 | 67.80 | 0.25 | 0.533 | 37.64 |
| $25 *$ | 0.00 | 0.55 | 0.530 | 53.05 | 15 | 67.40 | 0.50 | 0.525 | 39.46 |
| 2 | 0.00 | 0.75 | 0.529 | 56.91 | 37 * | 69.91 | 0.55 | 0.528 | 38.06 |
| 26 | 0.00 | 1.00 | 0.532 | 53.28 | 38 | 67.42 | 0.75 | 0.528 | 31.78 |
|  |  |  |  |  | 39 * | 70.56 | 0.79 | 0.531 | 34.83 |
| 3 | 22.41 | 0.00 | 0.523 | 39.33 | 40 * | 73.08 | 0.80 | 0.541 | 34.90 |
| 27 * | 24.04 | 0.02 | 0.510 | 36.31 | 17 | 68.21 | 1.00 | 0.532 | 38.33 |
| 28 | 23.69 | 0.23 | 0.531 | 43.4 |  |  |  |  |  |
|  | 23.73 | 0.27 | 0.548 | 47.25 | 41 | 90.00 | 0.00 | 0.523 | 33.24 |
| $5 *$ | 23.53 | 0.27 | 0.524 | 41.44 | 18 | 90.00 | 0.04 | 0.530 | 33.88 |
| 6 | 22.48 | 0.50 | 0.526 | 43.40 | 19 * | 90.00 | 0.07 | 0.538 | 38.16 |
| 29 | 22.21 | 0.75 | 0.531 | 46.64 | 42 | 90.00 | 0.32 | 0.530 | 45.04 |
| 30 * | 22.92 | 0.85 | 0.529 | 46.15 | 20 * | 90.00 | 0.54 | 0.530 | 45.24 |
| 8 | 22.47 | 0.99 | 0.541 | 42.92 | 43 | 90.00 | 0.78 | 0.520 | 39.16 |
|  |  |  |  |  | 21 * | 90.00 | 0.78 | 0.520 | 40.35 |
| 31 | 44.71 | 0.02 | 0.535 | 36.20 | $22 *$ | 90.00 | 0.99 | 0.520 | 36.69 |
| $32 *$ | 31.76 | 0.18 | 0.560 | 40.24 | 44 | 90.00 | 0.99 | 0.510 | 36.68 |
| 9 | 44.98 | 0.25 | 0.530 | 38.88 |  |  |  |  |  |
| 33 | 44.99 | 0.50 | 0.528 | 45.02 |  |  |  |  |  |
| 10 * | 42.60 | 0.54 | 0.555 | 39.89 |  |  |  |  |  |
| 11 | 44.98 | 0.75 | 0.540 | 40.40 |  |  |  |  |  |
| 12 * | 42.26 | 0.80 | 0.559 | 40.26 |  |  |  |  |  |
| 34 | 44.95 | 1.00 | 0.541 | 35.64 |  |  |  |  |  |



Figure 7.1.1. b-value vs. Friction Angle for $\alpha=0^{\circ}$ degrees for both Torsion Shear and True Triaxial Tests.

Figure 7.1.1 shows the test results for all $\alpha=0^{\circ}$ tests. The torsion shear tests have been separated and marked by open and closed circles. The open circles are torsion shear tests that were previously denoted with a (*). These tests did not have the uplift pressure accounted for prior to testing and therefore, followed a different initial stress path than originally desired. Tests with the closed circle were performed after and were performed with all corrections applied prior to and during testing. The true triaxial tests have also been added to the figure to provide a reference and comparison.

As can be seen, the true triaxial and torsion shear tests confirm each other. This shows the validity of the torsion shear apparatus and the test results. Low b-values for torsion shear tests show slightly less friction angles than that of the true triaxial tests but they were well within the scatter. As the $b$-value increases from zero up to 0.75 , there is an increase in friction angle of about 15 degrees. From $b=0.75$ to $b=1.0$, the friction angle decreases about 3.5 degrees. This is similar to results from Abelev and Lade (2004). For tests with rotation, other than at 90 degrees, there is no true triaxial test data to compare the torsion shear tests with. Therefore, Figures 7.1.2 through 7.1.4 do not have any additional test data besides that of the torsion shear tests performed.

Figure 7.1.2 shows torsion shear test data for $\alpha=22.5^{\circ}$ tests. The results follow the same behavior as seen in $\alpha=0^{\circ}$, with increasing friction angle as $b$-value increases. There is also a drop of about 4 degrees from $b=0.75$ to $b=1.0$. Overall, the friction angles are lower than $\alpha=0^{\circ}$ at the same $b$-values. The increase in friction angle is about 7 degrees from $b=0$ to $b=0.75$. This is only half of the increase seen in the $\alpha=0^{\circ}$ tests.


Figure 7.1.2. b-value vs Friction Angle for $\alpha=22.5^{\circ}$ degrees for Torsion Shear Tests.


Figure 7.1.3. $b$-value vs Friction Angle for $\alpha=45^{\circ}$ degrees for Torsion Shear Tests.

Test data for $\alpha=67.5^{\circ}$ are presented in Figure 7.1.3. The behavior of the soil under these conditions shows an increase in friction angle from $b=0$ to $b=0.5$, unlike $\alpha=0^{\circ}$ and $\alpha=22.5^{\circ}$ where there was an increase from $b=0$ to $b=0.75$. There is an increase of an average of 8 degrees. At $b=0.5$ to $b=1.0$, the strength decreases and the final strength at $b=1$ is 0.5 lower than that of $b=0$. Once again, the strength of the soil at $\alpha=45^{\circ}$ is less than that of corresponding b-values at $\alpha=22.5^{\circ}$.

Data for $\alpha=67.5^{\circ}$ tests (see Figure 7.1.4) showed similar behavior to $\alpha=45^{\circ}$. However, a sharp decrease in friction angle is seen $a t /$ near $b=0.75$. It is believed that at this principal stress direction and $b$-value, the bedding plane direction is close to/equal the principal stress direction. Due to this, strain localization may occur sooner and cause the lower friction angle. It is important to note that three tests conducted at this condition show this same behavior and very similar development of shear bands (as will be discussed further in Chapter 9).

Figure 7.1.5 shows $\alpha=90^{\circ}$ tests and also shows the true triaxial results for comparison. At low $b$-values, the tests seem to behave similarly. From $b=0.32$ to $b=1.0$, the torsion shear tests begin to lose strength and deviate from the true triaxial test results. The torsion shear tests show significantly less (from 12 to 15 degrees) strength than the true triaxial tests. Lade and Wang (2012) compared triaxial extension tests with flexible versus rigid boundaries. They
found that at $\alpha=90^{\circ}, \mathrm{b}=1.0$, there was an 8.2 degree difference in friction angle between tests performed with rigid long plates to stiffen the soft membrane than in conventional extension tests with flexible membranes. Showing higher strengths, the extension tests with rigid boundaries also showed higher rates of dilations and were more succesful in maintaining uniform strains. Therefore, the strengths obtained from different designs of true triaxial equipment may be affected by the stiffness of the loading plates. It is important to recall that in order to perform $\alpha=90^{\circ}$ tests in the true triaxial apparatus, the sand was poured and frozen so that the bedding planes could be rotated to create an $\alpha=90^{\circ}$ condition. However, the major principal stress was still applied in the vertical direction through two rigid boundaries. For torsion shear tests, the bedding planes are always horizontal. Therefore, the major principal stress is rotated and applied in the horizontal direction parallel to the bedding planes to the specimen. This stress acts through the flexible membranes. The difference in strengths seen between the true triaxial and torsion shear test results may be due to this flexible boundary effect in the torsion shear tests.


Figure 7.1.4. b-value vs. Friction Angle for $\alpha=67.5^{\circ}$ degrees for Torsion Shear Tests.


Figure 7.1.5. b-value vs. Friction Angle for $\alpha=90^{\circ}$ degrees for both Torsion Shear and True Triaxial tests.

To get an understanding of the strength of the soil while the b-value was kept constant, Figures 7.1.6 through 7.1.10 present the results with varying alpha values. Although not all b-values were kept exactly at the corresponding b (especially the tests designated with $\mathrm{a} *$ ), the test results were grouped and plotted at the nearest b-value.

When looking at the test data with b-value as constant, the tests all show very similar behavior. The strength decreases from $\alpha=0^{\circ}$ to $\alpha=67.5^{\circ}$ (where it is the lowest) and then increases to $\alpha=90^{\circ}$, ending always at a strength less than that of $\alpha=0^{\circ}$. Table 7.1.2 summarizes the tests results and shows the difference in friction angle compared to $\alpha=0^{\circ}$ for the corresponding b -value.

Table 7.1.2 shows an increase in difference of friction angle for $b$-values as alpha goes from 0 to 90 degrees. Overall, the drop from $\alpha=0^{\circ}$ to $\alpha=67.5^{\circ}$ gets more pronounced as the $b$-value increases. The greatest drop from $\alpha=0^{\circ}$ to $\alpha=67.5^{\circ}$ is seen at $b=0.75$, where there is an average decrease of 23.7 degrees. An increase in strength seen from $\alpha=67.5^{\circ}$ to $\alpha=90^{\circ}$ is the greatest at $b=0.25$ and $b=0.75$, increasing an average of 7.5 degrees.

Lade and Wasif (1988) performed tests on specimens of Cambria sand with height/diameter ratios of 1 and 2.5. They prepared the specimens by placing a mold in a bucket of deaired
water and then tilting the mold sitting in a cradle to the desired alpha value. The specimen was then placed upright and frozen. When thawed and tested, the major principal stress acted at the angle of the bedding planes. This is a similar procedure as explained previously when preparing true triaxial specimens at $\alpha=90^{\circ}$, except with the additional use of a cradle to tilt the mold while pouring the sand into the mold. For tests done at $b=0$, the tests showed that the friction angle consistently dropped by 5.5 degrees with increasing inclination of the bedding planes. Similarly, as can be seen in Table 7.1.2 and Figure 7.1.6, a drop of 7.6 degrees is seen where $\alpha=90^{\circ}$ and $b=0$.

Table 7.1.2. Summary of Torsion Shear test results with the b-value constant.

| Test No. | $\alpha$ | $\begin{gathered} \mathbf{b}- \\ \text { value } \end{gathered}$ | $\phi$ | $\Delta \phi$ | Test No. | $\alpha$ | bvalue | $\phi$ | $\Delta \phi$ |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 23 | 0.00 | 0.00 | 41.57 | 0 | 2 | 0.00 | 0.75 | 57.02 | 0 |
| 1 * | 0.00 | 0.01 | 37.36 | -4.21 | 29 | 22.77 | 0.75 | 46.75 | -10.27 |
| 3 | 22.38 | 0.00 | 39.4 | -2.17 | 30* | 22.92 | 0.85 | 46.18 | -10.84 |
| 27 * | 24.04 | 0.02 | 36.34 | -5.23 | 11 | 44.98 | 0.75 | 40.36 | -16.66 |
| 31 | 44.91 | 0.00 | 36.25 | -5.32 | 12 * | 41.54 | 0.81 | 41 | -16.02 |
| 14 | 67.50 | 0.00 | 35.01 | -6.56 | 38 | 67.54 | 0.75 | 32.98 | -24.04 |
| 41 | 90.00 | 0.00 | 34.27 | -7.3 | 39 * | 70.62 | 0.79 | 34.68 | -22.34 |
| 18 | 90.00 | 0.06 | 36.32 | -5.25 | 40* | 73.06 | 0.80 | 34.97 | -22.05 |
| 19* | 90.00 | 0.08 | 40.37 | -1.2 | 43 | 90.00 | 0.78 | 40.41 | -16.61 |
|  |  |  |  |  | 21 * | 90.00 | 0.78 | 41.27 | -15.75 |
| 24 * | 0.00 | 0.27 | 45.89 | 0 |  |  |  |  |  |
| 28 | 22.97 | 0.26 | 44.48 | -1.41 | 26 | 0.00 | 1.00 | 53.27 | 0 |
| 4* | 23.73 | 0.27 | 47.03 | 1.14 | 8 | 22.87 | 0.97 | 41.87 | -11.4 |
| 5 * | 24.18 | 0.27 | 41.39 | -4.5 | 34 | 44.95 | 1.00 | 35.61 | -17.66 |
| 32 * | 31.76 | 0.18 | 40.58 | -5.31 | 17 | 68.26 | 1.00 | 37.11 | -16.16 |
| 9 | 44.99 | 0.24 | 38.85 | -7.04 | 22 | 90.00 | 0.99 | 39.36 | -13.91 |
| 35 * | 69.85 | 0.16 | 37.74 | -8.15 | 44 | 90.00 | 0.99 | 35.72 | -17.55 |
| 36 | 67.68 | 0.25 | 37.57 | -8.32 |  |  |  |  |  |
| 42* | 90.00 | 0.32 | 45.1 | -0.79 |  |  |  |  |  |
|  |  |  |  |  |  |  |  |  |  |
| 25 * | 0.00 | 0.55 | 53.51 | 0 |  |  |  |  |  |
| 6 | 22.01 | 0.51 | 46.31 | -7.2 |  |  |  |  |  |
| 33 | 44.99 | 0.50 | 45.02 | -8.49 |  |  |  |  |  |
| 10 * | 41.96 | 0.55 | 40.55 | -13 |  |  |  |  |  |
| 15 | 67.40 | 0.50 | 39.52 | -14 |  |  |  |  |  |
| 37* | 70.03 | 0.56 | 38.19 | -15.3 |  |  |  |  |  |
| 20* | 90.00 | 0.52 | 41.81 | -11.7 |  |  |  |  |  |
|  |  |  |  |  |  |  |  |  |  |



Figure 7.1.6. Alpha vs Friction Angle for $b=0$ for Torsion Shear Tests.


Figure 7.1.7. Alpha vs Friction Angle for $\mathrm{b}=0.25$ for Torsion Shear Tests.


Figure 7.1.8. Alpha vs Friction Angle for $b=0.50$ for Torsion Shear Tests.


Figure 7.1.9. Alpha vs Friction Angle for $\mathrm{b}=0.75$ for Torsion Shear Tests.


Figure 7.1.10. Alpha vs Friction Angle for $\mathrm{b}=1.0$ for Torsion Shear Tests.


Figure 7.1.11.3D failure surface plot at increments of 0.25 for $b$-values and 22.5 degrees for alpha values.

Figure 7.1.11 shows the 3D representation of the test results. Representative points at the intersections of $b$-value and alpha values have been presented. As was seen in the individual plots (Figures 7.1.1 through 7.1.10), the soil shows the highest strength at $\alpha=0^{\circ}$ and higher bvalues. With increasing alpha values, there is a loss of strength, which at $\alpha=90^{\circ}$ increases a small amount. The loss in strength at $\alpha=67.5^{\circ}$ and $b=0.75$ is seen as a dip in Figure 7.1.11. From the highest to lowest point on the graph, there is a difference of $23.4^{\circ}$. For comparison in 3D, the true triaxial tests have been graphed with the torsion shear tests at $\alpha=0^{\circ}$ and $\alpha=90^{\circ}$ in Figure 7.1.12.


Figure 7.1.12. Comparison of 3D failure surface plot for True Triaxial and Torsion Shear tests.

### 7.2 Correction to Stress-Strain Curves for Piston Friction

When analyzing the stress strain curves for the test data, certain tests experienced an increase in major principal strain without much, if any increase in stress ratio. This only occurred in tests that had rotation. Tests where $\alpha=0^{\circ}$ or $\alpha=90^{\circ}$ did not experience this added strain. This was mainly seen in most of the preliminary set of tests (tests labeled with a *). If the specimen was not perfectly aligned and level with the top lid, a horizontal frictional force was applied to the piston. At the beginning of the test, where the stress level was still very low, this frictional force was picked up by the horizontal LVDT reading. Once the frictional
force was overcome, the piston would be free to move and the LVDT readings would be continuous and smooth. In order to verify that this was indeed what was occurring, other dial gages were set up and recorded. The readings of the horizontal LVDT and the dial gages were compared and proven to be the same.

Although it is difficult to quantify the amount of friction in each test, especially with the varying conditions unique to each test, it is believed that the friction did not affect the final strength reached for each specimen. This is confirmed by the true triaxial test results (shown in Chapter 5) for specimens at $\alpha=0^{\circ}$ and $\alpha=90^{\circ}$. Failure points for the torsion shear tests performed without rotation fit in within the range of scatter with the true triaxial results. This comparison can be seen in Figure 7.1.1 and 7.1.5.

With this in mind, the set-up of the specimen for the second set of testing (tests labeled without a *) was changed slightly. The procedure deviated from previous tests by ensuring that the bushing was removed before setting the top lid on the acrylic cylinder prior to saturation of the specimen. Without the bushing in place, it was possible to physically see if the specimen, the acrylic cylinder, and the piston were completely centered and aligned with the load cell. Once this was centered, as there was a little bit of tolerance in moving the acrylic cylinder on its O-ring, the top lid was removed to allow the saturation of the inner cell after which the top lid was placed atop the acrylic cylinder again. The bushing was connected and the rest of procedure was completed as described in Section 6.2.

It is believed that this alignment process helped to minimize the effect of piston friction when rotating the specimen. Although every effort was made to create a specimen that was as plum as possible, in certain cases, the horizontal deviation still occurred. Most of the new set of tests did not have this occur. However, four tests (Tests 14, 17, 36, and 38) needed to be corrected. The previous set of tests, as well as these four tests were corrected and an example of the correction is shown in Figures 7.2.1 through 7.2.4. Figures 7.2.1 and 7.2.2 show an example of a correction done to a previous test where there is a pronounced increase in strain. Figures 7.2.3 ad 7.2.4 show an example of a test where a small correction was applied. The stress-strain and volume change plots presented in the sections that follow have all been corrected in this manner if needed.


Figure 7.2.1. Stress Strain Curve for Tests 32* showing the correction applied to account for piston friction.


Figure 7.2.2. Volume Change Curve for Tests $32 *$ showing the correction applied to account for piston friction.


Figure 7.2.3. Stress Strain Curve for Tests 3 showing the correction applied to account for piston friction.


Figure 7.2.4. Volume Change Curve for Tests 3 showing the correction applied to account for piston friction.
7.3 Tests with $\alpha=0^{\circ}$

Tests without rotation were performed under stress controlled conditions. In order to change the b-value, internal and external pressures were changed, as well as the vertical load applied. Vertical load was added to the specimen, creating a vertical and principal stress over the area of the specimen. The stress was added until the specimen could no longer hold that stress and shearing occurred. In most instances, shear bands occurred at or near failure. Therefore, a well-pronounced softening regime in the stress-strain curves is not seen.

Isotropic compression was performed on each specimen from 48 kPa to 101 kPa prior to shearing. The isotropic compression charts for $\alpha=0^{\circ}$ tests can be seen in Figure 7.2.1. All specimens have very similar behavior during isotropic compression in terms of volume change and no obvious leaks or problems can be detected from looking at Figure 7.3.1.


Figure 7.3.1. Isotropic Compression Chart for $\boldsymbol{\alpha}=0^{\circ}$ Torsion Shear Tests.

The stress-strain plot for these tests is presented in Figure 7.3.2. Since the Tests 25*, 2 and 26 all had very high stress ratios, the entire curve is shown in Figure 7.3.2. The scale on Figure 7.3.3 is cut off at a stress ratio of 7 for consistency and ease when comparing the other tests in Sections 7.4 to 7.7. When looking at Figure 7.3.2 and 7.3.3, a pattern can be seen. As the $b$-value increases, so does the friction angle. At $b=1$ however, the friction angle is much lower. This pattern is shown in Figure 7.1.1. The strain to failure also decreases as the bvalue is increased. This is typical and was also seen in the true triaxial test results presented in Chapter 5. Section 8.2 explains in detail explanations for why the strain to failure is different between the true triaxial tests and torsion shear tests.


Figure 7.3.2. Stress-Strain Curves for $\boldsymbol{\alpha}=0^{\circ}$ Torsion Shear Tests.


Figure 7.3.3. Stress-Strain Curves for $\boldsymbol{\alpha}=0^{\circ}$ Torsion Shear Tests only showing up to $\sigma_{1} / \sigma_{3}=7$.

The volume change curves for this set of tests are presented in Figure 7.3.4. With increasing b-value, it is thought that there would also be increasing volumetric strain. The volume change does not show such a consistent pattern as did Figure 7.3.3. It is seen here that Tests 24* and Test 25 fall out of the pattern described above. Test $24^{*}$ shows less volumetric strain than what would be expected. Conversely, Test 25 shows much more dilation than expected. However, the rest of the tests follow the expected pattern. For tests with $\boldsymbol{\alpha}=0^{\circ}$, there is a slight increase in dilation angles from $b=0.5$ to $b=1.0$ as seen in Figure 7.3.5.


Figure 7.3.4 Volume Curves for $\boldsymbol{\alpha}=0^{\circ}$ Torsion Shear Tests.


Figure 7.3.5. Dilation angle versus b-value for $\boldsymbol{\alpha}=0^{\circ}$ Torsion Shear Tests.

### 7.4 Tests with $\alpha=22.5^{\circ}$

The isotropic compression data for tests with rotation at $\alpha=22.5^{\circ}$ degrees is presented in Figure 7.4.1. All but three tests seem to have a very similar slope and total volumetric strain as the confining pressure was increased. Tests $27^{*}$ and 28 have quite different slopes and show less total volumetric strain than the rest. Test $27^{*}$ had a void ratio of 0.51 , where the other tests had void ratios near 0.53 . The higher relative density of Test 27* can be a reason why there is less volumetric strain. Test 28 has a void ratio of 0.531 , which is the targeted void ratio of the tests. However, accidentally, the specimen did not have a total back pressure of 101 kPa before isotropic compression was performed. With only 48 kPa of back pressure, the change in volumetric strain is seen in Figure 7.4.1. Test 29's data was only recorded half way through the isotropic compression and that is why it begins at 81 kPa .

The stress strain, volume change data and dilation angles are presented in Figures 7.4.2, 7.4.3 and 7.4.4. When looking at the stress-strain curves, once again, there is an increase in stress ratio as the b -value increases. Although there is some scatter in the results from repeat tests at similar b-values, the trend is still seen. Just as in the $\alpha=0^{\circ}$ torsion shear tests, the strength at b-values of 1 dropped significantly. The friction angle can be seen in Figure 7.1.2. There is also decreased strain to failure with increasing b-values.


Figure 7.4.1. Isotropic Compression Chart for $\boldsymbol{\alpha}=22.5^{\circ}$ Torsion Shear Tests.

The volume change curves are presented in Figure 7.4.3. The tests show less total volumetric strain as the b-value increases. Test $5^{*}$ is the only test that is slightly out of order. There is a
little less volumetric strain than tests near $b=0$. With this exception the rest of the volume change curves correspond to the stress strain behavior seen in Figure 7.4.2.


Figure 7.4.2. Stress-Strain Curves for $\boldsymbol{\alpha}=22.5^{\circ}$ Torsion Shear Tests.


Figure 7.4.3. Volume Change Curves for $\boldsymbol{\alpha}=22.5^{\circ}$ Torsion Shear Tests.


Figure 7.4.4. Dilation angle versus $b$-value for $\boldsymbol{\alpha}=22.5^{\circ}$ Torsion Shear Tests.

### 7.5 Tests with $\alpha=45^{\circ}$

Isotropic compression data is shown in Figure 7.5.1. All but Test 12* show similar behavior. Test 12* had a void ratio of 0.56 corresponding to a slightly lower relative density than the rest of the tests. This may be a possible explanation for the increase in volumetric strain with increasing pressure.


Figure 7.5.1. Isotropic Compression Chart for $\boldsymbol{\alpha}=45^{\circ}$ Torsion Shear Tests.

In terms of the vertical load in tests at $\alpha=45^{\circ}$, it is important to recall Figure 6.6.1, which shows the boundary between compression and extension tests. The $\alpha=45^{\circ}$ tests at $b=0$ and $\mathrm{b}=0.25$ are in compression while those at $\mathrm{b}=0.75$ to $\mathrm{b}=1.0$ are in extension. $\alpha=45^{\circ}$ and $\mathrm{b}=0.5$ is the boundary dividing compression and extension tests. Referring to Figure 7.5.3, there is an increase in strength until $b=0.5$. For tests with $b$-values greater than $b=0.5$, there is a decrease in friction angle. As the b-value increases, the strain to failure decreases. When looking at the volume change, most of the curves are very close to each other, with the exception of Test 31 that had the most strain and therefore, most volumetric strain. Most of
the tests have very similar rates of dilation and all show very little total volumetric strain. However, Test 10* and Test 12* show an average of 15 degree increase in angle of dilation above the values for the other tests.


Figure 7.5.2. Dilation Angle versus b-value for $\boldsymbol{\alpha}=45^{\circ}$ Torsion Shear Tests.


Figure 7.5.3. Stress strain plot for $\boldsymbol{\alpha}=45^{\circ}$ Torsion Shear Tests.


Figure 7.5.4. Volume change plot for $\boldsymbol{\alpha}=45^{\circ}$ Torsion Shear Tests.

Presented below are the isotropic compression, stress strain and volume change plots for all tests with $\boldsymbol{\alpha}=67.5^{\circ}$ degrees. All isotropic compression curves have similar slopes and show similar volumetric strain. Prior to isotropic compression, Test 37* did not have the weight of the cross bar and torque arms set up. Therefore, the weight of these pieces was missing in the recorded values of isotropic compression. Without this weight, the specimen was able to show a little bit more volumetric strain. As is seen at the end of isotropic compression for Test 17, a leak was detected after the sample reached 101 kPa confining stress. Although the rate of the leak could be calculated, it is impossible to calibrate it out because the inner and outer pressures for $b=1$ tests vary throughout shearing. Fortunately, since the $b=1$ tests have very short total strain to failure, any additional specimen volume change throughout the test was not detrimental to the test results and final strength of the specimen. Because of this leak, the volume change is not presented for Test 17.


Figure 7.6.1. Isotropic Compression Chart for $\boldsymbol{\alpha}=67.5^{\circ}$ Torsion Shear Tests.

The stress-strain behavior of tests with their major principal stress at an inclination of 67.5 degrees is presented in Figure 7.6.3. Volumetric strain and dilation angles are presented in Figure 7.6.4 and 7.6.2, respectively. The curves show typical behavior when looking at strain to failure with increasing $b$-values. The higher the $b$-value, the less strain to failure. The curves also show increasing strengths from $b=0$ to $b=0.5$. At $b=0.75$, there is a drop in strength and then at $b=1.0$, the strength increases slightly. The decrease in strength at $b=0.75$ and $\boldsymbol{\alpha}=67.5^{\circ}$ was the only location where there is a dip in the strength of the soil. This condition was tested three times and all the test data shows consistency in the results,
confirming that this decrease in strength at the condition stated is real. At this condition, for all three tests, large horizontal deep shear bands developed along the top cap. These horizontal shear bands coincide with the shear band angle, when using the Coulomb angle for shear band inclination (Equation 2.5.3).


Figure 7.6.2. Dilation Angle versus b-value for $\boldsymbol{\alpha}=67.5^{\circ}$ Torsion Shear Tests.


Figure 7.6.3. Stress strain plot for $\boldsymbol{\alpha}=67.5^{\circ}$ Torsion Shear Tests.


Figure 7.6.4. Volume change plot for $\alpha=67.5^{\circ}$ Torsion Shear Tests.

### 7.7 Tests with $\alpha=90^{\circ}$

Isotropic compression data, the stress-strain curves and the volumetric strain curves for tests with $\alpha=90^{\circ}$ are presented in Figures 7.7.1 through 7.7.3, respectively. The pattern for this set of tests is a bit different than what was previously seen for $\boldsymbol{\alpha}=0^{\circ}$ tests which were also without rotation. The strength increases for $b$-values from 0 to 0.25 . At $b=0.5$, there is a drop in strength until $b=1.0$ where the lowest strength is reached. Friction angle failure points can be seen in Figure 7.1.5. As stated when describing Figure 7.1.5, tests done in extension at higher b-values tend to be very unstable. There is quite a deviation in terms of strength when compared to the true triaxial tests done at $\alpha=90^{\circ}$. Once again, Lade and Wang (2012) observed a decrease of 8.2 degrees when performing tests with flexible versus stiff boundaries. This is seen when comparing true triaxial results presented in Chapter 5 (with stiff boundaries in the $\alpha=90^{\circ}$ direction) versus the torsion shear tests (with flexible boundaries in the $\alpha=90^{\circ}$ direction). A difference of about 10 to 13 degrees is found.

Notably, the two tests that seem to be out of this pattern (Tests 20* and 43*) are also the tests that deviate from the isotropic compression chart shown in Figure 7.7.1. Test 20* shows a greater rate of dilation compared to all the other tests (seen in Figure 7.7.3 and 7.7.4). Test 43* although failing at a similar stress ratio as Test 21, which had similar conditions, shows almost twice the amount of strain to failure as test 21 . The stress ratios show a drop in strength after $\mathrm{b}=0.32$.


Figure 7.7.1. Isotropic Compression Chart for $\boldsymbol{\alpha}=90^{\circ}$ Torsion Shear Tests.


Figure 7.7.2. Dilation Angle versus b-value for $\boldsymbol{\alpha}=90^{\circ}$ Torsion Shear Tests.


Figure 7.7.3. Stress strain plot for $\alpha=90^{\circ}$ Torsion Shear Tests.


Figure 7.7.4. Volume change plot for $\alpha=90^{\circ}$ Torsion Shear Tests.

### 7.8 Conclusion

In summary, torsion shear test results have been provided. Friction angle results for tests in Sector I and III have been compared directly to the true triaxial test results presented in Chapter 5. Tests for $\boldsymbol{\alpha}=0^{\circ}$ confirmed each other while there was more scatter in the $\boldsymbol{\alpha}=90^{\circ}$ tests starting at $b=0.5$ to 1 . Torsion shear tests with rotation show that the friction angle decreases as alpha varies from $0^{\circ}$ to $90^{\circ}$, showing a dip in friction angle at $67.5^{\circ}$. From bvalue $=0$ to 0.5 , there is an increase in friction angle, and then a slight decrease until $b=1.0$. This occurs for all alpha values besides $\boldsymbol{\alpha}=0^{\circ}$ and $\boldsymbol{\alpha}=22.5^{\circ}$. In these cases, the friction angle continues to increase with increasing b-value until 0.75 and then slightly decreases as it reaches 1.0. Dilation angles tend to stay constant for constant alpha values and changing bvalues. As is seen in the stress-strain curves presented, as b-value increases, the stress strain curves become much steeper and fail quickly, with very little strain. Shear bands (discussed in Chapter 9) develop at/near failure, not allowing further progress in the stress-strain curve to develop.

## 8. Interpretation of Strains in True Triaxial and Torsion Shear Tests

### 8.1 Introduction

In the previous chapters, stress analysis has been the primary topic when looking at the anisotropic strength and failure surface of Fine Nevada Sand. Although stress analysis is very important, one of the most important aspects when modeling the behavior of frictional materials is the direction of the strain increment vector compared to the direction of principal stress and principal stress increment and the role that anisotropy plays in these directions.

When studying material behavior during stress rotation, it is important to consider the predictions that can be made using both elasticity and plasticity. Stress strain relations are made up of both elastic and plastic strains. Elastic strains are fundamentally very small when compared to plastic strains. However, just because they are small, the elastic strain components cannot be ignored because constitutive models should include elastic strain components, so that the true plastic strain relations can be found.

Fundamentally, incremental elasticity suggests that the increment of elastic strain coincides with the direction of the increment of stress (Figure 8.1.1a). Plasticity theory is based upon the condition that the plastic strain increment coincides with the total stress direction. In 1870, St. Venant (see Figure 8.1.2) suggested the first approach to plastic
stress-strain relations. After conducting experiments, he observed that the axes of strain increments coincided with the axes of stress (Figure 8.1.1b). This relationship is also seen in Melan's (1938) equation that is for plastic behavior of isotropic materials.

$$
d \varepsilon_{i j}^{p}=d \lambda \frac{\partial g}{\partial \sigma_{i j}}
$$

## Eq. 8.1

where $\mathrm{d} \lambda$ is a proportionality constant, and g is the plastic potential function ( $\mathrm{g}=\mathrm{f}$ for associated flow, $\mathrm{g} \nexists \mathrm{f}$ for non-associated flow). $\left\{\partial \mathrm{g} / \partial \sigma_{\mathrm{ij}}\right\}$ is normal to the plastic potential surface.

For frictional materials such as soils, elasticity is dominant far away from failure. This occurs at low stress levels near the isotropic state of stress. Elastic strains can also be seen upon unloading and/or reloading. Near higher stress levels and failure, plasticity theory is more adequately applied. When the strain increment vector is perpendicular to the yield surface, then associated flow is obtained. For isotropic materials such as metals, that is correct. However, associated flow can be questioned for frictional materials. From previous data on tests of frictional materials, experimental evidence shows that when using associated flow rules, too large volumetric expansion is predicted. Non-associated flow should be used in constitutive models for frictional materials in order to capture volume change correctly as well as, observed instabilities such as shear banding and liquefaction (Lade et al. 1987, Hong and Lade 1989, Lade et. al 2009). In order to determine associated or non-associated flow, the directions of strain increment vectors relative to yield surfaces are of importance.

(a) Elastic Behavior
(b) Plastic Behavior

Figure 8.1.1. (a) Elastic, and (b) plastic behavior of soil element during rotation of stress axes (after Lade 2005b).

(a) State of

Stress Near
Failure

Figure 8.1.2. (a) Stresses on a soil element near failure, its (b) plastic behavior (St. Venant's Principle), and its (c) elastic behavior. (after Lade 2005b)
8.2 Strain Increment Directions in True Triaxial Tests and Torsion Shear in $\alpha=0^{\circ}$ and $\alpha=90^{\circ}$ Tests

Lade and Duncan (1973) considered the directions of the strain increments at failure. By definition, the strains at failure are all plastic so there was no need to separate out elastic strains. They found that the normality condition (showing associated flow) was not satisfied. Ko and Scott (1967) also found the same result for tests on Ottowa sand. Although strain increments and the relationship between principal strains in true triaxial tests have been studied before, they will also be analyzed in this thesis.

The intermediate and minor strains plotted versus the major principal strain for each sector of true triaxial tests are presented in Figures 8.2.1 through 8.2.3. For all sectors, the intermediate to major principal strain ratio increases as the $b$-value increases (seen in part (a) of the graphs). The ratio between minor principal strain and major principal strain increases as bvalues increase from 0 to 1 . This is seen in part (b) of the graphs.

For $b=0$ values, there is more strain in the $\varepsilon_{1}$ direction than in the $\varepsilon_{2}$ direction for Sector III than in Sector I. The bedding planes for Sector III are vertical $\left(\alpha=90^{\circ}\right)$ and therefore, are in the same direction as the stress, $\sigma_{1}$. When looking at the graphs, it is important to remember that tests at b-values of near 0.25 are very close to the plain strain condition. This causes the
intermediate principal strain to be close to or near zero. There is a slight increase when moving through Sector I, II and III. For $b=0.25$ tests, failure is right around $\varepsilon_{1}=2 \%$ strain for all three sectors. As the b-value increases, there is an increase in the intermediate principal stain. This increase is also seen when looking from Sector I to II and III. Sector II shows slightly less strain for the same b-value tests than Sector III but they are very close to each other.

It is important to recall that certain stress paths (TT \#5, TT \#6, TT \#11, and TT \#17), all started as $b=1.0$ tests. Since there was not enough pressure in the air supply line to keep this b-value constant up until failure, the horizontal stress was kept constant as the vertical stress increased. These tests failed at around $b=0.7$. The strain paths are very similar to those of $b=1$ tests until the drop in stress occurs. This is why there is a break in the slopes of strain for these tests. However, they all end at about the same $\varepsilon_{2}$ as the $\mathrm{b}=0.75$ tests.

At $\mathrm{b}=1$, in Sector III, the bedding planes are aligned so that the major principal stress and minor principal stress are also in the direction of the bedding planes. This would lead to the expectation that the intermediate and major principal strains would be equal $\left(\varepsilon_{1}=\varepsilon_{2}\right)$. However, this was not what was seen in Test TT \#18. There was almost double the amount of horizontal strain, $\varepsilon_{2}$ than axial strain, $\varepsilon_{1}$.

(b)

Figure 8.2.1. Plots of (a) Intermediate versus Major Principal Strain and (b) Minor versus Major Principal Strain True Triaxial Tests in Sector I.



Figure 8.2.2. Plots of (a) Intermediate versus Major Principal Strain and (b) Minor versus Major Principal Strain True Triaxial Tests in Sector II.

(a)

(b)

Figure 8.2.3. Plots of (a) Intermediate versus Major Principal Strain and (b) Minor versus Major Principal Strain True Triaxial Tests in Sector III

In order to plot the strain increment directions on the octahedral plane, the principal strain increments for each test can be considered as a vector, $\left(\Delta \varepsilon_{1}, \Delta \varepsilon_{2}, \Delta \varepsilon_{3}\right)$. This vector can be superimposed on the principal stress space. When considering the origin as the starting point of the strain increment vector, the coordinates of the vector will be known $\left(\Delta \varepsilon_{1}, \Delta \varepsilon_{2}, \Delta \varepsilon_{3}\right)$. Using a similar procedure as was done to plot stresses on the octahedral plane, the strain increments directions were calculated.

Figure 8.2.4 shows the total stress directions and strain increment directions plotted out on the octahedral plane. The dotted arrows are drawn to go from the origin through the failure point of the different tests. The smaller solid arrows represent the strain increment vectors. In Sector I, the greatest deviation of the strain increment directions from the total stress directions are seen for $b=0.25$ and $b=0.5$. At $b=0.75$ and $b=1$, the strain increment directions become almost the same as the stress directions. In Sector II, the greatest difference is seen also at $b=0.25$. Sector III shows overall the most deviation from the strain increment and stress directions coinciding. As was the case in Sector I and Sector II, as b approaches 1, the difference gets smaller until the strain increment coincides with the stress direction.


Figure 8.2.4. Octahedral Plane showing Strain Increment Directions and Total Stress Directions for True Triaxial Tests.

The intermediate and minor principal strains for torsion shear tests at $\alpha=0^{\circ}$ and $\alpha=90^{\circ}$ were plotted to be compared with the true triaxial results. Figures 8.2 .5 and 8.2.6 present the strains for these tests. As is seen, the strains in torsion shear tests are much smaller than those in the true triaxial tests. Chapter 7 presented the friction angle results from both true triaxial and torsion shear tests. The strength results were very similar for $\alpha=0^{\circ}$ and deviated after $\mathrm{b}=0.5$ for $\alpha=90^{\circ}$. Recalling that the stress paths were different for torsion shear tests and true triaxial tests, the stress-strain curves were not superimposed and compared in the previous chapters. Torsion shear tests all had constant mean stress while true triaxial tests had a
constant minor principal stress. The true triaxial tests increase the major principal stress while staying with a constant minor principal stress. Test TT \#7 was done at a lower confining pressure than TT\#1 due to the limitations of the air supply system that has already been discussed in Chapter 5. Torsion shear tests have a different stress path that approaches failure much more quickly. This deviation in stress path when approaching the failure surface is an important factor when looking at the difference in strain to failure for the two types of tests.

Another more subtle difference between the tests is the use of lubricated ends in the true triaxial tests compared to full frictional ends in torsion shear tests. Lade (1982) shows the comparison of stress-strain relations for tests with lubricated caps and bases compared to those with no lubrication, as seen in Figure 8.2.7. Dense specimens with lubrication had almost double the strain to failure than those without lubrication. A similar case is seen in the true triaxial tests (with double lubrication at both top and bottom plates) and the torsion shear tests (with no lubrication; fully frictional end restraints).

(a)
$\varepsilon_{3}$ vs $\varepsilon_{1}$ for $\alpha=0^{\circ}$ (Sector I) Torsion Shear Tests


| -Test $23(\mathrm{a}=0, \mathrm{~b}=0.0)$ |
| :--- |
| -Test $1^{*}(\mathrm{a}=0, \mathrm{~b}=0.01)$ |
| —Test $24^{*}(\mathrm{a}=0, \mathrm{~b}=0.27)$ |
| —Test $25^{*}(\mathrm{a}=0, \mathrm{~b}=0.55)$ |
| ——Test $2(\mathrm{a}=0, \mathrm{~b}=0.75)$ |
| —Test $26(\mathrm{a}=0, \mathrm{~b}=1.0)$ |

(b)

Figure 8.2.5. Plots of (a) Intermediate versus Major Principal Strain and (b) Minor versus Major Principal Strain Torsion Shear Tests in Sector I.

(a)

(b)

Figure 8.2.6. Plots of (a) Intermediate versus Major Principal Strain and (b) Minor versus Major Principal Strain Torsion Shear Tests in Sector III.


Figure 8.2.7. Comparison of Stress-Strain relations and void ratio changes in triaxial compression tests on specimens with $\mathrm{H} / \mathrm{D}=1.0$ and 2.7 for (a) dense and (b) loose Santa Monica Beach Sand (after Lade 1982).

### 8.3 Strains and Strain Increment vectors in Torsion Shear Tests

## Mohr's Circles

Using Mohr's circles, the direction of major principal stress, major principal stress increment and major principal strain increment can be found. Figure 8.3.1 shows the Mohr's circles diagrams used in determining these directions. From these diagrams, relationships can be derived for these angles shown below.

$$
\tan (2 \psi)=\frac{2 \tau_{z \theta}}{\sigma_{z}-\sigma_{\theta}}
$$

Eq. 8.3
$\tan (2 \chi)=\frac{2 \Delta \tau_{z \theta}}{\Delta \sigma_{z}-\Delta \sigma_{\theta}}$
$\tan (2 \zeta)=\frac{2 \Delta \varepsilon_{z \theta}}{\Delta \varepsilon_{z}-\Delta \varepsilon_{\theta}}$
where $\psi$ is the angle between $\sigma_{1}$ and vertical, $\chi$ is the angle between $\Delta \sigma_{1}$ and vertical and $\zeta$ is the angle between $\Delta \varepsilon_{1}$ and vertical. Note that the angle labeled $\psi$ in Figure 8.3.1 is indicated by $\alpha$ in this presentation.
(a)


Figure 8.3.1. Directions of Major Principal (a) stress, (b) stress increment and (c) strain increment direction in Torsion Shear Tests (after Hong and Lade, 1989).

## Relations between strains

The normal strain differences and shear strains were calculated for torsion shear tests. It is important to state that these are total strains, both elastic and plastic. These have been plotted in Figures 8.3.2 through 8.3.6. The plots are separated by b-value=constant so that the changes with varying alpha directions can be seen. Tests with $\alpha=0^{\circ}$ and $\alpha=90^{\circ}$ do not have any shear strain and therefore run along the $\varepsilon_{z \theta}=0$ lines when plotted. By taking the slope of these plots, the strain increment direction can be calculated (Eq. 8.3). Test with $\alpha=0^{\circ}$ and $\alpha=22.5^{\circ}$ have a positive strain difference. The other alpha values $\left(\alpha=45^{\circ}, \alpha=67.5^{\circ}\right.$ and $\alpha=90^{\circ}$ ) have negative strain differences. This is because $\varepsilon_{z}$ is less than $\varepsilon_{\theta}$ for these tests. As can be seen from looking at the graphs presented below in a series of increasing b-values, the shear strains and normal strain differences get smaller and smaller as the b-value increases. This was also seen when looking at the stress-strain curves presented in Chapter 7.


Figure 8.3.2. Normal Strain Differences versus Shear strain for Torsion Shear Tests with $b$-value $=0.0$.


Figure 8.3.3. Normal Strain Differences versus Shear strain for Torsion Shear Tests with b-value $=0.25$.


Figure 8.3.4. Normal Strain Differences versus Shear strain for Torsion Shear Tests with b -value $=0.50$.


Figure 8.3.5. Normal Strain Differences versus Shear strain for Torsion Shear Tests with b-value $=0.75$.


Figure 8.3.6. Normal Strain Differences versus Shear strain for Torsion Shear Tests with b -value $=1.0$.

When looking at tests with $b=0$, TS 31 (where $\alpha=45^{\circ}$ ) showed the largest amount of shear strain. TS 3 and $27^{*}\left(\right.$ at $\alpha=22.4^{\circ}$ and $\alpha=24^{\circ}$ ) showed a similar amount of shear strain as TS $14\left(\alpha=67.3^{\circ}\right)$. Tests with $\alpha=0^{\circ}$ have less total strain difference than that of $\alpha=90^{\circ}$ tests. For $\mathrm{b}=0.25$ tests, shear strains were more scattered. Two or more tests are presented for each direction. The four tests that have less strain (TS 4*, TS 28, TS 32* and TS 36) show a decrease in shear strain as the alpha value increases. Tests TS 5*, TS 9 and TS 35* all fail at approximately the same shear strain of $2.25 \%$. Tests with $\alpha=0^{\circ}$ have almost the same strain difference as $\alpha=90^{\circ}$ tests.

Tests with $\mathrm{b}=0.5$ show an increase in shear strain as alpha goes from $22.5^{\circ}$ to $45^{\circ}$. TS $37^{*}$ has a large increase in shear strain in it's last two points as it reaches failure. Without these two points, TS 15 and TS 37* have very similar curves, ending at about $1.5 \%$ shear strain. The general trend shows a decrease in shear strain from $45^{\circ}$ to $67.5^{\circ}$. This is similar to $\mathrm{b}=0$ strains. Tests with $b=0.75$ strains a little shorter that $b=0.5$ strains. The highest strain occurs at $\alpha=45^{\circ}$. Once again, there is scatter within the various tests at $\alpha=22.5^{\circ}$ and $\alpha=67.5^{\circ}$, showing tests TS $30^{*}$ and $39^{*}$ having more strain than TS 29, TS 40 and TS 38. Lastly, $\mathrm{b}=1.0$ tests showed the least amount of strains (both normal shear difference and shear strain). There is an increase in shear strain as alpha increases.

## Strain Increment Vector Directions

When looking at Figures 8.3.2 through 8.3.6, the initial part of most slopes is not linear. This is due to the presence of elastic strains in these plots. The elastic strains were not subtracted from the total strains. As is seen in Figures 8.3.7 through 8.3.11, the elastic strains create an initial deviation of strain increment direction from the major stress direction. As there is more stress applied, and the specimen is in the plastic range of the stress-strain curve, the strain increment direction coincides with the major stress direction. This deviation is mostly seen in $\alpha=22.5^{\circ}$ conditions. Since there are no shear strains for $\alpha=0^{\circ}$ and $\alpha=90^{\circ}$ tests, they are not plotted on Figures 8.3.7 through 8.3.11. Points only until failure are plotted. Strains after failure are not considered as there are great non-uniformities after failure. Only tests that had the correct stress path have been presented in Figure 8.3.7 through 8.3.11. These tests were part of the second set of tests in which the uplift correction was accounted
for and the targeted alpha and b-values were reached. Although, the other tests are not wrong, these tests were chosen as representative tests to show the observed patterns.

As seen in the b -value $=0$ tests, TS 3 starts out having a significantly higher strain increment direction than principal stress direction. As the specimen is sheared, the strain increment direction approaches the principal stress direction. TS 3 levels off near $\gamma_{z \theta}=1.5 \%$ $\left(\varepsilon_{z \theta}=0.75 \%\right)$. Looking closely at Figure 8.3.2, the slope of shear strain to normal strain difference begins to become linear near this same point as well. This is where the elastic strains no longer are so large and plastic strains dominate. There is a small deviation also seen in TS 14 up until about $\gamma_{z \theta}=1.0 \%\left(\varepsilon_{z \theta}=0.50 \%\right)$. It is important to remember that $\gamma_{z \theta}=2 \varepsilon_{\mathrm{z} \mathrm{\theta}}$. This change of slope is also seen in Figure 8.3.2. There is a slight change of strain increment direction for TS 31 at around $\gamma_{\mathrm{z}_{0}}=0.5 \%\left(\varepsilon_{\mathrm{z}_{0}}=0.25 \%\right)$. This change is so small that it is not very pronounced in Figure 8.3.2. Although there is a slight deviation from the principal stress direction, it becomes smaller as alpha value increases.

It is interesting to note that the strain increment directions for $\alpha=22.5^{\circ}$ and $\alpha=45^{\circ}$ are slightly above the stress direction. For $\alpha=67.5^{\circ}$, the strain increment directions are slightly below. This is due to the cross-anisotropic nature of the specimen in correlation with its bedding planes. At $\alpha=22.5^{\circ}$, the horizontal bedding planes cause the strain increment direction to become more horizontal and therefore show a slightly higher angle. At $\alpha=45^{\circ}$, this also
occurs but to a lesser extent. However, at $\alpha=67.5^{\circ}$, the strain increment direction becomes less than the principal stress direction. This pattern occurs for all $b$-values.

Similar to Figure 8.3.7, tests with other b-values show elastic behavior towards the beginning of the test. As strains become more plastic, they tend to line up with the principal stress direction. In a similar pattern, all $\alpha=22.5^{\circ}$ and $\alpha=45^{\circ}$ tests show slightly higher strain increment directions and $\alpha=67.5^{\circ}$ show slightly lower. Although not large, the relative biggest variation occurs at $\alpha=22.5^{\circ}$. Tests with $\alpha=45^{\circ}$ have the closest strain increment directions and principal stress directions throughout the entire straining of the specimen. Tests with $\mathrm{b}=1$ (seen in Figure 8.3.11) show the most variation towards the beginning of the test (in the elastic region). Table 8.3.1 summarizes the deviations of strain increment directions and principal stress directions at failure. Figure 8.3 .12 presents a drawing showing the bedding planes of a specimen. The general pattern of what occurred with stress directions and strain increment directions for all b-values is represented in the drawing.

Table 8.3.1. Summary of Strain Increment directions and principal stress directions at failure.

|  |  |  |  |  |
| :---: | :---: | :---: | :---: | :---: |
| $\mathbf{b}=\mathbf{0}$ |  |  | $\alpha$ |  |
| TS 3 | 28.3 | 22.4 | 5.9 |  |
| TS 31 | 48.14 | 44.95 | 3.19 |  |
| TS 14 | 66.09 | 67.33 | -1.24 |  |
| $\mathbf{b}=\mathbf{0 . 2 5}$ |  |  |  |  |
| TS 28 | 30.1 | 23.7 | 6.4 |  |
| TS 9 | 46.9 | 45 | 1.9 |  |
| TS 36 | 63.7 | 67.8 | -4.1 |  |
| $\mathbf{b}=\mathbf{0 . 5 0}$ |  |  |  |  |
| TS 6 | 26.7 | 22.5 | 4.2 |  |
| TS 33 | 47.24 | 45 | 2.24 |  |
| TS 15 | 61.8 | 67.5 | -5.7 |  |
| b=0.75 |  |  |  |  |
| TS 29 | 25.9 | 22.2 | 3.7 |  |
| TS 11 | 48.6 | 45 | 3.6 |  |
| TS 38 | 64.2 | 67.4 | -3.2 |  |
| b=1.0 |  |  |  |  |
| TS 8 | 29.6 | 22.6 | 7 |  |
| TS 34 | 47.9 | 45 | 2.9 |  |
| TS 17 | 68.2 | 62.2 | 6 |  |



Figure 8.3.7. Principal Stress and Strain Increment Directions versus Engineering Shear Strain for $\mathrm{b}=0$ Torsion Shear Tests.


Figure 8.3.8. Principal Stress and Strain Increment Directions versus Engineering Shear Strain for $b=0.25$ Torsion Shear Tests.


Figure 8.3.9. Principal Stress and Strain Increment Directions versus Engineering Shear Strain for $\mathrm{b}=0.50$ Torsion Shear Tests.


Figure 8.3.10. Principal Stress and Strain Increment Directions versus Engineering Shear Strain for $b=0.75$ Torsion Shear Tests.


Figure 8.3.11. Principal Stress and Strain Increment Directions versus Engineering Shear Strain for $\mathrm{b}=1.0$ Torsion Shear Tests.

*Note: Strain Increment Directions not to scale. Only a conceptual drawing
Figure 8.3.12. Patterns of Strain Increment Direction and Principal Stress Directions observed for Torsion Shear Tests.

Looking at the strain increment directions and the stress directions from a different perspective, they can be plotted showing their difference. Figures 8.3.13 though 8.3.17 plot the principal stress directions versus the shear strain. The stress paths are plotted and arrows representing the strain increment directions are overlaid. These arrows show the variation from the stress direction. As can be seen from the graphs, the difference is only slightly noticeable towards the beginning of the tests, as also seen in Figures 8.3.7 through 8.3.11.


Figure 8.3.13. Principal Stress and Strain Directions versus Engineering Strain for $b=0$ Torsion Shear Tests.


Figure 8.3.14. Principal Stress and Strain Directions versus Engineering Strain for $b=0.25$ Torsion Shear Tests.

Principal Stress and Strain Increment Directions vs Engineering Strain for $\mathbf{b}=\mathbf{0 . 5 0}$ Tests


Figure 8.3.15. Principal Stress and Strain Directions versus Engineering Strain for $b=0.50$.


Figure 8.3.16. Principal Stress and Strain Directions versus Engineering Strain for $b=0.75$ Torsion Shear Tests.


Figure 8.3.17. Principal Stress and Strain Directions versus Engineering Strain for $\mathrm{b}=1.0$ Torsion Shear Tests.

### 8.4 Conclusion

The preceding pages have summarized and analyzed the strains for both true triaxial and torsion shear tests. For the cases that were possible, where $\alpha=0^{\circ}$ and $\alpha=90^{\circ}$ in Sector I and III, torsion shear and true triaxial strains were directly compared. Although the patterns were similar, the total amount of strains was significantly less in the torsion shear tests. Possible explanations are the difference in stress paths for each type of test (constant mean confining stress for torsion shear and constant minor principal stress for true triaxial) and the different end restraints (full frictional surfaces for torsion shear and double lubricated ends for true triaxial tests). Shear strains have been presented for torsion shear tests with rotation. Although no single pattern exists when looking at the various b -values, when looking at the magnitudes of the shear strains as the alpha values increased, the strain increment directions all show non-associated flow in the region of plastic strains and the stress directions become asymptotic to each other but never reached the same directions completely.

## 9. Shear Band Analysis

Shear bands were observed for a number of triaxial, true triaxial and torsion shear tests. After the specimen had reached failure, it was held under vacuum in order to measure and record any shear bands that had developed. In general, shear bands for triaxial and true triaxial tests were seen after peak failure (in the softening regime). Shear bands in torsion shear tests occurred (with few exceptions) at peak failure. In the sections that follow, the recorded shear band inclination angles will be compared to the existing theories (Coulomb, Roscoe and Arthur), which predict shear band inclination angle. As was stated in Chapter 2, Section 2.6, the Coulomb, Roscoe and Arthur equations are as follows:

$$
\begin{equation*}
\alpha_{c}=45+\left(\frac{\phi}{2}\right) \tag{Eq. 9.1}
\end{equation*}
$$

$\alpha_{R}=45+\frac{\psi}{2}$
$\alpha_{A}=45+\left(\frac{\phi+\psi}{4}\right)$
where $\alpha_{C}$ is the Coulomb angle inclination, $\alpha_{R}$ is the Roscoe angle inclination and $\alpha_{A}$ is the Arthur angle inclination. $\phi$ and $\psi$ are the friction angle and dilation angle, respectively.

As already mentioned, the dilation angle can be calculated by the following equation,

$$
\begin{equation*}
\psi=\sin ^{-1}\left(\frac{\left(\Delta \varepsilon_{v} / \Delta \varepsilon_{1}\right)}{\left(\Delta \varepsilon_{v} / \Delta \varepsilon_{1}\right)-2}\right) \tag{Eq. 9.4}
\end{equation*}
$$

where $\Delta \varepsilon_{1}$ is the change in major principal strain and $\Delta \varepsilon_{\mathrm{v}}$ is the change in volumetric strain. The slope of the volume change curve is calculated over several points at/near failure.

### 9.1 Shear Band Analysis for Triaxial Tests

As described in Chapter 3, a series of 10 triaxial compression tests were performed under different confining pressures and bedding plane directions of $\alpha=0^{\circ}$ and $\alpha=90^{\circ}$ with the major principal stress to determine the necessary parameters for calibration a constitutive model for cross-anisotropic sand deposits. Shear bands developed and their inclinations were recorded. In most cases for triaxial compression, the shear band plane that was seen was skewed relative to the surface of the specimen. It is possible that the direction of the plane changed by a small amount as it hit the corners of the specimen. Therefore, to calculate the true angles of the shear bands for each plane, vector analysis was performed using the measured heights, widths and directions of the shear bands recorded after failure. An example picture of where the plane developed is shown in Figure 9.1.1. In cases where this occurred, the angle of each plane with respect to the horizontal was calculated. Instead of computing an average between both planes, each plane is plotted in the results. This shows a range and the reader can get a better idea of the variation in shear band inclination angles. In some cases, only one plane developed. Figure 9.1.2 shows an example of this occurrence.

Therefore, vector analysis was not needed for this case since the angle of inclination was directly measured from the horizontal on the specimen after failure.

Table 9.1.1 summarizes the angles of inclination for the 10 triaxial tests. The angle of inclination is measured from the horizontal to the direction of shear band plane. A sketch of the measured shear band angles, $\alpha_{s b}$, is shown in Figure 9.1.3. For tests where vectors were calculated, the angles were also from the horizontal to the developed shear band planes (seen in Figure 9.1.4).


Figure 9.1.1. Example of sheared specimen with two distinct shear band planes after failure in triaxial testing.


Figure 9.1.2. Example of shear specimen with only one distinct shear band plane after failure in triaxial testing.


Figure 9.1.3. Sketch of the measured shear band angle for all three sectors in triaxial and true triaxial tests. $\alpha_{\mathrm{sb}}$ is measured from the horizontal to the shear plane.


Figure 9.1.4. Schematic showing two planes in a tall prismatic specimen. Shear band inclination angles, $\alpha_{\mathrm{sb}}$ (upper) and $\alpha_{\mathrm{sb} \text { (lower) }}$ are measured from the horizontal plane to the corresponding shear plane.

In order to calculate the angle between the shear band plane and the horizontal plane for each of the upper and lower shear band planes, the angle between the normal vectors of the horizontal plane and the shear plane was calculated. The normal vector of each horizontal plane is $(\mathrm{x}, \mathrm{y}, \mathrm{z})=(0,0,1)$. The normal vector, $\mathrm{n}_{\text {sp }}$, of the respective shear plane is calculated as the cross product (using the determinant method) of the two measured vectors of the shear plane at the sides of the specimen. The geometric extensions of the visible shear bands for each specimen were measured and put into the following equations to calculate the corresponding vectors, $\mathrm{v}_{1}$ and $\mathrm{v}_{2}$.

$$
\begin{align*}
& \vec{v}_{1}=\left(x_{1}, y_{1}, z_{1}\right) \\
& \vec{v}_{2}=\left(x_{2}, y_{2}, z_{2}\right) \tag{Eq. 9.1.1}
\end{align*}
$$

Then, the cross product was calculated using the following equation:

$$
n_{s p}=\left(y_{1} z_{2}-z_{1} y_{2}\right) i-\left(x_{1} z_{2}-z_{1} x_{2}\right) j+\left(x_{1} y_{2}-y_{1} x_{2}\right) k
$$

where
$i=\left[\begin{array}{l}1 \\ 0 \\ 0\end{array}\right]$
$j=\left[\begin{array}{l}0 \\ 1 \\ 0\end{array}\right]$
$k=\left[\begin{array}{l}0 \\ 0 \\ 1\end{array}\right]$

Using a reference plane, $\mathrm{v}_{3}$

$$
\begin{aligned}
& \vec{v}_{3}=\left(a_{1}, b_{1}, c_{1}\right) \\
& \vec{v}_{3}=(0,0,1)
\end{aligned}
$$

Eq. 9.1.2

Eq. 9.1.3
the shear plane inclination angle can be calculated using

$$
\begin{aligned}
& \cos \alpha_{s p}=\frac{a_{1}\left(y_{1} z_{2}-z_{1} y_{2}\right)-b_{1}\left(x_{1} z_{2}-z_{1} x_{2}\right)+c_{1}\left(x_{1} y_{2}-y_{1} x_{2}\right)}{\sqrt{\left(y_{1} z_{2}-z_{1} y_{2}\right)^{2}+\left(x_{1} z_{2}-z_{1} x_{2}\right)^{2}+\left(x_{1} y_{2}-y_{1} x_{2}\right)^{2}} \sqrt{a_{1}^{2}+b_{1}^{2}+c_{1}^{2}}} \\
& \cos \alpha_{s p}=\frac{1\left(x_{1} y_{2}-y_{1} x_{2}\right)}{\sqrt{\left(y_{1} z_{2}-z_{1} y_{2}\right)^{2}+\left(x_{1} z_{2}-z_{1} x_{2}\right)^{2}+\left(x_{1} y_{2}-y_{1} x_{2}\right)^{2}}}
\end{aligned}
$$

Table 9.1.1. Summary of Triaxial Test Shear Band Inclination Angles.

| Test No | $\alpha(\mathrm{deg})$ | $\alpha_{\text {sb }}$ (deg) |
| :--- | :---: | :---: |
| 1 (TT\#1) (lower plane) | 0 | 51.00 |
| 1 (TT\#1) (upper plane) | 0 | 57.82 |
| 2 (lower plane) | 0 | 57.40 |
| 2 (upper plane) | 0 | 56.89 |
| 3 (lower plane) | 0 | 57.04 |
| 3 (upper plane) | 0 | 58.88 |
| 4 (lower plane) | 0 | 52.97 |
| 4 (upper plane) | 90 | 60.29 |
| 5 (one plane) | 90 | 60.00 |
| 6 (lower plane) | 90 | 59.13 |
| 6 (upper plane) | 90 | 66.86 |
| 7 (lower plane) | 90 | 60.56 |
| 7 (upper plane) | 90 | 63.00 |
| 8 (TT\#13) (one plane) | 90 | 58.40 |
| 9(W1) (lower plane) | 90 | 59.70 |
| 9(W1) (upper plane) | 00.00 |  |
| 10 (A7) (one plane) | 00 | 60 |

In Figures 9.1.5 and 9.1.6, the measured shear band inclination angles are compared to the predictions of Coulomb, Roscoe and Arthur. As can be seen, for $\alpha=0^{\circ}$ tests, the inclination angles are between the Arthur and Roscoe shear band inclination predictions. Specimens
with vertical bedding planes $\left(\alpha=90^{\circ}\right)$ have slightly higher inclination angles that favor the Coulomb and Arthur predictions. The stress-strain curves have also been presented in Figures 9.1.6 and 9.1.7. The first time that a shear band was seen is indicated with an arrow for each test. For all triaxial tests, the shear bands developed after peak failure well into the softening regime. Therefore, shear bands did not affect the peak strength of the specimens.


Figure 9.1.5. Shear Band Inclination Angles for Triaxial Compression Tests with horizontal bedding planes $\left(\alpha=0^{\circ}\right)$.


Figure 9.1.5. Shear Band Inclination Angles for Triaxial Compression Tests with vertical bedding planes $\left(\alpha=90^{\circ}\right)$. Test 9 is an extension test with $\alpha=90^{\circ}$ bedding planes.


Figure 9.1.6. Stress-Strain curves for Triaxial compression tests indicating Shear Band development for $\alpha=0^{\circ}$ tests.


Figure 9.1.7. Stress-Strain curves for Triaxial compression tests indicating Shear Band development for $\alpha=90^{\circ}$ tests.

### 9.2 Shear Band Analysis for True Triaxial Tests

Eighteen true triaxial tests were sheared in this experimental program. Shear bands were seen in all but two tests. For Tests TT \#16 and TT\#18, after the specimen was sheared, a vacuum was inadvertently not applied to the specimen. Therefore, when the horizontal loading plates were removed and the specimen was unloaded, the specimen collapsed. No shear bands could be seen prior to unloading. However, for the other 16 tests, vacuum was applied, and shear bands were recorded. Most failed in one plane as shown in Figure 9.1.3. The normal vector to this failure plane was always in the plane of the major and minor principal stresses. A typical picture of a sheared specimen is shown in Figure 9.2.1. Tests that
did not fail along one plane were analyzed using vector analysis (as explained in Section 9.1). This only occurred for Test TT\#1. For two tests, Tests TT\#7 and TT\#12, where $b=1$, the shear bands developed vertically. It is speculated that this also occurred for Test TT\#18 but it is not known for certain since the specimen was not held under vacuum once unloaded. The behavior of Tests TT \#7 and TT \#12 show both specimens' shearing parallel to the loading direction (Figure 9.2.2). This occurs mid-height of the specimen. Towards the top and bottom of the specimen (towards the end restraints), some shear bands are seen at slightly different angles.

Table 9.2.1 summarizes the shear band inclination angles recorded for true triaxial tests. Figures 9.2.3 through 9.2.4 show the measured shear band inclination angles versus b-value for each of the three sectors. The predicted Coulomb, Arthur and Roscoe inclination angles are also presented. Although there is some scatter in the results, most of the triaxial tests for all three sectors are closest to the Arthur shear band inclination angle prediction. The Arthur prediction takes into account the friction angle and the dilation angle when predicting the shear band inclination. It is important to note, that as seen in Figure 9.2.1, shear bands were free to develop fully across the entire height of the specimen. In cases where the shear band was not restricted by the top and bottom plates, the ends of the shear bands could be seen a couple of centimeters above or below the bottom and top plate. Clear visible shear bands were seen after peak failure in the softening regime of the stress-strain curve for all but $b=1$ triaxial tests. At $b=1$, shear bands developed at failure, causing a sharp decrease in strength seen on the stress-strain behavior presented in Chapter 5 (see Figures 9.2.6, 9.2.7, and 9.2.8).


Figure 9.2.1. Typical example of shear band development for most True Triaxial tests.


Specimen TT\#12 $\left(\alpha=90^{\circ}, b=1\right)$
Figure 9.2.2. Pictures of Tests TT\#7 (Sector I) and TT\#12 (Sector II) at $\mathrm{b}=1$ where shear bands developed vertically at failure.

Table 9.2.1. Summary of Shear Band Inclination Angles for True Triaxial Tests.

| Sector I | $\alpha$ (deg) | b-value | $\alpha_{\text {sb }}$ (deg) |
| :---: | :---: | :---: | :---: |
| TT\#1 (lower)/(upper) | $0^{\circ}$ | 0.00 | 51.00/57.82 |
| TT\#2 | $0^{\circ}$ | 0.24 | 61.50 |
| TT\#3 | $0^{\circ}$ | 0.51 | 62.00 |
| TT\#4 | $0^{\circ}$ | 0.75 | 64.00 |
| TT\#5 | $0^{\circ}$ | 0.70 | 64.50 |
| TT\#6 | $0^{\circ}$ | 0.72 | 62.00 |
| TT\#7 | $0^{\circ}$ | 1.00 | 90, 70 |
| Sector II |  |  |  |
| TT\#8 | $90^{\circ}$ | 0.25 | 64.50 |
| TT\#9 | $90^{\circ}$ | 0.49 | 60.00 |
| TT\#10 | $90^{\circ}$ | 0.70 | 62.00 |
| TT\#11 | $90^{\circ}$ | 0.69 | 62.00 |
| TT\#12 | $90^{\circ}$ | 1.00 | 90.00 |
| Sector III |  |  |  |
| TT\#13 | $90^{\circ}$ | 0.00 | 57.50 |
| TT\#14 | $90^{\circ}$ | 0.25 | 59.50 |
| TT\#15 | $90^{\circ}$ | 0.49 | 65.00 |
| TT\#16 | $90^{\circ}$ | 0.72 | n/a |
| TT\#17 | $90^{\circ}$ | 0.72 | 63.00 |
| TT\#18 | $90^{\circ}$ | 0.95 | n/a |

Looking at Figure 9.2.3, there is a slight positive slope in shear band inclination angle as bvalue increases in Sector I. At $b=1$, the shear band inclination is vertical at 90 degrees. The failure occurs in the horizontal direction. This shows that as the intermediate stress is increased, shear bands tend to develop at steeper angles in the major-minor principal stress plane. Looking at the anisotropy of the bedding planes, the strongest direction for the sand grains to slide in Sector I, would be in the $\sigma_{2}$ and $\sigma_{3}$ directions. The sand grains would tend to slide less in the $\sigma_{1}$ direction for this configuration. At the point where the intermediate and major principal stress directions are the same $(b=1.0)$, the failure direction switches and creates vertical shear bands. Sector II does not show an increased inclination angle as the bvalue increases. In Sector II, the $\sigma_{2}$ direction is the direction where the bedding planes create a condition where the sand grains would want to slide less than in the other directions. By changing the intermediate principal stress, there may not be too much of an influence on the shear band inclination angle due to the way the bedding planes are aligned. By switching the bedding planes in Sector III so that the $\sigma_{3}$ direction is the direction where the sand grains have the hardest time sliding, there is once again (as seen in Sector I) an increase in shear band inclination angle as $b$-value increases, allowing for sliding to occur easily between the sand grains.


Figure 9.2.3. True Triaxial shear band inclination angles for Sector I tests, including Coulomb, Arthur and Roscoe predictions.


Figure 9.2.4. True Triaxial shear band inclination angles for Sector II tests, including Coulomb, Arthur and Roscoe predictions.


Figure 9.2.5. True Triaxial shear band inclination angles for Sector III tests, including Coulomb, Arthur and Roscoe predictions.


Figure 9.2.6. Stress-strain behavior for Sector I tests $\left(\alpha=0^{\circ}\right)$ with initial point of shear band development is indicated with arrows.


Figure 9.2.7. Stress-strain behavior for Sector II tests $\left(\alpha=90^{\circ}\right)$ with initial point of shear band development is indicated with arrows.


Figure 9.2.8. Stress-strain behavior for Sector III tests $\left(\alpha=90^{\circ}\right)$ with initial point of shear band development is indicated with arrows.

### 9.3 Shear Band Analysis for Torsion Shear Tests

The true triaxial shear band inclination angles described in Section 9.2 had only one failure plane (all except TT \#1 which had two planes). Shear bands that developed in torsion shear specimens were not always in just one direction as was seen in the true triaxial tests. In cases where more than one direction was measured, the span of directions for each test is presented in the results. Out of the 44 torsion shear tests performed, in 41 tests, shear bands were observed. These shear bands are presented in the section that follows. In Test 1*, there were no shear bands observed. For Tests 13* and 34, the top cap seemed to slip during shearing and therefore, no shear bands were seen.

Several shear band patterns developed in the torsion shear specimens. The shear bands were able to develop freely and were not affected by the rubber membranes that are on the specimen's vertical sides. Most shear bands were seen near or right at failure. In some cases (usually for $\mathrm{b}=0.75$ and $\mathrm{b}=1.0$ ), the shear bands developed right at failure and grew extremely quickly, deforming the specimen. In cases where that did not happen, although peak failure was reached, the specimen was sheared more to develop the shear bands more fully. However, there was no gain in strength after peak failure. By just apparently looking at the stress-strain and volume change curves, it is hard to tell where the shear bands developed. Therefore, the time that they first appeared was recorded manually. There is no sharp drop in
stress (as seen in the shear bands that developed for the true triaxial tests) and softening in the stress-strain curves.

Shear bands were measured on the specimen from the horizontal up to the shear band. The grid drawn on the specimen prior to shearing helped in measuring correctly the angles of the shear bands. When the major principal stress was inclined, the major principal stress was added to the shear band angle that was measured (as shown in Figure 9.3.1a). When the shear band was negative of the horizontal, the shear band inclination angle was considered negative and was then found as shown in Figure 9.3.1b. For the sake of plotting all positive numbers, if the shear band was clockwise to the horizontal, it was made positive once the correct inclination had been calculated.


Figure 9.3.1. Inclination of shear bands relative to the major principal stress plane for torsion shear tests (after Lade et al. 2008).

Representative patterns are seen in Table 9.3.1. In many occasions, a torsion shear specimen showed more than just one type of pattern. However, in order to group them, these 6 representative patterns have been named as follows: (a) zig-zag bands, (b) inclined bands, (c) horizontal bands, (d) crossed bands, (e) collapse and (f) r-theta failures. Z-theta failures failed in the z-theta plane and are somewhat similar to when necking occurs. R-theta failures are classified as such because the shear bands failed in the r-theta plane. In all other cases, the shear bands developed in the $z$-theta plane where the principal stress was applied. The specimen had a sufficient height to diameter ratio to allow the shear bands to develop completely. In cases where the shear bands did not hit the end restraints, they wrapped around the entire specimen.


Shear band inclination angles are shown in Figures 9.3.2, 9.3.5, 9.3.8, 9.3.11 and 9.3.14. Tests are separated by alpha=constant values so that the change of shear band inclination angle with b-value can be studied. The summary of shear band inclination angles for all torsion shear tests is presented in Table 9.3.2.

Table 9.3.2. Summary of Shear Band Inclination for Torsion Shear Tests.

| Test | alpha | b-value | a |  |
| :---: | :---: | :---: | :---: | :--- |
| sb | Patterns |  |  |  |
| 23 | 0 | 0 | $65-70$ | inclined, zig-zag (all around) |
| $1^{*}$ | 0 | 0.01 | n/a | none observed |
| $24^{*}$ | 0 | 0.27 | $63-71$ | zig zag (all around) |
| $25^{*}$ | 0 | 0.55 | $63-70$ | zig zag (all around) |
| 2 | 0 | 0.75 | $65-68$ | zig zag (all around) with r-theta |
| 26 | 0 | 1 | 90 | vertical r-theta |
| 3 | 22.41 | 0.00 | $41-45$ | inclined (wrap around) |
| $27^{*}$ | 24.04 | 0.02 | 54 | inclined (wrap around) |
| 28 | 22.97 | 0.26 | $64-66$ | inclined (wrap around) |
| $5^{*}$ | 24.18 | 0.27 | 58 | inclined (wrap around) |
| $4^{*}$ | 23.73 | 0.28 | 64 | inclined (wrap around) |
| 6 | 22.01 | 0.51 | $70-72$ | inclined (wrap around) |
| 29 | 22.77 | 0.75 | 83 | inclined r-theta (ffont) |
| $7 *$ | 24.11 | 0.83 | $89-91$ | inclined r-theta (front and back) |
| $30^{*}$ | 22.92 | 0.85 | $78-88$ | inclined r-theta (front) |
| 8 | 22.87 | 0.97 | 90 | inclined r-theta (front) |
| 31 | 44.91 | 0.00 | 60 | inclined (front) |
| $32^{*}$ | 31.76 | 0.18 | 47 | inclined (wrap around) (mid\&top) |
| 9 | 44.99 | 0.24 | $64-70$ | inclined (not very deep) |
| 33 | 44.99 | 0.50 | 65 | inclined (front bottom) |
| $10^{*}$ | 41.96 | 0.55 | 52 | inclined (wrap around) (bottom) |
| 11 | 44.98 | 0.75 | $65-70$ | inclined (wrap around) |
| $12^{*}$ | 41.54 | 0.81 | $60-64$ | inclined (wrap around) |
| $13^{*}$ | 33.86 | 0.94 | $n / a$ | under top cap (slip) |
| 34 | 44.97 | 1.00 | n/a | under top cap (slip) |
| 14 | 67.51 | 0.00 | 68 | sucked in at bottom |
| $35^{*}$ | 69.89 | 0.16 | 70 | under top cap |
| 36 | 67.80 | 0.25 | 68 | under top cap |
| 15 | 67.45 | 0.50 | 72 | under top cap and bottom ring |
| $37^{*}$ | 69.91 | 0.55 | $60-61$ | under top cap (deep) |
| 38 | 67.54 | 0.75 | $58-68$ | under top cap (deep) |
| $39^{*}$ | 70.62 | 0.79 | 71 | under top cap (deep) |
| $40^{*}$ | 73.06 | 0.80 | $60-63$ | horizontal wrap around r-theta |
| $16^{*}$ | 72.06 | 0.96 | 72 | under top cap (deep) |
| 17 | 68.26 | 1.00 | $62-63$ | horizontal wrap around r-theta |
| 41 | 90 | 0.00 | $67-69$ | collapse inward, crossed |
| 18 | 90 | 0.04 | $65-70$ | collapse inward, crossed |
| $19^{*}$ | 90 | 0.07 | 65 | collapse inward, crossed |
| $42^{*}$ | 90 | 0.32 | $64-66$ | crossed all the way around |
| $20^{*}$ | 90 | 0.54 | $62-70$ | crossed all way around, w r-theta |
| $43^{*}$ | 90 | 0.78 | $57-58$ | slanted r-theta all around |
| $21^{*}$ | 90 | 0.78 | $64-72$ | r-theta (almost horizontal), slanted |
| $44^{*}$ | 90 | 0.99 | 90 | deep r-theta around under top cap |
| $22^{*}$ | 90 | 0.99 | $65-70$ | r-theta around top, crossed |
|  |  |  |  |  |

For $\alpha=0^{\circ}$ tests (Figure 9.3.2), if a trend line would be drawn through all the tests, the shear band inclination angle is almost constant. The spans of shear band angles are slightly higher than the Coulomb prediction at $\mathrm{b}=0$ and then get closer to the Arthur prediction as $\mathrm{b}=1$. Tests $23,24^{*}, 25^{*}$, and 2 all showed a zig-zag pattern wrapping around the middle of the specimen. For TS 2, a vertical "canyon" (r-theta band) developed at the edge of one of the zig-zag patterns. Test 26 had a vertical r-theta band develop at failure. Pictures of these two tests are shown in Figure 9.3.3 (a) and (b), respectively. Since the failure angles are within the specimen wall and are not visible on the outside surface of the specimen, they have not been included in the summary plots that follow. A direct comparison to the prediction angles is not possible for these situations.


Figure 9.3.2. Shear band inclination angle for $\alpha=0^{\circ}$ Torsion Shear Tests.


Figures 9.3.3. Pictures of the outer and inner shear bands that developed for (a) Tests 2 and (b) 26.

The stress-strain curves presented in Section 7.3 has been modified slightly to show the initiation of the shear band development (Figure 9.3.4). For tests with lower b-values of 0 and 0.27 where shear bands were seen, Tests 23 and $24^{*}$, respectively, after the shear band was first seen, the test continued to develop the shear band more. However, there was not any significant gain in strength for the specimen. For Test 25*, after the shear band developed there was a significant decrease in strength. Tests 2 and 26* shear bands developed extremely quickly at failure. No shear bands were seen before the last points on the curve. These two tests had large deep "canyons" (r-theta bands) indicating failure in the intermediate stress direction (pictures are shown in Figure 9.3.3). Once these deep r-theta bands developed, the volume change was so large in the inner cell, that the test had to be stopped. However, it was clear that the sample had failed at the point of the shear band developments.


Figure 9.3.4. Stress-strain curves for $\alpha=0^{\circ}$ Torsion shear tests indicating the onset of Shear band development.

The shear band inclination angles for torsion shear tests with $\alpha=22.5^{\circ}$ are presented in Figure 9.3.5. The angles clearly increase as the b-value increases but do not follow any one of the three predictions at all. Tests 3 and 27* are below or at the Roscoe inclination angle. Tests 28, 4* and 5* appear to gather around the Arthur angle. Test 6 is near the Coulomb angle. Tests 7*, 8, 29 and 30 are all almost near 90 degrees, far away from the Coulomb angle. These tests developed r-theta shear bands at different angles. They all developed right at failure and are shown in Figure 9.3.6. For these tests, the failure direction was radially, in the intermediate principal stress direction.


Figure 9.3.5. Shear band inclination angle for $\alpha=22.5^{\circ}$ Torsion Shear Tests.

The stress-strain curves for $\alpha=22.5^{\circ}$ are presented in Figure 9.3.7. In this figure, the onset of shear band development is pointed out. Tests where the b -value was less than 0.5 had shear bands that developed past failure. Tests with $b=0$ shear bands developed shear bands at further strain after failure than the $b=0.25$ tests. At $b=0.5$ and $b<0.5$, the tests show a fast decrease in strength after the shear band is developed. Once the shear band develops at these high b-values, there is no further increase in strength and the strength cannot be kept constant like that of the $b=0$ tests.


Figure 9.3.6. Pictures of Shear band r-theta bands developed for Tests (a) 7*, (b) 8, (c) 29 and (d) $30^{*}$.


Figure 9.3.7. Stress-strain curves for $\alpha=22.5^{\circ}$ Torsion shear tests indicating the onset of Shear band development.

Figure 9.3.8 shows the shear band inclination angles for $\alpha=45^{\circ}$ tests. At $\alpha=45^{\circ}$, shear bands were all inclined but they were more horizontal than those at $\alpha=0^{\circ}$ and $\alpha=22.5^{\circ}$. An example is shown in Figure 9.3.9a. With the exception of Test $32^{*}$ and Test $10^{*}$, most of the shear band inclination angles follow the Coulomb prediction. It should be pointed out that Test 32* actually was at $\alpha=32^{\circ}$ yet it is presented in the $\alpha=45^{\circ}$ degree tests. Both of these tests had shear bands that wrapped around the entire specimen. The circular band around the specimen that formed was mostly horizontal but with a slight angle. Pictures of Test $32^{*}$ and $10^{*}$ are shown in Figure 9.3.9b and 9.3.9c, respectively. As can be seen in the pictures, shear bands
also developed right below the top ring. This development also happened in most of the $\alpha=67.5^{\circ}$ tests that will be described in the next section.


Figure 9.3.8. Shear band inclination angle for $\alpha=45^{\circ}$ Torsion Shear Tests.


Figure 9.3.9. Examples of shear band that occurred at $\alpha=45^{\circ}$. (a) Test 11 shear bands are inclined and do not wrap all around specimen, except at top ring; (b) Test 10* and (c) Test 32* shear bands are more horizontal and wrap all around specimen.

The stress-strain curves are presented in Figures 9.3.10a and 9.3.10b. To avoid having too many tests in one graph, they were separated so that the shear band development location could be easily shown. The arrow for Test 31 indicates where the membrane was first seen to have kinks on the previously drawn grid. The kinks began forming well before the specimen failed. During Test $32^{*}$, shear bands near the top cap were first noticed and then secondly, parallel shear bands formed in the middle of the specimen (see Figure 9.3.9c). Test 9 had the shear bands develop right at peak failure and then after rotating the specimen more, then a shear band developed underneath the top cap. Test 33 had a shear band develop prior to reaching ultimate peak strength.


Figure 9.3.10a. Stress-strain curves for $\alpha=45^{\circ}$ Torsion shear tests indicating the onset of Shear band development. (Tests 31, 32*, 9, 33 and 10*).

In Test 11, a shear band was seen prior to peak failure. The stress ratio dropped a little and then increased to peak failure. At peak failure, the top cap slipped, and there was a drop in strength. This is seen in Figure 9.3.10b. Test $12 *$ 's shear band developed right after the peak stress ratio had been reached. Test 13* and Test 34 had shear bands develop quickly at failure, and were so large that an immediate stress drop was seen.


Figure 9.3.10b. Stress-strain curves for $\alpha=45^{\circ}$ Torsion shear tests indicating the onset of Shear band development. (Tests 11, 12* and 34).

Tests with inclination angles of $\alpha=67.5^{\circ}$ are presented in Figure 9.3.11. These angles are slightly higher than the Coulomb prediction. For all but three tests (Tests 14, 40* and 17), the shear bands developed right underneath the top cap. The bottom half of Test 14 collapsed inward at failure. This also happened for the $\alpha=90^{\circ}$ tests at $b=0$. Tests $35^{*}, 36$ and 15 had shear bands develop right underneath the top cap but they were not very thick. It looked as though the top cap had slipped but further inspection showed that it was not the top cap slipping but the shear band forming right at the end restraint. Tests $16^{*}, 37^{*}, 38$ and $39^{*}$ all had deeper and thicker shear bands form underneath the top cap ring. Tests $40^{*}$ and 17 had thick shear bands wrap all the way around the specimen in the shape of a ring (not
underneath the top cap ring). Pictures of these different shear band patterns are shown in Figure 9.3.12.

It is important to remember that at $\alpha=67.5^{\circ}$ tests, the friction angle/stress ratio of these tests were the lowest (see Figure 7.1.9 and 7.1.11). At $67.5^{\circ}$, the shear band direction is close to the bedding planes. In other words, when $\omega=0^{\circ}, \alpha=\alpha_{\mathrm{sb}}$ (see Figure 9.3.1). When these two planes line up exactly, the sand grains are allowed to move easily and therefore, can create less strength in the soil.


Figure 9.3.11. Shear band inclination angle for $\alpha=67.5^{\circ}$ Torsion Shear Tests.


Figure 9.3.12. Examples of shear band that occurred at $\alpha=67.5^{\circ}$. (a) Test 14 showing inward collapse at failure; (b) Test $37^{*}$ showing deep top cap shear band and a thin shear band along bottom base and (c) Test 40* showing horizontal deep shear band around the entire specimen.

Figure 9.3.13a shows the stress strain behavior tests from $b=0$ to $b=0.56$ for $\alpha=67.5^{\circ}$ tests. The first time that a shear band was physically seen is indicated as well. For these tests, shear bands occurred at different parts of the stress-strain curves. For example, Test 14's shear band was visible at about 4\% major principal strain. Failure occurred at $2.9 \%$. Test $35^{*}$ had the top cap shear band visible very near peak failure. Test 36 's shear band was seen right after peak failure, showing a significant decrease in strength once it developed. Test 15 had a top cap shear band develop at less that $1 \%$ major principal strain and then another shear band at the base of the specimen near failure. Test 37* also had a top cap shear band develop around $1 \%$ major principal strain. However, it kept increasing in strength until it failed at $1.8 \%$.


Figure 9.3.13a. Stress-strain curves for $\alpha=67.5^{\circ}$ Torsion shear tests indicating the onset of Shear band development (Tests 14, 35*, 36, 15, and 37*).

Similarly, as seen in Figure 9.3.13b, Tests 38 and 39* had shear bands develop very close to peak failure. No increase in strength was seen after the shear band developed. Tests 17 and 40* developed right at peak failure. A drop in strength was seen for these two tests right after the shear band developed.


Figure 9.3.13b. Stress-strain curves for $\alpha=67.5^{\circ}$ Torsion shear tests indicating the onset of Shear band development (Tests 38, 39*, 40, 17).

The last set of tests where $\alpha=90^{\circ}$ follows the Coulomb prediction as well. The average angle of shear band inclination for these tests is at 67.5 degrees. Tests at $\alpha=90^{\circ}$ showed crossed shear band patterns. For Tests 18, 19* and 41 (all at or near $b=0$ ), the specimen collapsed inward at failure. Crossed shear bands also developed coming out of the collapse (see Figure 9.3.15a). Tests $42^{*}$ and $20^{*}$ had shear bands criss-cross each other and wrap all the way around the specimen (see Figure 9.3.15b). Tests $21^{*}, 43^{*}, 44^{*}$ and 22* all had deep r-theta bands (inclined for $\mathrm{b}=0.75$ and almost horizontal at $\mathrm{b}=1$ tests). Tests $22^{*}$ and $44^{*}$ had deep r theta bands develop right underneath the top ring. For Test 22*, additional crossed shear
bands developed coming out of the top cap r-theta band. (see Figure 9.3.16c). Test 21* was similar (having a top cap r-theta band) but the additional shear bands were all around the body of the specimen (Figure 9.3.16a). Test 43* had a deep shear band r-theta band where two shear bands came together (almost crossing) but became horizontal at the meeting point. All shear bands occurred right at peak ultimate strength (Figures 9.3.17a and 9.3.17b).


Figure 9.3.14. Shear band inclination angle for $\alpha=90^{\circ}$ Torsion Shear Tests.

(a) Pictures of Test 18 (outside and inside) $\left(\alpha=90^{\circ}, b=0.04\right)$

(b) Pictures of Test 42* (front and back) $\left(\alpha=90^{\circ}, b=0.32\right)$

Figure 9.3.15. Examples of shear band that occurred at $\alpha=90$ (low b-values). (a) Collapse and two crossing shear bands occurred at failure of Test 18 (b) crossed shear bands at front and back of Test 42*.


Figure 9.3.16. Examples of shear band that occurred at $\alpha=90$ (high $b$-values). (a) Test 21* shows top cap r-theta shear band and smaller crossed bands across the specimen; (b) Tests 43* shows two deep slanted crossed r-theta bands gathering horizontally at front of the specimen; (c) Test 22* shows deep r-theta bands at the top cap with smaller crossed shear bands; (d) Test 44* shows a horizontal deep r-theta band at top cap.


Figure 9.3.17a. Stress-strain curves for $\alpha=90^{\circ}$ Torsion shear tests indicating the onset of Shear band development (Tests 41, 18, 19*, 42*and 20*).


Figure 9.3.17b. Stress-strain curves for $\alpha=90^{\circ}$ Torsion shear tests indicating the onset of Shear band development (Tests 43, 21*, 22* and 44).

### 9.4 Discussion of Shear Band Angles

Different shear band patterns and inclination angles have been presented for the various tests done as part of this experimental program. Triaxial and true triaxial shear bands developed well into the softening regime of the stress-strain behavior of the specimens. In triaxial tests, where $b=1$, shear bands developed right after peak failure showing a very quick drop in strength once developed. The shear band inclination angles for triaxial and true triaxial tests followed the Arthur shear band inclination angle prediction in most cases. Torsion shear tests had more variation in the shear band inclination angle, depending on alpha and b-values. In most cases, shear bands developed and were first seen at or near peak
failure. For different alpha values, different patterns of shear band inclinations were seen in torsion shear tests. Although somewhat scattered, $\alpha=0^{\circ}$ showed angles between Coulomb and Arthur. Alpha $=22.5^{\circ}$ spanned from Roscoe all the way up to Coulomb as the b -value increased. For $\alpha=45^{\circ}, \alpha=67.5^{\circ}$ and $\alpha=90^{\circ}$, the shear band inclination angles were all near or above Coulomb's prediction.

Although Fine Nevada sand at the same relative density was tested for all tests in different apparatuses (true triaxial and torsion shear), shear bands at $\alpha=0^{\circ}$ and $\alpha=90^{\circ}$ were on average 7 degrees higher in torsion shear tests than in true triaxial. Shear bands were free to develop for the triaxial and true triaxial tests, with a $\mathrm{H} / \mathrm{D}$ ratio was 2.5 . Taking the average diameter of the torsion shear specimen, the H/D ratio was 2. Perhaps a greater H/D ratio allowed the shear bands to develop more freely and at steeper angles. Also, the thickness of the hollow torsion shear specimens was only 2 cm . For true triaxial tests, it was 7.6 cm . It is speculated that specimen geometry may affect the development of shear bands. Additionally, the conditions of torsion shear and true triaxial tests were not exactly the same. Torsion shear tests were performed with constant mean stress, and true triaxial tests had constant minor principal stress.
9.5 Conditions where the intermediate stress is not the radial stress

As previously stated in Section 6.7, for torsion shear tests, the radial stress, $\sigma_{\mathrm{r}}$, is assumed always to be equal to the intermediate principal stress, $\sigma_{2}$. For most cases in the
experimental program, this was the case. However, in the original tests (torsion shear tests marked with $\mathrm{a}^{*}$ ), the uplift correction was not initially correctly accounted for. Therefore, what was planned to be $\mathrm{a} b=1.0$ test where $\sigma_{2}$ was supposed to be the same as $\sigma_{1}$, the actual uplift pressure that had not been accounted for, created a condition where the radial stress, $\sigma_{\mathrm{r}}$, was greater than the largest normal stress in the wall. Since the major principal stress is always the biggest, when this condition occurred, the radial stress (traditionally $\sigma_{2}$ ) became $\sigma_{1}$. This difference is important because although failure occurred in the $\mathrm{z}-\theta$ plane, the friction angle is always calculated from the stress ratio, $\sigma_{1} / \sigma_{3}$. What was determined to be $\sigma_{1}$ will affect the friction angle.

In order to mathematically prove that there is a case where typically the radial stress can be bigger than $\sigma_{1}$ in the wall, the following equations were derived. It is important to note that these equations are based on the equations used in order to maintain constant b-value, alpha and mean normal stress. These were the conditions of the torsion shear experimental program prepared for this thesis.

Rearranging the intermediate stress ratio where $\mathrm{b}=1$ and setting it to an inequality yields,
$\left(\sigma_{2}-\sigma_{3}\right)>\left(\sigma_{1}-\sigma_{3}\right)$
$\sigma_{2}>\sigma_{1}$

In order to see if $\sigma_{2}$ can be in fact be greater than $\sigma_{1}$, it is necessary to use the equations provided in Chapter 6. These derived expressions are in terms of force, inner and outer pressure, inner and outer radii, and moment. Therefore,

$$
\frac{p_{o} r_{o}+p_{i} r_{i}}{r_{o}+r_{i}}>\frac{F_{v}}{2 \pi\left(r_{o}^{2}-r_{i}^{2}\right)}+\frac{2 p_{o} r_{o}^{2}-2 p_{i} r_{i}^{2}+r_{i} r_{o}\left(p_{i}-p_{o}\right)}{2\left(r_{o}^{2}-r_{i}^{2}\right)}+\sqrt{\left(\frac{F_{v}+\pi r_{i} r_{o}\left(p_{i}-p_{o}\right)}{2 \pi\left(r_{o}^{2}-r_{i}^{2}\right)}\right)^{2}+\left(\frac{3 M}{2 \pi\left(r_{o}^{3}-r_{i}^{3}\right)}\right)^{2}}
$$

Eq. 9.5.2

Through some simplification and solving for force, this equation becomes:
$F_{v}<\pi r_{i} r_{o}\left(p_{i}-p_{o}\right)-\frac{\left(\frac{3 M\left(r_{o}^{2}-r_{i}^{2}\right)}{2 \pi\left(r_{o}^{3}-r_{i}^{3}\right)}\right)^{2}}{8 \pi r_{i} r_{o}\left(p_{i}-p_{o}\right)}$
Eq. 9.5.3

In the case where the applied vertical force is less that the calculated amount with respect to inner and outer pressures, inner and outer radii and moment, the radial stress will become the major principal stress.

### 9.6 Effects of Radial Stress on Cross-Anisotropic Sand at high b-values

As per the conditions described in Section 9.5, for a given inner and outer pressure and moment, a certain force is needed to keep the balance where the radial and vertical stresses are equal to each other at $b=1$. In other words, this force is needed to sustain the assumed relationship where $\left(\sigma_{1}>\sigma_{2}=\sigma_{r}>\sigma_{3}\right)$. If the vertical force needed for these conditions is less than the vertical force applied, then $\sigma_{2}=\sigma_{r}>\sigma_{1}$ and this may cause out of plane shear bands to occur in the r- $\theta$ direction where usually the shear occurs in the $z-\theta$ direction. Due to
cross-anisotropy of the specimen, under these conditions where b-values are close to unity, shear bands were seen to develop in the r- $\theta$ plane. Due to the cross-anisotropic behavior of the specimen with all horizontal directions being weaker, the specimen shear bands may develop as seen in Figure 9.6.1. This condition may develop even if $\sigma_{l \text { wall }}>\sigma_{\text {radial }}$ due to anisotropy because $\sigma_{\mathrm{r}}$ is sufficiently large to cause failure in the horizontal r-direction before failure in the wall-plane.


Figure 9.6.1. Schematic of Shear Band Development for high b-value tests where failure occurs in the radial direction.

In tests with $b$-values approaching unity, the radial stress gets closer and closer to the major principal stress (since at $\mathrm{b}=1, \sigma_{\mathrm{r}}=\sigma_{2}=\sigma_{1}$ ). With a cross-anisotropic material, the material will be strongest in the plane normal to the direction of deposition (the vertical plane). Therefore, if the grains are aligned in a manner where the radial stress causes the most strain, as compared to the circumferential stress causing circumferential strain, the specimen will fail in the radial direction. This was seen in some torsion shear tests. Figures 9.6.2 through 9.6.5 show sample test pictures of tests in which this shear banding pattern occurred. Table 9.3.2 lists all the tests that had this condition occur. As can be seen in the table, in all but two cases (where $\alpha=67.5^{\circ} \mathrm{b}=0.55$ and $\alpha=90^{\circ}, b=0.54$ ), the $r-\theta$ shear bands occurred at $b$-values between 0.75 and 1.0. It is interesting to note that sometimes, more than one shear band developed. Where the shear band was very large and pronounced, usually only one large shear band in the radial direction occurred, affecting a large surface of the specimen. However, in some cases, where shear bands might have been developing before the radial failure, both types are seen, as in Test 20*.


Outside and Inside of Specimen - Test $26\left(\alpha=0^{\circ}, b=1\right)$

Figure 9.6.2. Pictures of tests with Shear bands occurring in the $\mathrm{r}-\theta$ plane for $\alpha=0^{\circ}$ Torsion Shear Tests.


Test $29\left(\alpha=22.8^{\circ}, \mathrm{b}=0.75\right)-$ Front View


Test $29\left(\alpha=22.8^{\circ}, \mathrm{b}=0.75\right)-$ Zoomed in Lower Half


Test $29\left(\alpha=22.8^{\circ}, \mathrm{b}=0.75\right)$ - Inside w/o Membrane

Figure 9.6.3. Pictures of tests with Shear bands occurring in the r- $\theta$ plane for $\alpha=22.5^{\circ}$ Torsion Shear Tests.


Test $17\left(\alpha=68.3^{\circ}, b=1\right)$ (Front View)


Test $17\left(\alpha=68.3^{\circ}, \mathrm{b}=1\right)$
(Right Side View)


Test 17 ( $\alpha=68.3^{\circ}$, $b=1)($ Back View)


Test $17\left(\alpha=68.3^{\circ}, b=1\right)$ (Left Side View)


Test $17\left(\alpha=68.3^{\circ}, \mathrm{b}=1\right)$ (Inside View)

Figure 9.6.4. Pictures of tests with Shear bands occurring in the r- $\theta$ plane for $\alpha=67.5^{\circ}$ Torsion Shear Tests.


Test 20* $\left(\alpha=90^{\circ}, \mathrm{b}=0.54\right)$ Front View


Test 20* $\left(\alpha=90^{\circ}, b=0.54\right)$ Front View


Test 20* $\left(\alpha=90^{\circ}, \mathrm{b}=0.54\right)$
Back View


Test 20* ( $\alpha=90^{\circ}, \mathrm{b}=0.54$ )
Inside view w/o membrane membrane

Figure 9.6.5. Pictures of tests with Shear bands occurring in the $\mathrm{r}-\theta$ plane for $\alpha=90^{\circ}$ Torsion Shear Tests

## 10. Failure Criterion for Cross-Anisotropic Sand Deposits

### 10.1 Failure Criterion and Parameter Determination

Lade $(2007,2008)$ presented a general 3D failure criterion for cross-anisotropic soils (Eq. 10.1).

$$
\begin{equation*}
f=\left(\frac{I_{1}^{3}}{I_{3}}-27\right)\left(\frac{I_{1}}{p_{a}}\right)^{m}=\eta_{0}\left[1+\Omega_{1}\left(1-3 l_{2}^{2}\right)\right] \tag{Eq. 10.1}
\end{equation*}
$$

where $I_{1}$ and $I_{3}$ are the first and third stress invariants, respectively and $p_{a}$ is atmospheric pressure, $m$ indicates the curvature of the failure surface in the meridian planes $\eta_{0}$ is the average value of the opening angle at the stress origin and $\Omega_{1}$ describes the variation of this opening angle (Lade 2008). $1_{2}$ is the loading vector calculated by
$l_{2}=\sqrt{\frac{\left(\sigma_{y}^{2} \sin ^{2} \beta+\sigma_{z}^{2} \cos ^{2} \beta\right)}{\sigma_{x}^{2}+\sigma_{y}^{2}+\sigma_{z}^{2}}}$
Eq. 10.2
where $\sigma_{x}, \sigma_{y}$, and $\sigma_{z}$ are the principal stresses.

The loading vector can also be written as a function of stress ratios for true triaxial tests where

For Sector I:
$l_{2}^{2}=\frac{R^{2}}{R^{2}\left(1+b^{2}\right)+2 R\left(b-b^{2}\right)+\left(2-2 b+b^{2}\right)}$
Eq. 10.3

For Sector II:
$l_{2}^{2}=\frac{b^{2} R^{2}+2 R\left(b-b^{2}\right)+\left(1-2 b+b^{2}\right)}{R^{2}\left(1+b^{2}\right)+2 R\left(b-b^{2}\right)+\left(2-2 b+b^{2}\right)}$

And for Sector III:
$l_{2}^{2}=\frac{1}{R^{2}\left(1+b^{2}\right)+2 R\left(b-b^{2}\right)+\left(2-2 b+b^{2}\right)}$
Eq. 10.5
where $R$ is the stress ratio and $b$ is the $b$-value for each test. This criterion is based on $a$ function of stress, which has been previously used in developing a 3D failure criterion for isotropic soils. For a given situation, this function is set to a constant scalar value. This criterion may potentially be used for true triaxial tests and torsion shear tests. Three triaxial compression tests with horizontal bedding planes, and three triaxial compression tests with vertical beddings planes are needed to determine the parameters $\eta_{0}, \mathrm{~m}, \Omega_{1}$. The tests described in Chapter 3 provide this information and the determined parameters will be determined in the following section. The loading directions are related to the principal directions of the cross-anisotropic microstructure of the soil by $1_{2}$ at the right hand side (Eq. 10.1). The expression for $l_{2}{ }^{2}$ can be written in terms of $b$-values and stress ratios for all three sectors of the octahedral plane as given in Eq. 10.3 through 10.5.

As described in Lade (2007), the first step in determining the value of m , is to plot $\left(\mathrm{I}_{1}{ }^{3} / \mathrm{I}_{3}-27\right)$ versus $\left(p_{a} / I_{1}\right)$ on $\log -\log$ scales. Then, the best fitting line can be drawn and $m$ can be calculated as the geometric slope of the best fitting line. The intercept of the line with $\mathrm{p}_{\mathrm{a}} / \mathrm{I}_{1}=1$ is $\eta_{1 v}$. It is assumed that the curvature for both the horizontal and the vertical specimens are the same and therefore, they both have the same m parameter. Therefore, a parallel line can be drawn through a point for the Sector III tests and $\eta_{1 \mathrm{~h}}$ can be determined. Table 10.1.1 summarizes the data used from Tests 1-8 (presented in Section 3.3). Figure 10.1.1 shows the points plotted in the log-log diagram. Torsion shear results are also presented in this figure to show a comparison between the two sets of tests. Torsion shear tests were all performed at $\mathrm{p}_{\mathrm{a}} / \mathrm{I}_{1}=0.33$. As can be seen, the torsion shear results are below the true triaxial tests. Since all torsion shear tests had constant mean confining stress, the value of $m$ determined from the true triaxial tests was used for the torsion shear tests.

Table 10.1.1. Data used in determination of parameter, m from Tests 1-8.

| Test No. | $\boldsymbol{\alpha}$ | $\boldsymbol{\sigma}_{\mathbf{1}}$ | $\boldsymbol{\sigma}_{\mathbf{3}}$ | $\mathbf{I}_{\mathbf{1}}$ | $\mathbf{I}_{\mathbf{3}}$ | $\left(\mathbf{I}_{\mathbf{1}} \mathbf{}^{\mathbf{3}} \mathbf{I}_{\mathbf{3}} \mathbf{- 2 7}\right.$ | $\mathbf{p}_{\mathbf{a}} / \mathbf{I}_{\mathbf{1}}$ |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| $1, \mathrm{TT} \# 1$ | 0 | 264.21 | 50.00 | 364.21 | 660525.00 | 46.14 | 0.28 |
| 2 | 0 | 157.18 | 25.00 | 207.18 | 98237.50 | 63.52 | 0.49 |
| 3 | 0 | 593.90 | 130.00 | 853.90 | 10036910.00 | 35.03 | 0.12 |
| 4 | 0 | 339.07 | 70.00 | 479.07 | 1661426.24 | 39.18 | 0.21 |
| 5 | 90 | 111.88 | 25.00 | 161.82 | 69886.18 | 33.63 | 0.62 |
| 6 | 90 | 317.88 | 75.00 | 467.89 | 1788126.68 | 30.28 | 0.22 |
| 7 | 90 | 521.37 | 130.00 | 781.37 | 8811153.00 | 27.14 | 0.13 |
| $8(\mathrm{TT} \# 13)$ | 90 | 206.51 | 50.00 | 306.51 | 516282.38 | 28.78 | 0.33 |



Figure 10.1.1. Log-log plot to determine parameters, $m, \eta_{1 v}$ and $\eta_{1 \mathrm{~h}}$.

In order to obtain $\eta_{0}$ and $\Omega_{1}$ corresponding to the range in which the stresses in the experimental program were performed, parameters were determined at $\mathrm{I}_{1} / \mathrm{p}_{\mathrm{a}}=2.97$ rather than $\mathrm{I}_{1} / \mathrm{p}_{\mathrm{a}}=1.0$. In order to find $\eta_{0}$ and $\Omega_{1}, \eta_{1 \mathrm{~h}}, \eta_{\mathrm{h}}, \eta_{1 \mathrm{v}}$ and $\eta_{\mathrm{v}}$ need to be found. These numbers are shown on Figure 10.1.1. By solving two linear equations based on the right hand side of Eq. 10.1 and knowing the $1_{2}$, which is determined from Eq. 10.3 and 10.5, the parameters can be determined. To solve for $l_{2}$ a cubical equation based on the left hand side of Eq. 10.1 is used to determine the stress ratio. This expression is written as follows:
$\frac{I_{1}^{3}}{I_{3}}-27=\frac{\left(\sigma_{1}+2 \sigma_{3}\right)^{3}}{\sigma_{1} \cdot \sigma_{3}^{2}}-27=\frac{(R+2)^{3}}{R}-27=\eta$
Eq. 10.6
where $I_{1}$ and $I_{3}$ are the first and third stress invariants, respectively. $\sigma_{1}, \sigma_{2}$ and $\sigma_{3}$ are principal stresses and R is the stress ratio. The values $\mathrm{R}_{\mathrm{v}}$ and $\mathrm{R}_{\mathrm{h}}$ are obtained from Eq. 10.6 and then substituted into Eq. 10.3 and 10.5. With two equations and two unknowns, the equations can be solved simultaneously so that $\eta_{0}$ and $\Omega_{1}$ can be determined. $\eta_{0}$ and $\Omega_{1}$ are found to be 37.229 and -0.196 , respectively. At $\mathrm{I}_{1}=300 \mathrm{kPa}$ there is a difference of $5.3^{\circ}$ in triaxial compression tests on specimens with $\alpha=0^{\circ}$ and $\alpha=90^{\circ}$. Lade and Wasif (1988) observed a $5.5^{\circ}$ difference in tests with Cambria sand for $\alpha=0^{\circ}$ and $\alpha=90^{\circ}$.

### 10.2 Pietrusczack Model based on Torsion Shear Results

Lade's model, which was used in section 10.1, was based on the developments by Pietruszczak and Mroz (2000, 2001). This model was recently expanded by Pietruszack (2011) by using higher order terms and multipliers in the equation. The approach defined the failure criterion in terms of traction components that acted on a physical plane. A spatial distribution of strength parameters and the direction of the physical plane can be solved for by using a constrained optimization technique. This involves searching for the orientation that maximizes the value of the failure function being used. His equations have been slightly modified in order to be consistent with Lade (2007) (see Equation 10.6).

$$
\eta_{f}=\eta_{0}\left[1+\Omega_{1}\left(1-3 l_{2}^{2}\right)+a_{1}\left(\Omega_{1}\left(1-3 l_{2}^{2}\right)\right)^{2}+a_{2}\left(\Omega_{1}\left(1-3 l_{2}^{2}\right)\right)^{3}+a_{3}\left(\Omega_{1}\left(1-3 l_{2}^{2}\right)\right)^{4}+\ldots\right] \quad \text { Eq. } 10.6
$$

In order to solve for the unknowns, $\eta_{0}, \Omega_{1}, a_{1}, a_{2}$, and $\mathrm{a}_{3}$ (for a fourth order equation), 25 torsion shear test results were used. These parameters were determined using the Least Squares Method (Chapra and Canale 2010) and were performed for different orders of Eq. 11.6. The results are presented in Table 11.2.1. Depending on which order equation is desired, the parameters will change. It is important to do an independent Least Squares based Polynomial Regression (see Appendix K) for each desired order to get the correct parameters. The only parameter needed from the triaxial test results (Figure 10.1.1) is the geometric slope, $m$. The other terms are determined when solving the optimization problem.

The $\eta_{\mathrm{f}}$ values that are calculated are shown in Table 10.2.2. As can be seen, there is not much difference in the predicted $\eta_{f}$ value when looking at the third and fourth order iteration.

Table 10.2.1. Determined Parameters using 25 Torsion Shear Test results based on Pietrusczack (2011) model.

| $\mathbf{4}^{\text {th }}$ Order Equation |  |
| :--- | :--- | :--- |
| $\eta_{\mathrm{o}}$ | 30.131 |
| $\Omega_{1}$ | -1.080 |
| $\mathrm{a}_{1}$ | 0.186 |
| $\mathrm{a}_{2}$ | -0.680 |
| $\mathrm{a}_{3}$ | 0.196 |\(\left|\begin{array}{ll}\mathbf{2}^{th} \& Order Equation <br>

\eta_{\mathrm{o}} \& 34.792 <br>
\mathbf{3}^{th} \& Order Equation <br>
\Omega_{1} \& -0.327 <br>
\mathrm{n}_{\mathrm{n}}= \& -0.171 <br>
\Omega_{1}= \& -0.827 <br>
\mathrm{a}_{1} \& 0.703 <br>

a_{2} \& -0.846\end{array}\right|\)| $\mathbf{1}^{\text {th }}$ | Order Equation |
| :--- | :--- |
| $\eta_{\mathrm{o}}$ | 34.368 |
| $\Omega_{1}$ | -0.321 |

Table 10.2.2. Calculated $\eta_{f}$ values for different order equations using 25 Torsion Shear Test results based on Pietrusczack (2011) model.

| Order of Equation |  | $\mathbf{0}$ | $\mathbf{1}$ | $\mathbf{2}$ | $\mathbf{3}$ | $\mathbf{4}$ |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| Test | $\mathbf{b}$ - | alpha | $\boldsymbol{\eta} \mathbf{f}$ | $\boldsymbol{\eta} \mathbf{f}$ | $\boldsymbol{\eta} \mathbf{f}$ | $\boldsymbol{\eta}_{\mathbf{f}}$ | $\boldsymbol{\eta} \mathbf{f}$ |
| 23 | 0.00 | 0.00 | 35.789 | 53.838 | 52.878 | 38.505 | 43.782 |
| $24^{*}$ | 0.27 | 0.00 | 35.789 | 51.362 | 50.797 | 48.365 | 46.539 |
| $25^{*}$ | 0.55 | 0.00 | 35.789 | 47.512 | 47.436 | 53.975 | 50.669 |
| 2 | 0.75 | 0.00 | 35.789 | 43.971 | 44.207 | 51.368 | 50.985 |
| 26 | 1.00 | 0.00 | 35.789 | 39.852 | 40.287 | 42.730 | 45.243 |
| 3 | 0.00 | 22.41 | 35.789 | 49.332 | 49.044 | 52.624 | 49.002 |
| 28 | 0.23 | 23.69 | 35.789 | 47.093 | 47.061 | 53.997 | 50.932 |
| 6 | 0.50 | 22.48 | 35.789 | 44.134 | 44.358 | 51.610 | 51.074 |
| 29 | 0.75 | 22.21 | 35.789 | 40.558 | 40.971 | 44.466 | 46.684 |
| 8 | 0.99 | 22.47 | 35.789 | 37.421 | 37.890 | 36.469 | 39.138 |
| 31 | 0.02 | 44.71 | 35.789 | 38.950 | 39.404 | 40.435 | 43.163 |
| 9 | 0.25 | 44.98 | 35.789 | 37.528 | 37.997 | 36.746 | 39.437 |
| 33 | 0.50 | 44.99 | 35.789 | 35.796 | 36.253 | 32.358 | 34.399 |
| 11 | 0.75 | 44.98 | 35.789 | 33.591 | 33.988 | 27.428 | 27.880 |
| 34 | 1.00 | 44.95 | 35.789 | 31.924 | 32.242 | 24.560 | 23.542 |
| 14 | 0.00 | 67.33 | 35.789 | 29.457 | 29.604 | 22.346 | 19.534 |
| 36 | 0.25 | 67.80 | 35.789 | 28.486 | 28.550 | 22.315 | 19.203 |
| 15 | 0.50 | 67.47 | 35.789 | 27.719 | 27.709 | 22.685 | 19.609 |
| 38 | 0.75 | 67.42 | 35.789 | 27.597 | 27.574 | 22.778 | 19.734 |
| 17 | 1.00 | 68.21 | 35.789 | 26.325 | 26.165 | 24.344 | 22.162 |
| 41 | 0.00 | 90.00 | 35.789 | 25.685 | 25.450 | 25.569 | 24.247 |
| $42^{*}$ | 0.32 | 90.00 | 35.789 | 24.129 | 23.693 | 29.896 | 32.174 |
| $20^{*}$ | 0.54 | 90.00 | 35.789 | 24.010 | 23.557 | 30.313 | 32.970 |
| $43^{*}$ | 0.78 | 90.00 | 35.789 | 24.174 | 23.744 | 29.743 | 31.882 |
| $22^{*}$ | 0.99 | 90.00 | 35.789 | 24.315 | 23.903 | 29.276 | 30.999 |

In order to compare the predictions to the attained experimental results, friction angles were calculated by using Eq. 10.1 and setting the calculated $\eta_{\mathrm{f}}=\mathrm{f}$ in the equation. This equation can be solved in terms of $b$-value and the various $\eta_{0}$ constants in the following manner:

$$
\left.\begin{array}{l}
f=\left(\frac{I_{1}^{3}}{I_{3}}-27\right)\left(\frac{I_{1}}{p_{a}}\right)^{m}=\eta_{f} \\
\left(\frac{I_{1}^{3}}{I_{3}}-27\right)=\frac{\eta_{f}}{\left(\frac{I_{1}}{p_{a}}\right)^{m}} \\
\frac{I_{1}^{3}}{I_{3}}=\frac{\eta_{f}}{\left(\frac{I_{1}}{p_{a}}\right)^{m}}+27=K
\end{array}\right\}
$$

Eq. 10.7

This can be written in terms of stresses by:
$\left(\sigma_{1}+\sigma_{2}+\sigma_{3}\right)^{3}=K\left(\sigma_{1} * \sigma_{2} * \sigma_{3}\right)=K\left(\sigma_{1} *\left(b\left(\sigma_{1}-\sigma_{3}\right)+\sigma_{3}\right) * \sigma_{3}\right)$
where

$$
b=\frac{\left(\sigma_{2}-\sigma_{3}\right)}{\left(\sigma_{1}-\sigma_{3}\right)}
$$

The stress ratio can then be determined by:

$$
\left.\begin{array}{l}
\frac{\sigma_{1}}{\sigma_{3}}=\frac{\left(\sigma_{1}+\sigma_{2}+\sigma_{3}\right)^{3}}{\sigma_{3}^{2}\left(b\left(\sigma_{1}-\sigma_{3}\right)+\sigma_{3}\right)^{*} K} \\
\frac{\sigma_{1}}{\sigma_{3}}=\frac{\left[\sigma_{1}+\left(b\left(\sigma_{1}-\sigma_{3}\right)+\sigma_{3}\right)+\sigma_{3}\right]^{3}}{\sigma_{3}^{2}\left(b\left(\sigma_{1}-\sigma_{3}\right)+\sigma_{3}\right)^{*}} \\
\frac{\sigma_{1}}{\sigma_{3}}=\frac{\left[\frac{\sigma_{1}}{\sigma_{3}}+\left(b\left(\frac{\sigma_{1}}{\sigma_{3}}-1\right)+1\right)+1\right]^{3}}{\left(b\left(\frac{\sigma_{1}}{\sigma_{3}}-1\right)+1\right) * K}
\end{array}\right\}
$$

This equation can be solved by solving for the cubical equation
$0=\left(\frac{\sigma_{1}}{\sigma_{3}}+b\left(\frac{\sigma_{1}}{\sigma_{3}}-1\right)+2\right)^{3}-K\left(\frac{\sigma_{1}}{\sigma_{3}}\right)\left(b\left(\frac{\sigma_{1}}{\sigma_{3}}-1\right)+1\right)$

With the procedure described above, the following results were attained from the $\eta_{f}$ values calculated. In the $\eta_{\mathrm{f}}$ values presented above, the second order function is far away from the calculated $\eta_{\mathrm{f}}$ values using Lade's isotropic criterion. The third and fourth order functions are almost exactly the same. It is possible to say that a third order function provides a sufficient simulation. The calculated friction angles for the corresponding order equations and experimental results are presented in Table 10.2.3 and Figures 10.2.1 through 10.2.5. A 3D graph of the experimental and the predicted friction angles is presented in Figure 10.2.6.

Table 10.2.3. Calculated $\phi$ values for different order equations using 25 Torsion Shear Test results based on Pietrusczack (2011) model.

| Test <br> No. | b- <br> value | alpha | $\phi$ <br> $\left(\mathbf{0}^{\text {th }} \mathbf{O r d e r}\right)$ | $\phi$ <br> $\left(\mathbf{1}^{\text {th }} \mathbf{O r d e r}\right)$ | $\left.\boldsymbol{\phi} \mathbf{2}^{\text {th }} \mathbf{O r d e r}\right)$ | $\phi$ <br> $\left(\mathbf{3}^{\text {th }} \mathbf{O r d e r}\right)$ | $\phi$ <br> (Results) |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 23 | 0.00 | 0.00 | 37.430 | 42.092 | 41.889 | 38.269 | 39.739 |
| $24^{*}$ | 0.27 | 0.00 | 44.366 | 49.473 | 49.318 | 48.628 | 48.086 |
| $25^{*}$ | 0.55 | 0.00 | 45.661 | 49.698 | 49.675 | 51.490 | 50.605 |
| 2 | 0.75 | 0.00 | 44.810 | 47.764 | 47.841 | 49.977 | 49.871 |
| 26 | 1.00 | 0.00 | 42.972 | 44.534 | 44.692 | 45.548 | 46.377 |
| 3 | 0.00 | 22.41 | 37.430 | 41.101 | 41.034 | 41.834 | 41.024 |
| 28 | 0.23 | 23.69 | 43.777 | 47.636 | 47.626 | 49.546 | 48.732 |
| 6 | 0.50 | 22.48 | 45.699 | 48.691 | 48.763 | 50.901 | 50.754 |
| 29 | 0.75 | 22.21 | 44.785 | 46.582 | 46.728 | 47.900 | 48.595 |
| 8 | 0.99 | 22.47 | 43.030 | 43.677 | 43.858 | 43.303 | 44.329 |
| 31 | 0.02 | 44.71 | 37.984 | 38.979 | 39.115 | 39.419 | 40.186 |
| 9 | 0.25 | 44.98 | 43.990 | 44.660 | 44.835 | 44.363 | 45.360 |
| 33 | 0.50 | 44.99 | 45.699 | 45.702 | 45.883 | 44.256 | 45.131 |
| 11 | 0.75 | 44.98 | 44.828 | 43.918 | 44.086 | 41.017 | 41.250 |
| 34 | 1.00 | 44.95 | 42.946 | 41.291 | 41.435 | 37.545 | 36.950 |
| 14 | 0.00 | 67.33 | 37.430 | 35.201 | 35.258 | 32.079 | 30.587 |
| 36 | 0.25 | 67.80 | 43.970 | 40.764 | 40.795 | 37.392 | 35.366 |
| 15 | 0.50 | 67.47 | 45.699 | 42.045 | 42.039 | 39.208 | 37.180 |
| 38 | 0.75 | 67.42 | 44.825 | 41.101 | 41.089 | 38.389 | 36.399 |
| 17 | 1.00 | 68.21 | 42.932 | 38.512 | 38.426 | 37.406 | 36.093 |
| 41 | 0.00 | 90.00 | 37.430 | 33.645 | 33.541 | 33.594 | 32.994 |
| $42^{*}$ | 0.32 | 90.00 | 44.954 | 39.366 | 39.110 | 42.390 | 43.435 |
| $20^{*}$ | 0.54 | 90.00 | 45.668 | 39.976 | 39.708 | 43.290 | 44.493 |
| $43^{*}$ | 0.78 | 90.00 | 44.647 | 39.045 | 38.792 | 41.990 | 42.986 |
| $22^{*}$ | 0.99 | 90.00 | 42.988 | 37.445 | 37.205 | 40.086 | 40.909 |



Figure 10.2.1. Predicted friction angle results for different order equations for torsion shear tests at $\mathrm{b}=0$.


Figure 10.2.2. Predicted friction angle results for different order equations for torsion shear tests at $\mathrm{b}=0.25$.


Figure 10.2.3. Predicted friction angle results for different order equations for torsion shear tests at $\mathrm{b}=0.50$.


Figure 10.2.4. Predicted friction angle results for different order equations for torsion shear tests at $\mathrm{b}=0.75$.


Figure 10.2.5. Predicted friction angle results for different order equations for torsion shear tests at $\mathrm{b}=1.0$.


Figure 10.2.6. Failure Surface for Fine Nevada Sand using third order equation with 25 Torsion Shear Data Points.

### 10.3 Pietrusczack Model based on Modified Torsion Shear Results

Section 10.2 presented the parameters and friction angles that were calculated by using 25 torsion shear points regardless of whether certain friction angles seemed somewhat out of the pattern that was expected. In order to minimize the skewing of parameters by including outlier data, three different versions of data in the paragraphs that follow. The first only uses a total of 20 points were used when calculating the parameters with the regression model. Test $24^{*}\left(\alpha=0^{\circ}, b=0.27\right)$ was deleted from the data because it was obviously low in the series of $\alpha=0^{\circ}$ tests. Tests $38^{*}\left(\alpha=67.42^{\circ}, b=0.75\right), 17\left(\alpha=68.21^{\circ}, b=1.0\right), 43^{*}\left(\alpha=90^{\circ}, b=0.78\right)$ and
$22^{*}\left(\alpha=90^{\circ}, b=0.99\right)$ were also deleted because of the effects discussed in Chapter 7 concerning soft boundary effects in that combination of stress path. Using the 20 other points (shown in Table 10.2.3), new parameters were determined so that $\eta_{f}$ values and friction angles could be calculated. These new parameters were applied to all 25 data points when determining $\eta_{\mathrm{f}}$ values and friction angles. Only the third order results are shown in Table 10.3.1 because the third order equation was found to be sufficient and showed only little difference with the fourth order equation in modeling the experimental 3D failure surface. The experimental friction angles are also presented for comparison. A 3D plot using these parameters in the third order equation is presented in Figure 10.3.1.

Table 10.3.1. Parameter determination and calculated $\eta_{\mathrm{f}}$ and $\phi$ values for third order equation using 20 Torsion Shear Test results based on Pietrusczack (2011) model.

| $4^{\text {th }}$ Order Equation |  |
| :---: | :---: |
| ๆо | 29.715 |
| $\Omega$ | -1.043 |
| a1 | 0.307 |
| a2 | -0.771 |
| a3 | 0.217 |
| $3^{\text {th }}$ Order Equation |  |
| ๆо | 29.151 |
| $\Omega$ | -0.768 |
| a1 | 0.934 |
| a2 | -1.046 |
| $2{ }^{\text {th }}$ Order Equation |  |
| по | 34.975 |
| $\Omega$ | -0.317 |
| a1 | 0.217 |
| $1^{\text {th }}$ Order Equation |  |
| ๆо | 35.379 |
| $\Omega$ | -0.326 |
| $0^{\text {th }}$ Order Equation |  |
| $\eta \mathrm{O}=$ | 38.186 |


| Test | b- | alpha | $\boldsymbol{\eta f}$ | $\phi$ | $\phi$ |
| :---: | :---: | :---: | :---: | :---: | :---: |
| 23 | 0.00 | 0.00 | 42.703 | 39.454 | 41.153 |
| $24^{*}$ | 0.27 | 0.00 | 51.230 | 49.437 | 45.842 |
| $25^{*}$ | 0.55 | 0.00 | 103.079 | 55.257 | 51.817 |
| 2 | 0.75 | 0.00 | 51.702 | 50.069 | 56.964 |
| 26 | 1.00 | 0.00 | 42.557 | 45.489 | 53.180 |
| 3 | 0.00 | 22.41 | 54.582 | 42.247 | 39.708 |
| 28 | 0.23 | 23.69 | 55.142 | 49.837 | 43.346 |
| 6 | 0.50 | 22.48 | 51.977 | 51.000 | 46.031 |
| 29 | 0.75 | 22.21 | 41.518 | 44.335 | 47.858 |
| 8 | 0.99 | 22.47 | 36.287 | 43.230 | 42.329 |
| 31 | 0.02 | 44.71 | 40.233 | 39.360 | 35.945 |
| 9 | 0.25 | 44.98 | 36.560 | 44.291 | 38.882 |
| 33 | 0.50 | 44.99 | 32.289 | 44.225 | 45.150 |
| 11 | 0.75 | 44.98 | 34.543 | 27.658 | 41.136 |
| 34 | 1.00 | 44.95 | 25.128 | 37.868 | 35.087 |
| 14 | 0.00 | 67.33 | 23.586 | 32.684 | 34.660 |
| 36 | 0.25 | 67.80 | 23.875 | 38.317 | 37.483 |
| 15 | 0.50 | 67.47 | 36.182 | 24.519 | 40.303 |
| 38 | 0.75 | 67.42 | 24.657 | 39.503 | 31.873 |
| 17 | 1.00 | 68.21 | 26.725 | 38.727 | 38.224 |
| 41 | 0.00 | 90.00 | 40.934 | 28.223 | 34.714 |
| $42^{*}$ | 0.32 | 90.00 | 33.266 | 43.911 | 45.042 |
| $20^{*}$ | 0.54 | 90.00 | 33.740 | 44.824 | 45.237 |
| $43^{*}$ | 0.78 | 90.00 | 33.090 | 43.520 | 40.887 |
| $22^{*}$ | 0.99 | 90.00 | 32.556 | 41.616 | 37.215 |



Figure 10.3.1. Failure Surface for Fine Nevada Sand using third order equation with 20 Torsion Shear Data Points.

In a similar fashion, the same iteration using the data presented above was done again. However, the second time, all tests with b-values equal to 0 were given doubled weights in the data used in the regression. This was done in order to add a weighting factor to these tests. These tests are known to be reliable since they were confirmed by both the true triaxial apparatus and the torsion shear apparatus (results were confirmed in Chapter 7). As can be seen by the data, certain friction angles do change based on the new set of parameters. Table 10.3.2 provides a summary of the parameters, $\eta_{\mathrm{f}}$ values and friction angles compared to the experimental results and Figure 10.3.2 shows the 3D results.

Table 10.3.2. Parameter determination and calculated $\eta_{\mathrm{f}}$ and $\phi$ values for third order equation using weighted Torsion Shear Test results based on Pietrusczack (2011) model.

|  |  | Test | b-value | alpha | $\eta{ }^{\text {f }}$ | $\phi$ | $\phi$ |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  |  | 23 | 0.00 | 0.00 | 47.660 | 40.708 | 41.153 |
|  |  | 24* | 0.27 | 0.00 | 51.349 | 49.469 | 45.842 |
| $4^{\text {th }}$ Order |  | 25* | 0.55 | 0.00 | 103.891 | 51.391 | 50.803 |
| $\eta \mathrm{O}$ | 29.699 | 2 | 0.75 | 0.00 | 47.071 | 48.737 | 56.964 |
| $\Omega$ | -1.033 | 26 | 1.00 | 0.00 | 39.265 | 44.319 | 53.180 |
| a1 | 0.133 | 3 | 0.00 | 22.41 | 52.121 | 41.725 | 39.708 |
| a2 | -0.887 | 28 | 0.23 | 23.69 | 51.060 | 48.767 | 43.346 |
| a3 | 0.350 | 6 | 0.50 | 22.48 | 47.334 | 49.683 | 46.031 |
| $3^{\text {th }}$ Order |  | 29 | 0.75 | 22.21 | 41.388 | 40.702 | 46.633 |
| $\eta$ \% | 28.946 | 8 | 0.99 | 22.47 | 34.350 | 42.434 | 42.329 |
| $\Omega$ | -0.578 | 31 | 0.02 | 44.71 | 37.420 | 38.508 | 35.945 |
| a1 | 1.313 | 9 | 0.25 | 44.98 | 34.560 | 43.497 | 38.882 |
| a2 | -1.640 | 33 | 0.50 | 44.99 | 31.302 | 43.781 | 45.150 |
| $2{ }^{\text {th }}$ Order |  | 11 | 0.75 | 44.98 | 33.941 | 27.835 | 41.227 |
| $\eta$ | 33.197 | 34 | 1.00 | 44.95 | 25.964 | 38.331 | 35.087 |
| $\Omega$ | -0.292 | 14 | 0.00 | 67.33 | 24.824 | 33.259 | 34.660 |
| a1 | 0.446 | 36 | 0.25 | 67.80 | 25.023 | 38.964 | 37.483 |
| $1^{\text {th }}$ Order |  | 15 | 0.50 | 67.47 | 35.533 | 25.478 | 40.847 |
| ךо | 33.883 | 38 | 0.75 | 67.42 | 25.576 | 40.021 | 31.873 |
| $\Omega$ | -0.316 | 17 | 1.00 | 68.21 | 27.041 | 38.894 | 38.224 |
| $0^{\text {th }}$ Order |  | 41 | 0.00 | 90.00 | 40.788 | 28.101 | 34.665 |
| $\eta$ ך | 37.778 | 42* | 0.32 | 90.00 | 31.662 | 43.207 | 45.042 |
|  |  | 20* | 0.54 | 90.00 | 31.997 | 44.064 | 45.237 |
|  |  | 43* | 0.78 | 90.00 | 31.539 | 42.831 | 40.887 |
|  |  | 22* | 0.99 | 90.00 | 31.162 | 40.984 | 37.215 |



Figure 10.3.2. Failure Surface for Fine Nevada Sand using third order equation with weighted Torsion Shear Data Points.

Realistically, torsion shear and/or true triaxial machines are not easily available. Therefore, without having all of the data from these tests, it is quite difficult to calculate the parameters that have been shown above. However, triaxial tests are very common in most geotechnical laboratories. Therefore, if it is possible to vary the bedding plane inclination, whether when depositing a sand specimen, carving a clay specimen or taking core samples at different inclinations, then this procedure might become more feasible. Therefore, the last parameter determination was performed by only using torsion shear data with $\mathrm{b}=0$ and varying the alpha values. A total of 5 tests were used (Test 23, 3, 31, 14, and 41).

As shown above, the parameters for the different order equations is shown in Table 10.3.3. For the regression model, it is not possible to use a $4^{\text {th }}$ order equation with only 5 points, as explained in Appendix K. Therefore, parameters up until $3^{\text {rd }}$ order were calculated. However, as seen in the previous examples, the third order provides a very close fit.

Table 10.3.3. Parameter determination and calculated $\eta_{\mathrm{f}}$ and $\phi$ values for third order equation using five $\mathrm{b}=0$ Torsion Shear Test results based on Pietrusczack (2011) model.

| $3{ }^{\text {th }}$ Order Equation |  |
| :---: | :---: |
| ךо | 23.764 |
| $\Omega$ | -0.193 |
| a1 | 4.487 |
| a2 | 2.080 |
| $2^{\text {th }}$ Order Equation |  |
| ךо | 23.602 |
| $\Omega$ | -0.196 |
| a1 | 5.022 |
| $1^{\text {th }}$ Order Equation |  |
| $\eta$ | 25.612 |
| $\Omega$ | -0.404 |
| $0{ }^{\text {th }}$ Order Equation |  |
| ךо | 34.826 |


| Test | b- | alpha | $\boldsymbol{\eta} \mathbf{f}$ | $\boldsymbol{\phi}$ | $\boldsymbol{\phi}$ |
| :---: | :---: | :---: | :---: | :---: | :---: |
| 23 | 0.00 | 0.00 | 46.096 | 40.328 | 41.153 |
| $24^{*}$ | 0.27 | 0.00 | 41.485 | 46.460 | 45.842 |
| $25^{*}$ | 0.55 | 0.00 | 35.426 | 45.514 | 53.054 |
| 2 | 0.75 | 0.00 | 30.976 | 42.736 | 56.964 |
| 26 | 1.00 | 0.00 | 27.059 | 38.944 | 53.180 |
| 3 | 0.00 | 22.41 | 38.127 | 38.155 | 39.708 |
| 28 | 0.23 | 23.69 | 34.845 | 43.401 | 43.346 |
| 6 | 0.50 | 22.48 | 31.158 | 43.715 | 46.031 |
| 29 | 0.75 | 22.21 | 27.639 | 41.082 | 46.587 |
| 8 | 0.99 | 22.47 | 25.341 | 38.071 | 42.329 |
| 31 | 0.02 | 44.71 | 26.372 | 34.417 | 35.945 |
| 9 | 0.25 | 44.98 | 25.408 | 39.193 | 38.882 |
| 33 | 0.50 | 44.99 | 24.423 | 40.248 | 45.150 |
| 11 | 0.75 | 44.98 | 23.461 | 38.807 | 39.867 |
| 34 | 1.00 | 44.95 | 22.941 | 36.588 | 35.087 |
| 14 | 0.00 | 67.33 | 22.479 | 32.145 | 34.660 |
| 36 | 0.25 | 67.80 | 22.395 | 37.440 | 37.483 |
| 15 | 0.50 | 67.47 | 22.365 | 39.009 | 39.727 |
| 38 | 0.75 | 67.42 | 22.363 | 38.132 | 31.873 |
| 17 | 1.00 | 68.21 | 22.393 | 36.236 | 38.224 |
| 41 | 0.00 | 90.00 | 22.440 | 32.125 | 33.589 |
| $42^{*}$ | 0.32 | 90.00 | 22.641 | 38.477 | 45.042 |
| $20^{*}$ | 0.54 | 90.00 | 22.661 | 39.163 | 45.237 |
| $43^{*}$ | 0.78 | 90.00 | 22.633 | 38.120 | 40.887 |
| $22^{*}$ | 0.99 | 90.00 | 22.610 | 36.427 | 37.215 |



Figure 10.3.4. Failure Surface for Fine Nevada Sand using third order equation with 5 Torsion Shear Data Points.

As can be seen from Figure 10.3.4, only the low b-value are modeled correctly.

The model that produces the best comparison with the experimental results is the weighted model. As a reminder, when calculating the parameters needed, this model omits 5 outlier tests and uses double weights for the front wall test where $b=0$. A 3D plot showing this model compared to all 44 torsion shear tests results is shown in Figure 10.3.5. One without any test results, just showing the actual surface of the prediction is shown in Figure 10.3.6.

The effect of cross-anisotropy on the failure surface is clearly captured in the model. For comparison, a 3D graph with Lade's isotropic criterion has also been shown in Figure 10.3.7. This criterion is only applicable in the first and third sectors ( $\alpha=0^{\circ}$ and $\alpha=90^{\circ}$ ).


Figure 10.3.5. Failure Surface for Fine Nevada Sand using third order equation with weighted Torsion Shear Data Points and all Torsion Shear Results.


Figure 10.3.6. Failure Surface for Fine Nevada Sand using only the third order equation with weighted Torsion Shear Data Points.


Figure 10.3.7. Failure Surface for Fine Nevada Sand using Lade's Isotropic Failure Criterion with Torsion Shear Data Points.

### 10.4 Conclusion

A cross-anisotropic criterion has been established from the torsion shear test data. Using the Pietrusczack model (2011), different parameters were established. These parameters were attained by doing a polynomial regression. The corresponding $\eta$ values were calculated and set to Lade's isotropic criterion in order to determine the friction angle for each point. After several iterations, consisting of using all the points, omitting outliers, weighing the $b=0$ tests by a factor of two and finally, only using the $\mathrm{b}=0$ tests, the best approximation model was
found to be the third order weighted approximation. This model shows the surface and the drops in friction angle closest to the actual experimental points.

## 11. Conclusion

This thesis consists of the presentation and comprehensive analysis of experimental results attained from a series of systematic drained, triaxial, true triaxial and torsion shear experiments performed on Fine Nevada Sand deposited with cross-anisotropic fabric. These tests were performed in order to be able to attain the needed data and knowledge for the future development of constitutive models that predict the behavior of soil under various conditions that occur in the field, including the rotation of principal stresses.

Triaxial tests were performed as basic tests to give fundamental data about the behavior of sand. Next, true triaxial tests, using a true triaxial apparatus were performed to see the crossanisotropic behavior of the sand with three different principal stresses, allowing for the variation of $b$-value in the stress paths chosen. By using a freezing technique, specimens could be rotated so the angle between the bedding planes and the principal stress direction could be changed from 0 to 90 degrees. These tests were then compared to results attained from a new torsion shear machine under similar conditions and stress paths.

As has been presented, the data attained from the true triaixal and torsion shear tests with the use of two independent apparatuses confirm the experimental results and show the reliability of the torsion shear tests performed. The torsion shear tests, all performed with stress paths of constant b-value, mean normal stress and principal stress direction, provide a complete 3D surface for different alpha and b-values.

When looking at this surface, it shows the clear effects of cross-anisotropy on the strength of the sand. When the principal stress direction is at 67.5 degrees, the soil becomes very weak and fails at low friction angles. The soil exhibits it's highest strength at $\alpha=0^{\circ}$ and $b=0.75$. Strengths vary over the surface but as $\alpha$ increases the strength decreases as well. Strain analysis also showed similar patterns with the strain to failure becoming increasingly smaller as the alpha values increased at constant $b$-values.

Torsion shear tests showed that shear bands developed at/near failure. As the b-value increased, shear bands developed quicker and were more pronounced. For certain cases of high b-values, shear bands developed in the non-typical r- $\theta$ plane, creating great troughs in the soil. Although it is hard to point to one theory that relates to the prediction of shear bands due to a range of shear band angles measured, the Coulomb theory seems to best model shear band directions for the torsion shear tests.

Finally, an already existing cross-anisotropic failure criterion was adapted and compared to the torsion shear results presented. It was found that Pietruszczak's third order model could be used to model the behavior of Fine Nevada sand under 3D conditions. However, the model proved to predict too low values for where alpha $=0$ and $b=0.75$ and 1 when compared to actual test results. In conclusion, the aim of providing the experimental basis for future modeling of cross-anisotropy in the field, shear banding and effects of principal stress rotation on shear strength of soil on frictional materials has been provided via this research.

Appendix A - X-Ray Diffraction Results- CUA Vitreous State Laboratory



## Appendix B - Specific Gravity Test for Fine Nevada Sand

Weight of Soil, $\mathrm{W}_{\mathrm{s}}$
Weight of Bottle, $\mathrm{W}_{\mathrm{b}}$
Weight of Water, Soil, and Bottle, $\mathrm{ww}_{\mathrm{w}+\mathrm{s}+\mathrm{b}}$
Temperature of Water, $\mathrm{T}_{\mathrm{w}, 26}$.
206.28 grams
179.6 grams
806.2 grams
26.0 deg. Celsius

Calibration Curve (Mass of Volumetric filled with Water over a range of Temperatures)

| $\mathrm{Mb}+\mathrm{wt}(\mathrm{g})$ | Temp (deg C) |  |
| ---: | ---: | ---: |
| 677.3 | 28.7 | 26 |
| 677.5 | 27.2 | 26 |
| 677.9 | 23.4 | 26 |
| 678.2 | 20.2 | 26 |

$\mathrm{M}_{\mathrm{b}+\mathrm{wt}}=-0.105(\mathrm{~T})+680.34$
$\mathrm{M}_{\mathrm{b}+\mathrm{wt}(26)}=77.61 \mathrm{grams}$

## Calculating Specific Gravity at Temperature ( $26^{\circ} \mathrm{C}$ )

$$
G_{s @ 26}=\frac{M_{s}}{\left(M_{(b+w)}+M_{s}\right)-M_{(b+w+s t)}}=\frac{206.28 \mathrm{grams}}{(677.61+206.28)-806.2}=2.655
$$

$$
\begin{aligned}
& \boldsymbol{\rho}_{\mathrm{w} 20 \cdot \mathrm{C}}=0.9982063 \\
& G_{s @_{20}}=G_{s @ 26} \frac{\rho_{\mathrm{w} 26^{\circ} \mathrm{C}}}{\rho_{w 20^{\circ} \mathrm{C}}}=2.655 * \frac{0.9967870}{0.9982063}=2.651 \\
& G_{s @ 20}=2.561
\end{aligned}
$$

## Appendix $\mathbf{C}-\mathbf{e}_{\text {min }}$ and $\mathbf{e}_{\text {max }}$ Fine Nevada Sand Test data and Results

Emin

| Total weight of sand, $\mathrm{W}_{\mathrm{t}}$ | 879.8 grams |
| :--- | :--- |
| Volume measured, $\mathrm{V}_{\text {emin }}$ | $500 \mathrm{~cm}^{\wedge} 3$ |

Emax
Total weight of sand, $\mathrm{W}_{\mathrm{t}}$
1413.8 grams

Volume measured, $\mathrm{V}_{\mathrm{emax}} \quad 945 \mathrm{~cm}^{\wedge} 3$

Specific Gravity, $\mathrm{G}_{\mathrm{s}} \quad 2.65$
Unit weight of Water, $\gamma_{w} \quad 1 \mathrm{~g} / \mathrm{cm}^{3}$
$\gamma_{d \max }=\frac{W_{t}}{V_{e \min }}=\frac{879.8 \mathrm{grams}}{500 \mathrm{~cm}^{3}}=1.759 \mathrm{~g} / \mathrm{cm}^{3}$
$e_{\text {min }}=\frac{G_{s} \gamma_{w}}{\gamma_{d, \text { max }}}-1=\frac{2.65 * 1 \mathrm{~g} / \mathrm{cm}^{3}}{1.759 \mathrm{~g} / \mathrm{cm}^{3}}-1=0.507$
$e_{\text {min }}=0.507$
$\gamma_{d \min }=\frac{W_{t}}{V_{e \max }}=\frac{1413.8 \mathrm{grams}}{945 \mathrm{~cm}^{3}}=1.496 \mathrm{~g} / \mathrm{cm}^{3}$
$e_{\max }=\frac{G_{s} \gamma_{w}}{\gamma_{d, \min }}-1=\frac{2.65 * 1 \mathrm{~g} / \mathrm{cm}^{3}}{1.496 \mathrm{~g} / \mathrm{cm}^{3}}-1=0.771$
$e_{\text {max }}=0.771$

# Appendix D - Grain Size Distribution Data (Sieve and Hydrometer) 

## Sieve Analysis for Fine Nevada Sand

Date 14-Sep-10
$\begin{array}{ll}\text { Shake Time } 25 \text { minutes } \\ \text { Sand }(\mathrm{g}) & 600\end{array}$

| Sieve No. | Sieve Opening <br> $[\mathrm{mm}]$ | Soil+Tare <br> $[\mathrm{gr}]$ | Tare <br> [gr] $]$ | Soil <br> $[\mathrm{gr}]$ |  |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |


| $D_{10}$ | 0.1144 |
| :---: | :---: |
| $D_{30}$ | 0.1688 |
| $D_{60}$ | 0.2375 |


| $\mathrm{C}_{\mathrm{c}}$ | 1.0487 | coefficient of uniformity |
| :--- | :--- | :--- |
| $\mathrm{C}_{u}$ | 2.0760 | coefficient of curvature |


| Hydrometer Data |  |
| :---: | :---: |
| 0.08 | 1.40 |
| 0.06 | 0.61 |
| 0.04 | 0.29 |
| 0.02 | 0.11 |
| 0.02 | 0.07 |
| 0.01 | 0.02 |
| 0.01 | 0.00 |
| 0.00 | 0.00 |
| 0.00 | 0.00 |
| 0.001915378 | 0.00 |
| 0.000850853 | 0.00 |

# Appendix D - Grain Size Distribution Data (Sieve and Hydrometer) 

## continued

## Hyrdometer Analysis for Fine Neveda Sand

| Soil $[\mathrm{g}]$ | 50 |
| :---: | :---: |
| $\mathrm{c}_{\mathrm{m}}$ | 0.5 |
| $\mu[\mathrm{mPa}-\mathrm{s}]_{24}$ | 0.916 |
| $\mathrm{G}_{\mathrm{S}}$ | 2.65 |
| $\rho_{\mathrm{w} 24}\left[\mathrm{~g} / \mathrm{cm}^{3}\right]_{24}$ | 0.9972994 |
| $\mathrm{~g}\left[\mathrm{~cm} / \mathrm{s}^{2}\right]$ | 980.67 |
| Hydrometer <br> in water | 1000.5 |
| Suspension <br> Volume [liter] | 1 |


| $\mathrm{h}[\mathrm{cm}]$ | 14.000 |
| :---: | :---: |
| $\mathrm{~V}_{\mathrm{h}}\left[\mathrm{cm}^{3}\right]$ | 50.000 |
| $\mathrm{~A}\left[\mathrm{~cm}^{2}\right]$ | 28.169 |


| Temperature | Elapsed Time <br> $[\mathrm{min}]$ | Observed <br> Hydrometer <br> Reading $\mathrm{R}^{\prime}$ | Corrected <br> Hydrometer <br> Reading $\mathrm{R}_{\mathrm{T}}=\mathrm{R}^{\prime}{ }_{\mathrm{T}}+\mathrm{c}_{\mathrm{m}}$ | Depth, H <br> $[\mathrm{cm}]$ | Grain Size [mm] | Percent <br> Finer [\%] |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 24.00 | 0.25 | 1020.00 | 1020.50 | 11.54 | 0.089 | 62.47 |
| 24.00 | 0.50 | 1014.00 | 1014.50 | 9.95 | 0.058 | 43.25 |
| 24.00 | 1.00 | 1007.00 | 1007.50 | 8.10 | 0.037 | 20.82 |
| 24.00 | 2.00 | 1003.00 | 1003.50 | 7.04 | 0.024 | 8.01 |
| 24.00 | 5.00 | 1002.00 | 1002.50 | 6.77 | 0.015 | 4.81 |
| 24.00 | 10.00 | 1001.00 | 1001.50 | 6.51 | 0.011 | 1.60 |
| 24.00 | 20.00 | 1000.50 | 1001.00 | 6.38 | 0.007 | 0.00 |
| 24.00 | 61.00 | 1000.50 | 1001.00 | 6.38 | 0.004 | 0.00 |
| 24.00 | 128.00 | 1000.50 | 1001.00 | 6.38 | 0.003 | 0.00 |
| 24.00 | 296.00 | 1000.50 | 1001.00 | 6.38 | 0.002 | 0.00 |
| 24.00 | 1500.00 | 1000.50 | 1001.00 | 6.38 | 0.001 | 0.00 |

Calibration Curve

| $R_{T}$ | $H_{1} \quad[\mathrm{~cm}]$ | $0.5^{*}\left(h-v_{h} / A\right)$ | H <br> $[\mathrm{cm}]$ |
| :---: | :---: | :---: | :---: |
| 1000.0 | 0 | 6.112 | 6.112 |
| 1040.0 | 10.6 | 6.112 | 16.712 |



# Appendix D - Grain Size Distribution Data (Sieve and Hydrometer) 

 continuedHydrometery Graph


Appendix E-Compiled List of Researchers who worked with Hollow Cylinder Specimens

Table 2.1. Researchers who worked with hollow cylin der (HC) specimens (expanded from Koester 1992) Contd.

| \# | Reference | Year | Specimen Dimensions (mmm) |  |  | Soil Type | Control Restrictions | Applications | Specimen Preparation Method | Uniformity Check |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  |  |  | H | $\mathrm{R}_{0}$ | $\mathrm{R}_{\mathrm{i}}$ |  |  |  |  |  |
| 23 | Ishibashi and Sherif | 1974 | 13 to 25 | 50.8 | 25.4 | Sand | $\mathrm{P}_{\mathrm{n}}=\mathrm{P}_{\mathrm{i}}$ | Liquefaction characteristics | WP | No |
| 24 | Tseng | 1974 | ? | 72.8 | 35.7 | Sand | $P_{0}=P_{i}$ | Liqucfaction, solid versus hollow spceimens | Compaction | NO |
| 25 | Ishilara and Yasuda | 1975 | 70 | 50 | 30 | Sand | $\mathrm{P}_{\mathrm{o}}=\mathrm{P}_{i} ; \varepsilon_{2}=0$ | Liquefaction of sands, irregular excitation | AP | NO |
| 26 | Lade | 1975 | 50 | 110 | 90 | Sand | $\mathrm{P}_{\mathrm{n}}=\mathrm{P}_{\mathrm{i}}$ | $\alpha$ effect on stress-strain behavior | No info | No |
| 27 | Al-Hussaini | 1981 | 101.6 | 76.2 | 50.8 | $\begin{gathered} \hline \text { Clay and Claycy } \\ \text { Sand } \end{gathered}$ | $\varepsilon^{\prime}=0$ | Tensile properties of compacted soils | Compaction | NO |
| 28 | Cheng | 1981 | 76.2 | 101.6 | 76.2 | Clay | $\mathrm{F}_{\mathrm{x}}=\mathrm{P}_{\mathrm{l}}=\mathrm{P}_{\mathrm{i}}=0$ | Strain rate effects in torsion | One-dimensional slury consolidation | No |
| 29 | Dusseault | 1981 | 200-240 | 50.8 | 25.4 | Dense Oil Sand | $\mathrm{T}_{\mathrm{h}}=0$ | Tunneling and pressuremeter paths | Trimming and coring | No |
| 30 | Lade | 1981 | 400 | 110 | 90 | Sand | $\mathrm{P}_{\mathrm{n}}=\mathrm{P}_{\mathrm{i}}$ | Influence of specimen height |  | No |
| 31 | Saada and Shook | 1981 | 140-152 | 35.6 | 25.4 | Clay | $\mathrm{P}_{0}=\mathrm{P}_{\mathrm{i}}$ | Slow cyclic and large shear strain | Ono-dimensional slurry consolidation | NO |
| 32 | Fukushima and Tatsuoka | 1982 | 200 | 50 | 30 | Sand | $\mathrm{P}_{\mathrm{n}}=\mathrm{P}_{1}$ | Deformation and strength behavior | AP | No |
| 33 | Symes et al. | 1982 | 254 | 127.5 | 101.5 | Sand | ---- | $\alpha$ and b effects on strain response | wp | No |
| 34 | Tatsuoka et al. | 1982 | 100-200 | 100 | 60 | Sand | $P_{0}=P_{1}, \varepsilon_{r}=0$ | Cyclic undrained stress-strain, dense sands | AP \& MT \& Static moist compaction | No |
| 35 | Donaghe and Gillbert | 1983 | 203.2 | 50.8 | 35.6 | Sand | $\mathrm{P}_{\mathrm{o}}=\mathrm{P}_{\text {i }}$ | Principal stress rotation: liquefaction | MT | No |
| 36 | Hight et al. | 1983 | 254 | 127.5 | 101.5 | Sand | $\cdots-$ | Specimen dimensions, $\alpha$ and $b$ effects on stress- strain | WP | NO |
| 37 | Ishilara and Towhata | 1983 | 104 | 50 | 30 | Sand | $\mathrm{P}_{\mathrm{n}}=\mathrm{P}_{\mathrm{i}}$ | Wave loading; principal stress rotation pore pressures | AP | No |
| 38 | Ishilara and Yamazaki | 1984 | 101 | 50 | 30 | Sand | $\mathrm{Pa}_{\mathrm{u}}=\mathrm{P}_{\mathrm{i}}$ | $\begin{array}{c}\text { Liquefaction in seabed deposits due to } \\ \text { wave loads }\end{array}$ | AP | No |
| 39 | Macky and Saada | 1984 | 108-152 | 25-35 | 18-25 | Clay | $\mathrm{P}_{\mathrm{n}}=\mathrm{P}_{\mathrm{i}}$ | Dynamics of anisotropic clays, large strains | One-dimensional slurry consolidation | No |
| 40 | Symes et al. | 1984 | 254 | 127 | 101.6 | Sand | $\cdots$ | Undrained anisotropy and principal stress rotation | wp | No |
| 41 | Tatsuoka et al. | 1984 | 100 | 50 | 30 | Sand | $\mathrm{P}_{\mathrm{o}}=\mathrm{P}_{\mathrm{i}}$ | Specimen preparation methods, cyclic undrained strength | AP, MT. wet-vibration, WP | No |
| 42 | Ishibashi ct al. | 1985 | 142 | 35.5 | 25.4 | Sand | $\mathrm{P}_{\mathrm{n}}=\mathrm{P}_{\mathrm{i}}$ | Liquefaction characteristics | AP + Tamping | No |

Table 2.1. Researchers who worked with hollow cylinder (HC) specimens (expanded from Koester 1992) Contd.

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| \# | Reference | Year | Specimen Dimensions (mm) |  |  | Soil Type | Control Restrictions | Applications | Specimen Preparation Method | Uniformity Check |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  |  |  | H | $\mathrm{R}_{0}$ | $\mathrm{R}_{\mathrm{i}}$ |  |  |  |  |  |
| 63 | Nakata et al. | 1998 | 200 | 50.0 | 30 | Sand | -- | Effect of $\alpha$ on undrained behavior | AP | NO |
| 64 | Lade and Kirkgard | 2000 | 250 | 110 | 90 | Clay | $\mathrm{P}_{\mathrm{o}}=\mathrm{P}_{\mathrm{i}}$ | $\alpha$ and $b$ effects on cross-anisotropic behavior | Trimming of undisturbed specimens | NO |
| 65 | Toyota et al. | 2001 | 160 | 40 | 25 | Clay | .-.- | Stress-strain behavior of clays under 3- $\qquad$ | One-dimensional slurry consolidation | NO |
| 66 | Zdravkovic\&Jardine | 2001 | 254 | 127.0 | 101.6 | Silt | --.- | Effect of rotating principal stresses during consolidation | SD | NO |
| 67 | Lee ct al. | 2002 | 230 | 49 | 39 | Mudstone | $\mathrm{P}_{\mathrm{o}}=\mathrm{P}_{\mathrm{i}}$ | Stress-strain behavior of a mudstone | Drilling | NO |
| 68 | Sivathayalan and Vaid | 2002 | 300 | 76 | 51 | Sand | $\cdots$ | Influence of initial state and $\alpha$ on undrained response | WP | NO |
| 69 | Uthayakumar and Vaid | 2002 | 300 | 76 | 51 | Sand | ---- | Static liquefaction under multiaxial $\qquad$ | WP + Vibration | NO |
| 70 | Chaudhary et al. | 2002 | 100 | 50 | 30 | Sand | ---- | Effect of initial fabric and shearing direction on cyclic deformation | AP, WP, dry rodding | NO |
| 71 | Shibuya et al. | 2003 | 254 | 127.5 | 101.5 | Sand | ---- | Four-dimensional local boundary surfaces | WP | NO |
| 72 | Rolo | 2003 | 254 | 127 | 102 | Clayey Sand | -..- | Anisotropic behavior of loose clayey sands | SD | In terms of fincs content |
| 73 | Chaudhary et al. | 2004 | 200 | 50 | 30 | Sand | ---- | Quasi-Elastic stifficss parameters in HC and TX testing | AP | NO |
| 74 | Naughton and O'Kelly | 2004 | 200 | 50 | 35.5 | Sand | --. | Induced anisotropy | WP | NO |
| 75 | Brown and Richardson | 2004 | 500 | 140 | 112 | Sand and granulated crushed slate | ---- | Cyclic loading behavior of dry granular material | AP | NO |
| 76 | Lin and Penumadu | 2005 | 230 | 25.4 | 17.8 | Clay | -- | Effect of principal stress rotation on Kaolin clay behavior | One-dimensional slurry consolidation | In terms of water content |
| 77 | Lin and Penumadu | 2005 | 230 | 25.4 | 17.8 | Clay | .... | Strain localization in combined axialtorsional loading | Onc-dimensional slurry consolidation | NO |
| 78 | Silvestri et al. | 2005 | 100 | 50-63.5 | 19-25 | Clay | --- | Expansion tests in clay using HC apparatus | Cutting with the technique of electroosmosis | NO |
| 79 | O'Kelly and Naughton | 2005 | 200 | 50 | 35.5 | Sand | ---- | Development of a hollow cylinder apparatus | WP | NO |
| 80 | O'Kelly and Naughton | 2005 | 200 | 50 | 35.5 | Sand | ---- | Engineering propertics of wet pluviated HC specimens | WP | NO |
| 81 | Altun et al. | 2005 | 200 | 50 | 30 | Silty sand | $\mathrm{P}_{\mathrm{o}}=\mathrm{P}_{\text {i }}$ | Effect of fincs content on liquefaction resistance | AP | NO |
| 82 | Yang ct al. | 2007 | 314 | 100 | 75 | Sand | .... | Anisotropic behavior in rotational shear | AP | NO |

## Appendix F - Description and Method to Project Stress points onto the Same Octahedral Plane

True Triaxial and Torsion Shear tests are conducted with three unequal principal stresses. Therefore, if the tests do not keep a constant mean confining stress, the specimens will not all fail on the same octahedral plane. The principal stresses however, can be modified so that they all fall on the same plane. This procedure will be described in the pages that follow.


Figure 1. Schematic Illustration of the Projection Procedure

Point A in Figure 1 shows the stress state, $\left(\sigma^{\prime}, \sigma^{\prime}{ }_{2}, \sigma^{\prime}{ }_{3}\right)$ which is known. Point B is where the stress state will be adjusted and projected to, $\left(\sigma^{*}{ }_{1}, \sigma^{*}{ }_{2}, \sigma^{*}\right)$. This is done by projecting Point A along a line with slope m to obtain a desired value of the first stress invariant, $\mathrm{I}_{12} . \mathrm{I}_{12}$ is the octahedral plane that is desired. m is the geometric slope attained from plotting three triaxial compression tests. The failure state of Point A is defined by
$\left(\frac{I_{1}^{3}}{I_{3}}-27\right)_{1} *\left(\frac{I_{1}}{p_{a}}\right)_{1}^{m}=\bar{\eta}_{1}$ Eq. F. 1
where $\mathrm{pa}=1 \mathrm{~kg} / \mathrm{cm} 2$; Therefore, Eq. F. 1 can be simplified to be
$\left(\frac{I_{1}^{3}}{I_{3}}-27\right)_{1} *\left(I_{1}\right)_{1}^{m}=\bar{\eta}_{1}$

Similarly, the failure state at Point B can be written as
$\left(\frac{I_{1}^{3}}{I_{3}}-27\right)_{2} *\left(I_{12}\right)^{m}=\bar{\eta}_{1}$
which can be rearranged as
$\left(\frac{I_{1}^{3}}{I_{3}}-27\right)_{2}=\frac{\bar{\eta}_{1}}{\left(I_{12}\right)^{m}}$
$\left(\frac{I_{1}^{3}}{I_{3}}\right)_{2}=\frac{\bar{\eta}_{1}}{\left(I_{12}\right)^{m}}+27$
$\left(I_{3}\right)_{2}=\frac{1}{\frac{\bar{\eta}_{1}}{\left(I_{12}\right)^{m}}+27} * I_{12}^{3}$

Eq. F. 4

The intermediate stress ratio parameter, $b$ can be defined as

$$
b=\frac{\left(\sigma_{2}^{*}-\sigma_{3}^{*}\right)}{\left(\sigma_{1}^{*}-\sigma_{3}^{*}\right)}
$$

Eq. F. 5

Rearranging the above equation, the intermediate principal stress, $\sigma^{*}{ }_{2}$ can be expressed as:

$$
\sigma_{2}^{*}=\sigma_{3}^{*}+b\left(\sigma_{1}^{*}-\sigma_{3}^{*}\right)
$$

The desired first stress invariant, I12 is expressed by

$$
I_{12}=\sigma_{1}^{*}+\sigma_{2}^{*}+\sigma_{3}^{*}
$$

Substituting the last two equations, the following is attained:
$I_{12}=\sigma_{1}^{*}+\sigma_{3}^{*}+b\left(\sigma_{1}^{*}-\sigma_{3}^{*}\right)+\sigma_{3}^{*}$
Eq. F. 8
$I_{12}=(1+b) \sigma_{1}^{*}+(2-b) \sigma_{3}^{*}$

Rearranging equation D.8, the desired major principal stress, $\sigma^{*}{ }_{1}$ can be expressed as

$$
\sigma_{1}^{*}=\frac{1}{(1+b)}\left\{I_{12}-(2-b) \sigma_{3}^{*}\right\}
$$

The desired third stress invariant $\mathrm{I}_{32}$ can be calculated as:

$$
I_{32}=\sigma_{1}^{*} * \sigma_{2}^{*} * \sigma_{3}^{*}
$$

Substituting Equation F. 6 into F. 10 yields:
$I_{32}=\sigma_{1}^{*} *\left\{\sigma_{3}^{*}+b^{*}\left(\sigma_{1}^{*}-\sigma_{3}^{*}\right)\right\}^{*} \sigma_{3}^{*}$
$I_{32}=\sigma_{1}^{*}\left(\sigma_{3}^{*}\right)^{2}+b\left(\sigma_{1}^{*}\right)^{2} * \sigma_{3}^{*}-b\left(\sigma_{1}^{*}\right)\left(\sigma_{3}^{*}\right)^{2}$
Eq. F. 11
$I_{\mathfrak{\gamma}}=(1-b) \sigma_{1}^{*}\left(\sigma_{\mathfrak{2}}^{*}\right)^{2}+b\left(\sigma_{1}^{*}\right)^{2} * \sigma_{2}^{*}$

Substituting Eq. F. 9 into F.11:
$I_{32}=(1-b) * \frac{1}{(1+b)} *\left\{I_{12}-(2-b) \sigma_{3}^{*}\right\}\left(\sigma_{3}^{*}\right)^{2}+b \frac{1}{(1+b)^{2}}\left\{I_{12}-(2-b) \sigma_{3}^{*}\right\}\left(\sigma_{3}^{*}\right)$

Eq. F. 12

Further rearrangement results in:
$I_{32}=\frac{b(2-b)^{2}-(1+b)(1-b)(2-b)}{(1+b)^{2}}\left(\sigma_{3}^{*}\right)^{3}+\frac{(1+b)(1-b)-2 b(2-b)}{(1+b)^{2}}\left(I_{12}^{2}\right)\left(\sigma_{3}^{*}\right)^{2}$
$+\frac{b\left(I_{12}^{2}\right)}{(1+b)^{2}}\left(\sigma_{3}^{*}\right)$
or
$\left\{b(2-b)^{2}-(1+b)(1-b)(2-b)\right\}\left(\sigma_{3}^{*}\right)^{3}+\{(1+b)(1-b)-2 b(2-b)\}\left(I_{12}^{2}\right)\left(\sigma_{3}^{*}\right)^{2}$
$+b\left(I_{1}^{2}\right)\left(\sigma_{3}^{*}\right)-(1+b)^{2}\left(I_{\imath}\right)=0$
Eq. F. 13

Equation F. 13 can be simplified into a cubical equation of $\sigma^{*}{ }_{3}$ by
$\left(\sigma_{3}^{*}\right)^{3}+\frac{\left(b^{2}-4 b+1\right) I_{12}}{(2-b)(2 b-1)}\left(\sigma_{3}^{*}\right)^{2}+\frac{(b) I_{12}^{2}}{(2-b)(2 b-1)}\left(\sigma_{3}^{*}\right)+\frac{-(1+b)^{2} I_{32}}{(2-b)(2 b-1)}=0 \quad$ Eq. F. 14

The cubical equation in the form of $x^{\wedge} 3,+A x^{\wedge} 2+B x+C=0$ can be solved by saying that
$A=\frac{\left(b^{2}-4 b+1\right) I_{12}}{(2-b)(2 b-1)}$
Eq. F. 15
$B=\frac{(b) I_{12}^{2}}{(2-b)(2 b-1)}$
Eq. F. 16
$C=\frac{-(1+b)^{2} I_{32}}{(2-b)(2 b-1)}$
Eq. F. 17

The solution to the cubical equation is given by (Korn and Korn, 1961):
$p=-\frac{A^{2}}{3}+B$
Eq. F. 18
$q=2\left(\frac{A}{3}\right)^{3}-\frac{A B}{3}+C$
Eq. F. 19
$x_{1}=2 \sqrt{\frac{-p}{3}} \cos \left(\frac{\alpha}{3}\right)-\left(\frac{A}{3}\right)$
Eq. F. 20
and
$x_{2,3}=-2 \sqrt{\frac{-p}{3}} \cos \left(\frac{\alpha}{3} \pm 60^{\circ}\right)-\left(\frac{A}{3}\right)$
Eq. F. 21
where

$$
\cos (\alpha)=-\frac{q}{2 \sqrt{-\left(\frac{p}{3}\right)^{3}}}
$$

The solution to F. 21 depends on the b-value.

For $\mathrm{b}<0.5$,
$\sigma_{3}^{*}=x_{2}=-2 \sqrt{\frac{-p}{3}} \cos \left(\frac{\alpha}{3}+60^{\circ}\right)-\left(\frac{A}{3}\right)$
Eq. F. 23

For $\mathrm{b}>0.5$,
$\sigma_{3}^{*}=x_{3}=-2 \sqrt{\frac{-p}{3}} \cos \left(\frac{\alpha}{3}-60^{\circ}\right)-\left(\frac{A}{3}\right)$
Eq. F. 24
and for $\mathrm{b}=0.5$, Equation F .14 can be simplified to be
$\left(b^{2}-4 b+1\right)\left(I_{12}\right)\left(\sigma_{3}^{*}\right)^{2}+b\left(I_{12}^{2}\right)\left(\sigma_{3}^{*}\right)-(1+b)^{2}\left(I_{32}\right)=0$
Eq. F. 25
where
$\sigma_{3}^{*}=\frac{1}{3} I_{12}-\frac{1}{3} \sqrt{\frac{I_{32}}{I_{12}}} \sqrt{\frac{I_{12}^{3}}{I_{32}}-27}$
Eq. F. 26

Once $\sigma^{*}{ }_{3}$ has been calculated, the major and intermediate principal stresses can also be determined by solving Eq. F. 6 and F. 7 simultaneously.

## Appendix G - Plotting Lade's Cross Anisotropic Failure Criterion (Lade, 2007)

In order to plot the failure criterion (Lade 2007) in terms of friction angle/stress ratio and bvalue, equations must be combined and rearranged. The steps for doing so are described in detail in the pages that follow.

As stated in Lade (2007), combining Pietrusczak's function for a cross-anisotropic material and Lade's (1977) isotropic three-dimensional failure criterion for soils results in

$$
f=\left(\frac{I_{1}^{3}}{I_{3}}-27\right)\left(\frac{I_{1}}{p_{a}}\right)^{m}=\eta_{0}\left[1+\Omega_{1}\left(1-3 l_{2}^{2}\right)^{-}\right.
$$

Eq. G. 1
where $I_{1}$ and $I_{3}$ are first and the third invariants of the stress tensor, pa is the atmospheric pressure (in the same units as $I_{1}$ ), $\eta_{0}$ and $\Omega_{1}$ are constant material properties and $1_{2}$ is the loading direction relative to the material axis in which up to three different orthogonal normal stresses and one shear stress are applied. The expression for $l_{2}$ is

$$
l_{2}=\sqrt{\frac{\sigma_{y}^{2} \sin ^{2} \beta+\sigma_{z}^{2} \cos ^{2} \beta}{\sigma_{x}^{2}+\sigma_{y}^{2}+\sigma_{z}^{2}}}
$$

Eq. G. 2
where $\sigma_{x}, \sigma_{y}$, and $\sigma_{z}$ are principal stresses and $\beta$ is the major principal stress direction.

Rearranging the terms in equation G. 1 in order to isolate the $\mathrm{I}_{1}$, the equation becomes,

$$
\begin{aligned}
& \left(\frac{I_{1}^{3}}{I_{3}}-27\right)=\frac{\eta_{0}\left[1+\Omega_{1}\left(1-3 l_{2}^{2}\right)\right]}{\left(\frac{I_{1}}{p_{a}}\right)^{m}} \\
& \left(\frac{I_{1}^{3}}{I_{3}}\right)=\frac{\eta_{0}\left[1+\Omega_{1}\left(1-3 l_{2}^{2}\right)\right]}{\left(\frac{I_{1}}{p_{a}}\right)^{m}}+27=K \\
& I_{1}^{3}=K^{*} I_{3}
\end{aligned}
$$

Eq. G. 3

Recalling that

$$
\begin{aligned}
& b=\frac{\left(\sigma_{2}-\sigma_{3}\right)}{\left(\sigma_{1}-\sigma_{3}\right)} \\
& I_{1}=\sigma_{1}+\sigma_{2}+\sigma_{3} \\
& I_{1}=\sigma_{1} * \sigma_{2} * \sigma_{3}
\end{aligned}
$$

Eq. G. 4

Eq. G. 5
Eq. G. 6

Equation G. 3 can be rearranged so that it is in terms of b-value and stresses.

$$
\begin{aligned}
& I_{1}^{3}=K * I_{3} \\
& \left(\sigma_{1}+\sigma_{2}+\sigma_{3}\right)^{3}=K\left(\sigma_{1} * \sigma_{2} * \sigma_{3}\right) \\
& \left(\sigma_{1}+\sigma_{2}+\sigma_{3}\right)^{3}=K * \sigma_{1} \sigma_{3}\left(b\left(\sigma_{1}-\sigma_{3}\right)+\sigma_{3}\right)
\end{aligned}
$$

Eq. G. 7

Rearranging the above equations in terms of $\sigma_{1} / \sigma_{3}$ results in

$$
\begin{aligned}
& \frac{\sigma_{1}}{\sigma_{3}}=\frac{\left(\sigma_{1}+\sigma_{2}+\sigma_{3}\right)^{3}}{K^{*} \sigma_{3}^{2}\left(b\left(\sigma_{1}-\sigma_{3}\right)+\sigma_{3}\right)} \\
& \frac{\sigma_{1}}{\sigma_{3}}=\frac{\left(\sigma_{1}+\left(b\left(\sigma_{1}-\sigma_{3}\right)+\sigma_{3}\right)+\sigma_{3}\right)^{3}}{K^{*} \sigma_{3}^{2}\left(b\left(\sigma_{1}-\sigma_{3}\right)+\sigma_{3}\right)} \\
& S R=\frac{(S R+(b(S R-1)+1)+1)^{3}}{(b(S R-1)+1)}
\end{aligned}
$$

where
$S R=\frac{\sigma_{1}}{\sigma_{3}}$
$0=[S R+(b(S R-1)+1)+1]^{3}-S R^{*} K(b(S R-1)+1)$
Eq. G. 8

Equation G. 8 can be set to zero and the cubical equation can be solved only with inputting the values of k (equation G. 3) and b -value. The stress ratios can then be converted into friction angles if desired.

## Appendix H - Equations used during Torsion Shear Testing where alpha, b-value and mean confining stress are kept constant.

The torsion shear experimental program involved conditions that kept b-value, alpha, and mean normal stress constant throughout the entire test. The inner pressure, outer pressure and vertical force therefore, had to be functions of the inputs given. By inputting the specific alpha, $b$-value and mean confining stress, while knowing the real-time inner and outer radius (calculated by inner volume change and specimen volume change), the stress path could be held constant. The mean normal stress was kept constant at 101.4 kPa . B-values varied in increments of 0.25 from 0 to 1 for each test and alpha values varied in 22.5 degree increments from 0 to 90 degrees. In the pages that follow, the equations used to derive the inner and outer pressure, as well as the vertical force required for the testing program is given.

As previously stated, in order to calculate stresses in a torsion shear specimen, the following equations are used:

$$
\begin{align*}
& \sigma_{1}=\frac{\sigma_{z}+\sigma_{\theta}}{2}+\sqrt{\left(\frac{\sigma_{z}+\sigma_{\theta}}{2}\right)^{2}+\tau_{z \theta}{ }^{2}} \\
& \sigma_{3}=\frac{\sigma_{z}+\sigma_{\theta}}{2}-\sqrt{\left(\frac{\sigma_{z}+\sigma_{\theta}}{2}\right)^{2}+\tau_{z \theta}{ }^{2}} \\
& \sigma_{2}=\sigma_{r}=\frac{p_{o} r_{o}+p_{i} r_{i}}{r_{o}+r_{i}}
\end{align*}
$$

Eq. H. 3
$\sigma_{\theta}=\frac{p_{o} r_{o}-p_{i} r_{i}}{r_{o}-r_{i}}$
Eq. H. 4
$\sigma_{z}=\frac{F_{v}}{\pi\left(r_{o}^{2}-r_{i}^{2}\right)}+\frac{p_{o} r_{o}^{2}-p_{i} r_{i}^{2}}{\left(r_{o}^{2}-r_{i}^{2}\right)}$
Eq. H. 5

$$
\tau_{z \theta}=\frac{3 M}{2 \pi\left(r_{o}^{3}-r_{i}^{3}\right)}
$$

The equations that were kept constant in the torsion shear experiments are:
$\sigma_{m}=\frac{\sigma_{1}+\sigma_{2}+\sigma_{3}}{3}$
Eq. H. 7
$b=\frac{\sigma_{2}-\sigma_{3}}{\sigma_{1}-\sigma_{3}}$
Eq. H. 8
$\alpha=\frac{1}{2} \tan ^{-1}\left(\frac{2 \tau_{z \theta}}{\sigma_{z}-\sigma_{\theta}}\right)$
Eq. H. 9

In order to express the vertical force $\left(F_{v}\right)$, the inner pressure $\left(p_{i}\right)$ and the outer pressure $\left(p_{o}\right)$ as a function of moment (M), b-value, alpha ( $\alpha$ ), and mean confining stress $\left(\sigma_{m}\right)$, the following equations were obtained.
$\frac{\sigma_{z}+\sigma_{\theta}}{2}=\frac{F_{v}}{2 \pi\left(r_{o}^{2}-r_{i}^{2}\right)}+\frac{p_{o} r_{o}^{2}-p_{i} r_{i}^{2}}{2\left(r_{o}^{2}-r_{i}^{2}\right)}+\frac{p_{o} r_{o}-p_{i} r_{i}}{2\left(r_{o}-r_{i}\right)}$
Eq. H. 10
which when simplified equals

$$
\frac{\sigma_{z}+\sigma_{\theta}}{2}=\frac{F_{v}}{2 \pi\left(r_{o}^{2}-r_{i}^{2}\right)}+\frac{2 p_{o} r_{o}^{2}-2 p_{i} r_{i}^{2}+r_{i} r_{o}\left(p_{o}-p_{i}\right)}{2\left(r_{o}^{2}-r_{i}^{2}\right)}
$$

Similarly,

$$
\frac{\sigma_{z}-\sigma_{\theta}}{2}=\frac{F_{v}}{2 \pi\left(r_{o}^{2}-r_{i}^{2}\right)}+\frac{p_{o} r_{o}^{2}-p_{i} r_{i}^{2}}{2\left(r_{o}^{2}-r_{i}^{2}\right)}-\frac{p_{o} r_{o}-p_{i} r_{i}}{2\left(r_{o}-r_{i}\right)}
$$

Eq. H. 12
$\frac{\sigma_{z}-\sigma_{\theta}}{2}=\frac{F_{v}+\pi r_{i} r_{o}\left(p_{i}-p_{o}\right)}{2 \pi\left(r_{o}^{2}-r_{i}^{2}\right)}$

Using Equation H.11, the mean normal stress can be simplified by a rearrangement of terms where,

$$
\sigma_{m}=\frac{1}{3}\left[\left(\sigma_{1}+\sigma_{3}\right)+\sigma_{2}\right]=\frac{1}{3}\left[2\left(\frac{\sigma_{z}+\sigma_{\theta}}{2}\right)+\sigma_{r}\right]
$$

Eq. H. 13

Substituting equations H. 11 and H. 3 into H. 13 results in

$$
\begin{aligned}
\sigma_{m} & =\frac{1}{3}\left[\frac{F_{v}}{\pi\left(r_{o}^{2}-r_{i}^{2}\right)}+\frac{2 p_{o} r_{o}^{2}-2 p_{i} r_{i}^{2}+r_{i} r_{o}\left(p_{o}-p_{i}\right)}{\left(r_{o}^{2}-r_{i}^{2}\right)}+\frac{p_{o} r_{o}+p_{i} r_{i}}{\left(r_{o}+r_{i}\right)}\right] \\
\sigma_{m} & =\frac{1}{3\left(r_{o}^{2}-r_{i}^{2}\right)}\left[\frac{F_{v}}{\pi}+3\left(p_{o} r_{o}^{2}-p i r_{i}^{2}\right)\right]
\end{aligned}
$$

Eq. H. 14

Equation H. 9 can be rearranged to get

$$
\begin{aligned}
& \tan 2 \alpha=\left(\frac{\tau_{z \theta}}{\frac{\sigma_{z}-\sigma_{\theta}}{2}}\right)=\frac{3 M}{2 \pi\left(r_{o}^{3}-r_{i}^{3}\right)} \frac{2 \pi\left(r_{o}^{2}-r_{i}^{2}\right)}{F_{v}+\pi r_{i} r_{o}\left(p_{i}-p_{o}\right)} \\
& 3 M \frac{\left(r_{o}^{2}-r_{i}^{2}\right)}{\left(r_{o}^{3}-r_{i}^{3}\right)}=\left[F_{v}+\pi r_{i} r_{o}\left(p_{i}-p_{o}\right)\right] \tan 2 \alpha
\end{aligned}
$$

Eq. H. 15

In order to get Eq. H. 8 in terms of $p_{o}, p_{i}$ and $F_{v}$, the numerator and denominator of the $b-$ value equation can be broken down into the following equations such that:
$\sigma_{1}-\sigma_{3}=2 \sqrt{\left(\frac{\sigma_{z}-\sigma_{\theta}}{2}\right)^{2}+\tau_{z \theta}^{2}}$
Eq. H. 16

$$
\begin{aligned}
& \left(\sigma_{1}-\sigma_{3}\right)^{2}=4\left[\left(\frac{\sigma_{z}-\sigma_{\theta}}{2}\right)^{2}+\tau_{z \theta}^{2}\right] \\
& \left(\sigma_{1}-\sigma_{3}\right)^{2}=4\left[\left(\frac{F_{v}+\pi r_{i} r_{o}\left(p_{i}-p_{o}\right)}{2 \pi\left(r_{o}^{2}-r_{i}^{2}\right)}\right)^{2}+\left(\frac{3 M}{2 \pi\left(r_{o}^{3}-r_{i}^{3}\right)}\right)^{2}\right] \\
& \left(\sigma_{1}-\sigma_{3}\right)^{2}=\frac{1}{\pi^{2}\left(r_{o}^{2}-r_{i}^{2}\right)^{2}}\left[\left(F_{v}+\pi r_{i} r_{o}\left(p_{i}-p_{o}\right)\right)^{2}+\left(F_{v}+\pi r_{i} r_{o}\left(p_{i}-p_{o}\right)\right)^{2} \tan ^{2} 2 \alpha .\right. \\
& \left(\sigma_{1}-\sigma_{3}\right)^{2}=\frac{1}{\pi^{2}\left(r_{o}^{2}-r_{i}^{2}\right)^{2}}\left[\left(F_{v}+\pi r_{i} r_{o}\left(p_{i}-p_{o}\right)\right)^{2} *\left(1+\tan ^{2} 2 \alpha\right)\right] \\
& \left(\sigma_{1}-\sigma_{3}\right)^{2}=\frac{1}{\pi^{2}\left(r_{o}^{2}-r_{i}^{2}\right)^{2}}\left[\left(F_{v}+\pi r_{i} r_{o}\left(p_{i}-p_{o}\right)\right)^{2} * \frac{1}{\cos ^{2} 2 \alpha}\right] \\
& \left(\sigma_{1}-\sigma_{3}\right)=\frac{\frac{F_{v}}{\pi}+r_{i} r_{o}\left(p_{i}-p_{o}\right)}{\cos 2 \alpha\left(r_{n}^{2}-r_{i}^{2}\right)}
\end{aligned}
$$

Eq. H. 17

In order to isolate $\sigma_{3}$, equations H. 11 and H. 17 can be used with Eq. H. 2 such that

$$
\begin{aligned}
& \sigma_{3}=\frac{F_{v}}{2 \pi\left(r_{o}^{2}-r_{i}^{2}\right)}+\frac{2 p_{o} r_{o}^{2}-2 p_{i} r_{i}^{2}+r_{i} r_{o}\left(p_{o}-p_{i}\right)}{2\left(r_{o}^{2}-r_{i}^{2}\right)}-\frac{\frac{F_{v}}{\pi}+r_{i} r_{o}\left(p_{i}-p_{o}\right)}{\cos 2 \alpha\left(r_{o}^{2}-r_{i}^{2}\right)} \\
& \sigma_{3}=\frac{\frac{F_{v}}{\pi}(\cos 2 \alpha-1)+2 \cos 2 \alpha\left(p_{o} r_{o}^{2}-p_{i} r_{i}^{2}\right)+r_{i} r_{o}\left(p_{o}-p_{i}\right)(\cos 2 \alpha+1)}{2\left(r_{o}^{2}-r_{i}^{2}\right) \cos 2 \alpha}
\end{aligned}
$$

Eq. H. 18

Combining Eq. H. 3 and H.18, $\left(\sigma_{2}-\sigma_{3}\right)$ is simplified to become

$$
\begin{aligned}
& \sigma_{2}-\sigma_{3}=\frac{p_{o} r_{o}+p_{i} r_{i}}{r_{o}+r_{i}}-\frac{\frac{F_{v}}{\pi}(\cos 2 \alpha-1)+2 \cos 2 \alpha\left(p_{o} r_{o}^{2}-p_{i} r_{i}^{2}\right)+r_{i} r_{o}\left(p_{o}-p_{i}\right)(\cos 2 \alpha+1)}{2\left(r_{o}^{2}-r_{i}^{2}\right) \cos 2 \alpha} \\
& \sigma_{2}-\sigma_{3}=\frac{\frac{F_{v}}{\pi}(1-\cos 2 \alpha)+3 \cos 2 \alpha * r_{i} r_{o}\left(p_{i}-p_{o}\right)+r_{i} r_{o}\left(p_{i}-p_{o}\right)}{2\left(r_{o}^{2}-r_{i}^{2}\right) \cos 2 \alpha}
\end{aligned}
$$

Eq. H. 19

Equation H. 19 can be combined with Equation H. 17 to solve for b, where

$$
\begin{aligned}
& \frac{\sigma_{2}-\sigma_{3}}{\sigma_{1}-\sigma_{3}}=\frac{\frac{F_{v}}{\pi}(1-\cos 2 \alpha)+3 \cos 2 \alpha^{*} r_{i} r_{o}\left(p_{i}-p_{o}\right)+r_{i} r_{o}\left(p_{i}-p_{o}\right)}{2\left(r_{o}^{2}-r_{i}^{2}\right) \cos 2 \alpha} \\
& b=\frac{\frac{F_{v}}{\pi}+r_{i} r_{o}\left(p_{i}-p_{o}\right)}{\cos 2 \alpha\left(r_{o}^{2}-r_{i}^{2}\right)} \\
& 2\left(\frac{F_{v}}{\pi}(1-\cos 2 \alpha)+(3 \cos 2 \alpha+1) r_{i} r_{o}\left(p_{i}-p_{o}\right)\right. \\
& 2\left(r_{i} r_{o}\left(p_{i}-p_{o}\right)\right)
\end{aligned}
$$

Eq. H. 20

Equations H. 14, H. 15 and H. 20 are a set of three linear equations with three unknowns ( $\mathrm{p}_{\mathrm{i}}$, $\mathrm{p}_{\mathrm{o}}$, and $\mathrm{F}_{\mathrm{v}}$ ). The three can be solved simultaneously resulting in three equations for $\mathrm{p}_{\mathrm{i}}, \mathrm{p}_{\mathrm{o}}$, and $\mathrm{F}_{\mathrm{v}}$ :

$$
p_{i}=\frac{3\left(r_{o}^{2}-r_{i}^{2}\right) \sigma_{m}-\left[\frac{3 r_{o}}{3 r_{o}+r_{i}}+\frac{3 \cos 2 \alpha-2 b+1}{1-2 b-\cos 2 \alpha} * \frac{r_{i}}{3 r_{o}+r_{i}}\right]\left(3\left(r_{o}^{2}-r_{i i}^{2}\right) \sigma_{m}-\left(\frac{3\left(r_{o}^{2}-r_{i}^{2}\right)}{\pi\left(r_{o}^{3}-r_{i}^{3}\right)}\right)\left(\frac{M}{\tan 2 \alpha}\right)\right)}{\frac{3 r_{i}\left(r_{o}^{2}-r_{i}^{2}\right)}{3 r_{o}+r_{i}} * \frac{4 \cos 2 \alpha}{\cos 2 \alpha+2 b-1}}
$$

Eq. H. 21

$$
p_{o}=\frac{3\left(r_{o}^{2}-r_{i}^{2}\right)}{r_{o}\left(3 r_{o}+r_{i}\right)} \sigma_{m}-\left(\frac{3\left(r_{o}^{2}-r_{i}^{2}\right)}{\pi\left(r_{o}^{3}-r_{i}^{3}\right)}\right)\left(\frac{1}{r_{o}\left(3 r_{o}+r_{i}\right)}\right)\left(\frac{M}{\tan 2 \alpha}\right)+\left(\frac{r_{i}\left(3 r_{i}+r_{o}\right)}{r_{o}\left(3 r_{o}+r_{i}\right)}\right) p_{i}
$$

Eq, H. 22

$$
F_{v}=\frac{3\left(r_{o}^{2}-r_{i}^{2}\right)}{r_{o}\left(3 r_{o}+r_{i}\right)} \pi \sigma_{m}-3 \pi r_{o}^{2} p_{o}+3 \pi r_{i}^{2} p_{i}
$$

Eq. H. 23

## Appendix Ia --- Measurement Error Corrections on Friction Angle for Torsion Shear Tests

Due to inaccuracies in the measurement devices in the torsion shear apparatus used, errors in the friction angle were calculated. The error estimation was calculated as follows.

The friction angle was calculated as
$\sin \phi=\frac{\left(\sigma_{1}-\sigma_{3}\right)}{\left(\sigma_{1}+\sigma_{3}\right)}$
Eq. I. 1

From Equation H. 17 found in Appendix H, we know that
$\left(\sigma_{1}-\sigma_{3}\right)^{2}=4\left[\left(\frac{F_{v}+\pi r_{i} r_{o}\left(p_{i}-p_{o}\right)}{2 \pi\left(r_{o}^{2}-r_{i}^{2}\right)}\right)^{2}+\left(\frac{3 M}{2 \pi\left(r_{o}^{3}-r_{i}^{3}\right)}\right)^{2}\right]$
Eq. I. 2
which simplifies to

$$
\left(\sigma_{1}-\sigma_{3}\right)=\frac{\sqrt{\left(F_{v}+\pi r_{i} r_{o}\left(p_{i}-p_{o}\right)\right)^{2}+\left(3 M \frac{\left(r_{o}^{2}-r_{i}^{2}\right)}{\left(r_{o}^{3}-r_{i}^{3}\right)}\right)^{2}}}{\pi\left(r_{o}^{2}-r_{i}^{2}\right)}
$$

We can also combine equations H. 1 and H. 2 to get
$\sigma_{1}+\sigma_{3}=2\left(\frac{\sigma_{z}+\sigma_{\theta}}{2}\right)$
Eq. I. 4

Combining the above equation and Equation H. 11 results in
$\sigma_{1}+\sigma_{3}=\frac{F_{v}+\pi\left[2 p_{o} r_{o}^{2}-2 p_{i} r_{i}^{2}+r_{i} r_{o}\left(p_{o}-p_{i}\right)\right]}{\pi\left(r_{o}^{2}-r_{i}^{2}\right)}$
Eq. I. 5

Substituting Equations I. 3 and I. 5 into I.1, the friction angle can be calculated as shown below.
$\sin \phi=\frac{\left(\sigma_{1}-\sigma_{3}\right)}{\left(\sigma_{1}+\sigma_{3}\right)}=\frac{\sqrt{\left(F_{v}+\pi r_{i} r_{o}\left(p_{i}-p_{o}\right)\right)^{2}+\left(3 M \frac{\left(r_{o}^{2}-r_{i}^{2}\right)}{\left(r_{o}^{3}-r_{i}^{3}\right)}\right)^{2}}}{F_{v}+\pi\left\lceil 2 p_{o} r_{o}^{2}-2 p_{i} r_{i}^{2}+r_{i} r_{o}\left(p_{o}-p_{i}\right)\right\rceil}$
Eq. I. 5

As previously defined, Fv is the vertical force. $\mathrm{r}_{\mathrm{o}}$ and $\mathrm{r}_{\mathrm{i}}$ are the outer and inner radii, respectively. $\mathrm{P}_{\mathrm{o}}$ and $\mathrm{P}_{\mathrm{i}}$ are the outer and inner cell pressures, respectively and alpha is the principal stress direction angle, measured from vertical. Because the radii do not change much throughout the test, they are considered to be constant in the error analysis. Therefore, Eq. I.5. can be simplified to
$\sin \phi=\frac{\left(\sigma_{1}-\sigma_{3}\right)}{\left(\sigma_{1}+\sigma_{3}\right)}=\frac{\sqrt{\left(F_{v}+A\left(p_{i}-p_{o}\right)\right)^{2}+(3 M B)^{2}}}{\left(F_{v}+\left(C p_{o}-D p_{i}\right)+A\left(p_{o}-p_{i}\right)\right)}$
Eq. I. 6
when

$$
\begin{aligned}
& A=\pi r_{i} r_{o} \\
& B=\frac{\left(r_{o}^{2}-r_{i}^{2}\right)}{\left(r_{o}^{3}-r_{i}^{3}\right)} \\
& C=2 \pi r_{o}^{2} \\
& D=2 \pi r_{i}^{2}
\end{aligned}
$$

The error in friction angle can be calculated using the Least Squares Method where

$$
\partial \sin \phi=\left[\left(\frac{\partial \sin \phi}{\partial F_{v}} \Delta F_{v}\right)^{2}+\left(\frac{\partial \sin \phi}{\partial p_{o}} \Delta p_{o}\right)^{2}+\left(\frac{\partial \sin \phi}{\partial p_{i}} \Delta p_{i}\right)^{2}+\left(\frac{\partial \sin \phi}{\partial M} \Delta M\right)^{2}\right]^{\frac{1}{2}}
$$

where

Eq. I. 8
$\frac{\partial \sin \phi}{\partial p_{i}}=\frac{\left(F_{v} A+A^{2}\left(p_{i}-p_{o}\right)\right)\left(F_{v}+\left(C p_{o}-D p_{i}\right)+A\left(p_{o}-p_{i}\right)\right)}{\sqrt{\left(F_{v}+A\left(p_{i}-p_{o}\right)\right)^{2}+(3 M B)^{2}}\left[\left(F_{v}+\left(C p_{o}-D p_{i}\right)+A\left(p_{o}-p_{i}\right)\right)\right]^{2}}-\frac{(-D-A) \sqrt{\left(F_{v}+A\left(p_{i}-p_{o}\right)\right)^{2}+(3 M B)^{2}}}{\left[\left(F_{v}+\left(C p_{o}-D p_{i}\right)+A\left(p_{o}-p_{i}\right)\right)\right]^{2}}$

Eq. I. 9
$\frac{\partial \sin \phi}{\partial p_{o}}=\frac{\left(-F_{v} A+A^{2}\left(p_{o}-p_{i}\right)\right)\left(F_{v}+\left(C p_{o}-D p_{i}\right)+A\left(p_{o}-p_{i}\right)\right)}{\sqrt{\left(F_{v}+A\left(p_{i}-p_{o}\right)\right)^{2}+(3 M B)^{2}}\left[\left(F_{v}+\left(C p_{o}-D p_{i}\right)+A\left(p_{o}-p_{i}\right)\right)\right]^{2}}-\frac{(C+A) \sqrt{\left(F_{v}+A\left(p_{i}-p_{o}\right)\right)^{2}+(3 M B)^{2}}}{\left[\left(\left(F_{v}+\left(C p_{o}-D p_{i}\right)+A\left(p_{o}-p_{i}\right)\right)\right)\right]^{2}}$

Eq. I. 10

$$
\frac{\partial \sin \phi}{\partial M}=\frac{\left(9 B^{2} M\right)\left(F_{v}+\left(C p_{o}-D p_{i}\right)+A\left(p_{o}-p_{i}\right)\right)}{\sqrt{\left(F_{v}+A\left(p_{i}-p_{o}\right)\right)^{2}+(3 M B)^{2}}\left[\left(F_{v}+\left(C p_{o}-D p_{i}\right)+A\left(p_{o}-p_{i}\right)\right)\right]^{2}}
$$

Eq. I. 11
and
$\Delta F_{v}$ is the estimated error in $F_{v}$ (vertical force)
$\Delta \mathrm{p}_{\mathrm{i}}$ is the estimated error in $\mathrm{p}_{\mathrm{i}}$ (inner pressure)
$\Delta \mathrm{p}_{\mathrm{o}}$ is the estimated error in $\mathrm{p}_{\mathrm{o}}$ (outer pressure)
$\Delta \mathrm{M}$ is the estimated error in M (moment)

The derivative of $\sin \phi$ is calculated by

$$
\partial \sin \phi=\cos \phi \Delta \phi
$$

Eq. I. 12

The error $\Delta \phi$ can be calculated by combining Equation I. 7 and I. 15.
$\Delta \phi=\frac{180}{\pi \cos \phi} \partial \sin \phi$
$\Delta \phi=\frac{180}{\pi \cos \phi}\left[\left(\frac{\partial \sin \phi}{\partial F_{v}} \Delta F_{v}\right)^{2}+\left(\frac{\partial \sin \phi}{\partial p_{o}} \Delta p_{o}\right)^{2}+\left(\frac{\partial \sin \phi}{\partial p_{i}} \Delta p_{i}\right)^{2}+\left(\frac{\partial \sin \phi}{\partial M} \Delta M\right)^{2}\right]^{\frac{1}{2}}$
Eq. I. 13

## Appendix Ib --- Measurement Error Corrections on b-value for Torsion Shear Tests

In a similar way as the error was calculated for friction angle, the error on the $b$-value can also be calculated by following a similar process. For the torsion shear tests, the b-value, ( $\sigma_{2^{-}}$ $\left.\sigma_{3}\right) /\left(\sigma_{1}-\sigma_{3}\right)$ can be written out in terms of $\mathrm{Fv}, \mathrm{M}, \mathrm{r}_{\mathrm{o}}, \mathrm{r}_{\mathrm{i}}, \mathrm{p}_{\mathrm{o}}$ and $\mathrm{p}_{\mathrm{i}}$. In order to get the numerator from the b-value equation, it is necessary to combine the terms as done below.

$$
\left(\sigma_{2}-\sigma_{3}\right)=\frac{p_{o} r_{o}+p_{i} r_{i}}{r_{o}+r_{i}}-\left[\frac{F_{v}}{2 \pi\left(r_{o}^{2}-r_{i}^{2}\right)}-\frac{2 p_{o} r_{o}^{2}-2 p_{i} r_{i}^{2}+r_{i} r_{o}\left(p_{o}-p_{i}\right)}{2\left(r_{o}^{2}-r_{i}^{2}\right)}\right]+\sqrt{\left(\frac{F_{v}+\pi r_{i} r_{o}\left(p_{i}-p_{o}\right)}{2 \pi\left(r_{o}^{2}-r_{i}^{2}\right)}\right)^{2}+\left(\frac{3 M}{2 \pi\left(r_{o}^{3}-r_{i}^{3}\right)}\right)^{2}}
$$

Eq. I. 14

When terms are combined,
$\left(\sigma_{2}-\sigma_{3}\right)=\left[\frac{-F_{v}+3 \pi r_{i} r_{o}\left(p_{i}-p_{o}\right)+\sqrt{\left(F_{v}+\pi r_{i} r_{o}\left(p_{i}-p_{o}\right)\right)^{2}+\left(\frac{3 M\left(r_{o}^{2}-r_{i}^{2}\right)}{\left(r_{o}^{3}-r_{i}^{3}\right)}\right)^{2}}}{2 \pi\left(r_{o}^{2}-r_{i}^{2}\right)}\right]$

Eq. I. 15

By combining Eq. I. 3 and I.15, the b-value can be written as,

$$
\frac{\left(\sigma_{2}-\sigma_{3}\right)}{\left(\sigma_{1}-\sigma_{3}\right)}=\left[\frac{-F_{v}+3 \pi r_{i} r_{o}\left(p_{i}-p_{o}\right)+\sqrt{\left(F_{v}+\pi r_{i} r_{o}\left(p_{i}-p_{o}\right)\right)^{2}+\left(\frac{3 M\left(r_{o}^{2}-r_{i}^{2}\right)}{\left(r_{o}^{3}-r_{i}^{3}\right)}\right)^{2}}}{2 \sqrt{\left(F_{v}+\pi r_{i} r_{o}\left(p_{i}-p_{o}\right)\right)^{2}+\left(\frac{3 M\left(r_{o}^{2}-r_{i}^{2}\right)}{\left(r_{o}^{3}-r_{i}^{3}\right)}\right)^{2}}}\right]
$$

Eq. I. 16
or
$\frac{\left(\sigma_{2}-\sigma_{3}\right)}{\left(\sigma_{1}-\sigma_{3}\right)}=\left[\frac{-F_{v}+3 \pi r_{i} r_{o}\left(p_{i}-p_{o}\right)}{2 \sqrt{\left(F_{v}+\pi r_{i} r_{o}\left(p_{i}-p_{o}\right)\right)^{2}+\left(\frac{3 M\left(r_{o}^{2}-r_{i}^{2}\right)}{\left(r_{o}^{3}-r_{i}^{3}\right)}\right)^{2}}}+\frac{1}{2}\right]$

Eq. I. 17

Using the same substituions used when calculating the friction angle measurement error,
equation I. 17 can be simplified to become

$$
\frac{\left(\sigma_{2}-\sigma_{3}\right)}{\left(\sigma_{1}-\sigma_{3}\right)}=\left[\frac{-F_{v}+3 A\left(p_{i}-p_{o}\right)}{2 \sqrt{\left(F_{v}+A\left(p_{i}-p_{o}\right)\right)^{2}+(3 M B)^{2}}}+\frac{1}{2}\right]
$$

Eq. I. 18

$$
\begin{aligned}
A & =\pi r_{i} r_{o} \\
B & =\frac{\left(r_{o}^{2}-r_{i}^{2}\right)}{\left(r_{o}^{3}-r_{i}^{3}\right)} \\
C & =2 \pi r_{o}^{2} \\
D & =2 \pi r_{i}^{2}
\end{aligned}
$$

Using the Least Squares Method, the error in b-value can be calculated by:

$$
\partial b=\left[\left(\frac{\partial b}{\partial F_{v}} \Delta F_{v}\right)^{2}+\left(\frac{\partial b}{\partial p_{o}} \Delta p_{o}\right)^{2}+\left(\frac{\partial b}{\partial p_{i}} \Delta p_{i}\right)^{2}+\left(\frac{\partial b}{\partial M} \Delta M\right)^{2}\right]^{\frac{1}{2}}
$$

Eq. I. 19

Where

$$
\frac{\partial b}{\partial F_{v}}=\frac{-1\left(2 \sqrt{\left(F_{v}+A\left(p_{i}-p_{o}\right)\right)^{2}+(3 M B)^{2}}\right)}{\left[2 \sqrt{\left(F_{v}+A\left(p_{i}-p_{o}\right)\right)^{2}+(3 M B)^{2}}\right]^{2}}-\frac{\left(2 F_{v}+2 A\left(p_{i}-p_{o}\right)\right)\left(-F_{v}+3 A\left(p_{i}-p_{o}\right)\right)}{\sqrt[3]{\left(F_{v}+A\left(p_{i}-p_{o}\right)\right)^{2}+(3 M B)^{2}}\left[2 \sqrt{\left(F_{v}+A\left(p_{i}-p_{o}\right)\right)^{2}+(3 M B)^{2}}\right]^{2}}
$$

Eq. I. 20

$$
\frac{\partial b}{\partial p_{i}}=\frac{3 A\left(2 \sqrt{\left(F_{v}+A\left(p_{i}-p_{o}\right)\right)^{2}+(3 M B)^{2}}\right)}{\left[2 \sqrt{\left(F_{v}+A\left(p_{i}-p_{o}\right)\right)^{2}+(3 M B)^{2}}\right]^{2}}-\frac{\left(2 F_{v} A+2 A^{2}\left(p_{i}-p_{o}\right)\right)\left(-F_{v}+3 A\left(p_{i}-p_{o}\right)\right)}{\sqrt[3]{\left(F_{v}+A\left(p_{i}-p_{o}\right)\right)^{2}+(3 M B)^{2}}\left[2 \sqrt{\left(F_{v}+A\left(p_{i}-p_{o}\right)\right)^{2}+(3 M B)^{2}}\right]^{2}}
$$

Eq. I. 21

$$
\frac{\partial b}{\partial p_{o}}=\frac{-3 A\left(2 \sqrt{\left(F_{v}+A\left(p_{i}-p_{o}\right)\right)^{2}+(3 M B)^{2}}\right)}{\left[2 \sqrt{\left(F_{v}+A\left(p_{i}-p_{o}\right)\right)^{2}+(3 M B)^{2}}\right]^{2}}-\frac{\left(-2 F_{v} A+2 A^{2}\left(p_{o}-p_{i}\right)\right)\left(-F_{v}+3 A\left(p_{i}-p_{o}\right)\right)}{\sqrt[3]{\left(F_{v}+A\left(p_{i}-p_{o}\right)\right)^{2}+(3 M B)^{2}}\left[2 \sqrt{\left(F_{v}+A\left(p_{i}-p_{o}\right)\right)^{2}+(3 M B)^{2}}\right]^{2}}
$$

Eq. I. 22

$$
\frac{\partial b}{\partial M}=-\frac{\left(18 B^{2} M\right)\left(-F_{v}+3 A\left(p_{i}-p_{o}\right)\right)}{\sqrt[3]{\left(F_{v}+A\left(p_{i}-p_{o}\right)\right)^{2}+(3 M B)^{2}}\left[2 \sqrt{\left(F_{v}+A\left(p_{i}-p_{o}\right)\right)^{2}+(3 M B)^{2}}\right]^{2}}
$$

Eq. I. 23
and
$\Delta \mathrm{F}_{\mathrm{v}}$ is the estimated error in $\mathrm{F}_{\mathrm{v}}$ (vertical force)
$\Delta \mathrm{p}_{\mathrm{i}}$ is the estimated error in $\mathrm{p}_{\mathrm{i}}$ (inner pressure)
$\Delta \mathrm{p}_{\mathrm{o}}$ is the estimated error in $\mathrm{p}_{\mathrm{o}}$ (outer pressure)
$\Delta \mathrm{M}$ is the estimated error in M (moment)

## Appendix Ic --- Measurement Error Corrections on alpha for Torsion Shear Tests

Similarly, error analysis can be performed to get the error on alpha for torsion shear tests. Below is the process for this error calculation.

Combining equations H. 6 and H.12, alpha can be written in terms of moment, inner and outer radii, inner and outer pressure and vertical force. This simplifies to:
$\alpha=\frac{1}{2} \tan ^{-1}\left(\frac{3 M\left(r_{o}^{2}-r_{i}^{2}\right)}{\left(r_{o}^{3}-r_{i}^{3}\right)\left(F_{v}+\pi r_{i} r_{o}\left(p_{i}-p_{o}\right)\right)}\right)$

Using the same substitutions shown in Appendix Ia and Ib, we can set the radii as constants and get
$\alpha=\frac{1}{2} \tan ^{-1}\left(\frac{3 M B}{\left(F_{v}+A\left(p_{i}-p_{o}\right)\right)}\right)$
Eq. I. 24
where
$A=\pi r_{i} r_{o}$
$B=\frac{\left(r_{o}^{2}-r_{i}^{2}\right)}{\left(r_{o}^{3}-r_{i}^{3}\right)}$

Using the Least Squares Method, the error in alpha can be calculated by

$$
\partial \alpha=\left[\left(\frac{\partial \alpha}{\partial F_{v}} \Delta F_{v}\right)^{2}+\left(\frac{\partial \alpha}{\partial p_{o}} \Delta p_{o}\right)^{2}+\left(\frac{\partial \alpha}{\partial p_{i}} \Delta p_{i}\right)^{2}+\left(\frac{\partial \alpha}{\partial M} \Delta M\right)^{2}\right]^{\frac{1}{2}}
$$

Eq. I. 25
where
$\frac{\partial a}{\partial F_{v}}=\frac{1}{2}\left(\frac{1}{1+\left(\frac{3 M B}{\left(F_{v}+A\left(p_{i}-p_{o}\right)\right)}\right)^{2}}\right)\left(\frac{-3 M B}{\left(F_{v}+A\left(p_{i}-p_{o}\right)\right)^{2}}\right)$
Eq. I. 26
$\frac{\partial a}{\partial p_{i}}=\frac{1}{2}\left(\frac{1}{1+\left(\frac{3 M B}{\left(F_{v}+A\left(p_{i}-p_{o}\right)\right)}\right)^{2}}\right)\left(\frac{-3 M B A}{\left(F_{v}+A\left(p_{i}-p_{o}\right)\right)^{2}}\right)$
$\frac{\partial a}{\partial p_{o}}=\frac{1}{2}\left(\frac{1}{1+\left(\frac{3 M B}{\left(F_{v}+A\left(p_{i}-p_{o}\right)\right)}\right)^{2}}\right)\left(\frac{3 M B A}{\left(F_{v}+A\left(p_{i}-p_{o}\right)\right)^{2}}\right)$
$\frac{\partial a}{\partial M}=\frac{1}{2}\left(\frac{1}{1+\left(\frac{3 M B}{\left(F_{v}+A\left(p_{i}-p_{o}\right)\right)}\right)^{2}}\right)\left(\frac{3 B\left(F_{v}+A\left(p_{i}-p_{o}\right)\right)}{\left(F_{v}+A\left(p_{i}-p_{o}\right)\right)^{2}}\right)$
Eq. I. 28

Eq. I. 29
and
$\Delta F_{v}$ is the estimated error in $F_{v}$ (vertical force)
$\Delta p_{i}$ is the estimated error in $p_{i}$ (inner pressure)
$\Delta \mathrm{p}_{\mathrm{o}}$ is the estimated error in $\mathrm{p}_{\mathrm{o}}$ (outer pressure)
$\Delta \mathrm{M}$ is the estimated error in M (moment)

## Appendix Ja ---Measurement Error Corrections for True Triaxial Tests

As described in Appendix I, when doing true triaxial tests, it is important to calculate the possible measurement error on the friction angle. In the true triaxial apparatus the minor principal stress, $\sigma_{3}$, is applied by a constant cell pressure. The major and intermediate principal stress, $\sigma_{1}$ and $\sigma_{2}$, respectively, are applied by the cell pressure and the deviator load. It is necessary to calculate the error as follows.

$$
\sin \phi=\frac{\sigma_{1}-\sigma_{3}}{\sigma_{1}+\sigma_{3}}=\frac{\left(\frac{F_{v}}{A}+\sigma_{c e l l}\right)-\sigma_{c e l l}}{\left(\frac{F_{v}}{A}+\sigma_{c e l l}\right)+\sigma_{c e l l}}=\frac{F_{v}}{F_{v}+2 A \sigma_{c e l l}}
$$

Eq. J.1.
where $F_{v}$ is the vertical load, $A$ is the specimen area where the deviator load is applied and $\sigma_{\text {cell }}$ is the cell pressure.

Using the Least Squares Method (as done in Appendix I), the error in friction angle can be calculated by:

$$
\partial \sin \phi=\left(\left(\frac{\partial \sin \phi}{\partial F_{v}} \Delta F_{v}\right)^{2}+\left(\frac{\partial \sin \phi}{\partial A} \Delta A\right)^{2}+\left(\frac{\partial \sin \phi}{\partial \sigma_{\text {cel }}} \Delta \sigma_{\text {cell }}\right)^{2}\right)^{\frac{1}{2}}
$$

where

$$
\frac{\partial \sin \phi}{\partial F_{v}}=\frac{2 A \sigma_{\text {cell }}}{\left(F+2 A \sigma_{c e l l}\right)^{2}}
$$

Eq. J. 3
$\frac{\partial \sin \phi}{\partial A}=\frac{-2 F_{v} \sigma_{c e l l}}{\left(F+2 A \sigma_{\text {cel }}\right)^{2}}$
Eq. J. 4
$\frac{\partial \sin \phi}{\partial \sigma_{\text {cel }}}=\frac{-2 F_{v} A}{\left(F+2 A \sigma_{\text {cel }}\right)^{2}}$
Eq. J. 5
where
$\Delta F_{v}$ is the estimated error in $F_{v}$
$\Delta \mathrm{A}$ is the estimated error in A
$\Delta \sigma_{\text {cell }}$ is the estimated error in $\sigma_{\text {cell }}$

The derivative of $\sin \varphi$ gives,
$\partial \sin \phi=\cos \phi \Delta \phi$
Eq. J. 6

The error can be calculated by

$$
\Delta \phi=\frac{180}{\pi \cos (\phi)}\left(\left(\frac{\partial \sin \phi}{\partial F_{v}} \Delta F_{v}\right)^{2}+\left(\frac{\partial \sin \phi}{\partial A} \Delta A\right)^{2}+\left(\frac{\partial \sin \phi}{\partial \sigma_{\text {cell }}} \Delta \sigma_{\text {cell }}\right)^{2}\right)^{\frac{1}{2}}
$$

Eq. J. 7

## Appendix Jb --- Measurement Error Corrections on b-value for True Triaxial Tests

In a similar way as the error was calculated for friction angle, the error on the $b$-value can also be calculated by following a similar process. For the true triaxial tests, the b-value, ( $\sigma_{2}{ }^{-}$ $\left.\sigma_{3}\right) /\left(\sigma_{1}-\sigma_{3}\right)$ can be written out in terms of $F_{v}, F_{h}, A_{v}, A_{h}$ and $\sigma_{\text {cell }}$ as follows:

$$
b=\frac{\left(\sigma_{2}-\sigma_{3}\right)}{\left(\sigma_{1}-\sigma_{3}\right)}=\frac{\left(\frac{F_{h}}{A_{v}}+\sigma_{\text {cell }}\right)-\sigma_{\text {cell }}}{\left(\frac{F_{v}}{A_{h}}+\sigma_{\text {cell }}\right)-\sigma_{\text {cell }}}=\frac{\frac{F_{h}}{A_{v}}}{\frac{F_{v}}{A_{h}}}=\frac{F_{h} A_{h}}{F_{v} A_{v}}
$$

Eq. J. 8

Therefore, the error in b-value using the Least Squares Method can be calculated by

$$
\partial b=\left(\left(\frac{\partial b}{\partial F_{h}} \Delta F_{h}\right)^{2}+\left(\frac{\partial b}{\partial F_{v}} \Delta F_{v}\right)^{2}+\left(\frac{\partial b}{\partial A_{h}} \Delta A_{h}\right)^{2}+\left(\frac{\partial b}{\partial A_{v}} \Delta A_{v}\right)^{2}\right)^{\frac{1}{2}}
$$

Eq. J. 9
where

$$
\begin{aligned}
& \frac{\partial b}{\partial F_{h}}=\frac{A_{h} F_{v} A_{v}}{\left(F_{v} A_{v}\right)^{2}} \\
& \frac{\partial b}{\partial F_{v}}=-\frac{A_{v} F_{h} A_{h}}{\left(F_{v} A_{v}\right)^{2}}
\end{aligned}
$$

Eq. J. 10

Eq. J. 11

$$
\begin{gather*}
\frac{\partial b}{\partial A_{h}}=\frac{F_{h} F_{v} A_{v}}{\left(F_{v} A_{v}\right)^{2}} \\
\frac{\partial b}{\partial A_{v}}=-\frac{F_{v} F_{h} A_{v}}{\left(F_{v} A_{v}\right)^{2}}
\end{gather*}
$$

Eq. J. 13
and
$\Delta F_{v}$ is the estimated error in vertical force, $F_{v}$
$\Delta \mathrm{F}_{\mathrm{h}}$ is the estimated error in horiziontal force, $\mathrm{F}_{\mathrm{h}}$
$\Delta A_{v}$ is the estimated error in vertical area, $\mathrm{A}_{\mathrm{v}}$ (length times height)
$\Delta \mathrm{A}_{\mathrm{h}}$ is the estimated error in vertical area, $\mathrm{A}_{\mathrm{h}}$ (length times width)

## Appendix K --- Polynomial Regression Explanation (after Chapra and Canale 2010).

In order to represent certain engineering data which cannot be captured with a straight line, a curve can be used to fit the data. A method to get this curve is to fit polynomial expressions to the data using a polynomial regression. By using the Least Squares procedure, the data can be fit to a higher order polynomial. The equations used that can be set to zero are shown below. In the case of a fourth order polynomial with 5 unknowns, 5 equations are required to solve the matrix. This is the case when using the fourth order Pietrusczack (2011) equation.

$$
\begin{aligned}
& (n) a_{0}+\left(\sum x_{i}\right) a_{1}+\left(\sum x_{i}^{2}\right) a_{2}+\left(\sum x_{i}^{3}\right) a_{3}+\left(\sum x_{i}^{4}\right) a_{4}=\left(\sum y_{i}\right) \\
& \left(\sum x_{i}\right) a_{0}+\left(\sum x_{i}^{2}\right) a_{1}+\left(\sum x_{i}^{3}\right) a_{2}+\left(\sum x_{i}^{4}\right) a_{3}+\left(\sum x_{i}^{5}\right) a_{4}=\left(\sum x_{i} y_{i}\right) \\
& \left(\sum x_{i}^{2}\right) a_{0}+\left(\sum x_{i}^{3}\right) a_{1}+\left(\sum x_{i}^{4}\right) a_{2}+\left(\sum x_{i}^{5}\right) a_{3}+\left(\sum x_{i}^{6}\right) a_{4}=\left(\sum x_{i}^{2} y_{i}\right) \\
& \left(\sum x_{i}^{3}\right) a_{0}+\left(\sum x_{i}^{4}\right) a_{1}+\left(\sum x_{i}^{5}\right) a_{2}+\left(\sum x_{i}^{6}\right) a_{3}+\left(\sum x_{i}^{7}\right) a_{4}=\left(\sum x_{i}^{3} y_{i}\right) \\
& \left(\sum x_{i}^{4}\right) a_{0}+\left(\sum x_{i}^{5}\right) a_{1}+\left(\sum x_{i}^{6}\right) a_{2}+\left(\sum x_{i}^{7}\right) a_{3}+\left(\sum x_{i}^{8}\right) a_{4}=\left(\sum x_{i}^{4} y_{i}\right) \\
& \left(\sum x_{i}^{5}\right) a_{0}+\left(\sum x_{i}^{6}\right) a_{1}+\left(\sum x_{i}^{7}\right) a_{2}+\left(\sum x_{i}^{8}\right) a_{3}+\left(\sum x_{i}^{9}\right) a_{4}=\left(\sum x_{i}^{5} y_{i}\right)
\end{aligned}
$$

Eq. K. 1
where n is the number of data points used, $\mathrm{a}_{1}$ through $\mathrm{a}_{4}$ are the coefficients of the polynomial expression that is being solved.

Solving the coefficients of an $m$ th order polynomial is equivalent to solving a system of $m+1$ simultaneous linear equations. Therefore, for this case, the standard error is
$s_{y / x}=\sqrt{\frac{S_{r}}{n-(m+1)}}$
Eq. K. 2
$\mathrm{S}_{\mathrm{r}}$ is the sums of the residuals and n is the number of points used. As can be seen in the equation, when using only 5 points, a $4^{\text {th }}$ order equation cannot be attained. For the particular case of this thesis (in Chapter 11), Equation K. 3 was separated so that
$\eta_{f}=\eta_{0}\left[1+\Omega_{1}\left(1-3 l_{2}^{2}\right)+a_{1}\left(\Omega_{1}\left(1-3 l_{2}^{2}\right)\right)^{2}+a_{2}\left(\Omega_{1}\left(1-3 l_{2}^{2}\right)\right)^{3}+a_{3}\left(\Omega_{1}\left(1-3 l_{2}^{2}\right)\right)^{4}+\ldots\right]$
Eq. K. 3
$\eta_{f}=\eta_{0}+\eta_{0} \Omega_{1} x+\eta_{0} \Omega_{1} a_{1} x^{2}+\eta_{0} \Omega_{1} a_{2} x^{3}+\eta_{0} \Omega_{1} a_{3} x^{4}$
where
$\eta_{f}=\left(\frac{I_{1}^{3}}{I_{3}}-27\right)\left(\frac{I_{1}}{p_{a}}\right)^{m}$
Eq. K. 4
and
$x=\left(1-3 l_{2}^{2}\right)$

By plugging in the corresponding values of points from the torsion shear results and solving the matrix, all the parameters can be determined.

## Appendix L-Test Data Sheets

$$
\text { Torsion Shear Test on Fine Nevada Sand with a Target Void Ratio }=0.530\left(D_{r}=91.28 \%\right)
$$

| Test No.: | $1^{*}$ |
| :--- | :--- |
| Test Date: | $5 / 10 / 10$ |
| Initial Height, $\mathrm{h}_{\mathrm{i}}:$ | 39.85 cm |
| Initial Void Ratio, e: | 0.510 |
| Final Mean Princpal Stress, $\sigma_{\mathrm{m}}:$ | 98.0 kPa |
| Max Friction Angle, $\phi:$ | 38.0 deg |
| b-value at failure: | 0.00 |
| Stress direction at failure, $\alpha:$ | 0 deg |$\quad$| Shear Band Notes |
| :--- |
| Point of Observation: |
| none observed. |
| Inclination (from Vertical) |
| $\mathrm{n} / \mathrm{a}$ |
|  |


| Point (No.) | $\begin{gathered} \varepsilon_{\mathrm{z}} \\ (\%) \\ \hline \end{gathered}$ | $\begin{gathered} \varepsilon_{\mathrm{r}} \\ (\%) \end{gathered}$ | $\begin{gathered} \varepsilon_{\theta} \\ (\%) \\ \hline \end{gathered}$ | $\begin{gathered} \varepsilon_{\mathrm{v}} \\ (\%) \\ \hline \end{gathered}$ | $\begin{array}{r} \gamma_{\theta z} \\ (\%) \\ \hline \end{array}$ | $\begin{gathered} \sigma_{z} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \sigma_{\mathrm{r}} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \sigma_{\theta} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \tau_{\theta z} \\ (\mathrm{kPa}) \end{gathered}$ | b | $\begin{gathered} \alpha \\ \left({ }^{\circ}\right) \end{gathered}$ | $\begin{gathered} \varphi \\ \left({ }^{\circ}\right) \end{gathered}$ |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 1 | 0.000 | 0.000 | 0.000 | 0.000 | 0.000 | 89.203 | 101.239 | 101.540 | 0.000 | 0.024 | 0.000 | -3.708 |
| 2 | 0.013 | -0.005 | -0.003 | 0.005 | 0.000 | 103.472 | 94.687 | 94.382 | 0.000 | 0.034 | 0.000 | 2.633 |
| 3 | 0.027 | -0.013 | -0.008 | 0.006 | 0.000 | 115.123 | 89.179 | 88.589 | 0.000 | 0.022 | 0.000 | 7.484 |
| 4 | 0.045 | -0.022 | -0.011 | 0.011 | 0.000 | 125.568 | 84.190 | 84.404 | 0.000 | -0.005 | 0.000 | 11.306 |
| 5 | 0.063 | -0.037 | -0.016 | 0.010 | 0.000 | 134.376 | 79.958 | 79.996 | 0.000 | -0.001 | 0.000 | 14.695 |
| 6 | 0.083 | -0.051 | -0.022 | 0.010 | 0.000 | 142.313 | 76.114 | 75.851 | 0.000 | 0.004 | 0.000 | 17.737 |
| 7 | 0.106 | -0.067 | -0.028 | 0.011 | 0.000 | 149.787 | 72.417 | 72.202 | 0.000 | 0.003 | 0.000 | 20.457 |
| 8 | 0.124 | -0.081 | -0.033 | 0.010 | 0.000 | 154.925 | 70.114 | 70.159 | 0.000 | -0.001 | 0.000 | 22.123 |
| 9 | 0.141 | -0.094 | -0.038 | 0.009 | 0.000 | 158.391 | 68.226 | 67.706 | 0.000 | 0.006 | 0.000 | 23.646 |
| 10 | 0.173 | -0.118 | -0.046 | 0.008 | 0.000 | 164.618 | 65.388 | 65.225 | 0.000 | 0.002 | 0.000 | 25.622 |
| 11 | 0.207 | -0.144 | -0.057 | 0.007 | 0.000 | 169.411 | 62.910 | 62.555 | 0.000 | 0.003 | 0.000 | 27.429 |
| 12 | 0.246 | -0.174 | -0.070 | 0.003 | 0.000 | 173.827 | 60.697 | 60.543 | 0.000 | 0.001 | 0.000 | 28.905 |
| 13 | 0.291 | -0.208 | -0.085 | -0.002 | 0.000 | 177.997 | 58.750 | 58.755 | 0.000 | 0.000 | 0.000 | 30.243 |
| 14 | 0.346 | -0.252 | -0.105 | -0.011 | 0.000 | 181.373 | 56.763 | 56.568 | 0.000 | 0.002 | 0.000 | 31.636 |
| 15 | 0.396 | -0.293 | -0.125 | -0.022 | 0.000 | 184.445 | 55.504 | 55.294 | 0.000 | 0.002 | 0.000 | 32.596 |
| 16 | 0.459 | -0.346 | -0.151 | -0.038 | 0.000 | 187.059 | 54.105 | 53.748 | 0.000 | 0.003 | 0.000 | 33.614 |
| 17 | 0.520 | -0.398 | -0.178 | -0.056 | 0.000 | 189.303 | 52.945 | 52.791 | 0.000 | 0.001 | 0.000 | 34.324 |
| 18 | 0.589 | -0.456 | -0.211 | -0.078 | 0.000 | 191.351 | 51.954 | 51.621 | 0.000 | 0.002 | 0.000 | 35.106 |
| 19 | 0.667 | -0.525 | -0.248 | -0.106 | 0.000 | 193.114 | 50.930 | 50.840 | 0.000 | 0.001 | 0.000 | 35.676 |
| 20 | 0.746 | -0.595 | -0.288 | -0.137 | 0.000 | 194.448 | 50.133 | 49.917 | 0.000 | 0.001 | 0.000 | 36.261 |
| 21 | 0.820 | -0.661 | -0.325 | -0.166 | 0.000 | 195.865 | 49.579 | 49.712 | 0.000 | -0.001 | 0.000 | 36.523 |
| 22 | 0.894 | -0.726 | -0.368 | -0.201 | 0.000 | 196.581 | 49.025 | 48.607 | 0.000 | 0.003 | 0.000 | 37.122 |
| 23 | 0.976 | -0.802 | -0.411 | -0.236 | 0.000 | 197.565 | 48.516 | 48.421 | 0.000 | 0.001 | 0.000 | 37.323 |
| 24 | 1.075 | -0.891 | -0.464 | -0.281 | 0.000 | 198.346 | 47.935 | 47.669 | 0.000 | 0.002 | 0.000 | 37.769 |
| 25 | 1.149 | -0.959 | -0.505 | -0.315 | 0.000 | 198.566 | 47.649 | 47.339 | 0.000 | 0.002 | 0.000 | 37.951 |
| 26 | 1.239 | -1.042 | -0.553 | -0.356 | 0.000 | 199.217 | 47.377 | 47.407 | 0.000 | 0.000 | 0.000 | 37.992 |
| 27 | 1.350 | -1.146 | -0.608 | -0.404 | 0.000 | 199.582 | 47.232 | 47.402 | 0.000 | -0.001 | 0.000 | 38.036 |

$$
\text { Torsion Shear Test on Fine Nevada Sand with a Target Void Ratio }=0.530\left(D_{r}=91.28 \%\right)
$$

| Test No.: | 2 |
| :--- | :---: |
| Test Date: | $10 / 25 / 11$ |
| Initial Height, $\mathrm{h}_{\mathrm{i}}$ : | 39.96 cm |
| Initial Void Ratio, e: | 0.529 |
| Final Mean Princpal Stress, $\sigma_{\mathrm{m}}:$ | 101.5 kPa |
| Max Friction Angle, $\phi:$ | 56.9 deg |
| b-value at failure: | 0.75 |
| Stress direction at failure, $\alpha:$ | 0 deg |


| Shear Band Notes | 33 |
| :--- | :--- |
| Point of Observation: |  |
|  |  |
| Inclination (from Vertical) |  |
| $7 @ 65^{\circ}, 67^{\circ}, 68^{\circ}, 75^{\circ}, 90^{\circ}$ |  |
| Failure Notes: |  |
| deep zig-zag patterns and vertical trough |  |


| Point (No.) | $\begin{gathered} \varepsilon_{\mathrm{z}} \\ (\%) \end{gathered}$ | $\begin{gathered} \varepsilon_{\mathrm{r}} \\ (\%) \end{gathered}$ | $\begin{gathered} \varepsilon_{\theta} \\ (\%) \end{gathered}$ | $\begin{gathered} \varepsilon_{\mathrm{v}} \\ (\%) \end{gathered}$ | $\begin{array}{r} \gamma_{\theta z} \\ (\%) \\ \hline \end{array}$ | $\begin{gathered} \sigma_{z} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \sigma_{\mathrm{r}} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \sigma_{\theta} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \tau_{\theta z} \\ (\mathrm{kPa}) \end{gathered}$ | b | $\begin{gathered} \alpha \\ \left(^{( }\right) \\ \hline \end{gathered}$ | $\begin{gathered} \varphi \\ \left({ }^{( }\right) \\ \hline \end{gathered}$ |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 1 | 0.000 | -0.001 | 0.000 | -0.001 | 0.000 | 100.540 | 101.315 | 101.401 | 0.000 | 0.900 | 0.000 | 0.244 |
| 2 | 0.001 | 0.002 | -0.001 | 0.002 | 0.000 | 102.952 | 102.105 | 98.779 | 0.000 | 0.797 | 0.000 | 1.185 |
| 3 | 0.038 | 0.031 | -0.043 | 0.026 | 0.000 | 134.866 | 114.985 | 53.710 | 0.000 | 0.755 | 0.000 | 25.491 |
| 4 | 0.048 | 0.034 | -0.055 | 0.027 | 0.000 | 139.094 | 116.736 | 47.973 | 0.000 | 0.755 | 0.000 | 29.150 |
| 5 | 0.052 | 0.036 | -0.061 | 0.027 | 0.000 | 140.615 | 117.426 | 45.427 | 0.000 | 0.756 | 0.000 | 30.773 |
| 6 | 0.058 | 0.037 | -0.068 | 0.026 | 0.000 | 142.514 | 118.022 | 43.310 | 0.000 | 0.753 | 0.000 | 32.267 |
| 7 | 0.064 | 0.038 | -0.076 | 0.025 | 0.000 | 144.165 | 118.667 | 41.107 | 0.000 | 0.753 | 0.000 | 33.797 |
| 8 | 0.069 | 0.040 | -0.084 | 0.024 | 0.000 | 145.559 | 119.215 | 39.343 | 0.000 | 0.752 | 0.000 | 35.061 |
| 9 | 0.074 | 0.041 | -0.093 | 0.022 | 0.000 | 146.719 | 119.658 | 37.753 | 0.000 | 0.752 | 0.000 | 36.206 |
| 10 | 0.080 | 0.044 | -0.103 | 0.021 | 0.000 | 147.757 | 120.074 | 36.127 | 0.000 | 0.752 | 0.000 | 37.378 |
| 11 | 0.088 | 0.047 | -0.115 | 0.020 | 0.000 | 148.922 | 120.576 | 34.325 | 0.000 | 0.753 | 0.000 | 38.710 |
| 12 | 0.094 | 0.050 | -0.125 | 0.018 | 0.000 | 149.756 | 120.959 | 33.094 | 0.000 | 0.753 | 0.000 | 39.645 |
| 13 | 0.100 | 0.051 | -0.136 | 0.016 | 0.000 | 150.734 | 121.301 | 31.998 | 0.000 | 0.752 | 0.000 | 40.525 |
| 14 | 0.107 | 0.055 | -0.148 | 0.013 | 0.000 | 151.594 | 121.675 | 30.675 | 0.000 | 0.753 | 0.000 | 41.560 |
| 15 | 0.114 | 0.058 | -0.161 | 0.010 | 0.000 | 152.316 | 121.932 | 29.782 | 0.000 | 0.752 | 0.000 | 42.291 |
| 16 | 0.127 | 0.066 | -0.188 | 0.005 | 0.000 | 153.624 | 122.437 | 27.975 | 0.000 | 0.752 | 0.000 | 43.781 |
| 17 | 0.136 | 0.071 | -0.207 | 0.001 | 0.000 | 154.231 | 122.764 | 26.792 | 0.000 | 0.753 | 0.000 | 44.749 |
| 18 | 0.151 | 0.080 | -0.236 | -0.005 | 0.000 | 155.226 | 123.132 | 25.430 | 0.000 | 0.753 | 0.000 | 45.929 |
| 19 | 0.158 | 0.083 | -0.250 | -0.009 | 0.000 | 155.838 | 123.366 | 24.786 | 0.000 | 0.752 | 0.000 | 46.515 |
| 20 | 0.166 | 0.087 | -0.266 | -0.014 | 0.000 | 156.627 | 123.617 | 24.090 | 0.000 | 0.751 | 0.000 | 47.172 |
| 21 | 0.182 | 0.096 | -0.303 | -0.024 | 0.000 | 157.515 | 123.994 | 22.641 | 0.000 | 0.751 | 0.000 | 48.474 |
| 22 | 0.193 | 0.103 | -0.328 | -0.032 | 0.000 | 158.029 | 124.180 | 22.112 | 0.000 | 0.751 | 0.000 | 48.982 |
| 23 | 0.207 | 0.113 | -0.362 | -0.043 | 0.000 | 158.190 | 124.475 | 21.017 | 0.000 | 0.754 | 0.000 | 49.947 |
| 24 | 0.227 | 0.125 | -0.407 | -0.055 | 0.000 | 159.225 | 124.688 | 20.236 | 0.000 | 0.752 | 0.000 | 50.758 |
| 25 | 0.252 | 0.141 | -0.467 | -0.074 | 0.000 | 160.225 | 125.137 | 19.228 | 0.000 | 0.751 | 0.000 | 51.786 |
| 26 | 0.255 | 0.142 | -0.474 | -0.077 | 0.000 | 160.395 | 125.169 | 19.025 | 0.000 | 0.751 | 0.000 | 51.992 |
| 27 | 0.309 | 0.176 | -0.609 | -0.123 | 1.000 | 161.312 | 125.561 | 17.230 | 0.000 | 0.752 | 0.000 | 53.804 |
| 28 | 0.317 | 0.180 | -0.627 | -0.130 | 2.000 | 161.648 | 125.609 | 17.113 | 0.000 | 0.751 | 0.000 | 53.953 |
| 29 | 0.375 | 0.216 | -0.778 | -0.187 | 3.000 | 162.832 | 126.107 | 15.597 | 0.000 | 0.751 | 0.000 | 55.606 |
| 30 | 0.389 | 0.225 | -0.816 | -0.202 | 4.000 | 163.065 | 126.203 | 15.351 | 0.000 | 0.750 | 0.000 | 55.885 |
| 31 | 0.408 | 0.238 | -0.866 | -0.221 | 5.000 | 163.315 | 126.283 | 15.023 | 0.000 | 0.750 | 0.000 | 56.256 |
| 32 | 0.424 | 0.248 | -0.911 | -0.239 | 6.000 | 163.487 | 126.378 | 14.735 | 0.000 | 0.751 | 0.000 | 56.579 |
| 33 | 0.441 | 0.258 | -0.956 | -0.256 | 7.000 | 163.736 | 126.435 | 14.452 | 0.000 | 0.750 | 0.000 | 56.907 |

$$
\text { Torsion Shear Test on Fine Nevada Sand with a Target Void Ratio }=0.530\left(\mathrm{D}_{\mathrm{r}}=91.28 \%\right)
$$

| Test No.: | 3 |
| :--- | :--- |
| Test Date: | $7 / 28 / 11$ |
| Initial Height, $\mathrm{h}_{\mathrm{i}}:$ | 40.15 cm |
| Initial Void Ratio, e: | 0.523 |
| Final Mean Princpal Stress, $\sigma_{\mathrm{m}}:$ | 100.8 kPa |
| Max Friction Angle, $\phi:$ | 39.9 deg |
| b-value at failure: | 0.00 |
| Stress direction at failure, $\alpha:$ | 22.4 deg |


| Shear Band Notes |
| :--- |
| Point of Observation: |
| Inclination (from Vertical) |
| $19^{\circ}, 20^{\circ}, 23^{\circ}$ |
|  |
| Failure Notes: |
| One thick shear band wrapping around |


| Point (No.) | $\begin{gathered} \varepsilon_{\mathrm{z}} \\ (\%) \end{gathered}$ | $\begin{gathered} \varepsilon_{\mathrm{r}} \\ (\%) \end{gathered}$ | $\begin{gathered} \varepsilon_{\theta} \\ (\%) \end{gathered}$ | $\begin{gathered} \varepsilon_{\mathrm{v}} \\ (\%) \end{gathered}$ | $\begin{aligned} & \gamma_{\theta z} \\ & (\%) \end{aligned}$ | $\begin{gathered} \sigma_{z} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \sigma_{\mathrm{r}} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \sigma_{\theta} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \tau_{\theta z} \\ (\mathrm{kPa}) \end{gathered}$ | b | $\begin{gathered} \alpha \\ \left({ }^{\circ}\right) \end{gathered}$ | $\begin{gathered} \varphi \\ \left(^{( }\right) \end{gathered}$ |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 1 | 0.000 | 0.000 | 0.000 | 0.000 | 0.017 | 102.617 | 100.056 | 101.076 | 2.500 | 0.158 | 36.436 | 1.472 |
| 2 | 0.001 | -0.001 | 0.000 | 0.000 | 0.018 | 105.370 | 98.310 | 99.539 | 4.308 | 0.102 | 27.957 | 2.910 |
| 3 | 0.030 | -0.062 | 0.012 | -0.020 | 0.245 | 155.886 | 65.996 | 81.373 | 40.069 | 0.019 | 23.541 | 27.465 |
| 4 | 0.050 | -0.089 | 0.006 | -0.033 | 0.327 | 160.685 | 62.858 | 79.788 | 43.191 | 0.015 | 23.439 | 29.482 |
| 5 | 0.095 | -0.132 | -0.013 | -0.050 | 0.421 | 165.095 | 59.909 | 77.787 | 44.981 | 0.009 | 22.929 | 31.074 |
| 6 | 0.119 | -0.162 | -0.023 | -0.066 | 0.499 | 167.730 | 58.287 | 76.810 | 46.285 | 0.007 | 22.758 | 32.046 |
| 7 | 0.245 | -0.315 | -0.083 | -0.153 | 0.786 | 174.531 | 53.872 | 74.373 | 50.371 | 0.003 | 22.583 | 34.802 |
| 8 | 0.331 | -0.417 | -0.130 | -0.216 | 0.980 | 177.238 | 52.196 | 73.702 | 51.844 | 0.000 | 22.521 | 35.727 |
| 9 | 0.394 | -0.501 | -0.166 | -0.273 | 1.135 | 179.002 | 50.970 | 72.864 | 52.582 | -0.002 | 22.368 | 36.387 |
| 10 | 0.427 | -0.535 | -0.185 | -0.293 | 1.194 | 179.265 | 50.892 | 72.979 | 53.083 | -0.001 | 22.484 | 36.553 |
| 11 | 0.488 | -0.612 | -0.221 | -0.345 | 1.328 | 180.367 | 50.183 | 72.759 | 53.852 | -0.002 | 22.513 | 36.975 |
| 12 | 0.518 | -0.650 | -0.240 | -0.372 | 1.397 | 180.871 | 49.869 | 72.450 | 53.998 | -0.002 | 22.444 | 37.164 |
| 13 | 0.582 | -0.729 | -0.280 | -0.427 | 1.537 | 181.693 | 49.404 | 72.178 | 54.587 | -0.001 | 22.455 | 37.526 |
| 14 | 0.619 | -0.782 | -0.304 | -0.467 | 1.632 | 182.254 | 49.071 | 72.129 | 54.969 | -0.002 | 22.476 | 37.713 |
| 15 | 0.680 | -0.855 | -0.344 | -0.518 | 1.751 | 182.768 | 48.674 | 71.740 | 55.317 | -0.001 | 22.449 | 38.014 |
| 16 | 0.736 | -0.928 | -0.381 | -0.572 | 1.881 | 183.353 | 48.368 | 71.496 | 55.707 | -0.001 | 22.443 | 38.279 |
| 17 | 0.796 | -1.004 | -0.421 | -0.629 | 2.011 | 183.844 | 47.963 | 71.296 | 56.065 | -0.001 | 22.447 | 38.512 |
| 18 | 0.823 | -1.042 | -0.440 | -0.658 | 2.078 | 183.986 | 47.809 | 71.235 | 56.200 | -0.001 | 22.455 | 38.594 |
| 19 | 0.894 | -1.130 | -0.489 | -0.725 | 2.228 | 184.639 | 47.512 | 71.215 | 56.515 | -0.002 | 22.450 | 38.745 |
| 20 | 0.916 | -1.161 | -0.505 | -0.749 | 2.282 | 184.570 | 47.344 | 71.174 | 56.580 | -0.003 | 22.470 | 38.785 |
| 21 | 0.966 | -1.224 | -0.538 | -0.796 | 2.388 | 184.939 | 47.203 | 71.064 | 56.786 | -0.002 | 22.462 | 38.920 |
| 22 | 1.029 | -1.303 | -0.584 | -0.858 | 2.525 | 185.322 | 46.975 | 70.738 | 56.999 | -0.001 | 22.426 | 39.141 |
| 23 | 1.122 | -1.417 | -0.649 | -0.944 | 2.722 | 185.793 | 46.655 | 70.203 | 56.497 | -0.003 | 22.175 | 39.156 |
| 24 | 1.222 | -1.556 | -0.727 | -1.061 | 2.957 | 186.050 | 46.337 | 70.052 | 57.610 | 0.000 | 22.404 | 39.673 |
| 25 | 1.225 | -1.560 | -0.729 | -1.064 | 2.965 | 186.037 | 46.338 | 70.087 | 57.605 | 0.000 | 22.408 | 39.657 |
| 26 | 1.265 | -1.615 | -0.760 | -1.110 | 3.063 | 186.203 | 46.190 | 70.278 | 57.777 | -0.001 | 22.454 | 39.656 |
| 27 | 1.322 | -1.681 | -0.799 | -1.158 | 3.168 | 186.539 | 46.121 | 70.460 | 57.880 | -0.003 | 22.461 | 39.634 |
| 28 | 1.358 | -1.732 | -0.827 | -1.201 | 3.259 | 186.787 | 46.034 | 70.483 | 57.986 | -0.003 | 22.459 | 39.673 |
| 29 | 1.423 | -1.808 | -0.872 | -1.257 | 3.381 | 186.943 | 45.944 | 70.434 | 58.065 | -0.003 | 22.453 | 39.728 |
| 30 | 1.466 | -1.868 | -0.904 | -1.306 | 3.486 | 187.282 | 45.883 | 70.480 | 58.091 | -0.004 | 22.424 | 39.728 |
| 31 | 1.501 | -1.909 | -0.929 | -1.337 | 3.554 | 187.143 | 45.859 | 70.313 | 58.171 | -0.003 | 22.440 | 39.823 |
| 32 | 1.545 | -1.964 | -0.963 | -1.382 | 3.653 | 187.045 | 45.869 | 70.297 | 58.245 | -0.002 | 22.468 | 39.857 |
| 33 | 1.620 | -2.057 | -1.020 | -1.456 | 3.813 | 187.571 | 45.754 | 70.368 | 58.244 | -0.004 | 22.412 | 39.839 |

$$
\text { Torsion Shear Test on Fine Nevada Sand with a Target Void Ratio = } 0.530\left(\mathrm{D}_{\mathrm{r}}=91.28 \%\right)
$$

| Test No.: | $4^{*}$ |
| :--- | :---: |
| Test Date: | $8 / 1 / 10$ |
| Initial Height, $\mathrm{h}_{\mathrm{i}}:$ | 40 cm |
| Initial Void Ratio, e: | 0.548 |
| Final Mean Princpal Stress, $\sigma_{\mathrm{m}}:$ | 97.5 kPa |
| Max Friction Angle, $\phi:$ | 46.5 deg |
| b-value at failure: | 0.27 |
| Stress direction at failure, $\alpha:$ | 24.0 deg |


| Shear Band Notes <br> Point of Observation: |  |
| :--- | :--- |
| Inclination (from Vertical) <br> along middle at $40^{\circ}$ and at $90^{\circ}$ |  |
| Failure Notes: <br> shearing | 30 |


| Point (No.) | $\begin{gathered} \varepsilon_{\mathrm{z}} \\ (\%) \end{gathered}$ | $\begin{gathered} \varepsilon_{\mathrm{r}} \\ (\%) \end{gathered}$ | $\begin{gathered} \varepsilon_{\theta} \\ (\%) \\ \hline \end{gathered}$ | $\begin{gathered} \varepsilon_{\mathrm{v}} \\ (\%) \\ \hline \end{gathered}$ | $\begin{aligned} & \gamma_{\theta z} \\ & (\%) \\ & \hline \end{aligned}$ | $\begin{gathered} \sigma_{z} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \sigma_{\mathrm{r}} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \sigma_{\theta} \\ (\mathrm{kPa}) \\ \hline \end{gathered}$ | $\begin{gathered} \tau_{\theta z} \\ (\mathrm{kPa}) \end{gathered}$ | b | $\begin{gathered} \alpha \\ \left({ }^{\circ}\right) \\ \hline \end{gathered}$ | $\begin{gathered} \varphi \\ \left({ }^{\circ}\right) \\ \hline \end{gathered}$ |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 1 | 0.000 | 0.000 | 0.000 | 0.000 | -0.003 | 89.995 | 101.394 | 101.498 | 0.779 | 0.986 | 86.142 | 3.475 |
| 2 | 0.001 | -0.001 | -0.001 | -0.001 | -0.003 | 90.825 | 100.594 | 100.006 | 2.440 | 0.998 | 76.003 | 3.123 |
| 3 | 0.022 | -0.008 | -0.008 | 0.006 | 0.006 | 111.812 | 93.042 | 87.841 | 17.397 | 0.339 | 27.717 | 12.218 |
| 4 | 0.026 | -0.007 | -0.010 | 0.009 | 0.018 | 116.891 | 91.202 | 84.646 | 21.485 | 0.322 | 26.558 | 15.460 |
| 5 | 0.027 | -0.004 | -0.012 | 0.011 | 0.039 | 121.624 | 89.398 | 81.619 | 25.853 | 0.313 | 26.136 | 18.763 |
| 6 | 0.028 | 0.000 | -0.014 | 0.013 | 0.071 | 125.426 | 87.968 | 79.595 | 30.732 | 0.310 | 26.645 | 21.960 |
| 7 | 0.030 | 0.005 | -0.021 | 0.013 | 0.113 | 129.511 | 86.551 | 77.222 | 34.566 | 0.306 | 26.449 | 24.789 |
| 8 | 0.037 | 0.008 | -0.034 | 0.011 | 0.176 | 134.327 | 84.728 | 74.169 | 38.198 | 0.299 | 25.891 | 27.800 |
| 9 | 0.048 | 0.015 | -0.055 | 0.008 | 0.295 | 138.767 | 82.993 | 71.321 | 41.575 | 0.294 | 25.477 | 30.638 |
| 10 | 0.062 | 0.021 | -0.085 | -0.001 | 0.422 | 143.087 | 81.333 | 68.975 | 44.431 | 0.287 | 25.086 | 33.069 |
| 11 | 0.078 | 0.028 | -0.122 | -0.016 | 0.554 | 146.508 | 80.003 | 66.540 | 46.670 | 0.284 | 24.706 | 35.234 |
| 12 | 0.103 | 0.032 | -0.171 | -0.036 | 0.697 | 149.177 | 78.905 | 64.770 | 48.597 | 0.282 | 24.514 | 36.991 |
| 13 | 0.161 | 0.037 | -0.304 | -0.106 | 1.031 | 154.148 | 76.874 | 61.193 | 51.625 | 0.278 | 24.002 | 40.177 |
| 14 | 0.198 | 0.034 | -0.379 | -0.147 | 1.210 | 156.793 | 76.103 | 61.151 | 53.365 | 0.271 | 24.068 | 41.115 |
| 15 | 0.248 | 0.022 | -0.451 | -0.181 | 1.335 | 157.886 | 75.680 | 60.103 | 53.888 | 0.271 | 23.892 | 41.880 |
| 16 | 0.259 | 0.022 | -0.485 | -0.203 | 1.410 | 158.183 | 75.473 | 59.581 | 54.144 | 0.272 | 23.840 | 42.263 |
| 17 | 0.293 | 0.015 | -0.547 | -0.239 | 1.537 | 159.183 | 75.181 | 59.224 | 55.147 | 0.271 | 23.907 | 42.963 |
| 18 | 0.309 | 0.016 | -0.601 | -0.275 | 1.646 | 160.181 | 74.843 | 58.721 | 55.644 | 0.270 | 23.822 | 43.469 |
| 19 | 0.326 | 0.016 | -0.641 | -0.300 | 1.728 | 160.591 | 74.610 | 58.684 | 55.985 | 0.269 | 23.847 | 43.667 |
| 20 | 0.333 | 0.017 | -0.665 | -0.315 | 1.780 | 160.581 | 74.557 | 58.541 | 56.222 | 0.269 | 23.888 | 43.864 |
| 21 | 0.384 | 0.002 | -0.747 | -0.361 | 1.917 | 161.475 | 74.291 | 57.892 | 56.540 | 0.269 | 23.755 | 44.352 |
| 22 | 0.387 | 0.002 | -0.756 | -0.366 | 1.936 | 161.400 | 74.292 | 57.658 | 56.678 | 0.271 | 23.768 | 44.545 |
| 23 | 0.399 | 0.002 | -0.798 | -0.396 | 2.020 | 161.779 | 74.130 | 57.349 | 56.898 | 0.271 | 23.729 | 44.817 |
| 24 | 0.402 | 0.002 | -0.806 | -0.402 | 2.037 | 161.933 | 74.121 | 57.648 | 56.922 | 0.269 | 23.755 | 44.677 |
| 25 | 0.440 | -0.006 | -0.887 | -0.453 | 2.194 | 162.734 | 73.877 | 57.696 | 57.523 | 0.267 | 23.802 | 44.969 |
| 26 | 0.456 | -0.006 | -0.942 | -0.492 | 2.306 | 162.738 | 73.778 | 57.255 | 57.796 | 0.269 | 23.809 | 45.343 |
| 27 | 0.502 | -0.020 | -1.025 | -0.543 | 2.449 | 163.404 | 73.536 | 56.930 | 58.054 | 0.267 | 23.739 | 45.642 |
| 28 | 0.530 | -0.021 | -1.116 | -0.606 | 2.626 | 163.914 | 73.380 | 56.485 | 58.292 | 0.268 | 23.670 | 45.997 |
| 29 | 0.554 | -0.030 | -1.182 | -0.658 | 2.754 | 163.666 | 73.316 | 56.120 | 58.388 | 0.270 | 23.678 | 46.246 |
| 30 | 0.656 | -0.133 | -1.356 | -0.833 | 3.255 | 162.734 | 73.457 | 56.304 | 58.996 | 0.273 | 23.975 | 46.506 |
| 31 | 0.727 | -0.140 | -1.567 | -0.981 | 3.667 | 163.030 | 73.379 | 56.134 | 58.790 | 0.272 | 23.863 | 46.475 |

$$
\text { Torsion Shear Test on Fine Nevada Sand with a Target Void Ratio }=0.530\left(D_{r}=91.28 \%\right)
$$

| Test No.: | $5^{*}$ |
| :--- | :---: |
| Test Date: | $12 / 3 / 10$ |
| Initial Height, $\mathrm{h}_{\mathrm{i}}$ : | 39.96 cm |
| Initial Void Ratio, e: | 0.524 |
| Final Mean Princpal Stress, $\sigma_{\mathrm{m}}:$ | 98.0 kPa |
| Max Friction Angle, $\phi:$ | 42.0 deg |
| b-value at failure: | 0.27 |
| Stress direction at failure, $\alpha:$ | 23.5 deg |


| Shear Band Notes |  |
| :--- | :--- |
| Point of Observation: | 33 |
| Inclination (from Vertical) |  |
| $2 @ 34^{\circ}$ |  |
| Failure Notes: |  |


| Point (No.) | $\begin{gathered} \varepsilon_{\mathrm{z}} \\ (\%) \end{gathered}$ | $\begin{gathered} \varepsilon_{\mathrm{r}} \\ (\%) \end{gathered}$ | $\begin{gathered} \varepsilon_{\theta} \\ (\%) \end{gathered}$ | $\begin{gathered} \varepsilon_{\mathrm{v}} \\ (\%) \end{gathered}$ | $\gamma_{\theta z}$ <br> (\%) | $\begin{gathered} \sigma_{z} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \sigma_{\mathrm{r}} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \sigma_{\theta} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \tau_{\theta z} \\ (\mathrm{kPa}) \end{gathered}$ | b | $\alpha$ <br> ${ }^{\circ}$ ) | $\begin{gathered} \varphi \\ \left({ }^{\circ}\right) \end{gathered}$ |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 1 | 0.001 | 0.000 | 0.000 | 0.001 | 0.001 | 92.963 | 100.025 | 99.151 | 3.698 | 0.912 | 64.958 | 2.877 |
| 2 | 0.001 | 0.001 | 0.000 | 0.002 | 0.002 | 95.382 | 99.307 | 98.249 | 5.574 | 0.716 | 52.212 | 3.408 |
| 3 | 0.004 | 0.004 | 0.004 | 0.012 | 0.043 | 111.140 | 93.322 | 88.249 | 17.237 | 0.346 | 28.208 | 11.978 |
| 4 | 0.005 | 0.005 | 0.004 | 0.014 | 0.078 | 116.277 | 91.416 | 85.120 | 21.781 | 0.327 | 27.214 | 15.422 |
| 5 | 0.013 | 0.003 | 0.000 | 0.016 | 0.120 | 120.116 | 89.946 | 82.715 | 26.004 | 0.321 | 27.139 | 18.411 |
| 6 | 0.032 | -0.001 | -0.011 | 0.019 | 0.204 | 125.989 | 87.718 | 79.135 | 31.013 | 0.309 | 26.466 | 22.269 |
| 7 | 0.045 | -0.005 | -0.021 | 0.020 | 0.260 | 129.507 | 86.468 | 77.195 | 33.502 | 0.301 | 26.010 | 24.283 |
| 8 | 0.060 | -0.008 | -0.032 | 0.020 | 0.316 | 132.580 | 85.283 | 75.143 | 35.447 | 0.296 | 25.493 | 26.056 |
| 9 | 0.073 | -0.010 | -0.042 | 0.021 | 0.361 | 135.228 | 84.411 | 73.984 | 36.845 | 0.289 | 25.135 | 27.258 |
| 10 | 0.084 | -0.012 | -0.054 | 0.018 | 0.407 | 136.857 | 83.731 | 72.681 | 38.016 | 0.289 | 24.917 | 28.349 |
| 11 | 0.100 | -0.013 | -0.074 | 0.013 | 0.496 | 139.797 | 82.683 | 71.125 | 40.027 | 0.284 | 24.688 | 30.004 |
| 12 | 0.125 | -0.017 | -0.103 | 0.005 | 0.589 | 141.901 | 81.832 | 69.649 | 41.637 | 0.283 | 24.527 | 31.409 |
| 13 | 0.146 | -0.021 | -0.130 | -0.004 | 0.667 | 143.789 | 81.199 | 68.663 | 42.751 | 0.280 | 24.348 | 32.394 |
| 14 | 0.165 | -0.024 | -0.155 | -0.014 | 0.739 | 144.993 | 80.759 | 68.149 | 43.829 | 0.279 | 24.380 | 33.156 |
| 15 | 0.191 | -0.029 | -0.193 | -0.031 | 0.842 | 146.484 | 80.212 | 66.937 | 44.744 | 0.279 | 24.183 | 34.126 |
| 16 | 0.244 | -0.031 | -0.284 | -0.072 | 1.075 | 148.877 | 79.273 | 65.212 | 46.732 | 0.279 | 24.083 | 35.868 |
| 17 | 0.300 | -0.083 | -0.331 | -0.114 | 1.296 | 149.900 | 78.727 | 64.700 | 48.064 | 0.278 | 24.224 | 36.767 |
| 18 | 0.320 | -0.088 | -0.371 | -0.138 | 1.376 | 151.743 | 78.106 | 62.929 | 48.222 | 0.277 | 23.679 | 37.643 |
| 19 | 0.361 | -0.096 | -0.448 | -0.183 | 1.547 | 152.683 | 77.629 | 62.710 | 49.359 | 0.275 | 23.827 | 38.324 |
| 20 | 0.397 | -0.100 | -0.519 | -0.223 | 1.692 | 154.092 | 77.327 | 61.964 | 49.865 | 0.274 | 23.634 | 38.933 |
| 21 | 0.426 | -0.104 | -0.574 | -0.252 | 1.803 | 154.451 | 77.082 | 61.567 | 50.249 | 0.274 | 23.627 | 39.309 |
| 22 | 0.490 | -0.120 | -0.711 | -0.341 | 2.078 | 155.781 | 76.635 | 60.850 | 51.167 | 0.273 | 23.575 | 40.117 |
| 23 | 0.525 | -0.125 | -0.787 | -0.388 | 2.233 | 156.399 | 76.407 | 60.630 | 51.651 | 0.272 | 23.584 | 40.471 |
| 24 | 0.685 | -0.140 | -1.169 | -0.625 | 2.985 | 158.441 | 75.764 | 59.645 | 53.114 | 0.271 | 23.538 | 41.697 |
| 25 | 0.692 | -0.142 | -1.186 | -0.636 | 3.016 | 158.410 | 75.724 | 59.491 | 53.044 | 0.271 | 23.501 | 41.734 |
| 26 | 0.694 | -0.142 | -1.192 | -0.640 | 3.027 | 158.410 | 75.730 | 59.490 | 53.112 | 0.271 | 23.519 | 41.769 |
| 27 | 0.732 | -0.151 | -1.292 | -0.711 | 3.223 | 158.714 | 75.586 | 59.336 | 53.338 | 0.271 | 23.514 | 41.961 |
| 28 | 0.759 | -0.148 | -1.359 | -0.748 | 3.353 | 158.999 | 75.544 | 59.452 | 53.502 | 0.270 | 23.534 | 41.992 |
| 29 | 0.869 | -0.152 | -1.583 | -0.866 | 3.723 | 157.671 | 75.728 | 59.434 | 52.685 | 0.272 | 23.503 | 41.571 |
| 30 | 0.916 | -0.159 | -1.675 | -0.919 | 3.877 | 158.189 | 75.735 | 59.707 | 52.992 | 0.270 | 23.551 | 41.603 |
| 31 | 0.967 | -0.169 | -1.769 | -0.971 | 4.022 | 158.630 | 75.712 | 59.946 | 53.164 | 0.269 | 23.568 | 41.581 |
| 32 | 1.017 | -0.176 | -1.861 | -1.020 | 4.158 | 158.819 | 75.645 | 59.948 | 53.275 | 0.268 | 23.570 | 41.639 |
| 33 | 1.117 | -0.180 | -2.045 | -1.108 | 4.426 | 158.938 | 75.549 | 59.481 | 53.045 | 0.269 | 23.424 | 41.742 |

Torsion Shear Test on Fine Nevada Sand with a Target Void Ratio $=0.530\left(D_{r}=91.28 \%\right)$

| Test No.: | 6 |
| :--- | :--- |
| Test Date: | $8 / 31 / 11$ |
| Initial Height, $\mathrm{h}_{\mathrm{i}}:$ | 39.96 cm |
| Initial Void Ratio, e: | 0.526 |
| Final Mean Princpal Stress, $\sigma_{\mathrm{m}}:$ | 101.3 kPa |
| Max Friction Angle, $\phi:$ | 45.9 deg |
| b-value at failure: | 0.50 |
| Stress direction at failure, $\alpha:$ | 22.5 deg |


| Shear Band Notes <br> Point of Observation: |
| :--- |
| Inclination (from Vertical) |
| $28^{\circ}, 43^{\circ}, 2 @ 48^{\circ}, 49^{\circ}, 50^{\circ}$ |
|  |
| Failure Notes: |
| 6 parallel shear bands |


| Point (No.) | $\begin{gathered} \varepsilon_{\mathrm{z}} \\ (\%) \end{gathered}$ | $\begin{gathered} \varepsilon_{\mathrm{r}} \\ (\%) \end{gathered}$ | $\begin{gathered} \varepsilon_{\theta} \\ (\%) \end{gathered}$ | $\begin{gathered} \varepsilon_{\mathrm{v}} \\ (\%) \end{gathered}$ | $\begin{gathered} \gamma_{\theta z} \\ (\%) \end{gathered}$ | $\begin{gathered} \sigma_{z} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \sigma_{\mathrm{r}} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \sigma_{\theta} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \boldsymbol{\tau}_{\theta z} \\ (\mathrm{kPa}) \end{gathered}$ | b | $\begin{gathered} \alpha \\ \left({ }^{\circ}\right) \end{gathered}$ | $\begin{gathered} \varphi \\ \left({ }^{\circ}\right) \end{gathered}$ |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 1 | 0.000 | 0.000 | 0.000 | 0.000 | 0.001 | 101.463 | 101.328 | 100.228 | 0.397 | 0.829 | 16.361 | 0.417 |
| 2 | 0.001 | -0.001 | 0.000 | 0.000 | 0.004 | 102.508 | 101.378 | 98.849 | 3.712 | 0.584 | 31.883 | 2.356 |
| 3 | 0.005 | 0.001 | -0.003 | 0.003 | 0.021 | 114.183 | 101.352 | 89.081 | 13.378 | 0.492 | 23.413 | 10.398 |
| 4 | 0.008 | 0.002 | -0.005 | 0.005 | 0.031 | 118.598 | 101.346 | 84.666 | 17.398 | 0.494 | 22.860 | 13.834 |
| 5 | 0.002 | 0.008 | -0.004 | 0.006 | 0.043 | 122.726 | 101.382 | 80.345 | 21.000 | 0.497 | 22.371 | 17.087 |
| 6 | 0.020 | 0.002 | -0.014 | 0.007 | 0.056 | 125.993 | 101.334 | 77.145 | 25.174 | 0.497 | 22.933 | 20.202 |
| 7 | 0.015 | 0.006 | -0.014 | 0.007 | 0.066 | 128.475 | 101.400 | 74.640 | 27.286 | 0.498 | 22.695 | 22.173 |
| 8 | 0.016 | 0.010 | -0.017 | 0.009 | 0.080 | 131.249 | 101.366 | 71.945 | 29.984 | 0.497 | 22.660 | 24.524 |
| 9 | 0.030 | 0.007 | -0.029 | 0.008 | 0.104 | 133.551 | 101.389 | 68.858 | 32.445 | 0.502 | 22.544 | 26.916 |
| 10 | 0.039 | 0.008 | -0.040 | 0.007 | 0.138 | 135.103 | 101.310 | 67.114 | 35.425 | 0.502 | 23.090 | 29.051 |
| 11 | 0.040 | 0.012 | -0.048 | 0.004 | 0.172 | 136.107 | 101.258 | 64.598 | 36.942 | 0.509 | 22.968 | 30.818 |
| 12 | 0.044 | 0.017 | -0.057 | 0.004 | 0.201 | 139.332 | 101.359 | 63.642 | 38.434 | 0.499 | 22.721 | 32.106 |
| 13 | 0.047 | 0.021 | -0.070 | -0.001 | 0.244 | 140.920 | 101.365 | 61.985 | 40.099 | 0.499 | 22.727 | 33.682 |
| 14 | 0.064 | 0.015 | -0.082 | -0.003 | 0.263 | 141.842 | 101.379 | 61.138 | 40.559 | 0.499 | 22.573 | 34.314 |
| 15 | 0.066 | 0.027 | -0.106 | -0.012 | 0.324 | 143.200 | 101.347 | 59.236 | 42.305 | 0.501 | 22.610 | 36.074 |
| 16 | 0.070 | 0.034 | -0.124 | -0.021 | 0.361 | 144.229 | 101.370 | 58.376 | 43.280 | 0.501 | 22.618 | 36.995 |
| 17 | 0.087 | 0.032 | -0.146 | -0.027 | 0.397 | 144.881 | 101.363 | 57.453 | 43.959 | 0.502 | 22.580 | 37.792 |
| 18 | 0.099 | 0.037 | -0.175 | -0.039 | 0.451 | 145.898 | 101.376 | 56.347 | 44.862 | 0.502 | 22.528 | 38.814 |
| 19 | 0.106 | 0.050 | -0.216 | -0.060 | 0.543 | 147.254 | 101.344 | 55.475 | 45.856 | 0.500 | 22.490 | 39.792 |
| 20 | 0.124 | 0.050 | -0.248 | -0.073 | 0.597 | 147.238 | 101.351 | 54.757 | 46.603 | 0.503 | 22.612 | 40.543 |
| 21 | 0.133 | 0.070 | -0.310 | -0.108 | 0.725 | 148.709 | 101.307 | 53.912 | 47.591 | 0.500 | 22.558 | 41.528 |
| 22 | 0.153 | 0.075 | -0.360 | -0.131 | 0.808 | 149.269 | 101.329 | 53.238 | 48.147 | 0.501 | 22.539 | 42.187 |
| 23 | 0.161 | 0.091 | -0.412 | -0.160 | 0.904 | 149.877 | 101.366 | 52.564 | 48.715 | 0.501 | 22.517 | 42.861 |
| 24 | 0.191 | 0.095 | -0.474 | -0.188 | 0.998 | 150.655 | 101.368 | 52.038 | 49.303 | 0.500 | 22.498 | 43.474 |
| 25 | 0.201 | 0.118 | -0.548 | -0.229 | 1.134 | 151.073 | 101.364 | 51.394 | 49.771 | 0.501 | 22.480 | 44.088 |
| 26 | 0.236 | 0.132 | -0.645 | -0.276 | 1.289 | 151.610 | 101.374 | 50.793 | 50.240 | 0.501 | 22.452 | 44.688 |
| 27 | 0.268 | 0.151 | -0.750 | -0.332 | 1.455 | 152.004 | 101.362 | 50.541 | 50.885 | 0.501 | 22.543 | 45.195 |
| 28 | 0.310 | 0.181 | -0.893 | -0.402 | 1.670 | 152.209 | 101.395 | 49.954 | 50.675 | 0.502 | 22.373 | 45.411 |
| 29 | 0.345 | 0.211 | -1.022 | -0.466 | 1.879 | 152.824 | 101.349 | 49.804 | 51.077 | 0.500 | 22.379 | 45.725 |
| 30 | 0.372 | 0.214 | -1.075 | -0.489 | 1.953 | 152.810 | 101.351 | 49.822 | 51.411 | 0.500 | 22.477 | 45.906 |

Torsion Shear Test on Fine Nevada Sand with a Target Void Ratio $=\mathbf{0 . 5 3 0}\left(\mathrm{D}_{\mathrm{r}}=\mathbf{9 1 . 2 8 \%}\right.$ )

| Test No.: | $7^{*}$ |
| :--- | :---: |
| Test Date: | $8 / 31 / 10$ |
| Initial Height, $\mathrm{h}_{\mathrm{i}}:$ | 40.13 cm |
| Initial Void Ratio, e: | 0.552 |
| Final Mean Princpal Stress, $\sigma_{\mathrm{m}}:$ | 96.67 kPa |
| Max Friction Angle, $\phi:$ | 43.1 deg |
| b-value at failure: | 0.89 |
| Stress direction at failure, $\alpha:$ | 24.5 deg |


| Shear Band Notes |  |
| :--- | :--- |
| Point of Observation: | 26 |
|  |  |
| Inclination (from Vertical) |  |
| 1 SB at $57^{\circ}-67^{\circ}$ |  |
| $30^{\circ}, 65^{\circ}, 66^{\circ}, 67^{\circ}$ |  |
| Failure Notes: |  |
| Deep trough varying direction |  |


| Point (No.) | $\begin{gathered} \varepsilon_{\mathrm{z}} \\ (\%) \end{gathered}$ | $\begin{gathered} \varepsilon_{\mathrm{r}} \\ (\%) \end{gathered}$ | $\begin{gathered} \varepsilon_{\theta} \\ (\%) \end{gathered}$ | $\begin{gathered} \varepsilon_{\mathrm{v}} \\ (\%) \end{gathered}$ | $\begin{aligned} & \gamma_{\theta z} \\ & (\%) \end{aligned}$ | $\begin{gathered} \sigma_{\mathrm{z}} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \sigma_{\mathrm{r}} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \sigma_{\theta} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \tau_{\theta z} \\ (\mathrm{kPa}) \end{gathered}$ | b | $\begin{gathered} \alpha \\ \left({ }^{\circ}\right) \end{gathered}$ | $\begin{gathered} \varphi \\ \left({ }^{\circ}\right) \end{gathered}$ |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 1 | -0.003 | 0.032 | -0.013 | 0.016 | 0.094 | 98.211 | 113.992 | 81.894 | 13.378 | 0.791 | 29.312 | 12.138 |
| 2 | -0.003 | 0.034 | -0.014 | 0.018 | 0.096 | 98.451 | 114.534 | 80.850 | 13.238 | 0.780 | 28.192 | 12.508 |
| 3 | 0.008 | 0.056 | -0.033 | 0.031 | 0.167 | 101.073 | 119.289 | 73.827 | 19.587 | 0.857 | 27.590 | 17.732 |
| 4 | 0.012 | 0.066 | -0.043 | 0.036 | 0.196 | 102.158 | 121.387 | 70.563 | 21.214 | 0.860 | 26.663 | 19.821 |
| 5 | 0.014 | 0.085 | -0.056 | 0.043 | 0.273 | 103.219 | 123.539 | 67.020 | 23.825 | 0.876 | 26.388 | 22.479 |
| 6 | 0.013 | 0.102 | -0.068 | 0.048 | 0.323 | 104.103 | 125.041 | 65.088 | 25.174 | 0.881 | 26.114 | 23.993 |
| 7 | 0.013 | 0.118 | -0.079 | 0.052 | 0.366 | 104.756 | 126.229 | 62.992 | 26.462 | 0.886 | 25.861 | 25.544 |
| 8 | 0.011 | 0.140 | -0.094 | 0.057 | 0.428 | 105.380 | 127.612 | 61.160 | 28.012 | 0.892 | 25.858 | 27.181 |
| 9 | 0.009 | 0.161 | -0.109 | 0.060 | 0.482 | 105.880 | 128.805 | 59.067 | 29.085 | 0.892 | 25.587 | 28.750 |
| 10 | 0.006 | 0.193 | -0.135 | 0.064 | 0.557 | 106.621 | 129.970 | 57.034 | 30.311 | 0.897 | 25.359 | 30.377 |
| 11 | 0.000 | 0.232 | -0.166 | 0.065 | 0.641 | 107.189 | 131.198 | 55.228 | 30.826 | 0.893 | 24.938 | 31.651 |
| 12 | -0.001 | 0.259 | -0.193 | 0.065 | 0.709 | 107.823 | 132.046 | 53.939 | 32.280 | 0.902 | 25.075 | 33.057 |
| 13 | -0.001 | 0.280 | -0.215 | 0.064 | 0.760 | 107.916 | 132.599 | 53.068 | 32.797 | 0.901 | 25.049 | 33.841 |
| 14 | -0.002 | 0.334 | -0.275 | 0.057 | 0.895 | 108.647 | 133.778 | 51.623 | 34.412 | 0.909 | 25.178 | 35.527 |
| 15 | -0.003 | 0.362 | -0.308 | 0.051 | 0.954 | 108.890 | 134.367 | 50.560 | 34.638 | 0.906 | 24.951 | 36.294 |
| 16 | -0.008 | 0.395 | -0.343 | 0.044 | 1.033 | 109.357 | 134.825 | 49.918 | 35.192 | 0.910 | 24.909 | 36.959 |
| 17 | -0.011 | 0.418 | -0.369 | 0.038 | 1.082 | 109.183 | 135.139 | 49.156 | 35.256 | 0.905 | 24.796 | 37.498 |
| 18 | -0.010 | 0.465 | -0.433 | 0.022 | 1.200 | 109.568 | 135.726 | 48.321 | 35.913 | 0.908 | 24.773 | 38.379 |
| 19 | -0.015 | 0.521 | -0.503 | 0.003 | 1.334 | 109.827 | 136.328 | 47.725 | 36.642 | 0.910 | 24.861 | 39.194 |
| 20 | -0.020 | 0.565 | -0.559 | -0.015 | 1.437 | 110.016 | 136.756 | 46.988 | 36.988 | 0.910 | 24.784 | 39.873 |
| 21 | -0.022 | 0.615 | -0.631 | -0.038 | 1.560 | 110.403 | 137.086 | 46.336 | 37.416 | 0.912 | 24.716 | 40.515 |
| 22 | -0.026 | 0.676 | -0.719 | -0.069 | 1.710 | 110.850 | 137.690 | 45.517 | 37.747 | 0.912 | 24.564 | 41.252 |
| 23 | -0.030 | 0.730 | -0.797 | -0.098 | 1.843 | 110.798 | 137.979 | 45.028 | 38.221 | 0.913 | 24.646 | 41.891 |
| 24 | -0.038 | 0.797 | -0.902 | -0.144 | 2.002 | 108.332 | 138.223 | 43.445 | 36.990 | 0.882 | 24.373 | 42.559 |
| 25 | -0.046 | 0.835 | -0.953 | -0.164 | 2.083 | 108.355 | 138.393 | 43.186 | 37.540 | 0.885 | 24.521 | 43.083 |
| 26 | -0.046 | 0.874 | -1.011 | -0.182 | 2.139 | 108.304 | 138.494 | 42.850 | 36.537 | 0.876 | 24.074 | 42.728 |

$$
\text { Torsion Shear Test on Fine Nevada Sand with a Target Void Ratio }=0.530\left(D_{r}=91.28 \%\right)
$$

| Test No.: | 8 |
| :--- | :--- |
| Test Date: | $9 / 2 / 11$ |
| Initial Height, $\mathrm{h}_{\mathrm{i}}:$ | 39.96 cm |
| Initial Void Ratio, e: | 0.541 |
| Final Mean Princpal Stress, $\sigma_{\mathrm{m}}:$ | 102.3 kPa |
| Max Friction Angle, $\phi:$ | 42.1 deg |
| b-value at failure: | 0.99 |
| Stress direction at failure, $\alpha:$ | 22.5 deg |


| Shear Band Notes |
| :--- |
| Point of Observation: |
|  |
| Inclination (from Vertical) |
| $67^{\circ}$ |
| Failure Notes: |
| Thick trough through specimen |


| Point (No.) | $\begin{gathered} \varepsilon_{\mathrm{z}} \\ (\%) \end{gathered}$ | $\begin{gathered} \varepsilon_{\mathrm{r}} \\ (\%) \end{gathered}$ | $\begin{gathered} \varepsilon_{\theta} \\ (\%) \end{gathered}$ | $\begin{gathered} \varepsilon_{\mathrm{v}} \\ (\%) \end{gathered}$ | $\begin{aligned} & \gamma_{\theta z} \\ & (\%) \end{aligned}$ | $\begin{gathered} \sigma_{z} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \sigma_{\mathrm{r}} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \sigma_{\theta} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \tau_{\text {өz }} \\ (\mathrm{kPa}) \end{gathered}$ | b | $\begin{gathered} \alpha \\ \left({ }^{\circ}\right) \end{gathered}$ | $\begin{gathered} \varphi \\ \left({ }^{\circ}\right) \\ \hline \end{gathered}$ |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 1 | 0.000 | 0.000 | 0.000 | 0.000 | 0.000 | 14.818 | 14.868 | 14.478 | 0.170 | 0.956 | 22.494 | 0.941 |
| 2 | 0.006 | 0.006 | -0.007 | 0.005 | 0.017 | 15.271 | 15.707 | 13.212 | 1.407 | 0.920 | 26.902 | 7.031 |
| 3 | 0.007 | 0.009 | -0.009 | 0.006 | 0.025 | 15.534 | 16.039 | 12.767 | 1.777 | 0.919 | 26.045 | 9.158 |
| 4 | 0.007 | 0.014 | -0.012 | 0.009 | 0.035 | 15.728 | 16.390 | 12.169 | 2.104 | 0.943 | 24.887 | 11.395 |
| 5 | 0.007 | 0.019 | -0.014 | 0.011 | 0.044 | 15.885 | 16.691 | 11.666 | 2.310 | 0.966 | 23.799 | 13.126 |
| 6 | 0.006 | 0.026 | -0.018 | 0.013 | 0.062 | 16.102 | 17.098 | 11.063 | 2.718 | 0.974 | 23.584 | 15.834 |
| 7 | 0.005 | 0.031 | -0.021 | 0.015 | 0.074 | 16.189 | 17.290 | 10.741 | 2.970 | 0.975 | 23.737 | 17.413 |
| 8 | 0.004 | 0.037 | -0.024 | 0.016 | 0.088 | 16.325 | 17.521 | 10.423 | 3.186 | 0.977 | 23.598 | 18.950 |
| 9 | 0.003 | 0.043 | -0.027 | 0.018 | 0.102 | 16.444 | 17.726 | 10.084 | 3.324 | 0.985 | 23.131 | 20.292 |
| 10 | 0.000 | 0.048 | -0.029 | 0.019 | 0.117 | 16.531 | 17.864 | 9.890 | 3.538 | 0.979 | 23.409 | 21.550 |
| 11 | 0.001 | 0.054 | -0.036 | 0.020 | 0.137 | 16.627 | 18.058 | 9.565 | 3.720 | 0.984 | 23.247 | 23.058 |
| 12 | 0.003 | 0.061 | -0.043 | 0.021 | 0.163 | 16.758 | 18.238 | 9.350 | 4.003 | 0.975 | 23.609 | 24.692 |
| 13 | 0.006 | 0.074 | -0.059 | 0.021 | 0.213 | 16.913 | 18.509 | 8.905 | 4.301 | 0.977 | 23.524 | 27.080 |
| 14 | 0.007 | 0.094 | -0.083 | 0.018 | 0.268 | 17.018 | 18.786 | 8.390 | 4.502 | 0.988 | 23.111 | 29.395 |
| 15 | 0.008 | 0.115 | -0.109 | 0.014 | 0.343 | 17.182 | 18.979 | 8.187 | 4.861 | 0.975 | 23.614 | 31.473 |
| 16 | 0.009 | 0.157 | -0.168 | -0.003 | 0.443 | 17.362 | 19.318 | 7.625 | 5.018 | 0.988 | 22.931 | 34.029 |
| 17 | 0.008 | 0.190 | -0.217 | -0.020 | 0.538 | 17.469 | 19.475 | 7.453 | 5.233 | 0.984 | 23.131 | 35.541 |
| 18 | 0.006 | 0.242 | -0.302 | -0.054 | 0.666 | 17.543 | 19.678 | 7.071 | 5.330 | 0.993 | 22.755 | 37.382 |
| 19 | 0.005 | 0.279 | -0.363 | -0.079 | 0.761 | 17.630 | 19.778 | 6.955 | 5.457 | 0.990 | 22.816 | 38.388 |
| 20 | 0.002 | 0.322 | -0.437 | -0.113 | 0.868 | 17.688 | 19.895 | 6.779 | 5.519 | 0.994 | 22.670 | 39.370 |
| 21 | 0.003 | 0.356 | -0.500 | -0.140 | 0.953 | 17.719 | 19.954 | 6.669 | 5.565 | 0.995 | 22.602 | 40.020 |
| 22 | 0.002 | 0.391 | -0.561 | -0.168 | 1.040 | 17.743 | 19.997 | 6.620 | 5.654 | 0.993 | 22.737 | 40.621 |
| 23 | 0.005 | 0.429 | -0.637 | -0.203 | 1.137 | 17.828 | 20.085 | 6.541 | 5.709 | 0.992 | 22.665 | 41.208 |
| 24 | 0.008 | 0.474 | -0.726 | -0.243 | 1.247 | 17.843 | 20.137 | 6.439 | 5.732 | 0.995 | 22.574 | 41.754 |
| 25 | 0.008 | 0.500 | -0.773 | -0.265 | 1.306 | 17.866 | 20.155 | 6.410 | 5.748 | 0.994 | 22.551 | 41.955 |
| 26 | 0.008 | 0.502 | -0.777 | -0.267 | 1.311 | 17.870 | 20.155 | 6.409 | 5.751 | 0.994 | 22.552 | 41.974 |
| 27 | 0.012 | 0.502 | -0.784 | -0.270 | 1.317 | 17.957 | 20.153 | 6.399 | 5.766 | 0.988 | 22.469 | 42.096 |

## Torsion Shear Test on Fine Nevada Sand with a Target Void Ratio $=0.530\left(D_{r}=\mathbf{9 1 . 2 8 \%}\right)$

| Test No.: | 9 |
| :--- | :--- |
| Test Date: | $9 / 9 / 11$ |
| Initial Height, $\mathrm{h}_{\mathrm{i}}:$ | 39.96 cm |
| Initial Void Ratio, e: | 0.526 |
| Final Mean Princpal Stress, $\sigma_{\mathrm{m}}:$ | 101.3 kPa |
| Max Friction Angle, $\phi:$ | 38.7 deg |
| b-value at failure: | 0.25 |
| Stress direction at failure, $\alpha:$ | 45.0 deg |


| Shear Band Notes <br> Point of Observation: |
| :--- |
| Inclination (from Vertical) |
| $16^{\circ}, 17^{\circ}, 2 @ 19^{\circ}, 21^{\circ}, 2 @ 21.5^{\circ}$ |
| Failure Notes: |


| Point (No.) | $\begin{gathered} \varepsilon_{\mathrm{z}} \\ (\%) \\ \hline \end{gathered}$ | $\begin{gathered} \varepsilon_{\mathrm{r}} \\ (\%) \\ \hline \end{gathered}$ | $\begin{gathered} \varepsilon_{\theta} \\ (\%) \end{gathered}$ | $\begin{gathered} \varepsilon_{\mathrm{v}} \\ (\%) \\ \hline \end{gathered}$ | $\begin{aligned} & \gamma_{\theta z} \\ & (\%) \\ & \hline \end{aligned}$ | $\begin{gathered} \sigma_{z} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \sigma_{\mathrm{r}} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \sigma_{\theta} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \tau_{\theta z} \\ (\mathrm{kPa}) \end{gathered}$ | b | $\alpha$ <br> $\left.{ }^{\circ}{ }^{\circ}\right)$ | $\begin{gathered} \varphi \\ \left.{ }^{( }\right) \end{gathered}$ |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 1 | 0.000 | 0.000 | 0.000 | 0.000 | 0.000 | 101.371 | 100.975 | 101.330 | 1.322 | 0.358 | 44.555 | 0.747 |
| 2 | 0.000 | -0.003 | 0.004 | 0.001 | 0.036 | 103.052 | 98.165 | 102.480 | 12.565 | 0.317 | 44.349 | 7.025 |
| 3 | 0.000 | -0.005 | 0.007 | 0.003 | 0.062 | 104.076 | 96.178 | 103.828 | 18.185 | 0.286 | 44.805 | 10.075 |
| 4 | 0.001 | -0.007 | 0.012 | 0.006 | 0.106 | 105.224 | 93.551 | 104.964 | 25.269 | 0.272 | 44.853 | 13.913 |
| 5 | 0.001 | -0.010 | 0.018 | 0.009 | 0.162 | 106.564 | 91.394 | 106.185 | 31.914 | 0.265 | 44.830 | 17.459 |
| 6 | 0.003 | -0.013 | 0.022 | 0.012 | 0.207 | 107.351 | 89.867 | 106.914 | 36.143 | 0.261 | 44.827 | 19.717 |
| 7 | 0.001 | -0.014 | 0.028 | 0.015 | 0.258 | 107.958 | 88.549 | 107.651 | 40.089 | 0.260 | 44.890 | 21.831 |
| 8 | -0.002 | -0.016 | 0.036 | 0.018 | 0.360 | 108.886 | 87.073 | 108.411 | 44.845 | 0.259 | 44.848 | 24.378 |
| 9 | -0.005 | -0.017 | 0.041 | 0.019 | 0.414 | 108.977 | 86.166 | 109.002 | 46.851 | 0.256 | 45.008 | 25.459 |
| 10 | -0.012 | -0.019 | 0.050 | 0.019 | 0.516 | 110.069 | 84.953 | 109.667 | 50.089 | 0.251 | 44.885 | 27.123 |
| 11 | -0.019 | -0.021 | 0.057 | 0.018 | 0.611 | 110.386 | 84.006 | 109.816 | 52.527 | 0.252 | 44.845 | 28.496 |
| 12 | -0.027 | -0.023 | 0.065 | 0.015 | 0.704 | 110.727 | 83.267 | 110.320 | 54.517 | 0.250 | 44.893 | 29.556 |
| 13 | -0.036 | -0.025 | 0.071 | 0.011 | 0.810 | 111.090 | 82.491 | 110.865 | 56.470 | 0.248 | 44.943 | 30.587 |
| 14 | -0.048 | -0.028 | 0.079 | 0.003 | 0.947 | 111.444 | 81.758 | 111.406 | 58.559 | 0.247 | 44.991 | 31.705 |
| 15 | -0.060 | -0.030 | 0.086 | -0.004 | 1.054 | 111.811 | 81.267 | 111.502 | 60.160 | 0.247 | 44.927 | 32.602 |
| 16 | -0.082 | -0.039 | 0.095 | -0.026 | 1.307 | 112.091 | 80.261 | 112.306 | 62.674 | 0.245 | 45.049 | 33.959 |
| 17 | -0.088 | -0.048 | 0.093 | -0.043 | 1.458 | 112.129 | 79.964 | 112.176 | 64.037 | 0.249 | 45.011 | 34.819 |
| 18 | -0.101 | -0.054 | 0.094 | -0.061 | 1.610 | 112.573 | 79.477 | 112.548 | 65.057 | 0.246 | 44.994 | 35.308 |
| 19 | -0.109 | -0.062 | 0.091 | -0.080 | 1.770 | 112.737 | 79.161 | 112.755 | 66.047 | 0.246 | 45.004 | 35.860 |
| 20 | -0.117 | -0.067 | 0.091 | -0.094 | 1.881 | 112.783 | 78.943 | 112.859 | 66.630 | 0.246 | 45.016 | 36.198 |
| 21 | -0.131 | -0.073 | 0.088 | -0.116 | 2.055 | 113.045 | 78.675 | 112.979 | 67.646 | 0.246 | 44.986 | 36.768 |
| 22 | -0.152 | -0.086 | 0.081 | -0.157 | 2.350 | 113.156 | 78.227 | 113.209 | 68.676 | 0.246 | 45.011 | 37.357 |
| 23 | -0.171 | -0.100 | 0.064 | -0.207 | 2.691 | 113.272 | 77.965 | 113.154 | 69.449 | 0.246 | 44.976 | 37.838 |
| 24 | -0.174 | -0.109 | 0.052 | -0.231 | 2.861 | 113.252 | 77.755 | 113.480 | 69.805 | 0.245 | 45.047 | 38.006 |
| 25 | -0.188 | -0.113 | 0.045 | -0.256 | 3.026 | 113.190 | 77.696 | 113.642 | 70.169 | 0.245 | 45.092 | 38.221 |
| 26 | -0.194 | -0.120 | 0.033 | -0.281 | 3.196 | 113.356 | 77.598 | 113.293 | 70.422 | 0.246 | 44.987 | 38.420 |
| 27 | -0.202 | -0.127 | 0.024 | -0.305 | 3.357 | 113.406 | 77.515 | 113.376 | 70.691 | 0.246 | 44.994 | 38.567 |
| 28 | -0.205 | -0.130 | 0.019 | -0.316 | 3.436 | 113.416 | 77.465 | 113.301 | 70.868 | 0.247 | 44.977 | 38.695 |
| 29 | -0.224 | -0.141 | 0.016 | -0.348 | 3.707 | 113.554 | 77.395 | 113.528 | 70.852 | 0.245 | 44.995 | 38.610 |
| 30 | -0.229 | -0.149 | 0.014 | -0.363 | 3.874 | 113.436 | 77.396 | 113.242 | 70.579 | 0.245 | 44.961 | 38.515 |
| 31 | -0.235 | -0.154 | 0.013 | -0.377 | 4.044 | 113.575 | 77.432 | 113.380 | 70.523 | 0.244 | 44.960 | 38.424 |
| 32 | -0.240 | -0.160 | 0.012 | -0.388 | 4.209 | 113.184 | 77.488 | 113.107 | 70.171 | 0.246 | 44.984 | 38.329 |
| 33 | -0.244 | -0.165 | 0.013 | -0.396 | 4.376 | 113.500 | 77.613 | 113.558 | 70.077 | 0.244 | 45.012 | 38.116 |

Torsion Shear Test on Fine Nevada Sand with a Target Void Ratio $=0.530\left(D_{r}=91.28 \%\right)$

| Test No.: | $10^{*}$ |
| :--- | :---: |
| Test Date: | $8 / 7 / 10$ |
| Initial Height, $\mathrm{h}_{\mathrm{i}}:$ | 40.09 cm |
| Initial Void Ratio, e: | 0.555 |
| Final Mean Princpal Stress, $\sigma_{\mathrm{m}}:$ | 98.2 kPa |
| Max Friction Angle, $\phi:$ | 38.9 deg |
| b-value at failure: | 0.54 |
| Stress direction at failure, $\alpha:$ | 47.4 deg |


| Shear Band Notes <br> Point of Observation: | 24 |
| :--- | :--- |
| Inclination (from Vertical) |  |
| $10^{\circ}, 3 @ 12^{\circ}$ |  |
|  |  |
| Failure Notes: |  |
| One SB along top cap |  |


| Point <br> $(\mathrm{No})$. | $\varepsilon_{\mathrm{z}}$ <br> $(\%)$ | $\varepsilon_{\mathrm{r}}$ <br> $(\%)$ | $\varepsilon_{\theta}$ <br> $(\%)$ | $\varepsilon_{\mathrm{v}}$ <br> $(\%)$ | $\gamma_{\theta \mathrm{z}}$ <br> $(\%)$ | $\sigma_{\mathrm{z}}$ <br> $(\mathrm{kPa})$ | $\sigma_{\mathrm{r}}$ <br> $(\mathrm{kPa})$ | $\sigma_{\theta}$ <br> $(\mathrm{kPa})$ | $\tau_{\mathrm{\theta z}}$ <br> $(\mathrm{kPa})$ | b | $\alpha$ <br> $\left({ }^{\circ}\right)$ | $\varphi$ <br> $\left({ }^{\circ}\right)$ |
| ---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 1 | 0.000 | 0.000 | 0.000 | 0.000 | 0.000 | 91.442 | 101.335 | 101.429 | 1.290 | 0.975 | 82.758 | 3.066 |
| 2 | -0.006 | 0.002 | 0.005 | 0.000 | 0.047 | 91.554 | 101.345 | 101.569 | 10.642 | 0.703 | 57.599 | 6.996 |
| 3 | -0.013 | 0.002 | 0.011 | 0.000 | 0.106 | 91.657 | 101.330 | 101.628 | 17.520 | 0.629 | 52.942 | 10.865 |
| 4 | -0.019 | 0.005 | 0.017 | 0.003 | 0.158 | 91.387 | 101.353 | 101.526 | 22.103 | 0.608 | 51.458 | 13.597 |
| 5 | -0.022 | 0.008 | 0.019 | 0.005 | 0.212 | 91.392 | 101.373 | 101.509 | 25.964 | 0.593 | 50.512 | 15.918 |
| 6 | -0.032 | 0.015 | 0.025 | 0.008 | 0.279 | 91.656 | 101.372 | 101.422 | 30.049 | 0.579 | 49.615 | 18.382 |
| 7 | -0.042 | 0.023 | 0.029 | 0.011 | 0.356 | 91.113 | 101.321 | 101.264 | 33.654 | 0.575 | 49.288 | 20.722 |
| 8 | -0.050 | 0.033 | 0.032 | 0.014 | 0.437 | 91.596 | 101.352 | 101.335 | 37.073 | 0.565 | 48.741 | 22.806 |
| 9 | -0.060 | 0.041 | 0.035 | 0.015 | 0.497 | 91.434 | 101.354 | 101.282 | 39.179 | 0.563 | 48.581 | 24.192 |
| 10 | -0.067 | 0.052 | 0.034 | 0.018 | 0.592 | 91.585 | 101.343 | 101.615 | 42.031 | 0.556 | 48.402 | 25.989 |
| 11 | -0.083 | 0.064 | 0.038 | 0.019 | 0.683 | 91.561 | 101.378 | 101.594 | 44.322 | 0.554 | 48.229 | 27.507 |
| 12 | -0.096 | 0.075 | 0.037 | 0.015 | 0.769 | 91.554 | 101.362 | 101.651 | 46.166 | 0.551 | 48.120 | 28.734 |
| 13 | -0.120 | 0.094 | 0.038 | 0.012 | 0.898 | 91.423 | 101.341 | 101.493 | 48.046 | 0.551 | 47.991 | 30.055 |
| 14 | -0.147 | 0.112 | 0.039 | 0.004 | 1.024 | 91.462 | 101.343 | 101.621 | 49.701 | 0.548 | 47.918 | 31.164 |
| 15 | -0.172 | 0.129 | 0.038 | -0.006 | 1.141 | 91.275 | 101.392 | 101.014 | 51.051 | 0.551 | 47.724 | 32.235 |
| 16 | -0.205 | 0.151 | 0.033 | -0.021 | 1.317 | 91.388 | 101.389 | 101.330 | 52.884 | 0.547 | 47.685 | 33.452 |
| 17 | -0.234 | 0.168 | 0.030 | -0.036 | 1.446 | 91.446 | 101.377 | 101.450 | 53.948 | 0.545 | 47.648 | 34.176 |
| 18 | -0.264 | 0.186 | 0.022 | -0.057 | 1.619 | 91.266 | 101.382 | 101.250 | 55.308 | 0.546 | 47.579 | 35.234 |
| 19 | -0.314 | 0.215 | 0.020 | -0.079 | 1.783 | 91.454 | 101.385 | 101.186 | 56.446 | 0.545 | 47.463 | 36.030 |
| 20 | -0.367 | 0.243 | 0.008 | -0.116 | 2.040 | 91.488 | 101.321 | 101.507 | 57.957 | 0.541 | 47.470 | 37.074 |
| 21 | -0.382 | 0.251 | 0.003 | -0.128 | 2.118 | 91.545 | 101.355 | 101.515 | 58.471 | 0.541 | 47.437 | 37.440 |
| 22 | -0.427 | 0.277 | -0.012 | -0.162 | 2.347 | 91.504 | 101.346 | 101.421 | 59.266 | 0.541 | 47.391 | 38.064 |
| 23 | -0.492 | 0.313 | -0.030 | -0.208 | 2.642 | 91.314 | 101.322 | 100.967 | 60.056 | 0.543 | 47.297 | 38.806 |
| 24 | -0.522 | 0.327 | -0.037 | -0.231 | 2.788 | 91.534 | 101.365 | 101.688 | 60.431 | 0.539 | 47.401 | 38.882 |

Torsion Shear Test on Fine Nevada Sand with a Target Void Ratio $=0.530\left(D_{r}=91.28 \%\right)$

| Test No.: | 11 |
| :--- | :--- |
| Test Date: | $9 / 22 / 11$ |
| Initial Height, $\mathrm{h}_{\mathrm{i}}$ : | 39.99 cm |
| Initial Void Ratio, e: | 0.540 |
| Final Mean Princpal Stress, $\sigma_{\mathrm{m}}:$ | 101.6 kPa |
| Max Friction Angle, $\phi:$ | 39.6 deg |
| b-value at failure: | 0.75 |
| Stress direction at failure, $\alpha:$ | 45.0 deg |


| Shear Band Notes <br> Point of Observation: <br>  <br> Inclination (from Vertical) <br> $25^{\circ}, 40^{\circ}, 50^{\circ}, 3 @ 60^{\circ}, 65^{\circ}, 69^{\circ}, 5 @ 70^{\circ}, 73^{\circ}$ <br> Failure Notes: |
| :--- |


| Point (No.) | $\begin{gathered} \varepsilon_{\mathrm{z}} \\ (\%) \end{gathered}$ | $\begin{gathered} \varepsilon_{\mathrm{r}} \\ (\%) \end{gathered}$ | $\begin{gathered} \varepsilon_{\theta} \\ (\%) \\ \hline \end{gathered}$ | $\begin{gathered} \varepsilon_{\mathrm{v}} \\ (\%) \end{gathered}$ | $\begin{aligned} & \gamma_{\theta z} \\ & (\%) \end{aligned}$ | $\begin{gathered} \sigma_{z} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \sigma_{\mathrm{r}} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \sigma_{\theta} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \tau_{\theta z} \\ (\mathrm{kPa}) \end{gathered}$ | b | $\begin{gathered} \alpha \\ \left({ }^{\circ}\right) \end{gathered}$ | $\begin{gathered} \varphi \\ \left({ }^{\circ}\right) \end{gathered}$ |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 1 | 0.000 | 0.001 | 0.001 | 0.002 | 0.005 | 100.860 | 101.643 | 101.450 | 1.710 | 0.641 | 49.889 | 0.983 |
| 2 | 0.000 | 0.007 | 0.000 | 0.007 | 0.038 | 99.992 | 104.435 | 99.872 | 10.187 | 0.721 | 44.831 | 5.851 |
| 3 | 0.000 | 0.014 | 0.002 | 0.016 | 0.076 | 98.665 | 106.978 | 98.565 | 17.094 | 0.745 | 44.917 | 9.982 |
| 4 | 0.002 | 0.019 | 0.001 | 0.022 | 0.124 | 98.137 | 108.853 | 97.971 | 23.786 | 0.727 | 44.900 | 14.039 |
| 5 | -0.001 | 0.026 | 0.002 | 0.027 | 0.164 | 97.265 | 110.399 | 97.183 | 27.914 | 0.736 | 44.958 | 16.685 |
| 6 | -0.007 | 0.038 | 0.004 | 0.035 | 0.228 | 96.808 | 111.870 | 96.741 | 33.196 | 0.727 | 44.971 | 20.062 |
| 7 | -0.012 | 0.047 | 0.003 | 0.038 | 0.268 | 96.159 | 112.950 | 96.196 | 35.743 | 0.735 | 45.015 | 21.816 |
| 8 | -0.023 | 0.065 | 0.002 | 0.044 | 0.356 | 95.270 | 114.220 | 95.199 | 39.202 | 0.742 | 44.974 | 24.308 |
| 9 | -0.035 | 0.081 | 0.001 | 0.048 | 0.430 | 94.936 | 114.985 | 94.845 | 41.426 | 0.743 | 44.968 | 25.885 |
| 10 | -0.048 | 0.098 | -0.002 | 0.048 | 0.515 | 94.470 | 115.695 | 94.280 | 43.373 | 0.746 | 44.937 | 27.360 |
| 11 | -0.065 | 0.119 | -0.005 | 0.049 | 0.617 | 94.410 | 116.512 | 94.359 | 45.674 | 0.742 | 44.984 | 28.942 |
| 12 | -0.085 | 0.141 | -0.011 | 0.045 | 0.723 | 93.810 | 117.071 | 93.590 | 47.128 | 0.748 | 44.933 | 30.197 |
| 13 | -0.114 | 0.171 | -0.018 | 0.039 | 0.870 | 93.559 | 117.682 | 93.521 | 49.050 | 0.746 | 44.989 | 31.626 |
| 14 | -0.142 | 0.197 | -0.026 | 0.030 | 0.999 | 93.387 | 118.143 | 93.288 | 50.326 | 0.746 | 44.972 | 32.628 |
| 15 | -0.189 | 0.238 | -0.039 | 0.010 | 1.221 | 93.624 | 118.588 | 93.680 | 52.776 | 0.736 | 45.015 | 34.300 |
| 16 | -0.201 | 0.249 | -0.045 | 0.004 | 1.272 | 93.076 | 119.044 | 93.090 | 52.631 | 0.747 | 45.004 | 34.432 |
| 17 | -0.241 | 0.283 | -0.059 | -0.017 | 1.439 | 92.831 | 119.236 | 92.643 | 53.551 | 0.747 | 44.950 | 35.271 |
| 18 | -0.280 | 0.315 | -0.074 | -0.039 | 1.604 | 92.714 | 119.540 | 92.586 | 54.520 | 0.747 | 44.967 | 36.047 |
| 19 | -0.337 | 0.360 | -0.092 | -0.069 | 1.818 | 92.435 | 119.890 | 92.356 | 55.380 | 0.748 | 44.980 | 36.826 |
| 20 | -0.375 | 0.390 | -0.107 | -0.092 | 1.969 | 92.303 | 120.088 | 92.238 | 56.050 | 0.748 | 44.983 | 37.406 |
| 21 | -0.424 | 0.429 | -0.125 | -0.120 | 2.157 | 92.352 | 120.327 | 92.102 | 56.819 | 0.747 | 44.937 | 38.030 |
| 22 | -0.467 | 0.466 | -0.150 | -0.151 | 2.362 | 92.332 | 120.656 | 92.054 | 57.664 | 0.747 | 44.931 | 38.717 |
| 23 | -0.504 | 0.498 | -0.171 | -0.178 | 2.532 | 92.309 | 120.819 | 91.988 | 58.145 | 0.747 | 44.921 | 39.124 |
| 24 | -0.543 | 0.532 | -0.202 | -0.213 | 2.743 | 92.094 | 120.976 | 92.062 | 58.580 | 0.747 | 44.992 | 39.509 |
| 25 | -0.561 | 0.545 | -0.208 | -0.224 | 2.816 | 91.992 | 120.956 | 91.925 | 58.675 | 0.747 | 44.984 | 39.648 |
| 26 | -0.600 | 0.567 | -0.213 | -0.246 | 3.002 | 92.080 | 120.553 | 91.720 | 56.614 | 0.753 | 44.909 | 38.028 |
| 27 | -0.608 | 0.572 | -0.216 | -0.252 | 3.104 | 92.492 | 120.272 | 92.238 | 55.953 | 0.749 | 44.935 | 37.285 |
| 28 | -0.621 | 0.578 | -0.213 | -0.257 | 3.210 | 92.469 | 120.000 | 92.211 | 55.477 | 0.749 | 44.933 | 36.927 |
| 29 | -0.628 | 0.581 | -0.212 | -0.259 | 3.265 | 92.417 | 120.028 | 92.069 | 55.377 | 0.751 | 44.910 | 36.894 |

## Torsion Shear Test on Fine Nevada Sand with a Target Void Ratio $=\mathbf{0 . 5 3 0}\left(\mathrm{D}_{\mathrm{r}}=\mathbf{9 1 . 2 8 \%}\right)$

| Test No.: | $12^{*}$ |
| :--- | :---: |
| Test Date: | $12 / 28 / 10$ |
| Initial Height, $\mathrm{h}_{\mathrm{i}}$ : | 40.09 cm |
| Initial Void Ratio, e: | 0.559 |
| Final Mean Princpal Stress, $\sigma_{\mathrm{m}}:$ | 98.2 kPa |
| Max Friction Angle, $\phi:$ | 38.9 deg |
| b-value at failure: | 0.80 |
| Stress direction at failure, $\alpha:$ | 47.9 deg |


| Shear Band Notes |
| :--- |
| Point of Observation: |
|  |
| Inclination (from Vertical) |
| $14^{\circ}, 15^{\circ}, 18^{\circ}, 19^{\circ}, 2 @ 20^{\circ}, 22^{\circ}, 30^{\circ}$ |
| Failure Notes: |
| SBs spriraled through specimen height |


| Point (No.) | $\begin{gathered} \varepsilon_{\mathrm{z}} \\ (\%) \end{gathered}$ | $\begin{gathered} \varepsilon_{\mathrm{r}} \\ (\%) \end{gathered}$ | $\begin{gathered} \varepsilon_{\theta} \\ (\%) \end{gathered}$ | $\begin{gathered} \varepsilon_{\mathrm{v}} \\ (\%) \\ \hline \end{gathered}$ | $\begin{aligned} & \gamma_{\theta z} \\ & (\%) \end{aligned}$ | $\begin{gathered} \sigma_{z} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \sigma_{\mathrm{r}} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \sigma_{\theta} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \tau_{\theta z} \\ (\mathrm{kPa}) \end{gathered}$ | b | $\alpha$ <br> ${ }^{\circ}$ ) | $\begin{gathered} \varphi \\ \left(^{( }\right) \\ \hline \end{gathered}$ |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 1 | 0.000 | 0.000 | 0.000 | 0.000 | 0.019 | 91.135 | 101.709 | 101.079 | 0.742 | 0.946 | 85.759 | 2.998 |
| 2 | 0.000 | 0.007 | 0.002 | 0.009 | 0.046 | 90.117 | 104.372 | 100.273 | 9.092 | 0.941 | 59.591 | 6.281 |
| 3 | 0.000 | 0.015 | 0.003 | 0.019 | 0.067 | 88.856 | 105.931 | 99.015 | 13.222 | 0.923 | 55.508 | 8.672 |
| 4 | 0.000 | 0.021 | 0.006 | 0.027 | 0.093 | 88.461 | 107.389 | 98.704 | 17.589 | 0.877 | 53.117 | 11.289 |
| 5 | 0.004 | 0.027 | 0.003 | 0.035 | 0.121 | 87.884 | 108.508 | 98.107 | 21.283 | 0.854 | 51.753 | 13.614 |
| 6 | 0.009 | 0.032 | 0.002 | 0.043 | 0.150 | 86.997 | 109.604 | 97.228 | 24.364 | 0.851 | 50.928 | 15.681 |
| 7 | 0.013 | 0.036 | 0.000 | 0.049 | 0.181 | 86.368 | 110.483 | 96.962 | 26.985 | 0.842 | 50.553 | 17.458 |
| 8 | 0.015 | 0.044 | -0.005 | 0.055 | 0.216 | 86.039 | 111.328 | 96.377 | 29.502 | 0.836 | 49.969 | 19.171 |
| 9 | 0.021 | 0.051 | -0.011 | 0.061 | 0.256 | 85.456 | 112.046 | 95.905 | 31.741 | 0.832 | 49.673 | 20.778 |
| 10 | 0.023 | 0.058 | -0.015 | 0.065 | 0.296 | 85.382 | 112.819 | 95.974 | 33.715 | 0.824 | 49.464 | 22.109 |
| 11 | 0.024 | 0.069 | -0.024 | 0.069 | 0.351 | 85.156 | 113.428 | 95.350 | 35.813 | 0.820 | 49.050 | 23.629 |
| 12 | 0.026 | 0.080 | -0.035 | 0.072 | 0.421 | 85.003 | 114.052 | 95.315 | 38.051 | 0.811 | 48.858 | 25.208 |
| 13 | 0.022 | 0.091 | -0.041 | 0.072 | 0.466 | 84.754 | 114.598 | 95.381 | 39.205 | 0.810 | 48.859 | 26.057 |
| 14 | 0.015 | 0.110 | -0.054 | 0.071 | 0.557 | 84.300 | 115.189 | 94.596 | 41.023 | 0.811 | 48.576 | 27.530 |
| 15 | 0.000 | 0.126 | -0.061 | 0.064 | 0.620 | 84.078 | 115.700 | 94.762 | 42.175 | 0.809 | 48.609 | 28.386 |
| 16 | -0.008 | 0.139 | -0.072 | 0.059 | 0.700 | 83.708 | 116.028 | 94.311 | 43.284 | 0.810 | 48.492 | 29.335 |
| 17 | -0.020 | 0.156 | -0.088 | 0.048 | 0.803 | 83.344 | 116.492 | 94.141 | 44.566 | 0.809 | 48.453 | 30.389 |
| 18 | -0.030 | 0.172 | -0.103 | 0.039 | 0.901 | 83.178 | 116.823 | 93.761 | 45.588 | 0.809 | 48.311 | 31.249 |
| 19 | -0.062 | 0.207 | -0.133 | 0.012 | 1.111 | 82.941 | 117.483 | 93.452 | 47.461 | 0.807 | 48.159 | 32.780 |
| 20 | -0.078 | 0.225 | -0.150 | -0.004 | 1.233 | 82.938 | 117.811 | 93.755 | 48.494 | 0.802 | 48.182 | 33.525 |
| 21 | -0.101 | 0.247 | -0.171 | -0.024 | 1.375 | 82.551 | 118.088 | 93.177 | 49.210 | 0.805 | 48.081 | 34.286 |
| 22 | -0.158 | 0.297 | -0.210 | -0.071 | 1.681 | 82.671 | 118.652 | 93.480 | 50.865 | 0.799 | 48.033 | 35.504 |
| 23 | -0.184 | 0.319 | -0.228 | -0.093 | 1.819 | 82.263 | 118.820 | 93.088 | 51.405 | 0.801 | 48.005 | 36.125 |
| 24 | -0.281 | 0.396 | -0.285 | -0.170 | 2.250 | 81.934 | 119.139 | 92.884 | 52.472 | 0.801 | 47.978 | 37.126 |
| 25 | -0.388 | 0.482 | -0.347 | -0.253 | 2.705 | 81.917 | 119.609 | 92.805 | 53.580 | 0.799 | 47.901 | 38.059 |
| 26 | -0.445 | 0.533 | -0.386 | -0.298 | 2.957 | 81.920 | 119.790 | 92.781 | 54.011 | 0.799 | 47.871 | 38.421 |
| 27 | -0.574 | 0.653 | -0.474 | -0.394 | 3.484 | 81.917 | 120.145 | 92.385 | 54.634 | 0.801 | 47.736 | 39.032 |
| 28 | -0.582 | 0.663 | -0.482 | -0.402 | 3.528 | 82.129 | 120.185 | 92.993 | 54.748 | 0.796 | 47.833 | 38.927 |
| 29 | -0.619 | 0.697 | -0.503 | -0.425 | 3.662 | 82.079 | 120.214 | 93.168 | 54.760 | 0.796 | 47.891 | 38.913 |
| 30 | -0.661 | 0.738 | -0.530 | -0.453 | 3.821 | 81.967 | 120.101 | 92.680 | 54.418 | 0.800 | 47.811 | 38.769 |
| 31 | -0.709 | 0.792 | -0.568 | -0.485 | 4.032 | 81.806 | 119.931 | 92.627 | 53.990 | 0.801 | 47.861 | 38.473 |
| 32 | -0.728 | 0.814 | -0.582 | -0.496 | 4.116 | 82.023 | 119.910 | 92.871 | 53.804 | 0.800 | 47.878 | 38.199 |
| 33 | -0.756 | 0.862 | -0.623 | -0.517 | 4.311 | 81.112 | 119.820 | 92.867 | 52.989 | 0.808 | 48.165 | 37.798 |

# Torsion Shear Test on Fine Nevada Sand with a Target Void Ratio $=\mathbf{0 . 5 3 0}\left(\mathrm{D}_{\mathrm{r}}=\mathbf{9 1 . 2 8 \%}\right)$ 

| Test No.: | $13^{*}$ |
| :--- | :---: |
| Test Date: | $9 / 3 / 11$ |
| Initial Height, $\mathrm{h}_{\mathrm{i}}$ : | 40.03 cm |
| Initial Void Ratio, e: | 0.553 |
| Final Mean Princpal Stress, $\sigma_{\mathrm{m}}:$ | 97.98 kPa |
| Max Friction Angle, $\phi:$ | 39.3 deg |
| b-value at failure: | 0.96 |
| Stress direction at failure, $\alpha:$ | 48.2 deg |


| Shear Band Notes |  |
| :--- | :--- |
| Point of Observation: | 31 |
| Inclination (from Vertical) |  |
| top cap slip none observed |  |
| Failure Notes: |  |


| Point (No.) | $\begin{gathered} \varepsilon_{\mathrm{z}} \\ (\%) \\ \hline \end{gathered}$ | $\begin{gathered} \varepsilon_{\mathrm{r}} \\ (\%) \\ \hline \end{gathered}$ | $\begin{gathered} \varepsilon_{\theta} \\ (\%) \\ \hline \end{gathered}$ | $\begin{gathered} \varepsilon_{\mathrm{v}} \\ (\%) \\ \hline \end{gathered}$ | $\begin{array}{r} \gamma_{\theta z} \\ (\%) \\ \hline \end{array}$ | $\begin{gathered} \sigma_{z} \\ (\mathrm{kPa}) \\ \hline \end{gathered}$ | $\begin{gathered} \sigma_{r} \\ (\mathrm{kPa}) \\ \hline \end{gathered}$ | $\begin{gathered} \sigma_{\theta} \\ (\mathrm{kPa}) \\ \hline \end{gathered}$ | $\begin{gathered} \tau_{\theta z} \\ (\mathrm{kPa}) \\ \hline \end{gathered}$ | b | $\begin{gathered} \alpha \\ \left({ }^{\circ}\right) \\ \hline \end{gathered}$ | $\begin{gathered} \varphi \\ \left(^{\circ}\right) \\ \hline \end{gathered}$ |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 1 | 0.000 | 0.000 | 0.000 | 0.000 | 0.011 | 91.758 | 101.378 | 101.416 | 1.483 | 0.974 | 81.464 | 2.924 |
| 2 | -0.002 | 0.007 | -0.001 | 0.004 | 0.021 | 89.212 | 105.418 | 99.737 | 7.445 | 0.909 | 62.627 | 6.036 |
| 3 | -0.003 | 0.009 | -0.001 | 0.005 | 0.027 | 88.590 | 106.719 | 98.649 | 9.932 | 0.919 | 58.429 | 7.359 |
| 4 | -0.004 | 0.013 | 0.000 | 0.008 | 0.032 | 87.445 | 108.364 | 97.732 | 11.895 | 0.902 | 56.692 | 8.792 |
| 5 | -0.008 | 0.019 | 0.000 | 0.012 | 0.041 | 86.536 | 110.538 | 96.904 | 14.894 | 0.912 | 54.596 | 10.689 |
| 6 | -0.010 | 0.023 | 0.000 | 0.013 | 0.046 | 86.114 | 111.752 | 96.464 | 16.727 | 0.922 | 53.596 | 11.810 |
| 7 | -0.014 | 0.029 | 0.001 | 0.017 | 0.053 | 85.176 | 112.983 | 95.995 | 18.688 | 0.930 | 53.072 | 13.139 |
| 8 | -0.019 | 0.036 | 0.001 | 0.018 | 0.070 | 84.095 | 114.291 | 95.163 | 21.000 | 0.936 | 52.382 | 14.747 |
| 9 | -0.027 | 0.045 | 0.004 | 0.021 | 0.096 | 83.508 | 115.969 | 93.933 | 23.738 | 0.943 | 51.193 | 16.606 |
| 10 | -0.045 | 0.065 | 0.007 | 0.027 | 0.166 | 82.419 | 117.755 | 93.281 | 28.187 | 0.980 | 50.453 | 19.349 |
| 11 | -0.070 | 0.093 | 0.009 | 0.032 | 0.267 | 80.750 | 120.676 | 91.884 | 31.821 | 0.969 | 49.962 | 22.433 |
| 12 | -0.087 | 0.113 | 0.008 | 0.034 | 0.354 | 79.803 | 123.139 | 90.683 | 33.792 | 0.949 | 49.573 | 24.465 |
| 13 | -0.113 | 0.139 | 0.008 | 0.034 | 0.434 | 78.455 | 123.892 | 90.787 | 35.564 | 0.958 | 49.918 | 25.919 |
| 14 | -0.146 | 0.170 | 0.007 | 0.032 | 0.535 | 78.587 | 125.645 | 89.892 | 37.507 | 0.956 | 49.285 | 27.476 |
| 15 | -0.216 | 0.233 | 0.002 | 0.019 | 0.739 | 77.988 | 126.877 | 89.384 | 40.338 | 0.971 | 49.020 | 29.618 |
| 16 | -0.324 | 0.322 | -0.006 | -0.007 | 1.013 | 76.400 | 128.770 | 87.911 | 42.345 | 0.957 | 48.870 | 32.089 |
| 17 | -0.420 | 0.403 | -0.027 | -0.045 | 1.295 | 76.018 | 130.175 | 87.783 | 44.372 | 0.962 | 48.776 | 33.782 |
| 18 | -0.525 | 0.487 | -0.052 | -0.090 | 1.582 | 75.205 | 131.539 | 86.877 | 45.744 | 0.955 | 48.635 | 35.477 |
| 19 | -0.609 | 0.552 | -0.070 | -0.127 | 1.794 | 74.359 | 132.378 | 85.923 | 46.157 | 0.942 | 48.570 | 36.506 |
| 20 | -0.625 | 0.565 | -0.065 | -0.124 | 1.809 | 74.385 | 132.794 | 85.833 | 43.110 | 0.904 | 48.782 | 34.588 |
| 21 | -0.623 | 0.564 | -0.065 | -0.124 | 1.809 | 75.686 | 130.833 | 86.521 | 42.795 | 0.929 | 48.607 | 33.378 |
| 22 | -0.623 | 0.565 | -0.065 | -0.122 | 1.809 | 75.543 | 130.672 | 87.058 | 42.714 | 0.932 | 48.838 | 33.202 |
| 23 | -0.623 | 0.565 | -0.065 | -0.122 | 1.809 | 75.184 | 130.687 | 86.614 | 41.874 | 0.918 | 48.886 | 32.931 |
| 24 | -0.620 | 0.564 | -0.059 | -0.115 | 1.794 | 75.704 | 130.056 | 87.135 | 36.422 | 0.862 | 49.459 | 29.320 |
| 25 | -0.619 | 0.561 | -0.058 | -0.116 | 1.794 | 77.784 | 126.387 | 88.651 | 37.294 | 0.932 | 49.145 | 28.055 |
| 26 | -0.617 | 0.562 | -0.059 | -0.115 | 1.794 | 77.730 | 126.852 | 88.387 | 37.533 | 0.928 | 49.040 | 28.360 |
| 27 | -0.557 | 0.548 | -0.025 | -0.034 | 1.683 | 78.058 | 125.684 | 89.759 | 40.787 | 0.993 | 49.081 | 29.524 |
| 28 | -0.603 | 0.575 | -0.041 | -0.069 | 1.836 | 76.785 | 129.138 | 88.236 | 47.575 | 0.987 | 48.431 | 35.272 |
| 29 | -0.756 | 0.683 | -0.081 | -0.154 | 2.204 | 74.139 | 133.955 | 85.434 | 47.653 | 0.939 | 48.379 | 38.047 |
| 30 | -0.786 | 0.701 | -0.082 | -0.167 | 2.370 | 74.123 | 133.951 | 85.684 | 48.126 | 0.946 | 48.425 | 38.309 |
| 31 | -0.862 | 0.756 | -0.110 | -0.216 | 2.859 | 74.274 | 134.156 | 85.449 | 49.396 | 0.956 | 48.227 | 39.271 |

Torsion Shear Test on Fine Nevada Sand with a Target Void Ratio $=\mathbf{0 . 5 3 0}\left(\mathrm{D}_{\mathrm{r}}=\mathbf{9 1 . 2 8 \%}\right.$ )

| Test No.: | 14 |
| :--- | :--- |
| Test Date: | $9 / 30 / 11$ |
| Initial Height, $\mathrm{h}_{\mathrm{i}}:$ | 39.99 cm |
| Initial Void Ratio, e: | 0.538 |
| Final Mean Princpal Stress, $\sigma_{\mathrm{m}}:$ | 101.7 kPa |
| Max Friction Angle, $\phi:$ | 34.7 deg |
| b-value at failure: | 0.01 |
| Stress direction at failure, $\alpha:$ | 67.3 deg |


| Shear Band Notes |
| :--- |
| Point of Observation: |
|  |
| Inclination (from Vertical) |
| $0^{\circ}$ |
|  |
| Failure Notes: |
| Slip along top cap |


| Point <br> (No.) | $\begin{gathered} \varepsilon_{\mathrm{z}} \\ (\%) \end{gathered}$ | $\begin{gathered} \varepsilon_{\mathrm{r}} \\ (\%) \\ \hline \end{gathered}$ | $\begin{gathered} \varepsilon_{\theta} \\ (\%) \end{gathered}$ | $\begin{gathered} \varepsilon_{\mathrm{v}} \\ (\%) \end{gathered}$ | $\begin{gathered} \gamma_{\theta z} \\ (\%) \\ \hline \end{gathered}$ | $\begin{gathered} \sigma_{z} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \sigma_{\mathrm{r}} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \sigma_{\theta} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \tau_{\theta z} \\ (\mathrm{kPa}) \end{gathered}$ | b | $\begin{gathered} \alpha \\ \left({ }^{\circ}\right) \end{gathered}$ | $\begin{gathered} \varphi \\ \left({ }^{\circ}\right) \\ \hline \end{gathered}$ |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 1 | -0.003 | 0.001 | 0.001 | 0.000 | 0.005 | 100.717 | 100.199 | 102.485 | 1.243 | 0.040 | 62.706 | 0.860 |
| 2 | -0.018 | 0.000 | 0.018 | 0.001 | 0.038 | 98.311 | 94.999 | 111.187 | 7.686 | 0.014 | 64.974 | 5.493 |
| 3 | -0.031 | 0.000 | 0.034 | 0.002 | 0.073 | 95.421 | 90.913 | 116.904 | 12.639 | 0.040 | 65.180 | 8.989 |
| 4 | -0.044 | -0.005 | 0.055 | 0.006 | 0.114 | 93.151 | 86.776 | 123.456 | 16.784 | 0.024 | 66.038 | 12.051 |
| 5 | -0.058 | -0.012 | 0.082 | 0.012 | 0.165 | 90.805 | 82.697 | 129.802 | 21.016 | 0.019 | 66.428 | 15.064 |
| 6 | -0.070 | -0.023 | 0.111 | 0.017 | 0.238 | 89.162 | 79.368 | 134.969 | 23.993 | 0.007 | 66.835 | 17.217 |
| 7 | -0.083 | -0.035 | 0.140 | 0.021 | 0.316 | 87.214 | 76.583 | 139.267 | 27.179 | 0.013 | 66.879 | 19.409 |
| 8 | -0.102 | -0.053 | 0.181 | 0.026 | 0.415 | 85.819 | 73.517 | 144.078 | 30.696 | 0.010 | 66.750 | 21.601 |
| 9 | -0.120 | -0.068 | 0.217 | 0.029 | 0.489 | 84.615 | 71.367 | 147.770 | 32.067 | 0.002 | 67.280 | 22.788 |
| 10 | -0.138 | -0.087 | 0.255 | 0.029 | 0.570 | 83.644 | 69.821 | 149.861 | 34.406 | 0.009 | 66.950 | 24.140 |
| 11 | -0.163 | -0.112 | 0.303 | 0.027 | 0.659 | 82.356 | 68.019 | 152.370 | 36.128 | 0.010 | 67.049 | 25.381 |
| 12 | -0.194 | -0.145 | 0.362 | 0.022 | 0.764 | 81.312 | 66.283 | 155.039 | 37.727 | 0.008 | 67.169 | 26.509 |
| 13 | -0.245 | -0.193 | 0.448 | 0.010 | 0.914 | 80.406 | 64.417 | 158.163 | 39.748 | 0.007 | 67.183 | 27.782 |
| 14 | -0.276 | -0.225 | 0.500 | -0.001 | 1.003 | 79.926 | 63.357 | 159.983 | 40.640 | 0.004 | 67.283 | 28.395 |
| 15 | -0.331 | -0.271 | 0.583 | -0.020 | 1.144 | 78.881 | 62.040 | 161.366 | 41.835 | 0.006 | 67.296 | 29.278 |
| 16 | -0.378 | -0.311 | 0.651 | -0.038 | 1.256 | 78.450 | 61.202 | 162.795 | 42.748 | 0.005 | 67.306 | 29.856 |
| 17 | -0.446 | -0.377 | 0.755 | -0.068 | 1.428 | 77.597 | 59.602 | 165.267 | 43.463 | -0.001 | 67.622 | 30.554 |
| 18 | -0.491 | -0.418 | 0.820 | -0.089 | 1.542 | 77.057 | 59.165 | 165.455 | 44.336 | 0.004 | 67.456 | 31.084 |
| 19 | -0.535 | -0.458 | 0.882 | -0.110 | 1.650 | 77.108 | 58.719 | 166.675 | 45.092 | 0.003 | 67.401 | 31.425 |
| 20 | -0.590 | -0.509 | 0.962 | -0.137 | 1.787 | 77.047 | 58.132 | 168.070 | 45.822 | 0.001 | 67.403 | 31.800 |
| 21 | -0.645 | -0.558 | 1.037 | -0.166 | 1.918 | 76.457 | 57.638 | 168.221 | 46.507 | 0.005 | 67.306 | 32.277 |
| 22 | -0.714 | -0.625 | 1.135 | -0.205 | 2.082 | 76.301 | 56.944 | 169.723 | 47.094 | 0.002 | 67.383 | 32.631 |
| 23 | -0.770 | -0.677 | 1.211 | -0.236 | 2.211 | 76.162 | 56.533 | 170.464 | 47.577 | 0.002 | 67.371 | 32.902 |
| 24 | -0.953 | -0.844 | 1.454 | -0.343 | 2.630 | 75.376 | 55.449 | 171.895 | 48.667 | 0.003 | 67.380 | 33.667 |
| 25 | -1.009 | -0.894 | 1.526 | -0.377 | 2.753 | 75.145 | 55.234 | 172.181 | 48.973 | 0.004 | 67.366 | 33.880 |
| 26 | -1.059 | -0.965 | 1.606 | -0.418 | 2.902 | 75.126 | 54.957 | 172.781 | 49.328 | 0.003 | 67.354 | 34.052 |
| 27 | -1.140 | -1.053 | 1.719 | -0.473 | 3.103 | 74.792 | 54.490 | 173.412 | 49.480 | 0.002 | 67.451 | 34.255 |
| 28 | -1.171 | -1.093 | 1.766 | -0.498 | 3.197 | 74.905 | 54.543 | 173.270 | 49.863 | 0.004 | 67.303 | 34.362 |
| 29 | -1.226 | -1.169 | 1.853 | -0.542 | 3.357 | 74.750 | 54.165 | 174.131 | 49.764 | 0.000 | 67.479 | 34.411 |
| 30 | -1.268 | -1.223 | 1.918 | -0.574 | 3.488 | 74.502 | 54.106 | 174.082 | 49.854 | 0.002 | 67.481 | 34.533 |
| 31 | -1.314 | -1.288 | 1.990 | -0.613 | 3.636 | 74.495 | 54.042 | 174.083 | 50.133 | 0.003 | 67.403 | 34.646 |
| 32 | -1.365 | -1.354 | 2.066 | -0.652 | 3.786 | 74.419 | 53.996 | 173.813 | 50.205 | 0.004 | 67.354 | 34.692 |
| 33 | -1.367 | -1.356 | 2.069 | -0.654 | 3.791 | 74.346 | 54.002 | 173.691 | 50.259 | 0.005 | 67.332 | 34.735 |

# Torsion Shear Test on Fine Nevada Sand with a Target Void Ratio $=\mathbf{0 . 5 3 0}\left(\mathrm{D}_{\mathrm{r}}=\mathbf{9 1 . 2 8 \%}\right)$ 

| Test No.: | 15 |
| :--- | :--- |
| Test Date: | $10 / 7 / 11$ |
| Initial Height, $\mathrm{h}_{\mathrm{h}}$ : | 39.96 cm |
| Initial Void Ratio, e: | 0.525 dPa |
| Final Mean Princpal Stress, $\sigma_{\mathrm{m}}:$ | 101.3 kPa |
| Max Friction Angle, $\phi:$ | 39.5 deg |
| b-value at failure: | 0.50 |
| Stress direction at failure, $\alpha:$ | 67.5 deg |$\quad$| Shear Band Notes |
| :--- |
| Point of Observation: |
|  |
| Inclination (from Vertical) |
|  |


| Point (No.) | $\begin{gathered} \varepsilon_{\mathrm{z}} \\ (\%) \\ \hline \end{gathered}$ | $\begin{gathered} \varepsilon_{\mathrm{r}} \\ (\%) \end{gathered}$ | $\begin{gathered} \varepsilon_{\theta} \\ (\%) \\ \hline \end{gathered}$ | $\begin{gathered} \varepsilon_{\mathrm{v}} \\ (\%) \end{gathered}$ | $\begin{array}{r} \gamma_{\theta z} \\ (\%) \\ \hline \end{array}$ | $\begin{gathered} \sigma_{z} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \sigma_{\mathrm{r}} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \sigma_{\theta} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \tau_{\theta z} \\ (\mathrm{kPa}) \end{gathered}$ | b | $\begin{gathered} \alpha \\ \left({ }^{\circ}\right) \end{gathered}$ | $\begin{gathered} \varphi \\ \left({ }^{\circ}\right) \\ \hline \end{gathered}$ |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 1 | 0.000 | 0.000 | 0.000 | 0.000 | 0.000 | 100.756 | 101.328 | 102.781 | 1.256 | 0.364 | 64.433 | 0.909 |
| 2 | 0.007 | -0.005 | 0.000 | 0.002 | 0.006 | 94.352 | 101.466 | 107.409 | 7.008 | 0.531 | 66.486 | 5.448 |
| 3 | -0.001 | -0.002 | 0.008 | 0.006 | 0.021 | 90.281 | 101.353 | 112.283 | 11.523 | 0.502 | 66.837 | 9.050 |
| 4 | -0.003 | -0.002 | 0.014 | 0.009 | 0.046 | 86.629 | 101.366 | 116.150 | 15.458 | 0.499 | 66.839 | 12.169 |
| 5 | -0.007 | -0.002 | 0.022 | 0.013 | 0.080 | 83.089 | 101.364 | 119.268 | 19.124 | 0.504 | 66.704 | 15.080 |
| 6 | -0.014 | 0.000 | 0.032 | 0.019 | 0.134 | 79.488 | 101.362 | 122.691 | 22.666 | 0.504 | 66.811 | 18.043 |
| 7 | -0.019 | 0.001 | 0.040 | 0.022 | 0.174 | 77.459 | 101.384 | 124.716 | 24.655 | 0.504 | 66.891 | 19.744 |
| 8 | -0.027 | 0.000 | 0.055 | 0.028 | 0.232 | 74.784 | 101.311 | 127.680 | 26.648 | 0.501 | 67.392 | 21.770 |
| 9 | -0.037 | -0.001 | 0.072 | 0.035 | 0.293 | 72.902 | 101.378 | 129.902 | 29.094 | 0.500 | 67.204 | 23.681 |
| 10 | -0.047 | 0.002 | 0.084 | 0.039 | 0.359 | 70.892 | 101.382 | 131.521 | 30.857 | 0.502 | 67.246 | 25.303 |
| 11 | -0.069 | 0.008 | 0.103 | 0.043 | 0.438 | 68.791 | 101.384 | 133.645 | 32.445 | 0.502 | 67.492 | 26.949 |
| 12 | -0.111 | 0.022 | 0.131 | 0.043 | 0.531 | 67.612 | 101.372 | 135.105 | 34.253 | 0.500 | 67.287 | 28.320 |
| 13 | -0.157 | 0.035 | 0.165 | 0.042 | 0.627 | 66.399 | 101.305 | 136.777 | 35.645 | 0.497 | 67.316 | 29.542 |
| 14 | -0.221 | 0.051 | 0.208 | 0.038 | 0.754 | 65.194 | 101.317 | 138.769 | 37.238 | 0.494 | 67.326 | 30.882 |
| 15 | -0.269 | 0.064 | 0.236 | 0.031 | 0.847 | 63.691 | 101.349 | 138.903 | 38.068 | 0.500 | 67.325 | 31.887 |
| 16 | -0.328 | 0.078 | 0.271 | 0.021 | 0.962 | 62.745 | 101.360 | 139.926 | 39.118 | 0.500 | 67.306 | 32.837 |
| 17 | -0.396 | 0.094 | 0.308 | 0.006 | 1.087 | 61.727 | 101.399 | 141.063 | 40.052 | 0.500 | 67.362 | 33.777 |
| 18 | -0.486 | 0.116 | 0.354 | -0.016 | 1.251 | 60.640 | 101.378 | 142.017 | 41.002 | 0.500 | 67.390 | 34.755 |
| 19 | -0.572 | 0.137 | 0.396 | -0.040 | 1.419 | 59.749 | 101.427 | 142.885 | 41.732 | 0.501 | 67.444 | 35.547 |
| 20 | -0.635 | 0.156 | 0.423 | -0.057 | 1.561 | 59.138 | 101.405 | 143.143 | 42.402 | 0.502 | 67.364 | 36.164 |
| 21 | -0.702 | 0.175 | 0.453 | -0.075 | 1.689 | 58.508 | 101.410 | 144.183 | 43.285 | 0.501 | 67.351 | 36.934 |
| 22 | -0.787 | 0.188 | 0.491 | -0.108 | 1.859 | 57.870 | 101.407 | 144.774 | 43.967 | 0.501 | 67.331 | 37.596 |
| 23 | -0.874 | 0.212 | 0.529 | -0.133 | 2.037 | 57.054 | 101.429 | 145.376 | 44.389 | 0.502 | 67.426 | 38.216 |
| 24 | -0.984 | 0.239 | 0.580 | -0.166 | 2.268 | 56.591 | 101.374 | 146.460 | 44.859 | 0.499 | 67.524 | 38.711 |
| 25 | -1.095 | 0.264 | 0.627 | -0.204 | 2.532 | 55.921 | 101.356 | 146.612 | 45.139 | 0.501 | 67.566 | 39.185 |
| 26 | -1.183 | 0.285 | 0.663 | -0.235 | 2.752 | 55.807 | 101.403 | 146.523 | 45.583 | 0.502 | 67.429 | 39.468 |
| 27 | -1.211 | 0.291 | 0.673 | -0.247 | 2.833 | 55.774 | 101.397 | 146.874 | 45.658 | 0.501 | 67.466 | 39.532 |
| 28 | -1.230 | 0.296 | 0.681 | -0.253 | 2.875 | 55.857 | 101.436 | 146.864 | 45.612 | 0.501 | 67.466 | 39.467 |
| 29 | -1.235 | 0.296 | 0.682 | -0.257 | 2.905 | 55.948 | 101.431 | 146.938 | 45.704 | 0.500 | 67.434 | 39.472 |

$$
\text { Torsion Shear Test on Fine Nevada Sand with a Target Void Ratio = } 0.530\left(\mathrm{D}_{\mathrm{r}}=91.28 \%\right)
$$

| Test No.: | $16^{*}$ |
| :--- | :---: |
| Test Date: | $9 / 7 / 10$ |
| Initial Height, $\mathrm{h}_{\mathrm{i}}$ : | 40.13 cm |
| Initial Void Ratio, e: | 0.536 |
| Final Mean Princpal Stress, $\sigma_{\mathrm{m}}:$ | 96.6 kPa |
| Max Friction Angle, $\phi:$ | 29.2 deg |
| b-value at failure: | 0.96 |
| Stress direction at failure, $\alpha:$ | 71.5 deg |


| Shear Band Notes |  |
| :--- | :--- |
| Point of Observation: | 30 |
| Inclination (from Vertical) |  |
| top cap slip none observed |  |
| Failure Notes: |  |


| Point <br> (No.) | $\begin{gathered} \varepsilon_{\mathrm{z}} \\ (\%) \\ \hline \end{gathered}$ | $\begin{gathered} \varepsilon_{\mathrm{r}} \\ (\%) \\ \hline \end{gathered}$ | $\begin{gathered} \varepsilon_{\theta} \\ (\%) \end{gathered}$ | $\begin{gathered} \varepsilon_{\mathrm{v}} \\ (\%) \\ \hline \end{gathered}$ | $\begin{aligned} & \gamma_{\theta z} \\ & (\%) \\ & \hline \end{aligned}$ | $\begin{gathered} \sigma_{z} \\ (\mathrm{kPa}) \\ \hline \end{gathered}$ | $\begin{gathered} \sigma_{r} \\ (\mathrm{kPa}) \\ \hline \end{gathered}$ | $\begin{gathered} \sigma_{\theta} \\ (\mathrm{kPa}) \\ \hline \end{gathered}$ | $\begin{gathered} \tau_{\text {өz }} \\ (\mathrm{kPa}) \\ \hline \end{gathered}$ | b | $\begin{gathered} \alpha \\ \left({ }^{\circ}\right) \\ \hline \end{gathered}$ | $\begin{gathered} \varphi \\ \left({ }^{\circ}\right) \\ \hline \end{gathered}$ |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 1 | 0.000 | 0.000 | 0.000 | 0.000 | 0.012 | 91.467 | 101.356 | 101.210 | 1.076 | 0.997 | 83.771 | 2.969 |
| 2 | -0.002 | 0.001 | -0.001 | -0.002 | 0.022 | 88.350 | 102.857 | 101.813 | 4.139 | 0.992 | 74.208 | 4.767 |
| 3 | -0.006 | 0.006 | 0.001 | 0.001 | 0.031 | 83.817 | 105.941 | 103.642 | 5.731 | 0.968 | 74.984 | 7.017 |
| 4 | -0.012 | 0.011 | 0.005 | 0.004 | 0.041 | 81.257 | 107.457 | 104.712 | 9.788 | 0.974 | 70.075 | 9.455 |
| 5 | -0.021 | 0.021 | 0.010 | 0.011 | 0.051 | 77.023 | 110.477 | 106.707 | 9.898 | 0.979 | 73.150 | 11.198 |
| 6 | -0.022 | 0.022 | 0.010 | 0.010 | 0.059 | 74.843 | 111.410 | 106.762 | 11.264 | 0.973 | 72.393 | 12.423 |
| 7 | -0.030 | 0.028 | 0.013 | 0.012 | 0.072 | 72.706 | 112.628 | 107.118 | 12.542 | 0.968 | 71.955 | 13.698 |
| 8 | -0.037 | 0.036 | 0.019 | 0.017 | 0.082 | 70.981 | 113.826 | 107.942 | 13.096 | 0.964 | 72.339 | 14.666 |
| 9 | -0.043 | 0.040 | 0.022 | 0.019 | 0.091 | 70.168 | 114.303 | 108.562 | 13.774 | 0.973 | 72.171 | 15.331 |
| 10 | -0.054 | 0.049 | 0.027 | 0.022 | 0.109 | 68.339 | 115.148 | 108.297 | 15.563 | 0.971 | 71.042 | 16.664 |
| 11 | -0.068 | 0.061 | 0.034 | 0.026 | 0.133 | 66.279 | 116.699 | 109.857 | 16.924 | 0.981 | 71.081 | 18.257 |
| 12 | -0.090 | 0.075 | 0.045 | 0.030 | 0.162 | 64.150 | 117.965 | 110.402 | 18.053 | 0.978 | 71.011 | 19.643 |
| 13 | -0.114 | 0.095 | 0.054 | 0.035 | 0.193 | 62.396 | 119.031 | 110.924 | 18.876 | 0.974 | 71.059 | 20.778 |
| 14 | -0.131 | 0.107 | 0.062 | 0.037 | 0.216 | 61.264 | 119.740 | 111.675 | 19.497 | 0.978 | 71.139 | 21.624 |
| 15 | -0.151 | 0.120 | 0.069 | 0.038 | 0.239 | 60.103 | 120.424 | 111.424 | 19.664 | 0.965 | 71.268 | 22.145 |
| 16 | -0.151 | 0.121 | 0.068 | 0.039 | 0.239 | 60.258 | 120.431 | 111.898 | 19.677 | 0.972 | 71.344 | 22.157 |
| 17 | -0.152 | 0.122 | 0.069 | 0.039 | 0.241 | 60.200 | 120.442 | 111.795 | 19.633 | 0.970 | 71.364 | 22.146 |
| 18 | -0.159 | 0.127 | 0.072 | 0.040 | 0.249 | 60.233 | 120.625 | 112.402 | 20.064 | 0.979 | 71.217 | 22.411 |
| 19 | -0.201 | 0.153 | 0.089 | 0.041 | 0.302 | 58.769 | 121.685 | 113.014 | 21.413 | 0.982 | 70.855 | 23.724 |
| 20 | -0.224 | 0.168 | 0.096 | 0.040 | 0.328 | 57.583 | 122.188 | 113.187 | 21.349 | 0.976 | 71.240 | 24.238 |
| 21 | -0.247 | 0.183 | 0.105 | 0.041 | 0.355 | 57.637 | 122.421 | 113.481 | 21.965 | 0.982 | 70.905 | 24.533 |
| 22 | -0.289 | 0.210 | 0.119 | 0.040 | 0.405 | 56.297 | 123.099 | 113.569 | 22.586 | 0.977 | 70.868 | 25.430 |
| 23 | -0.322 | 0.231 | 0.127 | 0.036 | 0.452 | 55.555 | 123.617 | 114.002 | 22.981 | 0.978 | 70.909 | 26.009 |
| 24 | -0.344 | 0.243 | 0.135 | 0.034 | 0.479 | 55.076 | 123.774 | 113.809 | 23.280 | 0.976 | 70.797 | 26.346 |
| 25 | -0.401 | 0.275 | 0.153 | 0.027 | 0.544 | 53.826 | 124.401 | 114.230 | 23.613 | 0.974 | 70.990 | 27.145 |
| 26 | -0.456 | 0.308 | 0.168 | 0.020 | 0.607 | 53.077 | 125.181 | 114.441 | 23.609 | 0.966 | 71.211 | 27.530 |
| 27 | -0.498 | 0.332 | 0.178 | 0.013 | 0.660 | 52.278 | 125.506 | 114.421 | 23.780 | 0.963 | 71.286 | 27.997 |
| 28 | -0.563 | 0.365 | 0.198 | 0.000 | 0.735 | 51.273 | 126.023 | 114.909 | 23.543 | 0.959 | 71.751 | 28.448 |
| 29 | -0.670 | 0.421 | 0.221 | -0.028 | 0.900 | 50.978 | 126.056 | 115.002 | 24.700 | 0.968 | 71.174 | 29.157 |
| 30 | -0.722 | 0.448 | 0.229 | -0.045 | 1.014 | 50.579 | 126.553 | 115.105 | 24.275 | 0.960 | 71.521 | 29.169 |

Torsion Shear Test on Fine Nevada Sand with a Target Void Ratio $=\mathbf{0 . 5 3 0}\left(\mathrm{D}_{\mathrm{r}}=\mathbf{9 1 . 2 8 \%}\right)$

| Test No.: | 17 |
| :--- | :---: |
| Test Date: | $10 / 5 / 11$ |
| Initial Height, $\mathrm{h}_{\mathrm{i}}:$ | 39.96 cm |
| Initial Void Ratio, e: | 0.532 |
| Final Mean Princpal Stress, $\sigma_{\mathrm{m}}:$ | 101.8 kPa |
| Max Friction Angle, $\phi:$ | 37.9 deg |
| b-value at failure: | 1.01 |
| Stress direction at failure, $\alpha:$ | 68.2 deg |


| Shear Band Notes |
| :--- |
| Point of Observation: |
| Inclination (from Vertical) |
| $0^{\circ}$ |
| Failure Notes: |
| Horizontal trough along specimen |


| Point (No.) | $\begin{gathered} \varepsilon_{\mathrm{z}} \\ (\%) \\ \hline \end{gathered}$ | $\begin{gathered} \varepsilon_{\mathrm{r}} \\ (\%) \\ \hline \end{gathered}$ | $\begin{gathered} \varepsilon_{\theta} \\ (\%) \end{gathered}$ | $\begin{gathered} \varepsilon_{\mathrm{v}} \\ (\%) \\ \hline \end{gathered}$ | $\begin{gathered} \gamma_{\theta z} \\ (\%) \end{gathered}$ | $\begin{gathered} \sigma_{z} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \sigma_{\mathrm{r}} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \sigma_{\theta} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \boldsymbol{\tau}_{\theta z} \\ (\mathrm{kPa}) \\ \hline \end{gathered}$ | b | $\alpha$ <br> ${ }^{\circ}$ ) | $\begin{gathered} \varphi \\ \left({ }^{\circ}\right) \\ \hline \end{gathered}$ |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 1 | -0.011 | 0.013 | 0.006 | 0.007 | 0.000 | 99.819 | 102.771 | 102.300 | 1.403 | 0.957 | 65.740 | . 062 |
| 2 | 0.002 | 0.002 | 0.000 | 0.004 | 0.031 | 85.516 | 111.943 | 107.893 | 12.556 | 0.953 | 65.851 | 10.015 |
| 3 | 0.000 | 0.004 | 0.000 | 0.004 | 0.056 | 80.337 | 114.983 | 109.146 | 15.335 | 0.981 | 66.604 | 12.831 |
| 4 | -0.003 | 0.006 | 0.002 | 0.005 | 0.082 | 77.585 | 116.930 | 110.699 | 17.742 | 0.969 | 66.511 | 14.938 |
| 5 | -0.006 | 0.009 | 0.003 | 0.006 | 0.102 | 75.968 | 118.162 | 111.674 | 19.193 | 0.964 | 66.464 | 16.224 |
| 6 | -0.017 | 0.017 | 0.009 | 0.009 | 0.160 | 70.991 | 120.901 | 112.751 | 21.776 | 0.981 | 66.898 | 19.171 |
| 7 | -0.022 | 0.021 | 0.011 | 0.010 | 0.188 | 69.988 | 121.981 | 113.819 | 23.077 | 0.973 | 66.761 | 20.261 |
| 8 | -0.030 | 0.027 | 0.015 | 0.012 | 0.235 | 67.442 | 123.257 | 113.975 | 24.502 | 0.982 | 66.759 | 21.870 |
| 9 | -0.043 | 0.037 | 0.021 | 0.015 | 0.277 | 65.672 | 124.538 | 114.463 | 25.595 | 0.987 | 66.813 | 23.115 |
| 10 | -0.062 | 0.052 | 0.029 | 0.019 | 0.327 | 64.160 | 125.558 | 115.485 | 26.762 | 0.982 | 66.899 | 24.380 |
| 11 | -0.080 | 0.065 | 0.037 | 0.023 | 0.369 | 62.642 | 126.690 | 116.380 | 27.250 | 0.986 | 67.298 | 25.311 |
| 12 | -0.088 | 0.072 | 0.041 | 0.025 | 0.400 | 61.557 | 127.136 | 116.187 | 27.689 | 0.992 | 67.305 | 25.954 |
| 13 | -0.096 | 0.077 | 0.044 | 0.026 | 0.425 | 60.879 | 127.574 | 116.442 | 28.020 | 0.993 | 67.378 | 26.426 |
| 14 | -0.101 | 0.081 | 0.047 | 0.027 | 0.445 | 60.242 | 127.829 | 116.311 | 28.252 | 0.997 | 67.389 | 26.799 |
| 15 | -0.128 | 0.101 | 0.058 | 0.031 | 0.511 | 59.047 | 128.820 | 116.840 | 29.352 | 0.996 | 67.276 | 27.928 |
| 16 | -0.155 | 0.120 | 0.069 | 0.035 | 0.571 | 57.782 | 129.618 | 117.426 | 29.928 | 0.997 | 67.449 | 28.834 |
| 17 | -0.227 | 0.170 | 0.098 | 0.041 | 0.675 | 56.613 | 130.521 | 117.735 | 31.708 | 0.992 | 66.972 | 30.343 |
| 18 | -0.257 | 0.190 | 0.109 | 0.042 | 0.715 | 55.240 | 131.447 | 118.381 | 31.310 | 1.002 | 67.619 | 30.810 |
| 19 | -0.342 | 0.244 | 0.138 | 0.039 | 0.867 | 54.579 | 131.957 | 118.289 | 33.100 | 0.995 | 66.951 | 32.106 |
| 20 | -0.382 | 0.267 | 0.150 | 0.035 | 0.917 | 52.906 | 132.772 | 119.098 | 32.322 | 1.005 | 67.839 | 32.541 |
| 21 | -0.484 | 0.324 | 0.178 | 0.018 | 1.063 | 51.929 | 133.501 | 119.285 | 32.872 | 1.009 | 67.847 | 33.349 |
| 22 | -0.582 | 0.373 | 0.201 | -0.008 | 1.199 | 51.055 | 133.884 | 119.432 | 33.533 | 1.008 | 67.777 | 34.179 |
| 23 | -0.648 | 0.410 | 0.214 | -0.024 | 1.296 | 50.611 | 134.375 | 119.833 | 34.043 | 1.006 | 67.737 | 34.727 |
| 24 | -0.709 | 0.444 | 0.225 | -0.041 | 1.388 | 51.044 | 133.996 | 119.488 | 35.453 | 0.994 | 66.994 | 35.303 |
| 25 | -0.850 | 0.527 | 0.244 | -0.080 | 1.568 | 49.347 | 135.467 | 120.770 | 35.211 | 1.003 | 67.702 | 36.129 |
| 26 | -0.986 | 0.614 | 0.255 | -0.117 | 1.752 | 50.442 | 134.457 | 120.171 | 36.997 | 0.983 | 66.650 | 36.579 |
| 27 | -1.076 | 0.676 | 0.258 | -0.142 | 1.973 | 49.145 | 135.802 | 121.178 | 36.505 | 0.994 | 67.307 | 37.026 |
| 28 | -1.183 | 0.752 | 0.259 | -0.172 | 2.138 | 49.125 | 136.006 | 120.600 | 37.438 | 0.994 | 66.834 | 37.582 |
| 29 | -1.267 | 0.816 | 0.256 | -0.195 | 2.232 | 46.746 | 137.191 | 121.598 | 35.620 | 1.013 | 68.208 | 37.867 |

# Torsion Shear Test on Fine Nevada Sand with a Target Void Ratio $=\mathbf{0 . 5 3 0}\left(\mathrm{D}_{\mathrm{r}}=\mathbf{9 1 . 2 8 \%}\right)$ 

| Test No.: | 18 |
| :--- | :---: |
| Test Date: | $10 / 13 / 11$ |
| Initial Height, $\mathrm{h}_{\mathrm{i}}:$ | 39.96 cm |
| Initial Void Ratio, e: | 0.53 |
| Final Mean Princpal Stress, $\sigma_{\mathrm{m}}:$ | 100.1 kPa |
| Max Friction Angle, $\phi:$ | 33.9 deg |
| b-value at failure: | 0.00 |
| Stress direction at failure, $\alpha:$ | 90.0 deg |


| Shear Band Notes <br> Point of Observation: | 30 |
| :--- | :--- |
| Inclination (from Vertical) |  |
| $2 @ 25^{\circ}$ |  |
| Failure Notes: |  |
| lower half bulged inwards |  |


| Point (No.) | $\begin{gathered} \varepsilon_{\mathrm{z}} \\ (\%) \end{gathered}$ | $\begin{gathered} \varepsilon_{\mathrm{r}} \\ (\%) \end{gathered}$ | $\begin{gathered} \varepsilon_{\theta} \\ (\%) \\ \hline \end{gathered}$ | $\begin{gathered} \varepsilon_{\mathrm{v}} \\ (\%) \end{gathered}$ | $\begin{gathered} \gamma_{\theta z} \\ (\%) \end{gathered}$ | $\begin{gathered} \sigma_{z} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \sigma_{\mathrm{r}} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \sigma_{\theta} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \tau_{\text {өz }} \\ (\mathrm{kPa}) \end{gathered}$ | b | $\begin{gathered} \alpha \\ \left({ }^{\circ}\right) \end{gathered}$ | $\begin{gathered} \varphi \\ \left({ }^{\circ}\right) \end{gathered}$ |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 1 | 0.000 | 0.000 | 0.000 | 0.000 | 0.000 | 94.902 | 101.305 | 101.611 | 0.000 | 0.954 | 90.000 | 1.956 |
| 2 | -0.002 | -0.012 | 0.009 | -0.004 | 0.000 | 91.093 | 97.562 | 108.963 | 0.000 | 0.362 | 90.000 | 5.125 |
| 3 | -0.007 | -0.023 | 0.022 | -0.008 | 0.000 | 87.734 | 94.173 | 115.920 | 0.000 | 0.228 | 90.000 | 7.955 |
| 4 | -0.014 | -0.033 | 0.037 | -0.009 | 0.000 | 84.676 | 91.091 | 122.250 | 0.000 | 0.171 | 90.000 | 10.462 |
| 5 | -0.023 | -0.040 | 0.057 | -0.007 | 0.000 | 81.875 | 88.119 | 128.292 | 0.000 | 0.135 | 90.000 | 12.759 |
| 6 | -0.034 | -0.049 | 0.079 | -0.004 | 0.000 | 79.205 | 85.441 | 133.805 | 0.000 | 0.114 | 90.000 | 14.852 |
| 7 | -0.045 | -0.058 | 0.103 | -0.001 | 0.000 | 76.668 | 83.051 | 138.810 | 0.000 | 0.103 | 90.000 | 16.762 |
| 8 | -0.057 | -0.067 | 0.127 | 0.003 | 0.000 | 74.356 | 80.812 | 143.139 | 0.000 | 0.094 | 90.000 | 18.436 |
| 9 | -0.072 | -0.079 | 0.159 | 0.007 | 0.000 | 72.198 | 78.460 | 148.017 | 0.000 | 0.083 | 90.000 | 20.139 |
| 10 | -0.095 | -0.091 | 0.198 | 0.012 | 0.000 | 70.301 | 76.551 | 151.691 | 0.000 | 0.077 | 90.000 | 21.508 |
| 11 | -0.119 | -0.105 | 0.241 | 0.017 | 0.000 | 68.787 | 74.819 | 155.527 | 0.000 | 0.070 | 90.000 | 22.749 |
| 12 | -0.150 | -0.119 | 0.291 | 0.022 | 0.000 | 67.081 | 73.209 | 158.617 | 0.000 | 0.067 | 90.000 | 23.927 |
| 13 | -0.186 | -0.137 | 0.349 | 0.026 | 0.000 | 65.550 | 71.647 | 161.784 | 0.000 | 0.063 | 90.000 | 25.044 |
| 14 | -0.221 | -0.156 | 0.405 | 0.028 | 0.000 | 64.163 | 70.373 | 164.138 | 0.000 | 0.062 | 90.000 | 25.970 |
| 15 | -0.267 | -0.176 | 0.473 | 0.030 | 0.000 | 63.180 | 69.302 | 166.604 | 0.000 | 0.059 | 90.000 | 26.750 |
| 16 | -0.320 | -0.202 | 0.553 | 0.030 | 0.000 | 61.972 | 68.040 | 168.975 | 0.000 | 0.057 | 90.000 | 27.602 |
| 17 | -0.360 | -0.223 | 0.612 | 0.028 | 0.000 | 61.100 | 67.220 | 170.671 | 0.000 | 0.056 | 90.000 | 28.213 |
| 18 | -0.432 | -0.261 | 0.715 | 0.023 | 0.000 | 59.822 | 66.015 | 172.847 | 0.000 | 0.055 | 90.000 | 29.063 |
| 19 | -0.480 | -0.283 | 0.782 | 0.019 | 0.000 | 59.385 | 65.508 | 174.190 | 0.000 | 0.053 | 90.000 | 29.440 |
| 20 | -0.542 | -0.314 | 0.867 | 0.011 | 0.000 | 58.802 | 64.762 | 175.853 | 0.000 | 0.051 | 90.000 | 29.922 |
| 21 | -0.594 | -0.341 | 0.938 | 0.003 | 0.000 | 57.979 | 64.134 | 176.872 | 0.000 | 0.052 | 90.000 | 30.414 |
| 22 | -0.664 | -0.378 | 1.031 | -0.010 | 0.000 | 57.295 | 63.471 | 178.281 | 0.000 | 0.051 | 90.000 | 30.902 |
| 23 | -0.713 | -0.403 | 1.098 | -0.019 | 0.000 | 56.969 | 63.043 | 179.293 | 0.000 | 0.050 | 90.000 | 31.181 |
| 24 | -0.765 | -0.432 | 1.167 | -0.030 | 0.000 | 56.450 | 62.548 | 180.093 | 0.000 | 0.049 | 90.000 | 31.514 |
| 25 | -0.844 | -0.476 | 1.270 | -0.050 | 0.000 | 55.495 | 61.855 | 180.968 | 0.000 | 0.051 | 90.000 | 32.048 |
| 26 | -0.887 | -0.501 | 1.326 | -0.061 | 0.000 | 55.056 | 61.588 | 181.231 | 0.000 | 0.052 | 90.000 | 32.276 |
| 27 | -1.016 | -0.565 | 1.491 | -0.089 | 0.000 | 54.826 | 61.062 | 183.609 | 0.000 | 0.048 | 90.000 | 32.692 |
| 28 | -1.065 | -0.594 | 1.555 | -0.104 | 0.000 | 54.537 | 60.573 | 184.213 | 0.000 | 0.047 | 90.000 | 32.898 |
| 29 | -1.104 | -0.614 | 1.605 | -0.113 | 0.000 | 54.405 | 60.353 | 184.685 | 0.000 | 0.046 | 90.000 | 33.018 |
| 30 | -1.175 | -0.653 | 1.694 | -0.134 | 0.000 | 53.795 | 59.953 | 185.132 | 0.000 | 0.047 | 90.000 | 33.346 |
| 31 | -1.222 | -0.681 | 1.754 | -0.149 | 0.000 | 53.481 | 59.729 | 185.355 | 0.000 | 0.047 | 90.000 | 33.515 |
| 32 | -1.274 | -0.710 | 1.819 | -0.164 | 0.000 | 53.349 | 59.558 | 185.874 | 0.000 | 0.047 | 90.000 | 33.641 |
| 33 | -1.400 | -0.787 | 1.993 | -0.194 | 0.000 | 53.365 | 59.068 | 187.747 | 0.000 | 0.042 | 90.000 | 33.872 |

$$
\text { Torsion Shear Test on Fine Nevada Sand with a Target Void Ratio }=0.530\left(D_{r}=91.28 \%\right)
$$

| Test No.: | $19^{*}$ |
| :--- | :---: |
| Test Date: | $7 / 5 / 10$ |
| Initial Height, $\mathrm{h}_{\mathrm{i}}$ : | 39.99 cm |
| Initial Void Ratio, e: | 0.523 |
| Final Mean Princpal Stress, $\sigma_{\mathrm{m}}:$ | 96.7 kPa |
| Max Friction Angle, $\phi:$ | 38.1 deg |
| b-value at failure: | 0.04 |
| Stress direction at failure, $\alpha:$ | 90.0 deg |


| Shear Band Notes |
| :--- |
| Point of Observation: |
|  |
| Inclination (from Vertical) |
| $5 @ 20^{\circ}, 5 @ 25^{\circ}, 15^{\circ}, 30^{\circ}$ |
| Failure Notes: |
| middle caved inwards |


| $\begin{aligned} & \text { Point } \\ & \text { (No.) } \end{aligned}$ | $\begin{gathered} \varepsilon_{\mathrm{z}} \\ (\%) \\ \hline \end{gathered}$ | $\begin{gathered} \varepsilon_{\mathrm{r}} \\ (\%) \\ \hline \end{gathered}$ | $\begin{gathered} \varepsilon_{\theta} \\ (\%) \end{gathered}$ | $\begin{gathered} \varepsilon_{\mathrm{v}} \\ (\%) \\ \hline \end{gathered}$ | $\begin{gathered} \gamma_{\theta z} \\ (\%) \\ \hline \end{gathered}$ | $\begin{gathered} \sigma_{z} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \sigma_{r} \\ (\mathrm{kPa}) \\ \hline \end{gathered}$ | $\begin{gathered} \sigma_{\theta} \\ (\mathrm{kPa}) \\ \hline \end{gathered}$ | $\begin{gathered} \boldsymbol{\tau}_{\theta \mathrm{tz}} \\ (\mathrm{kPa}) \\ \hline \end{gathered}$ | b | $\begin{gathered} \alpha \\ \left({ }^{\circ}\right) \\ \hline \end{gathered}$ | $\begin{gathered} \varphi \\ \left({ }^{\circ}\right) \\ \hline \end{gathered}$ |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 1 | 0.000 | 0.004 | 0.000 | 0.004 | 0.000 | 91.352 | 101.284 | 101.575 | 0.000 | 0.972 | 90.000 | 3.037 |
| 2 | 0.000 | -0.001 | 0.008 | 0.008 | 0.000 | 87.117 | 97.203 | 109.351 | 0.000 | 0.454 | 90.000 | 6.498 |
| 3 | 0.000 | -0.006 | 0.018 | 0.012 | 0.000 | 83.587 | 93.439 | 116.949 | 0.000 | 0.295 | 90.000 | 9.576 |
| 4 | -0.003 | -0.012 | 0.028 | 0.014 | 0.000 | 80.284 | 89.848 | 124.096 | 0.000 | 0.218 | 90.000 | 12.378 |
| 5 | -0.008 | -0.017 | 0.044 | 0.019 | 0.000 | 77.312 | 86.856 | 130.132 | 0.000 | 0.181 | 90.000 | 14.751 |
| 6 | -0.015 | -0.024 | 0.061 | 0.022 | 0.000 | 74.534 | 83.998 | 135.880 | 0.000 | 0.154 | 90.000 | 16.951 |
| 7 | -0.025 | -0.032 | 0.082 | 0.026 | 0.000 | 71.768 | 81.241 | 140.793 | 0.000 | 0.137 | 90.000 | 18.949 |
| 8 | -0.037 | -0.038 | 0.107 | 0.031 | 0.000 | 69.458 | 78.774 | 145.962 | 0.000 | 0.122 | 90.000 | 20.802 |
| 9 | -0.053 | -0.048 | 0.136 | 0.036 | 0.000 | 67.185 | 76.415 | 151.158 | 0.000 | 0.110 | 90.000 | 22.618 |
| 10 | -0.072 | -0.057 | 0.169 | 0.041 | 0.000 | 65.360 | 74.444 | 154.851 | 0.000 | 0.102 | 90.000 | 23.978 |
| 11 | -0.094 | -0.067 | 0.207 | 0.045 | 0.000 | 63.501 | 72.434 | 158.841 | 0.000 | 0.094 | 90.000 | 25.391 |
| 12 | -0.124 | -0.080 | 0.253 | 0.049 | 0.000 | 61.803 | 70.601 | 162.356 | 0.000 | 0.088 | 90.000 | 26.653 |
| 13 | -0.158 | -0.094 | 0.303 | 0.051 | 0.000 | 60.038 | 69.056 | 165.439 | 0.000 | 0.086 | 90.000 | 27.869 |
| 14 | -0.199 | -0.110 | 0.361 | 0.051 | 0.000 | 58.654 | 67.595 | 168.457 | 0.000 | 0.081 | 90.000 | 28.913 |
| 15 | -0.248 | -0.131 | 0.427 | 0.049 | 0.000 | 57.390 | 66.106 | 171.131 | 0.000 | 0.077 | 90.000 | 29.850 |
| 16 | -0.299 | -0.153 | 0.496 | 0.044 | 0.000 | 56.016 | 64.954 | 173.423 | 0.000 | 0.076 | 90.000 | 30.778 |
| 17 | -0.357 | -0.178 | 0.572 | 0.037 | 0.000 | 54.889 | 63.741 | 175.762 | 0.000 | 0.073 | 90.000 | 31.605 |
| 18 | -0.418 | -0.206 | 0.651 | 0.027 | 0.000 | 53.819 | 62.700 | 177.712 | 0.000 | 0.072 | 90.000 | 32.351 |
| 19 | -0.484 | -0.237 | 0.734 | 0.012 | 0.000 | 53.076 | 61.773 | 179.864 | 0.000 | 0.069 | 90.000 | 32.977 |
| 20 | -0.545 | -0.268 | 0.810 | -0.003 | 0.000 | 52.196 | 61.010 | 181.441 | 0.000 | 0.068 | 90.000 | 33.586 |
| 21 | -0.649 | -0.320 | 0.935 | -0.033 | 0.000 | 51.345 | 60.290 | 183.282 | 0.000 | 0.068 | 90.000 | 34.217 |
| 22 | -0.725 | -0.352 | 1.023 | -0.054 | 0.000 | 50.942 | 59.539 | 184.365 | 0.000 | 0.064 | 90.000 | 34.542 |
| 23 | -0.795 | -0.386 | 1.106 | -0.075 | 0.000 | 50.292 | 58.875 | 185.536 | 0.000 | 0.063 | 90.000 | 34.993 |
| 24 | -0.870 | -0.426 | 1.194 | -0.102 | 0.000 | 49.644 | 58.231 | 186.808 | 0.000 | 0.063 | 90.000 | 35.457 |
| 25 | -0.944 | -0.463 | 1.278 | -0.128 | 0.000 | 49.139 | 57.762 | 187.534 | 0.000 | 0.062 | 90.000 | 35.785 |
| 26 | -1.016 | -0.503 | 1.362 | -0.158 | 0.000 | 48.485 | 57.321 | 188.357 | 0.000 | 0.063 | 90.000 | 36.197 |
| 27 | -1.127 | -0.560 | 1.487 | -0.200 | 0.000 | 48.131 | 56.812 | 189.590 | 0.000 | 0.061 | 90.000 | 36.517 |
| 28 | -1.208 | -0.604 | 1.580 | -0.232 | 0.000 | 47.617 | 56.382 | 190.109 | 0.000 | 0.062 | 90.000 | 36.826 |
| 29 | -1.288 | -0.645 | 1.668 | -0.266 | 0.000 | 47.383 | 56.043 | 190.959 | 0.000 | 0.060 | 90.000 | 37.042 |
| 30 | -1.451 | -0.722 | 1.841 | -0.332 | 0.000 | 45.746 | 55.464 | 189.634 | 0.000 | 0.068 | 90.000 | 37.684 |
| 31 | -1.500 | -0.747 | 1.893 | -0.354 | 0.000 | 45.436 | 55.418 | 189.617 | 0.000 | 0.069 | 90.000 | 37.835 |
| 32 | -1.551 | -0.773 | 1.947 | -0.377 | 0.000 | 45.274 | 55.231 | 189.936 | 0.000 | 0.069 | 90.000 | 37.954 |
| 33 | -1.624 | -0.809 | 2.024 | -0.409 | 0.000 | 45.073 | 55.098 | 190.087 | 0.000 | 0.069 | 90.000 | 38.073 |

$$
\text { Torsion Shear Test on Fine Nevada Sand with a Target Void Ratio }=0.530\left(D_{r}=91.28 \%\right)
$$

| Test No.: | $20^{*}$ |
| :--- | :---: |
| Test Date: | $4 / 22 / 10$ |
| Initial Height, $\mathrm{h}_{\mathrm{i}}$ : | 39.99 cm |
| Initial Void Ratio, e: | 0.53 |
| Final Mean Princpal Stress, $\sigma_{\mathrm{m}}:$ | 97.6 kPa |
| Max Friction Angle, $\phi:$ | 45.5 deg |
| b-value at failure: | 0.54 |
| Stress direction at failure, $\alpha:$ | 90.0 deg |


| Shear Band Notes <br> Point of Observation: <br>  <br> Inclination (from Vertical) <br> $3 @ 20^{\circ}, 2 @ 25^{\circ}, 26^{\circ}, 2 @ 28^{\circ}$ <br> Failure Notes: <br> Trough in X shape that spiraled through |
| :--- |


| Point <br> (No.) | $\begin{gathered} \varepsilon_{\mathrm{z}} \\ (\%) \\ \hline \end{gathered}$ | $\begin{gathered} \varepsilon_{\mathrm{r}} \\ (\%) \\ \hline \end{gathered}$ | $\begin{gathered} \varepsilon_{\theta} \\ (\%) \end{gathered}$ | $\begin{gathered} \varepsilon_{\mathrm{v}} \\ (\%) \\ \hline \end{gathered}$ | $\begin{array}{r} \gamma_{\theta z} \\ (\%) \\ \hline \end{array}$ | $\begin{gathered} \sigma_{z} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \sigma_{\mathrm{r}} \\ (\mathrm{kPa}) \\ \hline \end{gathered}$ | $\begin{gathered} \sigma_{\theta} \\ (\mathrm{kPa}) \\ \hline \end{gathered}$ | $\begin{gathered} \boldsymbol{\tau}_{\theta z} \\ (\mathrm{kPa}) \\ \hline \end{gathered}$ | b | $\begin{gathered} \alpha \\ \left({ }^{\circ}\right) \\ \hline \end{gathered}$ | $\begin{gathered} \varphi \\ \left({ }^{\circ}\right) \\ \hline \end{gathered}$ |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 1 | -0.003 | 0.001 | 0.001 | 0.000 | 0.000 | 92.156 | 101.367 | 101.343 | 0.000 | 0.997 | 90.000 | 2.721 |
| 2 | -0.003 | 0.001 | 0.001 | 0.000 | 0.000 | 91.956 | 101.395 | 100.822 | 0.000 | 0.939 | 90.000 | 2.636 |
| 3 | -0.008 | 0.001 | 0.009 | 0.003 | 0.000 | 85.511 | 101.413 | 107.125 | 0.000 | 0.736 | 90.000 | 6.442 |
| 4 | -0.012 | 0.002 | 0.015 | 0.005 | 0.000 | 79.771 | 101.341 | 111.957 | 0.000 | 0.670 | 90.000 | 9.664 |
| 5 | -0.017 | 0.001 | 0.022 | 0.006 | 0.000 | 75.012 | 101.356 | 116.615 | 0.000 | 0.633 | 90.000 | 12.539 |
| 6 | 0.000 | -0.010 | 0.019 | 0.009 | 0.000 | 71.154 | 101.390 | 121.397 | 0.000 | 0.602 | 90.000 | 15.125 |
| 7 | -0.003 | -0.012 | 0.027 | 0.011 | 0.000 | 67.286 | 101.347 | 124.611 | 0.000 | 0.594 | 90.000 | 17.381 |
| 8 | -0.011 | -0.010 | 0.038 | 0.017 | 0.000 | 63.941 | 101.344 | 127.707 | 0.000 | 0.587 | 90.000 | 19.434 |
| 9 | -0.019 | -0.011 | 0.050 | 0.020 | 0.000 | 61.119 | 101.400 | 131.025 | 0.000 | 0.576 | 90.000 | 21.335 |
| 10 | -0.031 | -0.011 | 0.065 | 0.023 | 0.000 | 58.248 | 101.360 | 133.552 | 0.000 | 0.573 | 90.000 | 23.117 |
| 11 | -0.045 | -0.007 | 0.079 | 0.026 | 0.000 | 55.753 | 101.368 | 135.813 | 0.000 | 0.570 | 90.000 | 24.704 |
| 12 | -0.063 | -0.003 | 0.097 | 0.031 | 0.000 | 53.621 | 101.348 | 137.970 | 0.000 | 0.566 | 90.000 | 26.120 |
| 13 | -0.084 | 0.001 | 0.117 | 0.035 | 0.000 | 51.628 | 101.381 | 140.303 | 0.000 | 0.561 | 90.000 | 27.517 |
| 14 | -0.110 | 0.007 | 0.140 | 0.037 | 0.000 | 49.371 | 101.362 | 141.962 | 0.000 | 0.562 | 90.000 | 28.942 |
| 15 | -0.139 | 0.012 | 0.165 | 0.038 | 0.000 | 47.829 | 101.324 | 143.506 | 0.000 | 0.559 | 90.000 | 30.003 |
| 16 | -0.176 | 0.019 | 0.197 | 0.040 | 0.000 | 46.430 | 101.366 | 145.286 | 0.000 | 0.556 | 90.000 | 31.040 |
| 17 | -0.215 | 0.029 | 0.226 | 0.040 | 0.000 | 45.074 | 101.369 | 146.877 | 0.000 | 0.553 | 90.000 | 32.030 |
| 18 | -0.259 | 0.039 | 0.259 | 0.039 | 0.000 | 43.786 | 101.363 | 147.861 | 0.000 | 0.553 | 90.000 | 32.892 |
| 19 | -0.307 | 0.050 | 0.292 | 0.035 | 0.000 | 42.425 | 101.354 | 149.106 | 0.000 | 0.552 | 90.000 | 33.848 |
| 20 | -0.359 | 0.060 | 0.328 | 0.030 | 0.000 | 41.245 | 101.357 | 150.065 | 0.000 | 0.552 | 90.000 | 34.668 |
| 21 | -0.412 | 0.073 | 0.363 | 0.025 | 0.000 | 40.372 | 101.371 | 151.055 | 0.000 | 0.551 | 90.000 | 35.324 |
| 22 | -0.481 | 0.089 | 0.409 | 0.018 | 0.000 | 39.458 | 101.381 | 152.314 | 0.000 | 0.549 | 90.000 | 36.050 |
| 23 | -0.605 | 0.116 | 0.486 | -0.003 | 0.000 | 37.607 | 101.401 | 154.001 | 0.000 | 0.548 | 90.000 | 37.406 |
| 24 | -0.725 | 0.141 | 0.559 | -0.026 | 0.000 | 36.699 | 101.409 | 155.260 | 0.000 | 0.546 | 90.000 | 38.144 |
| 25 | -0.886 | 0.176 | 0.648 | -0.062 | 0.000 | 35.317 | 101.361 | 156.444 | 0.000 | 0.545 | 90.000 | 39.172 |
| 26 | -1.097 | 0.221 | 0.764 | -0.112 | 0.000 | 33.771 | 101.364 | 157.844 | 0.000 | 0.545 | 90.000 | 40.354 |
| 27 | -1.263 | 0.253 | 0.852 | -0.158 | 0.000 | 32.610 | 101.379 | 158.771 | 0.000 | 0.545 | 90.000 | 41.240 |
| 28 | -1.466 | 0.292 | 0.956 | -0.218 | 0.000 | 31.784 | 101.388 | 159.963 | 0.000 | 0.543 | 90.000 | 41.950 |
| 29 | -1.778 | 0.355 | 1.114 | -0.310 | 0.000 | 30.490 | 101.360 | 161.199 | 0.000 | 0.542 | 90.000 | 42.990 |
| 30 | -2.157 | 0.419 | 1.296 | -0.442 | 0.000 | 28.889 | 101.360 | 162.352 | 0.000 | 0.543 | 90.000 | 44.257 |
| 31 | -2.807 | 0.530 | 1.595 | -0.682 | 0.000 | 27.474 | 101.396 | 163.984 | 0.000 | 0.542 | 90.000 | 45.479 |
| 32 | -2.913 | 0.545 | 1.643 | -0.726 | 0.000 | 27.810 | 101.411 | 164.242 | 0.000 | 0.539 | 90.000 | 45.267 |
| 33 | -3.027 | 0.546 | 1.697 | -0.784 | 0.000 | 27.819 | 101.494 | 164.239 | 0.000 | 0.540 | 90.000 | 45.260 |

$$
\text { Torsion Shear Test on Fine Nevada Sand with a Target Void Ratio }=0.530\left(\mathrm{D}_{\mathrm{r}}=91.28 \%\right)
$$

| Test No.: | $21^{*}$ |
| :--- | :---: |
| Test Date: | $6 / 26 / 10$ |
| Initial Height, $\mathrm{h}_{\mathrm{i}}:$ | 39.96 cm |
| Initial Void Ratio, e: | 0.52 |
| Final Mean Princpal Stress, $\sigma_{\mathrm{m}}:$ | 97.0 kPa |
| Max Friction Angle, $\phi:$ | 40.1 deg |
| b-value at failure: | 0.78 |
| Stress direction at failure, $\alpha:$ | 90.0 deg |


| Shear Band Notes |
| :--- |
| Point of Observation: |
| Inclination (from Vertical) |
| $18^{\circ}, 2 @ 24^{\circ}, 25^{\circ}, 4 @ 26^{\circ}$ |
|  |
| Failure Notes: |
| SB crossed each other (not very deep) |


| Point <br> (No.) | $\begin{gathered} \varepsilon_{\mathrm{z}} \\ (\%) \end{gathered}$ | $\begin{gathered} \varepsilon_{\mathrm{r}} \\ (\%) \end{gathered}$ | $\begin{gathered} \varepsilon_{\theta} \\ (\%) \\ \hline \end{gathered}$ | $\begin{gathered} \varepsilon_{\mathrm{v}} \\ (\%) \end{gathered}$ | $\begin{gathered} \gamma_{\theta z} \\ (\%) \end{gathered}$ | $\begin{gathered} \sigma_{z} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \sigma_{r} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \sigma_{\theta} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \tau_{\theta z} \\ (\mathrm{kPa}) \end{gathered}$ | b | $\begin{gathered} \alpha \\ \left({ }^{\circ}\right) \end{gathered}$ | $\begin{gathered} \varphi \\ { }_{\left({ }^{\circ}\right)} \\ \hline \end{gathered}$ |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 1 | 0.000 | 0.000 | 0.000 | 0.000 | 0.000 | 91.924 | 101.262 | 101.153 | 0.000 | 0.988 | 90.000 | 2.740 |
| 2 | 0.006 | -0.005 | -0.002 | -0.001 | 0.000 | 85.804 | 102.871 | 105.123 | 0.000 | 0.883 | 90.000 | 5.807 |
| 3 | 0.008 | -0.007 | 0.001 | 0.002 | 0.000 | 80.574 | 104.322 | 108.869 | 0.000 | 0.839 | 90.000 | 8.590 |
| 4 | 0.004 | -0.005 | 0.007 | 0.006 | 0.000 | 75.867 | 105.597 | 112.016 | 0.000 | 0.822 | 90.000 | 11.093 |
| 5 | -0.007 | 0.001 | 0.017 | 0.011 | 0.000 | 71.810 | 106.792 | 115.016 | 0.000 | 0.810 | 90.000 | 13.371 |
| 6 | -0.023 | 0.009 | 0.029 | 0.015 | 0.000 | 67.875 | 107.816 | 117.211 | 0.000 | 0.810 | 90.000 | 15.459 |
| 7 | -0.049 | 0.022 | 0.046 | 0.019 | 0.000 | 64.625 | 108.710 | 119.976 | 0.000 | 0.796 | 90.000 | 17.448 |
| 8 | -0.080 | 0.037 | 0.065 | 0.023 | 0.000 | 61.736 | 109.535 | 121.621 | 0.000 | 0.798 | 90.000 | 19.063 |
| 9 | -0.116 | 0.056 | 0.088 | 0.028 | 0.000 | 59.267 | 110.344 | 124.036 | 0.000 | 0.789 | 90.000 | 20.692 |
| 10 | -0.158 | 0.077 | 0.112 | 0.032 | 0.000 | 56.666 | 110.986 | 125.322 | 0.000 | 0.791 | 90.000 | 22.164 |
| 11 | -0.195 | 0.096 | 0.136 | 0.037 | 0.000 | 54.704 | 111.655 | 127.457 | 0.000 | 0.783 | 90.000 | 23.540 |
| 12 | -0.247 | 0.122 | 0.166 | 0.041 | 0.000 | 52.196 | 112.193 | 128.292 | 0.000 | 0.788 | 90.000 | 24.936 |
| 13 | -0.308 | 0.153 | 0.199 | 0.044 | 0.000 | 50.652 | 112.650 | 129.848 | 0.000 | 0.783 | 90.000 | 26.025 |
| 14 | -0.382 | 0.189 | 0.239 | 0.047 | 0.000 | 48.855 | 113.158 | 130.955 | 0.000 | 0.783 | 90.000 | 27.168 |
| 15 | -0.459 | 0.227 | 0.281 | 0.050 | 0.000 | 47.131 | 113.565 | 131.514 | 0.000 | 0.787 | 90.000 | 28.187 |
| 16 | -0.550 | 0.271 | 0.329 | 0.051 | 0.000 | 45.782 | 114.014 | 132.846 | 0.000 | 0.784 | 90.000 | 29.170 |
| 17 | -0.639 | 0.315 | 0.377 | 0.053 | 0.000 | 44.273 | 114.357 | 134.115 | 0.000 | 0.780 | 90.000 | 30.241 |
| 18 | -0.719 | 0.353 | 0.418 | 0.053 | 0.000 | 43.295 | 114.670 | 134.805 | 0.000 | 0.780 | 90.000 | 30.918 |
| 19 | -0.818 | 0.400 | 0.470 | 0.052 | 0.000 | 42.163 | 114.919 | 135.376 | 0.000 | 0.781 | 90.000 | 31.670 |
| 20 | -0.926 | 0.451 | 0.523 | 0.049 | 0.000 | 41.028 | 115.218 | 136.094 | 0.000 | 0.780 | 90.000 | 32.461 |
| 21 | -1.030 | 0.499 | 0.577 | 0.045 | 0.000 | 40.022 | 115.505 | 136.480 | 0.000 | 0.783 | 90.000 | 33.127 |
| 22 | -1.272 | 0.610 | 0.698 | 0.035 | 0.000 | 38.255 | 115.971 | 137.847 | 0.000 | 0.780 | 90.000 | 34.439 |
| 23 | -1.553 | 0.739 | 0.834 | 0.020 | 0.000 | 36.567 | 116.426 | 138.803 | 0.000 | 0.781 | 90.000 | 35.660 |
| 24 | -1.816 | 0.856 | 0.962 | 0.001 | 0.000 | 35.120 | 116.730 | 139.790 | 0.000 | 0.780 | 90.000 | 36.757 |
| 25 | -1.923 | 0.905 | 1.012 | -0.006 | 0.000 | 34.752 | 116.913 | 139.970 | 0.000 | 0.781 | 90.000 | 37.028 |
| 26 | -2.083 | 0.977 | 1.089 | -0.017 | 0.000 | 33.940 | 117.059 | 140.780 | 0.000 | 0.778 | 90.000 | 37.697 |
| 27 | -2.157 | 1.009 | 1.124 | -0.024 | 0.000 | 33.416 | 117.183 | 140.898 | 0.000 | 0.779 | 90.000 | 38.068 |
| 28 | -2.268 | 1.057 | 1.176 | -0.034 | 0.000 | 32.671 | 117.364 | 141.244 | 0.000 | 0.780 | 90.000 | 38.630 |
| 29 | -2.407 | 1.118 | 1.241 | -0.048 | 0.000 | 32.106 | 117.516 | 141.638 | 0.000 | 0.780 | 90.000 | 39.081 |
| 30 | -2.546 | 1.177 | 1.306 | -0.064 | 0.000 | 31.585 | 117.597 | 141.760 | 0.000 | 0.781 | 90.000 | 39.463 |
| 31 | -2.687 | 1.239 | 1.371 | -0.077 | 0.000 | 31.248 | 117.694 | 142.121 | 0.000 | 0.780 | 90.000 | 39.756 |
| 32 | -2.819 | 1.296 | 1.433 | -0.090 | 0.000 | 30.811 | 117.786 | 142.385 | 0.000 | 0.780 | 90.000 | 40.106 |

# Torsion Shear Test on Fine Nevada Sand with a Target Void Ratio $=\mathbf{0 . 5 3 0}\left(\mathrm{D}_{\mathrm{r}}=\mathbf{9 1 . 2 8 \%}\right)$ 

| Test No.: | $22^{*}$ |
| :--- | :---: |
| Test Date: | $6 / 29 / 10$ |
| Initial Height, $\mathrm{h}_{\mathrm{i}}:$ | 39.96 cm |
| Initial Void Ratio, e: | 0.520 |
| Final Mean Princpal Stress, $\sigma_{\mathrm{m}}:$ | 97.9 kPa |
| Max Friction Angle, $\phi:$ | 37.4 deg |
| b-value at failure: | 0.99 |
| Stress direction at failure, $\alpha:$ | 90.0 deg |


| Shear Band Notes |
| :--- |
| Point of Observation: |
|  |
| Inclination (from Vertical) |
| vertical deep SB at top cap |
| crossed SB at $20^{\circ}$ and $25^{\circ}$ |
| Failure Notes: |


| Point <br> (No.) | $\begin{gathered} \varepsilon_{\mathrm{z}} \\ (\%) \\ \hline \end{gathered}$ | $\begin{gathered} \varepsilon_{\mathrm{r}} \\ (\%) \\ \hline \end{gathered}$ | $\begin{gathered} \varepsilon_{\theta} \\ (\%) \\ \hline \end{gathered}$ | $\begin{gathered} \varepsilon_{\mathrm{v}} \\ (\%) \\ \hline \end{gathered}$ | $\begin{gathered} \gamma_{\theta z} \\ (\%) \\ \hline \end{gathered}$ | $\begin{gathered} \sigma_{z} \\ (\mathrm{kPa}) \\ \hline \end{gathered}$ | $\begin{gathered} \sigma_{r} \\ (\mathrm{kPa}) \\ \hline \end{gathered}$ | $\begin{gathered} \sigma_{\theta} \\ (\mathrm{kPa}) \\ \hline \end{gathered}$ | $\begin{gathered} \boldsymbol{\tau}_{\theta z} \\ (\mathrm{kPa}) \\ \hline \end{gathered}$ | b | $\begin{gathered} \alpha \\ \left({ }^{\circ}\right) \\ \hline \end{gathered}$ | $\begin{gathered} \varphi \\ \left({ }^{\circ}\right) \\ \hline \end{gathered}$ |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 1 | -0.003 | 0.002 | 0.002 | 0.000 | 0.000 | 92.721 | 101.068 | 101.147 | 0.000 | 0.991 | 90.000 | 2.469 |
| 2 | -0.007 | 0.004 | 0.004 | 0.001 | 0.000 | 87.188 | 103.814 | 103.959 | 0.000 | 0.991 | 90.000 | 4.994 |
| 3 | -0.012 | 0.010 | 0.007 | 0.005 | 0.000 | 81.967 | 106.230 | 105.801 | 0.000 | 0.982 | 90.000 | 7.407 |
| 4 | -0.017 | 0.017 | 0.011 | 0.010 | 0.000 | 77.650 | 108.305 | 107.991 | 0.000 | 0.990 | 90.000 | 9.489 |
| 5 | -0.024 | 0.023 | 0.016 | 0.015 | 0.000 | 73.887 | 110.181 | 110.162 | 0.000 | 0.999 | 90.000 | 11.372 |
| 6 | -0.035 | 0.032 | 0.023 | 0.020 | 0.000 | 70.343 | 111.914 | 111.948 | 0.000 | 0.999 | 90.000 | 13.185 |
| 7 | -0.048 | 0.044 | 0.033 | 0.029 | 0.000 | 67.041 | 113.397 | 113.223 | 0.000 | 0.996 | 90.000 | 14.887 |
| 8 | -0.061 | 0.056 | 0.042 | 0.037 | 0.000 | 64.370 | 114.903 | 115.059 | 0.000 | 0.997 | 90.000 | 16.372 |
| 9 | -0.078 | 0.070 | 0.052 | 0.044 | 0.000 | 61.349 | 116.119 | 116.541 | 0.000 | 0.992 | 90.000 | 17.976 |
| 10 | -0.093 | 0.081 | 0.062 | 0.049 | 0.000 | 59.104 | 117.355 | 117.448 | 0.000 | 0.998 | 90.000 | 19.275 |
| 11 | -0.111 | 0.094 | 0.073 | 0.055 | 0.000 | 56.808 | 118.461 | 118.709 | 0.000 | 0.996 | 90.000 | 20.595 |
| 12 | -0.131 | 0.108 | 0.083 | 0.061 | 0.000 | 54.612 | 119.463 | 119.519 | 0.000 | 0.999 | 90.000 | 21.873 |
| 13 | -0.150 | 0.123 | 0.095 | 0.068 | 0.000 | 52.662 | 120.434 | 120.351 | 0.000 | 0.999 | 90.000 | 23.050 |
| 14 | -0.174 | 0.140 | 0.107 | 0.073 | 0.000 | 50.919 | 121.325 | 121.252 | 0.000 | 0.999 | 90.000 | 24.127 |
| 15 | -0.202 | 0.159 | 0.122 | 0.079 | 0.000 | 49.156 | 122.119 | 122.073 | 0.000 | 0.999 | 90.000 | 25.214 |
| 16 | -0.232 | 0.179 | 0.137 | 0.083 | 0.000 | 47.417 | 122.857 | 122.245 | 0.000 | 0.992 | 90.000 | 26.299 |
| 17 | -0.264 | 0.197 | 0.153 | 0.087 | 0.000 | 46.233 | 123.581 | 123.600 | 0.000 | 1.000 | 90.000 | 27.096 |
| 18 | -0.307 | 0.224 | 0.171 | 0.089 | 0.000 | 44.756 | 124.245 | 124.059 | 0.000 | 0.998 | 90.000 | 28.057 |
| 19 | -0.346 | 0.248 | 0.189 | 0.091 | 0.000 | 43.518 | 124.908 | 124.771 | 0.000 | 0.998 | 90.000 | 28.897 |
| 20 | -0.388 | 0.272 | 0.208 | 0.092 | 0.000 | 42.208 | 125.450 | 125.456 | 0.000 | 1.000 | 90.000 | 29.769 |
| 21 | -0.436 | 0.299 | 0.230 | 0.093 | 0.000 | 41.067 | 126.029 | 125.857 | 0.000 | 0.998 | 90.000 | 30.562 |
| 22 | -0.488 | 0.329 | 0.251 | 0.091 | 0.000 | 40.036 | 126.541 | 126.608 | 0.000 | 0.999 | 90.000 | 31.286 |
| 23 | -0.544 | 0.358 | 0.274 | 0.088 | 0.000 | 39.205 | 127.011 | 127.202 | 0.000 | 0.998 | 90.000 | 31.888 |
| 24 | -0.602 | 0.389 | 0.297 | 0.085 | 0.000 | 38.105 | 127.436 | 127.621 | 0.000 | 0.998 | 90.000 | 32.658 |
| 25 | -0.676 | 0.428 | 0.325 | 0.077 | 0.000 | 37.287 | 127.927 | 128.165 | 0.000 | 0.997 | 90.000 | 33.272 |
| 26 | -0.751 | 0.467 | 0.352 | 0.068 | 0.000 | 36.236 | 128.389 | 128.533 | 0.000 | 0.998 | 90.000 | 34.040 |
| 27 | -0.816 | 0.500 | 0.375 | 0.059 | 0.000 | 35.506 | 128.712 | 128.984 | 0.000 | 0.997 | 90.000 | 34.581 |
| 28 | -0.879 | 0.530 | 0.398 | 0.049 | 0.000 | 34.819 | 129.083 | 129.272 | 0.000 | 0.998 | 90.000 | 35.108 |
| 29 | -0.965 | 0.574 | 0.426 | 0.035 | 0.000 | 34.063 | 129.467 | 129.727 | 0.000 | 0.997 | 90.000 | 35.690 |
| 30 | -1.051 | 0.618 | 0.452 | 0.019 | 0.000 | 33.370 | 129.791 | 129.889 | 0.000 | 0.999 | 90.000 | 36.225 |
| 31 | -1.247 | 0.722 | 0.502 | -0.022 | 0.000 | 32.593 | 130.189 | 130.626 | 0.000 | 0.996 | 90.000 | 36.838 |
| 32 | -1.439 | 0.829 | 0.546 | -0.064 | 0.000 | 32.071 | 130.430 | 130.981 | 0.000 | 0.994 | 90.000 | 37.249 |
| 33 | -1.565 | 0.912 | 0.556 | -0.097 | 0.000 | 31.884 | 130.568 | 131.218 | 0.000 | 0.993 | 90.000 | 37.407 |

$$
\text { Torsion Shear Test on Fine Nevada Sand with a Target Void Ratio }=0.530\left(\mathrm{D}_{\mathrm{r}}=91.28 \%\right)
$$

| Test No.: | 23 |
| :--- | ---: |
| Test Date: | $9 / 22 / 11$ |
| Initial Height, $\mathrm{h}_{\mathrm{i}}:$ | 39.94 cm |
| Initial Void Ratio, e: | 0.531 |
| Final Mean Princpal Stress, $\sigma_{\mathrm{m}}:$ | 101.4 kPa |
| Max Friction Angle, $\phi:$ | 41.4 deg |
| b-value at failure: | 0.00 |
| Stress direction at failure, $\alpha:$ | 0 deg |


| Shear Band Notes |
| :--- |
| Point of Observation: |
| Inclination (from Vertical): |
| $6 @ 70^{\circ}, 3 @ 60^{\circ}, 65^{\circ}, 73^{\circ}, 50^{\circ}, 25^{\circ}, 40^{\circ}$ |
| Failure Notes: |
| Large bulge around middle of specimen |


| Point (No.) | $\begin{gathered} \varepsilon_{\mathrm{z}} \\ (\%) \end{gathered}$ | $\begin{gathered} \varepsilon_{\mathrm{r}} \\ (\%) \end{gathered}$ | $\begin{gathered} \varepsilon_{\theta} \\ (\%) \end{gathered}$ | $\begin{gathered} \varepsilon_{\mathrm{v}} \\ (\%) \\ \hline \end{gathered}$ | $\begin{aligned} & \gamma_{\theta z} \\ & (\%) \end{aligned}$ | $\begin{gathered} \sigma_{z} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \sigma_{\mathrm{r}} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \sigma_{\theta} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \tau_{\theta z} \\ (\mathrm{kPa}) \end{gathered}$ | b | $\alpha$ <br> ${ }^{\circ}$ ) | $\varphi$ $\left(^{\circ}\right)$ |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 1 | 0.000 | 0.000 | 0.000 | 0.000 | 0.000 | 101.486 | 101.355 | 101.145 | 0.000 | 0.616 | 0.000 | 0.097 |
| 2 | 0.051 | -0.003 | -0.009 | 0.039 | 0.000 | 156.852 | 73.764 | 73.743 | 0.000 | 0.000 | 0.000 | 21.125 |
| 3 | 0.101 | -0.034 | -0.027 | 0.040 | 0.000 | 177.032 | 63.542 | 63.414 | 0.000 | 0.001 | 0.000 | 28.199 |
| 4 | 0.152 | -0.074 | -0.047 | 0.031 | 0.000 | 188.959 | 57.530 | 57.456 | 0.000 | 0.001 | 0.000 | 32.253 |
| 5 | 0.201 | -0.116 | -0.071 | 0.014 | 0.000 | 194.107 | 54.839 | 54.736 | 0.000 | 0.001 | 0.000 | 34.061 |
| 6 | 0.250 | -0.158 | -0.097 | -0.005 | 0.000 | 197.833 | 53.048 | 53.246 | 0.000 | -0.001 | 0.000 | 35.160 |
| 7 | 0.302 | -0.202 | -0.125 | -0.026 | 0.000 | 200.331 | 51.657 | 52.138 | 0.000 | -0.003 | 0.000 | 35.943 |
| 8 | 0.358 | -0.251 | -0.159 | -0.052 | 0.000 | 202.631 | 50.598 | 50.475 | 0.000 | 0.001 | 0.000 | 36.953 |
| 9 | 0.414 | -0.300 | -0.193 | -0.080 | 0.000 | 204.019 | 49.829 | 49.652 | 0.000 | 0.001 | 0.000 | 37.483 |
| 10 | 0.458 | -0.339 | -0.220 | -0.101 | 0.000 | 205.124 | 49.306 | 49.393 | 0.000 | -0.001 | 0.000 | 37.725 |
| 11 | 0.507 | -0.381 | -0.250 | -0.125 | 0.000 | 206.241 | 48.796 | 48.921 | 0.000 | -0.001 | 0.000 | 38.065 |
| 12 | 0.563 | -0.430 | -0.285 | -0.153 | 0.000 | 206.963 | 48.321 | 48.240 | 0.000 | 0.001 | 0.000 | 38.459 |
| 13 | 0.620 | -0.479 | -0.322 | -0.182 | 0.000 | 207.940 | 47.825 | 47.840 | 0.000 | 0.000 | 0.000 | 38.750 |
| 14 | 0.658 | -0.513 | -0.348 | -0.203 | 0.000 | 208.604 | 47.580 | 47.548 | 0.000 | 0.000 | 0.000 | 38.958 |
| 15 | 0.731 | -0.576 | -0.396 | -0.241 | 0.000 | 209.450 | 47.076 | 47.078 | 0.000 | 0.000 | 0.000 | 39.269 |
| 16 | 0.761 | -0.604 | -0.416 | -0.259 | 0.000 | 209.986 | 46.843 | 47.022 | 0.000 | -0.001 | 0.000 | 39.352 |
| 17 | 0.825 | -0.663 | -0.459 | -0.297 | 0.000 | 210.563 | 46.461 | 46.531 | 0.000 | 0.000 | 0.000 | 39.645 |
| 18 | 0.873 | -0.707 | -0.492 | -0.327 | 0.000 | 211.124 | 46.275 | 46.413 | 0.000 | -0.001 | 0.000 | 39.759 |
| 19 | 0.926 | -0.755 | -0.529 | -0.358 | 0.000 | 211.587 | 46.082 | 46.229 | 0.000 | -0.001 | 0.000 | 39.895 |
| 20 | 0.993 | -0.814 | -0.576 | -0.397 | 0.000 | 211.764 | 45.852 | 45.670 | 0.000 | 0.001 | 0.000 | 40.180 |
| 21 | 1.049 | -0.864 | -0.614 | -0.429 | 0.000 | 212.451 | 45.651 | 45.739 | 0.000 | -0.001 | 0.000 | 40.218 |
| 22 | 1.115 | -0.923 | -0.661 | -0.469 | 0.000 | 212.790 | 45.387 | 45.517 | 0.000 | -0.001 | 0.000 | 40.359 |
| 23 | 1.196 | -0.997 | -0.719 | -0.519 | 0.000 | 213.216 | 45.163 | 45.217 | 0.000 | 0.000 | 0.000 | 40.547 |
| 24 | 1.266 | -1.059 | -0.769 | -0.562 | 0.000 | 213.720 | 44.949 | 45.138 | 0.000 | -0.001 | 0.000 | 40.636 |
| 25 | 1.341 | -1.128 | -0.823 | -0.610 | 0.000 | 214.137 | 44.773 | 44.864 | 0.000 | -0.001 | 0.000 | 40.810 |
| 26 | 1.446 | -1.226 | -0.899 | -0.679 | 0.000 | 214.539 | 44.507 | 44.911 | 0.000 | -0.002 | 0.000 | 40.829 |
| 27 | 1.542 | -1.314 | -0.972 | -0.744 | 0.000 | 215.073 | 44.349 | 44.669 | 0.000 | -0.002 | 0.000 | 41.000 |
| 28 | 1.719 | -1.470 | -1.109 | -0.860 | 0.000 | 214.568 | 44.071 | 43.782 | 0.000 | 0.002 | 0.000 | 41.381 |
| 29 | 1.759 | -1.506 | -1.139 | -0.886 | 0.000 | 215.460 | 44.077 | 44.101 | 0.000 | 0.000 | 0.000 | 41.314 |
| 30 | 1.900 | -1.634 | -1.240 | -0.975 | 0.000 | 215.708 | 43.890 | 43.975 | 0.000 | 0.000 | 0.000 | 41.401 |

$$
\text { Torsion Shear Test on Fine Nevada Sand with a Target Void Ratio }=0.530\left(D_{r}=91.28 \%\right)
$$

| Test No.: | $24^{*}$ |
| :--- | :---: |
| Test Date: | $5 / 17 / 10$ |
| Initial Height, $\mathrm{h}_{\mathrm{i}}$ : | 39.93 cm |
| Initial Void Ratio, e: | 0.530 |
| Final Mean Princpal Stress, $\sigma_{\mathrm{m}}:$ | 97.28 kPa |
| Max Friction Angle, $\phi:$ | 46.1 deg |
| b-value at failure: | 0.27 |
| Stress direction at failure, $\alpha:$ | 0 deg |


| Shear Band Notes |
| :--- |
| Point of Observation: |
|  |
| Inclination (from Vertical): |
| $6 @ 62^{\circ}, 2 @ 63^{\circ}, 2 @ 64^{\circ}$, |
| $2 @ 65^{\circ}, 66^{\circ}, 67^{\circ}, 68^{\circ}, 70^{\circ}, 71^{\circ}$ |
| Failure Notes: |
| zig-zag paattern in middle of specimen |


| Point (No.) | $\begin{gathered} \varepsilon_{\mathrm{z}} \\ (\%) \end{gathered}$ | $\begin{gathered} \varepsilon_{\mathrm{r}} \\ (\%) \\ \hline \end{gathered}$ | $\begin{gathered} \varepsilon_{\ominus} \\ (\%) \end{gathered}$ | $\begin{gathered} \varepsilon_{\mathrm{v}} \\ (\%) \end{gathered}$ | $\begin{gathered} \gamma_{\theta z} \\ (\%) \end{gathered}$ | $\begin{gathered} \sigma_{z} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \sigma_{\mathrm{r}} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \sigma_{\theta} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \tau_{\theta z} \\ (\mathrm{kPa}) \end{gathered}$ | b | $\alpha$ <br> ${ }^{\circ}$ ) | $\begin{gathered} \varphi \\ \left({ }^{\circ}\right) \\ \hline \end{gathered}$ |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 1 | 0.000 | 0.000 | 0.000 | 0.000 | 0.000 | 89.056 | 101.298 | 101.759 | 0.000 | 0.036 | 0.000 | -3.817 |
| 2 | 0.080 | -0.021 | -0.052 | 0.008 | 0.000 | 145.957 | 85.647 | 61.874 | 0.000 | 0.283 | 0.000 | 23.864 |
| 3 | 0.121 | -0.031 | -0.083 | 0.007 | 0.000 | 155.755 | 82.947 | 54.934 | 0.000 | 0.278 | 0.000 | 28.590 |
| 4 | 0.160 | -0.040 | -0.117 | 0.003 | 0.000 | 162.345 | 81.049 | 50.381 | 0.000 | 0.274 | 0.000 | 31.758 |
| 5 | 0.201 | -0.050 | -0.154 | -0.003 | 0.000 | 167.011 | 79.701 | 46.848 | 0.000 | 0.273 | 0.000 | 34.186 |
| 6 | 0.281 | -0.067 | -0.232 | -0.019 | 0.000 | 173.203 | 77.892 | 42.716 | 0.000 | 0.270 | 0.000 | 37.181 |
| 7 | 0.321 | -0.078 | -0.274 | -0.031 | 0.000 | 175.442 | 77.290 | 41.171 | 0.000 | 0.269 | 0.000 | 38.307 |
| 8 | 0.401 | -0.095 | -0.361 | -0.056 | 0.000 | 178.360 | 76.358 | 38.799 | 0.000 | 0.269 | 0.000 | 39.991 |
| 9 | 0.480 | -0.115 | -0.453 | -0.087 | 0.000 | 180.476 | 75.760 | 36.955 | 0.000 | 0.270 | 0.000 | 41.306 |
| 10 | 0.521 | -0.124 | -0.499 | -0.103 | 0.000 | 181.572 | 75.507 | 36.427 | 0.000 | 0.269 | 0.000 | 41.744 |
| 11 | 0.602 | -0.142 | -0.597 | -0.137 | 0.000 | 183.130 | 74.974 | 35.560 | 0.000 | 0.267 | 0.000 | 42.438 |
| 12 | 0.684 | -0.162 | -0.698 | -0.176 | 0.000 | 184.242 | 74.538 | 34.413 | 0.000 | 0.268 | 0.000 | 43.254 |
| 13 | 0.720 | -0.169 | -0.743 | -0.191 | 0.000 | 184.049 | 74.511 | 33.844 | 0.000 | 0.271 | 0.000 | 43.579 |
| 14 | 0.800 | -0.183 | -0.843 | -0.225 | 0.000 | 184.191 | 74.473 | 33.488 | 0.000 | 0.272 | 0.000 | 43.814 |
| 15 | 0.925 | -0.212 | -1.008 | -0.295 | 0.000 | 186.143 | 73.898 | 32.341 | 0.000 | 0.270 | 0.000 | 44.745 |
| 16 | 0.968 | -0.222 | -1.066 | -0.319 | 0.000 | 186.328 | 73.826 | 32.097 | 0.000 | 0.271 | 0.000 | 44.919 |
| 17 | 1.006 | -0.229 | -1.115 | -0.339 | 0.000 | 186.537 | 73.723 | 31.878 | 0.000 | 0.271 | 0.000 | 45.080 |
| 18 | 1.121 | -0.250 | -1.267 | -0.396 | 0.000 | 186.379 | 73.664 | 31.434 | 0.000 | 0.273 | 0.000 | 45.346 |
| 19 | 1.161 | -0.260 | -1.320 | -0.419 | 0.000 | 186.737 | 73.523 | 31.514 | 0.000 | 0.271 | 0.000 | 45.334 |
| 20 | 1.200 | -0.268 | -1.374 | -0.441 | 0.000 | 187.166 | 73.483 | 31.485 | 0.000 | 0.270 | 0.000 | 45.399 |
| 21 | 1.285 | -0.283 | -1.489 | -0.486 | 0.000 | 187.528 | 73.310 | 31.378 | 0.000 | 0.269 | 0.000 | 45.506 |
| 22 | 1.331 | -0.295 | -1.552 | -0.515 | 0.000 | 187.586 | 73.267 | 31.174 | 0.000 | 0.269 | 0.000 | 45.643 |
| 23 | 1.365 | -0.300 | -1.601 | -0.535 | 0.000 | 187.576 | 73.189 | 31.062 | 0.000 | 0.269 | 0.000 | 45.714 |
| 24 | 1.419 | -0.306 | -1.675 | -0.562 | 0.000 | 187.546 | 73.159 | 31.211 | 0.000 | 0.268 | 0.000 | 45.615 |
| 25 | 1.475 | -0.303 | -1.781 | -0.610 | 0.000 | 186.726 | 72.909 | 30.265 | 0.000 | 0.273 | 0.000 | 46.141 |
| 26 | 1.539 | -0.310 | -1.871 | -0.641 | 0.000 | 186.828 | 73.095 | 30.441 | 0.000 | 0.273 | 0.000 | 46.036 |
| 27 | 1.578 | -0.320 | -1.923 | -0.665 | 0.000 | 186.779 | 73.093 | 30.340 | 0.000 | 0.273 | 0.000 | 46.098 |
| 28 | 1.617 | -0.325 | -1.974 | -0.682 | 0.000 | 186.862 | 73.087 | 30.629 | 0.000 | 0.272 | 0.000 | 45.918 |
| 29 | 1.657 | -0.333 | -2.026 | -0.703 | 0.000 | 186.930 | 73.088 | 30.760 | 0.000 | 0.271 | 0.000 | 45.840 |
| 30 | 1.735 | -0.347 | -2.127 | -0.739 | 0.000 | 187.183 | 73.093 | 31.303 | 0.000 | 0.268 | 0.000 | 45.517 |
| 31 | 1.775 | -0.353 | -2.178 | -0.756 | 0.000 | 187.262 | 73.071 | 31.488 | 0.000 | 0.267 | 0.000 | 45.407 |
| 32 | 1.814 | -0.362 | -2.229 | -0.776 | 0.000 | 187.141 | 73.070 | 31.288 | 0.000 | 0.268 | 0.000 | 45.522 |
| 33 | 1.882 | -0.373 | -2.315 | -0.807 | 0.000 | 187.353 | 73.094 | 31.703 | 0.000 | 0.266 | 0.000 | 45.280 |

Torsion Shear Test on Fine Nevada Sand with a Target Void Ratio $=0.530\left(\mathrm{D}_{\mathrm{r}}=\mathbf{9 1 . 2 8 \%}\right.$ )

| Test No.: | $25^{*}$ |
| :--- | :---: |
| Test Date: | $6 / 5 / 10$ |
| Initial Height, $\mathrm{h}_{\mathrm{i}}$ : | 39.99 cm |
| Initial Void Ratio, e: | 0.530 |
| Final Mean Princpal Stress, $\sigma_{\mathrm{m}}:$ | 96.69 kPa |
| Max Friction Angle, $\phi:$ | 53.2 deg |
| b-value at failure: | 0.55 |
| Stress direction at failure, $\alpha:$ | 0 deg |


| Shear Band Notes |
| :--- |
| Point of Observation: |
| Inclination (from Vertical) |
| $63^{\circ}, 3 @ 64^{\circ}, 3 @ 65^{\circ}, 66^{\circ}, 2 @ 67^{\circ}, 2 @ 68^{\circ}$, |
| $3 @ 70^{\circ}, 2 @ 80^{\circ}$ |
| Failure Notes: |
| zig-zag pattern across middle of specimen |


| Point (No.) | $\begin{gathered} \varepsilon_{\mathrm{z}} \\ (\%) \end{gathered}$ | $\begin{gathered} \varepsilon_{\mathrm{r}} \\ (\%) \\ \hline \end{gathered}$ | $\begin{gathered} \varepsilon_{\theta} \\ (\%) \\ \hline \end{gathered}$ | $\begin{gathered} \varepsilon_{\mathrm{v}} \\ (\%) \end{gathered}$ | $\begin{gathered} \gamma_{\text {өz }} \\ (\%) \end{gathered}$ | $\begin{gathered} \sigma_{z} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \sigma_{\mathrm{r}} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \sigma_{\theta} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \tau_{\text {өz }} \\ (\mathrm{kPa}) \end{gathered}$ | b | $\begin{gathered} \alpha \\ \left(^{\circ}\right) \end{gathered}$ | $\begin{gathered} \varphi \\ \left(^{\circ}\right) \\ \hline \end{gathered}$ |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 1 | 0.000 | 0.000 | 0.000 | 0.000 | 0.000 | 89.751 | 101.394 | 101.675 | 0.000 | 0.976 | 0.000 | 3.571 |
| 2 | 0.020 | -0.003 | -0.017 | 0.000 | 0.000 | 123.051 | 101.340 | 68.858 | 0.000 | 0.599 | 0.000 | 16.403 |
| 3 | 0.040 | 0.003 | -0.044 | -0.001 | 0.000 | 141.797 | 101.317 | 50.105 | 0.000 | 0.559 | 0.000 | 28.542 |
| 4 | 0.061 | 0.017 | -0.090 | -0.012 | 0.000 | 152.767 | 101.303 | 39.010 | 0.000 | 0.548 | 0.000 | 36.382 |
| 5 | 0.081 | 0.027 | -0.129 | -0.021 | 0.000 | 156.789 | 101.333 | 34.937 | 0.000 | 0.545 | 0.000 | 39.461 |
| 6 | 0.101 | 0.037 | -0.170 | -0.033 | 0.000 | 158.914 | 101.363 | 32.401 | 0.000 | 0.545 | 0.000 | 41.398 |
| 7 | 0.122 | 0.045 | -0.213 | -0.046 | 0.000 | 160.862 | 101.386 | 30.444 | 0.000 | 0.544 | 0.000 | 42.979 |
| 8 | 0.142 | 0.053 | -0.255 | -0.061 | 0.000 | 162.424 | 101.412 | 29.106 | 0.000 | 0.542 | 0.000 | 44.113 |
| 9 | 0.161 | 0.061 | -0.297 | -0.075 | 0.000 | 163.402 | 101.335 | 28.132 | 0.000 | 0.541 | 0.000 | 44.931 |
| 10 | 0.181 | 0.068 | -0.342 | -0.093 | 0.000 | 164.427 | 101.337 | 27.065 | 0.000 | 0.541 | 0.000 | 45.834 |
| 11 | 0.200 | 0.074 | -0.383 | -0.109 | 0.000 | 165.071 | 101.337 | 26.322 | 0.000 | 0.541 | 0.000 | 46.464 |
| 12 | 0.223 | 0.084 | -0.437 | -0.129 | 0.000 | 165.820 | 101.397 | 25.486 | 0.000 | 0.541 | 0.000 | 47.185 |
| 13 | 0.244 | 0.094 | -0.487 | -0.149 | 0.000 | 166.663 | 101.383 | 24.866 | 0.000 | 0.540 | 0.000 | 47.760 |
| 14 | 0.272 | 0.104 | -0.554 | -0.179 | 0.000 | 166.909 | 101.401 | 23.796 | 0.000 | 0.542 | 0.000 | 48.629 |
| 15 | 0.281 | 0.109 | -0.578 | -0.188 | 0.000 | 167.106 | 101.390 | 23.475 | 0.000 | 0.542 | 0.000 | 48.907 |
| 16 | 0.307 | 0.119 | -0.636 | -0.210 | 0.000 | 167.737 | 101.355 | 22.937 | 0.000 | 0.542 | 0.000 | 49.412 |
| 17 | 0.325 | 0.126 | -0.681 | -0.229 | 0.000 | 168.159 | 101.354 | 22.730 | 0.000 | 0.541 | 0.000 | 49.628 |
| 18 | 0.363 | 0.144 | -0.779 | -0.272 | 0.000 | 168.211 | 101.415 | 21.878 | 0.000 | 0.544 | 0.000 | 50.337 |
| 19 | 0.370 | 0.146 | -0.795 | -0.279 | 0.000 | 168.267 | 101.390 | 21.702 | 0.000 | 0.544 | 0.000 | 50.490 |
| 20 | 0.382 | 0.151 | -0.826 | -0.292 | 0.000 | 168.713 | 101.372 | 21.748 | 0.000 | 0.542 | 0.000 | 50.500 |
| 21 | 0.401 | 0.158 | -0.870 | -0.312 | 0.000 | 169.204 | 101.359 | 21.518 | 0.000 | 0.541 | 0.000 | 50.747 |
| 22 | 0.422 | 0.165 | -0.925 | -0.338 | 0.000 | 169.513 | 101.351 | 21.056 | 0.000 | 0.541 | 0.000 | 51.171 |
| 23 | 0.452 | 0.178 | -1.003 | -0.373 | 0.000 | 169.096 | 101.380 | 20.257 | 0.000 | 0.545 | 0.000 | 51.817 |
| 24 | 0.466 | 0.184 | -1.040 | -0.390 | 0.000 | 169.383 | 101.353 | 20.289 | 0.000 | 0.544 | 0.000 | 51.819 |
| 25 | 0.508 | 0.202 | -1.151 | -0.441 | 0.000 | 169.300 | 101.409 | 19.483 | 0.000 | 0.547 | 0.000 | 52.522 |
| 26 | 0.516 | 0.205 | -1.172 | -0.451 | 0.000 | 169.554 | 101.378 | 19.314 | 0.000 | 0.546 | 0.000 | 52.700 |
| 27 | 0.524 | 0.208 | -1.192 | -0.461 | 0.000 | 170.011 | 101.380 | 19.780 | 0.000 | 0.543 | 0.000 | 52.332 |
| 28 | 0.551 | 0.220 | -1.266 | -0.495 | 0.000 | 169.794 | 101.390 | 19.245 | 0.000 | 0.546 | 0.000 | 52.787 |
| 29 | 0.567 | 0.226 | -1.307 | -0.514 | 0.000 | 169.843 | 101.373 | 19.158 | 0.000 | 0.546 | 0.000 | 52.870 |
| 30 | 0.583 | 0.231 | -1.347 | -0.533 | 0.000 | 169.984 | 101.351 | 19.276 | 0.000 | 0.545 | 0.000 | 52.779 |
| 31 | 0.606 | 0.239 | -1.405 | -0.560 | 0.000 | 169.835 | 101.335 | 18.833 | 0.000 | 0.546 | 0.000 | 53.164 |
| 32 | 0.629 | 0.250 | -1.470 | -0.591 | 0.000 | 169.940 | 101.375 | 18.759 | 0.000 | 0.546 | 0.000 | 53.242 |

# Torsion Shear Test on Fine Nevada Sand with a Target Void Ratio $=\mathbf{0 . 5 3 0}\left(\mathrm{D}_{\mathrm{r}}=\mathbf{9 1 . 2 8 \%}\right)$ 

| Test No.: | 26 |
| :--- | :---: |
| Test Date: | $10 / 19 / 11$ |
| Initial Height, $\mathrm{h}_{\mathrm{i}}$ : | 39.99 cm |
| Initial Void Ratio, e: | 0.532 |
| Final Mean Princpal Stress, $\sigma_{\mathrm{m}}:$ | 101.8 kPa |
| Max Friction Angle, $\phi:$ | 53.1 deg |
| b-value at failure: | 1.00 |
| Stress direction at failure, $\alpha:$ | 0 deg |


| Shear Band Notes | 25 |
| :--- | :--- |
| Point of Observation: |  |
| Inclination (from Vertical) |  |
| $90^{\circ}$ |  |
|  |  |
| Failure Notes: |  |
| deep vertical trough |  |


| Point <br> $($ No. $)$ | $\varepsilon_{z}$ <br> $(\%)$ | $\varepsilon_{\mathrm{r}}$ <br> $(\%)$ | $\varepsilon_{\theta}$ <br> $(\%)$ | $\varepsilon_{\mathrm{v}}$ <br> $(\%)$ | $\gamma_{\theta z}$ <br> $(\%)$ | $\sigma_{z}$ <br> $(\mathrm{kPa})$ | $\sigma_{\mathrm{r}}$ <br> $(\mathrm{kPa})$ | $\sigma_{\theta}$ <br> $(\mathrm{kPa})$ | $\tau_{\theta z}$ <br> $(\mathrm{kPa})$ | b | $\alpha$ <br> $\left({ }^{\circ}\right)$ | $\varphi$ <br> $\left({ }^{\circ}\right)$ |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 1 | 0.000 | 0.000 | 0.000 | 0.000 | 0.000 | 101.522 | 101.416 | 101.420 | 0.000 | -0.040 | 0.000 | 0.029 |
| 2 | 0.010 | 0.009 | -0.012 | 0.007 | 0.000 | 112.146 | 111.676 | 80.616 | 0.000 | 0.985 | 0.000 | 9.414 |
| 3 | 0.020 | 0.028 | -0.037 | 0.011 | 0.000 | 124.980 | 124.531 | 55.338 | 0.000 | 0.994 | 0.000 | 22.719 |
| 4 | 0.030 | 0.039 | -0.061 | 0.008 | 0.000 | 130.227 | 130.001 | 44.592 | 0.000 | 0.997 | 0.000 | 29.331 |
| 5 | 0.040 | 0.053 | -0.090 | 0.003 | 0.000 | 133.686 | 133.310 | 37.748 | 0.000 | 0.996 | 0.000 | 34.029 |
| 6 | 0.051 | 0.070 | -0.123 | -0.002 | 0.000 | 136.156 | 135.757 | 33.140 | 0.000 | 0.996 | 0.000 | 37.481 |
| 7 | 0.061 | 0.086 | -0.156 | -0.009 | 0.000 | 138.020 | 137.343 | 30.142 | 0.000 | 0.994 | 0.000 | 39.904 |
| 8 | 0.071 | 0.102 | -0.190 | -0.017 | 0.000 | 138.900 | 138.342 | 28.033 | 0.000 | 0.995 | 0.000 | 41.617 |
| 9 | 0.081 | 0.119 | -0.225 | -0.026 | 0.000 | 139.927 | 139.193 | 26.384 | 0.000 | 0.994 | 0.000 | 43.056 |
| 10 | 0.091 | 0.136 | -0.264 | -0.037 | 0.000 | 140.231 | 139.800 | 25.017 | 0.000 | 0.996 | 0.000 | 44.205 |
| 11 | 0.102 | 0.153 | -0.304 | -0.049 | 0.000 | 141.165 | 140.575 | 23.739 | 0.000 | 0.995 | 0.000 | 45.405 |
| 12 | 0.110 | 0.170 | -0.342 | -0.061 | 0.000 | 141.389 | 141.036 | 22.675 | 0.000 | 0.997 | 0.000 | 46.351 |
| 13 | 0.121 | 0.189 | -0.384 | -0.075 | 0.000 | 142.034 | 141.423 | 21.896 | 0.000 | 0.995 | 0.000 | 47.126 |
| 14 | 0.133 | 0.209 | -0.432 | -0.090 | 0.000 | 142.398 | 141.857 | 20.927 | 0.000 | 0.996 | 0.000 | 48.051 |
| 15 | 0.142 | 0.224 | -0.467 | -0.101 | 0.000 | 142.978 | 142.144 | 20.421 | 0.000 | 0.993 | 0.000 | 48.595 |
| 16 | 0.152 | 0.242 | -0.510 | -0.116 | 0.000 | 142.771 | 142.428 | 19.701 | 0.000 | 0.997 | 0.000 | 49.243 |
| 17 | 0.160 | 0.257 | -0.544 | -0.127 | 0.000 | 142.990 | 142.553 | 19.362 | 0.000 | 0.996 | 0.000 | 49.594 |
| 18 | 0.170 | 0.272 | -0.583 | -0.140 | 0.000 | 143.509 | 142.882 | 19.032 | 0.000 | 0.995 | 0.000 | 49.980 |
| 19 | 0.183 | 0.293 | -0.633 | -0.158 | 0.000 | 144.002 | 143.338 | 18.414 | 0.000 | 0.995 | 0.000 | 50.647 |
| 20 | 0.193 | 0.311 | -0.679 | -0.175 | 0.000 | 144.273 | 143.575 | 17.887 | 0.000 | 0.994 | 0.000 | 51.205 |
| 21 | 0.201 | 0.325 | -0.714 | -0.188 | 0.000 | 144.425 | 143.736 | 17.616 | 0.000 | 0.995 | 0.000 | 51.497 |
| 22 | 0.216 | 0.352 | -0.782 | -0.214 | 0.000 | 144.590 | 143.958 | 17.074 | 0.000 | 0.995 | 0.000 | 52.071 |
| 23 | 0.227 | 0.371 | -0.829 | -0.231 | 0.000 | 144.694 | 144.088 | 16.740 | 0.000 | 0.995 | 0.000 | 52.430 |
| 24 | 0.235 | 0.387 | -0.870 | -0.247 | 0.000 | 144.860 | 144.208 | 16.513 | 0.000 | 0.995 | 0.000 | 52.688 |
| 25 | 0.246 | 0.408 | -0.920 | -0.265 | 0.000 | 144.968 | 144.360 | 16.139 | 0.000 | 0.995 | 0.000 | 53.097 |

## Torsion Shear Test on Fine Nevada Sand with a Target Void Ratio $=\mathbf{0 . 5 3 0}\left(\mathrm{D}_{\mathrm{r}}=\mathbf{9 1 . 2 8 \%}\right)$

| Test No.: | $27^{*}$ |
| :--- | :---: |
| Test Date: | $4 / 15 / 11$ |
| Initial Height, $\mathrm{h}_{\mathrm{i}}$ : | 39.96 cm |
| Initial Void Ratio, e: | 0.510 |
| Final Mean Princpal Stress, $\sigma_{\mathrm{m}}:$ | 97.9 kPa |
| Max Friction Angle, $\phi:$ | 37.6 deg |
| b-value at failure: | 0.02 |
| Stress direction at failure, $\alpha:$ | 24.0 deg |


| Shear Band Notes <br> Point of Observation: | 26 |
| :--- | :--- |
| Inclination (from Vertical) |  |
| $30^{\circ}$ |  |
| Failure Notes: |  |


| Point (No.) | $\begin{gathered} \varepsilon_{\mathrm{z}} \\ (\%) \end{gathered}$ | $\begin{gathered} \varepsilon_{\mathrm{r}} \\ (\%) \end{gathered}$ | $\begin{gathered} \varepsilon_{\theta} \\ (\%) \end{gathered}$ | $\begin{gathered} \varepsilon_{\mathrm{v}} \\ (\%) \end{gathered}$ | $\begin{aligned} & \gamma_{\theta z} \\ & (\%) \end{aligned}$ | $\begin{gathered} \sigma_{z} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \sigma_{\mathrm{r}} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \sigma_{\theta} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \tau_{\theta z} \\ (\mathrm{kPa}) \end{gathered}$ | b | $\begin{gathered} \alpha \\ \left({ }^{\circ}\right) \end{gathered}$ | $\begin{gathered} \varphi \\ \left({ }^{\circ}\right) \end{gathered}$ |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 1 | 0.000 | 0.000 | 0.000 | 0.000 | 0.000 | 88.632 | 101.375 | 101.240 | -0.633 | -0.008 | 2.865 | -3.826 |
| 2 | 0.034 | -0.026 | -0.014 | -0.006 | 0.056 | 113.698 | 86.174 | 92.744 | 20.770 | 0.134 | 31.616 | 13.024 |
| 3 | 0.039 | -0.032 | -0.014 | -0.007 | 0.086 | 121.595 | 81.081 | 89.704 | 23.440 | 0.067 | 27.887 | 15.565 |
| 4 | 0.047 | -0.040 | -0.017 | -0.010 | 0.142 | 130.701 | 75.340 | 86.449 | 28.145 | 0.036 | 25.914 | 19.253 |
| 5 | 0.052 | -0.044 | -0.020 | -0.011 | 0.172 | 134.456 | 73.142 | 85.489 | 30.574 | 0.030 | 25.656 | 20.865 |
| 6 | 0.060 | -0.050 | -0.023 | -0.013 | 0.226 | 136.727 | 71.772 | 84.810 | 34.619 | 0.049 | 26.568 | 22.994 |
| 7 | 0.071 | -0.061 | -0.027 | -0.017 | 0.254 | 142.877 | 68.065 | 82.311 | 36.211 | 0.028 | 25.047 | 24.787 |
| 8 | 0.080 | -0.067 | -0.032 | -0.020 | 0.285 | 145.356 | 66.556 | 81.676 | 37.747 | 0.025 | 24.926 | 25.787 |
| 9 | 0.097 | -0.084 | -0.039 | -0.026 | 0.340 | 149.315 | 64.081 | 80.335 | 39.935 | 0.019 | 24.592 | 27.357 |
| 10 | 0.130 | -0.111 | -0.057 | -0.038 | 0.413 | 153.089 | 61.720 | 78.984 | 42.096 | 0.016 | 24.323 | 28.901 |
| 11 | 0.163 | -0.143 | -0.076 | -0.056 | 0.510 | 156.940 | 59.345 | 77.570 | 44.047 | 0.012 | 23.991 | 30.373 |
| 12 | 0.186 | -0.166 | -0.091 | -0.071 | 0.579 | 158.676 | 58.273 | 77.144 | 45.416 | 0.011 | 24.044 | 31.170 |
| 13 | 0.239 | -0.216 | -0.123 | -0.100 | 0.717 | 161.762 | 56.182 | 75.704 | 45.900 | 0.003 | 23.425 | 31.998 |
| 14 | 0.326 | -0.301 | -0.185 | -0.160 | 0.962 | 163.799 | 55.051 | 75.179 | 49.446 | 0.015 | 24.068 | 33.756 |
| 15 | 0.356 | -0.330 | -0.205 | -0.178 | 1.019 | 165.445 | 54.153 | 74.600 | 49.485 | 0.010 | 23.725 | 34.032 |
| 16 | 0.469 | -0.436 | -0.282 | -0.249 | 1.287 | 165.326 | 54.041 | 74.681 | 49.257 | 0.007 | 23.691 | 33.902 |
| 17 | 0.506 | -0.483 | -0.320 | -0.297 | 1.449 | 166.037 | 53.674 | 74.609 | 53.047 | 0.024 | 24.623 | 35.591 |
| 18 | 0.556 | -0.539 | -0.354 | -0.338 | 1.550 | 170.759 | 50.710 | 72.602 | 50.851 | -0.002 | 23.008 | 35.507 |
| 19 | 0.593 | -0.569 | -0.385 | -0.360 | 1.643 | 167.633 | 52.691 | 73.824 | 53.106 | 0.020 | 24.274 | 35.936 |
| 20 | 0.672 | -0.658 | -0.443 | -0.428 | 1.813 | 170.733 | 50.596 | 72.990 | 52.862 | 0.005 | 23.623 | 36.211 |
| 21 | 0.733 | -0.726 | -0.494 | -0.487 | 1.973 | 171.464 | 50.455 | 72.582 | 53.433 | 0.008 | 23.611 | 36.626 |
| 22 | 0.838 | -0.832 | -0.575 | -0.569 | 2.193 | 172.086 | 49.995 | 72.520 | 54.032 | 0.008 | 23.672 | 36.922 |
| 23 | 0.904 | -0.906 | -0.629 | -0.631 | 2.352 | 173.098 | 49.632 | 72.307 | 54.254 | 0.007 | 23.556 | 37.120 |
| 24 | 1.000 | -0.997 | -0.707 | -0.703 | 2.583 | 171.894 | 50.241 | 72.638 | 55.029 | 0.014 | 23.977 | 37.306 |
| 25 | 1.162 | -1.159 | -0.814 | -0.811 | 2.853 | 173.304 | 48.794 | 71.477 | 53.313 | 0.001 | 23.159 | 37.037 |
| 26 | 1.187 | -1.182 | -0.842 | -0.837 | 2.964 | 171.130 | 50.326 | 72.098 | 55.154 | 0.019 | 24.042 | 37.551 |
| 27 | 1.256 | -1.253 | -0.887 | -0.883 | 3.101 | 172.412 | 49.797 | 73.131 | 55.130 | 0.008 | 24.000 | 37.176 |
| 28 | 1.347 | -1.338 | -0.954 | -0.945 | 3.302 | 171.389 | 50.340 | 72.525 | 55.267 | 0.017 | 24.095 | 37.444 |
| 29 | 1.440 | -1.438 | -1.011 | -1.009 | 3.456 | 174.422 | 48.636 | 71.881 | 53.265 | -0.004 | 23.047 | 36.893 |
| 30 | 1.503 | -1.494 | -1.060 | -1.051 | 3.614 | 171.072 | 50.402 | 72.281 | 55.038 | 0.018 | 24.046 | 37.430 |
| 31 | 1.635 | -1.632 | -1.134 | -1.130 | 3.888 | 171.692 | 50.316 | 72.950 | 54.927 | 0.013 | 24.025 | 37.141 |

$$
\text { Torsion Shear Test on Fine Nevada Sand with a Target Void Ratio }=0.530\left(D_{r}=91.28 \%\right)
$$

| Test No.: | 28 |
| :--- | :---: |
| Test Date: | $8 / 29 / 11$ |
| Initial Height, $\mathrm{h}_{\mathrm{i}}$ : | 40.06 cm |
| Initial Void Ratio, e: | 0.531 |
| Final Mean Princpal Stress, $\sigma_{\mathrm{m}}:$ | 103.5 kPa |
| Max Friction Angle, $\phi:$ | 43.0 deg |
| b-value at failure: | 0.23 |
| Stress direction at failure, $\alpha$ : | 23.7 deg |


| Shear Band Notes <br> Point of Observation: | 32 |
| :--- | :--- |
| Inclination (from Vertical) |  |
| First SB: $43^{\circ}, 30^{\circ}, 41^{\circ}$ |  |
| Second SB: $32^{\circ}, 43^{\circ}$ |  |
| Failure Notes: |  |
| two parallel shear bands |  |


| Point (No.) | $\begin{gathered} \varepsilon_{z} \\ (\%) \end{gathered}$ | $\begin{gathered} \varepsilon_{\mathrm{r}} \\ (\%) \end{gathered}$ | $\begin{gathered} \varepsilon_{\theta} \\ (\%) \end{gathered}$ | $\begin{gathered} \varepsilon_{\mathrm{v}} \\ (\%) \end{gathered}$ | $\begin{gathered} \gamma_{\theta z} \\ (\%) \end{gathered}$ | $\begin{gathered} \sigma_{z} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \sigma_{\mathrm{r}} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \sigma_{\theta} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \tau_{\theta z} \\ (\mathrm{kPa}) \end{gathered}$ | b | $\alpha$ <br> ${ }^{\circ}$ ) | $\begin{gathered} \varphi \\ \left({ }^{\circ}\right) \end{gathered}$ |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 1 | 0.000 | 0.000 | 0.000 | 0.000 | 0.000 | 103.082 | 100.437 | 100.856 | 1.847 | 0.145 | 29.464 | 1.212 |
| 2 | 0.010 | 0.000 | -0.004 | 0.006 | 0.000 | 109.568 | 97.840 | 95.236 | 10.753 | 0.323 | 28.160 | 7.249 |
| 3 | 0.016 | -0.002 | -0.006 | 0.008 | 0.000 | 113.585 | 96.329 | 92.509 | 13.666 | 0.305 | 26.181 | 9.641 |
| 4 | 0.026 | -0.008 | -0.011 | 0.007 | 0.000 | 117.726 | 94.902 | 91.161 | 14.637 | 0.259 | 23.889 | 10.909 |
| 5 | 0.024 | -0.005 | -0.009 | 0.011 | 0.023 | 123.685 | 92.576 | 87.842 | 19.395 | 0.250 | 23.631 | 14.459 |
| 6 | 0.029 | -0.007 | -0.012 | 0.011 | 0.032 | 124.670 | 92.227 | 87.051 | 20.238 | 0.253 | 23.548 | 15.129 |
| 7 | 0.030 | -0.005 | -0.013 | 0.012 | 0.047 | 126.216 | 91.797 | 85.890 | 21.370 | 0.257 | 23.332 | 16.084 |
| 8 | 0.025 | -0.004 | -0.008 | 0.013 | 0.062 | 127.468 | 91.210 | 85.487 | 22.805 | 0.254 | 23.686 | 16.923 |
| 9 | 0.025 | -0.002 | -0.008 | 0.014 | 0.081 | 129.292 | 90.583 | 85.008 | 24.202 | 0.247 | 23.773 | 17.826 |
| 10 | 0.033 | -0.005 | -0.013 | 0.016 | 0.099 | 130.539 | 89.908 | 83.142 | 25.479 | 0.257 | 23.537 | 19.007 |
| 11 | 0.034 | -0.005 | -0.013 | 0.017 | 0.101 | 132.503 | 89.328 | 82.962 | 25.054 | 0.239 | 22.663 | 19.089 |
| 12 | 0.036 | -0.004 | -0.015 | 0.017 | 0.113 | 131.038 | 89.592 | 81.486 | 26.481 | 0.270 | 23.453 | 19.955 |
| 13 | 0.036 | -0.004 | -0.014 | 0.018 | 0.145 | 133.883 | 88.743 | 82.506 | 28.035 | 0.244 | 23.751 | 20.576 |
| 14 | 0.038 | -0.004 | -0.015 | 0.019 | 0.179 | 135.598 | 88.030 | 81.001 | 29.320 | 0.247 | 23.522 | 21.710 |
| 15 | 0.039 | -0.003 | -0.016 | 0.019 | 0.213 | 137.051 | 87.443 | 79.515 | 30.383 | 0.251 | 23.282 | 22.732 |
| 16 | 0.040 | -0.004 | -0.016 | 0.020 | 0.224 | 138.191 | 86.993 | 78.428 | 30.068 | 0.249 | 22.589 | 23.041 |
| 17 | 0.045 | 0.039 | -0.013 | 0.071 | 0.244 | 138.353 | 86.897 | 78.843 | 31.504 | 0.250 | 23.318 | 23.518 |
| 18 | 0.051 | 0.038 | -0.016 | 0.073 | 0.279 | 140.973 | 86.060 | 77.119 | 33.404 | 0.251 | 23.148 | 25.071 |
| 19 | 0.056 | 0.039 | -0.020 | 0.075 | 0.318 | 143.229 | 85.065 | 76.436 | 34.140 | 0.241 | 22.815 | 25.774 |
| 20 | 0.057 | 0.041 | -0.023 | 0.075 | 0.360 | 143.955 | 84.677 | 74.728 | 36.456 | 0.255 | 23.242 | 27.371 |
| 21 | 0.065 | 0.042 | -0.030 | 0.077 | 0.404 | 148.327 | 83.176 | 73.157 | 37.919 | 0.242 | 22.627 | 28.824 |
| 22 | 0.069 | 0.044 | -0.038 | 0.075 | 0.449 | 150.669 | 82.889 | 70.584 | 39.597 | 0.254 | 22.340 | 30.600 |
| 23 | 0.083 | 0.040 | -0.050 | 0.073 | 0.510 | 151.781 | 81.971 | 71.315 | 42.416 | 0.247 | 23.257 | 31.608 |
| 24 | 0.089 | 0.042 | -0.061 | 0.070 | 0.551 | 152.857 | 81.537 | 69.578 | 43.322 | 0.253 | 23.067 | 32.703 |
| 25 | 0.112 | 0.039 | -0.090 | 0.061 | 0.660 | 155.334 | 80.491 | 68.555 | 45.525 | 0.250 | 23.188 | 34.180 |
| 26 | 0.141 | 0.037 | -0.135 | 0.043 | 0.800 | 157.928 | 79.480 | 66.409 | 47.583 | 0.252 | 23.059 | 36.053 |
| 27 | 0.161 | 0.033 | -0.168 | 0.026 | 0.911 | 159.231 | 78.945 | 65.822 | 49.443 | 0.253 | 23.316 | 37.187 |
| 28 | 0.198 | 0.029 | -0.237 | -0.010 | 1.087 | 162.402 | 78.159 | 64.835 | 51.371 | 0.250 | 23.240 | 38.574 |
| 29 | 0.234 | 0.029 | -0.307 | -0.043 | 1.253 | 163.164 | 77.588 | 62.942 | 52.016 | 0.254 | 23.034 | 39.708 |
| 30 | 0.311 | 0.014 | -0.448 | -0.122 | 1.569 | 167.634 | 76.665 | 63.035 | 54.594 | 0.244 | 23.115 | 40.958 |
| 31 | 0.414 | 0.001 | -0.652 | -0.238 | 1.988 | 167.640 | 75.761 | 60.118 | 55.360 | 0.253 | 22.920 | 42.659 |
| 32 | 0.515 | -0.023 | -0.889 | -0.397 | 2.456 | 172.144 | 74.962 | 63.352 | 59.095 | 0.234 | 23.686 | 43.010 |

Torsion Shear Test on Fine Nevada Sand with a Target Void Ratio $=\mathbf{0 . 5 3 0}\left(\mathrm{D}_{\mathrm{r}}=\mathbf{9 1 . 2 8 \%}\right)$

| Test No.: | 29 |
| :--- | :---: |
| Test Date: | $9 / 1 / 11$ |
| Initial Height, $\mathrm{h}_{\mathrm{i}}:$ | 39.96 cm |
| Initial Void Ratio, e: | 0.531 |
| Final Mean Princpal Stress, $\sigma_{\mathrm{m}}:$ | 101.4 kPa |
| Max Friction Angle, $\phi:$ | 46.5 deg |
| b-value at failure: | 0.75 |
| Stress direction at failure, $\alpha:$ | 22.2 deg |


| Shear Band Notes |
| :--- |
| Point of Observation: |
| Inclination (from Vertical) |
| one SB at $20^{\circ}, 48^{\circ}, 60^{\circ}, 76^{\circ}$ |
| Failure Notes: |
| thick SB (two SB's crossing each other) |


| Point (No.) | $\begin{gathered} \varepsilon_{\mathrm{z}} \\ (\%) \end{gathered}$ | $\begin{gathered} \varepsilon_{\mathrm{r}} \\ (\%) \end{gathered}$ | $\begin{gathered} \varepsilon_{\theta} \\ (\%) \\ \hline \end{gathered}$ | $\begin{gathered} \varepsilon_{\mathrm{v}} \\ (\%) \end{gathered}$ | $\begin{aligned} & \gamma_{\theta z} \\ & (\%) \end{aligned}$ | $\begin{gathered} \sigma_{z} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \sigma_{\mathrm{r}} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \sigma_{\theta} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \boldsymbol{\tau}_{\theta z} \\ (\mathrm{kPa}) \end{gathered}$ | b | $\begin{gathered} \alpha \\ \left({ }^{\circ}\right) \\ \hline \end{gathered}$ | $\begin{gathered} \varphi \\ \left({ }^{\circ}\right) \end{gathered}$ |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 1 | 0.000 | 0.000 | 0.000 | 0.000 | 0.000 | 100.314 | 101.302 | 101.092 | 0.261 | 1.139 | -16.924 | 0.267 |
| 2 | 0.017 | 0.010 | -0.014 | 0.014 | 0.000 | 111.843 | 108.028 | 84.187 | 13.965 | 0.755 | 22.642 | 11.567 |
| 3 | 0.020 | 0.015 | -0.017 | 0.017 | 0.000 | 113.970 | 109.536 | 80.317 | 17.846 | 0.753 | 23.341 | 14.625 |
| 4 | 0.025 | 0.022 | -0.026 | 0.022 | 0.000 | 117.612 | 111.660 | 74.627 | 22.539 | 0.749 | 23.181 | 18.906 |
| 5 | 0.028 | 0.026 | -0.029 | 0.025 | 0.000 | 118.440 | 112.371 | 72.791 | 25.191 | 0.746 | 23.911 | 20.826 |
| 6 | 0.030 | 0.032 | -0.036 | 0.026 | 0.036 | 120.582 | 113.625 | 69.657 | 26.797 | 0.750 | 23.231 | 22.869 |
| 7 | 0.032 | 0.037 | -0.041 | 0.028 | 0.064 | 121.768 | 114.401 | 67.657 | 28.059 | 0.753 | 23.022 | 24.302 |
| 8 | 0.033 | 0.042 | -0.047 | 0.029 | 0.092 | 122.770 | 114.980 | 66.099 | 29.441 | 0.751 | 23.048 | 25.639 |
| 9 | 0.042 | 0.045 | -0.057 | 0.030 | 0.121 | 123.966 | 115.683 | 64.428 | 30.933 | 0.750 | 23.049 | 27.114 |
| 10 | 0.039 | 0.056 | -0.064 | 0.031 | 0.157 | 124.871 | 116.348 | 62.368 | 32.370 | 0.753 | 23.003 | 28.725 |
| 11 | 0.042 | 0.065 | -0.077 | 0.031 | 0.192 | 126.112 | 117.069 | 60.719 | 33.797 | 0.752 | 22.974 | 30.225 |
| 12 | 0.046 | 0.074 | -0.088 | 0.031 | 0.222 | 127.181 | 117.683 | 59.097 | 34.760 | 0.752 | 22.799 | 31.491 |
| 13 | 0.057 | 0.081 | -0.109 | 0.028 | 0.266 | 127.940 | 118.143 | 57.845 | 36.233 | 0.750 | 22.976 | 32.866 |
| 14 | 0.062 | 0.098 | -0.136 | 0.024 | 0.325 | 128.994 | 118.872 | 55.812 | 37.449 | 0.753 | 22.832 | 34.515 |
| 15 | 0.065 | 0.115 | -0.162 | 0.018 | 0.384 | 129.979 | 119.414 | 54.456 | 38.553 | 0.752 | 22.797 | 35.817 |
| 16 | 0.078 | 0.127 | -0.194 | 0.011 | 0.439 | 130.809 | 119.946 | 53.160 | 39.221 | 0.753 | 22.646 | 36.867 |
| 17 | 0.080 | 0.137 | -0.211 | 0.006 | 0.474 | 131.587 | 120.253 | 52.513 | 39.693 | 0.752 | 22.556 | 37.490 |
| 18 | 0.083 | 0.147 | -0.231 | 0.000 | 0.509 | 131.881 | 120.485 | 51.933 | 40.132 | 0.752 | 22.556 | 38.048 |
| 19 | 0.094 | 0.154 | -0.255 | -0.007 | 0.549 | 132.158 | 120.692 | 51.291 | 40.568 | 0.753 | 22.547 | 38.641 |
| 20 | 0.101 | 0.163 | -0.279 | -0.015 | 0.589 | 132.450 | 120.895 | 50.756 | 40.991 | 0.753 | 22.550 | 39.178 |
| 21 | 0.104 | 0.175 | -0.303 | -0.024 | 0.632 | 132.623 | 121.104 | 50.029 | 41.320 | 0.755 | 22.508 | 39.768 |
| 22 | 0.108 | 0.195 | -0.343 | -0.040 | 0.705 | 133.316 | 121.426 | 49.521 | 41.997 | 0.753 | 22.534 | 40.460 |
| 23 | 0.113 | 0.216 | -0.386 | -0.057 | 0.776 | 133.431 | 121.673 | 48.493 | 42.327 | 0.756 | 22.452 | 41.237 |
| 24 | 0.133 | 0.231 | -0.441 | -0.077 | 0.862 | 134.037 | 121.929 | 48.014 | 43.036 | 0.754 | 22.508 | 41.947 |
| 25 | 0.137 | 0.249 | -0.478 | -0.092 | 0.926 | 134.451 | 122.149 | 47.531 | 43.441 | 0.754 | 22.494 | 42.479 |
| 26 | 0.144 | 0.272 | -0.528 | -0.113 | 1.006 | 134.747 | 122.411 | 46.847 | 43.751 | 0.755 | 22.435 | 43.078 |
| 27 | 0.165 | 0.300 | -0.612 | -0.146 | 1.135 | 135.132 | 122.598 | 46.335 | 44.581 | 0.753 | 22.559 | 43.903 |
| 28 | 0.169 | 0.313 | -0.640 | -0.159 | 1.183 | 135.566 | 122.800 | 46.103 | 44.871 | 0.752 | 22.545 | 44.228 |
| 29 | 0.183 | 0.338 | -0.707 | -0.187 | 1.282 | 135.815 | 123.046 | 45.253 | 45.097 | 0.754 | 22.441 | 44.901 |
| 30 | 0.193 | 0.363 | -0.771 | -0.215 | 1.381 | 136.143 | 123.184 | 44.934 | 45.599 | 0.753 | 22.498 | 45.423 |
| 31 | 0.198 | 0.388 | -0.829 | -0.243 | 1.470 | 136.350 | 123.387 | 44.394 | 45.831 | 0.754 | 22.454 | 45.919 |
| 32 | 0.295 | 0.517 | -1.242 | -0.430 | 1.953 | 137.015 | 123.529 | 43.523 | 45.807 | 0.754 | 22.209 | 46.472 |

$$
\text { Torsion Shear Test on Fine Nevada Sand with a Target Void Ratio }=0.530\left(D_{r}=91.28 \%\right)
$$

| Test No.: | $30^{*}$ |
| :--- | :---: |
| Test Date: | $5 / 6 / 11$ |
| Initial Height, $\mathrm{h}_{\mathrm{i}}:$ | 39.96 cm |
| Initial Void Ratio, e: | 0.529 |
| Final Mean Princpal Stress, $\sigma_{\mathrm{m}}:$ | 97.9 kPa |
| Max Friction Angle, $\phi:$ | 46.4 deg |
| b-value at failure: | 0.85 |
| Stress direction at failure, $\alpha:$ | 22.9 deg |


| Shear Band Notes <br> Point of Observation: <br>  <br> Inclination (from Vertical) <br> big trough at $62^{\circ}, 40^{\circ}$ and $62^{\circ}$ <br> other SBs @ $40^{\circ}, 49^{\circ}, 2 @ 62^{\circ}, 70^{\circ}$ <br> Failure Notes: <br> trough changed slope in middle of specimen |
| :--- |


| Point (No.) | $\begin{gathered} \varepsilon_{\mathrm{z}} \\ (\%) \end{gathered}$ | $\begin{gathered} \varepsilon_{\mathrm{r}} \\ (\%) \end{gathered}$ | $\begin{gathered} \varepsilon_{\theta} \\ (\%) \end{gathered}$ | $\begin{gathered} \varepsilon_{\mathrm{v}} \\ (\%) \end{gathered}$ | $\begin{gathered} \gamma_{\theta z} \\ (\%) \end{gathered}$ | $\begin{gathered} \sigma_{z} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \sigma_{\mathrm{r}} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \sigma_{\theta} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \tau_{\theta z} \\ (\mathrm{kPa}) \end{gathered}$ | b | $\begin{gathered} \alpha \\ \left({ }^{( }\right) \\ \hline \end{gathered}$ | $\begin{gathered} \varphi \\ \left({ }^{\circ}\right) \end{gathered}$ |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 1 | 0.000 | 0.000 | 0.000 | 0.000 | 0.000 | 89.389 | 101.779 | 100.232 | 0.301 | 0.001 | 0.170 | 3.283 |
| 2 | 0.006 | -0.002 | -0.003 | 0.000 | 0.000 | 91.346 | 101.785 | 100.110 | 0.310 | 0.001 | 0.175 | 2.630 |
| 3 | 0.009 | -0.004 | -0.004 | 0.001 | 0.000 | 89.845 | 101.844 | 100.091 | 2.838 | 0.063 | 1.594 | 3.535 |
| 4 | 0.011 | -0.002 | -0.006 | 0.003 | 0.000 | 91.457 | 103.051 | 96.872 | 3.741 | 0.207 | 2.076 | 2.811 |
| 5 | 0.011 | -0.002 | -0.006 | 0.004 | 0.000 | 92.433 | 103.445 | 96.062 | 4.326 | 0.307 | 2.390 | 2.853 |
| 6 | 0.013 | -0.002 | -0.008 | 0.004 | 0.000 | 92.821 | 103.804 | 95.173 | 4.957 | 0.385 | 2.728 | 3.107 |
| 7 | 0.018 | -0.003 | -0.010 | 0.005 | 0.000 | 94.438 | 104.703 | 92.793 | 7.328 | 0.556 | 25.451 | 4.518 |
| 8 | 0.020 | -0.003 | -0.012 | 0.005 | 0.000 | 95.193 | 105.130 | 91.627 | 8.353 | 0.604 | 25.526 | 5.246 |
| 9 | 0.022 | -0.003 | -0.014 | 0.006 | 0.000 | 95.753 | 105.658 | 90.174 | 9.921 | 0.635 | 26.017 | 6.365 |
| 10 | 0.025 | -0.002 | -0.016 | 0.007 | 0.000 | 97.179 | 106.345 | 88.561 | 11.362 | 0.677 | 25.976 | 7.518 |
| 11 | 0.028 | -0.002 | -0.018 | 0.007 | 0.000 | 98.149 | 107.041 | 86.668 | 12.572 | 0.708 | 25.491 | 8.601 |
| 12 | 0.031 | -0.002 | -0.021 | 0.008 | 0.000 | 99.195 | 107.591 | 85.038 | 13.226 | 0.736 | 24.775 | 9.373 |
| 13 | 0.036 | -0.003 | -0.024 | 0.009 | 0.000 | 99.249 | 107.930 | 84.278 | 15.915 | 0.713 | 26.692 | 11.050 |
| 14 | 0.046 | -0.001 | -0.031 | 0.013 | 0.000 | 101.449 | 109.242 | 80.893 | 20.452 | 0.725 | 27.638 | 14.540 |
| 15 | 0.047 | 0.007 | -0.038 | 0.016 | 0.055 | 104.873 | 111.317 | 75.364 | 24.485 | 0.758 | 26.857 | 18.494 |
| 16 | 0.047 | 0.018 | -0.046 | 0.018 | 0.108 | 107.892 | 113.274 | 70.108 | 27.149 | 0.786 | 25.758 | 21.816 |
| 17 | 0.040 | 0.030 | -0.050 | 0.020 | 0.145 | 109.965 | 114.541 | 67.151 | 29.994 | 0.790 | 25.846 | 24.589 |
| 18 | 0.042 | 0.040 | -0.061 | 0.021 | 0.185 | 111.616 | 115.904 | 63.357 | 30.785 | 0.808 | 24.760 | 26.557 |
| 19 | 0.048 | 0.048 | -0.076 | 0.020 | 0.237 | 112.953 | 116.313 | 63.048 | 34.301 | 0.794 | 26.086 | 28.816 |
| 20 | 0.055 | 0.069 | -0.108 | 0.015 | 0.318 | 115.435 | 117.942 | 58.786 | 36.247 | 0.808 | 25.393 | 31.876 |
| 21 | 0.062 | 0.106 | -0.170 | -0.002 | 0.472 | 116.701 | 118.823 | 55.900 | 39.288 | 0.806 | 25.657 | 35.143 |
| 22 | 0.068 | 0.152 | -0.250 | -0.031 | 0.627 | 118.770 | 120.183 | 51.771 | 40.247 | 0.820 | 24.819 | 37.887 |
| 23 | 0.072 | 0.181 | -0.305 | -0.052 | 0.746 | 119.926 | 120.735 | 51.457 | 41.856 | 0.817 | 25.360 | 39.126 |
| 24 | 0.075 | 0.234 | -0.396 | -0.087 | 0.915 | 120.675 | 121.475 | 48.723 | 42.701 | 0.822 | 24.943 | 41.241 |
| 25 | 0.078 | 0.260 | -0.453 | -0.115 | 1.025 | 121.402 | 121.851 | 47.717 | 43.455 | 0.823 | 24.854 | 42.356 |
| 26 | 0.087 | 0.325 | -0.589 | -0.177 | 1.260 | 122.898 | 122.654 | 46.369 | 44.414 | 0.826 | 24.627 | 43.842 |
| 27 | 0.090 | 0.338 | -0.613 | -0.185 | 1.293 | 121.933 | 122.574 | 44.197 | 42.139 | 0.839 | 23.656 | 43.642 |
| 28 | 0.090 | 0.362 | -0.661 | -0.209 | 1.430 | 121.222 | 121.686 | 48.978 | 47.110 | 0.804 | 26.260 | 44.234 |
| 29 | 0.097 | 0.439 | -0.809 | -0.273 | 1.600 | 124.420 | 123.912 | 41.645 | 43.039 | 0.847 | 23.060 | 45.981 |
| 30 | 0.102 | 0.590 | -1.074 | -0.383 | 1.999 | 126.088 | 124.238 | 41.538 | 43.529 | 0.848 | 22.919 | 46.384 |
| 31 | 0.106 | 0.665 | -1.202 | -0.430 | 2.234 | 122.934 | 122.448 | 47.476 | 48.086 | 0.809 | 25.941 | 45.835 |
| 32 | 0.111 | 0.535 | -1.217 | -0.571 | 2.648 | 125.385 | 124.498 | 40.873 | 41.664 | 0.856 | 22.298 | 45.550 |
| 33 | 0.099 | 0.655 | -1.368 | -0.614 | 2.930 | 120.883 | 121.606 | 48.597 | 46.856 | 0.805 | 26.178 | 44.293 |

$$
\text { Torsion Shear Test on Fine Nevada Sand with a Target Void Ratio }=0.530\left(D_{r}=91.28 \%\right)
$$

| Test No.: | 31 |
| :--- | :---: |
| Test Date: | $9 / 6 / 11$ |
| Initial Height, $\mathrm{h}_{\mathrm{i}}:$ | 39.99 cm |
| Initial Void Ratio, e: | 0.535 |
| Final Mean Princpal Stress, $\sigma_{\mathrm{m}}:$ | 101.1 kPa |
| Max Friction Angle, $\phi:$ | 35.9 deg |
| b-value at failure: | 0.00 |
| Stress direction at failure, $\alpha:$ | 44.9 deg |


| Shear Band Notes |
| :--- |
| Point of Observation: |
|  |
| Inclination (from Vertical) |
| $15^{\circ}, 10^{\circ}, 15^{\circ}, 13^{\circ}, 10^{\circ}$ |
|  |
| Failure Notes: |
| SB's not so prominent |


| Point <br> (No.) | $\begin{gathered} \varepsilon_{\mathrm{z}} \\ (\%) \end{gathered}$ | $\begin{gathered} \varepsilon_{\mathrm{r}} \\ (\%) \end{gathered}$ | $\begin{gathered} \varepsilon_{\theta} \\ (\%) \end{gathered}$ | $\varepsilon_{\mathrm{v}}$ <br> (\%) | $\begin{gathered} \gamma_{\theta z} \\ (\%) \end{gathered}$ | $\begin{gathered} \sigma_{z} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \sigma_{\mathrm{r}} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \sigma_{\theta} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \tau_{\theta z} \\ (\mathrm{kPa}) \end{gathered}$ | b | $\alpha$ <br> ${ }^{\circ}$ ) | $\begin{gathered} \varphi \\ \left({ }^{\circ}\right) \\ \hline \end{gathered}$ |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 1 | 0.000 | 0.000 | 0.000 | 0.000 | 0.000 | 102.144 | 100.233 | 101.482 | 1.774 | 0.062 | 39.715 | 1.015 |
| 2 | 0.006 | -0.015 | 0.006 | -0.003 | 0.092 | 106.067 | 92.440 | 105.775 | 14.610 | 0.039 | 44.714 | 7.929 |
| 3 | 0.019 | -0.034 | 0.012 | -0.002 | 0.200 | 108.864 | 86.528 | 108.420 | 25.211 | 0.061 | 44.748 | 13.419 |
| 4 | 0.026 | -0.059 | 0.034 | 0.001 | 0.355 | 113.725 | 78.464 | 112.820 | 35.035 | 0.003 | 44.630 | 18.019 |
| 5 | 0.031 | -0.075 | 0.049 | 0.005 | 0.451 | 114.265 | 76.020 | 114.019 | 40.220 | 0.026 | 44.912 | 20.632 |
| 6 | 0.033 | -0.087 | 0.060 | 0.006 | 0.504 | 114.710 | 74.228 | 114.538 | 42.668 | 0.027 | 44.942 | 21.854 |
| 7 | 0.035 | -0.095 | 0.067 | 0.007 | 0.544 | 115.675 | 72.893 | 115.466 | 42.877 | 0.002 | 44.930 | 21.778 |
| 8 | 0.041 | -0.129 | 0.095 | 0.007 | 0.711 | 116.871 | 69.987 | 116.353 | 49.278 | 0.027 | 44.849 | 24.998 |
| 9 | 0.045 | -0.160 | 0.119 | 0.004 | 0.832 | 118.162 | 67.569 | 117.916 | 52.030 | 0.015 | 44.932 | 26.154 |
| 10 | 0.047 | -0.178 | 0.132 | 0.001 | 0.905 | 118.715 | 66.603 | 118.285 | 53.457 | 0.015 | 44.885 | 26.815 |
| 11 | 0.052 | -0.225 | 0.164 | -0.009 | 1.082 | 119.607 | 64.186 | 119.144 | 56.368 | 0.010 | 44.882 | 28.177 |
| 12 | 0.058 | -0.280 | 0.196 | -0.025 | 1.281 | 120.686 | 62.673 | 120.208 | 58.665 | 0.008 | 44.883 | 29.148 |
| 13 | 0.056 | -0.329 | 0.228 | -0.044 | 1.463 | 121.186 | 61.330 | 120.962 | 60.540 | 0.007 | 44.947 | 30.002 |
| 14 | 0.060 | -0.395 | 0.263 | -0.072 | 1.692 | 121.918 | 59.967 | 121.484 | 62.416 | 0.005 | 44.900 | 30.855 |
| 15 | 0.060 | -0.450 | 0.292 | -0.097 | 1.877 | 122.366 | 58.978 | 121.871 | 63.698 | 0.004 | 44.889 | 31.440 |
| 16 | 0.057 | -0.515 | 0.327 | -0.131 | 2.107 | 122.547 | 58.127 | 122.377 | 65.453 | 0.009 | 44.963 | 32.309 |
| 17 | 0.056 | -0.588 | 0.363 | -0.168 | 2.355 | 123.436 | 57.121 | 123.098 | 66.461 | 0.002 | 44.927 | 32.627 |
| 18 | 0.053 | -0.624 | 0.381 | -0.190 | 2.501 | 122.664 | 57.501 | 122.583 | 67.303 | 0.016 | 44.983 | 33.289 |
| 19 | 0.044 | -0.742 | 0.439 | -0.259 | 2.891 | 124.005 | 55.668 | 123.671 | 68.528 | 0.003 | 44.930 | 33.598 |
| 20 | 0.044 | -0.766 | 0.449 | -0.273 | 2.971 | 124.068 | 55.510 | 123.638 | 68.793 | 0.003 | 44.911 | 33.741 |
| 21 | 0.051 | -0.866 | 0.484 | -0.331 | 3.280 | 124.351 | 54.722 | 123.874 | 69.882 | 0.004 | 44.902 | 34.267 |
| 22 | 0.058 | -0.927 | 0.503 | -0.366 | 3.461 | 124.632 | 54.442 | 124.105 | 70.345 | 0.003 | 44.893 | 34.446 |
| 23 | 0.062 | -1.003 | 0.529 | -0.412 | 3.699 | 124.900 | 54.059 | 124.578 | 70.884 | 0.001 | 44.935 | 34.629 |
| 24 | 0.079 | -1.103 | 0.555 | -0.469 | 3.989 | 126.129 | 53.635 | 124.629 | 71.574 | -0.001 | 44.700 | 34.812 |
| 25 | 0.084 | -1.175 | 0.573 | -0.518 | 4.246 | 124.462 | 54.597 | 124.099 | 72.428 | 0.019 | 44.928 | 35.646 |
| 26 | 0.090 | -1.231 | 0.591 | -0.551 | 4.384 | 125.209 | 52.941 | 124.750 | 72.035 | 0.000 | 44.909 | 35.196 |
| 27 | 0.100 | -1.292 | 0.604 | -0.588 | 4.583 | 124.506 | 53.956 | 124.722 | 72.973 | 0.016 | 44.958 | 35.845 |
| 28 | 0.111 | -1.410 | 0.639 | -0.660 | 4.919 | 125.640 | 52.655 | 125.361 | 72.955 | 0.001 | 44.945 | 35.543 |
| 29 | 0.123 | -1.485 | 0.656 | -0.705 | 5.132 | 125.483 | 52.514 | 125.017 | 73.137 | 0.003 | 44.909 | 35.728 |
| 30 | 0.134 | -1.564 | 0.677 | -0.754 | 5.369 | 125.587 | 52.427 | 125.267 | 73.331 | 0.002 | 44.937 | 35.779 |
| 31 | 0.137 | -1.626 | 0.697 | -0.793 | 5.555 | 125.731 | 52.304 | 125.301 | 73.421 | 0.001 | 44.916 | 35.800 |
| 32 | 0.153 | -1.717 | 0.718 | -0.846 | 5.814 | 125.481 | 52.292 | 125.021 | 73.440 | 0.003 | 44.910 | 35.898 |
| 33 | 0.161 | -1.779 | 0.735 | -0.882 | 6.005 | 125.683 | 52.288 | 125.186 | 73.443 | 0.002 | 44.903 | 35.839 |

# Torsion Shear Test on Fine Nevada Sand with a Target Void Ratio $=\mathbf{0 . 5 3 0}\left(\mathrm{D}_{\mathrm{r}}=\mathbf{9 1 . 2 8 \%}\right)$ 

| Test No.: | $32^{*}$ |
| :--- | :---: |
| Test Date: | $4 / 30 / 11$ |
| Initial Height, $\mathrm{h}_{\mathrm{i}}$ : | 40.06 cm |
| Initial Void Ratio, e: | 0.560 |
| Final Mean Princpal Stress, $\sigma_{\mathrm{m}}:$ | 84.0 kPa |
| Max Friction Angle, $\phi:$ | 40.9 deg |
| b-value at failure: | 0.18 |
| Stress direction at failure, $\alpha:$ | 31.6 deg |


| Shear Band Notes |  |
| :--- | :--- |
| Point of Observation: | 31 |
| Inclination (from Vertical) |  |
| $3 @ 15^{\circ}, 10^{\circ}$ |  |
|  |  |
| Failure Notes: |  |
| One SB along top cap |  |


| Point (No.) | $\begin{gathered} \varepsilon_{\mathrm{z}} \\ (\%) \\ \hline \end{gathered}$ | $\begin{gathered} \varepsilon_{\mathrm{r}} \\ (\%) \\ \hline \end{gathered}$ | $\begin{gathered} \varepsilon_{\theta} \\ (\%) \\ \hline \end{gathered}$ | $\begin{gathered} \varepsilon_{\mathrm{v}} \\ (\%) \\ \hline \end{gathered}$ | $\begin{array}{r} \gamma_{\theta z} \\ (\%) \\ \hline \end{array}$ | $\begin{gathered} \sigma_{z} \\ (\mathrm{kPa}) \\ \hline \end{gathered}$ | $\begin{gathered} \sigma_{r} \\ (\mathrm{kPa}) \\ \hline \end{gathered}$ | $\begin{gathered} \sigma_{\theta} \\ (\mathrm{kPa}) \\ \hline \end{gathered}$ | $\begin{gathered} \boldsymbol{\tau}_{\theta z} \\ (\mathrm{kPa}) \\ \hline \end{gathered}$ | b | $\begin{gathered} \alpha \\ \left(^{( }\right) \\ \hline \end{gathered}$ | $\begin{gathered} \varphi \\ \left({ }^{\circ}\right) \\ \hline \end{gathered}$ |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 1 | 0.000 | 0.000 | 0.000 | 0.000 | 0.000 | 90.037 | 101.355 | 101.552 | 2.082 | 0.843 | 43.646 | 4.116 |
| 2 | 0.040 | -0.052 | 0.017 | 0.005 | 0.143 | 85.909 | 82.540 | 110.159 | 28.804 | 0.256 | 32.192 | 18.380 |
| 3 | 0.043 | -0.066 | 0.032 | 0.009 | 0.239 | 85.031 | 78.914 | 111.636 | 33.863 | 0.239 | 32.106 | 21.350 |
| 4 | 0.043 | -0.079 | 0.045 | 0.010 | 0.325 | 84.381 | 76.565 | 112.736 | 37.833 | 0.235 | 32.225 | 23.749 |
| 5 | 0.043 | -0.088 | 0.054 | 0.010 | 0.384 | 84.188 | 75.085 | 113.641 | 40.002 | 0.228 | 32.135 | 25.015 |
| 6 | 0.044 | -0.117 | 0.080 | 0.007 | 0.559 | 83.488 | 72.109 | 114.922 | 45.291 | 0.226 | 32.351 | 28.286 |
| 7 | 0.045 | -0.144 | 0.100 | 0.001 | 0.680 | 82.844 | 68.851 | 116.556 | 48.002 | 0.205 | 31.789 | 29.971 |
| 8 | 0.046 | -0.165 | 0.113 | -0.005 | 0.774 | 82.362 | 67.360 | 116.883 | 49.984 | 0.204 | 31.824 | 31.313 |
| 9 | 0.049 | -0.185 | 0.125 | -0.011 | 0.864 | 82.161 | 66.142 | 117.595 | 51.692 | 0.200 | 31.770 | 32.390 |
| 10 | 0.052 | -0.217 | 0.141 | -0.024 | 1.000 | 81.875 | 64.699 | 118.292 | 53.728 | 0.197 | 31.746 | 33.717 |
| 11 | 0.048 | -0.267 | 0.170 | -0.050 | 1.218 | 81.415 | 62.623 | 119.149 | 56.448 | 0.193 | 31.702 | 35.551 |
| 12 | 0.042 | -0.303 | 0.189 | -0.072 | 1.380 | 80.907 | 61.864 | 119.743 | 57.908 | 0.194 | 31.723 | 36.656 |
| 13 | 0.039 | -0.321 | 0.198 | -0.083 | 1.454 | 80.797 | 61.391 | 119.932 | 58.425 | 0.193 | 31.695 | 37.019 |
| 14 | 0.036 | -0.348 | 0.212 | -0.101 | 1.575 | 80.552 | 60.727 | 120.062 | 59.282 | 0.193 | 31.707 | 37.670 |
| 15 | 0.033 | -0.370 | 0.221 | -0.116 | 1.673 | 80.548 | 60.428 | 120.274 | 60.232 | 0.194 | 31.791 | 38.304 |
| 16 | 0.025 | -0.431 | 0.249 | -0.157 | 1.931 | 80.481 | 59.178 | 121.041 | 61.431 | 0.188 | 31.637 | 39.049 |
| 17 | 0.024 | -0.440 | 0.253 | -0.163 | 1.970 | 80.428 | 59.034 | 121.053 | 61.595 | 0.188 | 31.639 | 39.180 |
| 18 | 0.022 | -0.473 | 0.265 | -0.186 | 2.114 | 80.344 | 58.622 | 121.247 | 62.207 | 0.188 | 31.640 | 39.614 |
| 19 | 0.011 | -0.489 | 0.277 | -0.201 | 2.199 | 80.250 | 58.262 | 121.374 | 62.656 | 0.187 | 31.634 | 39.947 |
| 20 | 0.011 | -0.517 | 0.285 | -0.221 | 2.318 | 80.213 | 57.995 | 121.489 | 62.987 | 0.186 | 31.625 | 40.178 |
| 21 | 0.010 | -0.532 | 0.290 | -0.232 | 2.384 | 80.313 | 57.900 | 121.531 | 63.246 | 0.186 | 31.648 | 40.315 |
| 22 | 0.010 | -0.533 | 0.291 | -0.232 | 2.388 | 80.371 | 57.890 | 121.643 | 63.248 | 0.186 | 31.626 | 40.279 |
| 23 | 0.010 | -0.534 | 0.291 | -0.234 | 2.391 | 80.408 | 57.890 | 121.697 | 63.277 | 0.185 | 31.622 | 40.278 |
| 24 | 0.009 | -0.543 | 0.294 | -0.240 | 2.426 | 80.347 | 57.753 | 121.710 | 63.371 | 0.185 | 31.612 | 40.365 |
| 25 | 0.009 | -0.543 | 0.293 | -0.240 | 2.430 | 80.291 | 57.745 | 121.599 | 63.384 | 0.186 | 31.633 | 40.411 |
| 26 | 0.009 | -0.545 | 0.294 | -0.242 | 2.433 | 80.311 | 57.747 | 121.634 | 63.389 | 0.186 | 31.628 | 40.402 |
| 27 | 0.009 | -0.545 | 0.294 | -0.242 | 2.437 | 80.300 | 57.752 | 121.595 | 63.397 | 0.186 | 31.637 | 40.418 |
| 28 | 0.009 | -0.546 | 0.294 | -0.243 | 2.441 | 80.255 | 57.740 | 121.510 | 63.390 | 0.186 | 31.649 | 40.441 |
| 29 | -0.003 | -0.623 | 0.322 | -0.304 | 2.810 | 80.295 | 57.191 | 122.079 | 64.147 | 0.184 | 31.585 | 40.885 |
| 30 | -0.003 | -0.626 | 0.323 | -0.306 | 2.825 | 80.315 | 57.177 | 122.061 | 64.128 | 0.183 | 31.583 | 40.865 |
| 31 | -0.010 | -0.662 | 0.336 | -0.337 | 3.029 | 80.513 | 57.117 | 121.937 | 64.114 | 0.182 | 31.592 | 40.785 |
| 32 | -0.016 | -0.712 | 0.354 | -0.374 | 3.360 | 79.984 | 57.369 | 121.803 | 63.144 | 0.182 | 31.484 | 40.326 |
| 33 | -0.005 | -0.642 | 0.327 | -0.320 | 2.909 | 80.396 | 57.142 | 121.676 | 64.147 | 0.184 | 31.648 | 40.891 |

# Torsion Shear Test on Fine Nevada Sand with a Target Void Ratio $=\mathbf{0 . 5 3 0}\left(\mathrm{D}_{\mathrm{r}}=\mathbf{9 1 . 2 8 \%}\right)$ 

| Test No.: | 33 |
| :--- | :--- |
| Test Date: | $9 / 21 / 11$ |
| Initial Height, $\mathrm{h}_{\mathrm{i}}$ : | 39.96 cm |
| Initial Void Ratio, e: | 0.540 |
| Final Mean Princpal Stress, $\sigma_{\mathrm{m}}:$ | 101.6 kPa |
| Max Friction Angle, $\phi:$ | 45.0 deg |
| b-value at failure: | 0.50 |
| Stress direction at failure, $\alpha:$ | 45.0 deg |$\quad$| Shear Band Notes |  |
| :--- | :--- |
| Point of Observation: |  |
|  | 33 |
| Inclination (from Vertical) |  |
| $10^{\circ}, 2 @ 20^{\circ}$ |  |
|  |  |


| Point <br> (No.) | $\begin{gathered} \varepsilon_{\mathrm{z}} \\ (\%) \end{gathered}$ | $\begin{array}{r} \varepsilon_{\mathrm{r}} \\ (\%) \end{array}$ | $\begin{gathered} \varepsilon_{\theta} \\ (\%) \end{gathered}$ | $\begin{gathered} \varepsilon_{\mathrm{v}} \\ (\%) \end{gathered}$ | $\begin{gathered} \gamma_{\theta z} \\ (\%) \end{gathered}$ | $\begin{gathered} \sigma_{z} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \sigma_{\mathrm{r}} \\ (\mathrm{kPa}) \\ \hline \end{gathered}$ | $\begin{gathered} \sigma_{\theta} \\ (\mathrm{kPa}) \\ \hline \end{gathered}$ | $\begin{gathered} \boldsymbol{\tau}_{\theta z} \\ (\mathrm{kPa}) \\ \hline \end{gathered}$ | b | $\begin{gathered} \alpha \\ \left({ }^{\circ}\right) \end{gathered}$ | $\begin{gathered} \varphi \\ \left({ }^{\circ}\right) \end{gathered}$ |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 1 | 0.000 | 0.000 | 0.000 | 0.000 | 0.000 | 101.569 | 101.341 | 101.463 | 1.771 | 0.451 | 44.135 | 1.000 |
| 2 | 0.001 | 0.000 | 0.002 | 0.002 | 0.009 | 101.264 | 101.401 | 101.186 | 9.032 | 0.510 | 44.877 | 5.119 |
| 3 | 0.002 | 0.000 | 0.002 | 0.004 | 0.017 | 101.392 | 101.322 | 101.356 | 14.010 | 0.498 | 44.964 | 7.944 |
| 4 | 0.003 | -0.001 | 0.003 | 0.006 | 0.044 | 101.351 | 101.419 | 101.299 | 18.356 | 0.503 | 44.960 | 10.437 |
| 5 | 0.005 | -0.001 | 0.004 | 0.007 | 0.059 | 101.594 | 101.402 | 101.350 | 22.470 | 0.498 | 44.844 | 12.794 |
| 6 | 0.006 | 0.000 | 0.004 | 0.009 | 0.095 | 101.582 | 101.429 | 101.116 | 26.123 | 0.502 | 44.744 | 14.937 |
| 7 | 0.007 | -0.001 | 0.006 | 0.012 | 0.133 | 101.359 | 101.302 | 101.494 | 29.253 | 0.498 | 44.934 | 16.763 |
| 8 | 0.007 | 0.000 | 0.007 | 0.014 | 0.194 | 101.570 | 101.295 | 101.537 | 32.244 | 0.496 | 44.985 | 18.512 |
| 9 | 0.007 | 0.001 | 0.008 | 0.016 | 0.266 | 101.634 | 101.308 | 101.538 | 34.819 | 0.496 | 44.961 | 20.045 |
| 10 | 0.005 | 0.003 | 0.011 | 0.019 | 0.335 | 101.723 | 101.383 | 101.224 | 39.598 | 0.499 | 44.819 | 22.969 |
| 11 | 0.005 | 0.003 | 0.010 | 0.019 | 0.339 | 101.801 | 101.384 | 101.236 | 39.727 | 0.498 | 44.796 | 23.038 |
| 12 | 0.003 | 0.005 | 0.011 | 0.019 | 0.406 | 101.915 | 101.393 | 101.522 | 42.195 | 0.496 | 44.867 | 24.508 |
| 13 | 0.001 | 0.007 | 0.013 | 0.020 | 0.464 | 101.862 | 101.374 | 101.270 | 44.122 | 0.498 | 44.808 | 25.749 |
| 14 | -0.002 | 0.009 | 0.015 | 0.022 | 0.521 | 102.006 | 101.317 | 101.542 | 46.073 | 0.495 | 44.856 | 26.917 |
| 15 | -0.004 | 0.011 | 0.016 | 0.022 | 0.565 | 101.953 | 101.280 | 101.523 | 47.580 | 0.495 | 44.870 | 27.883 |
| 16 | -0.008 | 0.014 | 0.016 | 0.022 | 0.621 | 101.946 | 101.347 | 101.457 | 49.419 | 0.496 | 44.858 | 29.073 |
| 17 | -0.012 | 0.017 | 0.017 | 0.022 | 0.676 | 101.762 | 101.341 | 101.237 | 51.120 | 0.498 | 44.853 | 30.242 |
| 18 | -0.015 | 0.019 | 0.017 | 0.021 | 0.711 | 101.740 | 101.366 | 101.229 | 52.170 | 0.499 | 44.860 | 30.935 |
| 19 | -0.019 | 0.022 | 0.017 | 0.020 | 0.761 | 101.493 | 101.375 | 101.442 | 53.553 | 0.499 | 44.987 | 31.856 |
| 20 | -0.025 | 0.026 | 0.018 | 0.019 | 0.813 | 101.475 | 101.386 | 101.471 | 54.809 | 0.499 | 44.999 | 32.693 |
| 21 | -0.032 | 0.030 | 0.017 | 0.016 | 0.869 | 101.435 | 101.420 | 101.471 | 56.082 | 0.500 | 44.991 | 33.559 |
| 22 | -0.039 | 0.035 | 0.017 | 0.013 | 0.931 | 101.471 | 101.436 | 101.535 | 57.251 | 0.499 | 44.984 | 34.335 |
| 23 | -0.047 | 0.039 | 0.017 | 0.009 | 0.988 | 101.546 | 101.461 | 101.642 | 58.378 | 0.499 | 44.976 | 35.073 |
| 24 | -0.066 | 0.049 | 0.014 | -0.002 | 1.125 | 101.909 | 101.426 | 101.732 | 60.539 | 0.497 | 44.958 | 36.481 |
| 25 | -0.077 | 0.055 | 0.012 | -0.010 | 1.200 | 101.857 | 101.460 | 101.515 | 61.465 | 0.498 | 44.920 | 37.190 |
| 26 | -0.106 | 0.069 | 0.009 | -0.028 | 1.356 | 101.548 | 101.301 | 101.487 | 63.115 | 0.498 | 44.986 | 38.442 |
| 27 | -0.140 | 0.085 | 0.002 | -0.053 | 1.545 | 101.821 | 101.400 | 101.733 | 64.781 | 0.497 | 44.981 | 39.531 |
| 28 | -0.160 | 0.094 | -0.002 | -0.068 | 1.655 | 102.019 | 101.406 | 102.005 | 65.625 | 0.495 | 44.997 | 40.039 |
| 29 | -0.199 | 0.112 | -0.011 | -0.098 | 1.853 | 101.876 | 101.435 | 101.722 | 66.894 | 0.497 | 44.967 | 41.080 |
| 30 | -0.241 | 0.133 | -0.023 | -0.132 | 2.059 | 101.631 | 101.438 | 101.165 | 67.851 | 0.500 | 44.902 | 42.002 |
| 31 | -0.334 | 0.177 | -0.083 | -0.240 | 2.676 | 102.001 | 101.419 | 101.810 | 70.208 | 0.497 | 44.961 | 43.548 |
| 32 | -0.439 | 0.228 | -0.169 | -0.379 | 3.377 | 101.934 | 101.355 | 101.895 | 71.869 | 0.496 | 44.992 | 44.845 |
| 33 | -0.456 | 0.236 | -0.182 | -0.402 | 3.490 | 02.033 | 01.382 | 102.065 | 72.118 | 0.495 | 44.994 | 44.967 |

Torsion Shear Test on Fine Nevada Sand with a Target Void Ratio $=\mathbf{0 . 5 3 0}\left(\mathrm{D}_{\mathrm{r}}=\mathbf{9 1 . 2 8 \%}\right)$

| Test No.: | 34 |
| :--- | :---: |
| Test Date: | $9 / 29 / 11$ |
| Initial Height, $\mathrm{h}_{\mathrm{i}}$ : | 39.99 cm |
| Initial Void Ratio, e: | 0.541 |
| Final Mean Princpal Stress, $\sigma_{\mathrm{m}}:$ | 101.5 kPa |
| Max Friction Angle, $\phi:$ | 34.9 deg |
| b-value at failure: | 1.00 |
| Stress direction at failure, $\alpha$ : | 48.7 deg |


| Shear Band Notes <br> Point of Observation: <br>  <br>  <br> Inclination (from Vertical) <br> $2^{\circ}, 5^{\circ}, 10^{\circ}, 20^{\circ}, 25^{\circ}$ <br> Failure Notes: |
| :--- |


| Point (No.) | $\begin{gathered} \varepsilon_{\mathrm{z}} \\ (\%) \\ \hline \end{gathered}$ | $\begin{gathered} \varepsilon_{\mathrm{r}} \\ (\%) \\ \hline \end{gathered}$ | $\begin{gathered} \varepsilon_{\theta} \\ (\%) \\ \hline \end{gathered}$ | $\begin{gathered} \varepsilon_{\mathrm{v}} \\ (\%) \\ \hline \end{gathered}$ | $\begin{aligned} & \gamma_{\theta z} \\ & (\%) \\ & \hline \end{aligned}$ | $\begin{gathered} \sigma_{z} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \sigma_{\mathrm{r}} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \sigma_{\theta} \\ (\mathrm{kPa}) \\ \hline \end{gathered}$ | $\begin{gathered} \tau_{\theta z} \\ (\mathrm{kPa}) \\ \hline \end{gathered}$ | b | $\begin{gathered} \alpha \\ \left({ }^{\circ}\right) \\ \hline \end{gathered}$ | $\begin{gathered} \varphi \\ \left({ }^{\circ}\right) \\ \hline \end{gathered}$ |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 1 | 0.000 | 0.000 | 0.000 | 0.000 | 0.000 | 101.564 | 102.066 | 101.639 | 1.344 | 0.673 | 44.205 | 0.758 |
| 2 | -0.012 | 0.057 | -0.008 | 0.036 | 0.221 | 91.553 | 122.113 | 91.642 | 32.286 | 0.973 | 44.961 | 20.639 |
| 3 | -0.027 | 0.084 | -0.014 | 0.043 | 0.330 | 89.771 | 125.335 | 89.755 | 36.650 | 0.985 | 44.994 | 24.098 |
| 4 | -0.046 | 0.114 | -0.021 | 0.047 | 0.445 | 88.713 | 127.298 | 88.727 | 39.533 | 0.988 | 44.995 | 26.461 |
| 5 | -0.050 | 0.120 | -0.023 | 0.047 | 0.474 | 88.514 | 127.684 | 88.460 | 39.982 | 0.990 | 44.981 | 26.862 |
| 6 | -0.053 | 0.126 | -0.026 | 0.047 | 0.502 | 88.222 | 128.020 | 88.258 | 40.457 | 0.992 | 44.987 | 27.289 |
| 7 | -0.060 | 0.133 | -0.027 | 0.047 | 0.528 | 88.217 | 128.342 | 88.078 | 40.875 | 0.992 | 44.951 | 27.627 |
| 8 | -0.063 | 0.139 | -0.030 | 0.047 | 0.557 | 87.986 | 128.600 | 87.882 | 41.325 | 0.992 | 44.964 | 28.032 |
| 9 | -0.068 | 0.146 | -0.032 | 0.046 | 0.587 | 87.817 | 128.905 | 87.754 | 41.779 | 0.992 | 44.978 | 28.420 |
| 10 | -0.075 | 0.155 | -0.034 | 0.046 | 0.616 | 87.636 | 129.247 | 87.539 | 42.175 | 0.994 | 44.967 | 28.785 |
| 11 | -0.082 | 0.162 | -0.036 | 0.044 | 0.646 | 87.629 | 129.469 | 87.551 | 42.621 | 0.991 | 44.974 | 29.117 |
| 12 | -0.087 | 0.169 | -0.039 | 0.044 | 0.675 | 87.460 | 129.795 | 87.196 | 43.015 | 0.994 | 44.912 | 29.510 |
| 13 | -0.094 | 0.177 | -0.041 | 0.042 | 0.708 | 87.408 | 130.043 | 87.454 | 43.475 | 0.990 | 44.985 | 29.818 |
| 14 | -0.099 | 0.185 | -0.045 | 0.040 | 0.742 | 87.296 | 130.346 | 87.283 | 43.804 | 0.991 | 44.996 | 30.121 |
| 15 | -0.105 | 0.193 | -0.049 | 0.038 | 0.775 | 86.972 | 130.579 | 86.943 | 44.015 | 0.996 | 44.991 | 30.409 |
| 16 | -0.110 | 0.198 | -0.051 | 0.036 | 0.798 | 87.014 | 130.708 | 86.942 | 44.244 | 0.994 | 44.977 | 30.576 |
| 17 | -0.118 | 0.206 | -0.054 | 0.035 | 0.827 | 86.942 | 130.883 | 86.818 | 44.525 | 0.994 | 44.960 | 30.830 |
| 18 | -0.168 | 0.261 | -0.077 | 0.016 | 1.044 | 86.241 | 132.247 | 85.935 | 46.443 | 0.997 | 44.906 | 32.649 |
| 19 | -0.241 | 0.332 | -0.110 | -0.019 | 1.348 | 85.804 | 133.335 | 85.606 | 48.062 | 0.996 | 44.941 | 34.110 |
| 20 | -0.261 | 0.349 | -0.118 | -0.029 | 1.427 | 85.629 | 133.446 | 85.431 | 48.212 | 0.997 | 44.941 | 34.311 |
| 21 | -0.272 | 0.359 | -0.120 | -0.034 | 1.469 | 85.651 | 133.548 | 85.620 | 48.403 | 0.995 | 44.991 | 34.418 |
| 22 | -0.273 | 0.359 | -0.121 | -0.035 | 1.472 | 85.679 | 133.554 | 85.620 | 48.422 | 0.995 | 44.983 | 34.427 |
| 23 | -0.274 | 0.360 | -0.121 | -0.035 | 1.476 | 85.681 | 133.562 | 85.609 | 48.426 | 0.995 | 44.978 | 34.432 |
| 24 | -0.280 | 0.368 | -0.128 | -0.040 | 1.519 | 85.551 | 133.669 | 85.425 | 48.546 | 0.996 | 44.963 | 34.602 |
| 25 | -0.296 | 0.381 | -0.134 | -0.049 | 1.581 | 85.464 | 133.832 | 85.386 | 48.667 | 0.997 | 44.977 | 34.730 |
| 26 | -0.296 | 0.381 | -0.134 | -0.049 | 1.584 | 85.504 | 133.818 | 85.444 | 48.697 | 0.996 | 44.982 | 34.731 |
| 27 | -0.308 | 0.391 | -0.138 | -0.054 | 1.631 | 85.347 | 133.825 | 85.181 | 48.727 | 0.998 | 44.951 | 34.854 |

# Torsion Shear Test on Fine Nevada Sand with a Target Void Ratio $=\mathbf{0 . 5 3 0}\left(\mathrm{D}_{\mathrm{r}}=\mathbf{9 1 . 2 8 \%}\right)$ 

| Test No.: | $35^{*}$ |
| :--- | :---: |
| Test Date: | $5 / 4 / 11$ |
| Initial Height, $\mathrm{h}_{\mathrm{i}}:$ | 40.06 cm |
| Initial Void Ratio, e: | 0.531 |
| Final Mean Princpal Stress, $\sigma_{\mathrm{m}}:$ | 96.2 kPa |
| Max Friction Angle, $\phi:$ | 38.1 deg |
| b-value at failure: | 0.17 |
| Stress direction at failure, $\alpha:$ | 65.1 deg |


| Shear Band Notes |  |
| :--- | :--- |
| Point of Observation: | 21 |
| Inclination (from Vertical) |  |
| $0^{\circ}$ |  |
| Failure Notes: |  |
| SB along top cap |  |


| Point (No.) | $\begin{gathered} \varepsilon_{\mathrm{z}} \\ (\%) \end{gathered}$ | $\begin{array}{r} \varepsilon_{\mathrm{r}} \\ (\%) \end{array}$ | $\begin{gathered} \varepsilon_{\theta} \\ (\%) \end{gathered}$ | $\begin{gathered} \varepsilon_{\mathrm{v}} \\ (\%) \end{gathered}$ | $\begin{gathered} \gamma_{\theta z} \\ (\%) \end{gathered}$ | $\begin{gathered} \sigma_{z} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \sigma_{\mathrm{r}} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \sigma_{\theta} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \tau_{\theta z} \\ (\mathrm{kPa}) \end{gathered}$ | b | $\begin{gathered} \alpha \\ \left({ }^{\circ}\right) \end{gathered}$ | $\begin{gathered} \varphi \\ \left({ }^{\circ}\right) \end{gathered}$ |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 1 | 0.000 | 0.000 | 0.000 | 0.000 | 0.000 | 89.142 | 101.408 | 101.417 | 0.480 | 0.925 | 89.738 | 0.271 |
| 2 | -0.015 | -0.001 | 0.020 | 0.003 | 0.068 | 85.654 | 95.103 | 110.925 | 10.699 | 0.132 | 71.760 | 7.421 |
| 3 | -0.037 | -0.001 | 0.043 | 0.006 | 0.068 | 83.282 | 90.543 | 118.196 | 14.804 | 0.020 | 68.478 | 11.191 |
| 4 | -0.056 | -0.002 | 0.069 | 0.011 | 0.069 | 81.059 | 86.698 | 124.058 | 18.739 | 0.040 | 67.545 | 14.542 |
| 5 | -0.065 | -0.010 | 0.086 | 0.012 | 0.069 | 79.532 | 82.933 | 130.070 | 19.176 | 0.056 | 64.567 | 16.576 |
| 6 | -0.102 | -0.013 | 0.130 | 0.015 | 0.166 | 77.639 | 79.925 | 134.545 | 25.752 | 0.106 | 66.659 | 20.490 |
| 7 | -0.145 | -0.028 | 0.188 | 0.015 | 0.215 | 75.982 | 76.326 | 140.391 | 29.631 | 0.129 | 66.385 | 23.747 |
| 8 | -0.183 | -0.047 | 0.242 | 0.012 | 0.215 | 73.949 | 72.623 | 145.903 | 28.450 | 0.119 | 63.914 | 25.123 |
| 9 | -0.202 | -0.049 | 0.258 | 0.008 | 0.215 | 74.209 | 73.650 | 143.746 | 32.046 | 0.137 | 66.219 | 25.906 |
| 10 | -0.255 | -0.073 | 0.325 | -0.003 | 0.215 | 72.511 | 70.391 | 148.955 | 32.661 | 0.136 | 64.871 | 27.762 |
| 11 | -0.334 | -0.101 | 0.412 | -0.024 | 0.214 | 71.682 | 68.584 | 151.895 | 34.659 | 0.144 | 64.881 | 29.443 |
| 12 | -0.424 | -0.131 | 0.504 | -0.051 | 0.395 | 70.947 | 67.187 | 154.039 | 36.213 | 0.149 | 64.913 | 30.743 |
| 13 | -0.487 | -0.161 | 0.573 | -0.076 | 0.536 | 70.383 | 65.939 | 156.164 | 37.091 | 0.152 | 64.713 | 31.729 |
| 14 | -0.595 | -0.195 | 0.675 | -0.115 | 0.746 | 69.655 | 64.647 | 158.064 | 37.956 | 0.154 | 64.549 | 32.717 |
| 15 | -0.650 | -0.215 | 0.727 | -0.137 | 0.903 | 69.534 | 64.426 | 158.246 | 39.030 | 0.157 | 64.881 | 33.237 |
| 16 | -0.744 | -0.184 | 0.773 | -0.156 | 1.109 | 76.927 | 64.195 | 175.278 | 39.196 | 0.185 | 62.605 | 34.593 |
| 17 | -0.932 | -0.240 | 0.922 | -0.251 | 1.252 | 67.364 | 62.458 | 159.103 | 41.054 | 0.158 | 65.175 | 34.916 |
| 18 | -1.005 | -0.265 | 0.986 | -0.284 | 1.491 | 67.253 | 62.177 | 159.755 | 41.405 | 0.158 | 65.160 | 35.216 |
| 19 | -1.101 | -0.299 | 1.070 | -0.330 | 1.816 | 67.000 | 61.726 | 160.287 | 41.868 | 0.160 | 65.175 | 35.629 |
| 20 | -1.170 | -0.330 | 1.133 | -0.366 | 2.084 | 66.746 | 61.523 | 160.565 | 42.105 | 0.159 | 65.186 | 35.829 |
| 21 | -1.258 | -0.367 | 1.213 | -0.412 | 2.398 | 66.629 | 60.942 | 161.668 | 42.647 | 0.162 | 65.129 | 36.364 |
| 22 | -1.310 | -0.392 | 1.261 | -0.441 | 2.584 | 66.503 | 60.716 | 162.082 | 43.058 | 0.162 | 65.175 | 36.654 |
| 23 | -1.394 | -0.427 | 1.335 | -0.486 | 2.875 | 66.077 | 60.410 | 162.094 | 43.291 | 0.162 | 65.207 | 36.886 |
| 24 | -1.469 | -0.462 | 1.401 | -0.529 | 3.164 | 66.056 | 60.156 | 162.630 | 43.397 | 0.162 | 65.132 | 37.069 |
| 25 | -1.528 | -0.490 | 1.453 | -0.565 | 3.382 | 65.904 | 59.965 | 162.907 | 43.716 | 0.163 | 65.171 | 37.301 |
| 26 | -1.607 | -0.523 | 1.521 | -0.609 | 3.629 | 65.771 | 59.746 | 163.209 | 43.975 | 0.163 | 65.183 | 37.521 |
| 27 | -1.675 | -0.556 | 1.582 | -0.649 | 3.892 | 65.702 | 59.455 | 163.744 | 44.236 | 0.164 | 65.155 | 37.788 |
| 28 | -1.746 | -0.591 | 1.645 | -0.692 | 4.124 | 65.537 | 59.251 | 164.039 | 44.350 | 0.164 | 65.123 | 37.940 |
| 29 | -1.818 | -0.626 | 1.708 | -0.737 | 4.373 | 65.414 | 59.027 | 164.213 | 44.480 | 0.165 | 65.111 | 38.105 |

$$
\text { Torsion Shear Test on Fine Nevada Sand with a Target Void Ratio }=0.530\left(D_{r}=91.28 \%\right)
$$

| Test No.: | 36 |
| :--- | :---: |
| Test Date: | $10 / 4 / 11$ |
| Initial Height, $\mathrm{h}_{\mathrm{i}}$ : | 39.96 cm |
| Initial Void Ratio, e: | 0.533 |
| Final Mean Princpal Stress, $\sigma_{\mathrm{m}}:$ | 101.7 kPa |
| Max Friction Angle, $\phi:$ | 37.2 deg |
| b-value at failure: | 0.25 |
| Stress direction at failure, $\alpha$ : | 67.8 deg |


| Shear Band Notes |  |
| :--- | :--- |
| Point of Observation: | 33 |
| Inclination (from Vertical) |  |
| Failure Notes: |  |
| Slip along top cap |  |


| Point (No.) | $\begin{gathered} \varepsilon_{\mathrm{z}} \\ (\%) \end{gathered}$ | $\begin{gathered} \varepsilon_{\mathrm{r}} \\ (\%) \end{gathered}$ | $\begin{gathered} \varepsilon_{\theta} \\ (\%) \end{gathered}$ | $\begin{gathered} \varepsilon_{\mathrm{v}} \\ (\%) \end{gathered}$ | $\begin{aligned} & \gamma_{\theta z} \\ & (\%) \end{aligned}$ | $\begin{gathered} \sigma_{z} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \sigma_{\mathrm{r}} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \sigma_{\theta} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \tau_{\text {өz }} \\ (\mathrm{kPa}) \end{gathered}$ | b | $\begin{gathered} \alpha \\ \left(^{\circ}\right) \end{gathered}$ | $\begin{gathered} \varphi \\ \left({ }^{\circ}\right) \end{gathered}$ |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 1 | 0.000 | 0.000 | 0.000 | 0.000 | 0.000 | 101.588 | 101.304 | 101.542 | 1.680 | 0.423 | 65.691 | 0.948 |
| 2 | -0.001 | -0.005 | 0.015 | 0.010 | 0.034 | 93.181 | 96.239 | 114.491 | 12.093 | 0.264 | 65.691 | 8.929 |
| 3 | -0.004 | -0.008 | 0.026 | 0.013 | 0.058 | 90.350 | 94.556 | 119.143 | 15.532 | 0.259 | 66.413 | 11.665 |
| 4 | -0.010 | -0.010 | 0.040 | 0.019 | 0.089 | 87.669 | 92.921 | 123.313 | 18.983 | 0.259 | 66.597 | 14.290 |
| 5 | -0.020 | -0.015 | 0.061 | 0.026 | 0.139 | 84.509 | 90.799 | 129.254 | 22.531 | 0.247 | 67.399 | 17.282 |
| 6 | -0.024 | -0.016 | 0.068 | 0.029 | 0.159 | 83.580 | 90.209 | 130.873 | 23.731 | 0.246 | 67.449 | 18.206 |
| 7 | -0.029 | -0.016 | 0.076 | 0.031 | 0.178 | 82.714 | 89.697 | 131.950 | 24.916 | 0.248 | 67.328 | 19.047 |
| 8 | -0.032 | -0.018 | 0.082 | 0.032 | 0.194 | 82.308 | 89.362 | 133.185 | 25.545 | 0.245 | 67.441 | 19.548 |
| 9 | -0.038 | -0.020 | 0.091 | 0.033 | 0.218 | 81.353 | 89.015 | 134.010 | 26.755 | 0.251 | 67.270 | 20.401 |
| 10 | -0.043 | -0.021 | 0.099 | 0.036 | 0.245 | 80.745 | 88.378 | 135.835 | 27.602 | 0.245 | 67.470 | 21.106 |
| 11 | -0.049 | -0.023 | 0.110 | 0.038 | 0.276 | 79.747 | 87.961 | 136.865 | 28.800 | 0.249 | 67.380 | 21.993 |
| 12 | -0.056 | -0.024 | 0.120 | 0.040 | 0.305 | 79.163 | 87.461 | 138.052 | 29.673 | 0.247 | 67.389 | 22.638 |
| 13 | -0.059 | -0.024 | 0.126 | 0.042 | 0.318 | 78.835 | 87.301 | 138.639 | 29.981 | 0.247 | 67.462 | 22.918 |
| 14 | -0.063 | -0.026 | 0.131 | 0.042 | 0.339 | 78.545 | 87.164 | 139.225 | 30.890 | 0.249 | 67.242 | 23.431 |
| 15 | -0.070 | -0.026 | 0.142 | 0.045 | 0.368 | 78.025 | 86.639 | 140.141 | 31.366 | 0.246 | 67.358 | 23.870 |
| 16 | -0.073 | -0.027 | 0.146 | 0.046 | 0.374 | 77.545 | 86.496 | 140.823 | 30.934 | 0.244 | 67.823 | 23.908 |
| 17 | -0.077 | -0.029 | 0.151 | 0.045 | 0.400 | 77.445 | 86.329 | 141.424 | 32.352 | 0.246 | 67.339 | 24.566 |
| 18 | -0.085 | -0.030 | 0.162 | 0.047 | 0.426 | 77.039 | 85.987 | 142.253 | 32.136 | 0.242 | 67.708 | 24.679 |
| 19 | -0.091 | -0.030 | 0.169 | 0.048 | 0.452 | 76.324 | 85.856 | 142.448 | 33.128 | 0.249 | 67.471 | 25.333 |
| 20 | -0.095 | -0.031 | 0.176 | 0.049 | 0.466 | 76.174 | 85.659 | 142.951 | 32.806 | 0.245 | 67.752 | 25.292 |
| 21 | -0.110 | -0.033 | 0.194 | 0.050 | 0.529 | 75.575 | 85.178 | 144.410 | 34.412 | 0.245 | 67.502 | 26.262 |
| 22 | -0.129 | -0.036 | 0.217 | 0.052 | 0.601 | 74.899 | 84.644 | 145.704 | 35.557 | 0.244 | 67.438 | 27.059 |
| 23 | -0.161 | -0.041 | 0.253 | 0.051 | 0.722 | 73.574 | 83.944 | 147.315 | 37.007 | 0.246 | 67.447 | 28.229 |
| 24 | -0.187 | -0.043 | 0.281 | 0.051 | 0.816 | 73.761 | 83.768 | 148.166 | 38.571 | 0.246 | 66.983 | 28.878 |
| 25 | -0.227 | -0.050 | 0.323 | 0.047 | 0.922 | 71.785 | 82.840 | 150.703 | 39.494 | 0.246 | 67.488 | 30.123 |
| 26 | -0.272 | -0.056 | 0.367 | 0.039 | 1.033 | 70.599 | 82.190 | 151.780 | 40.579 | 0.247 | 67.504 | 31.078 |
| 27 | -0.325 | -0.063 | 0.418 | 0.030 | 1.155 | 70.021 | 81.685 | 153.432 | 41.967 | 0.246 | 67.411 | 31.975 |
| 28 | -0.386 | -0.070 | 0.473 | 0.017 | 1.283 | 69.113 | 81.183 | 154.779 | 43.073 | 0.247 | 67.420 | 32.863 |
| 29 | -0.443 | -0.078 | 0.524 | 0.003 | 1.406 | 68.572 | 80.710 | 156.236 | 44.019 | 0.245 | 67.439 | 33.549 |
| 30 | -0.547 | -0.092 | 0.613 | -0.026 | 1.612 | 67.379 | 80.046 | 157.925 | 45.257 | 0.245 | 67.505 | 34.628 |
| 31 | -0.629 | -0.100 | 0.680 | -0.049 | 1.770 | 66.593 | 79.515 | 159.091 | 46.180 | 0.245 | 67.521 | 35.394 |
| 32 | -0.747 | -0.112 | 0.773 | -0.086 | 2.001 | 65.745 | 78.987 | 160.553 | 47.286 | 0.245 | 67.536 | 36.282 |
| 33 | -0.897 | -0.132 | 0.883 | -0.146 | 2.326 | 64.296 | 78.390 | 162.275 | 47.958 | 0.245 | 67.805 | 37.241 |

Torsion Shear Test on Fine Nevada Sand with a Target Void Ratio $=0.530\left(D_{r}=\mathbf{9 1 . 2 8 \%}\right)$

| Test No.: | $37^{*}$ |
| :--- | :--- |
| Test Date: | $5 / 10 / 11 \mathrm{~cm}$ |
| Initial Height, $\mathrm{h}_{\mathrm{i}}$ : | 39.96 cm |
| Initial Void Ratio, e: | 0.528 kPa |
| Final Mean Princpal Stress, $\sigma_{\mathrm{m}}:$ | 97.2 kPa |
| Max Friction Angle, $\phi:$ | 38.7 deg |
| b-value at failure: | 0.56 |
| Stress direction at failure, $\alpha:$ | 65.0 deg |$\quad$| Shear Band Notes |
| :--- |
| Point of Observation: |
|  |
| Inclination (from Vertical) |
| One SB around bottom half at $0^{\circ}, 10^{\circ}, 15^{\circ}$ |
|  |
| Failure Notes: |
| After SB developed, top cap slipped |


| Point (No.) | $\begin{gathered} \varepsilon_{\mathrm{z}} \\ (\%) \end{gathered}$ | $\begin{gathered} \varepsilon_{\mathrm{r}} \\ (\%) \end{gathered}$ | $\begin{gathered} \varepsilon_{\theta} \\ (\%) \end{gathered}$ | $\begin{gathered} \varepsilon_{\mathrm{v}} \\ (\%) \end{gathered}$ | $\begin{aligned} & \gamma_{\theta z} \\ & (\%) \end{aligned}$ | $\begin{gathered} \sigma_{z} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \sigma_{r} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \sigma_{\theta} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \tau_{\theta z} \\ (\mathrm{kPa}) \end{gathered}$ | b | $\begin{gathered} \alpha \\ \left({ }^{\circ}\right) \end{gathered}$ | $\begin{gathered} \varphi \\ \left({ }^{\circ}\right) \end{gathered}$ |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 1 | 0.000 | 0.000 | 0.000 | 0.000 | 0.000 | 89.243 | 101.364 | 101.377 | 1.723 | 0.980 | 52.926 | 3.794 |
| 2 | -0.020 | 0.009 | 0.037 | 0.026 | 0.108 | 70.755 | 101.381 | 118.715 | 18.153 | 0.610 | 63.563 | 18.511 |
| 3 | -0.033 | 0.015 | 0.049 | 0.031 | 0.143 | 68.559 | 101.366 | 120.531 | 20.010 | 0.604 | 63.799 | 20.298 |
| 4 | -0.043 | 0.019 | 0.059 | 0.035 | 0.170 | 67.481 | 101.392 | 122.271 | 21.446 | 0.594 | 64.028 | 21.512 |
| 5 | -0.059 | 0.025 | 0.072 | 0.038 | 0.203 | 65.530 | 101.367 | 123.732 | 22.988 | 0.591 | 64.153 | 23.073 |
| 6 | -0.094 | 0.039 | 0.099 | 0.044 | 0.269 | 63.284 | 101.381 | 126.105 | 25.590 | 0.583 | 64.585 | 25.331 |
| 7 | -0.118 | 0.049 | 0.116 | 0.047 | 0.310 | 62.077 | 101.396 | 127.263 | 26.668 | 0.580 | 64.645 | 26.413 |
| 8 | -0.145 | 0.059 | 0.134 | 0.048 | 0.351 | 61.038 | 101.373 | 128.475 | 27.555 | 0.576 | 64.628 | 27.358 |
| 9 | -0.164 | 0.067 | 0.145 | 0.048 | 0.382 | 60.343 | 101.367 | 129.035 | 28.182 | 0.575 | 64.685 | 27.982 |
| 10 | -0.203 | 0.082 | 0.169 | 0.048 | 0.441 | 59.472 | 101.368 | 130.473 | 29.147 | 0.570 | 64.694 | 28.924 |
| 11 | -0.287 | 0.113 | 0.215 | 0.042 | 0.559 | 57.818 | 101.355 | 132.129 | 30.608 | 0.566 | 64.740 | 30.455 |
| 12 | -0.337 | 0.132 | 0.242 | 0.038 | 0.630 | 56.934 | 101.381 | 132.913 | 31.322 | 0.566 | 64.752 | 31.245 |
| 13 | -0.386 | 0.150 | 0.267 | 0.030 | 0.701 | 56.191 | 101.373 | 133.633 | 32.054 | 0.564 | 64.809 | 31.980 |
| 14 | -0.443 | 0.170 | 0.295 | 0.022 | 0.780 | 55.586 | 101.377 | 134.316 | 32.693 | 0.563 | 64.855 | 32.610 |
| 15 | -0.485 | 0.185 | 0.314 | 0.015 | 0.835 | 55.021 | 101.367 | 134.623 | 33.099 | 0.563 | 64.874 | 33.088 |
| 16 | -0.561 | 0.210 | 0.349 | -0.002 | 0.940 | 54.311 | 101.365 | 135.673 | 33.779 | 0.560 | 64.852 | 33.824 |
| 17 | -0.616 | 0.229 | 0.372 | -0.015 | 1.020 | 53.707 | 101.378 | 135.808 | 34.159 | 0.562 | 64.882 | 34.304 |
| 18 | -0.762 | 0.277 | 0.432 | -0.054 | 1.253 | 52.773 | 101.361 | 136.915 | 35.117 | 0.559 | 64.926 | 35.296 |
| 19 | -0.863 | 0.309 | 0.470 | -0.083 | 1.487 | 52.601 | 101.408 | 137.530 | 35.459 | 0.557 | 64.931 | 35.587 |
| 20 | -0.917 | 0.330 | 0.489 | -0.098 | 1.704 | 52.259 | 101.358 | 137.509 | 35.560 | 0.558 | 64.918 | 35.806 |
| 21 | -0.981 | 0.354 | 0.509 | -0.117 | 1.903 | 51.847 | 101.369 | 138.083 | 36.041 | 0.557 | 64.946 | 36.282 |
| 22 | -1.022 | 0.371 | 0.524 | -0.128 | 2.029 | 51.646 | 101.372 | 137.927 | 36.360 | 0.558 | 65.063 | 36.529 |
| 23 | -1.080 | 0.391 | 0.545 | -0.144 | 2.205 | 51.330 | 101.371 | 138.573 | 36.653 | 0.556 | 65.019 | 36.874 |
| 24 | -1.150 | 0.412 | 0.567 | -0.171 | 2.463 | 50.989 | 101.388 | 139.159 | 37.101 | 0.555 | 65.042 | 37.304 |
| 25 | -1.558 | 0.576 | 0.696 | -0.286 | 3.675 | 49.400 | 101.383 | 140.301 | 38.073 | 0.555 | 64.976 | 38.689 |
| 26 | -1.578 | 0.584 | 0.702 | -0.291 | 3.733 | 49.731 | 101.363 | 140.382 | 38.296 | 0.553 | 65.097 | 38.626 |
| 27 | -1.579 | 0.584 | 0.702 | -0.292 | 3.739 | 49.709 | 101.371 | 140.367 | 38.266 | 0.553 | 65.085 | 38.622 |
| 28 | -1.581 | 0.586 | 0.703 | -0.293 | 3.746 | 49.749 | 101.360 | 140.500 | 38.300 | 0.553 | 65.083 | 38.625 |
| 29 | -1.588 | 0.588 | 0.705 | -0.295 | 3.766 | 49.787 | 101.374 | 140.531 | 38.326 | 0.552 | 65.094 | 38.619 |

## Torsion Shear Test on Fine Nevada Sand with a Target Void Ratio $=\mathbf{0 . 5 3 0}\left(\mathrm{D}_{\mathrm{r}}=\mathbf{9 1 . 2 8 \%}\right)$

| Test No.: | 38 |
| :--- | :---: |
| Test Date: | $10 / 11 / 11$ |
| Initial Height, $\mathrm{h}_{\mathrm{i}}:$ | 39.96 cm |
| Initial Void Ratio, e: | 0.528 |
| Final Mean Princpal Stress, $\sigma_{\mathrm{m}}:$ | 101.5 kPa |
| Max Friction Angle, $\phi:$ | 31.5 deg |
| b-value at failure: | 0.75 |
| Stress direction at failure, $\alpha:$ | 65.1 deg |


| Shear Band Notes <br> Point of Observation: |
| :--- |
| Inclination (from Vertical) |
|  |
| Failure Notes: |
| top cap SB (deep) |


| Point (No.) | $\begin{gathered} \varepsilon_{\mathrm{z}} \\ (\%) \\ \hline \end{gathered}$ | $\begin{array}{r} \varepsilon_{\mathrm{r}} \\ (\%) \end{array}$ | $\begin{gathered} \varepsilon_{\theta} \\ (\%) \end{gathered}$ | $\varepsilon_{\mathrm{v}}$ <br> (\%) | $\begin{aligned} & \gamma_{\theta z} \\ & (\%) \end{aligned}$ | $\begin{gathered} \sigma_{z} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \sigma_{\mathrm{r}} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \sigma_{\theta} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \tau_{\theta z} \\ (\mathrm{kPa}) \end{gathered}$ | b | $\alpha$ <br> ${ }^{\circ}$ ) | $\begin{gathered} \varphi \\ \left({ }^{\circ}\right) \\ \hline \end{gathered}$ |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 1 | 0.000 | 0.000 | 0.000 | 0.000 | 0.000 | 101.198 | 101.394 | 101.053 | 1.300 | 0.603 | 52.926 | 0.738 |
| 2 | -0.014 | 0.020 | 0.015 | 0.022 | 0.052 | 86.608 | 107.195 | 111.688 | 14.227 | 0.712 | 63.563 | 11.027 |
| 3 | -0.017 | 0.024 | 0.018 | 0.025 | 0.066 | 84.289 | 108.077 | 112.851 | 16.090 | 0.721 | 63.799 | 12.607 |
| 4 | -0.026 | 0.031 | 0.026 | 0.031 | 0.093 | 80.302 | 109.526 | 114.717 | 18.474 | 0.738 | 64.028 | 15.005 |
| 5 | -0.038 | 0.039 | 0.034 | 0.035 | 0.122 | 77.436 | 110.487 | 116.339 | 20.183 | 0.743 | 64.153 | 16.817 |
| 6 | -0.045 | 0.044 | 0.039 | 0.039 | 0.140 | 76.880 | 110.912 | 116.935 | 21.490 | 0.738 | 64.585 | 17.646 |
| 7 | -0.053 | 0.050 | 0.045 | 0.042 | 0.157 | 75.047 | 111.462 | 118.045 | 22.132 | 0.742 | 64.645 | 18.638 |
| 8 | -0.063 | 0.057 | 0.052 | 0.045 | 0.181 | 73.471 | 112.035 | 118.695 | 23.124 | 0.747 | 64.628 | 19.670 |
| 9 | -0.073 | 0.063 | 0.058 | 0.048 | 0.203 | 72.518 | 112.475 | 119.237 | 23.979 | 0.748 | 64.685 | 20.436 |
| 10 | -0.086 | 0.071 | 0.066 | 0.051 | 0.229 | 71.298 | 112.915 | 120.136 | 24.827 | 0.747 | 64.694 | 21.335 |
| 11 | -0.096 | 0.077 | 0.072 | 0.053 | 0.249 | 70.354 | 113.250 | 120.781 | 25.542 | 0.746 | 64.740 | 22.058 |
| 12 | -0.113 | 0.088 | 0.081 | 0.056 | 0.283 | 69.294 | 113.655 | 121.139 | 26.468 | 0.749 | 64.752 | 22.898 |
| 13 | -0.119 | 0.091 | 0.085 | 0.057 | 0.295 | 69.040 | 113.830 | 121.577 | 26.724 | 0.747 | 64.809 | 23.152 |
| 14 | -0.133 | 0.100 | 0.093 | 0.059 | 0.320 | 68.273 | 114.117 | 121.880 | 27.200 | 0.749 | 64.855 | 23.681 |
| 15 | -0.151 | 0.110 | 0.103 | 0.061 | 0.354 | 67.496 | 114.282 | 122.594 | 27.919 | 0.745 | 64.874 | 24.373 |
| 16 | -0.162 | 0.117 | 0.107 | 0.062 | 0.373 | 67.074 | 114.498 | 122.854 | 28.257 | 0.746 | 64.852 | 24.713 |
| 17 | -0.173 | 0.124 | 0.114 | 0.066 | 0.393 | 66.675 | 114.655 | 123.107 | 28.432 | 0.747 | 64.882 | 24.969 |
| 18 | -0.185 | 0.130 | 0.120 | 0.065 | 0.415 | 66.302 | 114.837 | 123.338 | 28.830 | 0.747 | 64.926 | 25.320 |
| 19 | -0.201 | 0.138 | 0.127 | 0.064 | 0.444 | 65.677 | 115.026 | 123.455 | 29.224 | 0.749 | 64.931 | 25.756 |
| 20 | -0.216 | 0.146 | 0.135 | 0.065 | 0.472 | 65.292 | 115.227 | 123.768 | 29.648 | 0.749 | 64.918 | 26.135 |
| 21 | -0.230 | 0.153 | 0.142 | 0.065 | 0.497 | 64.902 | 115.387 | 123.995 | 29.937 | 0.749 | 64.946 | 26.445 |
| 22 | -0.243 | 0.161 | 0.148 | 0.066 | 0.519 | 64.516 | 115.544 | 124.317 | 30.233 | 0.748 | 65.063 | 26.767 |
| 23 | -0.277 | 0.178 | 0.164 | 0.064 | 0.579 | 63.613 | 115.844 | 124.696 | 30.851 | 0.750 | 65.019 | 27.456 |
| 24 | -0.329 | 0.205 | 0.187 | 0.063 | 0.669 | 62.631 | 116.229 | 125.411 | 31.685 | 0.749 | 65.042 | 28.319 |
| 25 | -0.410 | 0.243 | 0.221 | 0.055 | 0.804 | 61.367 | 116.761 | 126.416 | 32.794 | 0.748 | 64.976 | 29.467 |
| 26 | -0.488 | 0.280 | 0.252 | 0.044 | 0.934 | 60.156 | 117.162 | 126.955 | 33.559 | 0.749 | 65.097 | 30.403 |
| 27 | -0.614 | 0.335 | 0.299 | 0.021 | 1.154 | 59.009 | 117.689 | 127.942 | 34.663 | 0.748 | 65.085 | 31.530 |

## Torsion Shear Test on Fine Nevada Sand with a Target Void Ratio $=\mathbf{0 . 5 3 0}\left(\mathrm{D}_{\mathrm{r}}=\mathbf{9 1 . 2 8 \%}\right)$

| Test No.: | $39^{*}$ |
| :--- | :---: |
| Test Date: | $5 / 12 / 11$ |
| Initial Height, $\mathrm{h}_{\mathrm{i}}$ : | 40.06 cm |
| Initial Void Ratio, e: | 0.531 |
| Final Mean Princpal Stress, $\sigma_{\mathrm{m}}:$ | 97.1 kPa |
| Max Friction Angle, $\phi:$ | deg |
| b-value at failure: |  |
| Stress direction at failure, $\alpha:$ |  |


| Shear Band Notes |
| :--- |
| Point of Observation: |
| Inclination (from Vertical) |
|  |
| Failure Notes: |


| Point (No.) | $\begin{gathered} \varepsilon_{\mathrm{z}} \\ (\%) \end{gathered}$ | $\begin{gathered} \varepsilon_{\mathrm{r}} \\ (\%) \end{gathered}$ | $\begin{gathered} \varepsilon_{\theta} \\ (\%) \\ \hline \end{gathered}$ | $\begin{gathered} \varepsilon_{\mathrm{v}} \\ (\%) \end{gathered}$ | $\begin{gathered} \gamma_{\theta z} \\ (\%) \\ \hline \end{gathered}$ | $\begin{gathered} \sigma_{z} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \sigma_{\mathrm{r}} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \sigma_{\theta} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \tau_{\theta z} \\ (\mathrm{kPa}) \end{gathered}$ | b | $\begin{gathered} \alpha \\ \left({ }^{\circ}\right) \\ \hline \end{gathered}$ | $\begin{gathered} \varphi \\ \left(^{\circ}\right) \end{gathered}$ |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 1 | 0.000 | 0.000 | 0.000 | 0.000 | 0.000 | 88.455 | 101.542 | 101.656 | -0.140 | 0.991 | 44.393 | 3.982 |
| 2 | -0.077 | 0.033 | 0.075 | 0.031 | 0.169 | 63.609 | 110.740 | 116.793 | 20.385 | 0.806 | 63.737 | 21.806 |
| 3 | -0.121 | 0.051 | 0.106 | 0.035 | 0.242 | 60.423 | 111.870 | 118.543 | 22.540 | 0.804 | 63.899 | 24.267 |
| 4 | -0.175 | 0.073 | 0.139 | 0.036 | 0.324 | 57.626 | 112.974 | 120.152 | 24.420 | 0.804 | 63.997 | 26.506 |
| 5 | -0.228 | 0.093 | 0.169 | 0.034 | 0.396 | 55.628 | 113.694 | 121.399 | 25.669 | 0.802 | 63.987 | 28.119 |
| 6 | -0.229 | 0.093 | 0.170 | 0.034 | 0.397 | 55.606 | 113.696 | 121.340 | 25.680 | 0.802 | 64.001 | 28.128 |
| 7 | -0.287 | 0.117 | 0.198 | 0.028 | 0.477 | 54.869 | 114.119 | 121.881 | 27.487 | 0.797 | 64.682 | 29.366 |
| 8 | -0.418 | 0.165 | 0.258 | 0.005 | 0.645 | 52.002 | 115.180 | 124.029 | 29.144 | 0.793 | 64.491 | 31.761 |
| 9 | -0.535 | 0.208 | 0.305 | -0.021 | 0.798 | 50.332 | 115.728 | 124.861 | 29.874 | 0.795 | 64.359 | 33.041 |
| 10 | -0.605 | 0.235 | 0.331 | -0.039 | 0.895 | 49.851 | 115.868 | 125.055 | 29.988 | 0.795 | 64.286 | 33.364 |
| 11 | -0.657 | 0.255 | 0.351 | -0.051 | 0.975 | 49.611 | 115.885 | 124.878 | 30.410 | 0.796 | 64.470 | 33.682 |
| 12 | -0.715 | 0.278 | 0.369 | -0.068 | 1.106 | 49.321 | 116.167 | 125.524 | 30.609 | 0.794 | 64.389 | 33.990 |
| 13 | -0.763 | 0.296 | 0.384 | -0.083 | 1.231 | 49.171 | 116.165 | 125.295 | 30.583 | 0.796 | 64.391 | 34.037 |
| 14 | -0.800 | 0.311 | 0.394 | -0.095 | 1.358 | 48.837 | 116.248 | 125.458 | 30.933 | 0.796 | 64.459 | 34.404 |
| 15 | -0.840 | 0.329 | 0.406 | -0.105 | 1.476 | 48.518 | 116.387 | 125.804 | 31.180 | 0.794 | 64.450 | 34.728 |
| 16 | -0.876 | 0.344 | 0.416 | -0.117 | 1.587 | 48.352 | 116.521 | 126.093 | 31.328 | 0.793 | 64.434 | 34.916 |
| 17 | -0.894 | 0.352 | 0.421 | -0.122 | 1.642 | 48.397 | 116.526 | 126.113 | 31.346 | 0.793 | 64.446 | 34.902 |
| 18 | -0.896 | 0.353 | 0.421 | -0.122 | 1.647 | 48.419 | 116.532 | 126.151 | 31.380 | 0.793 | 64.459 | 34.910 |
| 19 | -0.897 | 0.354 | 0.421 | -0.123 | 1.652 | 48.407 | 116.527 | 126.152 | 31.398 | 0.793 | 64.464 | 34.926 |
| 20 | -0.899 | 0.354 | 0.422 | -0.123 | 1.655 | 48.452 | 116.523 | 126.209 | 31.439 | 0.792 | 64.480 | 34.927 |
| 21 | -0.901 | 0.356 | 0.423 | -0.122 | 1.661 | 48.454 | 116.529 | 126.249 | 31.449 | 0.792 | 64.478 | 34.934 |
| 22 | -0.901 | 0.356 | 0.422 | -0.123 | 1.667 | 48.457 | 116.530 | 126.247 | 31.390 | 0.792 | 64.453 | 34.903 |
| 23 | -0.903 | 0.356 | 0.423 | -0.123 | 1.671 | 48.442 | 116.540 | 126.195 | 31.464 | 0.792 | 64.492 | 34.944 |
| 24 | -0.904 | 0.358 | 0.423 | -0.124 | 1.675 | 48.486 | 116.556 | 126.262 | 31.445 | 0.792 | 64.480 | 34.916 |
| 25 | -0.947 | 0.385 | 0.434 | -0.128 | 1.800 | 48.160 | 116.624 | 126.466 | 31.490 | 0.792 | 64.405 | 35.132 |
| 26 | -1.009 | 0.490 | 0.441 | -0.078 | 1.999 | 47.965 | 116.596 | 126.534 | 31.665 | 0.791 | 64.435 | 35.332 |
| 27 | -1.094 | 0.654 | 0.448 | 0.008 | 2.277 | 47.923 | 116.587 | 126.668 | 31.615 | 0.790 | 64.382 | 35.340 |
| 28 | -1.222 | 1.772 | 0.363 | 0.913 | 2.793 | 47.800 | 116.469 | 125.778 | 31.038 | 0.798 | 64.261 | 35.044 |

Torsion Shear Test on Fine Nevada Sand with a Target Void Ratio $=\mathbf{0 . 5 3 0}\left(\mathrm{D}_{\mathrm{r}}=\mathbf{9 1 . 2 8 \%}\right)$

| Test No.: | $40^{*}$ |
| :--- | :---: |
| Test Date: | $9 / 19 / 10$ |
| Initial Height, $\mathrm{h}_{\mathrm{i}}:$ | 40.03 cm |
| Initial Void Ratio, e: | 0.541 |
| Final Mean Princpal Stress, $\sigma_{\mathrm{m}}:$ | 97.35 kPa |
| Max Friction Angle, $\phi:$ | 34.9 deg |
| b-value at failure: | 0.80 |
| Stress direction at failure, $\alpha:$ | 61.9 deg |


| Shear Band Notes |
| :--- |
| Point of Observation: |
|  |
| Inclination (from Vertical) |
| $1 \mathrm{SB} @ 9.5^{\circ}, 10^{\circ}, 11^{\circ}, 13^{\circ}$ |
| Failure Notes: |
| One SB wrapping around specimen |


| Point (No.) | $\begin{gathered} \varepsilon_{\mathrm{z}} \\ (\%) \end{gathered}$ | $\begin{gathered} \varepsilon_{\mathrm{r}} \\ (\%) \\ \hline \end{gathered}$ | $\begin{gathered} \varepsilon_{\theta} \\ (\%) \\ \hline \end{gathered}$ | $\begin{gathered} \varepsilon_{\mathrm{v}} \\ (\%) \\ \hline \end{gathered}$ | $\begin{aligned} & \gamma_{\theta z} \\ & (\%) \end{aligned}$ | $\begin{gathered} \sigma_{z} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \sigma_{\mathrm{r}} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \sigma_{\theta} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \tau_{\theta z} \\ (\mathrm{kPa}) \end{gathered}$ | b | $\begin{gathered} \alpha \\ \left({ }^{\circ}\right) \\ \hline \end{gathered}$ | $\begin{gathered} \varphi \\ \left({ }^{\circ}\right) \\ \hline \end{gathered}$ |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 1 | 0.000 | 0.000 | 0.000 | 0.000 | 0.000 | 86.365 | 103.111 | 102.776 | 1.060 | 0.976 | 48.608 | 5.111 |
| 2 | -0.004 | 0.005 | 0.003 | 0.004 | 0.088 | 77.454 | 109.120 | 106.439 | 10.121 | 0.850 | 61.294 | 11.621 |
| 3 | -0.011 | 0.014 | 0.005 | 0.008 | 0.105 | 72.721 | 111.764 | 108.078 | 10.862 | 0.854 | 59.546 | 14.016 |
| 4 | -0.015 | 0.015 | 0.008 | 0.008 | 0.115 | 71.610 | 112.446 | 108.091 | 11.140 | 0.845 | 59.308 | 14.640 |
| 5 | -0.017 | 0.016 | 0.008 | 0.008 | 0.125 | 70.838 | 112.715 | 107.689 | 11.510 | 0.833 | 59.399 | 15.091 |
| 6 | -0.018 | 0.017 | 0.009 | 0.008 | 0.130 | 70.300 | 112.991 | 107.965 | 11.788 | 0.835 | 59.454 | 15.431 |
| 7 | -0.023 | 0.024 | 0.011 | 0.012 | 0.158 | 68.924 | 113.918 | 108.557 | 12.991 | 0.830 | 60.003 | 16.509 |
| 8 | -0.032 | 0.026 | 0.015 | 0.010 | 0.193 | 66.810 | 115.083 | 109.052 | 14.380 | 0.822 | 60.393 | 17.994 |
| 9 | -0.041 | 0.031 | 0.019 | 0.010 | 0.227 | 65.172 | 116.386 | 109.685 | 15.492 | 0.816 | 60.587 | 19.250 |
| 10 | -0.050 | 0.040 | 0.021 | 0.012 | 0.263 | 63.265 | 117.452 | 110.415 | 16.975 | 0.814 | 61.034 | 20.722 |
| 11 | -0.061 | 0.046 | 0.027 | 0.012 | 0.301 | 61.240 | 118.724 | 111.352 | 18.273 | 0.814 | 61.223 | 22.241 |
| 12 | -0.074 | 0.053 | 0.033 | 0.012 | 0.338 | 60.433 | 119.375 | 111.668 | 19.387 | 0.808 | 61.669 | 23.103 |
| 13 | -0.085 | 0.059 | 0.036 | 0.010 | 0.358 | 58.560 | 120.440 | 112.398 | 19.667 | 0.812 | 61.221 | 24.182 |
| 14 | -0.111 | 0.072 | 0.044 | 0.006 | 0.414 | 57.208 | 121.368 | 112.990 | 20.969 | 0.809 | 61.586 | 25.419 |
| 15 | -0.125 | 0.082 | 0.047 | 0.004 | 0.444 | 56.226 | 122.019 | 113.305 | 21.713 | 0.807 | 61.713 | 26.249 |
| 16 | -0.144 | 0.091 | 0.055 | 0.002 | 0.475 | 54.750 | 122.740 | 113.691 | 21.997 | 0.808 | 61.453 | 27.146 |
| 17 | -0.174 | 0.106 | 0.062 | -0.006 | 0.532 | 53.461 | 123.736 | 114.351 | 23.578 | 0.804 | 61.931 | 28.529 |
| 18 | -0.203 | 0.121 | 0.070 | -0.012 | 0.573 | 52.021 | 124.418 | 114.362 | 24.049 | 0.801 | 61.799 | 29.513 |
| 19 | -0.222 | 0.129 | 0.076 | -0.018 | 0.605 | 51.286 | 124.901 | 114.845 | 24.610 | 0.802 | 61.884 | 30.173 |
| 20 | -0.045 | 0.034 | 0.021 | 0.010 | 0.244 | 64.341 | 116.869 | 110.168 | 16.233 | 0.817 | 60.860 | 19.924 |
| 21 | -0.249 | 0.143 | 0.079 | -0.028 | 0.649 | 50.581 | 125.484 | 115.091 | 24.986 | 0.801 | 61.854 | 30.758 |
| 22 | -0.269 | 0.153 | 0.081 | -0.035 | 0.682 | 49.993 | 125.828 | 115.435 | 25.360 | 0.802 | 61.888 | 31.258 |
| 23 | -0.345 | 0.185 | 0.100 | -0.059 | 0.785 | 48.303 | 126.787 | 115.721 | 26.306 | 0.798 | 61.918 | 32.660 |
| 24 | -0.369 | 0.198 | 0.104 | -0.067 | 0.821 | 47.987 | 127.101 | 116.369 | 26.683 | 0.802 | 62.001 | 33.028 |
| 25 | -0.426 | 0.223 | 0.124 | -0.079 | 0.898 | 47.750 | 127.614 | 116.547 | 26.886 | 0.800 | 61.976 | 33.300 |
| 26 | -0.486 | 0.252 | 0.136 | -0.098 | 0.975 | 46.835 | 127.821 | 116.752 | 27.365 | 0.801 | 62.025 | 34.031 |
| 27 | -0.544 | 0.281 | 0.135 | -0.128 | 1.056 | 46.056 | 128.266 | 116.858 | 27.558 | 0.800 | 61.920 | 34.595 |
| 28 | -0.649 | 0.342 | 0.140 | -0.167 | 1.324 | 45.833 | 128.649 | 117.570 | 27.764 | 0.804 | 61.921 | 34.852 |

## Torsion Shear Test on Fine Nevada Sand with a Target Void Ratio $=\mathbf{0 . 5 3 0}\left(\mathrm{D}_{\mathrm{r}}=\mathbf{9 1 . 2 8 \%}\right)$

| Test No.: 41 |  |  |  |  |  |  |  | Shear Band Notes |  |  |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| Test Date: |  |  |  |  | 10/21/11 |  |  | Point of Observation: |  |  |  |  |
| Initial Height, $\mathrm{h}_{\mathrm{i}}$ : |  |  |  |  |  |  |  |  |  |  |  |  |
| Initial Void Ratio, e: |  |  |  |  | 0.523 |  |  | Inclination (from Vertical) |  |  |  |  |
| Final Mean Princpal Stress, $\sigma_{\mathrm{m}}$ : |  |  |  |  | 101.8 |  |  | $2 @ 16,21,21.5,22.5,23.5,32$ |  |  |  |  |
| Max Fric | ion Angle | $\text { , } \phi:$ |  |  |  |  |  |  |  |  |  |  |
| b-value at failure: <br> Stress direction at failure, $\alpha$ : |  |  |  |  |  |  |  | Failure Notes: |  |  |  |  |
|  |  |  |  |  |  |  |  | lower half bulged inwards, SBs crossed |  |  |  |  |
| Point | $\varepsilon_{\text {z }}$ | $\varepsilon_{\mathrm{r}}$ | $\varepsilon_{\text {ө }}$ | $\varepsilon_{\mathrm{v}}$ | $\gamma_{\theta z}$ | $\sigma_{z}$ | $\sigma_{\text {r }}$ | $\sigma_{\theta}$ | $\tau_{\text {өz }}$ | b | $\alpha$ | $\varphi$ |
| (No.) | (\%) | (\%) | (\%) | (\%) | (\%) | (kPa) | (kPa) | (kPa) | (kPa) |  | $\left({ }^{\circ}\right.$ ) | $\left({ }^{\circ}\right)$ |
| 1 | 0.000 | 0.000 | 0.000 | 0.000 | 0.000 | 101.597 | 101.317 | 101.512 | 0.000 | 0.698 | 90.000 | 0.024 |
| 2 | -0.049 | -0.025 | 0.101 | 0.027 | 0.000 | 80.939 | 80.978 | 141.832 | 0.000 | 0.001 | 90.000 | 15.863 |
| 3 | -0.099 | -0.061 | 0.201 | 0.041 | 0.000 | 73.698 | 73.669 | 156.439 | 0.000 | 0.000 | 90.000 | 21.071 |
| 4 | -0.155 | -0.099 | 0.302 | 0.048 | 0.000 | 69.790 | 69.955 | 163.668 | 0.000 | 0.002 | 90.000 | 23.711 |
| 5 | -0.213 | -0.138 | 0.402 | 0.051 | 0.000 | 67.511 | 67.620 | 168.335 | 0.000 | 0.001 | 90.000 | 25.309 |
| 6 | -0.274 | -0.182 | 0.504 | 0.047 | 0.000 | 65.390 | 65.726 | 171.955 | 0.000 | 0.003 | 90.000 | 26.679 |
| 7 | -0.342 | -0.229 | 0.611 | 0.040 | 0.000 | 64.132 | 64.339 | 174.813 | 0.000 | 0.002 | 90.000 | 27.594 |
| 8 | -0.407 | -0.277 | 0.712 | 0.028 | 0.000 | 62.989 | 63.166 | 177.245 | 0.000 | 0.002 | 90.000 | 28.399 |
| 9 | -0.476 | -0.329 | 0.816 | 0.011 | 0.000 | 61.913 | 62.146 | 179.177 | 0.000 | 0.002 | 90.000 | 29.104 |
| 10 | -0.538 | -0.374 | 0.908 | -0.004 | 0.000 | 61.127 | 61.424 | 180.423 | 0.000 | 0.002 | 90.000 | 29.596 |
| 11 | -0.670 | -0.479 | 1.104 | -0.045 | 0.000 | 59.709 | 59.953 | 183.459 | 0.000 | 0.002 | 90.000 | 30.591 |
| 12 | -0.777 | -0.566 | 1.258 | -0.085 | 0.000 | 58.558 | 59.052 | 184.777 | 0.000 | 0.004 | 90.000 | 31.245 |
| 13 | -0.821 | -0.598 | 1.319 | -0.099 | 0.000 | 58.588 | 58.884 | 185.606 | 0.000 | 0.002 | 90.000 | 31.343 |
| 14 | -0.896 | -0.653 | 1.425 | -0.124 | 0.000 | 58.232 | 58.378 | 186.668 | 0.000 | 0.001 | 90.000 | 31.631 |
| 15 | -0.967 | -0.707 | 1.525 | -0.150 | 0.000 | 57.700 | 57.918 | 187.562 | 0.000 | 0.002 | 90.000 | 31.971 |
| 16 | -1.033 | -0.761 | 1.618 | -0.176 | 0.000 | 57.225 | 57.390 | 188.525 | 0.000 | 0.001 | 90.000 | 32.295 |
| 17 | -1.112 | -0.828 | 1.730 | -0.210 | 0.000 | 56.621 | 56.999 | 189.037 | 0.000 | 0.003 | 90.000 | 32.617 |
| 18 | -1.182 | -0.885 | 1.827 | -0.240 | 0.000 | 56.469 | 56.748 | 189.779 | 0.000 | 0.002 | 90.000 | 32.777 |
| 19 | -1.266 | -0.953 | 1.945 | -0.275 | 0.000 | 56.175 | 56.427 | 190.431 | 0.000 | 0.002 | 90.000 | 32.985 |
| 20 | -1.347 | -1.019 | 2.049 | -0.317 | 0.000 | 55.291 | 56.058 | 190.759 | 0.000 | 0.006 | 90.000 | 33.406 |
| 21 | -1.432 | -1.102 | 2.179 | -0.356 | 0.000 | 55.340 | 55.653 | 191.383 | 0.000 | 0.002 | 90.000 | 33.463 |
| 22 | -1.503 | -1.154 | 2.271 | -0.386 | 0.000 | 55.089 | 55.629 | 191.748 | 0.000 | 0.004 | 90.000 | 33.617 |
| 23 | -1.576 | -1.185 | 2.349 | -0.412 | 0.000 | 56.092 | 55.403 | 194.419 | 0.000 | -0.005 | 90.000 | 33.517 |

## Torsion Shear Test on Fine Nevada Sand with a Target Void Ratio $=\mathbf{0 . 5 3 0}\left(\mathrm{D}_{\mathrm{r}}=\mathbf{9 1 . 2 8 \%}\right)$

| Test No.: | $42^{*}$ |
| :--- | :---: |
| Test Date: | $4 / 17 / 10$ |
| Initial Height, $\mathrm{h}_{\mathrm{i}}$ : | 39.96 cm |
| Initial Void Ratio, e: | 0.53 |
| Final Mean Princpal Stress, $\sigma_{\mathrm{m}}:$ | 95.9 kPa |
| Max Friction Angle, $\phi:$ | 45.5 deg |
| b-value at failure: | 0.32 |
| Stress direction at failure, $\alpha:$ | 90.0 deg |


| Shear Band Notes |
| :--- |
| Point of Observation: |
| Inclination (from Vertical) |
| $2 @ 24^{\circ}, 6 @ 25^{\circ}, 2 @ 26^{\circ}$ |
| Failure Notes: |
| SB crossed along entire specimen |


| Point <br> (No.) | $\begin{gathered} \varepsilon_{\mathrm{z}} \\ (\%) \end{gathered}$ | $\begin{gathered} \varepsilon_{\mathrm{r}} \\ (\%) \\ \hline \end{gathered}$ | $\begin{gathered} \varepsilon_{\theta} \\ (\%) \end{gathered}$ | $\begin{gathered} \varepsilon_{\mathrm{v}} \\ (\%) \\ \hline \end{gathered}$ | $\begin{gathered} \gamma_{\theta z} \\ (\%) \end{gathered}$ | $\begin{gathered} \sigma_{z} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \sigma_{\mathrm{r}} \\ (\mathrm{kPa}) \\ \hline \end{gathered}$ | $\begin{gathered} \sigma_{\theta} \\ (\mathrm{kPa}) \\ \hline \end{gathered}$ | $\begin{gathered} \boldsymbol{\tau}_{\theta z} \\ (\mathrm{kPa}) \\ \hline \end{gathered}$ | b | $\begin{gathered} \alpha \\ \left({ }^{\circ}\right) \end{gathered}$ | $\begin{gathered} \varphi \\ \left({ }^{\circ}\right) \\ \hline \end{gathered}$ |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 1 | 0.000 | 0.000 | 0.000 | 0.000 | 0.000 | 89.457 | 101.605 | 100.493 | 0.000 | 0.908 | 90.000 | 3.331 |
| 2 | -0.034 | -0.032 | 0.100 | 0.034 | 0.000 | 58.428 | 88.940 | 144.358 | 0.000 | 0.355 | 90.000 | 25.071 |
| 3 | -0.071 | -0.034 | 0.150 | 0.045 | 0.000 | 53.931 | 87.123 | 150.616 | 0.000 | 0.343 | 90.000 | 28.209 |
| 4 | -0.113 | -0.037 | 0.201 | 0.051 | 0.000 | 50.925 | 85.919 | 154.749 | 0.000 | 0.337 | 90.000 | 30.318 |
| 5 | -0.158 | -0.036 | 0.252 | 0.058 | 0.000 | 48.539 | 84.971 | 157.993 | 0.000 | 0.333 | 90.000 | 32.003 |
| 6 | -0.207 | -0.035 | 0.302 | 0.060 | 0.000 | 46.771 | 84.275 | 160.362 | 0.000 | 0.330 | 90.000 | 33.257 |
| 7 | -0.255 | -0.037 | 0.350 | 0.059 | 0.000 | 45.176 | 83.650 | 162.482 | 0.000 | 0.328 | 90.000 | 34.395 |
| 8 | -0.313 | -0.040 | 0.406 | 0.053 | 0.000 | 43.525 | 83.007 | 164.653 | 0.000 | 0.326 | 90.000 | 35.581 |
| 9 | -0.381 | -0.033 | 0.464 | 0.049 | 0.000 | 42.745 | 82.721 | 165.600 | 0.000 | 0.325 | 90.000 | 36.134 |
| 10 | -0.423 | -0.032 | 0.502 | 0.048 | 0.000 | 42.176 | 82.511 | 166.295 | 0.000 | 0.325 | 90.000 | 36.539 |
| 11 | -0.486 | -0.031 | 0.557 | 0.040 | 0.000 | 40.867 | 82.010 | 167.975 | 0.000 | 0.324 | 90.000 | 37.491 |
| 12 | -0.545 | -0.030 | 0.608 | 0.033 | 0.000 | 40.196 | 81.667 | 169.113 | 0.000 | 0.322 | 90.000 | 38.019 |
| 13 | -0.601 | -0.033 | 0.656 | 0.022 | 0.000 | 39.586 | 81.430 | 169.887 | 0.000 | 0.321 | 90.000 | 38.465 |
| 14 | -0.671 | -0.033 | 0.714 | 0.010 | 0.000 | 38.698 | 81.151 | 170.797 | 0.000 | 0.321 | 90.000 | 39.092 |
| 15 | -0.733 | -0.032 | 0.764 | -0.001 | 0.000 | 38.144 | 80.968 | 171.381 | 0.000 | 0.321 | 90.000 | 39.487 |
| 16 | -0.790 | -0.031 | 0.808 | -0.013 | 0.000 | 37.708 | 80.804 | 171.905 | 0.000 | 0.321 | 90.000 | 39.808 |
| 17 | -0.860 | -0.029 | 0.863 | -0.026 | 0.000 | 37.100 | 80.603 | 172.551 | 0.000 | 0.321 | 90.000 | 40.247 |
| 18 | -0.932 | -0.028 | 0.918 | -0.042 | 0.000 | 36.576 | 80.399 | 173.200 | 0.000 | 0.321 | 90.000 | 40.639 |
| 19 | -0.989 | -0.028 | 0.961 | -0.056 | 0.000 | 36.150 | 80.249 | 173.675 | 0.000 | 0.321 | 90.000 | 40.952 |
| 20 | -1.065 | -0.028 | 1.018 | -0.074 | 0.000 | 35.522 | 80.068 | 174.246 | 0.000 | 0.321 | 90.000 | 41.401 |
| 21 | -1.128 | -0.027 | 1.064 | -0.091 | 0.000 | 35.096 | 79.885 | 174.830 | 0.000 | 0.321 | 90.000 | 41.731 |
| 22 | -1.184 | -0.026 | 1.107 | -0.104 | 0.000 | 34.822 | 79.752 | 175.247 | 0.000 | 0.320 | 90.000 | 41.949 |
| 23 | -1.349 | -0.026 | 1.227 | -0.149 | 0.000 | 33.785 | 79.478 | 176.071 | 0.000 | 0.321 | 90.000 | 42.689 |
| 24 | -1.478 | -0.028 | 1.317 | -0.189 | 0.000 | 33.355 | 79.264 | 176.716 | 0.000 | 0.320 | 90.000 | 43.035 |
| 25 | -1.544 | -0.025 | 1.363 | -0.207 | 0.000 | 32.968 | 79.208 | 176.868 | 0.000 | 0.321 | 90.000 | 43.296 |
| 26 | -1.695 | -0.027 | 1.468 | -0.254 | 0.000 | 32.275 | 79.016 | 177.419 | 0.000 | 0.322 | 90.000 | 43.802 |
| 27 | -1.818 | -0.027 | 1.551 | -0.293 | 0.000 | 31.821 | 78.832 | 177.967 | 0.000 | 0.322 | 90.000 | 44.158 |
| 28 | -1.903 | -0.024 | 1.609 | -0.318 | 0.000 | 31.338 | 78.698 | 178.370 | 0.000 | 0.322 | 90.000 | 44.518 |
| 29 | -2.132 | -0.035 | 1.761 | -0.406 | 0.000 | 30.722 | 78.502 | 178.881 | 0.000 | 0.322 | 90.000 | 44.979 |
| 30 | -2.257 | -0.040 | 1.843 | -0.454 | 0.000 | 30.343 | 78.440 | 179.006 | 0.000 | 0.324 | 90.000 | 45.244 |
| 31 | -2.408 | -0.040 | 1.938 | -0.510 | 0.000 | 30.057 | 78.369 | 179.155 | 0.000 | 0.324 | 90.000 | 45.452 |

Torsion Shear Test on Fine Nevada Sand with a Target Void Ratio $=\mathbf{0 . 5 3 0}\left(\mathrm{D}_{\mathrm{r}}=\mathbf{9 1 . 2 8 \%}\right)$

| Test No.: | $43^{*}$ |
| :--- | :---: |
| Test Date: | $5 / 13 / 11$ |
| Initial Height, $\mathrm{h}_{\mathrm{i}}:$ | 39.96 cm |
| Initial Void Ratio, e: | 0.520 |
| Final Mean Princpal Stress, $\sigma_{\mathrm{m}}:$ | 96.9 kPa |
| Max Friction Angle, $\phi:$ | 41.3 deg |
| b-value at failure: | 0.78 |
| Stress direction at failure, $\alpha:$ | 90.0 deg |


| Shear Band Notes |
| :--- |
| Point of Observation: |
| Inclination (from Vertical) |
| 1 SB @ $33^{\circ}, 32^{\circ}$, and $33^{\circ}$ |
| Failure Notes: |
| thick continuous band around specimen |


| Point (No.) | $\begin{gathered} \varepsilon_{\mathrm{z}} \\ (\%) \end{gathered}$ | $\begin{gathered} \varepsilon_{\mathrm{r}} \\ (\%) \end{gathered}$ | $\begin{gathered} \varepsilon_{\theta} \\ (\%) \end{gathered}$ | $\begin{gathered} \varepsilon_{\mathrm{v}} \\ (\%) \end{gathered}$ | $\begin{gathered} \gamma_{\theta z} \\ (\%) \end{gathered}$ | $\begin{gathered} \sigma_{z} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \sigma_{\mathrm{r}} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \sigma_{\theta} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \tau_{\text {өz }} \\ (\mathrm{kPa}) \end{gathered}$ | b | $\begin{gathered} \alpha \\ \left({ }^{\circ}\right) \end{gathered}$ | $\begin{gathered} \varphi \\ \left({ }^{\circ}\right) \end{gathered}$ |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 1 | 0.000 | 0.000 | 0.000 | 0.000 | 0.000 | 90.074 | 101.251 | 101.287 | 0.000 | 0.997 | 90.000 | 3.359 |
| 2 | -0.047 | 0.037 | 0.050 | 0.041 | 0.000 | 63.531 | 108.301 | 118.713 | 0.000 | 0.811 | 90.000 | 17.625 |
| 3 | -0.108 | 0.068 | 0.101 | 0.061 | 0.000 | 56.616 | 110.205 | 123.593 | 0.000 | 0.800 | 90.000 | 21.818 |
| 4 | -0.177 | 0.103 | 0.153 | 0.079 | 0.000 | 51.082 | 111.779 | 127.490 | 0.000 | 0.794 | 90.000 | 25.334 |
| 5 | -0.251 | 0.138 | 0.203 | 0.090 | 0.000 | 48.243 | 112.554 | 129.407 | 0.000 | 0.792 | 90.000 | 27.185 |
| 6 | -0.328 | 0.173 | 0.252 | 0.097 | 0.000 | 45.689 | 113.295 | 131.152 | 0.000 | 0.791 | 90.000 | 28.900 |
| 7 | -0.410 | 0.208 | 0.302 | 0.100 | 0.000 | 43.468 | 113.833 | 132.814 | 0.000 | 0.788 | 90.000 | 30.453 |
| 8 | -0.498 | 0.245 | 0.351 | 0.098 | 0.000 | 41.731 | 114.369 | 133.819 | 0.000 | 0.789 | 90.000 | 31.639 |
| 9 | -0.592 | 0.283 | 0.401 | 0.092 | 0.000 | 40.084 | 114.769 | 134.890 | 0.000 | 0.788 | 90.000 | 32.808 |
| 10 | -0.699 | 0.325 | 0.457 | 0.083 | 0.000 | 38.801 | 115.152 | 136.004 | 0.000 | 0.785 | 90.000 | 33.784 |
| 11 | -0.797 | 0.362 | 0.506 | 0.070 | 0.000 | 37.699 | 115.471 | 136.713 | 0.000 | 0.785 | 90.000 | 34.590 |
| 12 | -0.898 | 0.399 | 0.554 | 0.055 | 0.000 | 36.873 | 115.668 | 137.230 | 0.000 | 0.785 | 90.000 | 35.199 |
| 13 | -0.997 | 0.435 | 0.600 | 0.039 | 0.000 | 36.042 | 115.898 | 137.666 | 0.000 | 0.786 | 90.000 | 35.805 |
| 14 | -1.124 | 0.480 | 0.660 | 0.016 | 0.000 | 34.996 | 116.225 | 138.744 | 0.000 | 0.783 | 90.000 | 36.666 |
| 15 | -1.244 | 0.520 | 0.713 | -0.011 | 0.000 | 33.903 | 116.522 | 139.281 | 0.000 | 0.784 | 90.000 | 37.480 |
| 16 | -1.384 | 0.566 | 0.773 | -0.044 | 0.000 | 33.186 | 116.688 | 139.848 | 0.000 | 0.783 | 90.000 | 38.056 |
| 17 | -1.465 | 0.591 | 0.807 | -0.067 | 0.000 | 32.823 | 116.779 | 140.070 | 0.000 | 0.783 | 90.000 | 38.338 |
| 18 | -1.601 | 0.639 | 0.863 | -0.099 | 0.000 | 32.431 | 116.906 | 140.194 | 0.000 | 0.784 | 90.000 | 38.628 |
| 19 | -1.717 | 0.681 | 0.913 | -0.123 | 0.000 | 31.912 | 116.962 | 140.488 | 0.000 | 0.783 | 90.000 | 39.035 |
| 20 | -1.810 | 0.711 | 0.952 | -0.147 | 0.000 | 31.945 | 117.037 | 140.238 | 0.000 | 0.786 | 90.000 | 38.972 |
| 21 | -2.056 | 0.793 | 1.053 | -0.211 | 0.000 | 30.852 | 117.284 | 141.366 | 0.000 | 0.782 | 90.000 | 39.920 |
| 22 | -2.186 | 0.836 | 1.105 | -0.246 | 0.000 | 30.459 | 117.381 | 141.601 | 0.000 | 0.782 | 90.000 | 40.237 |
| 23 | -2.314 | 0.882 | 1.156 | -0.276 | 0.000 | 30.108 | 117.513 | 141.781 | 0.000 | 0.783 | 90.000 | 40.518 |
| 24 | -2.396 | 0.908 | 1.187 | -0.301 | 0.000 | 29.995 | 117.577 | 142.362 | 0.000 | 0.779 | 90.000 | 40.688 |
| 25 | -2.514 | 0.941 | 1.232 | -0.341 | 0.000 | 29.850 | 117.686 | 143.068 | 0.000 | 0.776 | 90.000 | 40.900 |
| 26 | -2.731 | 1.006 | 1.315 | -0.410 | 0.000 | 29.389 | 117.840 | 143.407 | 0.000 | 0.776 | 90.000 | 41.288 |

Torsion Shear Test on Fine Nevada Sand with a Target Void Ratio = 0.530 ( $\mathrm{D}_{\mathrm{r}}=\mathbf{9 1 . 2 8 \%}$ )

| Test No.: | $44^{*}$ |
| :--- | :---: |
| Test Date: | $5 / 16 / 11$ |
| Initial Height, $\mathrm{h}_{\mathrm{i}}:$ | 39.96 cm |
| Initial Void Ratio, e: | 0.510 |
| Final Mean Princpal Stress, $\sigma_{\mathrm{m}}:$ | 96.9 kPa |
| Max Friction Angle, $\phi:$ | 37.9 deg |
| b-value at failure: | 0.99 |
| Stress direction at failure, $\alpha:$ | 90.0 deg |


| Shear Band Notes <br> Point of Observation: |
| :--- |
| Inclination (from Vertical) |
| $0^{\circ}$ |
| Failure Notes: |
| Deep trough SB along top cap |


| Point (No.) | $\begin{gathered} \varepsilon_{\mathrm{z}} \\ (\%) \end{gathered}$ | $\begin{gathered} \varepsilon_{\mathrm{r}} \\ (\%) \end{gathered}$ | $\begin{gathered} \varepsilon_{\theta} \\ (\%) \\ \hline \end{gathered}$ | $\begin{gathered} \varepsilon_{\mathrm{v}} \\ (\%) \end{gathered}$ | $\begin{aligned} & \gamma_{\theta z} \\ & (\%) \end{aligned}$ | $\begin{gathered} \sigma_{z} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \sigma_{\mathrm{r}} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \sigma_{\theta} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \tau_{\theta z} \\ (\mathrm{kPa}) \end{gathered}$ | b | $\begin{gathered} \alpha \\ \left({ }^{\circ}\right) \end{gathered}$ | $\begin{gathered} \varphi \\ \left({ }^{\circ}\right) \end{gathered}$ |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 1 | 0.000 | 0.000 | 0.000 | 0.000 | 0.000 | 88.412 | 101.393 | 101.525 | 0.000 | 0.989 | 90.000 | 3.959 |
| 2 | -0.019 | 0.023 | 0.021 | 0.025 | 0.000 | 62.027 | 114.095 | 114.086 | 0.000 | 1.000 | 90.000 | 17.193 |
| 3 | -0.034 | 0.033 | 0.030 | 0.029 | 0.000 | 58.361 | 115.970 | 116.143 | 0.000 | 0.997 | 90.000 | 19.337 |
| 4 | -0.051 | 0.043 | 0.040 | 0.032 | 0.000 | 55.380 | 117.361 | 116.887 | 0.000 | 1.008 | 90.000 | 20.919 |
| 5 | -0.069 | 0.054 | 0.051 | 0.036 | 0.000 | 53.089 | 118.551 | 118.549 | 0.000 | 1.000 | 90.000 | 22.420 |
| 6 | -0.088 | 0.066 | 0.061 | 0.038 | 0.000 | 51.091 | 119.530 | 119.681 | 0.000 | 0.998 | 90.000 | 23.681 |
| 7 | -0.104 | 0.075 | 0.070 | 0.041 | 0.000 | 49.305 | 120.322 | 120.493 | 0.000 | 0.998 | 90.000 | 24.787 |
| 8 | -0.125 | 0.086 | 0.081 | 0.042 | 0.000 | 47.433 | 121.243 | 121.300 | 0.000 | 0.999 | 90.000 | 25.962 |
| 9 | -0.144 | 0.097 | 0.090 | 0.043 | 0.000 | 46.331 | 121.802 | 122.027 | 0.000 | 0.997 | 90.000 | 26.719 |
| 10 | -0.169 | 0.110 | 0.102 | 0.042 | 0.000 | 45.430 | 122.239 | 122.296 | 0.000 | 0.999 | 90.000 | 27.276 |
| 11 | -0.189 | 0.121 | 0.111 | 0.043 | 0.000 | 44.666 | 122.580 | 122.677 | 0.000 | 0.999 | 90.000 | 27.786 |
| 12 | -0.211 | 0.132 | 0.120 | 0.042 | 0.000 | 43.805 | 123.004 | 122.885 | 0.000 | 1.002 | 90.000 | 28.321 |
| 13 | -0.235 | 0.145 | 0.131 | 0.042 | 0.000 | 43.147 | 123.362 | 123.552 | 0.000 | 0.998 | 90.000 | 28.838 |
| 14 | -0.285 | 0.170 | 0.153 | 0.038 | 0.000 | 41.719 | 123.974 | 123.934 | 0.000 | 1.000 | 90.000 | 29.756 |
| 15 | -0.337 | 0.197 | 0.174 | 0.034 | 0.000 | 40.487 | 124.630 | 124.518 | 0.000 | 1.001 | 90.000 | 30.614 |
| 16 | -0.377 | 0.216 | 0.190 | 0.030 | 0.000 | 39.568 | 125.020 | 125.116 | 0.000 | 0.999 | 90.000 | 31.296 |
| 17 | -0.438 | 0.245 | 0.213 | 0.020 | 0.000 | 38.466 | 125.552 | 125.556 | 0.000 | 1.000 | 90.000 | 32.071 |
| 18 | -0.490 | 0.269 | 0.233 | 0.011 | 0.000 | 37.655 | 125.942 | 126.093 | 0.000 | 0.998 | 90.000 | 32.690 |
| 19 | -0.575 | 0.309 | 0.263 | -0.003 | 0.000 | 36.794 | 126.384 | 126.495 | 0.000 | 0.999 | 90.000 | 33.322 |
| 20 | -0.648 | 0.340 | 0.291 | -0.017 | 0.000 | 35.982 | 126.848 | 127.103 | 0.000 | 0.997 | 90.000 | 33.968 |
| 21 | -0.709 | 0.366 | 0.311 | -0.031 | 0.000 | 35.224 | 127.152 | 127.312 | 0.000 | 0.998 | 90.000 | 34.511 |
| 22 | -0.806 | 0.410 | 0.343 | -0.053 | 0.000 | 34.460 | 127.504 | 127.786 | 0.000 | 0.997 | 90.000 | 35.115 |
| 23 | -0.910 | 0.457 | 0.377 | -0.076 | 0.000 | 33.509 | 127.936 | 127.833 | 0.000 | 1.001 | 90.000 | 35.776 |
| 24 | -0.986 | 0.490 | 0.403 | -0.093 | 0.000 | 32.665 | 128.392 | 128.535 | 0.000 | 0.999 | 90.000 | 36.493 |
| 25 | -1.020 | 0.504 | 0.412 | -0.103 | 0.000 | 32.252 | 128.571 | 128.673 | 0.000 | 0.999 | 90.000 | 36.811 |
| 26 | -1.127 | 0.552 | 0.442 | -0.133 | 0.000 | 31.682 | 128.941 | 129.211 | 0.000 | 0.997 | 90.000 | 37.314 |
| 27 | -1.348 | 0.654 | 0.492 | -0.203 | 0.000 | 31.116 | 129.368 | 130.161 | 0.000 | 0.992 | 90.000 | 37.889 |

True Triaxial Test on Fine Nevada Sand with a Target Void Ratio $=0.530\left(D_{r}=91.28 \%\right)$

| Test No.: | 1 |
| :--- | :--- |
| Test Date: | $3 / 21 / 11$ |
| Sector: | I |
| Initial Void Ratio, e: | 0.522 |
| Final Mean Princpal Stress, $\sigma_{\mathrm{m}}:$ | 121.4 kPa |
| Max Friction Angle, $\phi:$ | 43.0 deg |
| b-value at failure: | 0.00 |
| Stress direction at failure, $\alpha:$ | 0.0 deg |


| Shear Band Notes |
| :--- |
| Point of Observation: |
|  |
| Inclination (from Vertical): |
| $51^{\circ}, 58^{\circ}$ |
| Failure Notes: |


| Point (No.) | $\begin{gathered} \varepsilon_{\mathrm{z}} \\ (\%) \end{gathered}$ | $\begin{gathered} \varepsilon_{\mathrm{x}} \\ (\%) \\ \hline \end{gathered}$ | $\begin{array}{r} \varepsilon_{\mathrm{y}} \\ (\%) \\ \hline \end{array}$ | $\begin{gathered} \varepsilon_{\mathrm{v}} \\ (\%) \end{gathered}$ | $\begin{array}{r} \gamma_{\theta z} \\ (\%) \\ \hline \end{array}$ | $\begin{gathered} \sigma_{z} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \sigma_{x} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \sigma_{\mathrm{y}} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \tau_{\theta z} \\ (\mathrm{kPa}) \end{gathered}$ | b | $\begin{gathered} \alpha \\ \left(^{( }\right) \\ \hline \end{gathered}$ | $\begin{gathered} \varphi \\ \left({ }^{( }\right) \\ \hline \end{gathered}$ |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 1 | 0.000 | 0.000 | 0.000 | 0.000 | 0.000 | 51.627 | 50.000 | 50.000 | 0.000 | 0.000 | 0.000 | 0.917 |
| 2 | 0.014 | -0.005 | -0.005 | 0.004 | 0.000 | 63.297 | 50.000 | 50.000 | 0.000 | 0.000 | 0.000 | 6.740 |
| 3 | 0.041 | -0.014 | -0.014 | 0.013 | 0.000 | 76.423 | 50.000 | 50.000 | 0.000 | 0.000 | 0.000 | 12.064 |
| 4 | 0.126 | -0.045 | -0.045 | 0.037 | 0.000 | 104.831 | 50.000 | 50.000 | 0.000 | 0.000 | 0.000 | 20.741 |
| 5 | 0.201 | -0.075 | -0.075 | 0.050 | 0.000 | 123.372 | 50.000 | 50.000 | 0.000 | 0.000 | 0.000 | 25.037 |
| 6 | 0.421 | -0.181 | -0.181 | 0.059 | 0.000 | 162.830 | 50.000 | 50.000 | 0.000 | 0.000 | 0.000 | 32.015 |
| 7 | 0.620 | -0.296 | -0.296 | 0.028 | 0.000 | 186.502 | 50.000 | 50.000 | 0.000 | 0.000 | 0.000 | 35.252 |
| 8 | 0.809 | -0.415 | -0.415 | -0.020 | 0.000 | 202.092 | 50.000 | 50.000 | 0.000 | 0.000 | 0.000 | 37.108 |
| 9 | 1.053 | -0.582 | -0.582 | -0.111 | 0.000 | 215.641 | 50.000 | 50.000 | 0.000 | 0.000 | 0.000 | 38.576 |
| 10 | 1.211 | -0.693 | -0.693 | -0.176 | 0.000 | 222.463 | 50.000 | 50.000 | 0.000 | 0.000 | 0.000 | 39.270 |
| 11 | 1.421 | -0.844 | -0.844 | -0.268 | 0.000 | 230.904 | 50.000 | 50.000 | 0.000 | 0.000 | 0.000 | 40.091 |
| 12 | 1.637 | -1.007 | -1.007 | -0.376 | 0.000 | 237.817 | 50.000 | 50.000 | 0.000 | 0.000 | 0.000 | 40.735 |
| 13 | 1.811 | -1.140 | -1.140 | -0.470 | 0.000 | 243.016 | 50.000 | 50.000 | 0.000 | 0.000 | 0.000 | 41.202 |
| 14 | 2.007 | -1.292 | -1.292 | -0.577 | 0.000 | 248.111 | 50.000 | 50.000 | 0.000 | 0.000 | 0.000 | 41.648 |
| 15 | 2.237 | -1.476 | -1.476 | -0.716 | 0.000 | 250.914 | 50.000 | 50.000 | 0.000 | 0.000 | 0.000 | 41.888 |
| 16 | 2.421 | -1.618 | -1.618 | -0.816 | 0.000 | 254.573 | 50.000 | 50.000 | 0.000 | 0.000 | 0.000 | 42.196 |
| 17 | 2.611 | -1.772 | -1.772 | -0.933 | 0.000 | 256.402 | 50.000 | 50.000 | 0.000 | 0.000 | 0.000 | 42.348 |
| 18 | 2.806 | -1.931 | -1.931 | -1.055 | 0.000 | 258.896 | 50.000 | 50.000 | 0.000 | 0.000 | 0.000 | 42.553 |
| 19 | 3.053 | -2.130 | -2.130 | -1.208 | 0.000 | 259.450 | 50.000 | 50.000 | 0.000 | 0.000 | 0.000 | 42.598 |
| 20 | 3.211 | -2.255 | -2.255 | -1.299 | 0.000 | 261.012 | 50.000 | 50.000 | 0.000 | 0.000 | 0.000 | 42.724 |
| 21 | 3.421 | -2.423 | -2.423 | -1.425 | 0.000 | 263.069 | 50.000 | 50.000 | 0.000 | 0.000 | 0.000 | 42.889 |
| 22 | 3.632 | -2.594 | -2.594 | -1.556 | 0.000 | 263.024 | 50.000 | 50.000 | 0.000 | 0.000 | 0.000 | 42.886 |
| 23 | 3.842 | -2.764 | -2.764 | -1.686 | 0.000 | 264.355 | 50.000 | 50.000 | 0.000 | 0.000 | 0.000 | 42.991 |
| 24 | 4.135 | -3.000 | -3.000 | -1.865 | 0.000 | 262.299 | 50.000 | 50.000 | 0.000 | 0.000 | 0.000 | 42.828 |
| 25 | 4.263 | -3.101 | -3.101 | -1.939 | 0.000 | 263.231 | 50.000 | 50.000 | 0.000 | 0.000 | 0.000 | 42.902 |
| 26 | 4.421 | -3.223 | -3.223 | -2.026 | 0.000 | 263.039 | 50.000 | 50.000 | 0.000 | 0.000 | 0.000 | 42.887 |
| 27 | 4.668 | -3.417 | -3.417 | -2.165 | 0.000 | 261.858 | 50.000 | 50.000 | 0.000 | 0.000 | 0.000 | 42.792 |
| 28 | 4.881 | -3.584 | -3.584 | -2.287 | 0.000 | 261.135 | 50.000 | 50.000 | 0.000 | 0.000 | 0.000 | 42.734 |
| 29 | 5.026 | -3.696 | -3.696 | -2.365 | 0.000 | 258.281 | 50.000 | 50.000 | 0.000 | 0.000 | 0.000 | 42.502 |
| 30 | 5.250 | -3.866 | -3.866 | -2.482 | 0.000 | 257.552 | 50.000 | 50.000 | 0.000 | 0.000 | 0.000 | 42.443 |

True Triaxial Test on Fine Nevada Sand with a Target Void Ratio $=\mathbf{0 . 5 3 0}\left(\mathrm{D}_{\mathrm{r}}=\mathbf{9 1 . 2 8 \%}\right)$

| Test No.: | 2 |
| :--- | :--- |
| Test Date: | $3 / 22 / 11$ |
| Sector: | I |
| Initial Void Ratio, e: | 0.527 |
| Final Mean Princpal Stress, $\sigma_{\mathrm{m}}:$ | 158.3 kPa |
| Max Friction Angle, $\phi:$ | 46.3 deg |
| b-value at failure: | 0.24 |
| Stress direction at failure, $\alpha:$ | 0.0 deg |$\quad$| Shear Band Notes |  |
| :--- | :--- |
| Point of Observation: |  |
|  | 18 |
| Inclination (from Vertical): |  |
| $61^{\circ}, 62^{\circ}$ |  |
|  |  |
| Failure Notes: |  |


| Point <br> $($ No. $)$ | $\varepsilon_{\mathrm{z}}$ <br> $(\%)$ | $\varepsilon_{\mathrm{x}}$ <br> $(\%)$ | $\varepsilon_{\mathrm{y}}$ <br> $(\%)$ | $\varepsilon_{\mathrm{v}}$ <br> $(\%)$ | $\gamma_{\theta z}$ <br> $(\%)$ | $\sigma_{\mathrm{z}}$ <br> $(\mathrm{kPa})$ | $\sigma_{\mathrm{x}}$ <br> $(\mathrm{kPa})$ | $\sigma_{\mathrm{y}}$ <br> $(\mathrm{kPa})$ | $\tau_{\theta \mathrm{B}}$ <br> $(\mathrm{kPa})$ | b | $\alpha$ <br> $\left({ }^{\circ}\right)$ | $\varphi$ <br> $\left({ }^{\circ}\right)$ |
| ---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 1 | 0.000 | 0.000 | 0.000 | 0.000 | 0.000 | 51.617 | 50.404 | 50.000 | 0.000 | 0.250 | 0.000 | 0.911 |
| 2 | 0.000 | 0.002 | 0.002 | 0.003 | 0.000 | 60.631 | 52.497 | 50.000 | 0.000 | 0.235 | 0.000 | 5.514 |
| 3 | 0.023 | 0.000 | -0.008 | 0.015 | 0.000 | 72.888 | 55.563 | 50.000 | 0.000 | 0.243 | 0.000 | 10.734 |
| 4 | 0.049 | 0.000 | -0.024 | 0.026 | 0.000 | 85.862 | 58.809 | 50.000 | 0.000 | 0.246 | 0.000 | 15.305 |
| 5 | 0.208 | 0.206 | -0.350 | 0.065 | 0.000 | 136.227 | 71.397 | 50.000 | 0.000 | 0.248 | 0.000 | 27.582 |
| 6 | 0.406 | 0.253 | -0.584 | 0.075 | 0.000 | 174.880 | 81.108 | 50.000 | 0.000 | 0.249 | 0.000 | 33.733 |
| 7 | 0.602 | 0.253 | -0.799 | 0.056 | 0.000 | 204.021 | 88.480 | 50.000 | 0.000 | 0.250 | 0.000 | 37.325 |
| 8 | 0.807 | 0.253 | -1.047 | 0.013 | 0.000 | 230.115 | 95.120 | 50.000 | 0.000 | 0.251 | 0.000 | 40.016 |
| 9 | 1.009 | 0.253 | -1.305 | -0.043 | 0.000 | 251.423 | 100.580 | 50.000 | 0.000 | 0.251 | 0.000 | 41.931 |
| 10 | 1.201 | 0.253 | -1.565 | -0.112 | 0.000 | 269.414 | 105.220 | 50.000 | 0.000 | 0.252 | 0.000 | 43.387 |
| 11 | 1.422 | 0.253 | -1.877 | -0.202 | 0.000 | 286.130 | 109.573 | 50.000 | 0.000 | 0.252 | 0.000 | 44.628 |
| 12 | 1.628 | 0.253 | -2.186 | -0.305 | 0.000 | 294.965 | 111.938 | 50.000 | 0.000 | 0.253 | 0.000 | 45.244 |
| 13 | 1.813 | 0.253 | -2.466 | -0.400 | 0.000 | 304.174 | 113.556 | 50.000 | 0.000 | 0.250 | 0.000 | 45.861 |
| 14 | 2.082 | 0.253 | -2.881 | -0.546 | 0.000 | 309.824 | 113.473 | 50.000 | 0.000 | 0.244 | 0.000 | 46.227 |
| 15 | 2.205 | 0.253 | -3.073 | -0.615 | 0.000 | 311.095 | 113.435 | 50.000 | 0.000 | 0.243 | 0.000 | 46.308 |
| 16 | 2.408 | 0.253 | -3.405 | -0.744 | 0.000 | 310.257 | 113.363 | 50.000 | 0.000 | 0.243 | 0.000 | 46.255 |
| 17 | 2.623 | 0.253 | -3.762 | -0.886 | 0.000 | 298.910 | 112.225 | 50.000 | 0.000 | 0.250 | 0.000 | 45.512 |
| 18 | 2.810 | 0.253 | -4.052 | -0.989 | 0.000 | 273.898 | 106.351 | 50.000 | 0.000 | 0.252 | 0.000 | 43.730 |
| 19 | 3.012 | 0.253 | -4.319 | -1.054 | 0.000 | 249.415 | 98.391 | 50.000 | 0.000 | 0.243 | 0.000 | 41.760 |
| 20 | 3.239 | 0.253 | -4.582 | -1.091 | 0.000 | 239.581 | 96.796 | 50.000 | 0.000 | 0.247 | 0.000 | 40.895 |
| 21 | 3.452 | 0.253 | -4.823 | -1.119 | 0.000 | 233.627 | 95.734 | 50.000 | 0.000 | 0.249 | 0.000 | 40.348 |

## True Triaxial Test on Fine Nevada Sand with a Target Void Ratio $=\mathbf{0 . 5 3 0}\left(\mathrm{D}_{\mathrm{r}}=\mathbf{9 1 . 2 8 \%}\right)$

| Test No.: | 3 |
| :--- | :--- |
| Test Date: | $3 / 24 / 11$ |
| Sector: | I |
| Initial Void Ratio, e: | 0.527 |
| Final Mean Princpal Stress, $\sigma_{\mathrm{m}}:$ | 217.7 kPa |
| Max Friction Angle, $\phi:$ | 50.3 deg |
| b-value at failure: | 0.51 |
| Stress direction at failure, $\alpha$ : | 0.0 deg |


| Shear Band Notes |
| :--- |
| Point of Observation: |
| Inclination (from Vertical): |
| $61^{\circ}, 63^{\circ}$ |
| Failure Notes: |


| Point (No.) | $\begin{gathered} \varepsilon_{\mathrm{z}} \\ (\%) \end{gathered}$ | $\begin{gathered} \varepsilon_{\mathrm{x}} \\ (\%) \end{gathered}$ | $\begin{array}{r} \varepsilon_{y} \\ (\%) \end{array}$ | $\begin{gathered} \varepsilon_{\mathrm{v}} \\ (\%) \end{gathered}$ | $\begin{aligned} & \gamma_{\theta z} \\ & (\%) \end{aligned}$ | $\begin{gathered} \sigma_{z} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \sigma_{x} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \sigma_{\mathrm{y}} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \tau_{\theta z} \\ (\mathrm{kPa}) \end{gathered}$ | b | $\begin{gathered} \alpha \\ \left(^{\circ}\right) \\ \hline \end{gathered}$ | $\begin{gathered} \varphi \\ \left({ }^{\circ}\right) \\ \hline \end{gathered}$ |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 1 | 0.000 | 0.000 | 0.000 | 0.000 | 0.000 | 51.616 | 50.808 | 50.000 | 0.000 | 0.500 | 0.000 | 0.911 |
| 2 | 0.008 | 0.000 | -0.001 | 0.006 | 1.000 | 59.909 | 54.955 | 50.000 | 0.000 | 0.500 | 0.000 | 5.173 |
| 3 | 0.022 | 0.000 | -0.005 | 0.017 | 2.000 | 72.888 | 61.446 | 50.000 | 0.000 | 0.500 | 0.000 | 10.734 |
| 4 | 0.052 | 0.399 | -0.415 | 0.037 | 3.000 | 85.503 | 67.690 | 50.000 | 0.000 | 0.498 | 0.000 | 15.189 |
| 5 | 0.077 | 0.599 | -0.626 | 0.049 | 4.000 | 97.034 | 73.394 | 50.000 | 0.000 | 0.497 | 0.000 | 18.656 |
| 6 | 0.103 | 0.143 | -0.185 | 0.060 | 6.000 | 107.117 | 78.547 | 50.000 | 0.000 | 0.500 | 0.000 | 21.317 |
| 7 | 0.203 | 0.238 | -0.361 | 0.080 | 0.000 | 138.040 | 94.005 | 50.000 | 0.000 | 0.500 | 0.000 | 27.917 |
| 8 | 0.316 | 0.314 | -0.545 | 0.086 | 0.000 | 165.286 | 109.897 | 50.000 | 0.000 | 0.520 | 0.000 | 32.378 |
| 9 | 0.402 | 0.352 | -0.677 | 0.077 | 0.000 | 186.023 | 118.408 | 50.000 | 0.000 | 0.503 | 0.000 | 35.192 |
| 10 | 0.536 | 0.400 | -0.885 | 0.052 | 0.000 | 215.590 | 133.289 | 50.000 | 0.000 | 0.503 | 0.000 | 38.571 |
| 11 | 0.618 | 0.429 | -1.014 | 0.032 | 0.000 | 232.273 | 142.578 | 50.000 | 0.000 | 0.508 | 0.000 | 40.221 |
| 12 | 0.746 | 0.467 | -1.222 | -0.009 | 0.000 | 260.241 | 156.088 | 50.000 | 0.000 | 0.505 | 0.000 | 42.662 |
| 13 | 0.809 | 0.486 | -1.329 | -0.034 | 0.000 | 272.215 | 162.431 | 50.000 | 0.000 | 0.506 | 0.000 | 43.602 |
| 14 | 0.927 | 0.524 | -1.537 | -0.086 | 0.000 | 293.620 | 173.504 | 50.000 | 0.000 | 0.507 | 0.000 | 45.152 |
| 15 | 1.000 | 0.552 | -1.673 | -0.120 | 0.000 | 305.131 | 179.277 | 50.000 | 0.000 | 0.507 | 0.000 | 45.924 |
| 16 | 1.131 | 0.610 | -1.942 | -0.202 | 0.000 | 320.972 | 188.661 | 50.000 | 0.000 | 0.512 | 0.000 | 46.923 |
| 17 | 1.209 | 0.638 | -2.087 | -0.241 | 0.000 | 334.882 | 195.204 | 50.000 | 0.000 | 0.510 | 0.000 | 47.747 |
| 18 | 1.305 | 0.667 | -2.262 | -0.290 | 0.000 | 350.095 | 202.255 | 50.000 | 0.000 | 0.507 | 0.000 | 48.596 |
| 19 | 1.416 | 0.705 | -2.478 | -0.357 | 0.000 | 364.460 | 209.520 | 50.000 | 0.000 | 0.507 | 0.000 | 49.352 |
| 20 | 1.532 | 0.752 | -2.715 | -0.430 | 0.000 | 377.325 | 217.275 | 50.000 | 0.000 | 0.511 | 0.000 | 49.995 |
| 21 | 1.623 | 0.781 | -2.890 | -0.486 | 0.000 | 383.217 | 219.788 | 50.000 | 0.000 | 0.510 | 0.000 | 50.279 |
| 22 | 1.729 | 0.798 | -3.091 | -0.564 | 0.000 | 374.871 | 215.679 | 50.000 | 0.000 | 0.510 | 0.000 | 49.874 |
| 23 | 1.810 | 0.798 | -3.236 | -0.628 | 0.000 | 354.699 | 206.345 | 50.000 | 0.000 | 0.513 | 0.000 | 48.842 |
| 24 | 1.911 | 0.845 | -3.444 | -0.688 | 0.000 | 317.671 | 187.735 | 50.000 | 0.000 | 0.515 | 0.000 | 46.720 |
| 25 | 2.091 | 0.845 | -3.641 | -0.705 | 0.000 | 297.165 | 175.155 | 50.000 | 0.000 | 0.506 | 0.000 | 45.394 |
| 26 | 2.177 | 0.845 | -3.770 | -0.748 | 0.000 | 274.431 | 165.538 | 50.000 | 0.000 | 0.515 | 0.000 | 43.770 |
| 27 | 2.245 | 0.798 | -3.805 | -0.761 | 0.000 | 264.447 | 162.680 | 50.000 | 0.000 | 0.525 | 0.000 | 42.999 |
| 28 | 2.330 | 0.798 | -3.898 | -0.770 | 0.000 | 257.608 | 156.086 | 50.000 | 0.000 | 0.511 | 0.000 | 42.447 |
| 29 | 2.439 | 0.798 | -4.013 | -0.776 | 0.000 | 257.029 | 155.823 | 50.000 | 0.000 | 0.511 | 0.000 | 42.400 |
| 30 | 2.787 | 0.798 | -4.375 | -0.789 | 0.000 | 252.482 | 153.461 | 50.000 | 0.000 | 0.511 | 0.000 | 42.021 |
| 31 | 2.842 | 0.798 | -4.430 | -0.789 | 0.000 | 251.679 | 152.961 | 50.000 | 0.000 | 0.511 | 0.000 | 41.953 |
| 32 | 3.000 | 0.798 | -4.590 | -0.791 | 0.000 | 252.760 | 153.681 | 50.000 | 0.000 | 0.511 | 0.000 | 42.044 |

True Triaxial Test on Fine Nevada Sand with a Target Void Ratio $=\mathbf{0 . 5 3 0}\left(\mathrm{D}_{\mathrm{r}}=\mathbf{9 1 . 2 8 \%}\right)$

| Test No.: | 4 |
| :--- | :--- |
| Test Date: | $3 / 25 / 11$ |
| Sector: | I |
| Initial Void Ratio, e: | 0.528 |
| Final Mean Princpal Stress, $\sigma_{\mathrm{m}}:$ | 263.2 kPa |
| Max Friction Angle, $\phi:$ | 51.7 deg |
| b-value at failure: | 0.75 |
| Stress direction at failure, $\alpha:$ | 0.0 deg |


| Shear Band Notes |  |
| :--- | :--- |
| Point of Observation: | 24 |
| Inclination (from Vertical): |  |
| $64^{\circ}$ |  |
| Failure Notes: |  |


| Point <br> (No.) | $\begin{gathered} \varepsilon_{\mathrm{z}} \\ (\%) \\ \hline \end{gathered}$ | $\begin{gathered} \varepsilon_{\mathrm{x}} \\ (\%) \\ \hline \end{gathered}$ | $\begin{array}{r} \varepsilon_{y} \\ (\%) \\ \hline \end{array}$ | $\begin{gathered} \varepsilon_{\mathrm{v}} \\ (\%) \\ \hline \end{gathered}$ | $\begin{gathered} \gamma_{\theta z} \\ (\%) \\ \hline \end{gathered}$ | $\begin{gathered} \sigma_{z} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \sigma_{\mathrm{x}} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \sigma_{y} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \tau_{\theta z} \\ (\mathrm{kPa}) \end{gathered}$ | b | $\begin{gathered} \alpha \\ \left({ }^{\circ}\right) \\ \hline \end{gathered}$ | $\begin{gathered} \varphi \\ \left(^{\circ}\right) \\ \hline \end{gathered}$ |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 1 | 0.000 | 0.000 | 0.000 | 0.000 | 0.000 | 50.000 | 50.161 | 50.000 | 0.000 | 0.750 | 0.000 | 0.000 |
| 2 | 0.002 | 0.000 | 0.011 | 0.013 | 1.000 | 59.914 | 57.449 | 50.000 | 0.000 | 0.751 | 0.000 | 5.175 |
| 3 | 0.009 | 0.000 | 0.012 | 0.022 | 2.000 | 71.096 | 65.851 | 50.000 | 0.000 | 0.751 | 0.000 | 10.033 |
| 4 | 0.049 | 0.000 | -0.004 | 0.045 | 3.000 | 87.684 | 78.318 | 50.000 | 0.000 | 0.751 | 0.000 | 15.884 |
| 5 | 0.111 | 0.452 | -0.490 | 0.073 | 0.000 | 107.862 | 93.496 | 50.000 | 0.000 | 0.752 | 0.000 | 21.502 |
| 6 | 0.213 | 0.703 | -0.812 | 0.103 | 0.000 | 135.925 | 114.636 | 50.000 | 0.000 | 0.752 | 0.000 | 27.526 |
| 7 | 0.312 | 0.879 | -1.076 | 0.114 | 0.000 | 161.051 | 133.608 | 50.000 | 0.000 | 0.753 | 0.000 | 31.748 |
| 8 | 0.406 | 1.029 | -1.334 | 0.101 | 0.000 | 185.389 | 152.038 | 50.000 | 0.000 | 0.754 | 0.000 | 35.112 |
| 9 | 0.507 | 1.180 | -1.607 | 0.080 | 0.000 | 210.733 | 171.281 | 50.000 | 0.000 | 0.755 | 0.000 | 38.058 |
| 10 | 0.605 | 1.406 | -1.959 | 0.052 | 0.000 | 234.199 | 189.169 | 50.000 | 0.000 | 0.756 | 0.000 | 40.401 |
| 11 | 0.709 | 1.531 | -2.229 | 0.011 | 0.000 | 257.254 | 206.786 | 50.000 | 0.000 | 0.756 | 0.000 | 42.418 |
| 12 | 0.825 | 1.632 | -2.501 | -0.045 | 0.000 | 283.389 | 226.842 | 50.000 | 0.000 | 0.758 | 0.000 | 44.431 |
| 13 | 0.900 | 1.682 | -2.666 | -0.084 | 0.000 | 300.290 | 239.851 | 50.000 | 0.000 | 0.759 | 0.000 | 45.604 |
| 14 | 1.019 | 1.782 | -2.962 | -0.161 | 0.000 | 323.730 | 256.414 | 50.000 | 0.000 | 0.754 | 0.000 | 47.090 |
| 15 | 1.101 | 1.908 | -3.234 | -0.226 | 0.000 | 336.177 | 264.456 | 50.000 | 0.000 | 0.749 | 0.000 | 47.821 |
| 16 | 1.203 | 2.008 | -3.518 | -0.308 | 0.000 | 353.824 | 280.540 | 50.000 | 0.000 | 0.759 | 0.000 | 48.796 |
| 17 | 1.322 | 2.134 | -3.862 | -0.407 | 0.000 | 373.437 | 295.284 | 50.000 | 0.000 | 0.758 | 0.000 | 49.804 |
| 18 | 1.400 | 2.234 | -4.105 | -0.471 | 0.000 | 387.532 | 304.667 | 50.000 | 0.000 | 0.754 | 0.000 | 50.484 |
| 19 | 1.526 | 2.334 | -4.457 | -0.596 | 0.000 | 399.823 | 315.390 | 50.000 | 0.000 | 0.759 | 0.000 | 51.050 |
| 20 | 1.619 | 2.410 | -4.719 | -0.691 | 0.000 | 409.103 | 320.751 | 50.000 | 0.000 | 0.754 | 0.000 | 51.461 |
| 21 | 1.737 | 2.514 | -5.010 | -0.759 | 0.000 | 414.832 | 324.772 | 50.000 | 0.000 | 0.753 | 0.000 | 51.709 |
| 22 | 1.807 | 2.614 | -5.288 | -0.867 | 0.000 | 410.724 | 320.751 | 50.000 | 0.000 | 0.751 | 0.000 | 51.531 |
| 23 | 1.909 | 2.660 | -5.527 | -0.957 | 0.000 | 384.446 | 301.986 | 50.000 | 0.000 | 0.753 | 0.000 | 50.338 |
| 24 | 2.123 | 2.660 | -5.852 | -1.069 | 0.000 | 309.558 | 245.691 | 50.000 | 0.000 | 0.754 | 0.000 | 46.210 |
| 25 | 2.224 | 2.660 | -5.984 | -1.099 | 0.000 | 287.210 | 229.607 | 50.000 | 0.000 | 0.757 | 0.000 | 44.704 |
| 26 | 2.327 | 2.614 | -6.047 | -1.106 | 0.000 | 278.923 | 221.565 | 50.000 | 0.000 | 0.749 | 0.000 | 44.105 |
| 27 | 2.461 | 2.614 | -6.182 | -1.108 | 0.000 | 274.794 | 218.884 | 50.000 | 0.000 | 0.751 | 0.000 | 43.798 |
| 28 | 2.533 | 2.614 | -6.257 | -1.110 | 0.000 | 274.986 | 218.884 | 50.000 | 0.000 | 0.751 | 0.000 | 43.812 |
| 29 | 2.643 | 2.614 | -6.369 | -1.112 | 0.000 | 275.793 | 218.884 | 50.000 | 0.000 | 0.748 | 0.000 | 43.873 |
| 30 | 2.705 | 2.614 | -6.431 | -1.112 | 0.000 | 277.052 | 218.884 | 50.000 | 0.000 | 0.744 | 0.000 | 43.967 |

## True Triaxial Test on Fine Nevada Sand with a Target Void Ratio $=\mathbf{0 . 5 3 0}\left(\mathrm{D}_{\mathrm{r}}=\mathbf{9 1 . 2 8 \%}\right)$

| Test No.: | 5 |
| :--- | :--- |
| Test Date: | $3 / 26 / 11$ |
| Sector: | I |
| Initial Void Ratio, e: | 0.530 |
| Final Mean Princpal Stress, $\sigma_{\mathrm{m}}:$ | 272.6 kPa |
| Max Friction Angle, $\phi:$ | 52.8 deg |
| b-value at failure: | 0.70 |
| Stress direction at failure, $\alpha:$ | 0.0 deg |


| Point (No.) | $\begin{gathered} \varepsilon_{\mathrm{z}} \\ (\%) \\ \hline \end{gathered}$ | $\begin{gathered} \varepsilon_{\mathrm{x}} \\ (\%) \\ \hline \end{gathered}$ | $\begin{array}{r} \varepsilon_{\mathrm{y}} \\ (\%) \\ \hline \end{array}$ | $\begin{gathered} \varepsilon_{\mathrm{v}} \\ (\%) \end{gathered}$ | $\begin{array}{r} \gamma_{\theta z} \\ (\%) \\ \hline \end{array}$ | $\begin{gathered} \sigma_{z} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \sigma_{x} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \sigma_{\mathrm{y}} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \tau_{\theta z} \\ (\mathrm{kPa}) \end{gathered}$ | b | $\alpha$ <br> ${ }^{\circ}$ ) | $\begin{gathered} \varphi \\ \left({ }^{( }\right) \\ \hline \end{gathered}$ |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 1 | 0.000 | 0.000 | 0.000 | 0.000 | 0.000 | 51.632 | 51.632 | 50.000 | 0.000 | 1.000 | 0.000 | 0.920 |
| 2 | 0.007 | 0.000 | 0.006 | 0.013 | 0.000 | 59.275 | 59.276 | 50.000 | 0.000 | 1.000 | 0.000 | 4.869 |
| 3 | 0.033 | 0.000 | 0.006 | 0.039 | 0.000 | 73.106 | 73.114 | 50.000 | 0.000 | 1.000 | 0.000 | 10.818 |
| 4 | 0.065 | 0.721 | -0.716 | 0.069 | 0.000 | 86.939 | 86.696 | 50.000 | 0.000 | 0.993 | 0.000 | 15.649 |
| 5 | 0.101 | 0.541 | -0.551 | 0.091 | 0.000 | 102.945 | 102.677 | 50.000 | 0.000 | 0.995 | 0.000 | 20.253 |
| 6 | 0.206 | 0.811 | -0.913 | 0.104 | 0.000 | 141.806 | 143.401 | 50.000 | 0.000 | 1.017 | 0.000 | 28.597 |
| 7 | 0.316 | 1.081 | -1.332 | 0.065 | 0.000 | 188.153 | 191.245 | 50.000 | 0.000 | 1.022 | 0.000 | 35.458 |
| 8 | 0.402 | 1.284 | -1.677 | 0.009 | 0.000 | 224.954 | 229.345 | 50.000 | 0.000 | 1.025 | 0.000 | 39.517 |
| 9 | 0.511 | 1.555 | -2.152 | -0.087 | 0.000 | 260.450 | 265.706 | 50.000 | 0.000 | 1.025 | 0.000 | 42.679 |
| 10 | 0.622 | 1.791 | -2.649 | -0.237 | 0.000 | 289.179 | 295.181 | 50.000 | 0.000 | 1.025 | 0.000 | 44.843 |
| 11 | 0.702 | 1.926 | -2.953 | -0.326 | 0.000 | 305.744 | 311.731 | 50.000 | 0.000 | 1.023 | 0.000 | 45.964 |
| 12 | 0.804 | 2.095 | -3.412 | -0.512 | 0.000 | 312.241 | 320.002 | 50.000 | 0.000 | 1.030 | 0.000 | 46.381 |
| 13 | 0.905 | 2.265 | -3.738 | -0.569 | 0.000 | 347.405 | 327.115 | 50.000 | 0.000 | 0.932 | 0.000 | 48.449 |
| 14 | 1.002 | 2.311 | -3.916 | -0.604 | 0.000 | 374.276 | 327.244 | 50.000 | 0.000 | 0.855 | 0.000 | 49.845 |
| 15 | 1.138 | 2.365 | -4.171 | -0.669 | 0.000 | 397.287 | 326.822 | 50.000 | 0.000 | 0.797 | 0.000 | 50.935 |
| 16 | 1.223 | 2.410 | -4.345 | -0.712 | 0.000 | 408.298 | 326.720 | 50.000 | 0.000 | 0.772 | 0.000 | 51.426 |
| 17 | 1.328 | 2.458 | -4.555 | -0.769 | 0.000 | 420.245 | 323.024 | 50.000 | 0.000 | 0.737 | 0.000 | 51.938 |
| 18 | 1.404 | 2.458 | -4.665 | -0.803 | 0.000 | 428.410 | 323.674 | 50.000 | 0.000 | 0.723 | 0.000 | 52.277 |
| 19 | 1.501 | 2.458 | -4.814 | -0.855 | 0.000 | 435.005 | 324.942 | 50.000 | 0.000 | 0.714 | 0.000 | 52.544 |
| 20 | 1.614 | 2.505 | -5.035 | -0.916 | 0.000 | 441.831 | 325.998 | 50.000 | 0.000 | 0.704 | 0.000 | 52.814 |
| 21 | 1.726 | 2.505 | -5.234 | -1.003 | 0.000 | 438.252 | 322.984 | 50.000 | 0.000 | 0.703 | 0.000 | 52.673 |
| 22 | 1.830 | 2.505 | -5.403 | -1.068 | 0.000 | 428.434 | 320.325 | 50.000 | 0.000 | 0.714 | 0.000 | 52.278 |
| 23 | 1.918 | 2.505 | -5.547 | -1.125 | 0.000 | 409.524 | 304.627 | 50.000 | 0.000 | 0.708 | 0.000 | 51.479 |
| 24 | 2.075 | 2.552 | -5.808 | -1.181 | 0.000 | 369.264 | 273.393 | 50.000 | 0.000 | 0.700 | 0.000 | 49.595 |
| 25 | 2.156 | 2.552 | -5.910 | -1.203 | 0.000 | 352.382 | 262.987 | 50.000 | 0.000 | 0.704 | 0.000 | 48.719 |
| 26 | 2.268 | 2.552 | -6.049 | -1.229 | 0.000 | 336.138 | 252.581 | 50.000 | 0.000 | 0.708 | 0.000 | 47.819 |
| 27 | 2.368 | 2.552 | -6.158 | -1.237 | 0.000 | 329.865 | 247.394 | 50.000 | 0.000 | 0.705 | 0.000 | 47.455 |
| 28 | 2.469 | 2.552 | -6.267 | -1.246 | 0.000 | 325.711 | 242.208 | 50.000 | 0.000 | 0.697 | 0.000 | 47.209 |
| 29 | 2.586 | 2.552 | -6.389 | -1.250 | 0.000 | 323.289 | 242.220 | 50.000 | 0.000 | 0.703 | 0.000 | 47.064 |
| 30 | 2.725 | 2.552 | -6.531 | -1.255 | 0.000 | 322.572 | 242.235 | 50.000 | 0.000 | 0.705 | 0.000 | 47.020 |
| 31 | 2.870 | 2.552 | -6.685 | -1.263 | 0.000 | 322.179 | 242.245 | 50.000 | 0.000 | 0.706 | 0.000 | 46.997 |

## True Triaxial Test on Fine Nevada Sand with a Target Void Ratio $=\mathbf{0 . 5 3 0}\left(\mathrm{D}_{\mathrm{r}}=\mathbf{9 1 . 2 8 \%}\right)$

| Test No.: | 6 | Shear Band Notes |  |
| :---: | :---: | :---: | :---: |
| Test Date: | 3/28/11 | Point of Observation: | 22 |
| Sector: | I |  |  |
| Initial Void Ratio, e: | 0.530 | Inclination (from Vertical): |  |
| Final Mean Princpal Stress, $\sigma_{\mathrm{m}}$ : | 267.8 kPa | $62^{\circ}$ |  |
| Max Friction Angle, $\phi$ : | 52.3 deg |  |  |
| $b$-value at failure: | 0.72 | Failure Notes: |  |
| Stress direction at failure, $\alpha$ : | 0.0 deg |  |  |


| Point <br> $($ No. $)$ | $\varepsilon_{\mathrm{z}}$ <br> $(\%)$ | $\varepsilon_{\mathrm{x}}$ <br> $(\%)$ | $\varepsilon_{\mathrm{y}}$ <br> $(\%)$ | $\varepsilon_{\mathrm{v}}$ <br> $(\%)$ | $\gamma_{\theta z}$ <br> $(\%)$ | $\sigma_{\mathrm{z}}$ <br> $(\mathrm{kPa})$ | $\sigma_{\mathrm{x}}$ <br> $(\mathrm{kPa})$ | $\sigma_{\mathrm{y}}$ <br> $(\mathrm{kPa})$ | $\tau_{\theta z}$ <br> $(\mathrm{kPa})$ | b | $\alpha$ <br> $\left({ }^{\circ}\right)$ | $\varphi$ <br> $\left({ }^{\circ}\right)$ |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 1 | 0.000 | 0.000 | 0.000 | 0.000 | 0.000 | 51.631 | 51.631 | 50.000 | 0.000 | 1.000 | 0.000 | 0.920 |
| 2 | 0.005 | 0.000 | -0.003 | 0.002 | 1.000 | 54.178 | 54.178 | 50.000 | 0.000 | 1.000 | 0.000 | 2.298 |
| 3 | 0.016 | 0.856 | -0.007 | 0.865 | 3.000 | 65.820 | 65.822 | 50.000 | 0.000 | 1.000 | 0.000 | 7.851 |
| 4 | 0.029 | 4.174 | -0.003 | 4.200 | 6.000 | 75.280 | 75.288 | 50.000 | 0.000 | 1.000 | 0.000 | 11.642 |
| 5 | 0.044 | 4.709 | -0.348 | 4.405 | 9.000 | 87.288 | 87.175 | 50.000 | 0.000 | 0.997 | 0.000 | 15.760 |
| 6 | 0.063 | 5.030 | -0.888 | 4.205 | 12.000 | 100.023 | 99.613 | 50.000 | 0.000 | 0.992 | 0.000 | 19.477 |
| 7 | 0.079 | 5.244 | -1.187 | 4.136 | 14.000 | 108.756 | 108.111 | 50.000 | 0.000 | 0.989 | 0.000 | 21.722 |
| 8 | 0.106 | 0.200 | 0.076 | 0.382 | 0.000 | 121.482 | 121.211 | 50.000 | 0.000 | 1.000 | 0.000 | 24.636 |
| 9 | 0.218 | 0.426 | -0.560 | 0.085 | 0.000 | 165.056 | 164.954 | 50.000 | 0.000 | 1.000 | 0.000 | 32.344 |
| 10 | 0.309 | 0.748 | -0.986 | 0.071 | 0.000 | 195.113 | 195.367 | 50.000 | 0.000 | 1.000 | 0.000 | 36.301 |
| 11 | 0.405 | 0.962 | -1.319 | 0.048 | 0.000 | 229.805 | 228.965 | 50.000 | 0.000 | 1.000 | 0.000 | 39.987 |
| 12 | 0.517 | 1.283 | -1.806 | -0.007 | 0.000 | 259.990 | 259.628 | 50.000 | 0.000 | 1.000 | 0.000 | 42.641 |
| 13 | 0.615 | 1.497 | -2.203 | -0.091 | 0.000 | 288.232 | 288.105 | 50.000 | 0.000 | 1.000 | 0.000 | 44.777 |
| 14 | 0.706 | 1.711 | -2.560 | -0.143 | 0.000 | 298.429 | 297.328 | 50.000 | 0.000 | 1.000 | 0.000 | 45.479 |
| 15 | 0.817 | 1.925 | -3.011 | -0.269 | 0.000 | 314.859 | 315.942 | 50.000 | 0.000 | 1.000 | 0.000 | 46.546 |
| 16 | 0.901 | 2.139 | -3.417 | -0.378 | 0.000 | 329.917 | 326.106 | 50.000 | 0.000 | 0.999 | 0.000 | 47.458 |
| 17 | 1.017 | 2.353 | -3.839 | -0.469 | 0.000 | 367.845 | 325.009 | 50.000 | 0.000 | 0.997 | 0.000 | 49.524 |
| 18 | 1.101 | 2.460 | -4.078 | -0.517 | 0.000 | 391.889 | 325.255 | 50.000 | 0.000 | 0.995 | 0.000 | 50.687 |
| 19 | 1.215 | 2.550 | -4.248 | -0.483 | 0.000 | 415.370 | 323.402 | 50.000 | 0.000 | 0.993 | 0.000 | 51.732 |
| 20 | 1.307 | 2.567 | -4.499 | -0.625 | 0.000 | 429.566 | 323.781 | 50.000 | 0.000 | 0.992 | 0.000 | 52.324 |
| 21 | 1.455 | 2.674 | -4.898 | -0.768 | 0.000 | 415.344 | 324.226 | 50.000 | 0.000 | 0.990 | 0.000 | 51.730 |
| 22 | 1.675 | 2.674 | -5.278 | -0.929 | 0.000 | 339.025 | 260.409 | 50.000 | 0.000 | 0.989 | 0.000 | 47.983 |
| 23 | 1.841 | 2.674 | -5.491 | -0.977 | 0.000 | 306.144 | 234.843 | 50.000 | 0.000 | 1.003 | 0.000 | 45.990 |

True Triaxial Test on Fine Nevada Sand with a Target Void Ratio $=\mathbf{0 . 5 3 0}\left(\mathrm{D}_{\mathrm{r}}=\mathbf{9 1 . 2 8 \%}\right)$

| Test No.: | 7 |
| :--- | :--- |
| Test Date: | $3 / 30 / 11$ |
| Sector: | I |
| Initial Void Ratio, e: | 0.530 |
| Final Mean Princpal Stress, $\sigma_{\mathrm{m}}:$ | 170.0 kPa |
| Max Friction Angle, $\phi:$ | 50.6 deg |
| b-value at failure: | 1.00 |
| Stress direction at failure, $\alpha:$ | 0.0 deg |


| Point (No.) | $\begin{gathered} \varepsilon_{\mathrm{z}} \\ (\%) \end{gathered}$ | $\begin{gathered} \varepsilon_{\mathrm{x}} \\ (\%) \end{gathered}$ | $\begin{gathered} \varepsilon_{y} \\ (\%) \end{gathered}$ | $\begin{gathered} \varepsilon_{\mathrm{v}} \\ (\%) \end{gathered}$ | $\begin{aligned} & \gamma_{\theta z} \\ & (\%) \end{aligned}$ | $\begin{gathered} \sigma_{z} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \sigma_{\mathrm{x}} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \sigma_{\mathrm{y}} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \tau_{\text {өz }} \\ (\mathrm{kPa}) \end{gathered}$ | b | $\begin{gathered} \alpha \\ \left({ }^{\circ}\right) \\ \hline \end{gathered}$ | $\begin{gathered} \varphi \\ \left({ }^{\circ}\right) \\ \hline \end{gathered}$ |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 1 | 0.000 | 0.000 | 0.000 | 0.000 | 0.000 | 72.723 | 72.440 | 30.000 | 0.000 | 1.000 | 0.000 | 1.517 |
| 2 | 0.001 | 0.000 | -0.001 | 0.000 | 1.000 | 33.086 | 33.086 | 30.000 | 0.000 | 1.000 | 0.000 | 2.804 |
| 3 | 0.002 | 0.000 | 0.003 | 0.004 | 2.000 | 34.906 | 34.906 | 30.000 | 0.000 | 1.000 | 0.000 | 4.335 |
| 4 | 0.007 | 0.000 | 0.002 | 0.009 | 3.000 | 37.816 | 37.817 | 30.000 | 0.000 | 1.000 | 0.000 | 6.618 |
| 5 | 0.012 | 0.000 | -0.003 | 0.009 | 4.000 | 40.726 | 40.728 | 30.000 | 0.000 | 1.000 | 0.000 | 8.723 |
| 6 | 0.018 | 0.000 | -0.007 | 0.011 | 5.000 | 43.272 | 43.275 | 30.000 | 0.000 | 1.000 | 0.000 | 10.436 |
| 7 | 0.026 | 0.000 | -0.009 | 0.017 | 6.000 | 46.183 | 46.187 | 30.000 | 0.000 | 1.000 | 0.000 | 12.264 |
| 8 | 0.035 | 0.000 | -0.009 | 0.026 | 7.000 | 49.820 | 49.827 | 30.000 | 0.000 | 1.000 | 0.000 | 14.378 |
| 9 | 0.049 | 0.000 | -0.017 | 0.033 | 8.000 | 54.911 | 54.924 | 30.000 | 0.000 | 1.000 | 0.000 | 17.061 |
| 10 | 0.062 | 0.247 | -0.268 | 0.041 | 9.000 | 58.548 | 58.495 | 30.000 | 0.000 | 0.998 | 0.000 | 18.808 |
| 11 | 0.073 | 0.448 | -0.475 | 0.046 | 10.000 | 61.456 | 61.339 | 30.000 | 0.000 | 0.996 | 0.000 | 20.118 |
| 12 | 0.081 | 0.548 | -0.577 | 0.052 | 11.000 | 64.002 | 63.844 | 30.000 | 0.000 | 0.995 | 0.000 | 21.206 |
| 13 | 0.090 | 0.648 | -0.684 | 0.054 | 12.000 | 66.183 | 65.981 | 30.000 | 0.000 | 0.994 | 0.000 | 22.098 |
| 14 | 0.097 | 0.702 | -0.742 | 0.056 | 13.000 | 68.000 | 67.770 | 30.000 | 0.000 | 0.994 | 0.000 | 22.815 |
| 15 | 0.114 | 0.777 | -0.830 | 0.061 | 1.000 | 99.204 | 99.520 | 30.000 | 0.000 | 0.993 | 0.000 | 24.576 |
| 16 | 0.221 | 1.110 | -1.266 | 0.065 | 2.000 | 116.180 | 117.121 | 30.000 | 0.000 | 1.005 | 0.000 | 32.386 |
| 17 | 0.312 | 1.150 | -1.611 | -0.149 | 3.000 | 142.846 | 143.812 | 30.000 | 0.000 | 1.011 | 0.000 | 36.125 |
| 18 | 0.435 | 1.609 | -2.066 | -0.022 | 4.000 | 153.934 | 155.446 | 30.000 | 0.000 | 1.009 | 0.000 | 40.758 |
| 19 | 0.504 | 1.771 | -2.353 | -0.078 | 5.000 | 167.426 | 168.811 | 30.000 | 0.000 | 1.012 | 0.000 | 42.361 |
| 20 | 0.607 | 2.125 | -2.916 | -0.184 | 6.000 | 187.397 | 189.381 | 30.000 | 0.000 | 1.010 | 0.000 | 44.114 |
| 21 | 0.701 | 2.332 | -3.302 | -0.269 | 7.000 | 196.377 | 198.841 | 30.000 | 0.000 | 1.013 | 0.000 | 46.386 |
| 22 | 0.803 | 2.787 | -4.032 | -0.443 | 8.000 | 217.168 | 218.971 | 30.000 | 0.000 | 1.015 | 0.000 | 47.303 |
| 23 | 0.924 | 3.041 | -4.546 | -0.582 | 9.000 | 222.145 | 223.330 | 30.000 | 0.000 | 1.010 | 0.000 | 49.222 |
| 24 | 1.024 | 3.348 | -5.101 | -0.729 | 10.000 | 226.399 | 231.608 | 30.000 | 0.000 | 1.006 | 0.000 | 49.644 |
| 25 | 1.105 | 3.595 | -5.586 | -0.885 | 11.000 | 228.045 | 230.613 | 30.000 | 0.000 | 1.027 | 0.000 | 49.995 |
| 26 | 1.215 | 3.936 | -6.227 | -1.076 | 12.000 | 234.420 | 236.540 | 30.000 | 0.000 | 1.013 | 0.000 | 50.128 |
| 27 | 1.325 | 4.076 | -6.590 | -1.189 | 13.000 | 230.536 | 234.786 | 30.000 | 0.000 | 1.010 | 0.000 | 50.632 |
| 28 | 1.507 | 4.576 | -7.702 | -1.619 | 14.000 | 206.690 | 208.343 | 30.000 | 0.000 | 1.021 | 0.000 | 50.327 |
| 29 | 1.613 | 5.010 | -8.490 | -1.866 | 0.000 | 165.056 | 164.954 | 50.000 | 0.000 | 1.009 | 0.000 | 48.288 |

True Triaxial Test on Fine Nevada Sand with a Target Void Ratio $=0.530\left(D_{r}=91.28 \%\right)$

| Test No.: | 8 |
| :--- | :--- |
| Test Date: | $3 / 12 / 11$ |
| Sector: | II |
| Initial Void Ratio, e: | 0.527 |
| Final Mean Princpal Stress, $\sigma_{\mathrm{m}}:$ | 158.3 kPa |
| Max Friction Angle, $\phi:$ | 46.3 deg |
| b-value at failure: | 0.25 |
| Stress direction at failure, $\alpha:$ | 90.0 deg |


| Shear Band Notes |  |
| :--- | :--- |
| Point of Observation: | 25 |
|  |  |
| Inclination (from Vertical): |  |
| $65^{\circ}, 64^{\circ}$ |  |
| Failure Notes: |  |


| Point (No.) | $\begin{gathered} \varepsilon_{\mathrm{z}} \\ (\%) \end{gathered}$ | $\begin{gathered} \varepsilon_{\mathrm{x}} \\ (\%) \end{gathered}$ | $\begin{array}{r} \varepsilon_{\mathrm{y}} \\ (\%) \\ \hline \end{array}$ | $\begin{gathered} \varepsilon_{\mathrm{v}} \\ (\%) \\ \hline \end{gathered}$ | $\begin{gathered} \gamma_{\theta z} \\ (\%) \\ \hline \end{gathered}$ | $\begin{gathered} \sigma_{z} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \sigma_{x} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \sigma_{\mathrm{y}} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \tau_{\text {өz }} \\ (\mathrm{kPa}) \end{gathered}$ | b | $\alpha$ <br> ${ }^{\circ}$ ) | $\begin{gathered} \varphi \\ \left({ }^{\circ}\right) \\ \hline \end{gathered}$ |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 1 | 0.000 | 0.000 | 0.000 | 0.000 | 0.000 | 51.616 | 50.404 | 50.000 | 0.000 | 0.250 | 90.000 | 0.911 |
| 2 | 0.009 | 0.000 | -0.008 | 0.001 | 1.000 | 59.907 | 52.477 | 50.000 | 0.000 | 0.250 | 90.000 | 5.172 |
| 3 | 0.023 | 0.000 | -0.017 | 0.006 | 2.000 | 71.440 | 55.361 | 50.000 | 0.000 | 0.250 | 90.000 | 10.169 |
| 4 | 0.046 | 0.000 | -0.033 | 0.013 | 3.000 | 85.491 | 58.877 | 50.000 | 0.000 | 0.250 | 90.000 | 15.185 |
| 5 | 0.069 | 0.000 | -0.051 | 0.018 | 4.000 | 96.654 | 61.672 | 50.000 | 0.000 | 0.250 | 90.000 | 18.550 |
| 6 | 0.100 | 0.076 | -0.151 | 0.025 | 0.000 | 109.611 | 64.906 | 50.000 | 0.000 | 0.250 | 90.000 | 21.930 |
| 7 | 0.210 | 0.132 | -0.313 | 0.029 | 0.000 | 146.612 | 76.187 | 50.000 | 0.000 | 0.271 | 90.000 | 29.432 |
| 8 | 0.304 | 0.157 | -0.442 | 0.019 | 0.000 | 172.036 | 80.554 | 50.000 | 0.000 | 0.250 | 90.000 | 33.341 |
| 9 | 0.411 | 0.198 | -0.609 | 0.000 | 0.000 | 193.425 | 85.933 | 50.000 | 0.000 | 0.251 | 90.000 | 36.100 |
| 10 | 0.500 | 0.239 | -0.760 | -0.022 | 0.000 | 209.406 | 89.956 | 50.000 | 0.000 | 0.251 | 90.000 | 37.916 |
| 11 | 0.608 | 0.284 | -0.946 | -0.054 | 0.000 | 225.300 | 93.968 | 50.000 | 0.000 | 0.251 | 90.000 | 39.551 |
| 12 | 0.721 | 0.385 | -1.205 | -0.099 | 0.000 | 237.188 | 96.955 | 50.000 | 0.000 | 0.251 | 90.000 | 40.677 |
| 13 | 0.809 | 0.432 | -1.370 | -0.129 | 0.000 | 246.971 | 99.430 | 50.000 | 0.000 | 0.251 | 90.000 | 41.550 |
| 14 | 0.908 | 0.432 | -1.512 | -0.172 | 0.000 | 256.682 | 101.919 | 50.000 | 0.000 | 0.251 | 90.000 | 42.371 |
| 15 | 1.002 | 0.478 | -1.691 | -0.211 | 0.000 | 264.252 | 103.846 | 50.000 | 0.000 | 0.251 | 90.000 | 42.983 |
| 16 | 1.116 | 0.478 | -1.857 | -0.262 | 0.000 | 271.725 | 105.789 | 50.000 | 0.000 | 0.252 | 90.000 | 43.565 |
| 17 | 1.216 | 0.478 | -2.004 | -0.310 | 0.000 | 277.088 | 107.196 | 50.000 | 0.000 | 0.252 | 90.000 | 43.969 |
| 18 | 1.517 | 0.525 | -2.513 | -0.471 | 0.000 | 289.500 | 110.478 | 50.000 | 0.000 | 0.253 | 90.000 | 44.866 |
| 19 | 1.737 | 0.525 | -2.847 | -0.585 | 0.000 | 298.236 | 111.830 | 50.000 | 0.000 | 0.249 | 90.000 | 45.466 |
| 20 | 2.110 | 0.478 | -3.352 | -0.763 | 0.000 | 310.576 | 114.406 | 50.000 | 0.000 | 0.247 | 90.000 | 46.275 |
| 21 | 2.226 | 0.478 | -3.591 | -0.886 | 0.000 | 305.780 | 115.088 | 50.000 | 0.000 | 0.254 | 90.000 | 45.966 |
| 22 | 2.331 | 0.478 | -3.773 | -0.963 | 0.000 | 289.623 | 109.012 | 50.000 | 0.000 | 0.246 | 90.000 | 44.874 |
| 23 | 2.421 | 0.478 | -3.936 | -1.036 | 0.000 | 262.760 | 102.380 | 50.000 | 0.000 | 0.246 | 90.000 | 42.864 |
| 24 | 2.536 | 0.478 | -4.103 | -1.088 | 0.000 | 235.279 | 95.767 | 50.000 | 0.000 | 0.247 | 90.000 | 40.501 |
| 25 | 2.632 | 0.432 | -4.180 | -1.116 | 0.000 | 225.325 | 93.141 | 50.000 | 0.000 | 0.246 | 90.000 | 39.553 |
| 26 | 2.717 | 0.432 | -4.280 | -1.131 | 0.000 | 218.906 | 90.500 | 50.000 | 0.000 | 0.240 | 90.000 | 38.912 |
| 27 | 2.810 | 0.432 | -4.390 | -1.148 | 0.000 | 214.566 | 90.496 | 50.000 | 0.000 | 0.246 | 90.000 | 38.464 |
| 28 | 2.935 | 0.385 | -4.479 | -1.159 | 0.000 | 210.885 | 89.986 | 50.000 | 0.000 | 0.249 | 90.000 | 38.075 |
| 29 | 3.006 | 0.385 | -4.562 | -1.170 | 0.000 | 210.413 | 88.929 | 50.000 | 0.000 | 0.243 | 90.000 | 38.024 |
| 30 | 3.106 | 0.385 | -4.674 | -1.183 | 0.000 | 208.857 | 88.927 | 50.000 | 0.000 | 0.245 | 90.000 | 37.857 |
| 31 | 3.211 | 0.385 | -4.787 | -1.191 | 0.000 | 208.686 | 88.927 | 50.000 | 0.000 | 0.245 | 90.000 | 37.838 |

## True Triaxial Test on Fine Nevada Sand with a Target Void Ratio $=\mathbf{0 . 5 3 0}\left(\mathrm{D}_{\mathrm{r}}=\mathbf{9 1 . 2 8 \%}\right)$

| Test No.: | 9 |
| :--- | :---: |
| Test Date: | $3 / 13 / 11$ |
| Sector: | II |
| Initial Void Ratio, e: | 0.528 |
| Final Mean Princpal Stress, $\sigma_{\mathrm{m}}:$ | 203.0 kPa |
| Max Friction Angle, $\phi:$ | 49.1 deg |
| b-value at failure: | 0.49 |
| Stress direction at failure, $\alpha:$ | 90.0 deg |


| Shear Band Notes |
| :--- |
| Point of Observation: |
| Inclination (from Vertical): |
| $60^{\circ}$ |
| Failure Notes: |


| Point (No.) | $\begin{gathered} \varepsilon_{\mathrm{z}} \\ (\%) \end{gathered}$ | $\begin{gathered} \varepsilon_{\mathrm{x}} \\ (\%) \end{gathered}$ | $\begin{array}{r} \varepsilon_{\mathrm{y}} \\ (\%) \\ \hline \end{array}$ | $\begin{gathered} \varepsilon_{\mathrm{v}} \\ (\%) \end{gathered}$ | $\begin{gathered} \gamma_{\theta z} \\ (\%) \end{gathered}$ | $\begin{gathered} \sigma_{z} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \sigma_{x} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \sigma_{\mathrm{y}} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \tau_{\theta z} \\ (\mathrm{kPa}) \end{gathered}$ | b | $\begin{gathered} \alpha \\ \left({ }^{\circ}\right) \\ \hline \end{gathered}$ | $\begin{gathered} \varphi \\ \left(^{\circ}\right) \\ \hline \end{gathered}$ |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 1 | 0.000 | 0.000 | 0.000 | 0.000 | 0.000 | 51.617 | 50.808 | 50.000 | 0.000 | 0.500 | 90.000 | 0.912 |
| 2 | 0.005 | 0.000 | 0.003 | 0.009 | 0.000 | 60.633 | 55.317 | 50.000 | 0.000 | 0.500 | 90.000 | 5.515 |
| 3 | 0.021 | 0.000 | 0.005 | 0.026 | 0.000 | 73.617 | 61.811 | 50.000 | 0.000 | 0.500 | 90.000 | 11.014 |
| 4 | 0.039 | 0.499 | -0.495 | 0.043 | 0.000 | 85.880 | 67.858 | 50.000 | 0.000 | 0.498 | 90.000 | 15.311 |
| 5 | 0.079 | 0.951 | -0.962 | 0.068 | 0.000 | 104.270 | 76.898 | 50.000 | 0.000 | 0.496 | 90.000 | 20.596 |
| 6 | 0.111 | 0.125 | -0.159 | 0.077 | 0.000 | 116.511 | 83.151 | 50.000 | 0.000 | 0.498 | 90.000 | 23.543 |
| 7 | 0.211 | 0.275 | -0.387 | 0.099 | 0.000 | 143.480 | 96.569 | 50.000 | 0.000 | 0.498 | 90.000 | 28.891 |
| 8 | 0.302 | 0.375 | -0.565 | 0.112 | 0.000 | 165.727 | 107.647 | 50.000 | 0.000 | 0.498 | 90.000 | 32.442 |
| 9 | 0.412 | 0.500 | -0.800 | 0.112 | 0.000 | 187.180 | 118.322 | 50.000 | 0.000 | 0.498 | 90.000 | 35.337 |
| 10 | 0.519 | 0.575 | -0.991 | 0.103 | 0.000 | 207.498 | 128.468 | 50.000 | 0.000 | 0.498 | 90.000 | 37.709 |
| 11 | 0.617 | 0.675 | -1.196 | 0.097 | 0.000 | 225.281 | 137.325 | 50.000 | 0.000 | 0.498 | 90.000 | 39.549 |
| 12 | 0.703 | 0.775 | -1.392 | 0.086 | 0.000 | 240.177 | 144.732 | 50.000 | 0.000 | 0.498 | 90.000 | 40.949 |
| 13 | 0.816 | 0.875 | -1.631 | 0.060 | 0.000 | 260.330 | 154.784 | 50.000 | 0.000 | 0.498 | 90.000 | 42.669 |
| 14 | 0.951 | 0.975 | -1.909 | 0.017 | 0.000 | 288.205 | 168.713 | 50.000 | 0.000 | 0.498 | 90.000 | 44.775 |
| 15 | 1.033 | 1.025 | -2.071 | -0.013 | 0.000 | 304.015 | 174.646 | 50.000 | 0.000 | 0.491 | 90.000 | 45.851 |
| 16 | 1.110 | 1.100 | -2.257 | -0.047 | 0.000 | 316.942 | 179.793 | 50.000 | 0.000 | 0.486 | 90.000 | 46.675 |
| 17 | 1.211 | 1.150 | -2.447 | -0.086 | 0.000 | 330.475 | 187.593 | 50.000 | 0.000 | 0.491 | 90.000 | 47.491 |
| 18 | 1.316 | 1.203 | -2.653 | -0.133 | 0.000 | 343.225 | 192.731 | 50.000 | 0.000 | 0.487 | 90.000 | 48.219 |
| 19 | 1.421 | 1.250 | -2.854 | -0.183 | 0.000 | 355.930 | 200.498 | 50.000 | 0.000 | 0.492 | 90.000 | 48.908 |
| 20 | 1.505 | 1.350 | -3.087 | -0.232 | 0.000 | 358.741 | 200.286 | 50.000 | 0.000 | 0.487 | 90.000 | 49.056 |
| 21 | 1.644 | 1.350 | -3.308 | -0.314 | 0.000 | 358.791 | 200.178 | 50.000 | 0.000 | 0.486 | 90.000 | 49.058 |
| 22 | 1.755 | 1.350 | -3.479 | -0.374 | 0.000 | 354.499 | 197.498 | 50.000 | 0.000 | 0.484 | 90.000 | 48.832 |
| 23 | 1.846 | 1.350 | -3.615 | -0.419 | 0.000 | 348.985 | 194.835 | 50.000 | 0.000 | 0.484 | 90.000 | 48.535 |
| 24 | 1.901 | 1.350 | -3.694 | -0.443 | 0.000 | 344.573 | 192.185 | 50.000 | 0.000 | 0.483 | 90.000 | 48.293 |
| 25 | 2.000 | 1.350 | -3.832 | -0.482 | 0.000 | 336.843 | 190.571 | 50.000 | 0.000 | 0.490 | 90.000 | 47.859 |
| 26 | 2.105 | 1.350 | -3.971 | -0.516 | 0.000 | 327.306 | 184.235 | 50.000 | 0.000 | 0.484 | 90.000 | 47.304 |
| 27 | 2.211 | 1.350 | -4.107 | -0.546 | 0.000 | 316.736 | 178.957 | 50.000 | 0.000 | 0.483 | 90.000 | 46.663 |
| 28 | 2.417 | 1.350 | -4.354 | -0.587 | 0.000 | 307.961 | 176.308 | 50.000 | 0.000 | 0.490 | 90.000 | 46.107 |
| 29 | 2.494 | 1.350 | -4.444 | -0.600 | 0.000 | 305.625 | 173.673 | 50.000 | 0.000 | 0.484 | 90.000 | 45.956 |
| 30 | 2.895 | 1.350 | -4.903 | -0.658 | 0.000 | 299.993 | 171.036 | 50.000 | 0.000 | 0.484 | 90.000 | 45.584 |
| 31 | 3.229 | 1.350 | -5.280 | -0.701 | 0.000 | 298.730 | 171.028 | 50.000 | 0.000 | 0.487 | 90.000 | 45.500 |
| 32 | 3.684 | 1.350 | -5.757 | -0.723 | 0.000 | 297.644 | 168.378 | 50.000 | 0.000 | 0.478 | 90.000 | 45.426 |

True Triaxial Test on Fine Nevada Sand with a Target Void Ratio $=\mathbf{0 . 5 3 0}\left(\mathrm{D}_{\mathrm{r}}=\mathbf{9 1 . 2 8 \%}\right)$

| Test No.: | 10 |
| :--- | :--- |
| Test Date: | $3 / 15 / 11$ |
| Sector: | II |
| Initial Void Ratio, e: | 0.528 |
| Final Mean Princpal Stress, $\sigma_{\mathrm{m}}:$ | 265.3 kPa |
| Max Friction Angle, $\phi:$ | 52.4 deg |
| b-value at failure: | 0.70 |
| Stress direction at failure, $\alpha:$ | 90.0 deg |


| Shear Band Notes |
| :--- |
| Point of Observation: |
| Inclination (from Vertical): |
| $62^{\circ}$ |
| Failure Notes: |


| Point <br> (No.) | $\begin{gathered} \varepsilon_{\mathrm{z}} \\ (\%) \end{gathered}$ | $\begin{gathered} \varepsilon_{\mathrm{x}} \\ (\%) \end{gathered}$ | $\begin{array}{r} \varepsilon_{y} \\ (\%) \\ \hline \end{array}$ | $\begin{gathered} \varepsilon_{\mathrm{v}} \\ (\%) \end{gathered}$ | $\begin{aligned} & \gamma_{\theta z} \\ & (\%) \end{aligned}$ | $\begin{gathered} \sigma_{z} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \sigma_{x} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \sigma_{y} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \boldsymbol{\tau}_{\theta z} \\ (\mathrm{kPa}) \end{gathered}$ | b | $\begin{gathered} \alpha \\ \left({ }^{\circ}\right) \\ \hline \end{gathered}$ | $\begin{gathered} \varphi \\ \left({ }^{\circ}\right) \end{gathered}$ |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 1 | 0.000 | 0.000 | 0.000 | 0.000 | 0.000 | 51.617 | 51.213 | 50.000 | 0.000 | 0.750 | 90.000 | 0.912 |
| 2 | 0.056 | 0.000 | -0.047 | 0.009 | 0.000 | 76.492 | 69.880 | 50.000 | 0.000 | 0.750 | 90.000 | 12.089 |
| 3 | 0.061 | 0.000 | -0.048 | 0.013 | 0.000 | 80.097 | 72.586 | 50.000 | 0.000 | 0.750 | 90.000 | 13.376 |
| 4 | 0.066 | 0.000 | -0.040 | 0.026 | 0.000 | 85.507 | 76.648 | 50.000 | 0.000 | 0.750 | 90.000 | 15.191 |
| 5 | 0.070 | 0.485 | -0.521 | 0.034 | 0.000 | 89.115 | 79.214 | 50.000 | 0.000 | 0.747 | 90.000 | 16.330 |
| 6 | 0.073 | 0.685 | -0.721 | 0.037 | 0.000 | 92.361 | 81.576 | 50.000 | 0.000 | 0.745 | 90.000 | 17.311 |
| 7 | 0.077 | 0.785 | -0.819 | 0.043 | 0.000 | 95.968 | 84.232 | 50.000 | 0.000 | 0.745 | 90.000 | 18.356 |
| 8 | 0.081 | 0.885 | -0.920 | 0.045 | 0.000 | 99.214 | 86.613 | 50.000 | 0.000 | 0.744 | 90.000 | 19.258 |
| 9 | 0.086 | 0.984 | -1.021 | 0.049 | 0.000 | 103.902 | 90.063 | 50.000 | 0.000 | 0.743 | 90.000 | 20.502 |
| 10 | 0.106 | 0.209 | -0.255 | 0.060 | 0.000 | 115.789 | 99.291 | 50.000 | 0.000 | 0.749 | 90.000 | 23.380 |
| 11 | 0.201 | 0.419 | -0.542 | 0.077 | 0.000 | 157.167 | 130.200 | 50.000 | 0.000 | 0.748 | 90.000 | 31.151 |
| 12 | 0.303 | 0.609 | -0.833 | 0.080 | 0.000 | 193.053 | 156.961 | 50.000 | 0.000 | 0.748 | 90.000 | 36.055 |
| 13 | 0.405 | 0.780 | -1.121 | 0.065 | 0.000 | 224.526 | 180.402 | 50.000 | 0.000 | 0.747 | 90.000 | 39.475 |
| 14 | 0.519 | 0.914 | -1.384 | 0.049 | 0.000 | 243.338 | 194.429 | 50.000 | 0.000 | 0.747 | 90.000 | 41.231 |
| 15 | 0.612 | 1.142 | -1.764 | -0.011 | 0.000 | 277.478 | 217.981 | 50.000 | 0.000 | 0.738 | 90.000 | 43.998 |
| 16 | 0.707 | 1.277 | -2.036 | -0.052 | 0.000 | 300.814 | 234.861 | 50.000 | 0.000 | 0.737 | 90.000 | 45.639 |
| 17 | 0.813 | 1.423 | -2.353 | -0.116 | 0.000 | 322.573 | 248.981 | 50.000 | 0.000 | 0.730 | 90.000 | 47.020 |
| 18 | 0.923 | 1.569 | -2.699 | -0.207 | 0.000 | 340.612 | 261.677 | 50.000 | 0.000 | 0.728 | 90.000 | 48.073 |
| 19 | 1.022 | 1.716 | -3.030 | -0.293 | 0.000 | 361.834 | 276.949 | 50.000 | 0.000 | 0.728 | 90.000 | 49.217 |
| 20 | 1.113 | 1.862 | -3.343 | -0.368 | 0.000 | 377.358 | 289.562 | 50.000 | 0.000 | 0.732 | 90.000 | 49.996 |
| 21 | 1.219 | 1.955 | -3.648 | -0.473 | 0.000 | 389.128 | 298.522 | 50.000 | 0.000 | 0.733 | 90.000 | 50.559 |
| 22 | 1.316 | 2.055 | -3.921 | -0.551 | 0.000 | 406.297 | 308.285 | 50.000 | 0.000 | 0.725 | 90.000 | 51.338 |
| 23 | 1.416 | 2.254 | -4.316 | -0.645 | 0.000 | 418.764 | 316.676 | 50.000 | 0.000 | 0.723 | 90.000 | 51.876 |
| 24 | 1.504 | 2.301 | -4.528 | -0.723 | 0.000 | 428.785 | 315.853 | 50.000 | 0.000 | 0.702 | 90.000 | 52.292 |
| 25 | 1.626 | 2.394 | -4.846 | -0.826 | 0.000 | 430.464 | 315.530 | 50.000 | 0.000 | 0.698 | 90.000 | 52.360 |
| 26 | 1.719 | 2.440 | -5.054 | -0.895 | 0.000 | 428.130 | 315.825 | 50.000 | 0.000 | 0.703 | 90.000 | 52.265 |
| 27 | 1.812 | 2.487 | -5.262 | -0.964 | 0.000 | 408.588 | 310.165 | 50.000 | 0.000 | 0.726 | 90.000 | 51.438 |
| 28 | 1.919 | 2.533 | -5.473 | -1.020 | 0.000 | 365.239 | 278.430 | 50.000 | 0.000 | 0.725 | 90.000 | 49.392 |
| 29 | 2.013 | 2.533 | -5.581 | -1.035 | 0.000 | 346.690 | 263.361 | 50.000 | 0.000 | 0.719 | 90.000 | 48.410 |
| 30 | 2.113 | 2.533 | -5.690 | -1.043 | 0.000 | 332.378 | 253.015 | 50.000 | 0.000 | 0.719 | 90.000 | 47.602 |

True Triaxial Test on Fine Nevada Sand with a Target Void Ratio $=\mathbf{0 . 5 3 0}\left(\mathrm{D}_{\mathrm{r}}=\mathbf{9 1 . 2 8 \%}\right)$

| Test No.: | 11 |
| :--- | :--- |
| Test Date: | $3 / 17 / 11$ |
| Sector: | II |
| Initial Void Ratio, e: | 0.534 |
| Final Mean Princpal Stress, $\sigma_{\mathrm{m}}:$ | 265.6 kPa |
| Max Friction Angle, $\phi:$ | 52.4 deg |
| b-value at failure: | 0.69 |
| Stress direction at failure, $\alpha:$ | 90.0 deg |$\quad$| Shear Band Notes |  |
| :--- | :--- |
| Point of Observation: |  |
|  | 23 |
| Inclination (from Vertical): |  |
| $62^{\circ}$ |  |
|  |  |


| Point (No.) | $\begin{gathered} \varepsilon_{\mathrm{z}} \\ (\%) \end{gathered}$ | $\begin{gathered} \varepsilon_{\mathrm{x}} \\ (\%) \end{gathered}$ | $\begin{array}{r} \varepsilon_{y} \\ (\%) \\ \hline \end{array}$ | $\varepsilon_{\mathrm{v}}$ <br> (\%) | $\begin{aligned} & \gamma_{\theta z} \\ & (\%) \end{aligned}$ | $\begin{gathered} \sigma_{z} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \sigma_{x} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \sigma_{y} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \tau_{\text {өz }} \\ (\mathrm{kPa}) \end{gathered}$ | b | $\begin{gathered} \alpha \\ \left({ }^{\circ}\right) \end{gathered}$ | $\begin{gathered} \varphi \\ \left({ }^{\circ}\right) \\ \hline \end{gathered}$ |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 1 | 0.000 | 0.000 | 0.000 | 0.000 | 0.000 | 51.611 | 51.611 | 50.000 | 0.000 | 1.000 | 90.000 | 0.908 |
| 2 | 0.010 | 0.000 | -0.006 | 0.004 | 1.000 | 59.516 | 59.517 | 50.000 | 0.000 | 1.000 | 90.000 | 4.985 |
| 3 | 0.019 | 0.000 | -0.004 | 0.015 | 2.000 | 68.858 | 68.861 | 50.000 | 0.000 | 1.000 | 90.000 | 9.129 |
| 4 | 0.035 | 0.000 | -0.001 | 0.034 | 3.000 | 80.357 | 80.367 | 50.000 | 0.000 | 1.000 | 90.000 | 13.466 |
| 5 | 0.064 | 0.000 | -0.004 | 0.060 | 4.000 | 105.867 | 105.903 | 50.000 | 0.000 | 1.000 | 90.000 | 21.004 |
| 6 | 0.085 | 0.000 | -0.016 | 0.069 | 5.000 | 120.231 | 120.291 | 50.000 | 0.000 | 1.000 | 90.000 | 24.366 |
| 7 | 0.104 | 0.394 | -0.429 | 0.069 | 0.000 | 130.995 | 130.760 | 50.000 | 0.000 | 0.997 | 90.000 | 26.583 |
| 8 | 0.209 | 0.604 | -0.756 | 0.058 | 0.000 | 168.936 | 168.465 | 50.000 | 0.000 | 0.996 | 90.000 | 32.905 |
| 9 | 0.309 | 0.841 | -1.111 | 0.039 | 0.000 | 202.848 | 202.032 | 50.000 | 0.000 | 0.995 | 90.000 | 37.193 |
| 10 | 0.402 | 1.078 | -1.492 | -0.013 | 0.000 | 231.271 | 230.041 | 50.000 | 0.000 | 0.993 | 90.000 | 40.126 |
| 11 | 0.506 | 1.288 | -1.899 | -0.105 | 0.000 | 261.296 | 256.093 | 50.000 | 0.000 | 0.975 | 90.000 | 42.747 |
| 12 | 0.609 | 1.498 | -2.319 | -0.212 | 0.000 | 287.617 | 280.406 | 50.000 | 0.000 | 0.970 | 90.000 | 44.733 |
| 13 | 0.702 | 1.629 | -2.657 | -0.326 | 0.000 | 307.788 | 299.511 | 50.000 | 0.000 | 0.968 | 90.000 | 46.096 |
| 14 | 0.804 | 1.813 | -3.093 | -0.476 | 0.000 | 321.734 | 313.074 | 50.000 | 0.000 | 0.968 | 90.000 | 46.969 |
| 15 | 0.909 | 1.997 | -3.567 | -0.660 | 0.000 | 331.973 | 318.239 | 50.000 | 0.000 | 0.951 | 90.000 | 47.578 |
| 16 | 1.008 | 2.098 | -3.899 | -0.793 | 0.000 | 354.322 | 315.881 | 50.000 | 0.000 | 0.874 | 90.000 | 48.822 |
| 17 | 1.112 | 2.198 | -4.185 | -0.874 | 0.000 | 380.952 | 316.314 | 50.000 | 0.000 | 0.805 | 90.000 | 50.171 |
| 18 | 1.211 | 2.251 | -4.405 | -0.943 | 0.000 | 400.503 | 316.587 | 50.000 | 0.000 | 0.761 | 90.000 | 51.080 |
| 19 | 1.314 | 2.351 | -4.698 | -1.033 | 0.000 | 416.748 | 314.672 | 50.000 | 0.000 | 0.722 | 90.000 | 51.790 |
| 20 | 1.421 | 2.351 | -4.882 | -1.110 | 0.000 | 431.542 | 315.165 | 50.000 | 0.000 | 0.695 | 90.000 | 52.404 |
| 21 | 1.506 | 2.503 | -5.175 | -1.166 | 0.000 | 428.621 | 315.118 | 50.000 | 0.000 | 0.700 | 90.000 | 52.285 |
| 22 | 1.628 | 2.450 | -5.339 | -1.260 | 0.000 | 403.350 | 296.738 | 50.000 | 0.000 | 0.698 | 90.000 | 51.208 |
| 23 | 1.706 | 2.450 | -5.459 | -1.303 | 0.000 | 384.084 | 283.769 | 50.000 | 0.000 | 0.700 | 90.000 | 50.321 |
| 24 | 1.837 | 2.450 | -5.637 | -1.350 | 0.000 | 366.783 | 270.810 | 50.000 | 0.000 | 0.697 | 90.000 | 49.470 |
| 25 | 1.918 | 2.450 | -5.740 | -1.372 | 0.000 | 358.116 | 265.628 | 50.000 | 0.000 | 0.700 | 90.000 | 49.023 |
| 26 | 2.017 | 2.450 | -5.857 | -1.389 | 0.000 | 353.253 | 260.460 | 50.000 | 0.000 | 0.694 | 90.000 | 48.765 |
| 27 | 2.155 | 2.450 | -6.012 | -1.406 | 0.000 | 348.640 | 257.873 | 50.000 | 0.000 | 0.696 | 90.000 | 48.517 |
| 28 | 2.244 | 2.450 | -6.111 | -1.417 | 0.000 | 350.442 | 260.442 | 50.000 | 0.000 | 0.700 | 90.000 | 48.614 |
| 29 | 2.313 | 2.450 | -6.188 | -1.425 | 0.000 | 351.265 | 257.862 | 50.000 | 0.000 | 0.690 | 90.000 | 48.659 |
| 30 | 2.421 | 2.450 | -6.307 | -1.436 | 0.000 | 351.274 | 259.404 | 50.000 | 0.000 | 0.695 | 90.000 | 48.659 |
| 31 | 2.504 | 2.450 | -6.399 | -1.445 | 0.000 | 352.054 | 259.402 | 50.000 | 0.000 | 0.693 | 90.000 | 48.701 |

True Triaxial Test on Fine Nevada Sand with a Target Void Ratio $=0.530\left(D_{r}=91.28 \%\right)$

| Test No.: | 12 |
| :--- | :--- |
| Test Date: | $3 / 31 / 11$ |
| Sector: | II |
| Initial Void Ratio, e: | 0.530 |
| Final Mean Princpal Stress, $\sigma_{\mathrm{m}}:$ | 158.6 kPa |
| Max Friction Angle, $\phi:$ | 50.4 deg |
| b-value at failure: | 1.00 |
| Stress direction at failure, $\alpha:$ | 90.0 deg |


| Shear Band Notes |
| :--- |
| Point of Observation: |
| Inclination (from Vertical): |
| $90^{\circ}$ |
| Failure Notes: |
| vertical failure |


| Point (No.) | $\begin{gathered} \varepsilon_{\mathrm{z}} \\ (\%) \end{gathered}$ | $\begin{gathered} \varepsilon_{\mathrm{x}} \\ (\%) \end{gathered}$ | $\begin{array}{r} \varepsilon_{\mathrm{y}} \\ (\%) \\ \hline \end{array}$ | $\begin{gathered} \varepsilon_{\mathrm{v}} \\ (\%) \\ \hline \end{gathered}$ | $\begin{array}{r} \gamma_{\theta z} \\ (\%) \\ \hline \end{array}$ | $\begin{gathered} \sigma_{z} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \sigma_{x} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \sigma_{\mathrm{y}} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \tau_{\theta z} \\ (\mathrm{kPa}) \end{gathered}$ | b | $\alpha$ <br> ${ }^{\circ}$ ) | $\begin{gathered} \varphi \\ \left({ }^{\circ}\right) \\ \hline \end{gathered}$ |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 1 | 0.000 | 0.000 | 0.000 | 0.000 | 0.000 | 30.631 | 30.631 | 29.000 | 0.000 | 1.000 | 90.000 | 1.567 |
| 2 | 0.004 | 0.000 | 0.000 | 0.004 | 1.000 | 32.449 | 32.449 | 29.000 | 0.000 | 1.000 | 90.000 | 3.218 |
| 3 | 0.008 | 0.000 | 0.000 | 0.009 | 2.000 | 34.268 | 34.268 | 29.000 | 0.000 | 1.000 | 90.000 | 4.776 |
| 4 | 0.015 | 0.000 | -0.002 | 0.013 | 3.000 | 37.177 | 37.178 | 29.000 | 0.000 | 1.000 | 90.000 | 7.098 |
| 5 | 0.022 | 0.000 | -0.005 | 0.017 | 4.000 | 40.086 | 40.089 | 29.000 | 0.000 | 1.000 | 90.000 | 9.234 |
| 6 | 0.034 | 0.380 | -0.386 | 0.028 | 5.000 | 44.087 | 44.034 | 29.000 | 0.000 | 0.997 | 90.000 | 11.913 |
| 7 | 0.043 | 0.574 | -0.582 | 0.035 | 6.000 | 46.269 | 46.177 | 29.000 | 0.000 | 0.995 | 90.000 | 13.263 |
| 8 | 0.050 | 0.721 | -0.728 | 0.044 | 7.000 | 48.451 | 48.321 | 29.000 | 0.000 | 0.993 | 90.000 | 14.545 |
| 9 | 0.053 | 0.868 | -0.875 | 0.046 | 8.000 | 50.634 | 50.458 | 29.000 | 0.000 | 0.992 | 90.000 | 15.764 |
| 10 | 0.066 | 0.968 | -0.981 | 0.052 | 9.000 | 52.815 | 52.600 | 29.000 | 0.000 | 0.991 | 90.000 | 16.923 |
| 11 | 0.075 | 1.122 | -1.140 | 0.057 | 10.000 | 55.724 | 55.444 | 29.000 | 0.000 | 0.990 | 90.000 | 18.387 |
| 12 | 0.085 | 1.168 | -1.193 | 0.061 | 11.000 | 58.269 | 57.951 | 29.000 | 0.000 | 0.989 | 90.000 | 19.596 |
| 13 | 0.095 | 1.315 | -1.345 | 0.065 | 12.000 | 59.723 | 59.347 | 29.000 | 0.000 | 0.988 | 90.000 | 20.260 |
| 14 | 0.103 | 0.616 | -0.651 | 0.068 | 0.000 | 61.898 | 61.729 | 29.000 | 0.000 | 0.995 | 90.000 | 21.218 |
| 15 | 0.200 | 0.700 | -1.069 | -0.169 | 0.000 | 84.396 | 83.981 | 29.000 | 0.000 | 0.993 | 90.000 | 29.243 |
| 16 | 0.319 | 1.232 | -1.485 | 0.065 | 0.000 | 110.452 | 110.273 | 29.000 | 0.000 | 0.998 | 90.000 | 35.739 |
| 17 | 0.423 | 1.421 | -1.812 | 0.033 | 0.000 | 127.379 | 127.628 | 29.000 | 0.000 | 1.003 | 90.000 | 38.984 |
| 18 | 0.529 | 1.563 | -2.101 | -0.009 | 0.000 | 146.421 | 146.654 | 29.000 | 0.000 | 1.002 | 90.000 | 42.018 |
| 19 | 0.615 | 1.753 | -2.437 | -0.070 | 0.000 | 155.296 | 155.655 | 29.000 | 0.000 | 1.003 | 90.000 | 43.258 |
| 20 | 0.741 | 1.957 | -2.847 | -0.148 | 0.000 | 171.286 | 171.715 | 29.000 | 0.000 | 1.003 | 90.000 | 45.269 |
| 21 | 0.810 | 2.058 | -3.068 | -0.201 | 0.000 | 178.693 | 179.961 | 29.000 | 0.000 | 1.008 | 90.000 | 46.116 |
| 22 | 0.904 | 2.258 | -3.436 | -0.275 | 0.000 | 189.257 | 190.370 | 29.000 | 0.000 | 1.007 | 90.000 | 47.245 |
| 23 | 1.025 | 2.512 | -3.923 | -0.386 | 0.000 | 201.127 | 201.628 | 29.000 | 0.000 | 1.003 | 90.000 | 48.414 |
| 24 | 1.125 | 2.712 | -4.317 | -0.480 | 0.000 | 209.790 | 210.315 | 29.000 | 0.000 | 1.003 | 90.000 | 49.210 |
| 25 | 1.205 | 2.866 | -4.638 | -0.567 | 0.000 | 216.319 | 219.050 | 29.000 | 0.000 | 1.015 | 90.000 | 49.780 |
| 26 | 1.311 | 3.120 | -5.128 | -0.698 | 0.000 | 223.076 | 223.572 | 29.000 | 0.000 | 1.003 | 90.000 | 50.346 |
| 27 | 1.405 | 3.467 | -5.744 | -0.872 | 0.000 | 218.365 | 218.738 | 29.000 | 0.000 | 1.002 | 90.000 | 49.954 |
| 28 | 1.534 | 3.808 | -6.433 | -1.090 | 0.000 | 209.636 | 208.739 | 29.000 | 0.000 | 0.995 | 90.000 | 49.196 |
| 29 | 1.633 | 4.148 | -7.046 | -1.265 | 0.000 | 203.212 | 202.757 | 29.000 | 0.000 | 0.997 | 90.000 | 48.610 |
| 30 | 1.704 | 4.342 | -7.385 | -1.339 | 0.000 | 204.431 | 203.614 | 29.000 | 0.000 | 0.995 | 90.000 | 48.723 |
| 31 | 1.852 | 4.629 | -7.981 | -1.500 | 0.000 | 206.812 | 208.008 | 29.000 | 0.000 | 1.007 | 90.000 | 48.942 |
| 32 | 1.907 | 4.722 | -8.200 | -1.570 | 0.000 | 207.685 | 207.748 | 29.000 | 0.000 | 1.000 | 90.000 | 49.021 |
| 33 | 2.031 | 4.909 | -8.632 | -1.692 | 0.000 | 218.238 | 221.219 | 29.000 | 0.000 | 1.016 | 90.000 | 49.943 |

True Triaxial Test on Fine Nevada Sand with a Target Void Ratio $=0.530\left(D_{r}=91.28 \%\right)$

| Test No.: | 13 |
| :--- | :---: |
| Test Date: | $2 / 25 / 11$ |
| Sector: | III |
| Initial Void Ratio, e: | 0.535 |
| Final Mean Princpal Stress, $\sigma_{\mathrm{m}}:$ | 102.2 kPa |
| Max Friction Angle, $\phi:$ | 37.6 deg |
| b-value at failure: | 0.00 |
| Stress direction at failure, $\alpha:$ | 90.0 deg |


| Shear Band Notes |
| :--- |
| Point of Observation: |
| Inclination (from Vertical): |
| $63^{\circ}$ |
| Failure Notes: |


| Point (No.) | $\begin{gathered} \varepsilon_{\mathrm{z}} \\ (\%) \end{gathered}$ | $\begin{gathered} \varepsilon_{\mathrm{x}} \\ (\%) \end{gathered}$ | $\begin{array}{r} \varepsilon_{\mathrm{y}} \\ (\%) \\ \hline \end{array}$ | $\begin{gathered} \varepsilon_{\mathrm{v}} \\ (\%) \\ \hline \end{gathered}$ | $\begin{gathered} \gamma_{\theta z} \\ (\%) \\ \hline \end{gathered}$ | $\begin{gathered} \sigma_{z} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \sigma_{x} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \sigma_{\mathrm{y}} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \tau_{\text {өz }} \\ (\mathrm{kPa}) \end{gathered}$ | b | $\alpha$ <br> ${ }^{\circ}$ ) | $\begin{gathered} \varphi \\ \left({ }^{\circ}\right) \\ \hline \end{gathered}$ |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 1 | 0.000 | 0.000 | 0.000 | 0.000 | 0.000 | 52.164 | 50.000 | 50.000 | 0.000 | 0.000 | 90.000 | 1.213 |
| 2 | 0.017 | -0.006 | -0.006 | 0.004 | 0.000 | 59.669 | 50.000 | 50.000 | 0.000 | 0.000 | 90.000 | 5.058 |
| 3 | 0.031 | -0.011 | -0.011 | 0.009 | 0.000 | 70.658 | 50.000 | 50.000 | 0.000 | 0.000 | 90.000 | 9.858 |
| 4 | 0.072 | -0.019 | -0.019 | 0.035 | 0.000 | 87.137 | 50.000 | 50.000 | 0.000 | 0.000 | 90.000 | 15.712 |
| 5 | 0.155 | -0.049 | -0.049 | 0.057 | 0.000 | 106.515 | 50.000 | 50.000 | 0.000 | 0.000 | 90.000 | 21.167 |
| 6 | 0.246 | -0.084 | -0.084 | 0.079 | 0.000 | 120.010 | 50.000 | 50.000 | 0.000 | 0.000 | 90.000 | 24.318 |
| 7 | 0.541 | -0.222 | -0.222 | 0.096 | 0.000 | 141.338 | 50.000 | 50.000 | 0.000 | 0.000 | 90.000 | 28.513 |
| 8 | 0.615 | -0.260 | -0.260 | 0.096 | 0.000 | 144.915 | 50.000 | 50.000 | 0.000 | 0.000 | 90.000 | 29.141 |
| 9 | 0.876 | -0.399 | -0.399 | 0.079 | 0.000 | 155.554 | 50.000 | 50.000 | 0.000 | 0.000 | 90.000 | 30.898 |
| 10 | 1.077 | -0.510 | -0.510 | 0.057 | 0.000 | 161.845 | 50.000 | 50.000 | 0.000 | 0.000 | 90.000 | 31.867 |
| 11 | 1.215 | -0.590 | -0.590 | 0.035 | 0.000 | 165.647 | 50.000 | 50.000 | 0.000 | 0.000 | 90.000 | 32.431 |
| 12 | 1.421 | -0.711 | -0.711 | 0.000 | 0.000 | 171.145 | 50.000 | 50.000 | 0.000 | 0.000 | 90.000 | 33.217 |
| 13 | 1.846 | -0.967 | -0.967 | -0.087 | 0.000 | 177.703 | 50.000 | 50.000 | 0.000 | 0.000 | 90.000 | 34.113 |
| 14 | 2.221 | -1.200 | -1.200 | -0.179 | 0.000 | 184.610 | 50.000 | 50.000 | 0.000 | 0.000 | 90.000 | 35.013 |
| 15 | 2.647 | -1.470 | -1.470 | -0.293 | 0.000 | 190.629 | 50.000 | 50.000 | 0.000 | 0.000 | 90.000 | 35.762 |
| 16 | 2.942 | -1.670 | -1.670 | -0.398 | 0.000 | 192.537 | 50.000 | 50.000 | 0.000 | 0.000 | 90.000 | 35.993 |
| 17 | 3.307 | -1.909 | -1.909 | -0.511 | 0.000 | 197.127 | 50.000 | 50.000 | 0.000 | 0.000 | 90.000 | 36.538 |
| 18 | 3.736 | -2.194 | -2.194 | -0.651 | 0.000 | 199.425 | 50.000 | 50.000 | 0.000 | 0.000 | 90.000 | 36.804 |
| 19 | 4.018 | -2.385 | -2.385 | -0.752 | 0.000 | 201.979 | 50.000 | 50.000 | 0.000 | 0.000 | 90.000 | 37.095 |
| 20 | 4.402 | -2.645 | -2.645 | -0.887 | 0.000 | 204.640 | 50.000 | 50.000 | 0.000 | 0.000 | 90.000 | 37.394 |
| 21 | 5.038 | -3.077 | -3.077 | -1.115 | 0.000 | 205.674 | 50.000 | 50.000 | 0.000 | 0.000 | 90.000 | 37.508 |
| 22 | 5.286 | -3.242 | -3.242 | -1.198 | 0.000 | 206.513 | 50.000 | 50.000 | 0.000 | 0.000 | 90.000 | 37.601 |
| 23 | 5.856 | -3.629 | -3.629 | -1.403 | 0.000 | 205.256 | 50.000 | 50.000 | 0.000 | 0.000 | 90.000 | 37.462 |
| 24 | 6.362 | -3.961 | -3.961 | -1.561 | 0.000 | 194.047 | 50.000 | 50.000 | 0.000 | 0.000 | 90.000 | 36.174 |
| 25 | 6.685 | -4.155 | -4.155 | -1.626 | 0.000 | 187.065 | 50.000 | 50.000 | 0.000 | 0.000 | 90.000 | 35.322 |
| 26 | 6.886 | -4.271 | -4.271 | -1.657 | 0.000 | 183.708 | 50.000 | 50.000 | 0.000 | 0.000 | 90.000 | 34.898 |
| 27 | 7.104 | -4.391 | -4.391 | -1.679 | 0.000 | 181.023 | 50.000 | 50.000 | 0.000 | 0.000 | 90.000 | 34.551 |
| 28 | 7.803 | -4.767 | -4.767 | -1.731 | 0.000 | 175.653 | 50.000 | 50.000 | 0.000 | 0.000 | 90.000 | 33.838 |
| 29 | 8.442 | -5.097 | -5.097 | -1.753 | 0.000 | 170.470 | 50.000 | 50.000 | 0.000 | 0.000 | 90.000 | 33.122 |
| 30 | 8.839 | -5.301 | -5.301 | -1.764 | 0.000 | 170.262 | 50.000 | 50.000 | 0.000 | 0.000 | 90.000 | 33.093 |
| 31 | 9.088 | -5.428 | -5.428 | -1.768 | 0.000 | 169.601 | 50.000 | 50.000 | 0.000 | 0.000 | 90.000 | 32.999 |

True Triaxial Test on Fine Nevada Sand with a Target Void Ratio $=\mathbf{0 . 5 3 0}\left(\mathrm{D}_{\mathrm{r}}=\mathbf{9 1 . 2 8 \%}\right)$

| Test No.: | 14 |
| :--- | :---: |
| Test Date: | $3 / 5 / 11$ |
| Sector: | III |
| Initial Void Ratio, e: | 0.530 |
| Final Mean Princpal Stress, $\sigma_{\mathrm{m}}:$ | 151.6 kPa |
| Max Friction Angle, $\phi:$ | 45.1 deg |
| b-value at failure: | 0.25 |
| Stress direction at failure, $\alpha:$ | 90.0 deg |


| Shear Band Notes |
| :--- |
| Point of Observation: |
|  |
| Inclination (from Vertical): |
| $59^{\circ}, 60^{\circ}$ |
| Failure Notes: |


| Point (No.) | $\begin{gathered} \varepsilon_{\mathrm{z}} \\ (\%) \end{gathered}$ | $\begin{gathered} \varepsilon_{\mathrm{x}} \\ (\%) \end{gathered}$ | $\begin{gathered} \varepsilon_{y} \\ (\%) \end{gathered}$ | $\begin{gathered} \varepsilon_{\mathrm{v}} \\ (\%) \end{gathered}$ | $\gamma_{\theta z}$ <br> (\%) | $\begin{gathered} \sigma_{z} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \sigma_{x} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \sigma_{y} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \tau_{\text {өz }} \\ (\mathrm{kPa}) \end{gathered}$ | b | $\begin{gathered} \alpha \\ \left({ }^{\circ}\right) \\ \hline \end{gathered}$ | $\begin{gathered} \varphi \\ \left({ }^{\circ}\right) \\ \hline \end{gathered}$ |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 1 | 0.000 | 0.000 | 0.000 | 0.000 | 0.000 | 51.606 | 50.000 | 50.000 | 0.000 | 0.000 | 90.000 | 0.905 |
| 2 | 0.005 | 0.000 | -0.003 | 0.002 | 0.000 | 53.789 | 50.486 | 50.000 | 0.000 | 0.128 | 90.000 | 2.092 |
| 3 | 0.016 | 0.000 | -0.008 | 0.009 | 0.000 | 67.977 | 54.053 | 50.000 | 0.000 | 0.225 | 90.000 | 8.765 |
| 4 | 0.028 | 0.000 | -0.015 | 0.013 | 0.000 | 76.343 | 56.160 | 50.000 | 0.000 | 0.234 | 90.000 | 12.035 |
| 5 | 0.038 | 0.000 | -0.018 | 0.020 | 0.000 | 83.618 | 57.998 | 50.000 | 0.000 | 0.238 | 90.000 | 14.572 |
| 6 | 0.067 | 0.000 | -0.032 | 0.035 | 0.000 | 108.347 | 64.161 | 50.000 | 0.000 | 0.243 | 90.000 | 21.622 |
| 7 | 0.108 | 0.047 | -0.107 | 0.048 | 0.000 | 126.155 | 68.650 | 50.000 | 0.000 | 0.245 | 90.000 | 25.615 |
| 8 | 0.168 | 0.084 | -0.186 | 0.065 | 0.000 | 154.467 | 75.715 | 50.000 | 0.000 | 0.246 | 90.000 | 30.725 |
| 9 | 0.311 | 0.134 | -0.376 | 0.069 | 0.000 | 183.360 | 82.992 | 50.000 | 0.000 | 0.247 | 90.000 | 34.853 |
| 10 | 0.461 | 0.184 | -0.585 | 0.060 | 0.000 | 203.443 | 88.046 | 50.000 | 0.000 | 0.248 | 90.000 | 37.260 |
| 11 | 0.603 | 0.228 | -0.787 | 0.044 | 0.000 | 218.762 | 91.961 | 50.000 | 0.000 | 0.249 | 90.000 | 38.897 |
| 12 | 0.758 | 0.267 | -1.004 | 0.022 | 0.000 | 231.469 | 95.171 | 50.000 | 0.000 | 0.249 | 90.000 | 40.145 |
| 13 | 0.902 | 0.301 | -1.209 | -0.007 | 0.000 | 243.060 | 98.162 | 50.000 | 0.000 | 0.249 | 90.000 | 41.206 |
| 14 | 1.061 | 0.334 | -1.437 | -0.041 | 0.000 | 253.850 | 100.931 | 50.000 | 0.000 | 0.250 | 90.000 | 42.136 |
| 15 | 1.210 | 0.373 | -1.664 | -0.081 | 0.000 | 262.454 | 103.099 | 50.000 | 0.000 | 0.250 | 90.000 | 42.840 |
| 16 | 1.351 | 0.407 | -1.876 | -0.118 | 0.000 | 269.255 | 104.946 | 50.000 | 0.000 | 0.251 | 90.000 | 43.375 |
| 17 | 1.524 | 0.488 | -2.184 | -0.172 | 0.000 | 275.213 | 106.488 | 50.000 | 0.000 | 0.251 | 90.000 | 43.829 |
| 18 | 1.655 | 0.488 | -2.356 | -0.214 | 0.000 | 280.907 | 108.024 | 50.000 | 0.000 | 0.251 | 90.000 | 44.251 |
| 19 | 1.831 | 0.541 | -2.647 | -0.275 | 0.000 | 289.998 | 110.377 | 50.000 | 0.000 | 0.252 | 90.000 | 44.901 |
| 20 | 1.964 | 0.440 | -2.726 | -0.323 | 0.000 | 292.779 | 111.156 | 50.000 | 0.000 | 0.252 | 90.000 | 45.094 |
| 21 | 2.116 | 0.440 | -2.938 | -0.382 | 0.000 | 293.345 | 111.393 | 50.000 | 0.000 | 0.252 | 90.000 | 45.133 |
| 22 | 2.270 | 0.440 | -3.156 | -0.445 | 0.000 | 284.330 | 109.219 | 50.000 | 0.000 | 0.253 | 90.000 | 44.499 |
| 23 | 2.452 | 0.440 | -3.399 | -0.506 | 0.000 | 275.297 | 107.050 | 50.000 | 0.000 | 0.253 | 90.000 | 43.835 |
| 24 | 2.569 | 0.440 | -3.543 | -0.535 | 0.000 | 263.334 | 104.042 | 50.000 | 0.000 | 0.253 | 90.000 | 42.910 |
| 25 | 2.725 | 0.440 | -3.724 | -0.559 | 0.000 | 255.917 | 102.215 | 50.000 | 0.000 | 0.254 | 90.000 | 42.308 |
| 26 | 2.927 | 0.440 | -3.947 | -0.580 | 0.000 | 252.662 | 101.516 | 50.000 | 0.000 | 0.254 | 90.000 | 42.036 |
| 27 | 3.045 | 0.440 | -4.076 | -0.591 | 0.000 | 252.412 | 101.515 | 50.000 | 0.000 | 0.255 | 90.000 | 42.015 |
| 28 | 3.211 | 0.440 | -4.256 | -0.604 | 0.000 | 252.063 | 101.515 | 50.000 | 0.000 | 0.255 | 90.000 | 41.986 |
| 29 | 3.317 | 0.440 | -4.370 | -0.613 | 0.000 | 252.543 | 101.675 | 50.000 | 0.000 | 0.255 | 90.000 | 42.026 |
| 30 | 3.602 | 0.440 | -4.673 | -0.631 | 0.000 | 251.608 | 101.625 | 50.000 | 0.000 | 0.256 | 90.000 | 41.947 |
| 31 | 3.780 | 0.440 | -4.859 | -0.639 | 0.000 | 251.947 | 101.789 | 50.000 | 0.000 | 0.256 | 90.000 | 41.976 |

# True Triaxial Test on Fine Nevada Sand with a Target Void Ratio $=\mathbf{0 . 5 3 0}\left(\mathrm{D}_{\mathrm{r}}=\mathbf{9 1 . 2 8 \%}\right)$ 

| Test No.: | 15 |
| :--- | :---: |
| Test Date: | $3 / 8 / 11$ |
| Sector: | III |
| Initial Void Ratio, e: | 0.533 |
| Final Mean Princpal Stress, $\sigma_{\mathrm{m}}:$ | 165.0 kPa |
| Max Friction Angle, $\phi:$ | 44.3 deg |
| b-value at failure: | 0.49 |
| Stress direction at failure, $\alpha:$ | 90.0 deg |


| Shear Band Notes |  |
| :--- | :--- |
| Point of Observation: | 25 |
|  |  |
| Inclination (from Vertical): |  |
| $59^{\circ}, 60^{\circ}$ |  |
| Failure Notes: |  |


| Point (No.) | $\begin{gathered} \varepsilon_{\mathrm{z}} \\ (\%) \end{gathered}$ | $\begin{gathered} \varepsilon_{\mathrm{x}} \\ (\%) \end{gathered}$ | $\begin{array}{r} \varepsilon_{\mathrm{y}} \\ (\%) \\ \hline \end{array}$ | $\begin{gathered} \varepsilon_{\mathrm{v}} \\ (\%) \end{gathered}$ | $\begin{aligned} & \gamma_{\theta z} \\ & (\%) \end{aligned}$ | $\begin{gathered} \sigma_{z} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \sigma_{x} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \sigma_{\mathrm{y}} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \tau_{\theta z} \\ (\mathrm{kPa}) \end{gathered}$ | b | $\begin{gathered} \alpha \\ \left({ }^{\circ}\right) \\ \hline \end{gathered}$ | $\begin{gathered} \varphi \\ \left({ }^{\circ}\right) \\ \hline \end{gathered}$ |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 1 | 0.000 | 0.000 | 0.000 | 0.000 | 0.000 | 51.602 | 48.282 | 50.000 | 0.000 | 0.517 | 90.000 | 0.903 |
| 2 | 0.009 | 0.325 | 0.000 | 0.333 | 0.000 | 63.814 | 54.349 | 50.000 | 0.000 | 0.315 | 90.000 | 6.971 |
| 3 | 0.018 | 2.124 | -0.007 | 2.135 | 0.000 | 69.201 | 57.946 | 50.000 | 0.000 | 0.414 | 90.000 | 9.270 |
| 4 | 0.028 | 2.529 | -0.416 | 2.141 | 0.000 | 74.229 | 59.359 | 50.000 | 0.000 | 0.386 | 90.000 | 11.247 |
| 5 | 0.037 | 2.489 | -0.383 | 2.143 | 0.000 | 79.255 | 62.038 | 50.000 | 0.000 | 0.411 | 90.000 | 13.081 |
| 6 | 0.050 | 2.596 | -0.496 | 2.150 | 0.000 | 84.640 | 64.698 | 50.000 | 0.000 | 0.424 | 90.000 | 14.909 |
| 7 | 0.060 | 2.603 | -0.509 | 2.154 | 0.000 | 88.947 | 66.836 | 50.000 | 0.000 | 0.432 | 90.000 | 16.278 |
| 8 | 0.071 | 2.656 | -0.569 | 2.159 | 0.000 | 92.894 | 68.804 | 50.000 | 0.000 | 0.438 | 90.000 | 17.468 |
| 9 | 0.101 | 0.249 | -0.305 | 0.045 | 0.000 | 102.219 | 73.520 | 50.000 | 0.000 | 0.450 | 90.000 | 20.063 |
| 10 | 0.205 | 0.454 | -0.588 | 0.071 | 0.000 | 126.572 | 85.672 | 50.000 | 0.000 | 0.466 | 90.000 | 25.700 |
| 11 | 0.305 | 0.635 | -0.861 | 0.080 | 0.000 | 146.216 | 95.436 | 50.000 | 0.000 | 0.472 | 90.000 | 29.364 |
| 12 | 0.403 | 0.794 | -1.113 | 0.084 | 0.000 | 161.885 | 103.203 | 50.000 | 0.000 | 0.476 | 90.000 | 31.873 |
| 13 | 0.520 | 0.930 | -1.370 | 0.080 | 0.000 | 177.134 | 110.797 | 50.000 | 0.000 | 0.478 | 90.000 | 34.037 |
| 14 | 0.601 | 1.043 | -1.570 | 0.073 | 0.000 | 187.392 | 115.884 | 50.000 | 0.000 | 0.480 | 90.000 | 35.363 |
| 15 | 0.710 | 1.134 | -1.783 | 0.060 | 0.000 | 200.080 | 122.246 | 50.000 | 0.000 | 0.481 | 90.000 | 36.879 |
| 16 | 0.837 | 1.247 | -2.058 | 0.026 | 0.000 | 212.321 | 128.298 | 50.000 | 0.000 | 0.482 | 90.000 | 38.228 |
| 17 | 0.925 | 1.293 | -2.211 | 0.006 | 0.000 | 222.477 | 133.392 | 50.000 | 0.000 | 0.483 | 90.000 | 39.272 |
| 18 | 1.052 | 1.361 | -2.439 | -0.026 | 0.000 | 234.654 | 139.511 | 50.000 | 0.000 | 0.485 | 90.000 | 40.443 |
| 19 | 1.137 | 1.406 | -2.595 | -0.052 | 0.000 | 242.981 | 143.739 | 50.000 | 0.000 | 0.486 | 90.000 | 41.199 |
| 20 | 1.357 | 1.474 | -2.954 | -0.123 | 0.000 | 261.916 | 153.347 | 50.000 | 0.000 | 0.488 | 90.000 | 42.797 |
| 21 | 1.445 | 1.497 | -3.097 | -0.155 | 0.000 | 267.684 | 156.255 | 50.000 | 0.000 | 0.488 | 90.000 | 43.253 |
| 22 | 1.935 | 1.613 | -3.883 | -0.336 | 0.000 | 281.402 | 163.565 | 50.000 | 0.000 | 0.491 | 90.000 | 44.287 |
| 23 | 1.973 | 1.610 | -3.962 | -0.379 | 0.000 | 280.173 | 162.936 | 50.000 | 0.000 | 0.491 | 90.000 | 44.197 |
| 24 | 2.127 | 1.610 | -4.185 | -0.448 | 0.000 | 271.628 | 158.772 | 50.000 | 0.000 | 0.491 | 90.000 | 43.557 |
| 25 | 2.237 | 1.610 | -4.338 | -0.491 | 0.000 | 264.305 | 155.212 | 50.000 | 0.000 | 0.491 | 90.000 | 42.987 |
| 26 | 2.353 | 1.610 | -4.497 | -0.534 | 0.000 | 253.499 | 149.920 | 50.000 | 0.000 | 0.491 | 90.000 | 42.106 |
| 27 | 2.452 | 1.610 | -4.622 | -0.560 | 0.000 | 244.880 | 145.698 | 50.000 | 0.000 | 0.491 | 90.000 | 41.367 |
| 28 | 2.587 | 1.610 | -4.783 | -0.586 | 0.000 | 235.871 | 141.270 | 50.000 | 0.000 | 0.491 | 90.000 | 40.556 |
| 29 | 2.691 | 1.610 | -4.902 | -0.601 | 0.000 | 231.135 | 139.005 | 50.000 | 0.000 | 0.491 | 90.000 | 40.113 |
| 30 | 2.804 | 1.610 | -5.026 | -0.612 | 0.000 | 227.446 | 137.217 | 50.000 | 0.000 | 0.492 | 90.000 | 39.760 |
| 31 | 2.880 | 1.610 | -5.108 | -0.618 | 0.000 | 225.918 | 136.534 | 50.000 | 0.000 | 0.492 | 90.000 | 39.611 |
| 32 | 2.981 | 1.610 | -5.215 | -0.625 | 0.000 | 225.045 | 136.168 | 50.000 | 0.000 | 0.492 | 90.000 | 39.526 |
| 33 | 3.336 | 1.610 | -5.588 | -0.642 | 0.000 | 222.704 | 135.249 | 50.000 | 0.000 | 0.494 | 90.000 | 39.294 |

True Triaxial Test on Fine Nevada Sand with a Target Void Ratio $=\mathbf{0 . 5 3 0}\left(\mathrm{D}_{\mathrm{r}}=\mathbf{9 1 . 2 8 \%}\right)$

| Test No.: | 16 |
| :--- | :---: |
| Test Date: | $3 / 10 / 11$ |
| Sector: | III |
| Initial Void Ratio, e: | 0.533 |
| Final Mean Princpal Stress, $\sigma_{\mathrm{m}}:$ | 232.1 kPa |
| Max Friction Angle, $\phi:$ | 49.5 deg |
| b-value at failure: | 0.72 |
| Stress direction at failure, $\alpha$ : | 90.0 deg |


| Shear Band Notes |
| :--- |
| Point of Observation: |
| Inclination (from Vertical): |
| none observed |
| Failure Notes: |
| vacuum was not held after failure |


| Point (No.) | $\begin{gathered} \varepsilon_{\mathrm{z}} \\ (\%) \\ \hline \end{gathered}$ | $\begin{gathered} \varepsilon_{\mathrm{x}} \\ (\%) \\ \hline \end{gathered}$ | $\begin{array}{r} \varepsilon_{y} \\ (\%) \\ \hline \end{array}$ | $\begin{gathered} \varepsilon_{\mathrm{v}} \\ (\%) \end{gathered}$ | $\begin{aligned} & \gamma_{\theta z} \\ & (\%) \end{aligned}$ | $\begin{gathered} \sigma_{z} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \sigma_{\mathrm{x}} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \sigma_{\mathrm{y}} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \boldsymbol{\tau}_{\theta z} \\ (\mathrm{kPa}) \\ \hline \end{gathered}$ | b | $\alpha$ <br> ${ }^{\circ}$ ) | $\begin{gathered} \varphi \\ \left({ }^{\circ}\right) \\ \hline \end{gathered}$ |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 1 | 0.000 | 0.000 | 0.000 | 0.000 | 0.000 | 51.610 | 49.168 | 50.000 | 0.000 | 0.341 | 90.000 | 0.908 |
| 2 | 0.003 | 0.000 | 0.000 | 0.002 | 0.000 | 54.125 | 51.054 | 50.000 | 0.000 | 0.255 | 90.000 | 4.461 |
| 3 | 0.009 | 0.000 | -0.007 | 0.002 | 0.000 | 58.435 | 54.287 | 50.000 | 0.000 | 0.508 | 90.000 | 6.814 |
| 4 | 0.029 | 0.000 | -0.013 | 0.015 | 0.000 | 73.160 | 65.334 | 50.000 | 0.000 | 0.662 | 90.000 | 10.839 |
| 5 | 0.042 | 0.000 | -0.016 | 0.026 | 0.000 | 86.807 | 75.576 | 50.000 | 0.000 | 0.695 | 90.000 | 15.607 |
| 6 | 0.075 | 0.199 | -0.226 | 0.047 | 0.000 | 103.322 | 87.905 | 50.000 | 0.000 | 0.711 | 90.000 | 20.351 |
| 7 | 0.099 | 0.451 | -0.494 | 0.056 | 0.000 | 113.368 | 95.326 | 50.000 | 0.000 | 0.715 | 90.000 | 22.823 |
| 8 | 0.107 | 0.385 | -0.432 | 0.060 | 0.000 | 116.237 | 97.506 | 50.000 | 0.000 | 0.717 | 90.000 | 23.481 |
| 9 | 0.203 | 0.650 | -0.776 | 0.078 | 0.000 | 141.302 | 116.141 | 50.000 | 0.000 | 0.724 | 90.000 | 28.507 |
| 10 | 0.317 | 0.819 | -1.059 | 0.078 | 0.000 | 161.270 | 131.008 | 50.000 | 0.000 | 0.728 | 90.000 | 31.781 |
| 11 | 0.414 | 0.963 | -1.303 | 0.074 | 0.000 | 180.139 | 145.044 | 50.000 | 0.000 | 0.730 | 90.000 | 34.436 |
| 12 | 0.502 | 1.108 | -1.538 | 0.071 | 0.000 | 196.125 | 156.907 | 50.000 | 0.000 | 0.732 | 90.000 | 36.420 |
| 13 | 0.605 | 1.228 | -1.792 | 0.041 | 0.000 | 211.658 | 168.467 | 50.000 | 0.000 | 0.733 | 90.000 | 38.157 |
| 14 | 0.707 | 1.445 | -2.145 | 0.006 | 0.000 | 226.797 | 179.601 | 50.000 | 0.000 | 0.733 | 90.000 | 39.697 |
| 15 | 0.809 | 1.589 | -2.431 | -0.032 | 0.000 | 240.104 | 189.449 | 50.000 | 0.000 | 0.734 | 90.000 | 40.942 |
| 16 | 0.910 | 1.758 | -2.737 | -0.069 | 0.000 | 251.963 | 198.172 | 50.000 | 0.000 | 0.734 | 90.000 | 41.977 |
| 17 | 1.012 | 1.878 | -2.998 | -0.108 | 0.000 | 265.561 | 208.254 | 50.000 | 0.000 | 0.734 | 90.000 | 43.087 |
| 18 | 1.114 | 1.975 | -3.235 | -0.146 | 0.000 | 278.410 | 217.818 | 50.000 | 0.000 | 0.735 | 90.000 | 44.067 |
| 19 | 1.311 | 2.119 | -3.682 | -0.252 | 0.000 | 296.519 | 231.382 | 50.000 | 0.000 | 0.736 | 90.000 | 45.350 |
| 20 | 1.510 | 2.288 | -4.168 | -0.370 | 0.000 | 313.438 | 244.030 | 50.000 | 0.000 | 0.737 | 90.000 | 46.456 |
| 21 | 1.724 | 2.432 | -4.673 | -0.517 | 0.000 | 325.920 | 253.467 | 50.000 | 0.000 | 0.737 | 90.000 | 47.222 |
| 22 | 1.907 | 2.504 | -5.058 | -0.646 | 0.000 | 335.985 | 261.204 | 50.000 | 0.000 | 0.739 | 90.000 | 47.810 |
| 23 | 2.116 | 2.649 | -5.558 | -0.793 | 0.000 | 346.588 | 265.416 | 50.000 | 0.000 | 0.726 | 90.000 | 48.404 |
| 24 | 2.204 | 2.721 | -5.780 | -0.855 | 0.000 | 351.769 | 268.270 | 50.000 | 0.000 | 0.723 | 90.000 | 48.686 |
| 25 | 2.434 | 2.890 | -6.332 | -1.008 | 0.000 | 366.033 | 276.964 | 50.000 | 0.000 | 0.718 | 90.000 | 49.432 |
| 26 | 2.531 | 2.962 | -6.557 | -1.064 | 0.000 | 368.030 | 278.260 | 50.000 | 0.000 | 0.718 | 90.000 | 49.533 |
| 27 | 2.623 | 3.013 | -6.764 | -1.129 | 0.000 | 366.886 | 278.032 | 50.000 | 0.000 | 0.720 | 90.000 | 49.475 |
| 28 | 2.804 | 3.112 | -7.153 | -1.236 | 0.000 | 339.765 | 258.041 | 50.000 | 0.000 | 0.718 | 90.000 | 48.025 |
| 29 | 2.917 | 3.159 | -7.351 | -1.275 | 0.000 | 318.638 | 241.570 | 50.000 | 0.000 | 0.713 | 90.000 | 46.780 |
| 30 | 3.004 | 3.159 | -7.463 | -1.301 | 0.000 | 308.692 | 234.690 | 50.000 | 0.000 | 0.714 | 90.000 | 46.154 |
| 31 | 3.122 | 3.159 | -7.601 | -1.320 | 0.000 | 297.676 | 226.954 | 50.000 | 0.000 | 0.714 | 90.000 | 45.429 |
| 32 | 3.361 | 3.205 | -7.915 | -1.348 | 0.000 | 291.915 | 223.273 | 50.000 | 0.000 | 0.716 | 90.000 | 45.034 |
| 33 | 3.633 | 3.205 | -8.208 | -1.370 | 0.000 | 294.345 | 224.321 | 50.000 | 0.000 | 0.713 | 90.000 | 45.202 |

# True Triaxial Test on Fine Nevada Sand with a Target Void Ratio $=\mathbf{0 . 5 3 0}\left(\mathrm{D}_{\mathrm{r}}=\mathbf{9 1 . 2 8 \%}\right)$ 

| Test No.: | 17 |
| :--- | :--- |
| Test Date: | $3 / 11 / 11$ |
| Sector: | III |
| Initial Void Ratio, e: | 0.533 |
| Final Mean Princpal Stress, $\sigma_{\mathrm{m}}:$ | 238.7 kPa |
| Max Friction Angle, $\phi:$ | 50.1 deg |
| b-value at failure: | 0.72 |
| Stress direction at failure, $\alpha:$ | 90.0 deg |


| Point (No.) | $\begin{gathered} \varepsilon_{\mathrm{z}} \\ (\%) \end{gathered}$ | $\begin{gathered} \varepsilon_{\mathrm{x}} \\ (\%) \end{gathered}$ | $\begin{array}{r} \varepsilon_{y} \\ (\%) \\ \hline \end{array}$ | $\begin{gathered} \varepsilon_{\mathrm{v}} \\ (\%) \end{gathered}$ | $\begin{gathered} \gamma_{\theta z} \\ (\%) \end{gathered}$ | $\begin{gathered} \sigma_{z} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \sigma_{\mathrm{x}} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \sigma_{\mathrm{y}} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \tau_{\text {өz }} \\ (\mathrm{kPa}) \end{gathered}$ | b | $\alpha$ <br> $\left({ }^{\circ}\right)$ | $\begin{gathered} \varphi \\ \left({ }^{\circ}\right) \\ \hline \end{gathered}$ |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 1 | 0.000 | 0.000 | 0.000 | 0.000 | 0.000 | 51.610 | 51.020 | 50.000 | 0.000 | 0.633 | 90.000 | 0.908 |
| 2 | 0.012 | 0.000 | -0.010 | 0.002 | 0.000 | 59.871 | 59.282 | 50.000 | 0.000 | 0.940 | 90.000 | 5.155 |
| 3 | 0.024 | 0.000 | -0.015 | 0.009 | 0.000 | 67.054 | 66.467 | 50.000 | 0.000 | 0.966 | 90.000 | 8.377 |
| 4 | 0.026 | 0.000 | -0.009 | 0.017 | 0.000 | 74.238 | 73.654 | 50.000 | 0.000 | 0.976 | 90.000 | 11.250 |
| 5 | 0.028 | 0.000 | -0.002 | 0.026 | 0.000 | 81.424 | 80.842 | 50.000 | 0.000 | 0.981 | 90.000 | 13.834 |
| 6 | 0.029 | 0.000 | 0.006 | 0.034 | 0.000 | 90.407 | 89.828 | 50.000 | 0.000 | 0.986 | 90.000 | 16.725 |
| 7 | 0.030 | 0.000 | 0.006 | 0.037 | 0.000 | 98.310 | 97.734 | 50.000 | 0.000 | 0.988 | 90.000 | 19.010 |
| 8 | 0.032 | 0.000 | 0.006 | 0.039 | 0.000 | 105.136 | 104.563 | 50.000 | 0.000 | 0.990 | 90.000 | 20.818 |
| 9 | 0.037 | 0.252 | -0.248 | 0.041 | 0.000 | 110.882 | 110.162 | 50.000 | 0.000 | 0.988 | 90.000 | 22.236 |
| 10 | 0.106 | 0.300 | -0.358 | 0.047 | 0.000 | 142.438 | 141.514 | 50.000 | 0.000 | 0.990 | 90.000 | 28.708 |
| 11 | 0.204 | 0.710 | -0.885 | 0.028 | 0.000 | 172.814 | 171.582 | 50.000 | 0.000 | 0.990 | 90.000 | 33.449 |
| 12 | 0.305 | 1.055 | -1.365 | -0.004 | 0.000 | 195.578 | 194.043 | 50.000 | 0.000 | 0.989 | 90.000 | 36.356 |
| 13 | 0.404 | 1.300 | -1.752 | -0.047 | 0.000 | 216.128 | 214.214 | 50.000 | 0.000 | 0.988 | 90.000 | 38.626 |
| 14 | 0.508 | 1.539 | -2.155 | -0.108 | 0.000 | 234.443 | 232.132 | 50.000 | 0.000 | 0.987 | 90.000 | 40.424 |
| 15 | 0.605 | 1.785 | -2.566 | -0.177 | 0.000 | 251.990 | 249.212 | 50.000 | 0.000 | 0.986 | 90.000 | 41.979 |
| 16 | 0.703 | 1.977 | -2.935 | -0.254 | 0.000 | 270.884 | 267.691 | 50.000 | 0.000 | 0.986 | 90.000 | 43.500 |
| 17 | 0.813 | 2.216 | -3.391 | -0.362 | 0.000 | 282.524 | 275.632 | 50.000 | 0.000 | 0.970 | 90.000 | 44.368 |
| 18 | 0.916 | 2.362 | -3.735 | -0.457 | 0.000 | 301.603 | 289.808 | 50.000 | 0.000 | 0.953 | 90.000 | 45.691 |
| 19 | 1.012 | 2.508 | -4.028 | -0.508 | 0.000 | 319.685 | 289.728 | 50.000 | 0.000 | 0.889 | 90.000 | 46.844 |
| 20 | 1.104 | 2.554 | -4.227 | -0.569 | 0.000 | 332.028 | 289.493 | 50.000 | 0.000 | 0.849 | 90.000 | 47.582 |
| 21 | 1.204 | 2.654 | -4.492 | -0.633 | 0.000 | 343.608 | 287.411 | 50.000 | 0.000 | 0.809 | 90.000 | 48.240 |
| 22 | 1.323 | 2.700 | -4.735 | -0.711 | 0.000 | 353.992 | 288.445 | 50.000 | 0.000 | 0.784 | 90.000 | 48.805 |
| 23 | 1.422 | 2.747 | -4.948 | -0.780 | 0.000 | 363.370 | 288.716 | 50.000 | 0.000 | 0.762 | 90.000 | 49.296 |
| 24 | 1.621 | 2.840 | -5.378 | -0.918 | 0.000 | 374.668 | 286.771 | 50.000 | 0.000 | 0.729 | 90.000 | 49.864 |
| 25 | 1.711 | 2.840 | -5.524 | -0.974 | 0.000 | 375.619 | 287.333 | 50.000 | 0.000 | 0.729 | 90.000 | 49.911 |
| 26 | 1.825 | 2.886 | -5.754 | -1.043 | 0.000 | 378.562 | 287.507 | 50.000 | 0.000 | 0.723 | 90.000 | 50.055 |
| 27 | 1.904 | 2.886 | -5.902 | -1.111 | 0.000 | 372.167 | 287.366 | 50.000 | 0.000 | 0.737 | 90.000 | 49.741 |
| 28 | 2.011 | 2.886 | -6.064 | -1.167 | 0.000 | 348.659 | 263.839 | 50.000 | 0.000 | 0.716 | 90.000 | 48.518 |
| 29 | 2.101 | 2.886 | -6.184 | -1.198 | 0.000 | 332.298 | 253.455 | 50.000 | 0.000 | 0.721 | 90.000 | 47.597 |
| 30 | 2.248 | 2.886 | -6.364 | -1.230 | 0.000 | 316.522 | 240.497 | 50.000 | 0.000 | 0.715 | 90.000 | 46.649 |
| 31 | 2.453 | 2.886 | -6.589 | -1.249 | 0.000 | 310.754 | 237.911 | 50.000 | 0.000 | 0.721 | 90.000 | 46.286 |
| 32 | 2.529 | 2.886 | -6.669 | -1.254 | 0.000 | 312.291 | 237.657 | 50.000 | 0.000 | 0.715 | 90.000 | 46.384 |
| 33 | 2.622 | 2.886 | -6.766 | -1.258 | 0.000 | 315.170 | 237.665 | 50.000 | 0.000 | 0.708 | 90.000 | 46.565 |

## True Triaxial Test on Fine Nevada Sand with a Target Void Ratio $=\mathbf{0 . 5 3 0}\left(\mathrm{D}_{\mathrm{r}}=\mathbf{9 1 . 2 8 \%}\right)$

| Test No.: | 18 |
| :--- | :---: |
| Test Date: | $3 / 18 / 11$ |
| Sector: | III |
| Initial Void Ratio, e: | 0.526 |
| Final Mean Princpal Stress, $\sigma_{\mathrm{m}}:$ | 224.0 kPa |
| Max Friction Angle, $\phi:$ | 46.8 deg |
| b-value at failure: | 0.95 |
| Stress direction at failure, $\alpha:$ | 90.0 deg |


| Shear Band Notes <br> Point of Observation: <br> Inclination (from Vertical): <br> none observed |
| :--- |
| Failure Notes: |
| vacuum was not held at failure |


| Point (No.) | $\begin{gathered} \varepsilon_{\mathrm{z}} \\ (\%) \end{gathered}$ | $\begin{gathered} \varepsilon_{\mathrm{x}} \\ (\%) \end{gathered}$ | $\begin{gathered} \varepsilon_{\mathrm{y}} \\ (\%) \end{gathered}$ | $\begin{gathered} \varepsilon_{\mathrm{v}} \\ (\%) \end{gathered}$ | $\begin{aligned} & \gamma_{\theta z} \\ & (\%) \end{aligned}$ | $\begin{gathered} \sigma_{\mathrm{z}} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \sigma_{x} \\ (\mathrm{kPa}) \end{gathered}$ | $\underset{(\mathrm{kPa})}{\sigma_{\mathrm{y}}}$ | $\begin{gathered} \boldsymbol{\tau}_{\theta z} \\ (\mathrm{kPa}) \end{gathered}$ | b | $\begin{gathered} \alpha \\ \left({ }^{\circ}\right) \end{gathered}$ | $\begin{gathered} \varphi \\ \left({ }^{\circ}\right) \end{gathered}$ |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 1 | 0.000 | 0.000 | 0.000 | 0.000 | 0.000 | 51.611 | 51.611 | 50.000 | 0.000 | 1.000 | 90.000 | 0.908 |
| 2 | 0.001 | 0.000 | -0.001 | 0.000 | 0.000 | 53.048 | 51.389 | 50.000 | 0.000 | 0.456 | 90.000 | 1.695 |
| 3 | 0.009 | 0.000 | -0.007 | 0.002 | 0.000 | 58.077 | 58.078 | 50.000 | 0.000 | 1.000 | 90.000 | 4.286 |
| 4 | 0.017 | 0.000 | -0.013 | 0.004 | 0.000 | 63.106 | 63.108 | 50.000 | 0.000 | 1.000 | 90.000 | 6.654 |
| 5 | 0.023 | 0.000 | -0.016 | 0.008 | 0.000 | 68.854 | 68.858 | 50.000 | 0.000 | 1.000 | 90.000 | 9.127 |
| 6 | 0.031 | 0.000 | -0.020 | 0.011 | 0.000 | 73.882 | 73.889 | 50.000 | 0.000 | 1.000 | 90.000 | 11.115 |
| 7 | 0.036 | 0.000 | -0.019 | 0.017 | 0.000 | 78.193 | 78.203 | 50.000 | 0.000 | 1.000 | 90.000 | 12.705 |
| 8 | 0.043 | 0.000 | -0.015 | 0.028 | 0.000 | 84.301 | 84.315 | 50.000 | 0.000 | 1.000 | 90.000 | 14.797 |
| 9 | 0.052 | 0.297 | -0.310 | 0.039 | 0.000 | 89.690 | 89.593 | 50.000 | 0.000 | 0.998 | 90.000 | 16.507 |
| 10 | 0.105 | 0.519 | -0.543 | 0.081 | 0.000 | 112.676 | 112.416 | 50.000 | 0.000 | 0.996 | 90.000 | 22.661 |
| 11 | 0.165 | 0.631 | -0.691 | 0.105 | 0.000 | 129.173 | 128.803 | 50.000 | 0.000 | 0.995 | 90.000 | 26.224 |
| 12 | 0.201 | 0.779 | -0.866 | 0.114 | 0.000 | 138.487 | 137.974 | 50.000 | 0.000 | 0.994 | 90.000 | 27.999 |
| 13 | 0.258 | 0.928 | -1.066 | 0.120 | 0.000 | 149.927 | 149.256 | 50.000 | 0.000 | 0.993 | 90.000 | 29.988 |
| 14 | 0.316 | 1.076 | -1.271 | 0.121 | 0.000 | 161.351 | 160.502 | 50.000 | 0.000 | 0.992 | 90.000 | 31.793 |
| 15 | 0.358 | 1.187 | -1.427 | 0.119 | 0.000 | 169.550 | 166.798 | 50.000 | 0.000 | 0.977 | 90.000 | 32.992 |
| 16 | 0.402 | 1.262 | -1.552 | 0.111 | 0.000 | 175.584 | 172.514 | 50.000 | 0.000 | 0.976 | 90.000 | 33.828 |
| 17 | 0.461 | 1.341 | -1.700 | 0.102 | 0.000 | 185.528 | 182.432 | 50.000 | 0.000 | 0.977 | 90.000 | 35.129 |
| 18 | 0.511 | 1.487 | -1.908 | 0.090 | 0.000 | 192.968 | 189.604 | 50.000 | 0.000 | 0.976 | 90.000 | 36.045 |
| 19 | 0.646 | 1.832 | -2.431 | 0.047 | 0.000 | 217.035 | 212.697 | 50.000 | 0.000 | 0.974 | 90.000 | 38.720 |
| 20 | 0.678 | 1.932 | -2.565 | 0.045 | 0.000 | 221.987 | 216.994 | 50.000 | 0.000 | 0.971 | 90.000 | 39.223 |
| 21 | 0.762 | 2.131 | -2.882 | 0.011 | 0.000 | 236.793 | 231.006 | 50.000 | 0.000 | 0.969 | 90.000 | 40.641 |
| 22 | 0.858 | 2.377 | -3.269 | -0.034 | 0.000 | 243.688 | 236.763 | 50.000 | 0.000 | 0.964 | 90.000 | 41.262 |
| 23 | 0.906 | 2.430 | -3.391 | -0.056 | 0.000 | 252.468 | 246.016 | 50.000 | 0.000 | 0.968 | 90.000 | 42.020 |
| 24 | 1.059 | 2.729 | -3.925 | -0.137 | 0.000 | 267.331 | 259.597 | 50.000 | 0.000 | 0.964 | 90.000 | 43.225 |
| 25 | 1.200 | 2.981 | -4.430 | -0.249 | 0.000 | 276.417 | 267.940 | 50.000 | 0.000 | 0.963 | 90.000 | 43.919 |
| 26 | 1.281 | 3.127 | -4.721 | -0.313 | 0.000 | 283.566 | 273.971 | 50.000 | 0.000 | 0.959 | 90.000 | 44.444 |
| 27 | 1.404 | 3.326 | -5.148 | -0.418 | 0.000 | 294.759 | 284.931 | 50.000 | 0.000 | 0.960 | 90.000 | 45.230 |
| 28 | 1.561 | 3.625 | -5.752 | -0.566 | 0.000 | 312.448 | 299.355 | 50.000 | 0.000 | 0.950 | 90.000 | 46.394 |
| 29 | 1.665 | 3.970 | -6.381 | -0.746 | 0.000 | 318.516 | 304.524 | 50.000 | 0.000 | 0.948 | 90.000 | 46.773 |
| 30 | 1.706 | 4.169 | -6.724 | -0.849 | 0.000 | 318.209 | 303.812 | 50.000 | 0.000 | 0.946 | 90.000 | 46.754 |
| 31 | 1.771 | 5.590 | -9.091 | -1.730 | 0.000 | 267.555 | 232.427 | 50.000 | 0.000 | 0.839 | 90.000 | 43.243 |
| 32 | 1.858 | 9.123 | -12.233 | -1.252 | 0.000 | 213.973 | 201.835 | 50.000 | 0.000 | 0.926 | 90.000 | 38.402 |

# Triaxial Test on Fine Nevada Sand with a Target Void Ratio $=0.530\left(D_{r}=91.28 \%\right)$ 

| Test No.: | $1, \mathrm{TT} \# 1$ |
| :--- | :---: |
| Test Date: | $3 / 21 / 11$ |
| Sector: | I |
| Initial Void Ratio, e: | 0.522 |
| Final Mean Princpal Stress, $\sigma_{\mathrm{m}}:$ | 121.4 kPa |
| Max Friction Angle, $\phi:$ | 43.0 deg |
| b-value at failure: | 0.00 |
| Stress direction at failure, $\alpha:$ | 0.0 deg |


| Shear Band Notes |  |
| :--- | :--- |
| Point of Observation: | 30 |
| Inclination (from Vertical): |  |
| $51^{\circ}, 58^{\circ}$ |  |
|  |  |
| Failure Notes: |  |


| Point (No.) | $\begin{gathered} \varepsilon_{z} \\ (\%) \\ \hline \end{gathered}$ | $\begin{gathered} \varepsilon_{\mathrm{x}} \\ (\%) \\ \hline \end{gathered}$ | $\begin{array}{r} \varepsilon_{y} \\ (\%) \\ \hline \end{array}$ | $\begin{gathered} \varepsilon_{\mathrm{v}} \\ (\%) \end{gathered}$ | $\begin{array}{r} \gamma_{\theta z} \\ (\%) \\ \hline \end{array}$ | $\begin{gathered} \sigma_{z} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \sigma_{x} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \sigma_{y} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \tau_{\theta z} \\ (\mathrm{kPa}) \end{gathered}$ | b | $\begin{gathered} \alpha \\ \left(^{( }\right) \\ \hline \end{gathered}$ | $\begin{gathered} \varphi \\ \left({ }^{( }\right) \\ \hline \end{gathered}$ |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 1 | 0.000 | 0.000 | 0.000 | 0.000 | 0.000 | 51.627 | 50.000 | 50.000 | 0.000 | 0.000 | 0.000 | 0.917 |
| 2 | 0.014 | -0.005 | -0.005 | 0.004 | 0.000 | 63.297 | 50.000 | 50.000 | 0.000 | 0.000 | 0.000 | 6.740 |
| 3 | 0.041 | -0.014 | -0.014 | 0.013 | 0.000 | 76.423 | 50.000 | 50.000 | 0.000 | 0.000 | 0.000 | 12.064 |
| 4 | 0.126 | -0.045 | -0.045 | 0.037 | 0.000 | 104.831 | 50.000 | 50.000 | 0.000 | 0.000 | 0.000 | 20.741 |
| 5 | 0.201 | -0.075 | -0.075 | 0.050 | 0.000 | 123.372 | 50.000 | 50.000 | 0.000 | 0.000 | 0.000 | 25.037 |
| 6 | 0.421 | -0.181 | -0.181 | 0.059 | 0.000 | 162.830 | 50.000 | 50.000 | 0.000 | 0.000 | 0.000 | 32.015 |
| 7 | 0.620 | -0.296 | -0.296 | 0.028 | 0.000 | 186.502 | 50.000 | 50.000 | 0.000 | 0.000 | 0.000 | 35.252 |
| 8 | 0.809 | -0.415 | -0.415 | -0.020 | 0.000 | 202.092 | 50.000 | 50.000 | 0.000 | 0.000 | 0.000 | 37.108 |
| 9 | 1.053 | -0.582 | -0.582 | -0.11 | 0.000 | 215.641 | 50.000 | 50.000 | 0.000 | 0.000 | 0.000 | 38.576 |
| 10 | 1.21 | -0.693 | -0.693 | -0.17 | 0.00 | 222.463 | 50.000 | 50.000 | 0.000 | 0.000 | 0.000 | 39.270 |
| 11 | 1.421 | -0.844 | -0.844 | -0.268 | 0.000 | 230.904 | 50.000 | 50.000 | 0.000 | 0.000 | 0.000 | 40.091 |
| 12 | 1.637 | -1.007 | -1.007 | -0.376 | 0.000 | 237.817 | 50.000 | 50.000 | 0.000 | 0.000 | 0.000 | 40.735 |
| 13 | 1.811 | -1.140 | -1.140 | -0.470 | 0.000 | 243.016 | 50.000 | 50.000 | 0.000 | 0.000 | 0.000 | 41.202 |
| 14 | 2.007 | -1.292 | -1.292 | -0.577 | 0.000 | 248.111 | 50.000 | 50.000 | 0.000 | 0.000 | 0.000 | 41.648 |
| 15 | 2.237 | -1.476 | -1.476 | -0.716 | 0.000 | 250.914 | 50.000 | 50.000 | 0.000 | 0.000 | 0.000 | 41.888 |
| 16 | 2.421 | -1.618 | -1.618 | -0.816 | 0.000 | 254.573 | 50.000 | 50.000 | 0.000 | 0.000 | 0.000 | 42.196 |
| 17 | 2.611 | -1.772 | -1.772 | -0.933 | 0.000 | 256.402 | 50.000 | 50.000 | 0.000 | 0.000 | 0.000 | 42.348 |
| 18 | 2.806 | -1.931 | -1.931 | -1.055 | 0.000 | 258.896 | 50.000 | 50.000 | 0.000 | 0.000 | 0.000 | 42.553 |
| 19 | 3.053 | -2.130 | -2.130 | -1.208 | 0.000 | 259.450 | 50.000 | 50.000 | 0.000 | 0.000 | 0.000 | 42.598 |
| 20 | 3.211 | -2.255 | -2.255 | -1.299 | 0.000 | 261.012 | 50.000 | 50.000 | 0.000 | 0.000 | 0.000 | 42.724 |
| 21 | 3.421 | -2.423 | -2.423 | -1.425 | 0.000 | 263.069 | 50.000 | 50.000 | 0.000 | 0.000 | 0.000 | 42.889 |
| 22 | 3.632 | -2.594 | -2.594 | -1.556 | 0.000 | 263.024 | 50.000 | 50.000 | 0.000 | 0.000 | 0.000 | 42.886 |
| 23 | 3.842 | -2.764 | -2.764 | -1.686 | 0.000 | 264.355 | 50.000 | 50.000 | 0.000 | 0.000 | 0.000 | 42.991 |
| 24 | 4.135 | -3.000 | -3.000 | -1.865 | 0.000 | 262.299 | 50.000 | 50.000 | 0.000 | 0.000 | 0.000 | 42.828 |
| 25 | 4.263 | -3.101 | -3.101 | -1.939 | 0.000 | 263.231 | 50.000 | 50.000 | 0.000 | 0.000 | 0.000 | 42.902 |
| 26 | 4.421 | -3.223 | -3.223 | -2.026 | 0.000 | 263.039 | 50.000 | 50.000 | 0.000 | 0.000 | 0.000 | 42.887 |
| 27 | 4.668 | -3.417 | -3.417 | -2.165 | 0.000 | 261.858 | 50.000 | 50.000 | 0.000 | 0.000 | 0.000 | 42.792 |
| 28 | 4.881 | -3.584 | -3.584 | -2.287 | 0.000 | 261.135 | 50.000 | 50.000 | 0.000 | 0.000 | 0.000 | 42.734 |
| 29 | 5.026 | -3.696 | -3.696 | -2.365 | 0.000 | 258.281 | 50.000 | 50.000 | 0.000 | 0.000 | 0.000 | 42.502 |
| 30 | 5.250 | -3.866 | -3.866 | -2.482 | 0.000 | 257.552 | 50.000 | 50.000 | 0.000 | 0.000 | 0.000 | 42.443 |

# Triaxial Test on Fine Nevada Sand with a Target Void Ratio $=0.530\left(D_{r}=91.28 \%\right)$ 

| Test No.: | 2 |
| :--- | :---: |
| Test Date: | $4 / 1 / 11$ |
| Sector: | I |
| Initial Void Ratio, e: | 0.537 |
| Final Mean Princpal Stress, $\sigma_{\mathrm{m}}:$ | 69.1 kPa |
| Max Friction Angle, $\phi:$ | 46.5 deg |
| b-value at failure: | 0.00 |
| Stress direction at failure, $\alpha$ : | 0.0 deg |


| Shear Band Notes |
| :--- |
| Point of Observation: |
|  |
| Inclination (from Vertical): |
| $57.4^{\circ}, 56.9^{\circ}$ |
|  |
| Failure Notes: |


| Point (No.) | $\begin{gathered} \varepsilon_{\mathrm{z}} \\ (\%) \end{gathered}$ | $\begin{gathered} \varepsilon_{\mathrm{x}} \\ (\%) \end{gathered}$ | $\begin{array}{r} \varepsilon_{y} \\ (\%) \\ \hline \end{array}$ | $\begin{gathered} \varepsilon_{\mathrm{v}} \\ (\%) \\ \hline \end{gathered}$ | $\begin{array}{r} \gamma_{\theta z} \\ (\%) \\ \hline \end{array}$ | $\begin{gathered} \sigma_{z} \\ (\mathrm{kPa}) \\ \hline \end{gathered}$ | $\begin{gathered} \sigma_{x} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \sigma_{y} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \tau_{\theta z} \\ (\mathrm{kPa}) \end{gathered}$ | b | $\begin{gathered} \alpha \\ \left({ }^{\circ}\right) \\ \hline \end{gathered}$ | $\begin{gathered} \varphi \\ \left(^{\circ}\right) \\ \hline \end{gathered}$ |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 1 | 0.000 | 0.000 | 0.000 | 0.000 | 0.000 | 27.190 | 25.000 | 25.000 | 0.000 | 0.000 | 0.000 | 2.405 |
| 2 | 0.001 | 0.002 | 0.002 | 0.004 | 0.000 | 30.315 | 25.000 | 25.000 | 0.000 | 0.000 | 0.000 | 5.514 |
| 3 | 0.002 | 0.001 | 0.001 | 0.004 | 0.000 | 34.024 | 25.000 | 25.000 | 0.000 | 0.000 | 0.000 | 8.795 |
| 4 | 0.012 | -0.002 | -0.002 | 0.009 | 0.000 | 41.443 | 25.000 | 25.000 | 0.000 | 0.000 | 0.000 | 14.328 |
| 5 | 0.024 | -0.005 | -0.005 | 0.013 | 0.000 | 46.263 | 25.000 | 25.000 | 0.000 | 0.000 | 0.000 | 17.360 |
| 6 | 0.037 | -0.010 | -0.010 | 0.018 | 0.000 | 51.083 | 25.000 | 25.000 | 0.000 | 0.000 | 0.000 | 20.049 |
| 7 | 0.230 | -0.095 | -0.095 | 0.040 | 0.000 | 86.278 | 25.000 | 25.000 | 0.000 | 0.000 | 0.000 | 33.413 |
| 8 | 0.421 | -0.201 | -0.201 | 0.020 | 0.000 | 102.416 | 25.000 | 25.000 | 0.000 | 0.000 | 0.000 | 37.415 |
| 9 | 0.737 | -0.394 | -0.394 | -0.051 | 0.000 | 119.064 | 25.000 | 25.000 | 0.000 | 0.000 | 0.000 | 40.763 |
| 10 | 0.842 | -0.465 | -0.465 | -0.089 | 0.000 | 123.347 | 25.000 | 25.000 | 0.000 | 0.000 | 0.000 | 41.525 |
| 11 | 1.005 | -0.575 | -0.575 | -0.14 | 0.00 | 129.740 | 25.000 | 25.000 | 0.000 | 0.000 | 0.000 | 42.600 |
| 12 | 1.263 | -0.760 | -0.760 | -0.25 | 0.00 | 135.946 | 25.000 | 25.000 | 0.000 | 0.000 | 0.000 | 43.578 |
| 13 | 1.477 | -0.918 | -0.918 | -0.358 | 0.00 | 141.436 | 25.000 | 25.000 | 0.000 | 0.000 | 0.000 | 44.394 |
| 14 | 1.615 | -1.018 | -1.018 | -0.420 | 0.00 | 144.483 | 25.000 | 25.000 | 0.000 | 0.000 | 0.000 | 44.828 |
| 15 | 1.806 | -1.158 | -1.158 | -0.509 | 0.000 | 147.783 | 25.000 | 25.000 | 0.000 | 0.000 | 0.000 | 45.285 |
| 16 | 2.065 | -1.362 | -1.362 | -0.659 | 0.000 | 149.463 | 25.000 | 25.000 | 0.000 | 0.000 | 0.000 | 45.513 |
| 17 | 2.204 | -1.469 | -1.469 | -0.735 | 0.000 | 152.085 | 25.000 | 25.000 | 0.000 | 0.000 | 0.000 | 45.861 |
| 18 | 2.603 | -1.775 | -1.775 | -0.947 | 0.000 | 154.195 | 25.000 | 25.000 | 0.000 | 0.000 | 0.000 | 46.135 |
| 19 | 2.737 | -1.880 | -1.880 | -1.022 | 0.000 | 155.360 | 25.000 | 25.000 | 0.000 | 0.000 | 0.000 | 46.284 |
| 20 | 2.921 | -2.020 | -2.020 | -1.120 | 0.000 | 156.427 | 25.000 | 25.000 | 0.000 | 0.000 | 0.000 | 46.420 |
| 21 | 3.191 | -2.235 | -2.235 | -1.279 | 0.000 | 156.232 | 25.000 | 25.000 | 0.000 | 0.000 | 0.000 | 46.395 |
| 22 | 3.290 | -2.311 | -2.311 | -1.332 | 0.000 | 156.390 | 25.000 | 25.000 | 0.000 | 0.000 | 0.000 | 46.415 |
| 23 | 3.427 | -2.415 | -2.415 | -1.403 | 0.000 | 157.183 | 25.000 | 25.000 | 0.000 | 0.000 | 0.000 | 46.515 |
| 24 | 3.961 | -2.828 | -2.828 | -1.695 | 0.000 | 156.468 | 25.000 | 25.000 | 0.000 | 0.000 | 0.000 | 46.425 |
| 25 | 4.095 | -2.926 | -2.926 | -1.757 | 0.000 | 156.564 | 25.000 | 25.000 | 0.000 | 0.000 | 0.000 | 46.437 |
| 26 | 4.282 | -3.077 | -3.077 | -1.872 | 0.000 | 153.734 | 25.000 | 25.000 | 0.000 | 0.000 | 0.000 | 46.075 |
| 27 | 4.528 | -3.260 | -3.260 | -1.991 | 0.000 | 152.924 | 25.000 | 25.000 | 0.000 | 0.000 | 0.000 | 45.970 |
| 28 | 4.737 | -3.406 | -3.406 | -2.076 | 0.000 | 145.976 | 25.000 | 25.000 | 0.000 | 0.000 | 0.000 | 45.037 |
| 29 | 4.911 | -3.524 | -3.524 | -2.138 | 0.000 | 134.643 | 25.000 | 25.000 | 0.000 | 0.000 | 0.000 | 43.377 |
| 30 | 5.191 | -3.691 | -3.691 | -2.191 | 0.000 | 124.300 | 25.000 | 25.000 | 0.000 | 0.000 | 0.000 | 41.690 |
| 31 | 5.439 | -3.828 | -3.828 | -2.217 | 0.000 | 120.254 | 25.000 | 25.000 | 0.000 | 0.000 | 0.000 | 40.978 |
| 32 | 6.050 | -4.151 | -4.151 | -2.253 | 0.000 | 117.254 | 25.000 | 25.000 | 0.000 | 0.000 | 0.000 | 40.430 |
| 33 | 6.664 | -4.467 | -4.467 | -2.270 | 0.000 | 116.330 | 25.000 | 25.000 | 0.000 | 0.000 | 0.000 | 40.257 |

# Triaxial Test on Fine Nevada Sand with a Target Void Ratio $=0.530\left(D_{r}=91.28 \%\right)$ 

| Test No.: | 3 |  |
| :--- | :--- | :---: |
| Test Date: | $4 / 4 / 11$ |  |
| Sector: | I |  |
| Initial Void Ratio, e: | 0.542 |  |
| Final Mean Princpal Stress, $\sigma_{\mathrm{m}}:$ | 284.6 kPa |  |
| Max Friction Angle, $\phi:$ | 39.9 deg |  |
| b-value at failure: | 0.00 |  |
| Stress direction at failure, $\alpha:$ | 0.0 deg |  |


| Shear Band Notes |  |
| :--- | :--- |
| Point of Observation: | 26 |
|  |  |
| Inclination (from Vertical): |  |
| $57.0^{\circ}, 58.9^{\circ}$ |  |
|  |  |
| Failure Notes: |  |


| Point (No.) | $\begin{gathered} \varepsilon_{\mathrm{z}} \\ (\%) \\ \hline \end{gathered}$ | $\begin{gathered} \varepsilon_{\mathrm{x}} \\ (\%) \\ \hline \end{gathered}$ | $\begin{gathered} \varepsilon_{y} \\ (\%) \end{gathered}$ | $\begin{gathered} \varepsilon_{\mathrm{v}} \\ (\%) \\ \hline \end{gathered}$ | $\begin{gathered} \gamma_{\theta z} \\ (\%) \\ \hline \end{gathered}$ | $\begin{gathered} \sigma_{z} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \sigma_{x} \\ (\mathrm{kPa}) \\ \hline \end{gathered}$ | $\begin{gathered} \sigma_{y} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \tau_{\text {өz }} \\ (\mathrm{kPa}) \\ \hline \end{gathered}$ | b | $\begin{gathered} \alpha \\ \left(^{( }\right) \\ \hline \end{gathered}$ | $\begin{gathered} \varphi \\ \left(^{\circ}\right) \\ \hline \end{gathered}$ |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 1 | 0.000 | 0.000 | 0.000 | 0.000 | 0.000 | 130.000 | 130.000 | 25.000 | 0.000 | 0.000 | 0.000 | 0.000 |
| 2 | 0.000 | -0.005 | -0.005 | -0.011 | 0.000 | 151.214 | 130.000 | 25.000 | 0.000 | 0.000 | 0.000 | 5.45 |
| 3 | 0.000 | -0.015 | -0.015 | -0.029 | 0.000 | 187.606 | 130.000 | 25.000 | 0.000 | 0.000 | 0.000 | 10.3 |
| 4 | 0.000 | -0.030 | -0.030 | -0.060 | 0.000 | 217.430 | 130.000 | 25.000 | 0.000 | 0.000 | 0.000 | 15.7 |
| 5 | 0.039 | -0.041 | -0.041 | -0.043 | 0.000 | 238.876 | 130.000 | 25.000 | 0.000 | 0.000 | 0.000 | 18.2 |
| 6 | 0.017 | -0.045 | -0.045 | -0.073 | 0.000 | 250.143 | 130.000 | 25.000 | 0.000 | 0.000 | 0.000 | 19.5 |
| 7 | 0.205 | -0.062 | -0.062 | 0.081 | 0.000 | 276.283 | 130.000 | 25.000 | 0.000 | 0.000 | 0.000 | 21.103 |
| 8 | 0.412 | -0.150 | -0.150 | 0.112 | 0.000 | 351.891 | 130.000 | 25.000 | 0.000 | 0.000 | 0.000 | 27.417 |
| 9 | 0.619 | -0.253 | -0.253 | 0.114 | 0.000 | 406.486 | 130.000 | 25.000 | 0.000 | 0.000 | 0.000 | 31.022 |
| 10 | 0.875 | -0.394 | -0.394 | 0.087 | 0.000 | 453.011 | 130.000 | 25.000 | 0.000 | 0.000 | 0.000 | 33.645 |
| 11 | 1.130 | -0.548 | -0.548 | 0.035 | 0.000 | 486.208 | 130.000 | 25.000 | 0.000 | 0.000 | 0.000 | 35.315 |
| 12 | 1.417 | -0.73 | -0.739 | -0.06 | 0.000 | 513.871 | 130.000 | 25.000 | 0.000 | 0.000 | 0.000 | 36.598 |
| 13 | 1.656 | -0.896 | -0.896 | -0.13 | 0.000 | 533.381 | 130.000 | 25.000 | 0.000 | 0.000 | 0.000 | 37.450 |
| 14 | 1.853 | -1.029 | -1.029 | -0.205 | 0.000 | 546.547 | 130.000 | 25.000 | 0.000 | 0.000 | 0.000 | 38.002 |
| 15 | 2.227 | -1.292 | -1.292 | -0.358 | 0.000 | 564.165 | 130.000 | 25.000 | 0.000 | 0.000 | 0.000 | 38.715 |
| 16 | 3.145 | -1.966 | -1.966 | -0.786 | 0.000 | 592.484 | 130.000 | 25.000 | 0.000 | 0.000 | 0.000 | 39.802 |
| 17 | 3.514 | -2.238 | -2.238 | -0.961 | 0.000 | 593.722 | 130.000 | 25.000 | 0.000 | 0.000 | 0.000 | 39.847 |
| 18 | 3.966 | -2.581 | -2.581 | -1.197 | 0.000 | 593.897 | 130.000 | 25.000 | 0.000 | 0.000 | 0.000 | 39.854 |
| 19 | 4.231 | -2.779 | -2.779 | -1.328 | 0.000 | 592.688 | 130.000 | 25.000 | 0.000 | 0.000 | 0.000 | 39.809 |
| 20 | 4.410 | -2.921 | -2.921 | -1.433 | 0.000 | 588.587 | 130.000 | 25.000 | 0.000 | 0.000 | 0.000 | 39.656 |
| 21 | 4.725 | -3.158 | -3.158 | -1.590 | 0.000 | 585.317 | 130.000 | 25.000 | 0.000 | 0.000 | 0.000 | 39.533 |
| 22 | 4.843 | -3.247 | -3.247 | -1.651 | 0.000 | 584.471 | 130.000 | 25.000 | 0.000 | 0.000 | 0.000 | 39.501 |
| 23 | 5.053 | -3.398 | -3.398 | -1.743 | 0.000 | 579.305 | 130.000 | 25.000 | 0.000 | 0.000 | 0.000 | 39.305 |
| 24 | 5.293 | -3.573 | -3.573 | -1.852 | 0.000 | 574.286 | 130.000 | 25.000 | 0.000 | 0.000 | 0.000 | 39.112 |
| 25 | 5.437 | -3.682 | -3.682 | -1.926 | 0.000 | 570.236 | 130.000 | 25.000 | 0.000 | 0.000 | 0.000 | 38.954 |
| 26 | 5.641 | -3.812 | -3.812 | -1.983 | 0.000 | 558.920 | 130.000 | 25.000 | 0.000 | 0.000 | 0.000 | 38.506 |
| 27 | 5.929 | -4.011 | -4.011 | -2.092 | 0.000 | 486.676 | 130.000 | 25.000 | 0.000 | 0.000 | 0.000 | 35.337 |
| 28 | 6.036 | -4.075 | -4.075 | -2.114 | 0.000 | 464.747 | 130.000 | 25.000 | 0.000 | 0.000 | 0.000 | 34.252 |
| 29 | 6.355 | -4.248 | -4.248 | -2.141 | 0.000 | 429.453 | 130.000 | 25.000 | 0.000 | 0.000 | 0.000 | 32.362 |
| 30 | 6.639 | -4.391 | -4.391 | -2.143 | 0.000 | 420.202 | 130.000 | 25.000 | 0.000 | 0.000 | 0.000 | 31.833 |
| 31 | 6.903 | -4.522 | -4.522 | -2.141 | 0.000 | 417.716 | 130.000 | 25.000 | 0.000 | 0.000 | 0.000 | 31.689 |
| 32 | 7.106 | -4.621 | -4.621 | -2.136 | 0.000 | 417.751 | 130.000 | 25.000 | 0.000 | 0.000 | 0.000 | 31.691 |
| 33 | 7.446 | -4.788 | -4.788 | -2.130 | 0.000 | 418.019 | 130.000 | 25.000 | 0.000 | 0.000 | 0.000 | 31.706 |

# Triaxial Test on Fine Nevada Sand with a Target Void Ratio $=0.530\left(D_{r}=91.28 \%\right)$ 

| Test No.: | 4 |  |
| :--- | :--- | :---: |
| Test Date: | $4 / 5 / 11$ |  |
| Sector: | I |  |
| Initial Void Ratio, e: | 0.542 |  |
| Final Mean Princpal Stress, $\sigma_{\mathrm{m}}:$ | 159.7 kPa |  |
| Max Friction Angle, $\phi:$ | 41.1 deg |  |
| b-value at failure: | 0.00 |  |
| Stress direction at failure, $\alpha:$ | 0.0 deg |  |


| Shear Band Notes <br> Point of Observation: | 25 |
| :--- | :--- |
| Inclination (from Vertical): |  |
| $53.0^{\circ}, 60.3^{\circ}$ |  |
|  |  |
| Failure Notes: |  |


| Point (No.) | $\begin{gathered} \varepsilon_{z} \\ (\%) \\ \hline \end{gathered}$ | $\begin{gathered} \varepsilon_{\mathrm{x}} \\ (\%) \\ \hline \end{gathered}$ | $\begin{array}{r} \varepsilon_{y} \\ (\%) \\ \hline \end{array}$ | $\begin{gathered} \varepsilon_{\mathrm{v}} \\ (\%) \\ \hline \end{gathered}$ | $\begin{array}{r} \gamma_{\theta z} \\ (\%) \\ \hline \end{array}$ | $\begin{gathered} \sigma_{z} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \sigma_{x} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \sigma_{y} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \tau_{\text {өz }} \\ (\mathrm{kPa}) \end{gathered}$ | b | $\begin{gathered} \alpha \\ \left(^{( }\right) \\ \hline \end{gathered}$ | $\begin{gathered} \varphi \\ \left({ }^{( }\right) \\ \hline \end{gathered}$ |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 1 | 0.000 | 0.000 | 0.000 | 0.000 | 0.000 | 72.151 | 70.000 | 70.000 | 0.000 | 0.000 | 0.000 | 0.867 |
| 2 | 0.015 | -0.005 | -0.005 | 0.004 | 0.000 | 85.797 | 70.000 | 70.000 | 0.000 | 0.000 | 0.000 | 5.820 |
| 3 | 0.043 | -0.013 | -0.013 | 0.017 | 0.000 | 103.636 | 70.000 | 70.000 | 0.000 | 0.000 | 0.000 | 11.170 |
| 4 | 0.079 | -0.022 | -0.022 | 0.035 | 0.000 | 120.012 | 70.000 | 70.000 | 0.000 | 0.000 | 0.000 | 15.260 |
| 5 | 0.132 | -0.040 | -0.040 | 0.052 | 0.000 | 138.918 | 70.000 | 70.000 | 0.000 | 0.000 | 0.000 | 19.262 |
| 6 | 0.206 | -0.068 | -0.068 | 0.070 | 0.000 | 159.611 | 70.000 | 70.000 | 0.000 | 0.000 | 0.000 | 22.972 |
| 7 | 0.461 | -0.180 | -0.180 | 0.100 | 0.000 | 207.675 | 70.000 | 70.000 | 0.000 | 0.000 | 0.000 | 29.723 |
| 8 | 0.637 | -0.277 | -0.277 | 0.083 | 0.000 | 230.226 | 70.000 | 70.000 | 0.000 | 0.000 | 0.000 | 32.255 |
| 9 | 0.815 | -0.372 | -0.372 | 0.070 | 0.000 | 248.353 | 70.000 | 70.000 | 0.000 | 0.000 | 0.000 | 34.072 |
| 10 | 1.00 | -0.494 | -0.494 | 0.017 | 0.00 | 262.342 | 70.000 | 70.000 | 0.00 | 0.000 | 0.000 | 35.362 |
| 11 | 1.26 | -0.654 | -0.654 | -0.044 | 0.00 | 278.614 | 70.000 | 70.000 | 0.000 | 0.000 | 0.000 | 36.756 |
| 12 | 1.467 | -0.784 | -0.784 | -0.100 | 0.000 | 289.900 | 70.000 | 70.000 | 0.000 | 0.000 | 0.000 | 37.662 |
| 13 | 1.602 | -0.873 | -0.873 | -0.144 | 0.000 | 295.229 | 70.000 | 70.000 | 0.000 | 0.000 | 0.000 | 38.074 |
| 14 | 1.886 | -1.063 | -1.063 | -0.240 | 0.000 | 307.196 | 70.000 | 70.000 | 0.000 | 0.000 | 0.000 | 38.965 |
| 15 | 2.023 | -1.175 | -1.175 | -0.328 | 0.000 | 308.793 | 70.000 | 70.000 | 0.000 | 0.000 | 0.000 | 39.080 |
| 16 | 2.259 | -1.348 | -1.348 | -0.437 | 0.000 | 318.236 | 70.000 | 70.000 | 0.000 | 0.000 | 0.000 | 39.747 |
| 17 | 2.609 | -1.610 | -1.610 | -0.612 | 0.000 | 325.734 | 70.000 | 70.000 | 0.000 | 0.000 | 0.000 | 40.258 |
| 18 | 3.112 | -1.984 | -1.984 | -0.856 | 0.000 | 329.742 | 70.000 | 70.000 | 0.000 | 0.000 | 0.000 | 40.525 |
| 19 | 3.700 | -2.420 | -2.420 | -1.140 | 0.000 | 337.151 | 70.000 | 70.000 | 0.000 | 0.000 | 0.000 | 41.007 |
| 20 | 4.148 | -2.764 | -2.764 | -1.380 | 0.000 | 339.067 | 70.000 | 70.000 | 0.000 | 0.000 | 0.000 | 41.129 |
| 21 | 4.727 | -3.207 | -3.207 | -1.686 | 0.000 | 338.002 | 70.000 | 70.000 | 0.000 | 0.000 | 0.000 | 41.061 |
| 22 | 4.813 | -3.280 | -3.280 | -1.747 | 0.000 | 337.259 | 70.000 | 70.000 | 0.000 | 0.000 | 0.000 | 41.014 |
| 23 | 5.058 | -3.446 | -3.446 | -1.835 | 0.000 | 335.664 | 70.000 | 70.000 | 0.000 | 0.000 | 0.000 | 40.911 |
| 24 | 5.412 | -3.697 | -3.697 | -1.983 | 0.000 | 334.626 | 70.000 | 70.000 | 0.000 | 0.000 | 0.000 | 40.844 |
| 25 | 5.493 | -3.756 | -3.756 | -2.018 | 0.000 | 334.307 | 70.000 | 70.000 | 0.000 | 0.000 | 0.000 | 40.823 |
| 26 | 5.929 | -4.041 | -4.041 | -2.154 | 0.000 | 322.677 | 70.000 | 70.000 | 0.000 | 0.000 | 0.000 | 40.051 |
| 27 | 6.167 | -4.173 | -4.173 | -2.180 | 0.000 | 301.238 | 70.000 | 70.000 | 0.000 | 0.000 | 0.000 | 38.527 |
| 28 | 6.278 | -4.229 | -4.229 | -2.180 | 0.000 | 285.930 | 70.000 | 70.000 | 0.000 | 0.000 | 0.000 | 37.349 |
| 29 | 6.476 | -4.330 | -4.330 | -2.184 | 0.000 | 272.464 | 70.000 | 70.000 | 0.000 | 0.000 | 0.000 | 36.242 |
| 30 | 6.781 | -4.483 | -4.483 | -2.184 | 0.000 | 258.514 | 70.000 | 70.000 | 0.000 | 0.000 | 0.000 | 35.018 |
| 31 | 7.117 | -4.651 | -4.651 | -2.184 | 0.000 | 251.545 | 70.000 | 70.000 | 0.000 | 0.000 | 0.000 | 34.375 |
| 32 | 7.337 | -4.760 | -4.760 | -2.184 | 0.000 | 251.116 | 70.000 | 70.000 | 0.000 | 0.000 | 0.000 | 34.334 |
| 33 | 7.945 | -5.065 | -5.065 | -2.184 | 0.000 | 249.598 | 70.000 | 70.000 | 0.000 | 0.000 | 0.000 | 34.191 |

# Triaxial Test on Fine Nevada Sand with a Target Void Ratio $=\mathbf{0 . 5 3 0}\left(\mathrm{D}_{\mathrm{r}}=\mathbf{9 1 . 2 8 \%}\right)$ 

| Test No.: | 5 |
| :--- | :---: |
| Test Date: | $11 / 19 / 10$ |
| Sector: | III |
| Initial Void Ratio, e: | 0.533 |
| Final Mean Princpal Stress, $\sigma_{\mathrm{m}}:$ | 53.9 kPa |
| Max Friction Angle, $\phi:$ | 39.4 deg |
| b-value at failure: | 0.00 |
| Stress direction at failure, $\alpha$ : | 90.0 deg |


| Shear Band Notes |  |
| :--- | :--- |
| Point of Observation: | 29 |
| Inclination (from Vertical): |  |
| $60.0^{\circ}$ |  |
| Failure Notes: |  |


| Point (No.) | $\begin{gathered} \varepsilon_{\mathrm{z}} \\ (\%) \end{gathered}$ | $\begin{gathered} \varepsilon_{\mathrm{x}} \\ (\%) \end{gathered}$ | $\begin{array}{r} \varepsilon_{y} \\ (\%) \end{array}$ | $\begin{gathered} \varepsilon_{\mathrm{v}} \\ (\%) \end{gathered}$ | $\begin{aligned} & \gamma_{\theta z} \\ & (\%) \end{aligned}$ | $\begin{gathered} \sigma_{z} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \sigma_{x} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \sigma_{\mathrm{y}} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \tau_{\text {өz }} \\ (\mathrm{kPa}) \end{gathered}$ | b | $\begin{gathered} \alpha \\ \left(^{\circ}\right) \\ \hline \end{gathered}$ | $\begin{gathered} \varphi \\ \left({ }^{\circ}\right) \\ \hline \end{gathered}$ |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 1 | 0.000 | 0.000 | 0.000 | 0.000 | 0.000 | 25.000 | 25.000 | 25.000 | 0.000 | 0.000 | 90.000 | 0.000 |
| 2 | 0.017 | -0.008 | -0.008 | 0.000 | 0.000 | 31.377 | 25.000 | 25.000 | 0.000 | 0.000 | 90.000 | 6.495 |
| 3 | 0.036 | -0.015 | -0.015 | 0.007 | 0.000 | 39.043 | 25.000 | 25.000 | 0.000 | 0.000 | 90.000 | 12.666 |
| 4 | 0.056 | -0.022 | -0.022 | 0.013 | 0.000 | 43.221 | 25.000 | 25.000 | 0.000 | 0.000 | 90.000 | 15.491 |
| 5 | 0.105 | -0.040 | -0.040 | 0.026 | 0.000 | 48.787 | 25.000 | 25.000 | 0.000 | 0.000 | 90.000 | 18.807 |
| 6 | 0.224 | -0.090 | -0.090 | 0.044 | 0.000 | 56.766 | 25.000 | 25.000 | 0.000 | 0.000 | 90.000 | 22.861 |
| 7 | 0.324 | -0.137 | -0.137 | 0.050 | 0.000 | 60.212 | 25.000 | 25.000 | 0.000 | 0.000 | 90.000 | 24.408 |
| 8 | 0.403 | -0.175 | -0.175 | 0.052 | 0.000 | 64.005 | 25.000 | 25.000 | 0.000 | 0.000 | 90.000 | 25.991 |
| 9 | 0.506 | -0.225 | -0.225 | 0.057 | 0.000 | 67.437 | 25.000 | 25.000 | 0.000 | 0.000 | 90.000 | 27.328 |
| 10 | 0.632 | -0.288 | -0.288 | 0.056 | 0.000 | 71.195 | 25.000 | 25.000 | 0.000 | 0.000 | 90.000 | 28.700 |
| 11 | 0.738 | -0.343 | -0.343 | 0.052 | 0.000 | 73.567 | 25.000 | 25.000 | 0.000 | 0.000 | 90.000 | 29.520 |
| 12 | 0.895 | -0.426 | -0.426 | 0.044 | 0.000 | 76.942 | 25.000 | 25.000 | 0.000 | 0.000 | 90.000 | 30.632 |
| 13 | 0.998 | -0.482 | -0.482 | 0.035 | 0.000 | 78.954 | 25.000 | 25.000 | 0.000 | 0.000 | 90.000 | 31.266 |
| 14 | 1.147 | -0.565 | -0.565 | 0.017 | 0.000 | 81.965 | 25.000 | 25.000 | 0.000 | 0.000 | 90.000 | 32.178 |
| 15 | 1.442 | -0.732 | -0.732 | -0.022 | 0.000 | 86.581 | 25.000 | 25.000 | 0.000 | 0.000 | 90.000 | 33.497 |
| 16 | 1.623 | -0.838 | -0.838 | -0.052 | 0.000 | 89.191 | 25.000 | 25.000 | 0.000 | 0.000 | 90.000 | 34.204 |
| 17 | 2.007 | -1.074 | -1.074 | -0.140 | 0.000 | 92.295 | 25.000 | 25.000 | 0.000 | 0.000 | 90.000 | 35.010 |
| 18 | 2.463 | -1.356 | -1.356 | -0.249 | 0.000 | 96.996 | 25.000 | 25.000 | 0.000 | 0.000 | 90.000 | 36.168 |
| 19 | 2.871 | -1.615 | -1.615 | -0.358 | 0.000 | 101.002 | 25.000 | 25.000 | 0.000 | 0.000 | 90.000 | 37.098 |
| 20 | 3.662 | -2.134 | -2.134 | -0.607 | 0.000 | 105.871 | 25.000 | 25.000 | 0.000 | 0.000 | 90.000 | 38.166 |
| 21 | 4.006 | -2.344 | -2.344 | -0.681 | 0.000 | 107.849 | 25.000 | 25.000 | 0.000 | 0.000 | 90.000 | 38.582 |
| 22 | 4.851 | -2.904 | -2.904 | -0.956 | 0.000 | 109.853 | 25.000 | 25.000 | 0.000 | 0.000 | 90.000 | 38.993 |
| 23 | 4.975 | -2.983 | -2.983 | -0.991 | 0.000 | 110.369 | 25.000 | 25.000 | 0.000 | 0.000 | 90.000 | 39.097 |
| 24 | 5.236 | -3.161 | -3.161 | -1.087 | 0.000 | 110.053 | 25.000 | 25.000 | 0.000 | 0.000 | 90.000 | 39.034 |
| 25 | 5.487 | -3.331 | -3.331 | -1.174 | 0.000 | 111.057 | 25.000 | 25.000 | 0.000 | 0.000 | 90.000 | 39.235 |
| 26 | 5.665 | -3.450 | -3.450 | -1.236 | 0.000 | 111.818 | 25.000 | 25.000 | 0.000 | 0.000 | 90.000 | 39.387 |
| 27 | 6.002 | -3.682 | -3.682 | -1.362 | 0.000 | 108.814 | 25.000 | 25.000 | 0.000 | 0.000 | 90.000 | 38.781 |
| 28 | 6.298 | -3.874 | -3.874 | -1.450 | 0.000 | 105.258 | 25.000 | 25.000 | 0.000 | 0.000 | 90.000 | 38.035 |
| 29 | 6.542 | -4.024 | -4.024 | -1.506 | 0.000 | 102.116 | 25.000 | 25.000 | 0.000 | 0.000 | 90.000 | 37.348 |
| 30 | 6.777 | -4.172 | -4.172 | -1.567 | 0.000 | 99.956 | 25.000 | 25.000 | 0.000 | 0.000 | 90.000 | 36.860 |
| 31 | 6.938 | -4.267 | -4.267 | -1.596 | 0.000 | 99.487 | 25.000 | 25.000 | 0.000 | 0.000 | 90.000 | 36.752 |
| 32 | 7.144 | -4.386 | -4.386 | -1.629 | 0.000 | 98.980 | 25.000 | 25.000 | 0.000 | 0.000 | 90.000 | 36.635 |
| 33 | 7.636 | -4.667 | -4.667 | -1.698 | 0.000 | 95.055 | 25.000 | 25.000 | 0.000 | 0.000 | 90.000 | 35.699 |

# Triaxial Test on Fine Nevada Sand with a Target Void Ratio $=\mathbf{0 . 5 3 0}\left(\mathrm{D}_{\mathrm{r}}=\mathbf{9 1 . 2 8 \%}\right)$ 

| Test No.: | 6 |
| :--- | :--- |
| Test Date: | $11 / 24 / 10$ |
| Sector: | III |
| Initial Void Ratio, e: | 0.538 |
| Final Mean Princpal Stress, $\sigma_{\mathrm{m}}:$ | 155.9 kPa |
| Max Friction Angle, $\phi:$ | 38.2 deg |
| b-value at failure: | 0.00 |
| Stress direction at failure, $\alpha:$ | 90.0 deg |


| Shear Band Notes |
| :--- |
| Point of Observation: |
|  |
| Inclination (from Vertical): |
| $67.1^{\circ}$ |
| Failure Notes: |


| Point (No.) | $\begin{gathered} \varepsilon_{\mathrm{z}} \\ (\%) \\ \hline \end{gathered}$ | $\begin{gathered} \varepsilon_{\mathrm{x}} \\ (\%) \\ \hline \end{gathered}$ | $\begin{array}{r} \varepsilon_{y} \\ (\%) \\ \hline \end{array}$ | $\begin{gathered} \varepsilon_{\mathrm{v}} \\ (\%) \\ \hline \end{gathered}$ | $\begin{gathered} \gamma_{\theta z} \\ (\%) \\ \hline \end{gathered}$ | $\begin{gathered} \sigma_{z} \\ (\mathrm{kPa}) \\ \hline \end{gathered}$ | $\begin{gathered} \sigma_{x} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \sigma_{y} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \tau_{\theta z} \\ (\mathrm{kPa}) \end{gathered}$ | b | $\begin{gathered} \alpha \\ \left({ }^{( }\right) \\ \hline \end{gathered}$ | $\begin{gathered} \varphi \\ \left({ }^{\circ}\right) \\ \hline \end{gathered}$ |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 1 | 0.000 | 0.000 | 0.000 | 0.000 | 0.000 | 74.977 | 75.000 | 75.000 | 0.000 | 0.000 | 90.000 | -0.009 |
| 2 | 0.044 | -0.019 | -0.019 | 0.007 | 0.000 | 90.702 | 75.000 | 75.000 | 0.000 | 0.000 | 90.000 | 5.437 |
| 3 | 0.055 | -0.021 | -0.021 | 0.013 | 0.000 | 97.980 | 75.000 | 75.000 | 0.000 | 0.000 | 90.000 | 7.634 |
| 4 | 0.068 | -0.025 | -0.025 | 0.017 | 0.000 | 103.869 | 75.000 | 75.000 | 0.000 | 0.000 | 90.000 | 9.288 |
| 5 | 0.106 | -0.037 | -0.037 | 0.033 | 0.000 | 120.148 | 75.000 | 75.000 | 0.000 | 0.000 | 90.000 | 13.377 |
| 6 | 0.215 | -0.070 | -0.070 | 0.074 | 0.000 | 151.626 | 75.000 | 75.000 | 0.000 | 0.000 | 90.000 | 19.762 |
| 7 | 0.304 | -0.102 | -0.102 | 0.100 | 0.000 | 169.572 | 75.000 | 75.000 | 0.000 | 0.000 | 90.000 | 22.748 |
| 8 | 0.420 | -0.149 | -0.149 | 0.122 | 0.000 | 186.768 | 75.000 | 75.000 | 0.000 | 0.000 | 90.000 | 25.276 |
| 9 | 0.551 | -0.204 | -0.204 | 0.143 | 0.000 | 201.495 | 75.000 | 75.000 | 0.000 | 0.000 | 90.000 | 27.226 |
| 10 | 0.610 | -0.231 | -0.231 | 0.148 | 0.000 | 207.638 | 75.000 | 75.000 | 0.000 | 0.000 | 90.000 | 27.988 |
| 11 | 0.803 | -0.315 | -0.315 | 0.174 | 0.000 | 224.642 | 75.000 | 75.000 | 0.000 | 0.000 | 90.000 | 29.960 |
| 12 | 0.902 | -0.362 | -0.362 | 0.178 | 0.000 | 232.073 | 75.000 | 75.000 | 0.000 | 0.000 | 90.000 | 30.765 |
| 13 | 1.070 | -0.448 | -0.448 | 0.174 | 0.000 | 242.452 | 75.000 | 75.000 | 0.000 | 0.000 | 90.000 | 31.836 |
| 14 | 1.171 | -0.502 | -0.502 | 0.167 | 0.000 | 247.763 | 75.000 | 75.000 | 0.000 | 0.000 | 90.000 | 32.362 |
| 15 | 1.368 | -0.608 | -0.608 | 0.152 | 0.000 | 257.668 | 75.000 | 75.000 | 0.000 | 0.000 | 90.000 | 33.305 |
| 16 | 1.725 | -0.815 | -0.815 | 0.096 | 0.000 | 270.547 | 75.000 | 75.000 | 0.000 | 0.000 | 90.000 | 34.465 |
| 17 | 2.303 | -1.156 | -1.156 | -0.009 | 0.000 | 287.485 | 75.000 | 75.000 | 0.000 | 0.000 | 90.000 | 35.887 |
| 18 | 2.526 | -1.291 | -1.291 | -0.056 | 0.000 | 293.318 | 75.000 | 75.000 | 0.000 | 0.000 | 90.000 | 36.352 |
| 19 | 3.083 | -1.640 | -1.640 | -0.198 | 0.000 | 303.502 | 75.000 | 75.000 | 0.000 | 0.000 | 90.000 | 37.135 |
| 20 | 4.048 | -2.267 | -2.267 | -0.487 | 0.000 | 313.821 | 75.000 | 75.000 | 0.000 | 0.000 | 90.000 | 37.895 |
| 21 | 4.625 | -2.643 | -2.643 | -0.660 | 0.000 | 317.889 | 75.000 | 75.000 | 0.000 | 0.000 | 90.000 | 38.186 |
| 22 | 4.934 | -2.849 | -2.849 | -0.765 | 0.000 | 317.179 | 75.000 | 75.000 | 0.000 | 0.000 | 90.000 | 38.135 |
| 23 | 5.110 | -2.967 | -2.967 | -0.825 | 0.000 | 317.238 | 75.000 | 75.000 | 0.000 | 0.000 | 90.000 | 38.140 |
| 24 | 5.384 | -3.146 | -3.146 | -0.908 | 0.000 | 317.314 | 75.000 | 75.000 | 0.000 | 0.000 | 90.000 | 38.145 |
| 25 | 5.685 | -3.347 | -3.347 | -1.008 | 0.000 | 314.038 | 75.000 | 75.000 | 0.000 | 0.000 | 90.000 | 37.911 |
| 26 | 5.945 | -3.516 | -3.516 | -1.086 | 0.000 | 309.646 | 75.000 | 75.000 | 0.000 | 0.000 | 90.000 | 37.592 |
| 27 | 6.207 | -3.681 | -3.681 | -1.155 | 0.000 | 298.545 | 75.000 | 75.000 | 0.000 | 0.000 | 90.000 | 36.758 |
| 28 | 6.500 | -3.864 | -3.864 | -1.227 | 0.000 | 279.432 | 75.000 | 75.000 | 0.000 | 0.000 | 90.000 | 35.225 |
| 29 | 6.755 | -4.010 | -4.010 | -1.264 | 0.000 | 266.348 | 75.000 | 75.000 | 0.000 | 0.000 | 90.000 | 34.095 |
| 30 | 7.016 | -4.151 | -4.151 | -1.286 | 0.000 | 261.633 | 75.000 | 75.000 | 0.000 | 0.000 | 90.000 | 33.670 |
| 31 | 7.165 | -4.230 | -4.230 | -1.295 | 0.000 | 260.365 | 75.000 | 75.000 | 0.000 | 0.000 | 90.000 | 33.554 |
| 32 | 7.320 | -4.311 | -4.311 | -1.301 | 0.000 | 260.043 | 75.000 | 75.000 | 0.000 | 0.000 | 90.000 | 33.525 |
| 33 | 8.068 | -4.692 | -4.692 | -1.316 | 0.000 | 260.096 | 75.000 | 75.000 | 0.000 | 0.000 | 90.000 | 33.530 |

# Triaxial Test on Fine Nevada Sand with a Target Void Ratio $=\mathbf{0 . 5 3 0}\left(\mathrm{D}_{\mathrm{r}}=\mathbf{9 1 . 2 8 \%}\right)$ 

| Test No.: | 7 |
| :--- | :---: |
| Test Date: | $12 / 4 / 10$ |
| Sector: | III |
| Initial Void Ratio, e: | 0.532 |
| Final Mean Princpal Stress, $\sigma_{\mathrm{m}}:$ | 260.5 kPa |
| Max Friction Angle, $\phi:$ | 36.9 deg |
| b-value at failure: | 0.00 |
| Stress direction at failure, $\alpha:$ | 90.0 deg |


| Shear Band Notes |
| :--- |
| Point of Observation: |
|  |
| Inclination (from Vertical): |
| $66.9^{\circ}, 60.6^{\circ}$ |
|  |
| Failure Notes: |


| Point (No.) | $\begin{gathered} \varepsilon_{\mathrm{z}} \\ (\%) \\ \hline \end{gathered}$ | $\begin{gathered} \varepsilon_{\mathrm{x}} \\ (\%) \\ \hline \end{gathered}$ | $\begin{array}{r} \varepsilon_{y} \\ (\%) \\ \hline \end{array}$ | $\begin{gathered} \varepsilon_{\mathrm{v}} \\ (\%) \\ \hline \end{gathered}$ | $\begin{gathered} \gamma_{\theta z} \\ (\%) \\ \hline \end{gathered}$ | $\begin{gathered} \sigma_{z} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \sigma_{x} \\ (\mathrm{kPa}) \\ \hline \end{gathered}$ | $\begin{gathered} \sigma_{y} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \tau_{\theta z} \\ (\mathrm{kPa}) \end{gathered}$ | b | $\begin{gathered} \alpha \\ \left({ }^{( }\right) \\ \hline \end{gathered}$ | $\begin{gathered} \varphi \\ \left({ }^{\circ}\right) \\ \hline \end{gathered}$ |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 1 | 0.000 | 0.000 | 0.000 | 0.000 | 0.000 | 129.977 | 130.000 | 130.000 | 0.000 | 0.000 | 90.000 | -0.005 |
| 2 | 0.042 | -0.017 | -0.017 | 0.009 | 0.000 | 157.682 | 130.000 | 130.000 | 0.000 | 0.000 | 90.000 | . 522 |
| 3 | 0.061 | -0.018 | -0.018 | 0.024 | 0.000 | 174.088 | 130.000 | 130.000 | 0.000 | 0.000 | 90.000 | 8.336 |
| 4 | 0.090 | -0.028 | -0.028 | 0.035 | 0.000 | 197.463 | 130.000 | 130.000 | 0.000 | 0.000 | 90.000 | 11.889 |
| 5 | 0.104 | -0.030 | -0.030 | 0.044 | 0.000 | 207.231 | 130.000 | 130.000 | 0.000 | 0.000 | 90.000 | 13.239 |
| 6 | 0.224 | -0.068 | -0.068 | 0.088 | 0.000 | 260.526 | 130.000 | 130.000 | 0.000 | 0.000 | 90.000 | 19.526 |
| 7 | 0.305 | -0.096 | -0.096 | 0.114 | 0.000 | 285.549 | 130.000 | 130.000 | 0.000 | 0.000 | 90.000 | 21.982 |
| 8 | 0.428 | -0.142 | -0.142 | 0.144 | 0.000 | 313.958 | 130.000 | 130.000 | 0.000 | 0.000 | 90.000 | 24.479 |
| 9 | 0.501 | -0.172 | -0.172 | 0.158 | 0.000 | 329.856 | 130.000 | 130.000 | 0.000 | 0.000 | 90.000 | 25.760 |
| 10 | 0.620 | -0.223 | -0.223 | 0.175 | 0.000 | 345.294 | 130.000 | 130.000 | 0.000 | 0.000 | 90.000 | 26.934 |
| 11 | 0.935 | -0.371 | -0.371 | 0.193 | 0.000 | 382.777 | 130.000 | 130.000 | 0.000 | 0.000 | 90.000 | 29.535 |
| 12 | 1.048 | -0.427 | -0.427 | 0.195 | 0.000 | 393.571 | 130.000 | 130.000 | 0.000 | 0.000 | 90.000 | 30.226 |
| 13 | 1.134 | -0.470 | -0.470 | 0.195 | 0.000 | 401.300 | 130.000 | 130.000 | 0.000 | 0.000 | 90.000 | 30.706 |
| 14 | 1.213 | -0.510 | -0.510 | 0.193 | 0.000 | 407.645 | 130.000 | 130.000 | 0.000 | 0.000 | 90.000 | 31.092 |
| 15 | 1.460 | -0.640 | -0.640 | 0.179 | 0.000 | 425.527 | 130.000 | 130.000 | 0.000 | 0.000 | 90.000 | 32.139 |
| 16 | 1.783 | -0.818 | -0.818 | 0.147 | 0.000 | 446.103 | 130.000 | 130.000 | 0.000 | 0.000 | 90.000 | 33.277 |
| 17 | 2.242 | -1.082 | -1.082 | 0.079 | 0.000 | 466.958 | 130.000 | 130.000 | 0.000 | 0.000 | 90.000 | 34.365 |
| 18 | 2.710 | -1.364 | -1.364 | -0.018 | 0.000 | 487.440 | 130.000 | 130.000 | 0.000 | 0.000 | 90.000 | 35.374 |
| 19 | 2.952 | -1.513 | -1.513 | -0.074 | 0.000 | 495.831 | 130.000 | 130.000 | 0.000 | 0.000 | 90.000 | 35.771 |
| 20 | 3.169 | -1.648 | -1.648 | -0.127 | 0.000 | 502.251 | 130.000 | 130.000 | 0.000 | 0.000 | 90.000 | 36.070 |
| 21 | 3.719 | -1.999 | -1.999 | -0.278 | 0.000 | 513.660 | 130.000 | 130.000 | 0.000 | 0.000 | 90.000 | 36.588 |
| 22 | 4.112 | -2.251 | -2.251 | -0.390 | 0.000 | 519.341 | 130.000 | 130.000 | 0.000 | 0.000 | 90.000 | 36.841 |
| 23 | 4.850 | -2.731 | -2.731 | -0.613 | 0.000 | 521.105 | 130.000 | 130.000 | 0.000 | 0.000 | 90.000 | 36.919 |
| 24 | 5.020 | -2.842 | -2.842 | -0.665 | 0.000 | 517.896 | 130.000 | 130.000 | 0.000 | 0.000 | 90.000 | 36.777 |
| 25 | 5.229 | -2.980 | -2.980 | -0.731 | 0.000 | 509.562 | 130.000 | 130.000 | 0.000 | 0.000 | 90.000 | 36.404 |
| 26 | 5.483 | -3.142 | -3.142 | -0.801 | 0.000 | 492.563 | 130.000 | 130.000 | 0.000 | 0.000 | 90.000 | 35.618 |
| 27 | 5.774 | -3.318 | -3.318 | -0.862 | 0.000 | 471.327 | 130.000 | 130.000 | 0.000 | 0.000 | 90.000 | 34.585 |
| 28 | 6.166 | -3.541 | -3.541 | -0.917 | 0.000 | 438.878 | 130.000 | 130.000 | 0.000 | 0.000 | 90.000 | 32.885 |
| 29 | 6.245 | -3.584 | -3.584 | -0.923 | 0.000 | 432.761 | 130.000 | 130.000 | 0.000 | 0.000 | 90.000 | 32.547 |
| 30 | 6.667 | -3.800 | -3.800 | -0.932 | 0.000 | 416.838 | 130.000 | 130.000 | 0.000 | 0.000 | 90.000 | 31.637 |
| 31 | 6.761 | -3.847 | -3.847 | -0.934 | 0.000 | 416.868 | 130.000 | 130.000 | 0.000 | 0.000 | 90.000 | 31.639 |
| 32 | 6.929 | -3.931 | -3.931 | -0.934 | 0.000 | 415.063 | 130.000 | 130.000 | 0.000 | 0.000 | 90.000 | 31.533 |
| 33 | 7.150 | -4.040 | -4.040 | -0.930 | 0.000 | 411.826 | 130.000 | 130.000 | 0.000 | 0.000 | 90.000 | 31.342 |

# Triaxial Test on Fine Nevada Sand with a Target Void Ratio $=\mathbf{0 . 5 3 0}\left(\mathrm{D}_{\mathrm{r}}=\mathbf{9 1 . 2 8 \%}\right)$ 

| Test No.: | $8, \mathrm{TT} \# 13$ |
| :--- | :--- |
| Test Date: | $2 / 25 / 11$ |
| Sector: | III |
| Initial Void Ratio, e: | 0.535 |
| Final Mean Princpal Stress, $\sigma_{\mathrm{m}}:$ | 102.2 kPa |
| Max Friction Angle, $\phi:$ | 37.6 deg |
| b-value at failure: | 0.00 |
| Stress direction at failure, $\alpha:$ | 90.0 deg |$\quad$| Shear Band Notes |  |
| :--- | :--- |
| Point of Observation: |  |
|  | Inclination (from Vertical): |
| $63^{\circ}$ |  |
|  |  |


| Point (No.) | $\begin{gathered} \varepsilon_{\mathrm{z}} \\ (\%) \\ \hline \end{gathered}$ | $\begin{gathered} \varepsilon_{\mathrm{x}} \\ (\%) \\ \hline \end{gathered}$ | $\begin{array}{r} \varepsilon_{y} \\ (\%) \\ \hline \end{array}$ | $\begin{gathered} \varepsilon_{\mathrm{v}} \\ (\%) \\ \hline \end{gathered}$ | $\begin{gathered} \gamma_{\theta z} \\ (\%) \\ \hline \end{gathered}$ | $\begin{gathered} \sigma_{z} \\ (\mathrm{kPa}) \\ \hline \end{gathered}$ | $\begin{gathered} \sigma_{\mathrm{x}} \\ (\mathrm{kPa}) \\ \hline \end{gathered}$ | $\begin{gathered} \sigma_{y} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \tau_{\text {өz }} \\ (\mathrm{kPa}) \end{gathered}$ | b | $\begin{gathered} \alpha \\ \left({ }^{\circ}\right) \\ \hline \end{gathered}$ | $\begin{gathered} \varphi \\ \left({ }^{\circ}\right) \\ \hline \end{gathered}$ |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 1 | 0.000 | 0.000 | 0.000 | 0.000 | 0.000 | 52.164 | 50.000 | 50.000 | 0.000 | 0.000 | 90.000 | 1.213 |
| 2 | 0.017 | -0.006 | -0.006 | 0.004 | 0.000 | 59.669 | 50.000 | 50.000 | 0.000 | 0.000 | 90.000 | 5.058 |
| 3 | 0.031 | -0.011 | -0.011 | 0.009 | 0.000 | 70.658 | 50.000 | 50.000 | 0.000 | 0.000 | 90.000 | 9.858 |
| 4 | 0.072 | -0.019 | -0.019 | 0.035 | 0.000 | 87.137 | 50.000 | 50.000 | 0.000 | 0.000 | 90.000 | 15.712 |
| 5 | 0.155 | -0.049 | -0.049 | 0.057 | 0.000 | 106.515 | 50.000 | 50.000 | 0.000 | 0.000 | 90.000 | 21.167 |
| 6 | 0.246 | -0.084 | -0.084 | 0.079 | 0.000 | 120.010 | 50.000 | 50.000 | 0.000 | 0.000 | 90.000 | 24.318 |
| 7 | 0.541 | -0.222 | -0.222 | 0.096 | 0.000 | 141.338 | 50.000 | 50.000 | 0.000 | 0.000 | 90.000 | 28.513 |
| 8 | 0.615 | -0.260 | -0.260 | 0.096 | 0.000 | 144.915 | 50.000 | 50.000 | 0.000 | 0.000 | 90.000 | 29.141 |
| 9 | 0.876 | -0.399 | -0.399 | 0.079 | 0.000 | 155.554 | 50.000 | 50.000 | 0.000 | 0.000 | 90.000 | 30.898 |
| 10 | 1.077 | -0.510 | -0.510 | 0.057 | 0.000 | 161.845 | 50.000 | 50.000 | 0.000 | 0.000 | 90.000 | 31.867 |
| 11 | 1.215 | -0.590 | -0.590 | 0.035 | 0.000 | 165.647 | 50.000 | 50.000 | 0.000 | 0.000 | 90.000 | 32.431 |
| 12 | 1.421 | -0.711 | -0.711 | 0.000 | 0.000 | 171.145 | 50.000 | 50.000 | 0.000 | 0.000 | 90.000 | 33.217 |
| 13 | 1.846 | -0.967 | -0.967 | -0.087 | 0.000 | 177.703 | 50.000 | 50.000 | 0.000 | 0.000 | 90.000 | 34.113 |
| 14 | 2.221 | -1.200 | -1.200 | -0.179 | 0.000 | 184.610 | 50.000 | 50.000 | 0.000 | 0.000 | 90.000 | 35.013 |
| 15 | 2.647 | -1.470 | -1.470 | -0.293 | 0.000 | 190.629 | 50.000 | 50.000 | 0.000 | 0.000 | 90.000 | 35.762 |
| 16 | 2.942 | -1.670 | -1.670 | -0.398 | 0.000 | 192.537 | 50.000 | 50.000 | 0.000 | 0.000 | 90.000 | 35.993 |
| 17 | 3.307 | -1.909 | -1.909 | -0.511 | 0.000 | 197.127 | 50.000 | 50.000 | 0.000 | 0.000 | 90.000 | 36.538 |
| 18 | 3.736 | -2.194 | -2.194 | -0.651 | 0.000 | 199.425 | 50.000 | 50.000 | 0.000 | 0.000 | 90.000 | 36.804 |
| 19 | 4.018 | -2.385 | -2.385 | -0.752 | 0.000 | 201.979 | 50.000 | 50.000 | 0.000 | 0.000 | 90.000 | 37.095 |
| 20 | 4.402 | -2.645 | -2.645 | -0.887 | 0.000 | 204.640 | 50.000 | 50.000 | 0.000 | 0.000 | 90.000 | 37.394 |
| 21 | 5.038 | -3.077 | -3.077 | -1.115 | 0.000 | 205.674 | 50.000 | 50.000 | 0.000 | 0.000 | 90.000 | 37.508 |
| 22 | 5.286 | -3.242 | -3.242 | -1.198 | 0.000 | 206.513 | 50.000 | 50.000 | 0.000 | 0.000 | 90.000 | 37.601 |
| 23 | 5.856 | -3.629 | -3.629 | -1.403 | 0.000 | 205.256 | 50.000 | 50.000 | 0.000 | 0.000 | 90.000 | 37.462 |
| 24 | 6.362 | -3.961 | -3.961 | -1.561 | 0.000 | 194.047 | 50.000 | 50.000 | 0.000 | 0.000 | 90.000 | 36.174 |
| 25 | 6.685 | -4.155 | -4.155 | -1.626 | 0.000 | 187.065 | 50.000 | 50.000 | 0.000 | 0.000 | 90.000 | 35.322 |
| 26 | 6.886 | -4.271 | -4.271 | -1.657 | 0.000 | 183.708 | 50.000 | 50.000 | 0.000 | 0.000 | 90.000 | 34.898 |
| 27 | 7.104 | -4.391 | -4.391 | -1.679 | 0.000 | 181.023 | 50.000 | 50.000 | 0.000 | 0.000 | 90.000 | 34.551 |
| 28 | 7.803 | -4.767 | -4.767 | -1.731 | 0.000 | 175.653 | 50.000 | 50.000 | 0.000 | 0.000 | 90.000 | 33.838 |
| 29 | 8.442 | -5.097 | -5.097 | -1.753 | 0.000 | 170.470 | 50.000 | 50.000 | 0.000 | 0.000 | 90.000 | 33.122 |
| 30 | 8.839 | -5.301 | -5.301 | -1.764 | 0.000 | 170.262 | 50.000 | 50.000 | 0.000 | 0.000 | 90.000 | 33.093 |
| 31 | 9.088 | -5.428 | -5.428 | -1.768 | 0.000 | 169.601 | 50.000 | 50.000 | 0.000 | 0.000 | 90.000 | 32.999 |

## Triaxial Test on Fine Nevada Sand with a Target Void Ratio $=0.530\left(D_{r}=91.28 \%\right)$

| Test No.: | 9, W1 |
| :--- | :---: |
| Test Date: | $1 / 17 / 11$ |
| Sector: | III |
| Initial Void Ratio, e: | 0.534 |
| Final Mean Princpal Stress, $\sigma_{\mathrm{m}}:$ | 206.9 kPa |
| Max Friction Angle, $\phi:$ | 37.7 deg |
| b-value at failure: | 0.00 |
| Stress direction at failure, $\alpha:$ | 90.0 deg |


| Shear Band Notes |  |
| :--- | :--- |
| Point of Observation: | 26 |
| Inclination (from Vertical): |  |
| $60^{\circ}, 55^{\circ}$ |  |
|  |  |
| Failure Notes: |  |


| Point (No.) | $\begin{gathered} \varepsilon_{\mathrm{z}} \\ (\%) \end{gathered}$ | $\begin{gathered} \varepsilon_{\mathrm{x}} \\ (\%) \end{gathered}$ | $\begin{array}{r} \varepsilon_{\mathrm{y}} \\ (\%) \end{array}$ | $\begin{gathered} \varepsilon_{\mathrm{v}} \\ (\%) \end{gathered}$ | $\begin{gathered} \gamma_{\theta z} \\ (\%) \end{gathered}$ | $\begin{gathered} \sigma_{z} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \sigma_{x} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \sigma_{y} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \tau_{\theta z} \\ (\mathrm{kPa}) \end{gathered}$ | b | $\begin{gathered} \alpha \\ \left({ }^{\circ}\right) \end{gathered}$ | $\begin{gathered} \varphi \\ \left({ }^{\circ}\right) \end{gathered}$ |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 1 | 0.000 | 0.000 | 0.000 | 0.000 | 0.000 | 96.246 | 101.000 | 101.000 | 0.000 | 0.000 | 90.000 | -1.381 |
| 2 | 0.007 | 0.000 | 0.000 | 0.007 | 0.000 | 121.991 | 101.000 | 101.000 | 0.000 | 0.000 | 90.000 | 5.401 |
| 3 | 0.057 | -0.015 | -0.015 | 0.026 | 0.000 | 153.796 | 101.000 | 101.000 | 0.000 | 0.000 | 90.000 | 11.959 |
| 4 | 0.141 | -0.042 | -0.042 | 0.057 | 0.000 | 189.367 | 101.000 | 101.000 | 0.000 | 0.000 | 90.000 | 17.718 |
| 5 | 0.240 | -0.079 | -0.079 | 0.081 | 0.000 | 217.317 | 101.000 | 101.000 | 0.000 | 0.000 | 90.000 | 21.433 |
| 6 | 0.329 | -0.114 | -0.114 | 0.101 | 0.000 | 235.399 | 101.000 | 101.000 | 0.000 | 0.000 | 90.000 | 23.548 |
| 7 | 0.638 | -0.254 | -0.254 | 0.130 | 0.000 | 277.855 | 101.000 | 101.000 | 0.000 | 0.000 | 90.000 | 27.828 |
| 8 | 0.964 | -0.419 | -0.419 | 0.125 | 0.000 | 307.549 | 101.000 | 101.000 | 0.000 | 0.000 | 90.000 | 30.369 |
| 9 | 1.268 | -0.588 | -0.588 | 0.092 | 0.000 | 328.212 | 101.000 | 101.000 | 0.000 | 0.000 | 90.000 | 31.963 |
| 10 | 1.549 | -0.748 | -0.748 | 0.053 | 0.000 | 343.540 | 101.000 | 101.000 | 0.000 | 0.000 | 90.000 | 33.066 |
| 11 | 1.893 | -0.955 | -0.955 | -0.018 | 0.000 | 358.147 | 101.000 | 101.000 | 0.000 | 0.000 | 90.000 | 34.059 |
| 12 | 2.145 | -1.110 | -1.110 | -0.075 | 0.000 | 367.723 | 101.000 | 101.000 | 0.000 | 0.000 | 90.000 | 34.683 |
| 13 | 2.468 | -1.313 | -1.313 | -0.158 | 0.000 | 378.989 | 101.000 | 101.000 | 0.000 | 0.000 | 90.000 | 35.391 |
| 14 | 3.006 | -1.661 | -1.661 | -0.317 | 0.000 | 391.497 | 101.000 | 101.000 | 0.000 | 0.000 | 90.000 | 36.146 |
| 15 | 3.312 | -1.865 | -1.865 | -0.418 | 0.000 | 398.863 | 101.000 | 101.000 | 0.000 | 0.000 | 90.000 | 36.576 |
| 16 | 3.921 | -2.281 | -2.281 | -0.642 | 0.000 | 409.714 | 101.000 | 101.000 | 0.000 | 0.000 | 90.000 | 37.191 |
| 17 | 4.281 | -2.527 | -2.527 | -0.774 | 0.000 | 413.913 | 101.000 | 101.000 | 0.000 | 0.000 | 90.000 | 37.423 |
| 18 | 4.802 | -2.889 | -2.889 | -0.976 | 0.000 | 418.192 | 101.000 | 101.000 | 0.000 | 0.000 | 90.000 | 37.657 |
| 19 | 5.135 | -3.124 | -3.124 | -1.113 | 0.000 | 418.077 | 101.000 | 101.000 | 0.000 | 0.000 | 90.000 | 37.651 |
| 20 | 5.775 | -3.576 | -3.576 | -1.376 | 0.000 | 418.112 | 101.000 | 101.000 | 0.000 | 0.000 | 90.000 | 37.653 |
| 21 | 6.096 | -3.800 | -3.800 | -1.504 | 0.000 | 417.336 | 101.000 | 101.000 | 0.000 | 0.000 | 90.000 | 37.611 |
| 22 | 6.355 | -3.978 | -3.978 | -1.601 | 0.000 | 416.509 | 101.000 | 101.000 | 0.000 | 0.000 | 90.000 | 37.566 |
| 23 | 6.697 | -4.219 | -4.219 | -1.741 | 0.000 | 412.839 | 101.000 | 101.000 | 0.000 | 0.000 | 90.000 | 37.364 |
| 24 | 7.022 | -4.441 | -4.441 | -1.860 | 0.000 | 410.008 | 101.000 | 101.000 | 0.000 | 0.000 | 90.000 | 37.207 |
| 25 | 7.337 | -4.658 | -4.658 | -1.979 | 0.000 | 403.268 | 101.000 | 101.000 | 0.000 | 0.000 | 90.000 | 36.828 |
| 26 | 7.611 | -4.843 | -4.843 | -2.076 | 0.000 | 397.975 | 101.000 | 101.000 | 0.000 | 0.000 | 90.000 | 36.525 |
| 27 | 8.162 | -5.202 | -5.202 | -2.243 | 0.000 | 365.256 | 101.000 | 101.000 | 0.000 | 0.000 | 90.000 | 34.525 |
| 28 | 8.489 | -5.396 | -5.396 | -2.304 | 0.000 | 347.709 | 101.000 | 101.000 | 0.000 | 0.000 | 90.000 | 33.355 |
| 29 | 9.083 | -5.723 | -5.723 | -2.364 | 0.000 | 341.082 | 101.000 | 101.000 | 0.000 | 0.000 | 90.000 | 32.893 |
| 30 | 9.637 | -6.015 | -6.015 | -2.392 | 0.000 | 338.883 | 101.000 | 101.000 | 0.000 | 0.000 | 90.000 | 32.737 |
| 31 | 9.637 | -6.015 | -6.015 | -2.392 | 0.000 | 339.218 | 101.000 | 101.000 | 0.000 | 0.000 | 90.000 | 32.761 |
| 32 | 10.877 | -6.670 | -6.670 | -2.462 | 0.000 | 339.579 | 101.000 | 101.000 | 0.000 | 0.000 | 90.000 | 32.787 |

# Triaxial Test on Fine Nevada Sand with a Target Void Ratio $=0.530\left(D_{r}=91.28 \%\right)$ 

| Test No.: | $10, \mathrm{~A} 7$ |
| :--- | :---: |
| Test Date: | $12 / 21 / 10$ |
| Sector: | III |
| Initial Void Ratio, e: | 0.532 |
| Final Mean Princpal Stress, $\sigma_{\mathrm{m}}:$ | 200.0 kPa |
| Max Friction Angle, $\phi:$ | 37.2 deg |
| b-value at failure: | 0.00 |
| Stress direction at failure, $\alpha:$ | 90.0 deg |


| Shear Band Notes | 29 |
| :--- | :--- |
| Point of Observation: |  |
| Inclination (from Vertical): |  |
| $58^{\circ}$ |  |
| Failure Notes: |  |


| Point (No.) | $\begin{gathered} \varepsilon_{\mathrm{z}} \\ (\%) \end{gathered}$ | $\begin{gathered} \varepsilon_{\mathrm{x}} \\ (\%) \\ \hline \end{gathered}$ | $\begin{array}{r} \varepsilon_{\mathrm{y}} \\ (\%) \\ \hline \end{array}$ | $\begin{gathered} \varepsilon_{\mathrm{v}} \\ (\%) \end{gathered}$ | $\begin{aligned} & \gamma_{\theta z} \\ & (\%) \end{aligned}$ | $\begin{gathered} \sigma_{z} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \sigma_{x} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \sigma_{\mathrm{y}} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \tau_{\theta z} \\ (\mathrm{kPa}) \end{gathered}$ | b | $\begin{gathered} \alpha \\ \left(^{( }\right) \\ \hline \end{gathered}$ | $\begin{gathered} \varphi \\ \left({ }^{\circ}\right) \\ \hline \end{gathered}$ |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 1 | 0.000 | 0.000 | 0.000 | 0.000 | 0.000 | 98.977 | 99.000 | 99.000 | 0.000 | 0.000 | 90.000 | -0.007 |
| 2 | 0.023 | -0.006 | -0.006 | 0.011 | 0.000 | 121.459 | 99.000 | 99.000 | 0.000 | 0.000 | 90.000 | 5.847 |
| 3 | 0.067 | -0.018 | -0.018 | 0.031 | 0.000 | 146.596 | 99.000 | 99.000 | 0.000 | 0.000 | 90.000 | 11.175 |
| 4 | 0.122 | -0.034 | -0.034 | 0.053 | 0.000 | 172.064 | 99.000 | 99.000 | 0.000 | 0.000 | 90.000 | 15.637 |
| 5 | 0.171 | -0.051 | -0.051 | 0.070 | 0.000 | 189.489 | 99.000 | 99.000 | 0.000 | 0.000 | 90.000 | 18.280 |
| 6 | 0.256 | -0.081 | -0.081 | 0.095 | 0.000 | 211.058 | 99.000 | 99.000 | 0.000 | 0.000 | 90.000 | 21.187 |
| 7 | 0.516 | -0.188 | -0.188 | 0.141 | 0.000 | 253.622 | 99.000 | 99.000 | 0.000 | 0.000 | 90.000 | 26.008 |
| 8 | 0.793 | -0.315 | -0.315 | 0.163 | 0.000 | 284.120 | 99.000 | 99.000 | 0.000 | 0.000 | 90.000 | 28.894 |
| 9 | 1.023 | -0.432 | -0.432 | 0.158 | 0.000 | 302.729 | 99.000 | 99.000 | 0.000 | 0.000 | 90.000 | 30.473 |
| 10 | 1.299 | -0.586 | -0.586 | 0.128 | 0.000 | 319.359 | 99.000 | 99.000 | 0.000 | 0.000 | 90.000 | 31.784 |
| 11 | 1.505 | -0.701 | -0.701 | 0.102 | 0.000 | 329.865 | 99.000 | 99.000 | 0.000 | 0.000 | 90.000 | 32.569 |
| 12 | 1.926 | -0.940 | -0.940 | 0.046 | 0.000 | 347.259 | 99.000 | 99.000 | 0.000 | 0.000 | 90.000 | 33.801 |
| 13 | 2.147 | -1.067 | -1.067 | 0.013 | 0.000 | 354.481 | 99.000 | 99.000 | 0.000 | 0.000 | 90.000 | 34.290 |
| 14 | 2.681 | -1.385 | -1.385 | -0.088 | 0.000 | 368.453 | 99.000 | 99.000 | 0.000 | 0.000 | 90.000 | 35.200 |
| 15 | 2.796 | -1.459 | -1.459 | -0.123 | 0.000 | 370.076 | 99.000 | 99.000 | 0.000 | 0.000 | 90.000 | 35.303 |
| 16 | 3.040 | -1.612 | -1.612 | -0.185 | 0.000 | 377.682 | 99.000 | 99.000 | 0.000 | 0.000 | 90.000 | 35.777 |
| 17 | 3.275 | -1.763 | -1.763 | -0.251 | 0.000 | 383.226 | 99.000 | 99.000 | 0.000 | 0.000 | 90.000 | 36.115 |
| 18 | 3.623 | -1.990 | -1.990 | -0.356 | 0.000 | 391.969 | 99.000 | 99.000 | 0.000 | 0.000 | 90.000 | 36.635 |
| 19 | 3.836 | -2.134 | -2.134 | -0.431 | 0.000 | 394.115 | 99.000 | 99.000 | 0.000 | 0.000 | 90.000 | 36.760 |
| 20 | 4.066 | -2.284 | -2.284 | -0.502 | 0.000 | 397.539 | 99.000 | 99.000 | 0.000 | 0.000 | 90.000 | 36.959 |
| 21 | 4.294 | -2.435 | -2.435 | -0.576 | 0.000 | 399.598 | 99.000 | 99.000 | 0.000 | 0.000 | 90.000 | 37.077 |
| 22 | 4.736 | -2.731 | -2.731 | -0.726 | 0.000 | 400.410 | 99.000 | 99.000 | 0.000 | 0.000 | 90.000 | 37.123 |
| 23 | 5.197 | -3.043 | -3.043 | -0.889 | 0.000 | 402.735 | 99.000 | 99.000 | 0.000 | 0.000 | 90.000 | 37.255 |
| 24 | 5.528 | -3.272 | -3.272 | -1.016 | 0.000 | 401.943 | 99.000 | 99.000 | 0.000 | 0.000 | 90.000 | 37.211 |
| 25 | 5.679 | -3.374 | -3.374 | -1.069 | 0.000 | 401.954 | 99.000 | 99.000 | 0.000 | 0.000 | 90.000 | 37.211 |
| 26 | 6.509 | -3.930 | -3.930 | -1.351 | 0.000 | 398.452 | 99.000 | 99.000 | 0.000 | 0.000 | 90.000 | 37.011 |
| 27 | 6.676 | -4.042 | -4.042 | -1.408 | 0.000 | 395.823 | 99.000 | 99.000 | 0.000 | 0.000 | 90.000 | 36.860 |
| 28 | 6.794 | -4.116 | -4.116 | -1.439 | 0.000 | 390.541 | 99.000 | 99.000 | 0.000 | 0.000 | 90.000 | 36.551 |
| 29 | 7.065 | -4.296 | -4.296 | -1.527 | 0.000 | 368.339 | 99.000 | 99.000 | 0.000 | 0.000 | 90.000 | 35.192 |
| 30 | 7.422 | -4.494 | -4.494 | -1.566 | 0.000 | 347.776 | 99.000 | 99.000 | 0.000 | 0.000 | 90.000 | 33.836 |
| 31 | 8.479 | -5.033 | -5.033 | -1.586 | 0.000 | 339.222 | 99.000 | 99.000 | 0.000 | 0.000 | 90.000 | 33.242 |
| 32 | 9.623 | -5.606 | -5.606 | -1.588 | 0.000 | 337.768 | 99.000 | 99.000 | 0.000 | 0.000 | 90.000 | 33.139 |
| 32 | 10.117 | -5.852 | -5.852 | -1.586 | 0.000 | 335.849 | 99.000 | 99.000 | 0.000 | 0.000 | 90.000 | 33.002 |

# Triaxial Test on Fine Nevada Sand with a Target Void Ratio $=\mathbf{0 . 5 3 0}\left(\mathrm{D}_{\mathrm{r}}=\mathbf{9 1 . 2 8 \%}\right)$ 

| Test No.: | 11, W5 (ext) |
| :--- | :---: |
| Test Date: | $2 / 9 / 11$ |
| Sector: | III |
| Initial Void Ratio, e: | 0.534 |
| Final Mean Princpal Stress, $\sigma_{\mathrm{m}}:$ | 51.2 kPa |
| Max Friction Angle, $\phi:$ | 35.9 deg |
| b-value at failure: | 0.00 |
| Stress direction at failure, $\alpha:$ | 90.0 deg |


| Shear Band Notes |  |
| :--- | :--- |
| Point of Observation: | 32 |
| Inclination (from Vertical): |  |
| $20^{\circ}$ |  |
| Failure Notes: |  |


| Point (No.) | $\begin{gathered} \varepsilon_{\mathrm{z}} \\ (\%) \end{gathered}$ | $\begin{gathered} \varepsilon_{\mathrm{x}} \\ (\%) \end{gathered}$ | $\begin{array}{r} \varepsilon_{y} \\ (\%) \end{array}$ | $\begin{gathered} \varepsilon_{\mathrm{v}} \\ (\%) \end{gathered}$ | $\begin{aligned} & \gamma_{\theta z} \\ & (\%) \\ & \hline \end{aligned}$ | $\begin{gathered} \sigma_{z} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \sigma_{\mathrm{x}} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \sigma_{\mathrm{y}} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \tau_{\text {өz }} \\ (\mathrm{kPa}) \end{gathered}$ | b | $\begin{gathered} \alpha \\ \left(^{\circ}\right) \\ \hline \end{gathered}$ | $\begin{gathered} \varphi \\ \left({ }^{\circ}\right) \\ \hline \end{gathered}$ |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 1 | 0.000 | 0.000 | 0.000 | 0.000 | 0.000 | 114.587 | 101.000 | 101.000 | 0.000 | 0.000 | 90.000 | -3.613 |
| 2 | 0.002 | -0.003 | -0.003 | -0.004 | 0.000 | 96.611 | 101.000 | 101.000 | 0.000 | 0.000 | 90.000 | . 273 |
| 3 | 0.009 | -0.008 | -0.008 | -0.007 | 0.000 | 80.785 | 101.000 | 101.000 | 0.000 | 0.000 | 90.000 | 6.385 |
| 4 | 0.021 | -0.014 | -0.014 | -0.007 | 0.000 | 71.626 | 101.000 | 101.000 | 0.000 | 0.000 | 90.000 | 9.797 |
| 5 | 0.038 | -0.019 | -0.019 | 0.000 | 0.000 | 62.859 | 101.000 | 101.000 | 0.000 | 0.000 | 90.000 | 13.460 |
| 6 | 0.054 | -0.023 | -0.023 | 0.009 | 0.000 | 58.992 | 101.000 | 101.000 | 0.000 | 0.000 | 90.000 | 15.222 |
| 7 | 0.100 | -0.031 | -0.031 | 0.039 | 0.000 | 52.477 | 101.000 | 101.000 | 0.000 | 0.000 | 90.000 | 18.431 |
| 8 | 0.154 | -0.036 | -0.036 | 0.083 | 0.000 | 48.026 | 101.000 | 101.000 | 0.000 | 0.000 | 90.000 | 20.822 |
| 9 | 0.203 | -0.043 | -0.043 | 0.117 | 0.000 | 46.443 | 101.000 | 101.000 | 0.000 | 0.000 | 90.000 | 21.717 |
| 10 | 0.275 | -0.061 | -0.061 | 0.152 | 0.000 | 41.956 | 101.000 | 101.000 | 0.000 | 0.000 | 90.000 | 24.395 |
| 11 | 0.303 | -0.071 | -0.071 | 0.161 | 0.000 | 40.772 | 101.000 | 101.000 | 0.000 | 0.000 | 90.000 | 25.139 |
| 12 | 0.365 | -0.097 | -0.097 | 0.172 | 0.000 | 38.912 | 101.000 | 101.000 | 0.000 | 0.000 | 90.000 | 26.344 |
| 13 | 0.402 | -0.116 | -0.116 | 0.169 | 0.000 | 37.560 | 101.000 | 101.000 | 0.000 | 0.000 | 90.000 | 27.248 |
| 14 | 0.472 | -0.158 | -0.158 | 0.156 | 0.000 | 35.008 | 101.000 | 101.000 | 0.000 | 0.000 | 90.000 | 29.026 |
| 15 | 0.501 | -0.179 | -0.179 | 0.143 | 0.000 | 34.835 | 101.000 | 101.000 | 0.000 | 0.000 | 90.000 | 29.150 |
| 16 | 0.556 | -0.219 | -0.219 | 0.117 | 0.000 | 32.913 | 101.000 | 101.000 | 0.000 | 0.000 | 90.000 | 30.560 |
| 17 | 0.606 | -0.259 | -0.259 | 0.087 | 0.000 | 32.113 | 101.000 | 101.000 | 0.000 | 0.000 | 90.000 | 31.165 |
| 18 | 0.664 | -0.306 | -0.306 | 0.052 | 0.000 | 31.436 | 101.000 | 101.000 | 0.000 | 0.000 | 90.000 | 31.686 |
| 19 | 0.747 | -0.378 | -0.378 | -0.009 | 0.000 | 30.180 | 101.000 | 101.000 | 0.000 | 0.000 | 90.000 | 32.675 |
| 20 | 0.782 | -0.406 | -0.406 | -0.030 | 0.000 | 29.396 | 101.000 | 101.000 | 0.000 | 0.000 | 90.000 | 33.307 |
| 21 | 0.811 | -0.432 | -0.432 | -0.052 | 0.000 | 29.336 | 101.000 | 101.000 | 0.000 | 0.000 | 90.000 | 33.356 |
| 22 | 0.852 | -0.470 | -0.470 | -0.087 | 0.000 | 28.619 | 101.000 | 101.000 | 0.000 | 0.000 | 90.000 | 33.946 |
| 23 | 0.915 | -0.526 | -0.526 | -0.137 | 0.000 | 28.356 | 101.000 | 101.000 | 0.000 | 0.000 | 90.000 | 34.165 |
| 24 | 0.963 | -0.571 | -0.571 | -0.178 | 0.000 | 27.608 | 101.000 | 101.000 | 0.000 | 0.000 | 90.000 | 34.797 |
| 25 | 1.008 | -0.613 | -0.613 | -0.217 | 0.000 | 26.921 | 101.000 | 101.000 | 0.000 | 0.000 | 90.000 | 35.387 |
| 26 | 1.158 | -0.753 | -0.753 | -0.348 | 0.000 | 26.433 | 101.000 | 101.000 | 0.000 | 0.000 | 90.000 | 35.813 |
| 27 | 1.195 | -0.789 | -0.789 | -0.382 | 0.000 | 26.419 | 101.000 | 101.000 | 0.000 | 0.000 | 90.000 | 35.826 |
| 28 | 1.246 | -0.838 | -0.838 | -0.430 | 0.000 | 26.307 | 101.000 | 101.000 | 0.000 | 0.000 | 90.000 | 35.924 |
| 29 | 1.295 | -0.891 | -0.891 | -0.487 | 0.000 | 26.393 | 101.000 | 101.000 | 0.000 | 0.000 | 90.000 | 35.849 |
| 30 | 1.375 | -0.955 | -0.955 | -0.534 | 0.000 | 28.276 | 101.000 | 101.000 | 0.000 | 0.000 | 90.000 | 34.232 |
| 31 | 1.375 | -0.955 | -0.955 | -0.534 | 0.000 | 28.276 | 101.000 | 101.000 | 0.000 | 0.000 | 90.000 | 34.232 |
| 32 | 1.478 | -1.026 | -1.026 | -0.574 | 0.000 | 33.490 | 101.000 | 101.000 | 0.000 | 0.000 | 90.000 | 30.131 |
| 33 | 1.562 | -1.072 | -1.072 | -0.582 | 0.000 | 34.874 | 101.000 | 101.000 | 0.000 | 0.000 | 90.000 | 29.122 |

# Varying $\sigma_{3}$ Test on Fine Nevada Sand with a Target Void Ratio $=0.530\left(D_{r}=91.28 \%\right)$ 

| Test No.: | W1, Triaxial 9 |
| :--- | :---: |
| Test Date: | $1 / 17 / 11$ |
| Sector: | III |
| Initial Void Ratio, e: | 0.534 |
| Final Mean Princpal Stress, $\sigma_{\mathrm{m}}:$ | 206.9 kPa |
| Max Friction Angle, $\phi:$ | 37.7 deg |
| b-value at failure: | 0.00 |
| Stress direction at failure, $\alpha:$ | 90.0 deg |


| Shear Band Notes |  |
| :--- | :--- |
| Point of Observation: | 26 |
| Inclination (from Vertical): |  |
| $60^{\circ}, 55^{\circ}$ |  |
|  |  |
| Failure Notes: |  |


| Point (No.) | $\begin{gathered} \varepsilon_{\mathrm{z}} \\ (\%) \\ \hline \end{gathered}$ | $\begin{gathered} \varepsilon_{\mathrm{x}} \\ (\%) \\ \hline \end{gathered}$ | $\begin{array}{r} \varepsilon_{y} \\ (\%) \\ \hline \end{array}$ | $\begin{gathered} \varepsilon_{\mathrm{v}} \\ (\%) \\ \hline \end{gathered}$ | $\begin{gathered} \gamma_{\theta z} \\ (\%) \\ \hline \end{gathered}$ | $\begin{gathered} \sigma_{z} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \sigma_{x} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \sigma_{y} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \tau_{\text {өz }} \\ (\mathrm{kPa}) \end{gathered}$ | b | $\begin{gathered} \alpha \\ \left(^{\circ}\right) \\ \hline \end{gathered}$ | $\begin{gathered} \varphi \\ \left({ }^{\circ}\right) \\ \hline \end{gathered}$ |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 1 | 0.000 | 0.000 | 0.000 | 0.000 | 0.000 | 96.246 | 101.000 | 101.000 | 0.000 | 0.000 | 90.000 | -1.381 |
| 2 | 0.329 | -0.114 | -0.114 | 0.101 | 0.000 | 235.399 | 101.000 | 101.000 | 0.000 | 0.000 | 90.000 | 23.548 |
| 3 | 0.638 | -0.254 | -0.254 | 0.130 | 0.000 | 277.855 | 101.000 | 101.000 | 0.000 | 0.000 | 90.000 | 27.828 |
| 4 | 0.964 | -0.419 | -0.419 | 0.125 | 0.000 | 307.549 | 101.000 | 101.000 | 0.000 | 0.000 | 90.000 | 30.369 |
| 5 | 1.268 | -0.588 | -0.588 | 0.092 | 0.000 | 328.212 | 101.000 | 101.000 | 0.000 | 0.000 | 90.000 | 31.963 |
| 6 | 1.549 | -0.748 | -0.748 | 0.053 | 0.000 | 343.540 | 101.000 | 101.000 | 0.000 | 0.000 | 90.000 | 33.066 |
| 7 | 1.893 | -0.955 | -0.955 | -0.018 | 0.000 | 358.147 | 101.000 | 101.000 | 0.000 | 0.000 | 90.000 | 34.059 |
| 8 | 2.145 | -1.110 | -1.110 | -0.075 | 0.000 | 367.723 | 101.000 | 101.000 | 0.000 | 0.000 | 90.000 | 34.683 |
| 9 | 2.468 | -1.313 | -1.313 | -0.158 | 0.000 | 378.989 | 101.000 | 101.000 | 0.000 | 0.000 | 90.000 | 35.391 |
| 10 | 3.006 | -1.661 | -1.661 | -0.317 | 0.000 | 391.497 | 101.000 | 101.000 | 0.000 | 0.000 | 90.000 | 36.146 |
| 11 | 3.072 | -1.705 | -1.705 | -0.339 | 0.000 | 393.249 | 101.000 | 101.000 | 0.000 | 0.000 | 90.000 | 36.249 |
| 12 | 3.312 | -1.865 | -1.865 | -0.418 | 0.000 | 398.863 | 101.000 | 101.000 | 0.000 | 0.000 | 90.000 | 36.576 |
| 13 | 3.703 | -2.131 | -2.131 | -0.558 | 0.000 | 406.680 | 101.000 | 101.000 | 0.000 | 0.000 | 90.000 | 37.021 |
| 14 | 3.921 | -2.281 | -2.281 | -0.642 | 0.000 | 409.714 | 101.000 | 101.000 | 0.000 | 0.000 | 90.000 | 37.191 |
| 15 | 4.281 | -2.527 | -2.527 | -0.774 | 0.000 | 413.913 | 101.000 | 101.000 | 0.000 | 0.000 | 90.000 | 37.423 |
| 16 | 4.579 | -2.736 | -2.736 | -0.893 | 0.000 | 415.615 | 101.000 | 101.000 | 0.000 | 0.000 | 90.000 | 37.517 |
| 17 | 4.802 | -2.889 | -2.889 | -0.976 | 0.000 | 418.192 | 101.000 | 101.000 | 0.000 | 0.000 | 90.000 | 37.657 |
| 18 | 5.135 | -3.124 | -3.124 | -1.113 | 0.000 | 418.077 | 101.000 | 101.000 | 0.000 | 0.000 | 90.000 | 37.651 |
| 19 | 5.537 | -3.404 | -3.404 | -1.271 | 0.000 | 418.536 | 101.000 | 101.000 | 0.000 | 0.000 | 90.000 | 37.676 |
| 20 | 5.775 | -3.576 | -3.576 | -1.376 | 0.000 | 418.112 | 101.000 | 101.000 | 0.000 | 0.000 | 90.000 | 37.653 |
| 21 | 6.096 | -3.800 | -3.800 | -1.504 | 0.000 | 417.336 | 101.000 | 101.000 | 0.000 | 0.000 | 90.000 | 37.611 |
| 22 | 6.355 | -3.978 | -3.978 | -1.601 | 0.000 | 416.509 | 101.000 | 101.000 | 0.000 | 0.000 | 90.000 | 37.566 |
| 23 | 6.697 | -4.219 | -4.219 | -1.741 | 0.000 | 412.839 | 101.000 | 101.000 | 0.000 | 0.000 | 90.000 | 37.364 |
| 24 | 7.022 | -4.441 | -4.441 | -1.860 | 0.000 | 410.008 | 101.000 | 101.000 | 0.000 | 0.000 | 90.000 | 37.207 |
| 25 | 7.337 | -4.658 | -4.658 | -1.979 | 0.000 | 403.268 | 101.000 | 101.000 | 0.000 | 0.000 | 90.000 | 36.828 |
| 26 | 7.611 | -4.843 | -4.843 | -2.076 | 0.000 | 397.975 | 101.000 | 101.000 | 0.000 | 0.000 | 90.000 | 36.525 |
| 27 | 8.162 | -5.202 | -5.202 | -2.243 | 0.000 | 365.256 | 101.000 | 101.000 | 0.000 | 0.000 | 90.000 | 34.525 |
| 28 | 8.489 | -5.396 | -5.396 | -2.304 | 0.000 | 347.709 | 101.000 | 101.000 | 0.000 | 0.000 | 90.000 | 33.355 |
| 29 | 9.083 | -5.723 | -5.723 | -2.364 | 0.000 | 341.082 | 101.000 | 101.000 | 0.000 | 0.000 | 90.000 | 32.893 |
| 30 | 9.637 | -6.015 | -6.015 | -2.392 | 0.000 | 338.883 | 101.000 | 101.000 | 0.000 | 0.000 | 90.000 | 32.737 |
| 31 | 9.637 | -6.015 | -6.015 | -2.392 | 0.000 | 339.218 | 101.000 | 101.000 | 0.000 | 0.000 | 90.000 | 32.761 |
| 32 | 10.877 | -6.670 | -6.670 | -2.462 | 0.000 | 339.579 | 101.000 | 101.000 | 0.000 | 0.000 | 90.000 | 32.787 |

## Varying $\sigma_{3}$ Test on Fine Nevada Sand with a Target Void Ratio $=0.530\left(D_{r}=91.28 \%\right)$

| Test No.: | W 2 |
| :--- | :--- |
| Test Date: | $1 / 20 / 11$ |
| Sector: | III |
| Initial Void Ratio, e: | 0.528 |
| Final Mean Princpal Stress, $\sigma_{\mathrm{m}}:$ | 96.5 kPa |
| Max Friction Angle, $\phi:$ | 37.6 deg |
| b-value at failure: | 0.00 |
| Stress direction at failure, $\alpha:$ | 90.0 deg |$\quad$| Shear Band Notes |
| :--- |
| Point of Observation: |
| $\mathrm{n} / \mathrm{a}$ |
| Inclination (from Vertical): |
| $\mathrm{n} / \mathrm{a}$ |
|  |
| Failure Notes: |
| $\mathrm{n} / \mathrm{a}$ |


| Point (No.) | $\begin{gathered} \varepsilon_{\mathrm{z}} \\ (\%) \\ \hline \end{gathered}$ | $\begin{gathered} \varepsilon_{\mathrm{x}} \\ (\%) \\ \hline \end{gathered}$ | $\begin{array}{r} \varepsilon_{y} \\ (\%) \\ \hline \end{array}$ | $\begin{gathered} \varepsilon_{\mathrm{v}} \\ (\%) \\ \hline \end{gathered}$ | $\begin{gathered} \gamma_{\theta z} \\ (\%) \\ \hline \end{gathered}$ | $\begin{gathered} \sigma_{z} \\ (\mathrm{kPa}) \\ \hline \end{gathered}$ | $\begin{gathered} \sigma_{x} \\ (\mathrm{kPa}) \\ \hline \end{gathered}$ | $\begin{gathered} \sigma_{\mathrm{y}} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \tau_{\text {өz }} \\ (\mathrm{kPa}) \\ \hline \end{gathered}$ | b | $\begin{gathered} \alpha \\ \left(^{\circ}\right) \\ \hline \end{gathered}$ | $\begin{gathered} \varphi \\ \left({ }^{\circ}\right) \\ \hline \end{gathered}$ |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 1 | 0.000 | 0.000 | 0.000 | 0.000 | 0.000 | 101.014 | 101.036 | 101.036 | 0.000 | 0.000 | 90.000 | -0.006 |
| 2 | 0.028 | -0.012 | -0.012 | 0.004 | 0.000 | 118.519 | 91.418 | 91.418 | 0.000 | 0.000 | 90.000 | 7.417 |
| 3 | 0.061 | -0.027 | -0.027 | 0.006 | 0.000 | 130.722 | 84.566 | 84.566 | 0.000 | 0.000 | 90.000 | 12.380 |
| 4 | 0.096 | -0.044 | -0.044 | 0.008 | 0.000 | 139.864 | 79.610 | 79.610 | 0.000 | 0.000 | 90.000 | 15.935 |
| 5 | 0.163 | -0.073 | -0.073 | 0.017 | 0.000 | 149.934 | 74.506 | 74.506 | 0.000 | 0.000 | 90.000 | 19.638 |
| 6 | 0.233 | -0.106 | -0.106 | 0.021 | 0.000 | 153.888 | 69.353 | 69.353 | 0.000 | 0.000 | 90.000 | 22.251 |
| 7 | 0.343 | -0.160 | -0.160 | 0.023 | 0.000 | 160.108 | 65.967 | 65.967 | 0.000 | 0.000 | 90.000 | 24.608 |
| 8 | 0.470 | -0.223 | -0.223 | 0.023 | 0.000 | 165.223 | 63.710 | 63.710 | 0.000 | 0.000 | 90.000 | 26.322 |
| 9 | 0.616 | -0.301 | -0.301 | 0.015 | 0.000 | 169.293 | 61.551 | 61.551 | 0.000 | 0.000 | 90.000 | 27.822 |
| 10 | 0.759 | -0.377 | -0.377 | 0.004 | 0.000 | 172.644 | 59.784 | 59.784 | 0.000 | 0.000 | 90.000 | 29.050 |
| 11 | 0.917 | -0.468 | -0.468 | -0.019 | 0.000 | 174.457 | 58.165 | 58.165 | 0.000 | 0.000 | 90.000 | 29.995 |
| 12 | 1.084 | -0.564 | -0.564 | -0.044 | 0.000 | 176.389 | 56.693 | 56.693 | 0.000 | 0.000 | 90.000 | 30.900 |
| 13 | 1.243 | -0.659 | -0.659 | -0.074 | 0.000 | 178.344 | 55.613 | 55.613 | 0.000 | 0.000 | 90.000 | 31.640 |
| 14 | 1.419 | -0.765 | -0.765 | -0.112 | 0.000 | 180.073 | 54.534 | 54.534 | 0.000 | 0.000 | 90.000 | 32.351 |
| 15 | 1.574 | -0.862 | -0.862 | -0.150 | 0.000 | 181.175 | 53.896 | 53.896 | 0.000 | 0.000 | 90.000 | 32.783 |
| 16 | 1.737 | -0.964 | -0.964 | -0.191 | 0.000 | 182.699 | 53.159 | 53.159 | 0.000 | 0.000 | 90.000 | 33.314 |
| 17 | 1.888 | -1.059 | -1.059 | -0.231 | 0.000 | 183.934 | 52.669 | 52.669 | 0.000 | 0.000 | 90.000 | 33.696 |
| 18 | 2.046 | -1.161 | -1.161 | -0.277 | 0.000 | 184.115 | 51.687 | 51.687 | 0.000 | 0.000 | 90.000 | 34.167 |
| 19 | 2.213 | -1.271 | -1.271 | -0.328 | 0.000 | 185.040 | 51.295 | 51.295 | 0.000 | 0.000 | 90.000 | 34.466 |
| 20 | 2.397 | -1.390 | -1.390 | -0.383 | 0.000 | 184.715 | 51.295 | 51.295 | 0.000 | 0.000 | 90.000 | 34.424 |
| 21 | 2.549 | -1.491 | -1.491 | -0.434 | 0.000 | 185.953 | 50.313 | 50.313 | 0.000 | 0.000 | 90.000 | 35.036 |
| 22 | 2.699 | -1.590 | -1.590 | -0.481 | 0.000 | 186.920 | 49.773 | 49.773 | 0.000 | 0.000 | 90.000 | 35.410 |
| 23 | 2.869 | -1.703 | -1.703 | -0.536 | 0.000 | 187.662 | 49.234 | 49.234 | 0.000 | 0.000 | 90.000 | 35.757 |
| 24 | 3.025 | -1.808 | -1.808 | -0.591 | 0.000 | 188.772 | 48.694 | 48.694 | 0.000 | 0.000 | 90.000 | 36.149 |
| 25 | 3.180 | -1.913 | -1.913 | -0.646 | 0.000 | 189.973 | 49.136 | 49.136 | 0.000 | 0.000 | 90.000 | 36.087 |
| 26 | 3.336 | -2.016 | -2.016 | -0.697 | 0.000 | 192.045 | 49.037 | 49.037 | 0.000 | 0.000 | 90.000 | 36.369 |
| 27 | 3.481 | -2.116 | -2.116 | -0.750 | 0.000 | 192.868 | 48.743 | 48.743 | 0.000 | 0.000 | 90.000 | 36.632 |
| 28 | 3.642 | -2.224 | -2.224 | -0.807 | 0.000 | 193.284 | 48.252 | 48.252 | 0.000 | 0.000 | 90.000 | 36.965 |
| 29 | 3.789 | -2.327 | -2.327 | -0.864 | 0.000 | 194.423 | 47.418 | 47.418 | 0.000 | 0.000 | 90.000 | 37.355 |
| 30 | 3.937 | -2.427 | -2.427 | -0.917 | 0.000 | 195.032 | 47.124 | 47.124 | 0.000 | 0.000 | 90.000 | 37.613 |

$$
\text { Varying } \sigma_{3} \text { Test on Fine Nevada Sand with a Target Void Ratio }=0.530\left(D_{r}=91.28 \%\right)
$$

| Test No.: | W3 |
| :--- | :---: |
| Test Date: | $1 / 24 / 11$ |
| Sector: | III |
| Initial Void Ratio, e: | 0.530 |
| Final Mean Princpal Stress, $\sigma_{\mathrm{m}}:$ | 100.8 kPa |
| Max Friction Angle, $\phi:$ | 35.3 deg |
| b-value at failure: | 0.00 |
| Stress direction at failure, $\alpha:$ | 90.0 deg |


| Shear Band Notes <br> Point of Observation: <br> $\mathrm{n} / \mathrm{a}$ <br> Inclination (from Vertical): <br> $\mathrm{n} / \mathrm{a}$ <br> Failure Notes: <br> $\mathrm{n} / \mathrm{a}$ |
| :--- |


| Point (No.) | $\begin{gathered} \varepsilon_{\mathrm{z}} \\ (\%) \end{gathered}$ | $\begin{gathered} \varepsilon_{\mathrm{x}} \\ (\%) \end{gathered}$ | $\begin{gathered} \varepsilon_{\mathrm{y}} \\ (\%) \end{gathered}$ | $\begin{gathered} \varepsilon_{\mathrm{v}} \\ (\%) \end{gathered}$ | $\begin{aligned} & \gamma_{\theta z} \\ & (\%) \end{aligned}$ | $\begin{gathered} \sigma_{z} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \sigma_{x} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \sigma_{\mathrm{y}} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \tau_{\theta z} \\ (\mathrm{kPa}) \end{gathered}$ | b | $\begin{gathered} \alpha \\ \left({ }^{\circ}\right) \end{gathered}$ | $\begin{gathered} \varphi \\ \left({ }^{\circ}\right) \end{gathered}$ |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 1 | 0.000 | 0.000 | 0.000 | 0.000 | 0.000 | 100.277 | 100.300 | 100.300 | 0.000 | 0.000 | 90.000 | -0.007 |
| 2 | 0.025 | -0.009 | -0.009 | 0.007 | 0.000 | 117.064 | 90.945 | 90.945 | 0.000 | 0.000 | 90.000 | 7.213 |
| 3 | 0.055 | -0.023 | -0.023 | 0.009 | 0.000 | 130.658 | 85.547 | 85.547 | 0.000 | 0.000 | 90.000 | 12.043 |
| 4 | 0.112 | -0.047 | -0.047 | 0.017 | 0.000 | 141.748 | 79.168 | 79.168 | 0.000 | 0.000 | 90.000 | 16.456 |
| 5 | 0.196 | -0.083 | -0.083 | 0.030 | 0.000 | 151.010 | 74.751 | 74.751 | 0.000 | 0.000 | 90.000 | 19.742 |
| 6 | 0.292 | -0.127 | -0.127 | 0.039 | 0.000 | 157.579 | 71.807 | 71.807 | 0.000 | 0.000 | 90.000 | 21.958 |
| 7 | 0.423 | -0.189 | -0.189 | 0.046 | 0.000 | 163.902 | 68.666 | 68.666 | 0.000 | 0.000 | 90.000 | 24.173 |
| 8 | 0.531 | -0.241 | -0.241 | 0.048 | 0.000 | 167.948 | 66.066 | 66.066 | 0.000 | 0.000 | 90.000 | 25.809 |
| 9 | 0.710 | -0.331 | -0.331 | 0.048 | 0.000 | 174.192 | 64.446 | 64.446 | 0.000 | 0.000 | 90.000 | 27.380 |
| 10 | 0.845 | -0.401 | -0.401 | 0.043 | 0.000 | 177.062 | 62.238 | 62.238 | 0.000 | 0.000 | 90.000 | 28.675 |
| 11 | 0.981 | -0.474 | -0.474 | 0.034 | 0.000 | 179.497 | 61.109 | 61.109 | 0.000 | 0.000 | 90.000 | 29.475 |
| 12 | 1.158 | -0.570 | -0.570 | 0.017 | 0.000 | 182.922 | 59.735 | 59.735 | 0.000 | 0.000 | 90.000 | 30.508 |
| 13 | 1.274 | -0.636 | -0.636 | 0.002 | 0.000 | 184.939 | 58.754 | 58.754 | 0.000 | 0.000 | 90.000 | 31.185 |
| 14 | 1.452 | -0.739 | -0.739 | -0.026 | 0.000 | 187.396 | 57.576 | 57.576 | 0.000 | 0.000 | 90.000 | 32.001 |
| 15 | 1.612 | -0.832 | -0.832 | -0.052 | 0.000 | 189.195 | 56.840 | 56.840 | 0.000 | 0.000 | 90.000 | 32.544 |
| 16 | 1.790 | -0.938 | -0.938 | -0.087 | 0.000 | 190.604 | 55.760 | 55.760 | 0.000 | 0.000 | 90.000 | 33.184 |
| 17 | 1.948 | -1.035 | -1.035 | -0.122 | 0.000 | 191.854 | 55.613 | 55.613 | 0.000 | 0.000 | 90.000 | 33.404 |
| 18 | 2.113 | -1.141 | -1.141 | -0.169 | 0.000 | 192.075 | 55.024 | 55.024 | 0.000 | 0.000 | 90.000 | 33.686 |
| 19 | 2.263 | -1.234 | -1.234 | -0.204 | 0.000 | 192.601 | 54.337 | 54.337 | 0.000 | 0.000 | 90.000 | 34.050 |
| 20 | 2.421 | -1.333 | -1.333 | -0.246 | 0.000 | 193.517 | 53.699 | 53.699 | 0.000 | 0.000 | 90.000 | 34.442 |
| 21 | 2.592 | -1.441 | -1.441 | -0.290 | 0.000 | 194.378 | 53.123 | 53.123 | 0.000 | 0.000 | 90.000 | 34.801 |
| 22 | 2.742 | -1.537 | -1.537 | -0.332 | 0.000 | 195.264 | 52.644 | 52.644 | 0.000 | 0.000 | 90.000 | 35.120 |
| 23 | 2.880 | -1.627 | -1.627 | -0.374 | 0.000 | 195.994 | 52.607 | 52.607 | 0.000 | 0.000 | 90.000 | 35.224 |
| 24 | 3.063 | -1.740 | -1.740 | -0.417 | 0.000 | 196.593 | 52.423 | 52.423 | 0.000 | 0.000 | 90.000 | 35.377 |
| 25 | 3.225 | -1.846 | -1.846 | -0.467 | 0.000 | 196.891 | 52.730 | 52.730 | 0.000 | 0.000 | 90.000 | 35.276 |
| 26 | 3.415 | -1.982 | -1.982 | -0.550 | 0.000 | 194.942 | 55.490 | 55.490 | 0.000 | 0.000 | 90.000 | 33.838 |

## Varying $\sigma_{3}$ Test on Fine Nevada Sand with a Target Void Ratio $=0.530\left(D_{r}=91.28 \%\right)$

| Test No.: | W4 |
| :--- | :--- |
| Test Date: | $1 / 27 / 11$ |
| Sector: | III |
| Initial Void Ratio, e: | 0.554 |
| Final Mean Princpal Stress, $\sigma_{\mathrm{m}}:$ | 52.3 kPa |
| Max Friction Angle, $\phi:$ | 34.7 deg |
| b-value at failure: | 0.00 |
| Stress direction at failure, $\alpha:$ | 90.0 deg |$\quad$| Shear Band Notes |
| :--- |
| Point of Observation: |
| $\mathrm{n} / \mathrm{a}$ |
| Inclination (from Vertical): |
| $\mathrm{n} / \mathrm{a}$ |
|  |
| Failure Notes: |
| $\mathrm{n} / \mathrm{a}$ |


| Point (No.) | $\begin{gathered} \varepsilon_{\mathrm{z}} \\ (\%) \end{gathered}$ | $\begin{gathered} \varepsilon_{\mathrm{x}} \\ (\%) \\ \hline \end{gathered}$ | $\begin{array}{r} \varepsilon_{y} \\ (\%) \\ \hline \end{array}$ | $\begin{gathered} \varepsilon_{\mathrm{v}} \\ (\%) \\ \hline \end{gathered}$ | $\begin{gathered} \gamma_{\theta z} \\ (\%) \\ \hline \end{gathered}$ | $\begin{gathered} \sigma_{z} \\ (\mathrm{kPa}) \\ \hline \end{gathered}$ | $\begin{gathered} \sigma_{\mathrm{x}} \\ (\mathrm{kPa}) \\ \hline \end{gathered}$ | $\begin{gathered} \sigma_{y} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \tau_{\text {өz }} \\ (\mathrm{kPa}) \end{gathered}$ | b | $\begin{gathered} \alpha \\ \left({ }^{\circ}\right) \end{gathered}$ | $\begin{gathered} \varphi \\ \left({ }^{\circ}\right) \\ \hline \end{gathered}$ |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 1 | 0.000 | 0.000 | 0.000 | 0.000 | 0.000 | 100.977 | 101.000 | 101.000 | 0.000 | 0.000 | 90.000 | -0.007 |
| 2 | 0.004 | -0.007 | -0.007 | -0.009 | 0.000 | 102.031 | 79.100 | 79.100 | 0.000 | 0.000 | 90.000 | 7.273 |
| 3 | 0.012 | -0.021 | -0.021 | -0.031 | 0.000 | 101.033 | 68.812 | 68.812 | 0.000 | 0.000 | 90.000 | 10.936 |
| 4 | 0.012 | -0.028 | -0.028 | -0.044 | 0.000 | 101.693 | 60.809 | 60.809 | 0.000 | 0.000 | 90.000 | 14.572 |
| 5 | 0.024 | -0.043 | -0.043 | -0.062 | 0.000 | 100.992 | 54.149 | 54.149 | 0.000 | 0.000 | 90.000 | 17.574 |
| 6 | 0.051 | -0.062 | -0.062 | -0.073 | 0.000 | 100.967 | 49.021 | 49.021 | 0.000 | 0.000 | 90.000 | 20.263 |
| 7 | 0.115 | -0.098 | -0.098 | -0.081 | 0.000 | 101.024 | 44.793 | 44.793 | 0.000 | 0.000 | 90.000 | 22.682 |
| 8 | 0.220 | -0.154 | -0.154 | -0.088 | 0.000 | 101.028 | 41.698 | 41.698 | 0.000 | 0.000 | 90.000 | 24.563 |
| 9 | 0.346 | -0.222 | -0.222 | -0.097 | 0.000 | 100.964 | 39.308 | 39.308 | 0.000 | 0.000 | 90.000 | 26.075 |
| 10 | 0.470 | -0.291 | -0.291 | -0.112 | 0.000 | 100.959 | 37.497 | 37.497 | 0.000 | 0.000 | 90.000 | 27.281 |
| 11 | 0.641 | -0.389 | -0.389 | -0.137 | 0.000 | 100.983 | 36.296 | 36.296 | 0.000 | 0.000 | 90.000 | 28.113 |
| 12 | 0.784 | -0.473 | -0.473 | -0.161 | 0.000 | 101.034 | 35.214 | 35.214 | 0.000 | 0.000 | 90.000 | 28.888 |
| 13 | 0.921 | -0.553 | -0.553 | -0.185 | 0.000 | 101.063 | 34.301 | 34.301 | 0.000 | 0.000 | 90.000 | 29.551 |
| 14 | 1.043 | -0.626 | -0.626 | -0.209 | 0.000 | 100.961 | 33.570 | 33.570 | 0.000 | 0.000 | 90.000 | 30.062 |
| 15 | 1.189 | -0.716 | -0.716 | -0.242 | 0.000 | 101.024 | 32.770 | 32.770 | 0.000 | 0.000 | 90.000 | 30.673 |
| 16 | 1.370 | -0.824 | -0.824 | -0.278 | 0.000 | 100.091 | 32.292 | 32.292 | 0.000 | 0.000 | 90.000 | 30.806 |
| 17 | 1.417 | -0.857 | -0.857 | -0.297 | 0.000 | 101.132 | 31.417 | 31.417 | 0.000 | 0.000 | 90.000 | 31.732 |
| 18 | 1.530 | -0.926 | -0.926 | -0.322 | 0.000 | 101.414 | 30.249 | 30.249 | 0.000 | 0.000 | 90.000 | 32.718 |
| 19 | 1.667 | -1.016 | -1.016 | -0.366 | 0.000 | 101.172 | 29.959 | 29.959 | 0.000 | 0.000 | 90.000 | 32.893 |
| 20 | 1.816 | -1.117 | -1.117 | -0.417 | 0.000 | 101.816 | 29.586 | 29.586 | 0.000 | 0.000 | 90.000 | 33.346 |
| 21 | 2.055 | -1.266 | -1.266 | -0.477 | 0.000 | 100.774 | 29.505 | 29.505 | 0.000 | 0.000 | 90.000 | 33.164 |
| 22 | 2.116 | -1.309 | -1.309 | -0.503 | 0.000 | 100.858 | 29.815 | 29.815 | 0.000 | 0.000 | 90.000 | 32.933 |
| 23 | 2.247 | -1.406 | -1.406 | -0.566 | 0.000 | 102.218 | 31.908 | 31.908 | 0.000 | 0.000 | 90.000 | 33.336 |
| 24 | 2.335 | -1.458 | -1.458 | -0.581 | 0.000 | 101.068 | 28.227 | 28.227 | 0.000 | 0.000 | 90.000 | 34.289 |
| 25 | 2.474 | -1.551 | -1.551 | -0.629 | 0.000 | 100.771 | 28.128 | 28.128 | 0.000 | 0.000 | 90.000 | 34.303 |
| 26 | 2.384 | -1.491 | -1.491 | -0.599 | 0.000 | 101.344 | 27.769 | 27.769 | 0.000 | 0.000 | 90.000 | 34.740 |

$$
\text { Varying } \sigma_{3} \text { Test on Fine Nevada Sand with a Target Void Ratio }=0.530\left(D_{r}=91.28 \%\right)
$$

| Test No.: | W5, Triaxial 11 |
| :--- | :---: |
| Test Date: | $2 / 9 / 11$ |
| Sector: | III |
| Initial Void Ratio, e: | 0.534 |
| Final Mean Princpal Stress, $\sigma_{\mathrm{m}}:$ | 51.2 kPa |
| Max Friction Angle, $\phi:$ | 35.9 deg |
| b-value at failure: | 0.00 |
| Stress direction at failure, $\alpha:$ | 90.0 deg |


| Shear Band Notes <br> Point of Observation: <br> $\mathrm{n} / \mathrm{a}$ <br> Inclination (from Vertical): <br> $\mathrm{n} / \mathrm{a}$ <br> Failure Notes: <br> $\mathrm{n} / \mathrm{a}$ |
| :--- | :--- |


| Point (No.) | $\begin{gathered} \varepsilon_{\mathrm{z}} \\ (\%) \end{gathered}$ | $\begin{gathered} \varepsilon_{\mathrm{x}} \\ (\%) \end{gathered}$ | $\begin{array}{r} \varepsilon_{\mathrm{y}} \\ (\%) \end{array}$ | $\begin{gathered} \varepsilon_{\mathrm{v}} \\ (\%) \\ \hline \end{gathered}$ | $\begin{gathered} \gamma_{\theta z} \\ (\%) \\ \hline \end{gathered}$ | $\begin{gathered} \sigma_{z} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \sigma_{x} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \sigma_{\mathrm{y}} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \tau_{\theta z} \\ (\mathrm{kPa}) \\ \hline \end{gathered}$ | b | $\begin{gathered} \alpha \\ \left(^{\circ}\right) \\ \hline \end{gathered}$ | $\begin{gathered} \varphi \\ \left({ }^{\circ}\right) \\ \hline \end{gathered}$ |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 1 | 0.000 | 0.000 | 0.000 | 0.000 | 0.000 | 114.587 | 101.000 | 101.000 | 0.000 | 0.000 | 90.000 | -3.613 |
| 2 | 0.002 | -0.003 | -0.003 | -0.004 | 0.000 | 96.611 | 101.000 | 101.000 | 0.000 | 0.000 | 90.000 | 1.273 |
| 3 | 0.009 | -0.008 | -0.008 | -0.007 | 0.000 | 80.785 | 101.000 | 101.000 | 0.000 | 0.000 | 90.000 | 6.385 |
| 4 | 0.021 | -0.014 | -0.014 | -0.007 | 0.000 | 71.626 | 101.000 | 101.000 | 0.000 | 0.000 | 90.000 | 9.797 |
| 5 | 0.038 | -0.019 | -0.019 | 0.000 | 0.000 | 62.859 | 101.000 | 101.000 | 0.000 | 0.000 | 90.000 | 13.460 |
| 6 | 0.054 | -0.023 | -0.023 | 0.009 | 0.000 | 58.992 | 101.000 | 101.000 | 0.000 | 0.000 | 90.000 | 15.222 |
| 7 | 0.100 | -0.031 | -0.031 | 0.039 | 0.000 | 52.477 | 101.000 | 101.000 | 0.000 | 0.000 | 90.000 | 18.431 |
| 8 | 0.154 | -0.036 | -0.036 | 0.083 | 0.000 | 48.026 | 101.000 | 101.000 | 0.000 | 0.000 | 90.000 | 20.822 |
| 9 | 0.203 | -0.043 | -0.043 | 0.117 | 0.000 | 46.443 | 101.000 | 101.000 | 0.000 | 0.000 | 90.000 | 21.717 |
| 10 | 0.275 | -0.061 | -0.061 | 0.152 | 0.000 | 41.956 | 101.000 | 101.000 | 0.000 | 0.000 | 90.000 | 24.395 |
| 11 | 0.303 | -0.071 | -0.071 | 0.161 | 0.000 | 40.772 | 101.000 | 101.000 | 0.000 | 0.000 | 90.000 | 25.139 |
| 12 | 0.365 | -0.097 | -0.097 | 0.172 | 0.000 | 38.912 | 101.000 | 101.000 | 0.000 | 0.000 | 90.000 | 26.344 |
| 13 | 0.402 | -0.116 | -0.116 | 0.169 | 0.000 | 37.560 | 101.000 | 101.000 | 0.000 | 0.000 | 90.000 | 27.248 |
| 14 | 0.472 | -0.158 | -0.158 | 0.156 | 0.000 | 35.008 | 101.000 | 101.000 | 0.000 | 0.000 | 90.000 | 29.026 |
| 15 | 0.501 | -0.179 | -0.179 | 0.143 | 0.000 | 34.835 | 101.000 | 101.000 | 0.000 | 0.000 | 90.000 | 29.150 |
| 16 | 0.556 | -0.219 | -0.219 | 0.117 | 0.000 | 32.913 | 101.000 | 101.000 | 0.000 | 0.000 | 90.000 | 30.560 |
| 17 | 0.606 | -0.259 | -0.259 | 0.087 | 0.000 | 32.113 | 101.000 | 101.000 | 0.000 | 0.000 | 90.000 | 31.165 |
| 18 | 0.664 | -0.306 | -0.306 | 0.052 | 0.000 | 31.436 | 101.000 | 101.000 | 0.000 | 0.000 | 90.000 | 31.686 |
| 19 | 0.747 | -0.378 | -0.378 | -0.009 | 0.000 | 30.180 | 101.000 | 101.000 | 0.000 | 0.000 | 90.000 | 32.675 |
| 20 | 0.782 | -0.406 | -0.406 | -0.030 | 0.000 | 29.396 | 101.000 | 101.000 | 0.000 | 0.000 | 90.000 | 33.307 |
| 21 | 0.811 | -0.432 | -0.432 | -0.052 | 0.000 | 29.336 | 101.000 | 101.000 | 0.000 | 0.000 | 90.000 | 33.356 |
| 22 | 0.852 | -0.470 | -0.470 | -0.087 | 0.000 | 28.619 | 101.000 | 101.000 | 0.000 | 0.000 | 90.000 | 33.946 |
| 23 | 0.915 | -0.526 | -0.526 | -0.137 | 0.000 | 28.356 | 101.000 | 101.000 | 0.000 | 0.000 | 90.000 | 34.165 |
| 24 | 0.963 | -0.571 | -0.571 | -0.178 | 0.000 | 27.608 | 101.000 | 101.000 | 0.000 | 0.000 | 90.000 | 34.797 |
| 25 | 1.008 | -0.613 | -0.613 | -0.217 | 0.000 | 26.921 | 101.000 | 101.000 | 0.000 | 0.000 | 90.000 | 35.387 |
| 26 | 1.158 | -0.753 | -0.753 | -0.348 | 0.000 | 26.433 | 101.000 | 101.000 | 0.000 | 0.000 | 90.000 | 35.813 |
| 27 | 1.195 | -0.789 | -0.789 | -0.382 | 0.000 | 26.419 | 101.000 | 101.000 | 0.000 | 0.000 | 90.000 | 35.826 |
| 28 | 1.246 | -0.838 | -0.838 | -0.430 | 0.000 | 26.307 | 101.000 | 101.000 | 0.000 | 0.000 | 90.000 | 35.924 |
| 29 | 1.295 | -0.891 | -0.891 | -0.487 | 0.000 | 26.393 | 101.000 | 101.000 | 0.000 | 0.000 | 90.000 | 35.849 |
| 30 | 1.375 | -0.955 | -0.955 | -0.534 | 0.000 | 28.276 | 101.000 | 101.000 | 0.000 | 0.000 | 90.000 | 34.232 |
| 31 | 1.375 | -0.955 | -0.955 | -0.534 | 0.000 | 28.276 | 101.000 | 101.000 | 0.000 | 0.000 | 90.000 | 34.232 |
| 32 | 1.478 | -1.026 | -1.026 | -0.574 | 0.000 | 33.490 | 101.000 | 101.000 | 0.000 | 0.000 | 90.000 | 30.131 |
| 33 | 1.562 | -1.072 | -1.072 | -0.582 | 0.000 | 34.874 | 101.000 | 101.000 | 0.000 | 0.000 | 90.000 | 29.122 |

## Varying $\sigma_{3}$ Test on Fine Nevada Sand with a Target Void Ratio $=0.530\left(D_{r}=\mathbf{9 1 . 2 8 \%}\right)$

| Test No.: | W6 |
| :--- | :--- |
| Test Date: | $1 / 31 / 11$ |
| Sector: | I |
| Initial Void Ratio, e: | 0.534 |
| Final Mean Princpal Stress, $\sigma_{\mathrm{m}}:$ | 98.4 kPa |
| Max Friction Angle, $\phi:$ | 39.6 deg |
| b-value at failure: | 0.00 |
| Stress direction at failure, $\alpha$ : | 0.0 deg |$\quad$| Shear Band Notes |
| :--- |
| Point of Observation: |
| $\mathrm{n} / \mathrm{a}$ |
| Inclination (from Vertical): |
| $\mathrm{n} / \mathrm{a}$ |
|  |
|  |


| Point (No.) | $\begin{gathered} \varepsilon_{\mathrm{z}} \\ (\%) \end{gathered}$ | $\begin{gathered} \varepsilon_{\mathrm{x}} \\ (\%) \\ \hline \end{gathered}$ | $\begin{gathered} \varepsilon_{\mathrm{y}} \\ (\%) \end{gathered}$ | $\begin{gathered} \varepsilon_{\mathrm{v}} \\ (\%) \end{gathered}$ | $\begin{aligned} & \gamma_{\theta z} \\ & (\%) \end{aligned}$ | $\begin{gathered} \sigma_{z} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \sigma_{\mathrm{x}} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \sigma_{\mathrm{y}} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \tau_{\theta z} \\ (\mathrm{kPa}) \end{gathered}$ | b | $\begin{gathered} \alpha \\ \left(^{( }\right) \\ \hline \end{gathered}$ | $\begin{gathered} \varphi \\ \left({ }^{\circ}\right) \\ \hline \end{gathered}$ |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 1 | 0.000 | 0.000 | 0.000 | 0.000 | 0.000 | 101.013 | 101.036 | 101.036 | 0.000 | 0.000 | 0.000 | -0.007 |
| 2 | 0.046 | -0.022 | -0.022 | 0.003 | 0.000 | 120.076 | 90.945 | 90.945 | 0.000 | 0.000 | 0.000 | 7.935 |
| 3 | 0.076 | -0.035 | -0.035 | 0.007 | 0.000 | 132.285 | 85.057 | 85.057 | 0.000 | 0.000 | 0.000 | 12.551 |
| 4 | 0.107 | -0.049 | -0.049 | 0.009 | 0.000 | 141.467 | 80.149 | 80.149 | 0.000 | 0.000 | 0.000 | 16.062 |
| 5 | 0.156 | -0.073 | -0.073 | 0.011 | 0.000 | 152.063 | 75.340 | 75.340 | 0.000 | 0.000 | 0.000 | 19.718 |
| 6 | 0.208 | -0.098 | -0.098 | 0.013 | 0.000 | 159.533 | 71.611 | 71.611 | 0.000 | 0.000 | 0.000 | 22.357 |
| 7 | 0.290 | -0.140 | -0.140 | 0.010 | 0.000 | 167.279 | 67.440 | 67.440 | 0.000 | 0.000 | 0.000 | 25.173 |
| 8 | 0.372 | -0.185 | -0.185 | 0.002 | 0.000 | 173.368 | 64.299 | 64.299 | 0.000 | 0.000 | 0.000 | 27.317 |
| 9 | 0.488 | -0.255 | -0.255 | -0.022 | 0.000 | 179.600 | 60.619 | 60.619 | 0.000 | 0.000 | 0.000 | 29.690 |
| 10 | 0.590 | -0.311 | -0.311 | -0.033 | 0.000 | 183.503 | 59.343 | 59.343 | 0.000 | 0.000 | 0.000 | 30.749 |
| 11 | 0.701 | -0.380 | -0.380 | -0.059 | 0.000 | 187.347 | 57.674 | 57.674 | 0.000 | 0.000 | 0.000 | 31.954 |
| 12 | 0.837 | -0.466 | -0.466 | -0.095 | 0.000 | 190.969 | 55.466 | 55.466 | 0.000 | 0.000 | 0.000 | 33.357 |
| 13 | 0.981 | -0.561 | -0.561 | -0.141 | 0.000 | 193.818 | 54.043 | 54.043 | 0.000 | 0.000 | 0.000 | 34.328 |
| 14 | 1.102 | -0.643 | -0.643 | -0.185 | 0.000 | 196.138 | 53.012 | 53.012 | 0.000 | 0.000 | 0.000 | 35.061 |
| 15 | 1.259 | -0.753 | -0.753 | -0.247 | 0.000 | 198.448 | 51.687 | 51.687 | 0.000 | 0.000 | 0.000 | 35.925 |
| 16 | 1.441 | -0.883 | -0.883 | -0.326 | 0.000 | 200.041 | 50.853 | 50.853 | 0.000 | 0.000 | 0.000 | 36.486 |
| 17 | 1.586 | -0.990 | -0.990 | -0.394 | 0.000 | 201.871 | 49.823 | 49.823 | 0.000 | 0.000 | 0.000 | 37.164 |
| 18 | 1.741 | -1.105 | -1.105 | -0.469 | 0.000 | 203.438 | 48.939 | 48.939 | 0.000 | 0.000 | 0.000 | 37.747 |
| 19 | 1.947 | -1.264 | -1.264 | -0.581 | 0.000 | 203.676 | 48.743 | 48.743 | 0.000 | 0.000 | 0.000 | 37.864 |
| 20 | 2.090 | -1.373 | -1.373 | -0.656 | 0.000 | 204.876 | 48.056 | 48.056 | 0.000 | 0.000 | 0.000 | 38.317 |
| 21 | 2.253 | -1.499 | -1.499 | -0.744 | 0.000 | 205.781 | 47.320 | 47.320 | 0.000 | 0.000 | 0.000 | 38.761 |
| 22 | 2.429 | -1.634 | -1.634 | -0.839 | 0.000 | 205.422 | 47.025 | 47.025 | 0.000 | 0.000 | 0.000 | 38.862 |
| 23 | 2.590 | -1.760 | -1.760 | -0.929 | 0.000 | 206.421 | 46.584 | 46.584 | 0.000 | 0.000 | 0.000 | 39.180 |
| 24 | 2.775 | -1.905 | -1.905 | -1.035 | 0.000 | 206.661 | 46.191 | 46.191 | 0.000 | 0.000 | 0.000 | 39.393 |
| 25 | 2.958 | -2.050 | -2.050 | -1.142 | 0.000 | 205.301 | 45.799 | 45.799 | 0.000 | 0.000 | 0.000 | 39.436 |
| 26 | 3.123 | -2.182 | -2.182 | -1.241 | 0.000 | 205.324 | 44.977 | 44.977 | 0.000 | 0.000 | 0.000 | 39.838 |

$$
\text { Varying } \sigma_{3} \text { Test on Fine Nevada Sand with a Target Void Ratio }=0.530\left(D_{r}=91.28 \%\right)
$$

| Test No.: | A1 |
| :--- | :---: |
| Test Date: | $12 / 9 / 10$ |
| Sector: | III |
| Initial Void Ratio, e: | 0.522 |
| Final Mean Princpal Stress, $\sigma_{\mathrm{m}}:$ | 200.1 kPa |
| Max Friction Angle, $\phi:$ | 32.5 deg |
| b-value at failure: | 0.00 |
| Stress direction at failure, $\alpha:$ | 90.0 deg |


| Shear Band Notes |
| :--- |
| Point of Observation: |
| $\mathrm{n} / \mathrm{a}$ |
| Inclination (from Vertical): |
| $\mathrm{n} / \mathrm{a}$ |
| Failure Notes: |
| $\mathrm{n} / \mathrm{a}$ |


| Point (No.) | $\begin{gathered} \varepsilon_{z} \\ (\%) \end{gathered}$ | $\begin{gathered} \varepsilon_{\mathrm{x}} \\ (\%) \end{gathered}$ | $\begin{gathered} \varepsilon_{\mathrm{y}} \\ (\%) \end{gathered}$ | $\begin{gathered} \varepsilon_{\mathrm{v}} \\ (\%) \end{gathered}$ | $\begin{gathered} \gamma_{\theta z} \\ (\%) \end{gathered}$ | $\begin{gathered} \sigma_{z} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \sigma_{x} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \sigma_{y} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \tau_{\theta z} \\ (\mathrm{kPa}) \end{gathered}$ | b | $\begin{gathered} \alpha \\ \left({ }^{\circ}\right) \end{gathered}$ | $\begin{gathered} \varphi \\ \left({ }^{\circ}\right) \end{gathered}$ |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 1 | 0.000 | 0.000 | 0.000 | 0.000 | 0.000 | 100.786 | 102.088 | 102.088 | 0.000 | 0.000 | 90.000 | -0.368 |
| 2 | 0.015 | -0.003 | -0.003 | 0.009 | 0.000 | 111.526 | 91.243 | 91.243 | 0.000 | 0.000 | 90.000 | 5.741 |
| 3 | 0.037 | -0.011 | -0.011 | 0.015 | 0.000 | 124.753 | 85.680 | 85.680 | 0.000 | 0.000 | 90.000 | 10.701 |
| 4 | 0.057 | -0.020 | -0.020 | 0.017 | 0.000 | 132.871 | 83.461 | 83.461 | 0.000 | 0.000 | 90.000 | 13.203 |
| 5 | 0.086 | -0.031 | -0.031 | 0.024 | 0.000 | 140.311 | 80.400 | 80.400 | 0.000 | 0.000 | 90.000 | 15.750 |
| 6 | 0.127 | -0.048 | -0.048 | 0.030 | 0.000 | 146.492 | 76.842 | 76.842 | 0.000 | 0.000 | 90.000 | 18.172 |
| 7 | 0.176 | -0.070 | -0.070 | 0.036 | 0.000 | 152.151 | 74.962 | 74.962 | 0.000 | 0.000 | 90.000 | 19.869 |
| 8 | 0.225 | -0.092 | -0.092 | 0.041 | 0.000 | 156.538 | 72.932 | 72.932 | 0.000 | 0.000 | 90.000 | 21.367 |
| 9 | 0.316 | -0.135 | -0.135 | 0.045 | 0.000 | 161.792 | 70.156 | 70.156 | 0.000 | 0.000 | 90.000 | 23.270 |
| 10 | 0.406 | -0.179 | -0.179 | 0.048 | 0.000 | 165.769 | 67.775 | 67.775 | 0.000 | 0.000 | 90.000 | 24.809 |
| 11 | 0.492 | -0.222 | -0.222 | 0.047 | 0.000 | 169.213 | 65.788 | 65.788 | 0.000 | 0.000 | 90.000 | 26.111 |
| 12 | 0.616 | -0.287 | -0.287 | 0.043 | 0.000 | 172.199 | 63.950 | 63.950 | 0.000 | 0.000 | 90.000 | 27.284 |
| 13 | 0.725 | -0.346 | -0.346 | 0.032 | 0.000 | 175.519 | 62.259 | 62.259 | 0.000 | 0.000 | 90.000 | 28.446 |
| 14 | 0.822 | -0.403 | -0.403 | 0.017 | 0.000 | 177.371 | 63.303 | 63.303 | 0.000 | 0.000 | 90.000 | 28.291 |
| 15 | 0.854 | -0.418 | -0.418 | 0.017 | 0.000 | 178.198 | 61.612 | 61.612 | 0.000 | 0.000 | 90.000 | 29.088 |
| 16 | 1.042 | -0.528 | -0.528 | -0.013 | 0.000 | 181.338 | 60.069 | 60.069 | 0.000 | 0.000 | 90.000 | 30.155 |
| 17 | 1.188 | -0.613 | -0.613 | -0.039 | 0.000 | 183.228 | 59.969 | 59.969 | 0.000 | 0.000 | 90.000 | 30.453 |
| 18 | 1.309 | -0.684 | -0.684 | -0.060 | 0.000 | 184.916 | 58.376 | 58.376 | 0.000 | 0.000 | 90.000 | 31.340 |
| 19 | 1.411 | -0.748 | -0.748 | -0.086 | 0.000 | 185.178 | 59.521 | 59.521 | 0.000 | 0.000 | 90.000 | 30.898 |
| 20 | 1.503 | -0.786 | -0.786 | -0.069 | 0.000 | 186.253 | 57.429 | 57.429 | 0.000 | 0.000 | 90.000 | 31.915 |
| 21 | 1.648 | -0.880 | -0.880 | -0.112 | 0.000 | 186.167 | 59.571 | 59.571 | 0.000 | 0.000 | 90.000 | 31.009 |
| 22 | 1.772 | -0.952 | -0.952 | -0.133 | 0.000 | 187.884 | 55.907 | 55.907 | 0.000 | 0.000 | 90.000 | 32.776 |
| 23 | 1.854 | -1.006 | -1.006 | -0.159 | 0.000 | 188.539 | 53.738 | 53.738 | 0.000 | 0.000 | 90.000 | 33.807 |
| 24 | 1.937 | -1.056 | -1.056 | -0.176 | 0.000 | 188.577 | 52.340 | 52.340 | 0.000 | 0.000 | 90.000 | 34.437 |
| 25 | 2.014 | -1.105 | -1.105 | -0.197 | 0.000 | 189.125 | 50.378 | 50.378 | 0.000 | 0.000 | 90.000 | 35.402 |
| 26 | 2.124 | -1.169 | -1.169 | -0.214 | 0.000 | 191.642 | 47.815 | 47.815 | 0.000 | 0.000 | 90.000 | 37.130 |
| 27 | 2.209 | -1.220 | -1.220 | -0.232 | 0.000 | 170.998 | 59.607 | 59.607 | 0.000 | 0.000 | 90.000 | 34.525 |

$$
\text { Varying } \sigma_{3} \text { Test on Fine Nevada Sand with a Target Void Ratio }=0.530\left(D_{r}=91.28 \%\right)
$$

| Test No.: | A2 |
| :--- | :---: |
| Test Date: | $12 / 11 / 10$ |
| Sector: | III |
| Initial Void Ratio, e: | 0.536 |
| Final Mean Princpal Stress, $\sigma_{\mathrm{m}}:$ | 136.8 kPa |
| Max Friction Angle, $\phi:$ | 37.1 deg |
| b-value at failure: | 0.00 |
| Stress direction at failure, $\alpha:$ | 90.0 deg |


| Shear Band Notes |
| :--- |
| Point of Observation: |
| $\mathrm{n} / \mathrm{a}$ |
| Inclination (from Vertical): |
| $\mathrm{n} / \mathrm{a}$ |
| Failure Notes: |
| $\mathrm{n} / \mathrm{a}$ |


| Point <br> (No.) | $\begin{gathered} \varepsilon_{z} \\ (\%) \end{gathered}$ | $\begin{gathered} \varepsilon_{\mathrm{x}} \\ (\%) \end{gathered}$ | $\begin{gathered} \varepsilon_{\mathrm{y}} \\ (\%) \end{gathered}$ | $\begin{gathered} \varepsilon_{\mathrm{v}} \\ (\%) \end{gathered}$ | $\begin{aligned} & \gamma_{\theta z} \\ & (\%) \end{aligned}$ | $\begin{gathered} \sigma_{z} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \sigma_{x} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \sigma_{\mathrm{y}} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \tau_{\theta z} \\ (\mathrm{kPa}) \end{gathered}$ | b | $\begin{gathered} \alpha \\ \left({ }^{\circ}\right) \end{gathered}$ | $\begin{gathered} \varphi \\ \left(^{\circ}\right) \end{gathered}$ |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 1 | 0.000 | 0.000 | 0.000 | 0.000 | 0.000 | 100.282 | 100.305 | 100.305 | 0.000 | 0.000 | 90.000 | -0.006 |
| 2 | 0.006 | -0.001 | -0.001 | 0.004 | 0.000 | 115.905 | 96.677 | 96.677 | 0.000 | 0.000 | 90.000 | 5.190 |
| 3 | 0.054 | -0.016 | -0.016 | 0.022 | 0.000 | 149.778 | 90.735 | 90.735 | 0.000 | 0.000 | 90.000 | 14.211 |
| 4 | 0.084 | -0.027 | -0.027 | 0.030 | 0.000 | 161.439 | 88.570 | 88.570 | 0.000 | 0.000 | 90.000 | 16.946 |
| 5 | 0.141 | -0.047 | -0.047 | 0.048 | 0.000 | 177.527 | 85.616 | 85.616 | 0.000 | 0.000 | 90.000 | 20.443 |
| 6 | 0.231 | -0.084 | -0.084 | 0.063 | 0.000 | 193.247 | 82.905 | 82.905 | 0.000 | 0.000 | 90.000 | 23.551 |
| 7 | 0.307 | -0.119 | -0.119 | 0.069 | 0.000 | 203.149 | 80.931 | 80.931 | 0.000 | 0.000 | 90.000 | 25.482 |
| 8 | 0.436 | -0.179 | -0.179 | 0.078 | 0.000 | 214.321 | 79.005 | 79.005 | 0.000 | 0.000 | 90.000 | 27.472 |
| 9 | 0.555 | -0.239 | -0.239 | 0.078 | 0.000 | 222.741 | 77.523 | 77.523 | 0.000 | 0.000 | 90.000 | 28.923 |
| 10 | 0.726 | -0.326 | -0.326 | 0.074 | 0.000 | 232.699 | 75.644 | 75.644 | 0.000 | 0.000 | 90.000 | 30.621 |
| 11 | 0.838 | -0.387 | -0.387 | 0.063 | 0.000 | 237.151 | 74.902 | 74.902 | 0.000 | 0.000 | 90.000 | 31.328 |
| 12 | 0.995 | -0.475 | -0.475 | 0.045 | 0.000 | 242.924 | 73.912 | 73.912 | 0.000 | 0.000 | 90.000 | 32.238 |
| 13 | 1.243 | -0.624 | -0.624 | -0.004 | 0.000 | 250.891 | 72.031 | 72.031 | 0.000 | 0.000 | 90.000 | 33.634 |
| 14 | 1.330 | -0.676 | -0.676 | -0.022 | 0.000 | 251.283 | 72.427 | 72.427 | 0.000 | 0.000 | 90.000 | 33.540 |
| 15 | 1.521 | -0.793 | -0.793 | -0.065 | 0.000 | 256.001 | 71.486 | 71.486 | 0.000 | 0.000 | 90.000 | 34.293 |
| 16 | 1.685 | -0.894 | -0.894 | -0.104 | 0.000 | 258.155 | 70.891 | 70.891 | 0.000 | 0.000 | 90.000 | 34.688 |
| 17 | 1.857 | -1.010 | -1.010 | -0.164 | 0.000 | 260.001 | 70.792 | 70.792 | 0.000 | 0.000 | 90.000 | 34.889 |
| 18 | 2.014 | -1.113 | -1.113 | -0.212 | 0.000 | 262.725 | 70.247 | 70.247 | 0.000 | 0.000 | 90.000 | 35.314 |
| 19 | 2.166 | -1.210 | -1.210 | -0.255 | 0.000 | 265.003 | 69.801 | 69.801 | 0.000 | 0.000 | 90.000 | 35.664 |
| 20 | 2.329 | -1.318 | -1.318 | -0.307 | 0.000 | 267.861 | 69.255 | 69.255 | 0.000 | 0.000 | 90.000 | 36.095 |
| 21 | 2.484 | -1.421 | -1.421 | -0.358 | 0.000 | 268.934 | 69.106 | 69.106 | 0.000 | 0.000 | 90.000 | 36.238 |
| 22 | 2.689 | -1.560 | -1.560 | -0.432 | 0.000 | 270.297 | 68.858 | 68.858 | 0.000 | 0.000 | 90.000 | 36.437 |
| 23 | 2.857 | -1.673 | -1.673 | -0.488 | 0.000 | 272.072 | 68.561 | 68.561 | 0.000 | 0.000 | 90.000 | 36.688 |
| 24 | 3.112 | -1.843 | -1.843 | -0.574 | 0.000 | 274.215 | 68.164 | 68.164 | 0.000 | 0.000 | 90.000 | 37.000 |
| 25 | 3.228 | -1.921 | -1.921 | -0.613 | 0.000 | 274.230 | 67.966 | 67.966 | 0.000 | 0.000 | 90.000 | 37.068 |
| 26 | 3.354 | -2.014 | -2.014 | -0.674 | 0.000 | 273.267 | 68.114 | 68.114 | 0.000 | 0.000 | 90.000 | 36.938 |
| 27 | 3.502 | -2.116 | -2.116 | -0.730 | 0.000 | 272.938 | 68.214 | 68.214 | 0.000 | 0.000 | 90.000 | 36.877 |
| 28 | 3.660 | -2.221 | -2.221 | -0.782 | 0.000 | 273.602 | 68.065 | 68.065 | 0.000 | 0.000 | 90.000 | 36.983 |
| 29 | 3.817 | -2.325 | -2.325 | -0.834 | 0.000 | 274.571 | 67.866 | 67.866 | 0.000 | 0.000 | 90.000 | 37.130 |
| 30 | 4.001 | -2.458 | -2.458 | -0.916 | 0.000 | 273.415 | 68.164 | 68.164 | 0.000 | 0.000 | 90.000 | 36.934 |
| 31 | 4.146 | -2.555 | -2.555 | -0.963 | 0.000 | 274.232 | 67.966 | 67.966 | 0.000 | 0.000 | 90.000 | 37.069 |
| 32 | 4.320 | -2.671 | -2.671 | -1.021 | 0.000 | 274.407 | 67.916 | 67.916 | 0.000 | 0.000 | 90.000 | 37.100 |

## Varying $\sigma_{3}$ Test on Fine Nevada Sand with a Target Void Ratio $=0.530\left(D_{r}=91.28 \%\right)$

| Test No.: | A3 |
| :--- | :--- |
| Test Date: | $12 / 14 / 11$ |
| Sector: | III |
| Initial Void Ratio, e: | 0.533 |
| Final Mean Princpal Stress, $\sigma_{\mathrm{m}}:$ | 61.4 kPa |
| Max Friction Angle, $\phi:$ | 29.9 deg |
| b-value at failure: | 0.00 |
| Stress direction at failure, $\alpha:$ | 90.0 deg |$\quad$| Shear Band Notes |
| :--- |
| Point of Observation: |
| $\mathrm{n} / \mathrm{a}$ |
| Inclination (from Vertical): |
| $\mathrm{n} / \mathrm{a}$ |
|  |
| Failure Notes: |
| $\mathrm{n} / \mathrm{a}$ |


| Point (No.) | $\begin{gathered} \varepsilon_{\mathrm{z}} \\ (\%) \\ \hline \end{gathered}$ | $\begin{gathered} \varepsilon_{\mathrm{x}} \\ (\%) \\ \hline \end{gathered}$ | $\begin{array}{r} \varepsilon_{y} \\ (\%) \\ \hline \end{array}$ | $\begin{gathered} \varepsilon_{\mathrm{v}} \\ (\%) \\ \hline \end{gathered}$ | $\begin{gathered} \gamma_{\theta z} \\ (\%) \\ \hline \end{gathered}$ | $\begin{gathered} \sigma_{z} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \sigma_{x} \\ (\mathrm{kPa}) \\ \hline \end{gathered}$ | $\begin{gathered} \sigma_{y} \\ (\mathrm{kPa}) \\ \hline \end{gathered}$ | $\begin{gathered} \tau_{\theta z} \\ (\mathrm{kPa}) \end{gathered}$ | b | $\begin{gathered} \alpha \\ \left(^{\circ}\right) \\ \hline \end{gathered}$ | $\begin{gathered} \varphi \\ \left({ }^{\circ}\right) \\ \hline \end{gathered}$ |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 1 | 0.000 | 0.000 | 0.000 | 0.000 | 0.000 | 104.295 | 104.318 | 104.318 | 0.000 | 0.000 | 90.000 | -0.006 |
| 2 | 0.012 | -0.002 | -0.002 | 0.009 | 0.000 | 105.146 | 93.929 | 93.929 | 0.000 | 0.000 | 90.000 | 3.230 |
| 3 | 0.012 | -0.001 | -0.001 | 0.009 | 0.000 | 104.831 | 85.419 | 85.419 | 0.000 | 0.000 | 90.000 | 5.856 |
| 4 | 0.015 | -0.008 | -0.008 | -0.001 | 0.000 | 104.964 | 79.647 | 79.647 | 0.000 | 0.000 | 90.000 | 7.882 |
| 5 | 0.016 | -0.010 | -0.010 | -0.004 | 0.000 | 104.977 | 77.375 | 77.375 | 0.000 | 0.000 | 90.000 | 8.706 |
| 6 | 0.023 | -0.015 | -0.015 | -0.008 | 0.000 | 105.058 | 71.932 | 71.932 | 0.000 | 0.000 | 90.000 | 10.787 |
| 7 | 0.027 | -0.018 | -0.018 | -0.009 | 0.000 | 104.876 | 68.511 | 68.511 | 0.000 | 0.000 | 90.000 | 12.107 |
| 8 | 0.037 | -0.024 | -0.024 | -0.011 | 0.000 | 105.345 | 64.888 | 64.888 | 0.000 | 0.000 | 90.000 | 13.748 |
| 9 | 0.048 | -0.028 | -0.028 | -0.009 | 0.000 | 105.114 | 61.706 | 61.706 | 0.000 | 0.000 | 90.000 | 15.082 |
| 10 | 0.060 | -0.035 | -0.035 | -0.009 | 0.000 | 105.147 | 59.268 | 59.268 | 0.000 | 0.000 | 90.000 | 16.203 |
| 11 | 0.078 | -0.043 | -0.043 | -0.009 | 0.000 | 105.266 | 56.728 | 56.728 | 0.000 | 0.000 | 90.000 | 17.435 |
| 12 | 0.094 | -0.047 | -0.047 | -0.001 | 0.000 | 105.155 | 54.335 | 54.335 | 0.000 | 0.000 | 90.000 | 18.581 |
| 13 | 0.113 | -0.056 | -0.056 | 0.001 | 0.000 | 105.245 | 52.338 | 52.338 | 0.000 | 0.000 | 90.000 | 19.617 |
| 14 | 0.141 | -0.071 | -0.071 | 0.000 | 0.000 | 105.268 | 50.091 | 50.091 | 0.000 | 0.000 | 90.000 | 20.803 |
| 15 | 0.173 | -0.087 | -0.087 | 0.000 | 0.000 | 105.364 | 48.491 | 48.491 | 0.000 | 0.000 | 90.000 | 21.694 |
| 16 | 0.213 | -0.105 | -0.105 | 0.002 | 0.000 | 105.476 | 46.340 | 46.340 | 0.000 | 0.000 | 90.000 | 22.925 |
| 17 | 0.238 | -0.116 | -0.116 | 0.007 | 0.000 | 105.300 | 45.740 | 45.740 | 0.000 | 0.000 | 90.000 | 23.225 |
| 18 | 0.287 | -0.140 | -0.140 | 0.007 | 0.000 | 105.273 | 44.087 | 44.087 | 0.000 | 0.000 | 90.000 | 24.183 |
| 19 | 0.331 | -0.162 | -0.162 | 0.007 | 0.000 | 105.406 | 43.486 | 43.486 | 0.000 | 0.000 | 90.000 | 24.574 |
| 20 | 0.375 | -0.182 | -0.182 | 0.011 | 0.000 | 105.269 | 42.233 | 42.233 | 0.000 | 0.000 | 90.000 | 25.300 |
| 21 | 0.425 | -0.207 | -0.207 | 0.011 | 0.000 | 105.284 | 41.331 | 41.331 | 0.000 | 0.000 | 90.000 | 25.861 |
| 22 | 0.477 | -0.234 | -0.234 | 0.009 | 0.000 | 105.346 | 40.478 | 40.478 | 0.000 | 0.000 | 90.000 | 26.413 |
| 23 | 0.527 | -0.260 | -0.260 | 0.007 | 0.000 | 105.279 | 39.876 | 39.876 | 0.000 | 0.000 | 90.000 | 26.780 |
| 24 | 0.575 | -0.285 | -0.285 | 0.004 | 0.000 | 104.773 | 39.023 | 39.023 | 0.000 | 0.000 | 90.000 | 27.209 |
| 25 | 0.626 | -0.312 | -0.312 | 0.002 | 0.000 | 105.311 | 38.270 | 38.270 | 0.000 | 0.000 | 90.000 | 27.834 |
| 26 | 0.676 | -0.337 | -0.337 | 0.001 | 0.000 | 105.277 | 38.270 | 38.270 | 0.000 | 0.000 | 90.000 | 27.803 |
| 27 | 0.706 | -0.359 | -0.359 | -0.011 | 0.000 | 105.205 | 39.174 | 39.174 | 0.000 | 0.000 | 90.000 | 27.193 |
| 28 | 0.779 | -0.392 | -0.392 | -0.004 | 0.000 | 108.923 | 36.312 | 36.312 | 0.000 | 0.000 | 90.000 | 29.973 |

$$
\text { Varying } \sigma_{3} \text { Test on Fine Nevada Sand with a Target Void Ratio }=0.530\left(D_{r}=91.28 \%\right)
$$

| Test No.: | A4 |
| :--- | :---: |
| Test Date: | $12 / 15 / 10$ |
| Sector: | III |
| Initial Void Ratio, e: | 0.534 |
| Final Mean Princpal Stress, $\sigma_{\mathrm{m}}:$ | 49.1 kPa |
| Max Friction Angle, $\phi:$ | 27.8 deg |
| b-value at failure: | 0.00 |
| Stress direction at failure, $\alpha:$ | 90.0 deg |


| Shear Band Notes |
| :--- |
| Point of Observation: |
| $\mathrm{n} / \mathrm{a}$ |
| Inclination (from Vertical): |
| $\mathrm{n} / \mathrm{a}$ |
| Failure Notes: |
| $\mathrm{n} / \mathrm{a}$ |


| Point (No.) | $\begin{gathered} \varepsilon_{z} \\ (\%) \end{gathered}$ | $\begin{gathered} \varepsilon_{\mathrm{x}} \\ (\%) \end{gathered}$ | $\begin{gathered} \varepsilon_{\mathrm{y}} \\ (\%) \end{gathered}$ | $\begin{gathered} \varepsilon_{\mathrm{v}} \\ (\%) \end{gathered}$ | $\begin{gathered} \gamma_{\theta z} \\ (\%) \end{gathered}$ | $\begin{gathered} \sigma_{z} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \sigma_{x} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \sigma_{y} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \tau_{\theta z} \\ (\mathrm{kPa}) \end{gathered}$ | b | $\begin{gathered} \alpha \\ \left({ }^{\circ}\right) \end{gathered}$ | $\begin{gathered} \varphi \\ \left({ }^{\circ}\right) \end{gathered}$ |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 1 | 0.000 | 0.000 | 0.000 | 0.000 | 0.000 | 103.045 | 103.067 | 103.067 | 0.000 | 0.000 | 90.000 | -0.006 |
| 2 | 0.002 | -0.003 | -0.003 | -0.004 | 0.000 | 100.596 | 98.364 | 98.364 | 0.000 | 0.000 | 90.000 | 0.643 |
| 3 | 0.008 | -0.005 | -0.005 | -0.002 | 0.000 | 97.655 | 85.088 | 85.088 | 0.000 | 0.000 | 90.000 | 3.943 |
| 4 | 0.016 | -0.012 | -0.012 | -0.009 | 0.000 | 92.328 | 65.291 | 65.291 | 0.000 | 0.000 | 90.000 | 9.877 |
| 5 | 0.023 | -0.027 | -0.027 | -0.030 | 0.000 | 88.893 | 51.341 | 51.341 | 0.000 | 0.000 | 90.000 | 15.533 |
| 6 | 0.032 | -0.044 | -0.044 | -0.057 | 0.000 | 88.623 | 48.641 | 48.641 | 0.000 | 0.000 | 90.000 | 16.934 |
| 7 | 0.045 | -0.054 | -0.054 | -0.062 | 0.000 | 87.343 | 45.913 | 45.913 | 0.000 | 0.000 | 90.000 | 18.114 |
| 8 | 0.058 | -0.063 | -0.063 | -0.067 | 0.000 | 87.778 | 45.462 | 45.462 | 0.000 | 0.000 | 90.000 | 18.517 |
| 9 | 0.066 | -0.068 | -0.068 | -0.070 | 0.000 | 87.241 | 43.334 | 43.334 | 0.000 | 0.000 | 90.000 | 19.649 |
| 10 | 0.078 | -0.074 | -0.074 | -0.070 | 0.000 | 86.921 | 42.080 | 42.080 | 0.000 | 0.000 | 90.000 | 20.340 |
| 11 | 0.092 | -0.082 | -0.082 | -0.072 | 0.000 | 86.762 | 41.177 | 41.177 | 0.000 | 0.000 | 90.000 | 20.873 |
| 12 | 0.103 | -0.089 | -0.089 | -0.074 | 0.000 | 86.529 | 40.575 | 40.575 | 0.000 | 0.000 | 90.000 | 21.195 |
| 13 | 0.120 | -0.097 | -0.097 | -0.074 | 0.000 | 86.382 | 39.873 | 39.873 | 0.000 | 0.000 | 90.000 | 21.616 |
| 14 | 0.134 | -0.105 | -0.105 | -0.076 | 0.000 | 86.362 | 39.672 | 39.672 | 0.000 | 0.000 | 90.000 | 21.744 |
| 15 | 0.153 | -0.114 | -0.114 | -0.076 | 0.000 | 86.200 | 38.768 | 38.768 | 0.000 | 0.000 | 90.000 | 22.306 |
| 16 | 0.192 | -0.135 | -0.135 | -0.078 | 0.000 | 85.798 | 37.261 | 37.261 | 0.000 | 0.000 | 90.000 | 23.230 |
| 17 | 0.206 | -0.143 | -0.143 | -0.079 | 0.000 | 85.778 | 37.060 | 37.060 | 0.000 | 0.000 | 90.000 | 23.366 |
| 18 | 0.217 | -0.148 | -0.148 | -0.079 | 0.000 | 85.568 | 36.105 | 36.105 | 0.000 | 0.000 | 90.000 | 23.986 |
| 19 | 0.237 | -0.160 | -0.160 | -0.082 | 0.000 | 85.526 | 35.980 | 35.980 | 0.000 | 0.000 | 90.000 | 24.065 |
| 20 | 0.259 | -0.173 | -0.173 | -0.086 | 0.000 | 85.441 | 35.628 | 35.628 | 0.000 | 0.000 | 90.000 | 24.296 |
| 21 | 0.267 | -0.177 | -0.177 | -0.087 | 0.000 | 85.337 | 35.389 | 35.389 | 0.000 | 0.000 | 90.000 | 24.440 |
| 22 | 0.279 | -0.183 | -0.183 | -0.087 | 0.000 | 85.239 | 35.062 | 35.062 | 0.000 | 0.000 | 90.000 | 24.652 |
| 23 | 0.289 | -0.190 | -0.190 | -0.090 | 0.000 | 85.076 | 34.257 | 34.257 | 0.000 | 0.000 | 90.000 | 25.205 |
| 24 | 0.361 | -0.230 | -0.230 | -0.098 | 0.000 | 84.878 | 33.339 | 33.339 | 0.000 | 0.000 | 90.000 | 25.847 |
| 25 | 0.397 | -0.253 | -0.253 | -0.109 | 0.000 | 84.788 | 32.999 | 32.999 | 0.000 | 0.000 | 90.000 | 26.084 |
| 26 | 0.449 | -0.284 | -0.284 | -0.119 | 0.000 | 84.725 | 32.634 | 32.634 | 0.000 | 0.000 | 90.000 | 26.350 |
| 27 | 0.463 | -0.295 | -0.295 | -0.127 | 0.000 | 84.569 | 32.219 | 32.219 | 0.000 | 0.000 | 90.000 | 26.632 |
| 28 | 0.512 | -0.322 | -0.322 | -0.132 | 0.000 | 84.308 | 31.563 | 31.563 | 0.000 | 0.000 | 90.000 | 27.078 |
| 29 | 0.582 | -0.360 | -0.360 | -0.139 | 0.000 | 84.384 | 31.625 | 31.625 | 0.000 | 0.000 | 90.000 | 27.051 |
| 30 | 0.607 | -0.379 | -0.379 | -0.150 | 0.000 | 84.417 | 31.497 | 31.497 | 0.000 | 0.000 | 90.000 | 27.165 |
| 31 | 0.633 | -0.396 | -0.396 | -0.160 | 0.000 | 84.521 | 32.133 | 32.133 | 0.000 | 0.000 | 90.000 | 26.686 |
| 32 | 0.760 | -0.462 | -0.462 | -0.164 | 0.000 | 84.223 | 30.549 | 30.549 | 0.000 | 0.000 | 90.000 | 27.882 |
| 33 | 0.789 | -0.486 | -0.486 | -0.184 | 0.000 | 84.153 | 30.601 | 30.601 | 0.000 | 0.000 | 90.000 | 27.818 |

$$
\text { Varying } \sigma_{3} \text { Test on Fine Nevada Sand with a Target Void Ratio }=0.530\left(D_{r}=91.28 \%\right)
$$

| Test No.: | A5 |
| :--- | :---: |
| Test Date: | $12 / 17 / 10$ |
| Sector: | III |
| Initial Void Ratio, e: | 0.534 |
| Final Mean Princpal Stress, $\sigma_{\mathrm{m}}:$ | 89.02 kPa |
| Max Friction Angle, $\phi:$ | 22.6 deg |
| b-value at failure: | 0.00 |
| Stress direction at failure, $\alpha:$ | 90.0 deg |


| Shear Band Notes |
| :--- |
| Point of Observation: |
| $\mathrm{n} / \mathrm{a}$ |
| Inclination (from Vertical): |
| $\mathrm{n} / \mathrm{a}$ |
| Failure Notes: |
| $\mathrm{n} / \mathrm{a}$ |


| Point (No.) | $\begin{gathered} \varepsilon_{z} \\ (\%) \end{gathered}$ | $\begin{gathered} \varepsilon_{\mathrm{x}} \\ (\%) \end{gathered}$ | $\begin{gathered} \varepsilon_{y} \\ (\%) \end{gathered}$ | $\begin{gathered} \varepsilon_{\mathrm{v}} \\ (\%) \end{gathered}$ | $\begin{gathered} \gamma_{\theta z} \\ (\%) \end{gathered}$ | $\begin{gathered} \sigma_{z} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \sigma_{\mathrm{x}} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \sigma_{\mathrm{y}} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \tau_{\theta z} \\ (\mathrm{kPa}) \end{gathered}$ | b | $\begin{gathered} \alpha \\ \left(^{( }\right) \end{gathered}$ | $\begin{gathered} \varphi \\ \left(^{\circ}\right) \end{gathered}$ |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 1 | 0.000 | 0.000 | 0.000 | 0.000 | 0.000 | 101.478 | 101.501 | 101.501 | 0.000 | 0.000 | 90.000 | -0.006 |
| 2 | 0.001 | 0.002 | 0.002 | 0.004 | 0.000 | 102.800 | 99.737 | 99.737 | 0.000 | 0.000 | 90.000 | 0.867 |
| 3 | 0.002 | 0.001 | 0.001 | 0.004 | 0.000 | 105.946 | 95.763 | 95.763 | 0.000 | 0.000 | 90.000 | 2.894 |
| 4 | 0.004 | 0.000 | 0.000 | 0.004 | 0.000 | 108.773 | 92.472 | 92.472 | 0.000 | 0.000 | 90.000 | 4.646 |
| 5 | 0.010 | -0.001 | -0.001 | 0.007 | 0.000 | 113.038 | 86.961 | 86.961 | 0.000 | 0.000 | 90.000 | 7.492 |
| 6 | 0.011 | -0.003 | -0.003 | 0.004 | 0.000 | 113.193 | 87.059 | 87.059 | 0.000 | 0.000 | 90.000 | 7.499 |
| 7 | 0.016 | -0.004 | -0.004 | 0.009 | 0.000 | 117.260 | 81.832 | 81.832 | 0.000 | 0.000 | 90.000 | 10.250 |
| 8 | 0.017 | -0.007 | -0.007 | 0.004 | 0.000 | 117.322 | 81.832 | 81.832 | 0.000 | 0.000 | 90.000 | 10.265 |
| 9 | 0.022 | -0.007 | -0.007 | 0.009 | 0.000 | 120.534 | 77.188 | 77.188 | 0.000 | 0.000 | 90.000 | 12.664 |
| 10 | 0.030 | -0.010 | -0.010 | 0.009 | 0.000 | 121.934 | 75.457 | 75.457 | 0.000 | 0.000 | 90.000 | 13.619 |
| 11 | 0.035 | -0.013 | -0.013 | 0.009 | 0.000 | 123.413 | 73.724 | 73.724 | 0.000 | 0.000 | 90.000 | 14.599 |
| 12 | 0.039 | -0.015 | -0.015 | 0.009 | 0.000 | 123.757 | 72.783 | 72.783 | 0.000 | 0.000 | 90.000 | 15.032 |
| 13 | 0.045 | -0.018 | -0.018 | 0.009 | 0.000 | 124.892 | 71.990 | 71.990 | 0.000 | 0.000 | 90.000 | 15.587 |
| 14 | 0.048 | -0.020 | -0.020 | 0.009 | 0.000 | 125.042 | 71.842 | 71.842 | 0.000 | 0.000 | 90.000 | 15.677 |
| 15 | 0.053 | -0.020 | -0.020 | 0.013 | 0.000 | 126.783 | 69.710 | 69.710 | 0.000 | 0.000 | 90.000 | 16.885 |
| 16 | 0.059 | -0.023 | -0.023 | 0.013 | 0.000 | 126.726 | 69.759 | 69.759 | 0.000 | 0.000 | 90.000 | 16.854 |
| 17 | 0.066 | -0.025 | -0.025 | 0.015 | 0.000 | 127.906 | 68.333 | 68.333 | 0.000 | 0.000 | 90.000 | 17.672 |
| 18 | 0.072 | -0.028 | -0.028 | 0.017 | 0.000 | 128.792 | 67.377 | 67.377 | 0.000 | 0.000 | 90.000 | 18.244 |
| 19 | 0.079 | -0.031 | -0.031 | 0.018 | 0.000 | 129.533 | 66.496 | 66.496 | 0.000 | 0.000 | 90.000 | 18.758 |
| 20 | 0.088 | -0.035 | -0.035 | 0.019 | 0.000 | 130.254 | 65.552 | 65.552 | 0.000 | 0.000 | 90.000 | 19.296 |
| 21 | 0.093 | -0.036 | -0.036 | 0.020 | 0.000 | 130.582 | 65.166 | 65.166 | 0.000 | 0.000 | 90.000 | 19.523 |
| 22 | 0.099 | -0.039 | -0.039 | 0.021 | 0.000 | 131.469 | 64.058 | 64.058 | 0.000 | 0.000 | 90.000 | 20.167 |
| 23 | 0.110 | -0.043 | -0.043 | 0.023 | 0.000 | 131.912 | 63.557 | 63.557 | 0.000 | 0.000 | 90.000 | 20.469 |
| 24 | 0.134 | -0.054 | -0.054 | 0.027 | 0.000 | 133.243 | 64.276 | 64.276 | 0.000 | 0.000 | 90.000 | 20.436 |
| 25 | 0.147 | -0.052 | -0.052 | 0.042 | 0.000 | 140.617 | 62.702 | 62.702 | 0.000 | 0.000 | 90.000 | 22.533 |
| 26 | 0.160 | -0.056 | -0.056 | 0.048 | 0.000 | 141.417 | 62.819 | 62.819 | 0.000 | 0.000 | 90.000 | 22.634 |
| 27 | 0.271 | -0.173 | -0.173 | -0.075 | 0.000 | 166.093 | 73.957 | 73.957 | 0.000 | 0.000 | 90.000 | 22.570 |

## Varying $\sigma_{3}$ Test on Fine Nevada Sand with a Target Void Ratio $=0.530\left(D_{r}=91.28 \%\right)$

| Test No.: | A6 |
| :--- | :--- |
| Test Date: | $12 / 18 / 10$ |
| Sector: | III |
| Initial Void Ratio, e: | 0.538 |
| Final Mean Princpal Stress, $\sigma_{\mathrm{m}}:$ | 155.9 kPa |
| Max Friction Angle, $\phi:$ | 38.2 deg |
| b-value at failure: | 0.00 |
| Stress direction at failure, $\alpha:$ | 90.0 deg |$\quad$| Shear Band Notes |
| :--- |
| Point of Observation: |
| $\mathrm{n} / \mathrm{a}$ |
| Inclination (from Vertical): |
| $\mathrm{n} / \mathrm{a}$ |
|  |
|  |


| Point (No.) | $\begin{gathered} \varepsilon_{\mathrm{z}} \\ (\%) \\ \hline \end{gathered}$ | $\begin{gathered} \varepsilon_{\mathrm{x}} \\ (\%) \\ \hline \end{gathered}$ | $\begin{array}{r} \varepsilon_{\mathrm{y}} \\ (\%) \\ \hline \end{array}$ | $\begin{gathered} \varepsilon_{\mathrm{v}} \\ (\%) \\ \hline \end{gathered}$ | $\begin{array}{r} \gamma_{\theta z} \\ (\%) \\ \hline \end{array}$ | $\begin{gathered} \sigma_{z} \\ (\mathrm{kPa}) \\ \hline \end{gathered}$ | $\begin{gathered} \sigma_{\mathrm{x}} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \sigma_{y} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \tau_{\text {өz }} \\ (\mathrm{kPa}) \\ \hline \end{gathered}$ | b | $\begin{gathered} \alpha \\ \left({ }^{\circ}\right) \end{gathered}$ | $\begin{gathered} \varphi \\ \left({ }^{\circ}\right) \end{gathered}$ |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 1 | 0.000 | 0.000 | 0.000 | 0.000 | 0.000 | 101.086 | 101.109 | 101.109 | 0.000 | 0.000 | 90.000 | -0.006 |
| 2 | 0.012 | -0.004 | -0.004 | 0.004 | 0.000 | 111.297 | 95.518 | 95.518 | 0.000 | 0.000 | 90.000 | 4.376 |
| 3 | 0.023 | -0.007 | -0.007 | 0.009 | 0.000 | 120.430 | 91.292 | 91.292 | 0.000 | 0.000 | 90.000 | 7.910 |
| 4 | 0.036 | -0.012 | -0.012 | 0.013 | 0.000 | 126.786 | 88.241 | 88.241 | 0.000 | 0.000 | 90.000 | 10.326 |
| 5 | 0.058 | -0.018 | -0.018 | 0.022 | 0.000 | 135.686 | 84.348 | 84.348 | 0.000 | 0.000 | 90.000 | 13.492 |
| 6 | 0.078 | -0.024 | -0.024 | 0.030 | 0.000 | 141.911 | 81.734 | 81.734 | 0.000 | 0.000 | 90.000 | 15.609 |
| 7 | 0.107 | -0.034 | -0.034 | 0.039 | 0.000 | 147.800 | 79.165 | 79.165 | 0.000 | 0.000 | 90.000 | 17.602 |
| 8 | 0.143 | -0.049 | -0.049 | 0.046 | 0.000 | 153.202 | 76.496 | 76.496 | 0.000 | 0.000 | 90.000 | 19.508 |
| 9 | 0.181 | -0.063 | -0.063 | 0.054 | 0.000 | 157.596 | 74.516 | 74.516 | 0.000 | 0.000 | 90.000 | 20.973 |
| 10 | 0.240 | -0.087 | -0.087 | 0.065 | 0.000 | 162.989 | 72.238 | 72.238 | 0.000 | 0.000 | 90.000 | 22.693 |
| 11 | 0.315 | -0.119 | -0.119 | 0.076 | 0.000 | 167.733 | 69.710 | 69.710 | 0.000 | 0.000 | 90.000 | 24.383 |
| 12 | 0.380 | -0.149 | -0.149 | 0.083 | 0.000 | 171.375 | 67.775 | 67.775 | 0.000 | 0.000 | 90.000 | 25.671 |
| 13 | 0.442 | -0.178 | -0.178 | 0.086 | 0.000 | 173.751 | 66.831 | 66.831 | 0.000 | 0.000 | 90.000 | 26.386 |
| 14 | 0.529 | -0.221 | -0.221 | 0.087 | 0.000 | 176.567 | 65.987 | 65.987 | 0.000 | 0.000 | 90.000 | 27.123 |
| 15 | 0.600 | -0.256 | -0.256 | 0.087 | 0.000 | 178.543 | 65.043 | 65.043 | 0.000 | 0.000 | 90.000 | 27.772 |
| 16 | 0.710 | -0.313 | -0.313 | 0.085 | 0.000 | 180.868 | 63.751 | 63.751 | 0.000 | 0.000 | 90.000 | 28.605 |
| 17 | 0.789 | -0.354 | -0.354 | 0.080 | 0.000 | 182.556 | 63.104 | 63.104 | 0.000 | 0.000 | 90.000 | 29.094 |
| 18 | 0.889 | -0.410 | -0.410 | 0.070 | 0.000 | 183.397 | 62.209 | 62.209 | 0.000 | 0.000 | 90.000 | 29.566 |
| 19 | 0.988 | -0.466 | -0.466 | 0.057 | 0.000 | 184.517 | 61.413 | 61.413 | 0.000 | 0.000 | 90.000 | 30.037 |
| 20 | 1.093 | -0.525 | -0.525 | 0.043 | 0.000 | 185.887 | 61.065 | 61.065 | 0.000 | 0.000 | 90.000 | 30.361 |
| 21 | 1.200 | -0.585 | -0.585 | 0.030 | 0.000 | 187.051 | 60.517 | 60.517 | 0.000 | 0.000 | 90.000 | 30.738 |
| 22 | 1.269 | -0.623 | -0.623 | 0.022 | 0.000 | 187.867 | 59.571 | 59.571 | 0.000 | 0.000 | 90.000 | 31.232 |
| 23 | 1.369 | -0.682 | -0.682 | 0.004 | 0.000 | 188.889 | 59.073 | 59.073 | 0.000 | 0.000 | 90.000 | 31.569 |
| 24 | 1.478 | -0.747 | -0.747 | -0.015 | 0.000 | 189.645 | 58.973 | 58.973 | 0.000 | 0.000 | 90.000 | 31.708 |
| 25 | 1.580 | -0.811 | -0.811 | -0.041 | 0.000 | 189.735 | 59.160 | 59.160 | 0.000 | 0.000 | 90.000 | 31.643 |
| 26 | 1.693 | -0.877 | -0.877 | -0.061 | 0.000 | 190.256 | 59.795 | 59.795 | 0.000 | 0.000 | 90.000 | 31.449 |
| 27 | 1.758 | -0.915 | -0.915 | -0.072 | 0.000 | 191.994 | 57.977 | 57.977 | 0.000 | 0.000 | 90.000 | 32.421 |
| 28 | 1.828 | -0.957 | -0.957 | -0.087 | 0.000 | 192.554 | 57.728 | 57.728 | 0.000 | 0.000 | 90.000 | 32.595 |

$$
\text { Varying } \sigma_{3} \text { Test on Fine Nevada Sand with a Target Void Ratio }=0.530\left(D_{r}=91.28 \%\right)
$$

| Test No.: | A7, Triaxial 10 |
| :--- | :---: |
| Test Date: | $12 / 21 / 10$ |
| Sector: | III |
| Initial Void Ratio, e: | 0.532 |
| Final Mean Princpal Stress, $\sigma_{\mathrm{m}}:$ | 200.0 kPa |
| Max Friction Angle, $\phi:$ | 37.2 deg |
| b-value at failure: | 0.00 |
| Stress direction at failure, $\alpha:$ | 90.0 deg |


| Shear Band Notes |
| :--- |
| Point of Observation: |
|  |
| Inclination (from Vertical): |
| $58^{\circ}$ |
| Failure Notes: |


| Point (No.) | $\begin{gathered} \varepsilon_{\mathrm{z}} \\ (\%) \end{gathered}$ | $\begin{gathered} \varepsilon_{\mathrm{x}} \\ (\%) \end{gathered}$ | $\begin{array}{r} \varepsilon_{y} \\ (\%) \\ \hline \end{array}$ | $\begin{gathered} \varepsilon_{\mathrm{v}} \\ (\%) \\ \hline \end{gathered}$ | $\begin{aligned} & \gamma_{\theta z} \\ & (\%) \end{aligned}$ | $\begin{gathered} \sigma_{z} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \sigma_{x} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \sigma_{\mathrm{y}} \\ (\mathrm{kPa}) \end{gathered}$ | $\begin{gathered} \boldsymbol{\tau}_{\theta z} \\ (\mathrm{kPa}) \end{gathered}$ | b | $\alpha$ <br> ${ }^{\circ}$ ) | $\begin{gathered} \varphi \\ \left({ }^{( }\right) \\ \hline \end{gathered}$ |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 1 | 0.000 | 0.000 | 0.000 | 0.000 | 0.000 | 98.977 | 99.000 | 99.000 | 0.000 | 0.000 | 90.000 | -0.007 |
| 2 | 0.023 | -0.006 | -0.006 | 0.011 | 0.000 | 121.459 | 99.000 | 99.000 | 0.000 | 0.000 | 90.000 | 5.847 |
| 3 | 0.067 | -0.018 | -0.018 | 0.031 | 0.000 | 146.596 | 99.000 | 99.000 | 0.000 | 0.000 | 90.000 | 11.175 |
| 4 | 0.122 | -0.034 | -0.034 | 0.053 | 0.000 | 172.064 | 99.000 | 99.000 | 0.000 | 0.000 | 90.000 | 15.637 |
| 5 | 0.171 | -0.051 | -0.051 | 0.070 | 0.000 | 189.489 | 99.000 | 99.000 | 0.000 | 0.000 | 90.000 | 18.280 |
| 6 | 0.256 | -0.081 | -0.081 | 0.095 | 0.000 | 211.058 | 99.000 | 99.000 | 0.000 | 0.000 | 90.000 | 21.187 |
| 7 | 0.516 | -0.188 | -0.188 | 0.141 | 0.000 | 253.622 | 99.000 | 99.000 | 0.000 | 0.000 | 90.000 | 26.008 |
| 8 | 0.793 | -0.315 | -0.315 | 0.163 | 0.000 | 284.120 | 99.000 | 99.000 | 0.000 | 0.000 | 90.000 | 28.894 |
| 9 | 1.023 | -0.432 | -0.432 | 0.158 | 0.000 | 302.729 | 99.000 | 99.000 | 0.000 | 0.000 | 90.000 | 30.473 |
| 10 | 1.299 | -0.586 | -0.586 | 0.128 | 0.000 | 319.359 | 99.000 | 99.000 | 0.000 | 0.000 | 90.000 | 31.784 |
| 11 | 1.505 | -0.701 | -0.701 | 0.102 | 0.000 | 329.865 | 99.000 | 99.000 | 0.000 | 0.000 | 90.000 | 32.569 |
| 12 | 1.926 | -0.940 | -0.940 | 0.046 | 0.000 | 347.259 | 99.000 | 99.000 | 0.000 | 0.000 | 90.000 | 33.801 |
| 13 | 2.147 | -1.067 | -1.067 | 0.013 | 0.000 | 354.481 | 99.000 | 99.000 | 0.000 | 0.000 | 90.000 | 34.290 |
| 14 | 2.681 | -1.385 | -1.385 | -0.088 | 0.000 | 368.453 | 99.000 | 99.000 | 0.000 | 0.000 | 90.000 | 35.200 |
| 15 | 2.796 | -1.459 | -1.459 | -0.123 | 0.000 | 370.076 | 99.000 | 99.000 | 0.000 | 0.000 | 90.000 | 35.303 |
| 16 | 3.040 | -1.612 | -1.612 | -0.185 | 0.000 | 377.682 | 99.000 | 99.000 | 0.000 | 0.000 | 90.000 | 35.777 |
| 17 | 3.275 | -1.763 | -1.763 | -0.251 | 0.000 | 383.226 | 99.000 | 99.000 | 0.000 | 0.000 | 90.000 | 36.115 |
| 18 | 3.623 | -1.990 | -1.990 | -0.356 | 0.000 | 391.969 | 99.000 | 99.000 | 0.000 | 0.000 | 90.000 | 36.635 |
| 19 | 3.836 | -2.134 | -2.134 | -0.431 | 0.000 | 394.115 | 99.000 | 99.000 | 0.000 | 0.000 | 90.000 | 36.760 |
| 20 | 4.066 | -2.284 | -2.284 | -0.502 | 0.000 | 397.539 | 99.000 | 99.000 | 0.000 | 0.000 | 90.000 | 36.959 |
| 21 | 4.294 | -2.435 | -2.435 | -0.576 | 0.000 | 399.598 | 99.000 | 99.000 | 0.000 | 0.000 | 90.000 | 37.077 |
| 22 | 4.736 | -2.731 | -2.731 | -0.726 | 0.000 | 400.410 | 99.000 | 99.000 | 0.000 | 0.000 | 90.000 | 37.123 |
| 23 | 5.197 | -3.043 | -3.043 | -0.889 | 0.000 | 402.735 | 99.000 | 99.000 | 0.000 | 0.000 | 90.000 | 37.255 |
| 24 | 5.528 | -3.272 | -3.272 | -1.016 | 0.000 | 401.943 | 99.000 | 99.000 | 0.000 | 0.000 | 90.000 | 37.211 |
| 25 | 5.679 | -3.374 | -3.374 | -1.069 | 0.000 | 401.954 | 99.000 | 99.000 | 0.000 | 0.000 | 90.000 | 37.211 |
| 26 | 6.509 | -3.930 | -3.930 | -1.351 | 0.000 | 398.452 | 99.000 | 99.000 | 0.000 | 0.000 | 90.000 | 37.011 |
| 27 | 6.676 | -4.042 | -4.042 | -1.408 | 0.000 | 395.823 | 99.000 | 99.000 | 0.000 | 0.000 | 90.000 | 36.860 |
| 28 | 6.794 | -4.116 | -4.116 | -1.439 | 0.000 | 390.541 | 99.000 | 99.000 | 0.000 | 0.000 | 90.000 | 36.551 |
| 29 | 7.065 | -4.296 | -4.296 | -1.527 | 0.000 | 368.339 | 99.000 | 99.000 | 0.000 | 0.000 | 90.000 | 35.192 |
| 30 | 7.422 | -4.494 | -4.494 | -1.566 | 0.000 | 347.776 | 99.000 | 99.000 | 0.000 | 0.000 | 90.000 | 33.836 |
| 31 | 8.479 | -5.033 | -5.033 | -1.586 | 0.000 | 339.222 | 99.000 | 99.000 | 0.000 | 0.000 | 90.000 | 33.242 |
| 32 | 9.623 | -5.606 | -5.606 | -1.588 | 0.000 | 337.768 | 99.000 | 99.000 | 0.000 | 0.000 | 90.000 | 33.139 |
| 32 | 10.117 | -5.852 | -5.852 | -1.586 | 0.000 | 335.849 | 99.000 | 99.000 | 0.000 | 0.000 | 90.000 | 33.002 |

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[^0]:    *Friction angles listed in the table have been corrected for void ratio variation and have been shifted to $\mathrm{I}_{3}=300 \mathrm{kPa}$

